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**FLOW AND HEAT TRANSFER MEASUREMENTS INSIDE A
HEATED MULTIPLE ROTATING CAVITY WITH AXIAL
THROUGHFLOW**

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Submitted for the degree of Doctor of Philosophy

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September 2008

DECLARATION

I hereby declare that this thesis has not been submitted, either in the same or different form, to this or any other University for a degree.

Nicolas D. D. Miché

September 2008

PREFACE

The work reported in this thesis was carried out at the Thermo-Fluid Mechanics Research Centre (TFMRC), a Rolls-Royce University Technology Centre for aero-thermal systems. The experimental program was funded by the European Union and a consortium of European gas turbine manufacturers under the BRITE-EURAM Framework V scheme ICAS-GT2. Inevitably, certain aspects of the rig design and instrumentation, as well as the forming of the testing programme were carried out in collaboration with the author's colleagues in the industry and within the Centre. However, the experiments and evaluation of the results reported in this thesis were carried out by the author, except where indicated in the text.

UNIVERSITY OF SUSSEX

Nicolas D. D. Miché

Doctor of Philosophy

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SUMMARY

This thesis discusses experimental results of measurement of heat transfer and velocity flow in a heated multiple cavity test rig with axial throughflow. Of particular interest are the internal cylindrical cavities formed by adjacent discs and the interaction of these with a central axial throughflow of cooling air. Tests were carried out for a range of non-dimensional parameters representative of gas-turbine high pressure compressor internal air system flows (Re_ϕ up to 5×10^6 and Re_z up to 2×10^5). One configuration of the test rig was tested in the course of the reported study (Build 3) and test data from a previous rig configuration (Build 2) were processed, analysed and compared with the tested data. The most significant difference between the two builds of test rig was the size of the annular gap between the (non-rotating) shaft and the disc bores. Build 3 had a wider annular gap ratio, $d_h/b=0.164$, while Build 2 featured a gap ratio of $d_h/b=0.092$. Heat transfer data were obtained from thermocouples and a conduction analysis. Heat transfer results show differences between the versions of the rig, with the higher Nusselt number values in Build 3 attributed to the wider annular gap allowing more of the throughflow to penetrate into the cavity compared to Build 2. An attempt is made to correlate the average disc Nusselt numbers and this indicates the existence of different regimes. A two-component Laser Doppler Anemometry system was used on both rigs to measure cavity axial and tangential velocity components. Optical access in Build 3 also allowed for measurement of radial velocities. The axial and radial velocities inside the cavities are virtually zero. The size of the annular gap between disc bore and shaft has a significant effect on the radial distribution of tangential velocity. An analysis of the frequency spectrum obtained from the tangential velocity measurements shows evidence of periodicity in the flow consistent with the current understanding of the flow structure in a heated rotating cavity with axial throughflow.

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NOMENCLATURE

| | |
|--|--|
| a, b | inner and outer radii of a disc |
| A | area |
| C_p | specific heat at constant pressure |
| d | diameter |
| d_h | hydraulic diameter of the inlet |
| D | rotor outer diameter |
| f | frequency |
| $G = s/b$ | gap ratio |
| Gr_{sh} | shroud Grashof number |
| Gr_y | Grashof number |
| h | heat transfer coefficient |
| k | thermal conductivity |
| L | appropriate length scale |
| \dot{m} | mass flow rate of the throughflow |
| n | number of subcases |
| N | number of disc or shroud surfaces, or number of LDA fringes |
| $Nu = q r / \Delta T k$ | local Nusselt number |
| $Nu_{av} = q_{av} b / \Delta T_{av} k$ | average Nusselt number |
| $Nu_{sh} = q_{sh} (s/2) / \Delta T_{av} k$ | shroud Nusselt number |
| $Nu_y = q y / \Delta T k$ | Nusselt number based on y |
| p | pressure |
| $Pr = \mu C_p / k$ | Prandtl number |
| q | heat flux |
| r, z, Φ | radial, axial and tangential coordinates |
| r_i, r_o | inner and outer radii for evaluation of average disc heat transfer |
| r_s | shaft radius |
| $Re_z = W d_h / \nu$ | axial Reynolds number |
| $Re_\Phi = \Omega b^2 / \nu$ | rotational Reynolds number |
| $Ro = W / \Omega a$ | Rossby number |

| | |
|--------------------|--|
| s | axial width of cavity, standard deviation |
| Sk | Stokes number |
| t | thickness of disc |
| T | temperature |
| \bar{u} | mean value of velocity component |
| u, U | normal component velocity, standard uncertainty |
| U_α | uncertainty in the quantity α |
| V_r, V_z, V_ϕ | radial, axial and tangential velocity components |
| W | bulk mean axial velocity of the throughflow |
| $x = r/b$ | non dimensional radial coordinate |
| $y = b - r$ | distance radially inward from shroud |
| α | angle |
| $\beta = 1/T$ | volume expansion coefficient |
| δ | spacing or interval |
| ΔT | appropriate temperature difference |
| ε | emissivity |
| θ | angle |
| λ | wavelength |
| μ | dynamic viscosity |
| ν | kinematic viscosity |
| ρ | density |
| σ | R.M.S value |
| τ | characteristic response time |
| ω | angular frequency |
| Ω | rotational speed of the cavity |

Subscripts and superscripts

| | |
|------|--|
| air | value for air |
| av | radially weighted average |
| b | value at $r = b$ |
| c | constant, cut-off value |
| cav | value inside the cavity |
| conv | pertaining to convective heat transfer |
| d | pertaining to the downstream disc of the cavity, |

| | |
|------|--|
| | Doppler |
| f | pertaining to the LDA fringes, fluid |
| fin | pertaining to fin solution |
| g | gravitational acceleration |
| i, o | values on the inside and the outside of the shroud, respectively, increment value |
| in | value at inlet |
| j | increment value |
| p | particle |
| rad | pertaining to radiative heat transfer |
| r | value at radius r |
| ref | appropriate reference value |
| s | denotes a value at the surface |
| sh | pertaining to the shroud |
| T | total |
| t | pertaining to time |
| u | pertaining to the upstream disc of the cavity |
| 95 | 95% interval |

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1. Introduction

Gas turbines are internal combustion heat engines that use the thermal energy rise from a continuous combustion process on a mass flow of air to be converted into useful work such as thrust or shaft mechanical power. Engines operating at higher combustor output temperature also have a higher thermal efficiency. The obstacle to this maximisation of the power output of gas-turbines lies within the resistance of the turbine blade material and nozzle guide vanes to extreme temperatures and combined high tensile stresses induced by the turbine rotation (10000 rev/min for a typical high pressure turbine on a 0.6 m diameter). The use of air-cooled blades in the turbine assembly allows the heat (typically in the order of 1800 K in a modern civil engine such as the Rolls-Royce Trent 1000 featured in Fig1.1) that would compromise the material integrity of the turbine components to dissipate locally.

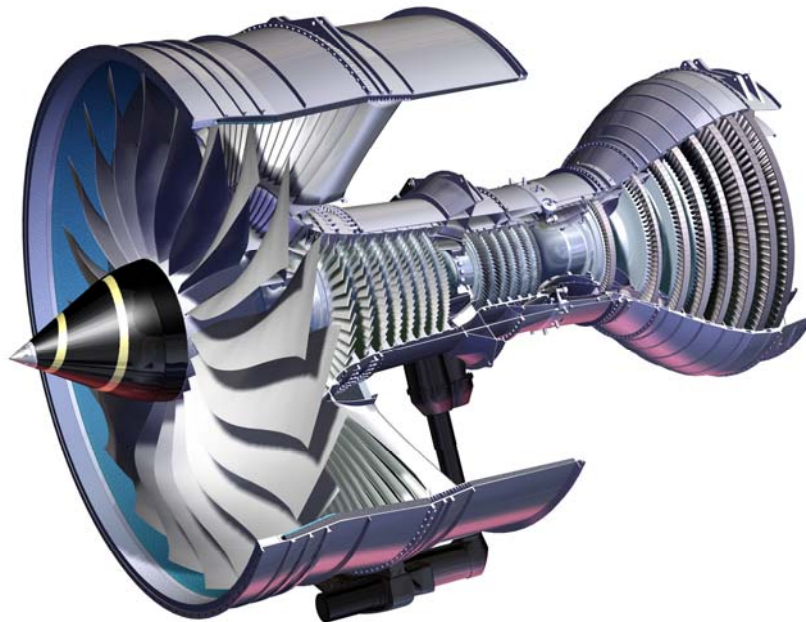


Fig 1.1 Trent 1000 engine cut-away, courtesy of Rolls-Royce plc.

Air cooled turbine blades and nozzle guide vanes are constructed with air passages in their cores being fed by a flow of cooling air allowing convective heat transfer to extract excessive heat from the material to the cooling air.

Part of this cooling air flow can also exit on the blade/vane and form a cooling film of air on the surfaces. Heat conduction from the turbine blade to its supporting disc also requires some cooling of the disc to prevent excessive mechanical growth and fatigue. The cooling air supply to the turbines originates from the internal part of the turbine assembly, away from the main stream of gases, in a zone where the cooling flow will act on the turbine assembly disc and be fed through holes on the blade stem into the blade itself. The pressure of this cooling air should be higher than the main gases pressure at the point in the turbine stages where it would exit in order to ensure positive flow and effective cooling. The mass flow rate of air to be fed to the cooled components, although small compared to the mainstream of gases, is by no means negligible. Gas-turbine manufacturers generally divert part of the airflow from the compressor stages to feed the air cooling system downstream of the combustors. This avoids the need for an external compressor system to power the cooling air path, it saves weight and simplifies the overall design of the gas-turbine.

However, the work carried out to compress the cooling air and the fact that this air does not participate directly in useful power generation are parasitic to the overall cycle of the engine (to the order of up to 6% of the specific fuel consumption in a modern turbofan engine according to Rolls-Royce, 2005). There is therefore a trade-off between thermal efficiency gains from increased turbine inlet temperatures and parasitic losses from cooling air and compression work to be provided for protection of the turbine assembly components from melting and excessive thermal distortion. A direct action in order to reduce the thermal efficiency losses due to the use of cooling air is to minimise the amount of this air required to ensure an effective and reliable use of the gas-turbine. An effective way to reduce the flow rate of cooling air is to deliver the cooling air at the lowest temperature possible in order to increase the cooling, thus using less air for the same cooling effectiveness.

Fig 1.2 shows the typical routes taken by this cooling air across a gas-turbine. This internal air path is referred to as the secondary air system of a gas-turbine because, as explained above, this air does not participate directly in the gas and thrust production.

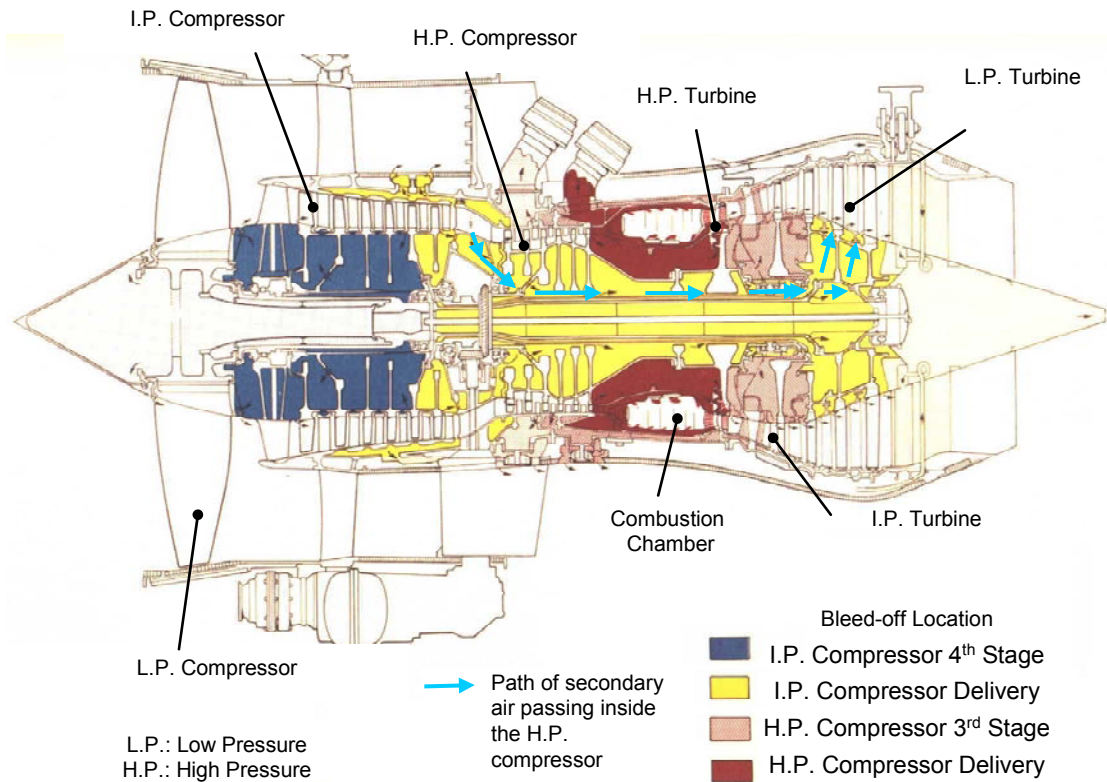


Fig 1.2 A typical gas-turbine aero-engine internal air system (Rolls-Royce plc.)

Of particular relevance to the subject of this thesis, is the air that is extracted from stages of the intermediate pressure compressor and used for the cooling and sealing of the low pressure turbine stages. This air travels from the compressor to the turbine area in the central part of the rotor and drive shafts as shown in Fig 1.3.

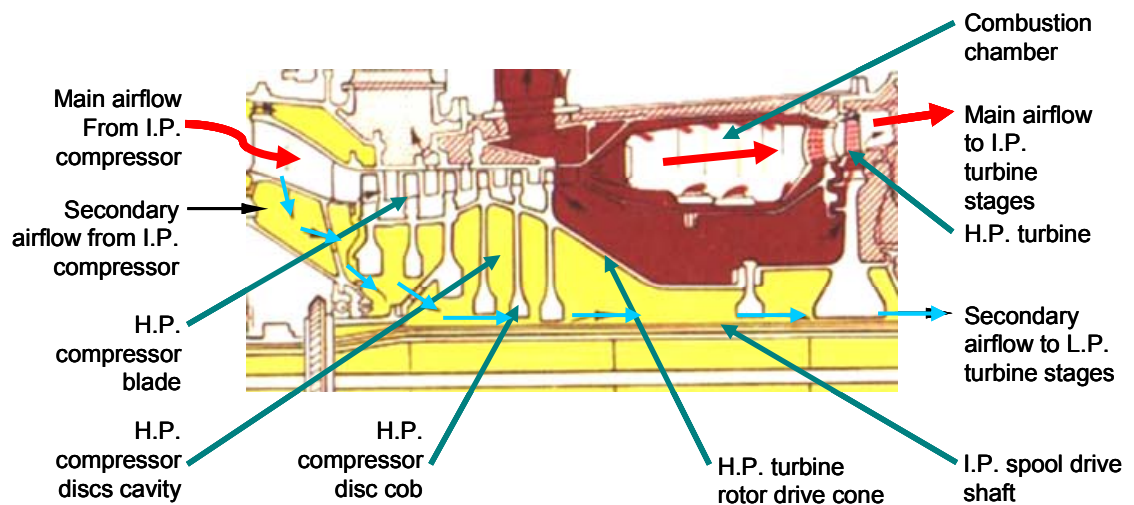


Fig 1.3 Typical layout of a H.P. compressor and turbine stage (Rolls-Royce plc.)

The rotor comprises a succession of narrow annular passages and cavities formed by the disc and hollow drive shafts. All these components also are subject to rotation, sometimes in different relative speeds and directions to each other. Viscous forces acting on the air through these flow obstacles and rotational forces also impair the pressure ratio across the internal air system and hence reduce the cooling air flow on the turbine side. Higher pressures in the compressor stage will also induce more heat rise of the air due to compression. As the cooling air passes through the internal components of the rotor, viscous dissipation ensues from the interaction of the cooling air with the various rotating parts (termed windage). Heat gain or losses are induced depending on the heat conducted through the metal of the components by the actual temperature of the main gas stream in the vicinity (e.g. heat conduction through the drive cone as a result of high temperatures in the combustion chamber). A knowledge of the heat transfer from the internal surfaces of the rotor components to the cooling air in the secondary air system is therefore important for the engine designer in order to predict not only the cooling efficiency of the turbine but also the thermal growth and metal stress and fatigue of the engine components.

The internal air system therefore provides sealing of the gas-turbine and acts on the overall safety and reliability of the gas-turbine. It is therefore of importance to understand the mechanisms involved with the flow characterisation and heat transfer occurring inside the secondary air system. The subject of the current thesis focuses on an investigation of the heat transfer and flows encountered inside the rotating cavities formed by the discs of the high pressure compressor stages of a typical gas-turbine engine. The surface temperatures and flow velocity measurements of the engine components were performed in an experimental apparatus representing the internal components of a high pressure compressor disc stack with a throughflow of air. The conditions of operation such as heating, secondary flow supply and rotational speed were near engine representative conditions.

This study follows on more than twenty years of previous studies, mostly performed at the TFMRC at the University of Sussex.

Notably Farthing et al. (1992 a and b) and Long (1994) established the fundamental and influential parameters governing convective heat transfer in cavities with superposed radial and axial flow on an idealised geometry experimental apparatus. Later, Alexiou (2000) performed experimental studies on engine representative geometry and operating conditions multiple cavity rig. The present work follows on from Alexiou's in a later generation of near engine geometry experimental rig with heat transfer instrumentation and provision for non-invasive airflow velocity measurement within the represented cooling air flow.

This work reported here is the embodiment of a leading edge experimental research programme which has successfully investigated the flows existing in the internal air systems of gas turbine engines. The results from this study have highlighted a dependency on the annular gap between the shaft and the discs on the in-cavity tangential velocity flow field and the discs heat transfer. Radial velocity flow measurements show evidence of periodicity in the flow, consistent with the passage of circulating bodies within the cavities. The shroud heat transfer measurements were consistent with earlier work and showed similarities to natural convection from a horizontal plate. The adoption of precision instrumentation and techniques, including LDA, has facilitated the quantification of highly complex flow structures which expands the understanding of the heat transfer mechanisms which are influenced by such flows. Such improvements are beneficial for the prediction of component temperature and life. The new data is also useful for the validation of CFD codes, where unsteady effects or buoyancy driven flows are inherently difficult to model accurately, which are used in the development of new engines.

In chapter 2, an introduction to the subject of flow and heat transfer in rotating cavities is provided. A description of the current literature is presented detailing the governing parameters and the current understanding of the flow and heat transfer characteristics. Chapter 3 provides a description of the experimental equipment used in this study, its instrumentation, operation and procedures used for the data recording. In chapter 4, a description of how the collected experimental data has been post-processed is presented. Then Chapter 5 will discuss the data analysis and uncertainties. Chapters 6 and 7 will exhibit and

discuss the flow and heat transfer results. Finally the conclusions and recommendations for future work will be provided in Chapter 8.

The work presented in this thesis also resulted in the publication of a journal article (Long et al. 2007) which is reproduced in Appendix 1.

2. Literature review

2.1 Generalities on Rotating Cavity Systems

2.1.1 Gas Turbine Co-rotating Discs Systems

Air delivery in a gas turbine is achieved in the compressor stage by the combined action of successive rows of rotating blades driven by a turbine downstream of the combustion chamber and static blades mounted on the external casing, as shown in Fig 2.1. Relative speed between the rotating and static blades ensures positive pressure rise of the inlet air to feed into the combustion chamber. Typically, rotating blades are individually dove-tail mounted onto the periphery of an assembly disc holding in place a row of blades. Rows of rotating blades are bolted to each other along the direction of the axis of rotation, forming what is referred to as a compressor disc stack and mechanically linked to their driving turbine. The ensemble of discs, turbines, links, and bearings under rotation represent the rotor of the gas turbine.

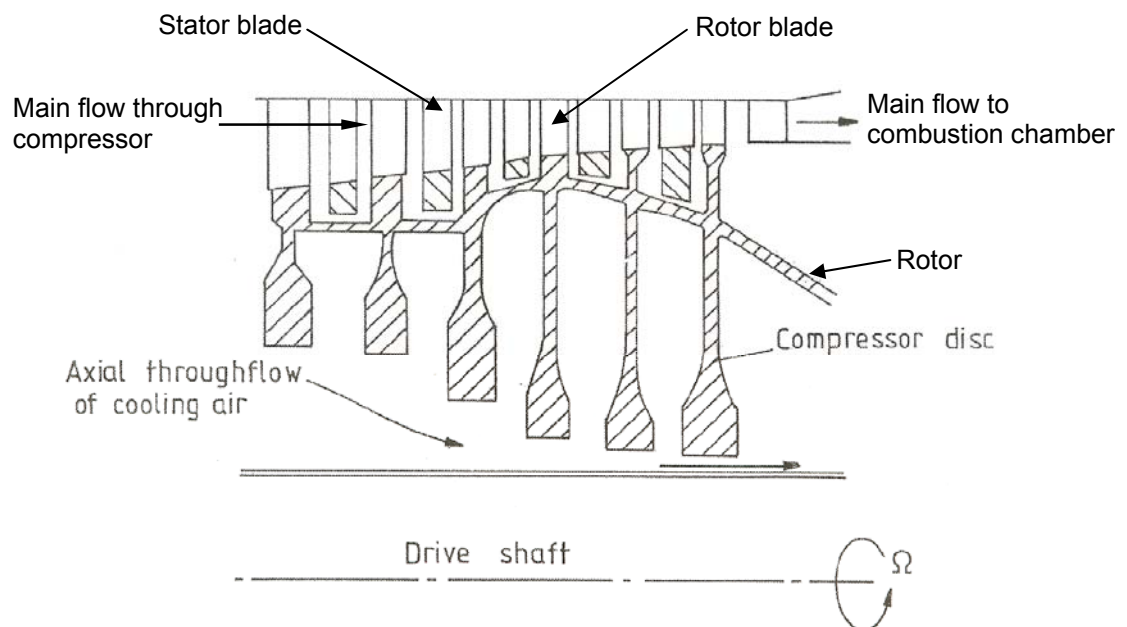


Fig 2. 1 Schematics of the internal layout of a H.P. compressor (Long 1994)

Mechanical damage to the blades while the gas turbine is operating is prevented by leaving a gap between the rotor blade tip and the casing (conversely for the stator blade and rotor). The efficiency of the compressor stages being affected by the blade tip clearance by about 1.5% of the affected stage efficiency for 1% of blade clearance change according to Freeman (1985), it is important to keep this value to a minimum. However, temperature rise of the air across the compressor, typically of 600 °C for a 30:1 compression ratio inevitably conducts heat through the rotor and stator components and affects their geometrical properties by thermal expansion; thus affecting blade tip clearance and overall power efficiency. Moreover hoop stresses on the rotor, thermal stress and high centripetal forces, with the typical magnitude of centrifugal acceleration in the order of 10^4g , justify the diametrical thickness of the discs and the expanded shape at their inner bore referred as the “cob” whose increase in material accommodates the hoop stress concentration of the bore.

In between rows of rotor blades lies a zone where the outer surface of the rotor acts as an end wall to the row of stator blades for axial flow sealing. This part is not mechanically submitted to such levels of mechanical stress as the disc, having no mechanical resistance requirements for securing blades in position and therefore presents the shape of a thin-walled cylinder compared to the disc's geometry. This cylinder, by design, is typically manufactured as part of a disc and is referred to as a shroud. Therefore, the typical geometry of a compressor disc as a whole comprises of a thick-walled cylinder part where the cylinder blades are mounted with material concentration at the core for increased mechanical resistance, and of a thin-walled cylinder part for the compressor stator blades sealing. Series of these two-part discs in a compressor stack will form distinct volumes between two discs and a shroud that are referred to as compressor disc stack cavities.

Rotating cavities can be of several types in gas turbines, depending on their location in the engine.

These cavities are the result of construction design, and their presence could also be put to use to the secondary air system as a point of collection, conveyance, or extraction of cooling air through the engine. The main types of rotating cavities are briefly described below and illustrated in Fig 2.2.

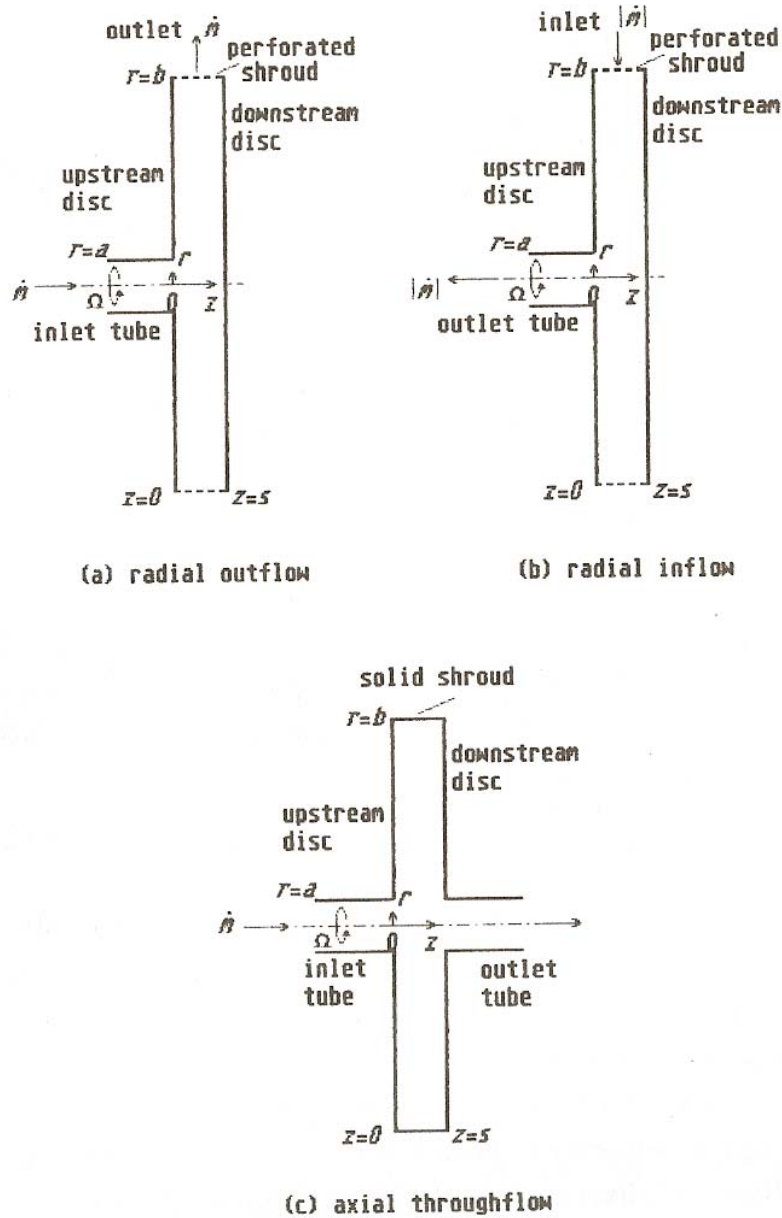


Fig 2. 2 Main types of cavity flow from Owen and Rogers (1995)

Totally enclosed cavities are sometimes found in turbine construction and less commonly in compressors where air flow across the components is generally encouraged as part of the secondary air system.

This configuration presents, however, by its simplicity a good base for modelling and understanding rotating cavity flows and has been studied in the literature as described in section 2.2.

Radial outflow cavities are a type of rotating cavity whereby air is introduced into the cavity near the centre of revolution and extracted through perforations in the shroud, as illustrated in Fig 2.2(a). This type of cavity is generally found in the turbine where it forms part of the turbine cooling air delivery system.

Conversely, radial inflow cavities are formed when the air circulation within the cavity originates from perforations in the shroud periphery and leaves the cavity by the centre of revolution, as shown in Fig 2.2(b). This is the arrangement found in some compressor stages to collect the pressurised air required by the secondary air system.

Both configurations of radial inflow and radial outflow cavities have been extensively described in the research literature and their internal flow and heat transfer mechanisms are now fairly well understood. Although a very interesting subject, this is beyond the scope of the current thesis. Therefore, the author suggests the reader wanting to know more about radially opened cavities to seek more information firstly in chapter 3 and 4 of the book by Owen and Rogers (1995), where a full description of these configurations' characteristics is provided, together with a selection of relevant publications.

Finally, the rotating cavity with axial throughflow configuration refers to when the air enters and leaves the cavity by the central axis. Fig 2.2(c) shows the schematics of a rotating cavity with axial throughflow when now a central shaft is also present. This type of cavity is commonly found within compressor stages of gas turbine engines. This configuration is the main focus of the present study and further description of its flow and heat transfer mechanisms will be found in sections 2.3 and 2.4.

2.1.2 Parameters and Dimensionless Groups

Below is a general nomenclature of the basic parameters pertaining to rotating cavities and particularly to the case of rotating cavities with axial throughflow as explained before.

Let us consider the schematics of a rotating cavity with axial throughflow as shown in Fig 2.3(a).

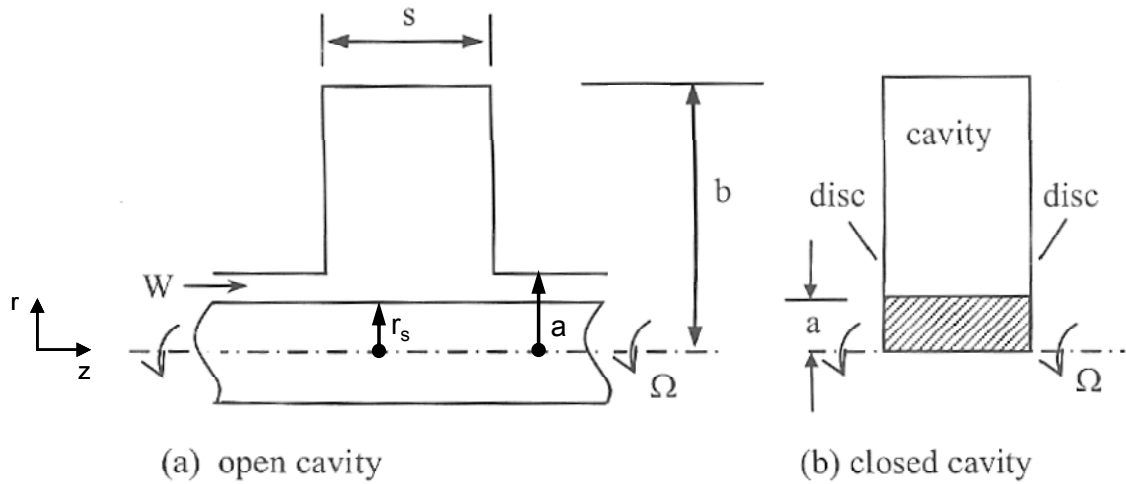


Fig 2. 3 Nomenclature of rotating cavities from King et al. (2005)

Two discs separated from each other by a shroud of axial gap s , rotate around the axis of revolution z at a rotational speed Ω . The outside radius of the discs is b , whereas the inside radius of the discs, or bore, is a . The gap ratio of the cavity G is defined as:

$$G = s/b \quad \text{Eq. 2.1}$$

The cavity generally rotates around a central shaft of diameter r_s , which can be stationary or rotate at a speed Ω_s . The axial throughflow of air runs across the cavity at a bulk average velocity W defined as:

$$W = \frac{\dot{m}}{\rho \times A_{an}} \quad \text{Eq. 2.2}$$

Where \dot{m} = mass flow rate of air delivered through the cavity

ρ = density of inlet air

$$A_{an} = \pi \times (a^2 - r_s^2) = \text{shaft annulus area}$$

The parameter relevant to the rotation of the cavity is the rotational Reynolds number, Re_ϕ defined as:

$$Re_\phi = \frac{\Omega \times b^2}{\nu} \quad \text{Eq. 2.3}$$

Likewise, the axial Reynolds number Re_z characterises the axial throughflow of air as:

$$Re_z = \frac{W \times d_h}{\nu} \quad \text{Eq. 2.4}$$

Where $d_h = 2(a - r_s)$ = hydraulic diameter

Relating the effects of the axial flow and the rotational flow can be expressed as the Rossby number Ro :

$$Ro = \frac{W}{\Omega \times a} \quad \text{Eq. 2.5}$$

This ratio of the axial velocity of the tangential solid body velocity at the disc's bore may also be expressed from equation 2.3 and equation 2.4 as:

$$Ro = \frac{b^2 \times Re_z}{2a \times (a - r_s) \times Re_\phi} \quad \text{Eq. 2.6}$$

The Grashof number Gr is a parameter relevant to buoyancy induced flow. In a rotating cavity application the expression of Grashof number based on the radial distance along the disc surface is based on the local surface temperature and its relative centripetal acceleration was expressed by Farthing et al. (1992b) by taking y as a height reference from the shroud as:

$$Gr_y = \frac{\beta \Delta T \times \Omega^2 r \times y^3}{\nu^2} \quad \text{Eq. 2.7}$$

Where $\beta = 1/T_{in}$ is the volume expansion coefficient with T_{in} the inlet temperature of the throughflow air. ΔT is the temperature difference between the supply air and the local surface temperature at the radius r considered, $\Delta T = (T_s - T_{in})$.

Also, $y = (b - r)$ is the distance radially inward from the shroud which is taken as the length parameter for this instance of Grashof number, with its value increasing against the gravitational field direction.

The Prandtl number is a ratio of the kinematic viscosity to the thermal diffusivity at a given point.

$$Pr = \frac{\nu}{\alpha} \quad \text{Eq. 2.8}$$

$$\text{Where } \alpha = \frac{k}{\rho \times C_p}$$

The Prandtl number is considered as the characteristic number for thermal boundary layers and heat transfer in forced convection. The kinematic viscosity (ν) is in relation with the transport properties of the fluid in respect to momentum, and is related to the thickness of the velocity boundary layer δ (non-slip condition). An increase of viscosity meaning an increase of δ . Similarly, the thickness of the thermal boundary layer is in relation with the thermal diffusivity. Hence, Schlichting and Gersten (2000) remark that the Prandtl number for forced convection is a direct measure of the ratio of the two boundary layers.

The Rayleigh number Ra is a parameter related to the heat transfer within the fluid. It is expressed as the product of the Grashof number and the Prandtl number.

$$Ra = Gr \times Pr \quad \text{Eq. 2.9}$$

The Nusselt number Nu is the non dimensional parameter related to the heat flux on a surface. For the case of a heat flux on a cavity disc, the Nusselt number is expressed as.

$$Nu_y = \frac{q \cdot y}{k \cdot \Delta T} \quad \text{Eq. 2.10}$$

Where q is the heat flux from the surface of the disc to the cavity fluid at location y

Following is a review of previous work related to flow and heat transfer in the sealed cavity configuration and an introduction on buoyancy induced flow.

2.2 Sealed cavities

2.2.1 Isothermal

A sealed cavity refers to the configuration shown in Fig 2.3(b) whereby the enclosure formed by the two disks is bounded on the disks' outer radius by the shroud and also at the inner diameter of the disks, if any, forming a cavity in which no air either enters or leaves. The cavity rotates about its axis of revolution and the trapped air within is accelerated as a result of friction with the cavity walls. When no temperature difference across the cavity and its walls exists (isothermal condition) the fluid within the cavity fluid is only subjected to mechanical forces and will end-up rotating at the same speed as the discs, which is referred as solid body rotation.

2.2.2 Axial heat flow

A sealed rotating cavity is subjected to axial heat flow when one of the discs is hotter than its opposite and the inner and outer cylindrical surfaces are considered adiabatic. The hotter disc is referred to as the hot disc and the cooler one as the cold disc. The lower cavity air density by the hot disc side induces greater buoyancy forces (directed against the centrifugal acceleration) than on the cold disc side. The circulation flow resulting from this buoyancy forces imbalance to the cavity air will be such as it will circulate centripetally oriented by the hot disc and centrifugally oriented by the cold disc, as illustrated by Fig 2.4.

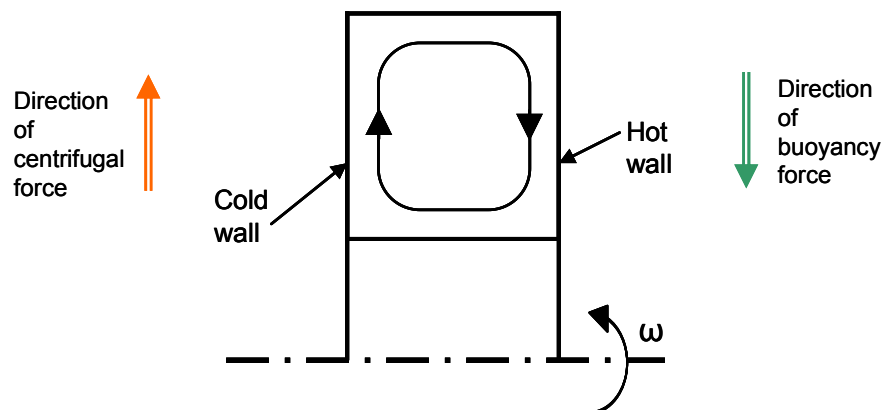


Fig 2. 4 Sealed rotating cavity with axial heat flow

Bohn et al. (1994) performed computations for a rotating sealed annulus with axial flow of heat. The temperature distributions on the discs were uniform and the inner and outer cylindrical surfaces were considered adiabatic. A fixed Prandtl number and up to 300 K temperature difference between the hot and cold discs were considered in the computations. Their results showed that the fluid within the cavities was circulating around the walls in the boundary layers, with nearly no motion occurring in the core region. The sudden transportation of a fluid element from the outer surface towards the inner radii from the buoyancy force will see its relative circumferential speed increasing from the shroud tangential velocity as the radius decreases (minus the reduction in the absolute speed due to viscous friction). In the solid body rotation frame of speed reference, this increase of tangential velocity will show the element to be spiralling towards the centre at increasing relative tangential speed, thus highlighting an outwards Coriolis force seen to balance against the centripetal buoyancy force (Fig 2.5). Similarly the Coriolis force will be seen to work against the radially-outwards circulating flow as shown in Fig 2.6.

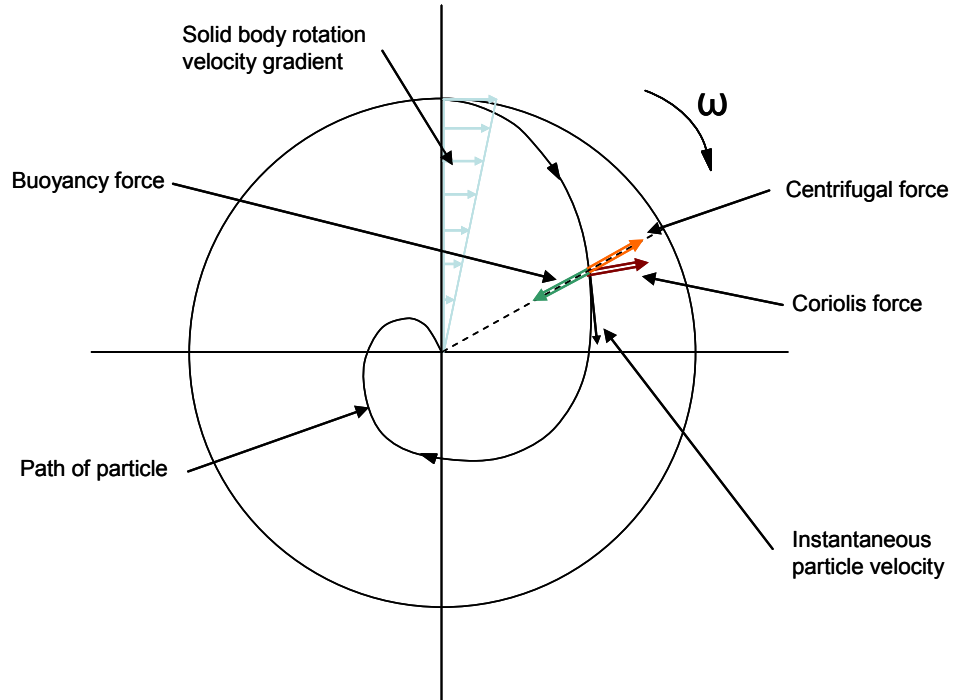


Fig 2. 5 Path of a particle driven inwards of cavity as result of buoyancy seen in rotating frame of reference with directions of acting forces upon it

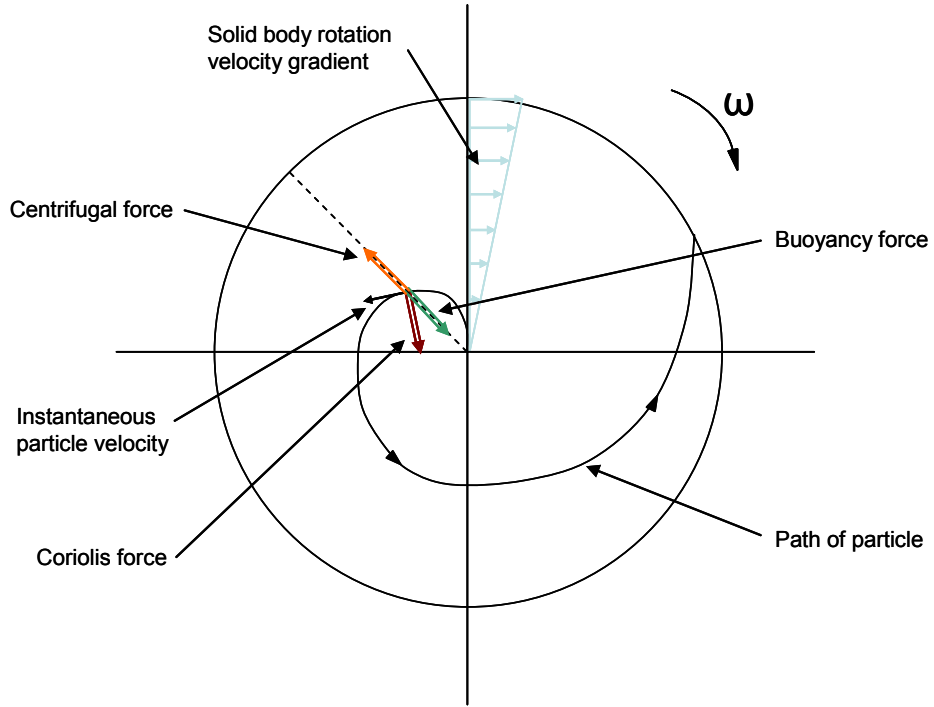


Fig 2. 6 Path of a particle driven outwards of cavity as result of centrifugal force seen in rotating frame of reference with directions of acting forces upon it

Thus Bohn et al. (1994) affirmed that the fluid motion in a rotating annulus with axial flow of heat is driven by buoyancy force and damped by the Coriolis effect. As a result their computations showed that an increase of the rotational Reynolds number at constant Grashof number reduced the flow circulation within the cavity (and consequently the heat transfer) from the damping effect of the Coriolis effect increasing with rotational speed. This can be seen in Fig 2.7, where:

$$Nu = \frac{q_{av} \times s}{k_{ref} \times (T_H - T_C)} \quad \text{Eq. 2.11}$$

Where T_H = hot disc temperature

T_C = cold disc temperature

$$Re_\phi = \frac{\Omega \times r_m \times s}{v_{ref}} \quad \text{Eq. 2.12}$$

Where r_m = average radius

$$Gr = \frac{\Omega^2 \times r_m \times s^3}{v_{ref}^2} \times \beta (T_H - T_C) \quad \text{Eq. 2.13}$$

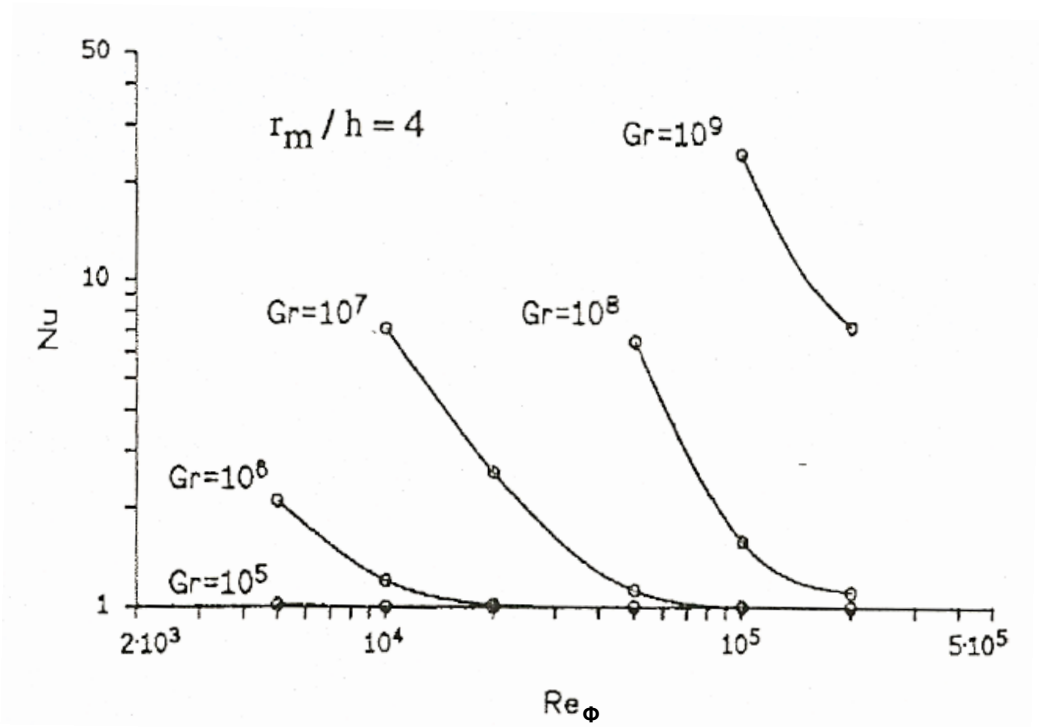


Fig 2. 7 Nu vs Re_ϕ for various Gr in an enclosed cavity from Bohn et al. (1994)

In Fig 2.7 it can be seen that an increase of Re_ϕ at constant Gr attenuates the heat transfer as it tends towards $Nu \rightarrow 1$ (pure conduction) at higher Re_ϕ values. Also Bohn et al. (1994) obtained heat transfer correlation for their stationary laminar model on experimental data up to values of $Ra=10^{11}$ which is significantly higher a value than the normally considered $Ra=10^9$ transition value from laminar to turbulent free convection on a stationary vertical plate. Later, Bohn and Gier (1998) confirmed by a combined computational and experimental study the dependence of the cavity heat transfer on Re_ϕ . Their experimental measurements were correlated for an air filled rotating annulus with axial flow of heat for $G=0.5$, $a/b=0.52$ and $2 \times 10^8 < Ra < 5 \times 10^{10}$ by:

$$Nu = 0.346 \times Ra^{0.124}$$

Eq. 2.14

2.2.3 Sealed cavity with radial flow of heat

A sealed cavity is subjected to a radial flow of heat when the annulus presents a temperature difference between the shroud and the inner cylinder walls (the discs being considered as adiabatic).

Bohn et al. (1995) conducted an experimental study on a sealed rotating annulus for the case of a radial heat flow where the shroud surface is hot and the inner cylindrical surface is cooler. The two discs were insulated. For conditions of $10^7 < Ra < 10^{12}$ the Nusselt numbers were correlated.

For a cavity with $G=0.34$ and $a/b=0.35$:

$$Nu = 0.246 \times Ra^{0.228} \quad \text{Eq. 2.15}$$

For a cavity with $G=0.5$ and $a/b=0.52$

$$Nu = 0.317 \times Ra^{0.213} \quad \text{Eq. 2.16}$$

$$\text{Where } Nu = \frac{q_{av} \times b \times \ln(b/a)}{k(T_H - T_C)}$$

$$Ra = Gr \cdot Pr = \frac{r_m \cdot \Omega^2 \cdot \Delta T \cdot (b-a)^3}{1/2 \cdot (T_H + T_C) \cdot \nu^2} \times \frac{\mu \cdot C_p}{k} \quad \text{Eq. 2.17}$$

$$Re = \frac{\Omega \cdot r_m \cdot (b-a)}{\nu} \quad \text{Eq. 2.18}$$

$$\text{With } r_m = \frac{(a+b)}{2} \quad \text{the mean annulus radius}$$

As seen previously, in the case of a sealed rotating annulus with axial flow of heat the cavity air was being driven into a tumbling motion going radially outwards on the hot disc side as a combined effect of buoyancy and centrifugal acceleration. In the typical case of radial heat flow where the shroud is hotter than the inner cylindrical wall of the annulus however, the temperature gradient generates centripetal buoyancy forces across the cavity length. As a result the flow can only circulate from the onset of flow instabilities allowing re-circulating flows oriented in the axis of rotation of the annulus to appear as illustrated by Fig 2.8.

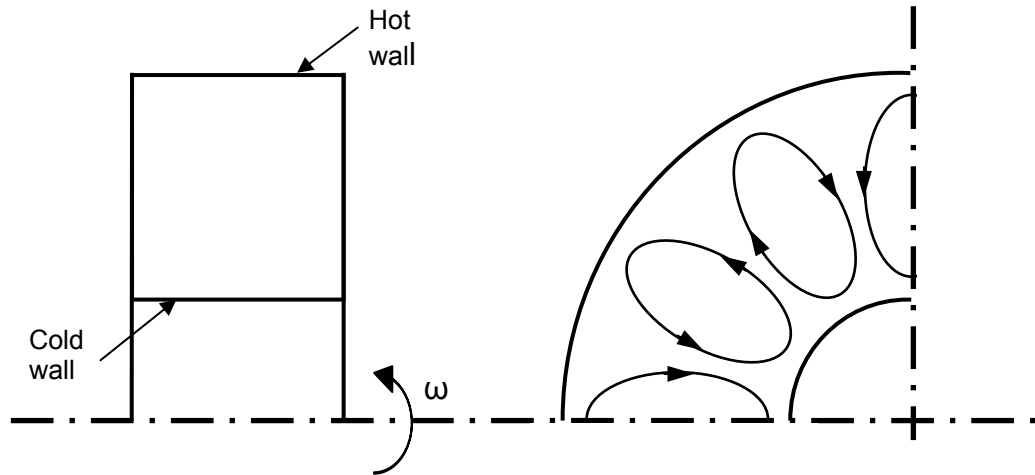


Fig 2. 8 Sealed rotating cavity with radial flow of heat

This flow behaviour is reminiscent of the Rayleigh-Bénard convection flow between two plates with a temperature gradient in the opposite direction to the gravitational force. Fig 2.9 shows the Rayleigh-Bénard convection flows in a horizontal stationary enclosure.

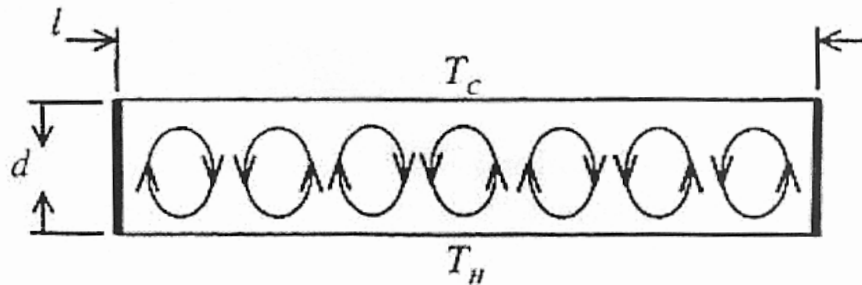


Fig 2. 9 Rayleigh-Bénard convection flow between two plates from King et al. (2005)

King et al. (2005)'s flow computations showed pairs of cyclonic and anti cyclonic vortices were created in the tangential plane of an inwardly heated sealed rotating cavity for Rayleigh number values up to $Ra=10^9$. Fig 2.10 shows their computed velocity contours and temperature isotherms. They remarked that the flow was in state of vacillation as the number of vortices pairs varied with time. The 3-D computations highlighted axial components of velocities associated to the vortices.

Their presence is linked to the cyclonic and anti-cyclonic nature of these vortices and show axial velocities away from the discs for a cyclonic vortex (low pressure) and towards the discs for an anti-cyclonic vortex as shown in Fig 2.10.

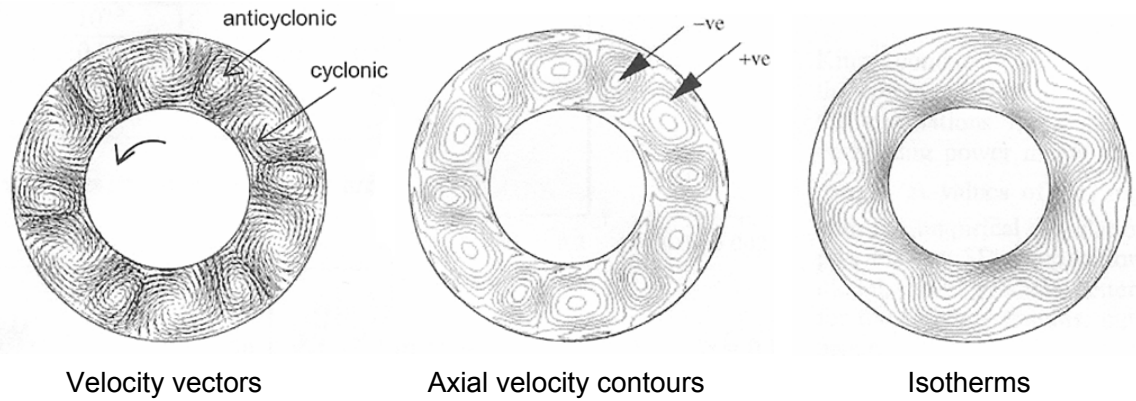


Fig 2. 10 Sealed cavity with radial flow of heat computations from King et al. (2005) for $Re_\phi=546$

The Nusselt number computations of King et al. were in good agreement with heat transfer from Rayleigh-Bénard convection in stationary enclosures but were significantly higher than the empirical correlations for a closed rotating annulus.

In summary it can be noted that the buoyancy forces influencing the flow structure within a sealed rotating annulus submitted to an inwardly radial flow of heat can induce time dependent three-dimensional pairs of cyclonic and anti-cyclonic vortices similar to Rayleigh-Bénard convection in a horizontal plate enclosure.

2.3 Rotating cavity with axial throughflow

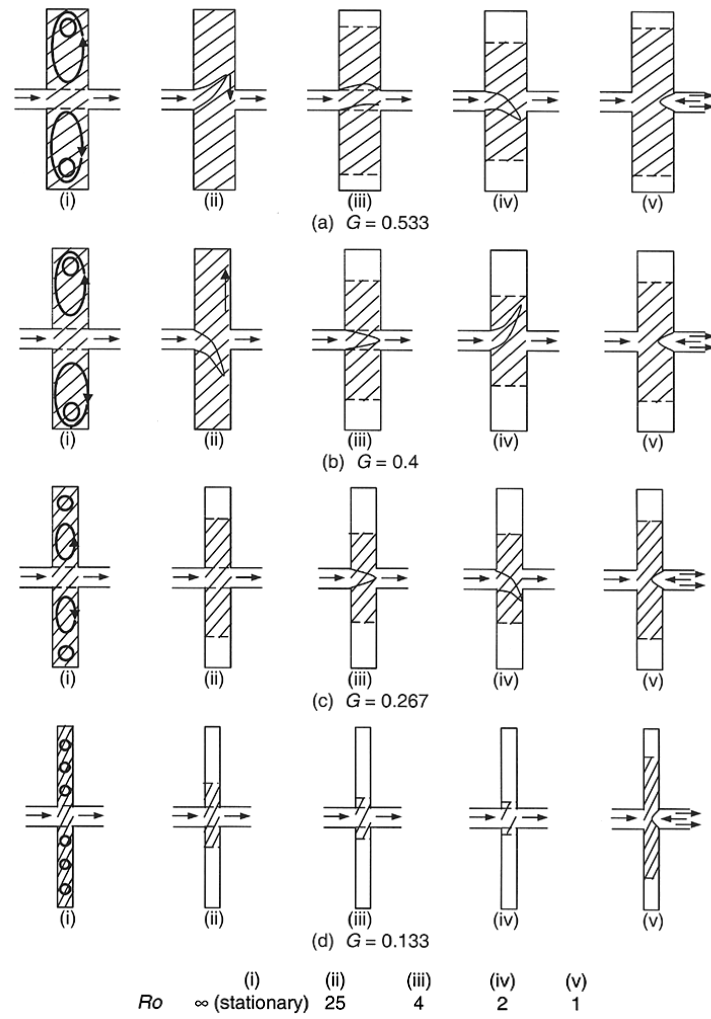
2.3.1 Isothermal throughflow

Farthing et al. (1992a) reported on their flow observations and measurements in a cavity under isothermal throughflow conditions. They used Laser Doppler Anemometry (LDA) in an experimental set-up reproducing a rotating cavity without central shaft which gap ratio G was varied from 0.133 to 0.533 and had a radius ratio $a/b \approx 0.1$. For a non-rotating condition ($Ro \rightarrow \infty$), the interaction of the throughflow of air with the cavity air was found to generate one or more toroidal vortices in the cavity with virtually no penetration of the throughflow into the cavity. The typical flow schematics for the case of throughflow of air with no rotation of the cavity for different values of G can be seen in the cases indexed as (i) in Fig 2.11.

At lower gap ratios ($G=0.267$ and $G=0.133$) Farthing et al. observed multiple contra-rotating vortices inside the cavity. The increase of rotational speed had the initial observed effect of suppressing the toroidal vortex in the cavity, plausibly as the result of vortex motion damping caused by generated Coriolis forces from the tumbling air motion driven by the interaction with the central throughflow. The central throughflow itself is also affected by rotational speed that would give a tendency to centrifuge the central flow out; and axi-symmetric and non axi-symmetric vortex breakdown were observed. The type of vortex breakdown and radial extent to which the throughflow penetrates the cavity was found to depend principally on the values of G and Ro , and that the transition from one type of vortex breakdown to another was subject to hysteresis.

For a case with $G=0.533$ in turbulent flow ($Re_z > 2000$), Farthing et al. observed no discernible signs of influence on the vortex and the central jet with rotational speed for values of Rossby number higher than 100.

For $100 > Ro > 21$, non axi-symmetric vortex breakdown where the jet of air gets diverted into the cavity could be observed. This is sketched in Fig 2.11 as condition (b). This mode of vortex breakdown was referred to as mode 1a shown in Fig 2.12.



(Shaded areas represent regions into which smoke is convected rapidly.)

Fig 2. 11 Visual impressions of smoke patterns in an isothermal rotating cavity with axial throughflow: $Re_z=5000$ (Farthing et al. 1992a)

For $21 > Ro > 2.6$, as the rotational speed increased, the jet would become axisymmetric again but with occasional oscillations (mode IIa in Fig 2.12, illustrated as case c in Fig 2.11).

$2.6 > Ro > 1.5$ showed a “flickering flame” appearance of the central jet with occasional excursions into the cavity.

For Rossby values below $Ro=1.5$ the flow turned axis-symmetric again and signs of reversed flow at the downstream edge of the jet was observed (case e in Fig 2.11).

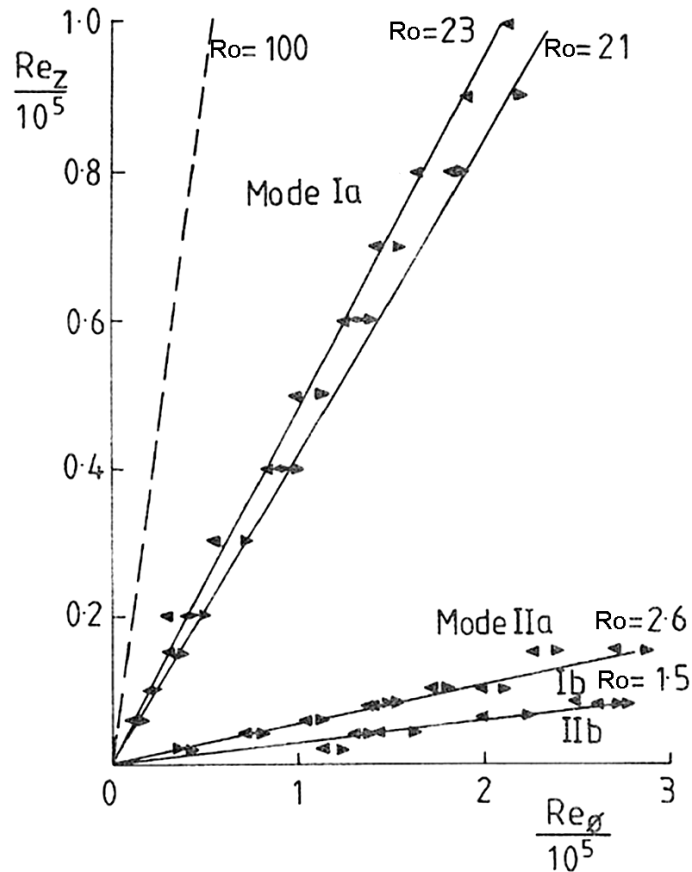


Fig 2. 12 Vortex breakdown modes in an isothermal rotating cavity according to Ro from Farthing et al. (1992a)

Observations in cavities with smaller gap ratio showed a weakening of the strength of the cavity vortex and vortex breakdown as G reduced.

These observations were made in a cavity without a central shaft. The presence of a central shaft within the throughflow is likely to act as a flow straightener to the central jet and therefore further reduce the effects of vortex breakdown.

Farthing et al.'s LDA measurements were performed for a fixed $Re_z = 8 \times 10^4$, varying Ro and G in the radial and tangential velocity components. The effects of Ro and G on the time averaged radial velocity across the cavity at $r/b=0.8$ is shown in Fig 2.13.

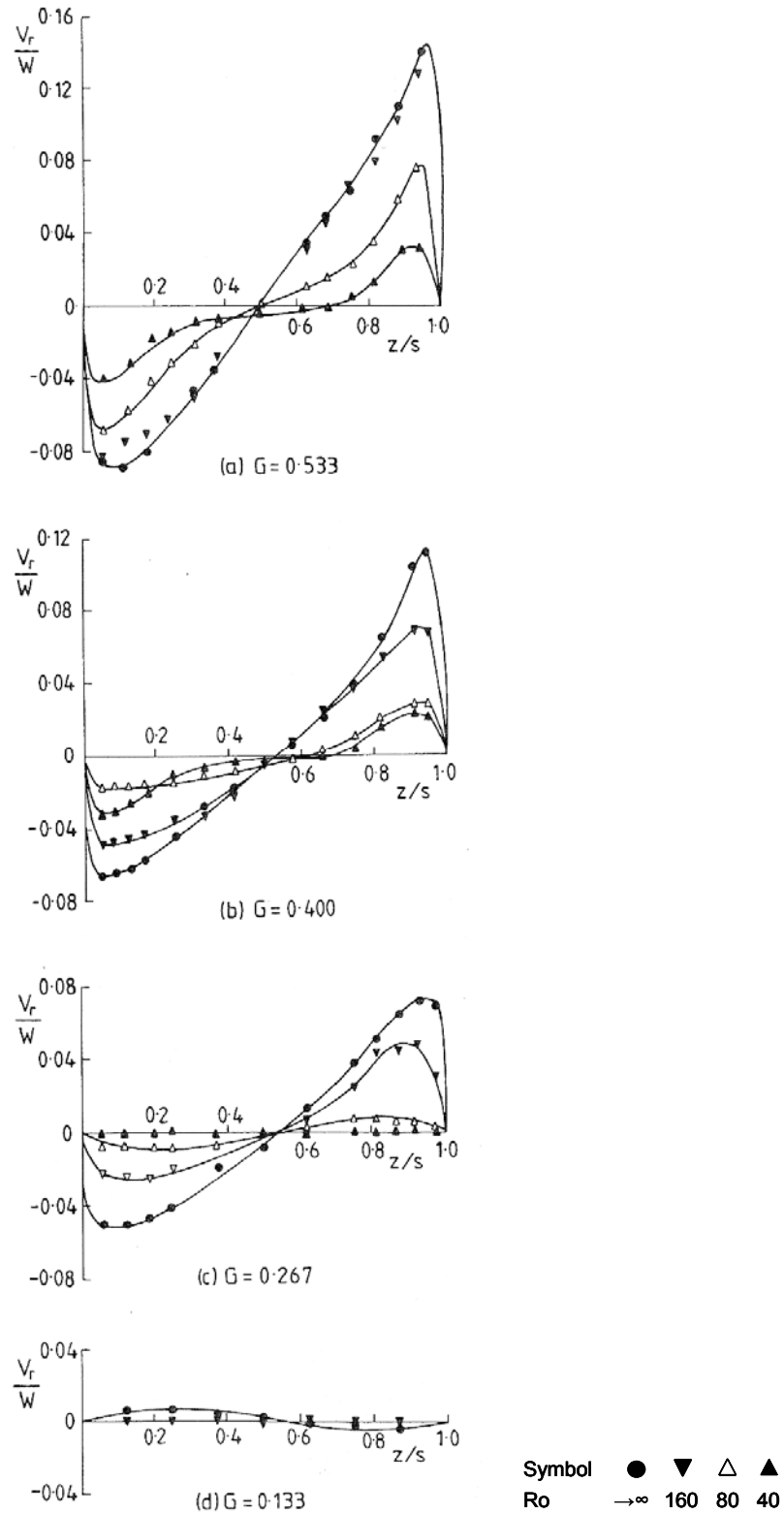


Fig 2. 13 Effects of Ro and G on radial velocities across an isothermal rotating cavity from Farthing et al. 1992a

Figure 2.13 shows, as seen before, that an increase in rotational speed or a reduction in gap ratio has the effect of weakening the toroidal vortex.

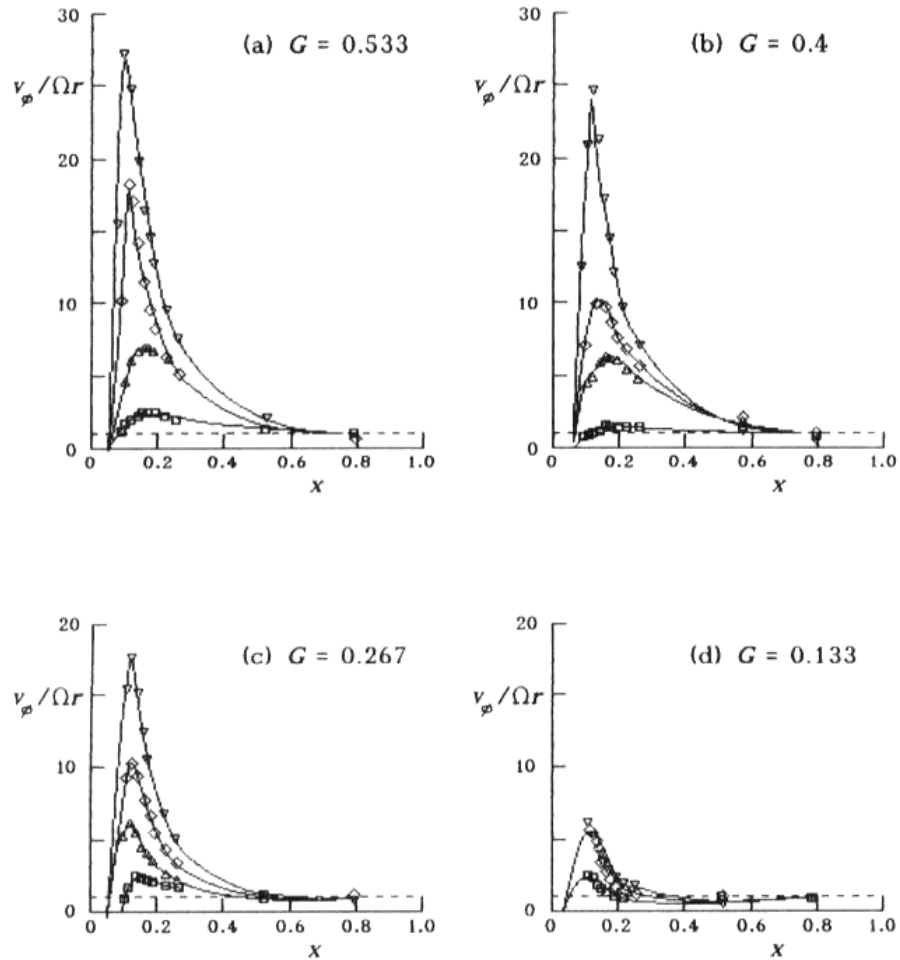
For $G=0.533$, $G=0.4$ and $G=0.267$, the radial flow is consistent with a toroidal vortex driven by the central throughflow, with radial outflow by the downstream disc, radial inflow by the upstream side of the cavity and a quiescent point of rotation by mid-cavity. For $G=0.133$ they observed a reverse flow consistent with a condition when contra-rotating vortices are present in the cavity; and the weakness of this circulation as a result of small gap ratio was further highlighted by the fact that at $Ro \leq 160$ zero velocity radial flow indicated stratification of the cavity flow in this outer region.

Tangential velocity flow measurements were also reported by Farthing et al. measured in the mid plane of cavity ($z/s=0.5$) for different gap ratio and Ro along the radial span of the cavity.

Fig 2.14 shows the effect of Ro and G on the normalized tangential velocity with Ωr for $Re_z = 4 \times 10^4$.

They observed that for all conditions, the outer part of the cavity air was at solid body rotation velocity and that the size for this zone was increasing with decreasing G . A free vortex flow where $V_\phi/\Omega r \gg 1$ is created by the centre of the cavity is said to be created by the inward coming part of the toroidal vortex by the upstream disc which turns to conserve its angular momentum.

As the toroidal vortex strength is weakened by decreasing Ro and G , so will the central vortex swirl be affected. The effect of z/s on the tangential velocities was found to be small in the zone outside the boundary layers of the discs.



Effect of Ro_z and G on variation with x of tangential component of velocity in an isothermal rotating cavity with axial throughflow: $Re_z = 4 \times 10^4$, $z/s = 0.5$.

| Symbol | ∇ | \diamond | \triangle | \square |
|--------|---|------------|-------------|-----------|
| Ro_z | 160 | 80 | 40 | 10 |
| ----- | solid-body rotation ($v_\phi / \Omega r = 1$) | | | |

Fig 2. 14 Effects of Ro and G on the tangential velocity components across an isothermal rotating cavity from Farthing et al. 1992a

2.3.2 Non-isothermal throughflow

Farthing et al. (1992a) used the same rig as described in 2.3.1 to produce flow visualisation inside a rotating cavity where the discs could be heated independently to produce temperature distributions that could increase, decrease or remain approximately constant with increasing radius. The shroud

was unheated. For $G=0.27$, $2.2 \times 10^3 \leq Re_z \leq 7.7 \times 10^3$, $1.3 \times 10^4 \leq Re_\phi \leq 8.7 \times 10^4$ they observed for all temperature distributions similar on both discs that the flow inside the cavity was seen to be non axis-symmetric. The throughflow was seen to enter the cavity in a “radial arm” and was then divided by the outer radii and tangentially transported around two contra-rotating re-circulation vortices towards a “dead zone” where a more diffusive inwards motion of the air would be seen. Fig 2.15 illustrates schematically their observations.

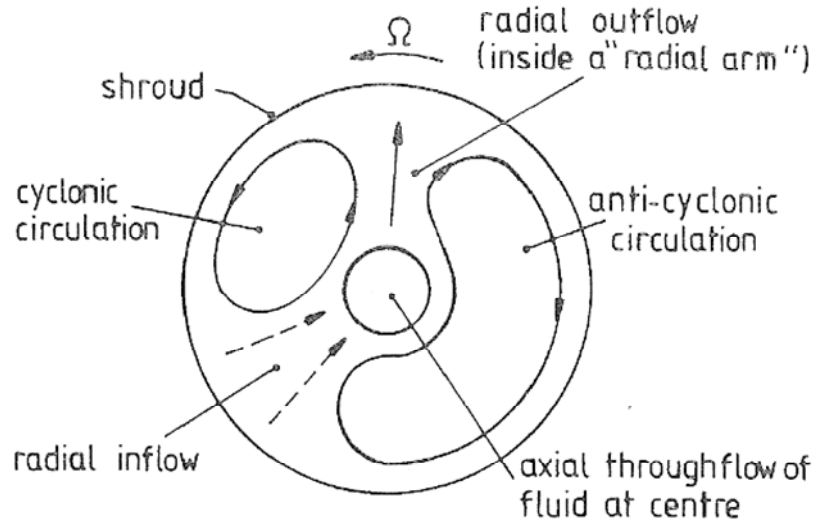


Fig 2. 15 Schematic of flow structure from Farthing et al. (1992a)

The flow ensemble was found to rotate at a speed ω slightly inferior to solid body rotation in the order of $0.9 \leq \omega/\Omega \leq 1$ which decreases with increasing G and temperature difference between max disc temperature and inlet air temperature. The relative speed of the circulation system was also found to reach a minimum at $Ro \approx 2$. For a symmetrically heated cavity (at $G=0.138$) with the temperature of the discs increasing with radius, Farthing et al. correlated the measured local Nusselt numbers on the discs by:

$$Nu_y = 0.0054 \times Re_z^{0.3} \times Gr_y^{0.25}$$

Eq. 2. 19

The 0.25 exponent of Gr_y in equation 2.19 suggests laminar free convection occurring in the narrow gap cavity by relation to the correlations for free convection for a stationary vertical plate for:

-laminar flow

$$Nu = 0.36 \times Gr^{0.25} \quad \text{Eq. 2. 20}$$

-turbulent flow

$$Nu = 0.022 \times Gr^{0.4} \quad \text{Eq. 2. 21}$$

Having the downstream disc heated while the upstream was not, showed broadly similar flows but unsymmetrical about the axial plane. A similar improvement in flow stability than in the case of an enclosed cavity with radial outflow of heat was seen in the axial throughflow condition with discs temperature decreasing with radius. Conversely, having a temperature profile increasing with radius would set an unstable density distribution like in the case of a closed cavity with radially inward heat flow.

Long (1994) used the same experimental rig geometry as described by Farthing et al. but where the disks and shroud could be symmetrically heated with the discs' temperature profiles increasing with radius and gap ratios of $G=0.36$ or $G=0.13$. He estimated the proportion of axial throughflow entering the cavity from a heat balance calculation using measured values of average disc heat transfer and cavity temperature. For $Ro < 1$, approximately 50% of the throughflow of air was estimated to be diverted into the cavity. For $Ro > 10$, this decreased to about 10%. These values were found to be broadly independent of gap ratio.

For conditions of $Ro > 4$, at $G=0.36$ the heat transfer was found to be increasing by several times compared to the case at $G=0.13$. Long concluded that the rotationally induced buoyancy and direct influence of the central throughflow are the two mechanisms responsible for the heat transfer inside a heated rotating cavity with axial throughflow.

Burkhardt et al. (1992) carried out heat transfer measurements on an engine geometry and running conditions representative five cavity rig involving measured surface temperatures and a conduction solution method. A central shaft was present and could rotate in either direction of the rotor. The

experimental conditions were in the range $2.7 \times 10^4 \leq Re_z \leq 9.5 \times 10^4$ and $1.9 \times 10^6 \leq Re_\phi \leq 5.6 \times 10^6$. They observed that under identical tests conditions the heat transfer for an upstream disc tended to be greater than that of its downstream disc. The temperatures of the discs tended to decrease as the shaft rotational speed approached the discs speed, implying higher heat transfer on the discs at this condition.

Alexiou et al. (2000) carried out engine representative heat transfer measurements on an experimental rig representing the last two cavities of the high-pressure compressor including the turbine drive cone on a scaled down version of a typical engine geometry. A central shaft could be stationary or either co-rotate or contra-rotate with the discs. Up to 85% of the throughflow could also be diverted away through peripheral holes located on the downstream side of the shaft. Surface temperatures of the last disc and cone were measured and a conduction solution method was applied in order to derive heat fluxes. The disc was instrumented on one face only, therefore a symmetrical temperature distribution across the disc thickness was assumed. Farthing et al. (1992b), Burkhardt et al. (1993) and Long et al. (1997) previously used this assumption on relatively thin discs experiencing a relatively large radial temperature gradient. Hence the name “fin solution” sometimes used for the description of an adiabatic plane assumed conduction solution on a thin disc.

Two regimes of heat transfer were revealed from the heat transfer measurements on the inner surface of the cone: a rotationally dominated and a throughflow dominated regime of heat transfer.

The rotationally dominated regime is when the heat transfer occurs by free convection induced by flow stratification on the outer part of the cone cavity with rotationally induced buoyancy forces. The local Nusselt numbers were correlated for $Ro < 3.5$ by:

$$Nu = 0.0243 \times Re_z^{0.086} \times \{Re_\phi^2 \times \beta \Delta T\}^{0.326} \times X^{-1.89} \times \{X^{-1} - 1\}^{-0.082} \quad \text{Eq. 2. 22}$$

Where $X = r/b$, is the non dimensional radial coordinate

The throughflow dominated regime on the cone heat transfer was suggested to be the result of a large vortex filling the cavity. The local Nusselt numbers were correlated for $Ro > 3.5$ by:

$$Nu = 8.93 \times 10^{-5} \times Re_z^{1.301} \times X^{-3.523} \quad \text{Eq. 2. 23}$$

Where $X = r/b$, is the non-dimensional radial coordinate

It was reported by Alexiou et al. (2000) that in the rotationally dominated regime higher heat transfer on the cone inner surface was found when the shaft was co-rotating with the cone. The disc heat transfer was however found to be relatively insensitive to the sense of rotation of the shaft and was found to be significantly lower than the heat transfer at the inner surface of the cone.

Long et al. (2003) extended their investigation on the previously cited rig to the heat transfer through the cavity shroud area. A variation of this engine representative multiple cavity rig was also used featuring four cylindrical cavities, a stationary shaft and no drive cone. This rig was referred to as the Multiple Cavity Rig, Build 2. Shaft and disc bore diameters remained identical to Build 1 used by Alexiou (2000). Figure 2.16 outlines the major geometrical differences between the different builds of the Multiple Cavity Rig.

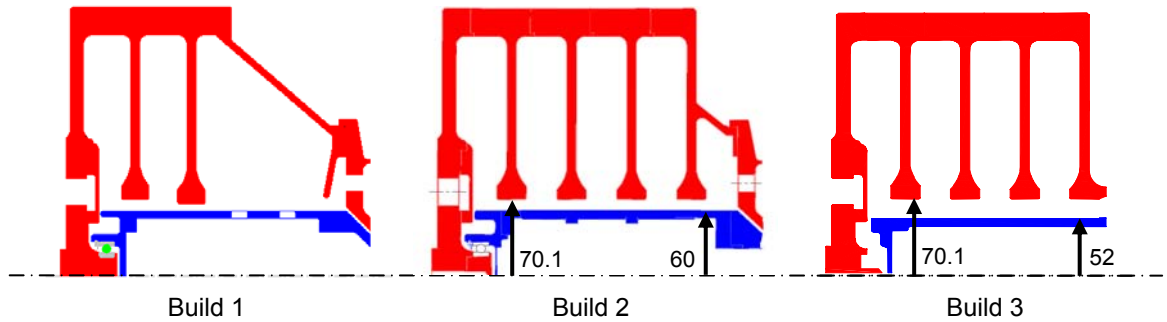


Fig 2. 16 Outline views of the Multiple Cavity Rig build 1, 2 and 3

Surface temperature measurements on the inner and outer surface of the shroud allowed heat flux calculations assuming one-dimensional radial conduction through the shroud. A total relative temperature of the cavity air by the shroud at $r=b$ was estimated as:

$$T_{T,rel,b} = T_{in} + \frac{\Omega^2 \times (b^2 - a^2)}{2 \times Cp} \quad \text{Eq. 2. 24}$$

This was then used in order to derive Nusselt numbers at the shroud's inner face. The characteristic length in the calculation of the shroud Nusselt and Grashof numbers was the ratio of the shroud area to the perimeter (resulting as $s/2$) consistent with the definitions used for correlations of free convection from a heated horizontal plate. The measurements were carried out in the range of $4 \times 10^4 \leq Re_z \leq 1.7 \times 10^5$ and $1.6 \times 10^8 \leq Gr_{sh} \leq 4 \times 10^9$. Experiments showed that the shroud heat transfer was governed by rotationally induced free convection and was increasing with Grashof number. The shroud Nusselt numbers showed similarities to correlations for free convection on a horizontal plate with a gravitational field; with a transition to turbulent free convection possibly occurring at $Gr_{sh} > 10^9$. Variations of Re_z appeared not to visibly influence the shroud heat transfer. Similarly so was the variation of the shaft rotation direction in the case of tests on Build 1 of the Multiple Cavity Rig. This indicates cavity air flow stratification at the outer cavity radii, making the air in the shroud vicinity impervious to the central flow influence. No differences in the shroud heat transfer between the two instrumented cavities in Build 2 were noticed. Following on this, Long and Childs (2007) extended their shroud heat transfer analysis on Build 3 of the Multiple Cavity Rig which was similar in operating and geometrical characteristics to Build 2 but with the difference of a smaller diameter shaft (104 mm diameter against 120 mm for Build 2) resulting in an annular gap almost twice as big as on Build 2. Similar conclusions were made as in Long et al. (2003). The shroud Nusselt number values of Build 3 were found to be greater than those of Build 2. This was attributed to a difference in throughflow interaction on the cavity flow as a result of an increased annular gap in Build 3. This is related to the increased measured tangential velocities found in Build 3's cavities compared to Build 2 as reported in Long et al. (2007). Recent flow velocity measurements were performed by Owen and Powell (2004) inside a single cavity rig with a heated downstream disc while the shroud and upstream disc were adiabatic. The shaft was co-rotating at disc speed. Their experiments covered the following range of axial and rotational Reynolds numbers, $0.147 \times 10^4 \leq Re_z \leq 4.78 \times 10^4$ and $0.299 \times 10^6 \leq Re_\phi \leq 3.2 \times 10^6$. The dimensional characteristics of the cavity were $a/b=0.4$ and $s/b=0.2$. Non-simultaneous LDA measurements for the radial and tangential velocity components were obtained through a Perspex upstream disc. The

measurements showed that for the case of a downstream heated disc, the core of fluid within the cavity tended to rotate at a slightly slower rate than solid-body rotation (by about 4%) and that the ratio of tangential fluid core velocity to solid body rotation tended to reduce even further as the fluid to disc surface temperature difference (ΔT) increased. This is in broad agreement with Farthing et al. (1992b)'s observations. Spectral analysis of the tangential velocities highlighted a multi-cell flow structure comprising one, two, or three pairs of vortices. This indicates complex three-dimensional flow within the cavity featuring vortex breakdown of the central flow which could perhaps be accentuated by the influence of a relatively big shaft diameter and an assymetrical disc heating. Radial velocity measurements were unfortunately not presented in Owen and Powell (2004) paper to illustrate this further.

Long et al. (2007) reported LDA measurements in the Multiple Cavity Rig for Build 2 and Build 3. Simultaneous axial and tangential velocity component measurements were performed on Build 2 from a window aperture inside the shaft. Build 3 featured a similar optical access for the axial and tangential velocity measurements, while also being able to perform simultaneous radial and tangential velocity measurements in the two last cavities through windows embedded in the last disc and downstream rotor casing. Radial velocities inside the cavities were found to be typically two orders of magnitude lower than the tangential velocity component. Tangential velocity spectra showed evidence of periodicity in the flow structure. The difference in flow structure between cavity 3 and 4 was attributed to differences in exit conditions from the cavities. A difference in tangential velocity profiles along the radii was also noted between Build 2 and Build 3.

This is attributed to a difference in the influence of the central throughflow and its breakdown on the flow inside the rotating cavities from the difference in annular gap size between Build 2 and Build 3. More details on the flow measurements are to be found in the present thesis.

Table 2.1 below summarises the reviewed experiments and operating conditions in rotating cavity with axial throughflow, while Figure 2.17 compares the range of experimental conditions to the operating condition of a typical engine in cruise.

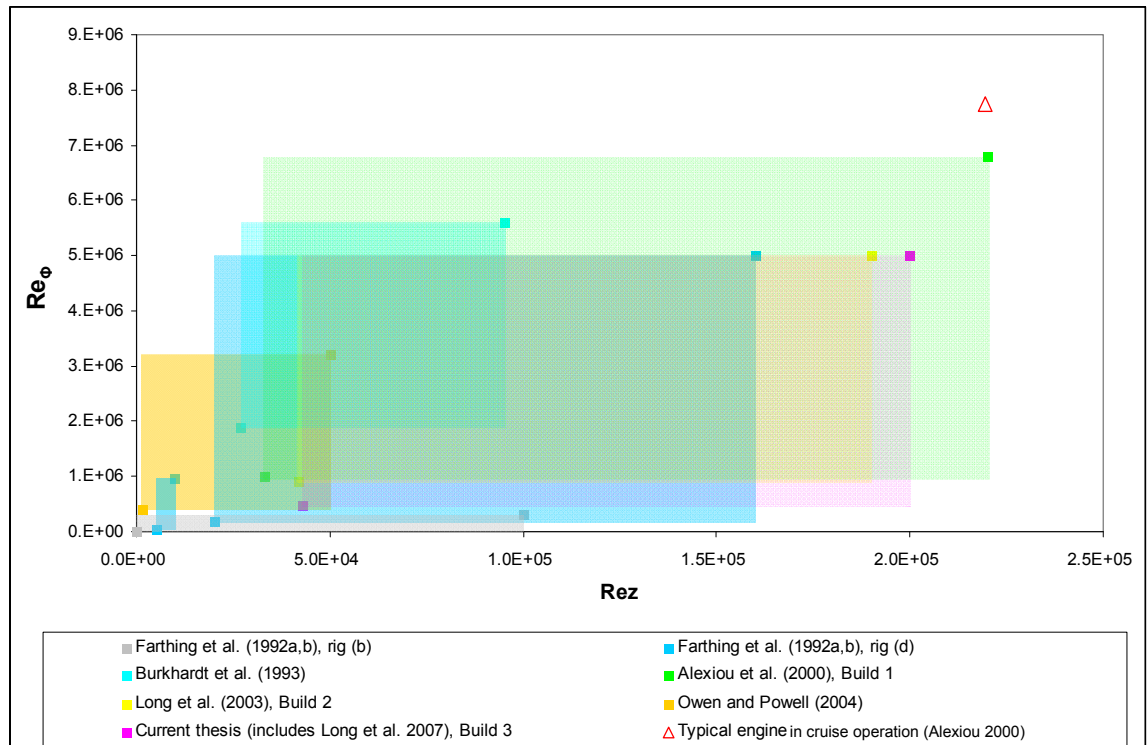


Fig 2. 17 Comparison of ranges of previous experiments to a typical engine in cruise operation

| Experiment | Number of cavities | Geometric description | Shaft | G | a/b | Heating | H/T | LDA | Re _φ range | Re _z range |
|---|--|---|------------------------|-----------------------------------|-------|---|-----|---------------------------|---|--|
| Farthing et al. (1992a,b) | rig a 1 cavity | idealised | no | 0.27 | 0.1 | air impingement 70°C max. | no | high speed photography | 1.3x10 ⁴ ≤Re _φ ≤ 8.7x10 ⁴ | 2.2x10 ³ ≤Re _z ≤ 7.7x10 ³ |
| | rig b 1 cavity | idealised | no | 0.133, 0.267, 0.4, 0.533 | 0.1 | isothermal | no | Vr, Vφ | 0≤Re _φ ≤ 3x10 ⁵ | 0≤Re _z ≤ 1x10 ⁵ |
| | rig c 1 cavity | idealised | yes | 0.267 | 0.1 | downstream disc | no | high speed photography | 0.5x10 ⁵ ≤Re _φ ≤ 9.5x10 ⁵ | 0.5x10 ⁴ ≤Re _z ≤ 1x10 ⁴ |
| | rig d 1 cavity | idealised | no | 0.12, 0.24, 0.36 | 0.1 | discs | no | Vφ | 0.5x10 ⁵ ≤Re _φ ≤ 9.5x10 ⁵ | 0.5x10 ⁴ ≤Re _z ≤ 1x10 ⁴ |
| | | | | | | | yes | no | 2x10 ⁵ ≤Re _φ ≤ 5x10 ⁶ | 2x10 ⁴ ≤Re _z ≤ 1.6x10 ⁵ |
| Burkhardt et al. (1993) | 5 cavities | engine parts | yes rotating | 0.256 | 0.286 | air impingement shroud, discs | yes | no | 1.9x10 ⁶ ≤Re _φ ≤ 5.6x10 ⁶ | 2.7x10 ⁴ ≤Re _z ≤ 9.5x10 ⁴ |
| Long (1994) | 1 cavity | idealised | no | 0.13, 0.36 | 0.1 | shroud, discs | yes | no | 2x10 ⁵ ≤Re _φ ≤ 5x10 ⁶ | 2x10 ⁴ ≤Re _z ≤ 1.6x10 ⁵ |
| Alexiou et al. (2000) | 2 cavities + turbine drive cone | geom.. similar (70% full size) | yes rotating | 0.195 | 0.3 | air impingement shroud, discs, cone | yes | no | 1x10 ⁶ ≤Re _φ ≤ 6.8x10 ⁶ | 0.33x10 ⁵ ≤Re _z ≤ 2.2x10 ⁵ |
| Long et al. (2003) | Build 1 2 cavities + cone | geom.. similar (70% full size) | yes rotating | 0.195 | 0.3 | air impingement shroud, discs, cone | yes | no | 1.1x10 ⁶ ≤Re _φ ≤ 7.7x10 ⁶ | 0.33x10 ⁵ ≤Re _z ≤ 2.2x10 ⁵ |
| | Build 2 5 cavities | geom.. similar (70% full size) | yes static | 0.195 | 0.32 | air impingement shroud, discs | yes | not reported | 0.9x10 ⁶ ≤Re _φ ≤ 5x10 ⁶ | 0.42x10 ⁵ ≤Re _z ≤ 1.9x10 ⁵ |
| Owen and Powell (2004) | 1 cavity | idealised | yes synchro nous | 0.2 | 0.4 | shroud, downstream disc | yes | Vφ Vr not reported | 4x10 ⁵ ≤Re _φ ≤ 3.2x10 ⁶ | 1.4x10 ³ ≤Re _z ≤ 5x10 ⁴ |
| Current thesis (includes Long et al.2007) | Build 2 5 cavities | geom.. similar (70% full size) | yes static | 0.195 | 0.32 | air impingement shroud, discs | yes | Vφ, Vz | 4x10 ⁵ ≤Re _φ ≤ 5x10 ⁶ | 4.2x10 ⁴ ≤Re _z ≤ 1.9x10 ⁵ |
| | Build 3 4 cavities | geom.. similar (70% full size) | yes static | 0.195 | 0.32 | air impingement shroud discs | yes | Vφ, Vz, Vr | 4.7x10 ⁵ ≤Re _φ ≤ 5x10 ⁶ | 4.3x10 ⁴ ≤Re _z ≤ 2x10 ⁵ |

Table 2. 1 Review of previous experiments in rotating cavities with axial throughflow

Progress in CFD modelling of gas turbines' components has been supported by experimental studies used to validate models and the parallel evolution of computing equipment. Sun et al. (2004) explained the difficulty involved in modelling buoyancy driven heat transfer in rotating cavities, whereas forced convection situations are comparatively easier to model accurately in CFD (due

to a lower importance of fluid density variations). The three-dimensional and unsteady nature of the flow within the compressor cavities requires a high-level of computational detail in order to define the overall behaviour of the flow and heat transfer. Cheaper computing facilities allow finer grids and smaller time steps to be used to try defining the flow better. However, the level of details provided by CFD simulations often falls within the measurement uncertainty zone of the experimental measurements they are trying to match. Therefore a somewhat paradoxical situation could occur in which a highly refined and detailed computation is necessary in order to be able to predict broader observations accurately. Tian et al. (2004)'s computations reproduced the geometry of the experimental rig used by Farthing et al. (1992a). Their modelling was three-dimensional, steady and a $k-\epsilon$ turbulence model was applied. In isothermal conditions, for a fixed Re_z and $\beta\Delta T$ (based on shroud temperature), rotational conditions were varied and the investigation was related to the flow structure evolution over a range of increasing rotational Rayleigh number. The simulations showed that the cavity flow and heat transfer could be separated into an outer radius zone where Rayleigh-Bénard type of convection could be encountered, and in the lower radius zone forced convection would occur. The Rayleigh-Bénard type zone size would increase with increasing Ra and a critical Rayleigh number Ra_c would mark the value over which the flow would become unstable and time dependent. From a stable axi-symmetric flow ($Ra_c=6.5\times 10^5$ in their case) the flow begins to oscillate and becomes more non axi-symmetric as Ra increases and three pairs of cyclonic and anti-cyclonic circulations would appear ($Ra\approx 8.5\times 10^6$).

Increasing further the value of Rayleigh number, two pairs of circulations would be found at $Ra=2.4\times 10^7$ and the flow would establish itself to one pair of circulations at $Ra\geq 2.6\times 10^7$.

Fig 2.18 shows a circulation pattern from Tian et al.'s computations for high Rayleigh number ($Ra=4.1\times 10^8$) very similar to the experimental observations of Farthing et al. (1992a).

Owen et al. (2006) performed CFD computations based on Owen and Powell (2004)'s experimental rig's geometry on three different running conditions. Similar circulating flow patterns to Farthing et al (1992a)'s observations were also highlighted in the simulations. The computed tangential velocity distribution

and radial Nu distribution were qualitatively similar to the experimental data. However, the computation's results at low Re_ϕ , Re_z , and Ro did not match its experimental equivalent, over predicting the tangential velocities and under predicting heat transfer values. The reason for this was not known by the authors.

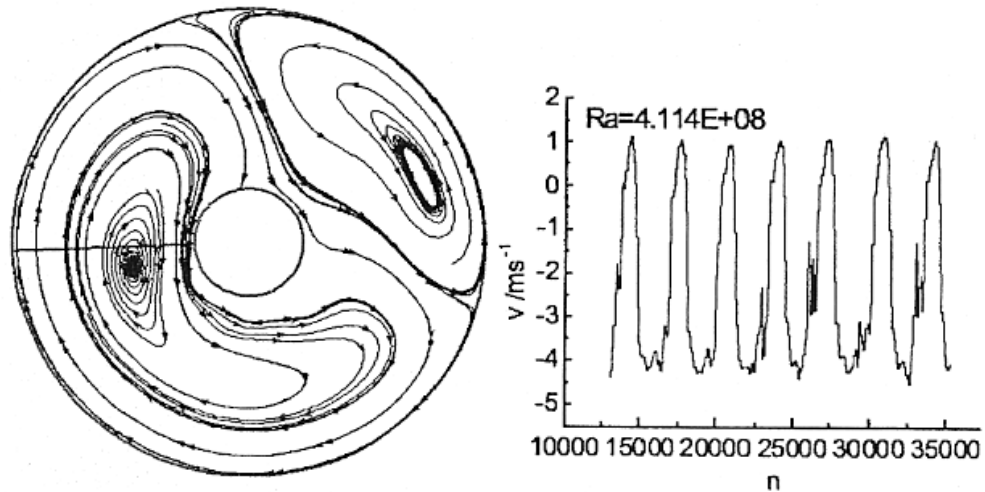


Fig 2. 18 Flow velocity field computations from Tian et al. 2004

Sun et al. (2006) performed a comparison simulation study on the computational ability of Large Eddy Simulation (LES) to obtain a better agreement with experimental measurements in buoyancy affected rotating cavity flow with axial throughflow, against a “regular” Navier-Stokes based computation using a $k-\epsilon$ type averaging turbulence model.

The geometry used was a 120 degree sector of a cavity geometry based on the Build 2 of the Multiple Cavity Rig. Three operating conditions were investigated in the range $1.87 \times 10^8 \leq Gr \leq 7.41 \times 10^8$ and $1.65 \leq Bo \leq 11.5$ (Gr being based on the shroud) from the experimental measurements performed by Alexiou (2002). Computations highlighted large Rayleigh-Bénard type flow structures within the cavity with the presence of unsteady circulating flows and radial arms. Overall, LES provided a finer flow structure description than the $k-\epsilon$ model and the velocity and temperature computations were in much better agreement with the experimental data, although still diverging on the computed heat transfer values by about 25%.

3. Experimental apparatus

As mentioned in the introduction, the experimental rig used in the present study follows on several generations of cavity rigs that were used in the TFMRC at the University of Sussex over the years. More precisely, the experimental rig used has several rotating cavities, and being the third variation of this rig set-up, is referred to as Multiple Cavity rig, Build 3, pictured in Figure 3.1.

Build 3 represents the internal set-up of a high pressure compressor on a 0.7 scale. A disc stack geometrically similar to a real engine is featured, forming the internal cavities. A shaft is represented in the inner radius, although it is stationary. Compressed air is blown across the rig as cooling air would inside a real high pressure compressor and thus forms the secondary air system in a compressor stack to be studied.

Build 2 will be seldom mentioned in this thesis. It was geometrically identical to Build 3 although the static shaft had a slightly bigger diameter. Results from Build 2 were reported in Alexiou (2001), Alexiou (2002) and Long et al. (2003). It is worth noting that some LDA measurement data had been produced in Build 2 but had not been published. Build 1 was the subject of the thesis of Alexiou (2000) and featured a shaft that could be rotated with or against the rotor direction at various speeds. Holes in the shaft allowed a portion of the cooling air to exit away from the cavity area. The rotor represented one compressor cavity and the turbine drive cone found on the end side of the high pressure compressor.

At the start of the work reported in this thesis, the Multiple Cavity Rig Build 3 had just finished being designed and manufactured by the technical team at the TFMRC and predecessor experimentalist on the Multiple Cavity Rigs, A. Alexiou. The author and A. Alexiou performed the commissioning of this rig with the LDA system using the shaft as optical access similarly to Build 2. Initial commissioning results were mentioned in Alexiou (2002) before the rig and project were handed over to the author late 2002. Provision for optical access through the discs had already been included in the design of Build 3. It was the author's subsequent task to source and set-up a suitable traverse system for LDA access through the discs before the experimental work would start. In the course of the currently reported experimental work, the author expanded the

flow measurement definition in greater details within the cavities. Measurements by the author within the shaft, cavity entrance and cob areas, together with the radial velocity measurements in cavities were the first of their kind to be reported.

Following is a description of the experimental apparatus. It is worth noting that most design features, instrumentation and test cell equipment related to Build 3 of the Multiple Cavity Rig had already been implemented and detailed in previous versions of the rig. More details on these are described in Alexiou (2000).

3.1 The Multiple Cavity Rig

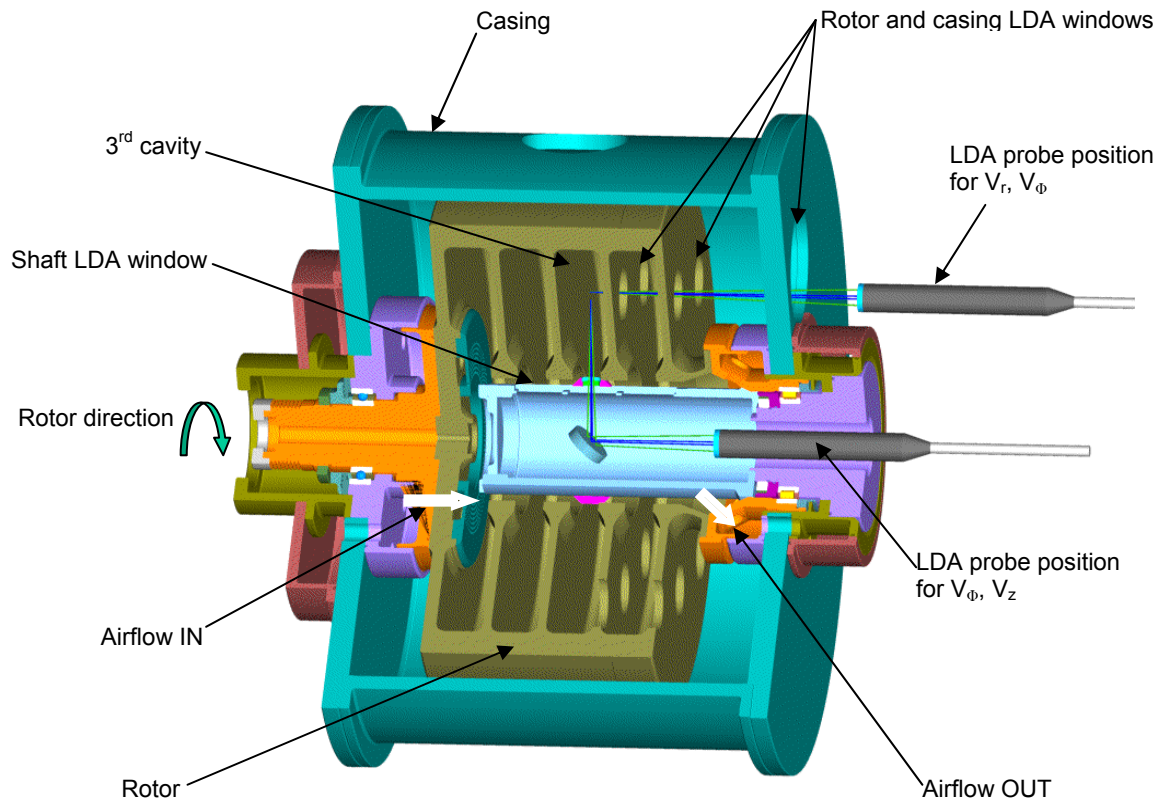


Fig 3 . 1 Cut-out view of the Multiple Cavity Rig build 3

Fig 3.1 shows a general view of the Multiple Cavity Rig Build 3. Figure 3.2 show a schematic of the multiple cavities and their dimensional parameters. The rotor is made of titanium 318 and forms four cylindrical cavities with an inner radius $a = 70.1\text{mm}$, an outer radius $b = 220.0\text{mm}$ ($a/b = 0.32$) and with a disc spacing $s = 42.9\text{mm}$ ($G = s/b = 0.195$). The last two discs carry glass windows to allow velocity measurements (radial and tangential components) in the last two cavities at $0.51 < r/b < 0.96$, also shown in Fig 3.3.

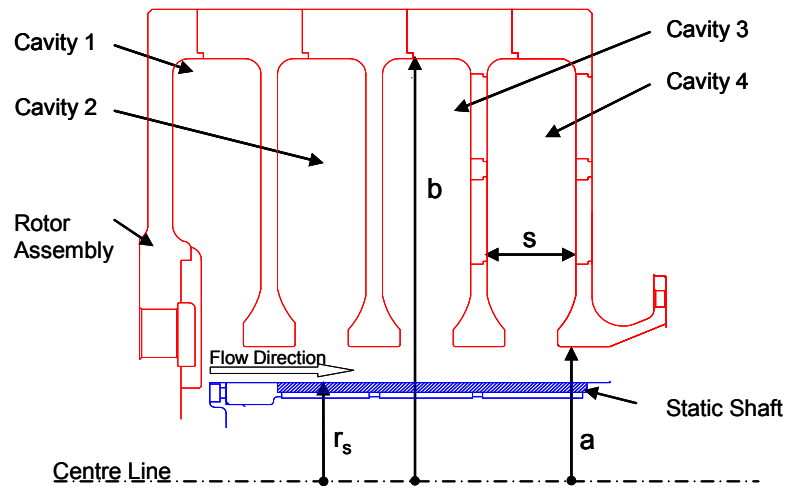


Fig 3. 2 Dimension parameters in the Multiple Cavity Rig, Build 3

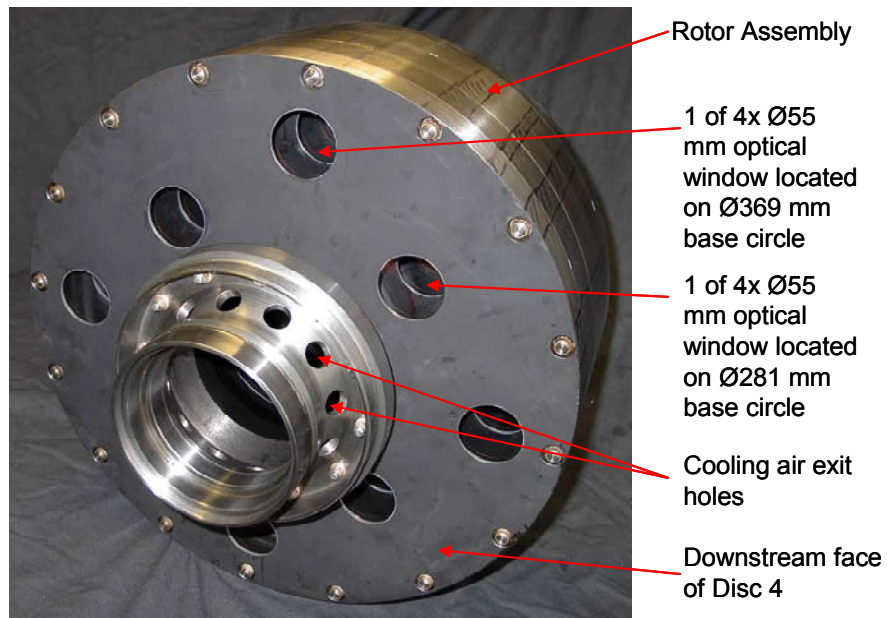


Fig 3. 3 External view of the downstream side of the rotor assembly showing the discs optical access windows

The non-rotating inner shaft has an outer radius $r_s = 52.0\text{mm}$ ($\Delta r = a - r_s = 18.1\text{mm}$). An axi-symmetric sleeve can be placed over the shaft as shown in Fig3.4. Its section geometry is shown in Fig 3.5.

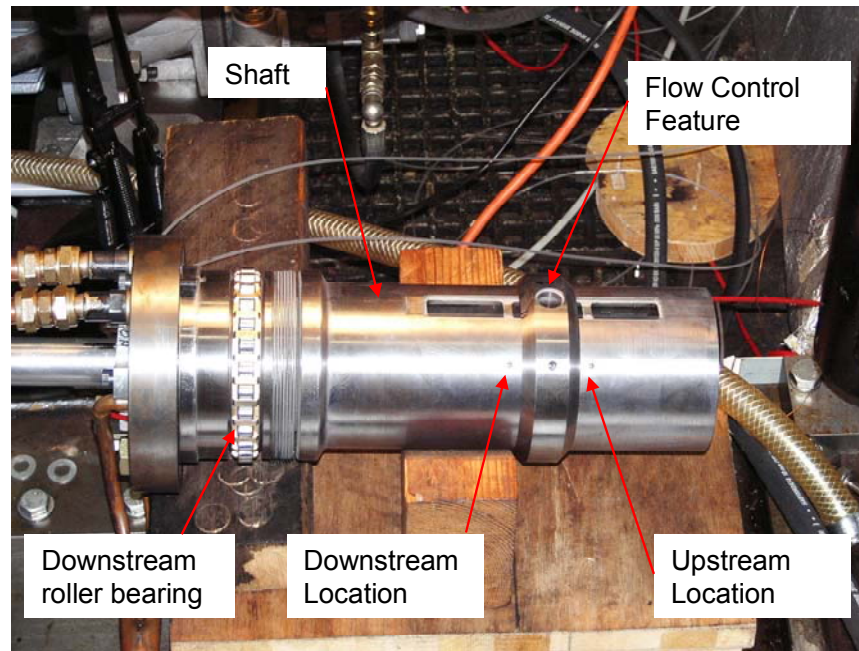


Fig 3. 4 View of the shaft and Flow Control Feature installed at mid-position

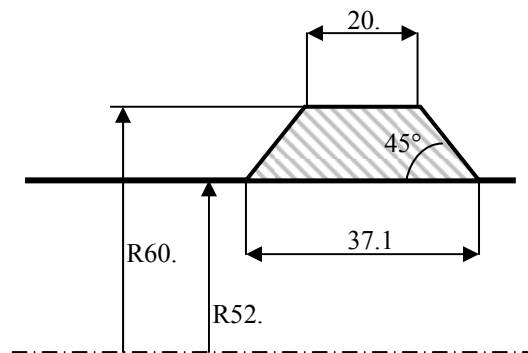


Fig 3. 5 Cross section sketch of the Flow Control Feature

This sleeve, named the 'Flow Control Feature', was aimed at influencing the behaviour of the central airflow and attempted to modify the amount of throughflow being diverted inside the third cavity. The Flow Control Feature could be positioned at three different axial locations as indicated in Fig 3.6, i.e. under disc 2, in the middle of cavity 3, or under disc 3.

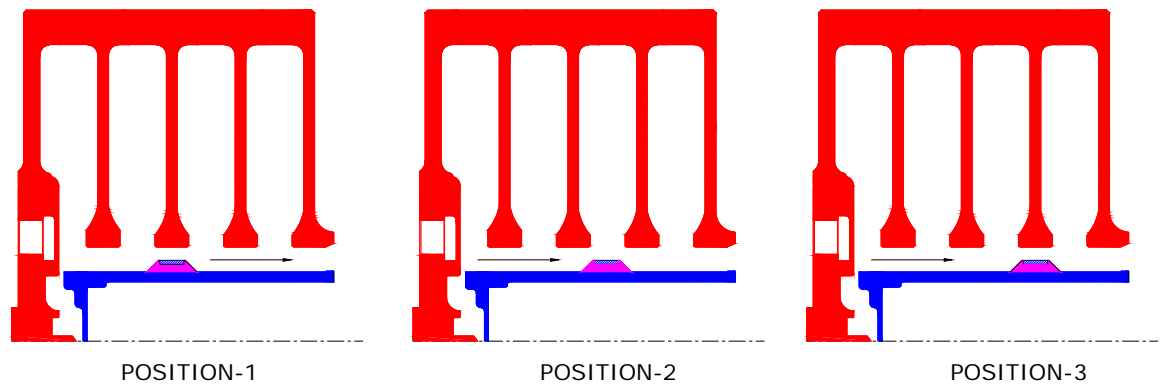


Fig 3. 6 Possible positions of the Flow Control Feature

The shaft has a glass window covering the axial length of the last three cavities. Through this window, the tangential and axial components of velocity in these cavities can be measured. The rig was designed so that the shaft can be removed without dismantling the rig. This allows for cleaning the optical access windows and positioning (or removing) the Flow Control Feature.

The rotor assembly is housed within a steel, non-pressurised casing and supported by a high precision grease lubricated ball bearing on the upstream side and an oil lubricated cylindrical roller bearing on the downstream side.

A 3 kW A/C motor was used to rotate the rotor in the clockwise direction (according to the direction of the cooling flow) via a 2.6 kW TR Tasc Unit driving a poly-vee belt. Operating speeds achieved were varied up to 6000 rev/min. A single pulse Hall effect sensor on the drive shaft provides speed feedback to the control desk.

The control desk, pictured in Fig 3.7, provides access to the data loggers, pump controls, speed controls, and includes the computer controlling the LDA system and the pressure and temperature data loggers.

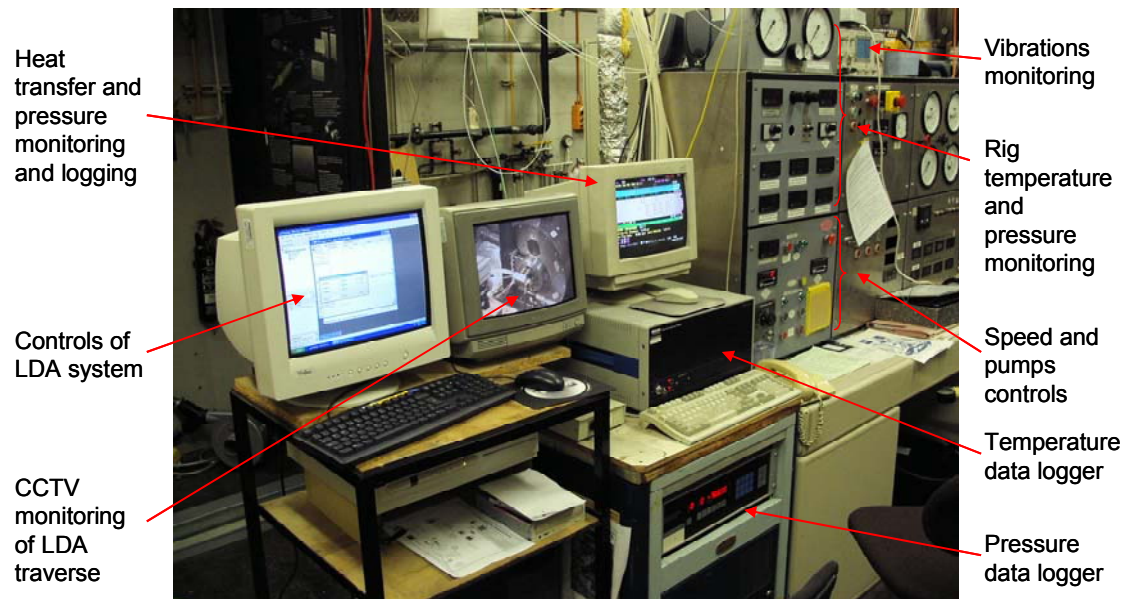


Fig 3. 7 Control desk of the Multiple Cavity Rig, Build 3

3.2 Air Supply

Air inside the rotor assembly is supplied to the rig by a single stage 275 kW Howden compressor which can deliver up to 1.1 kg/s of air at 4 bar abs. and 200°C, pictured in Figure 3.8.



Fig 3. 8 Photograph of the Howden screw compressor

Before entry to the rig, the air was cooled to typically 30-55°C (varied depending on air relative humidity and supply pressure in order to avoid condensation) by passing it through a single flow, shell and tube heat exchanger, with water flowing over the tubes at a variable rate to obtain the desired output air temperature. The air is delivered inside the rig via six circumferentially distributed holes of 25mm diameter in the upstream rotor disc, followed by a diffuser plate bolted onto the rotor in order to help create a uniform inlet velocity profile as described in Alexiou (2000) (pictured in Fig 3.9).



Fig 3. 9 Multiple Cavity Rig inlet diffuser plate

The air exits the rig in the downstream bearing support via 12 holes of 16mm diameter at 60° orientation outward to the axis of rotation. Sealing in the bearing supports and flanges of the pressurised rotor was provided by an arrangement of labyrinth seals designed to Rolls-Royce standard (Rolls-Royce, 1983), with the stator part lined with abradable material to increase sealing effectiveness. The labyrinth seals prevented lubricating oil ingestion into the rig provided the rotor was pressurised by at least 0.35 bar before the oil pressure pump was activated. More details on the lubrication and sealing systems can be found in Alexiou (2000).

The pressurisation of the rig was achieved by a manually operated ball valve downstream of the rig on the exhaust pipe leading to the outlet orifice plate.

Airflow across the rig was measured by an inlet and outlet orifice plate made to BS1042 specifications. The difference of the inlet and outlet airflows provided a measurement of the leakage flow through the rig's seals. Figure 3.11 illustrates the air system of the Multiple Cavity Rig.

The outer surface of the rotor assembly is heated by the impingement of hot air blown inside the rotor casing by two 10 kW Leister hot air heaters; each supplying up to 0.01 kg/s of air at 800 K.

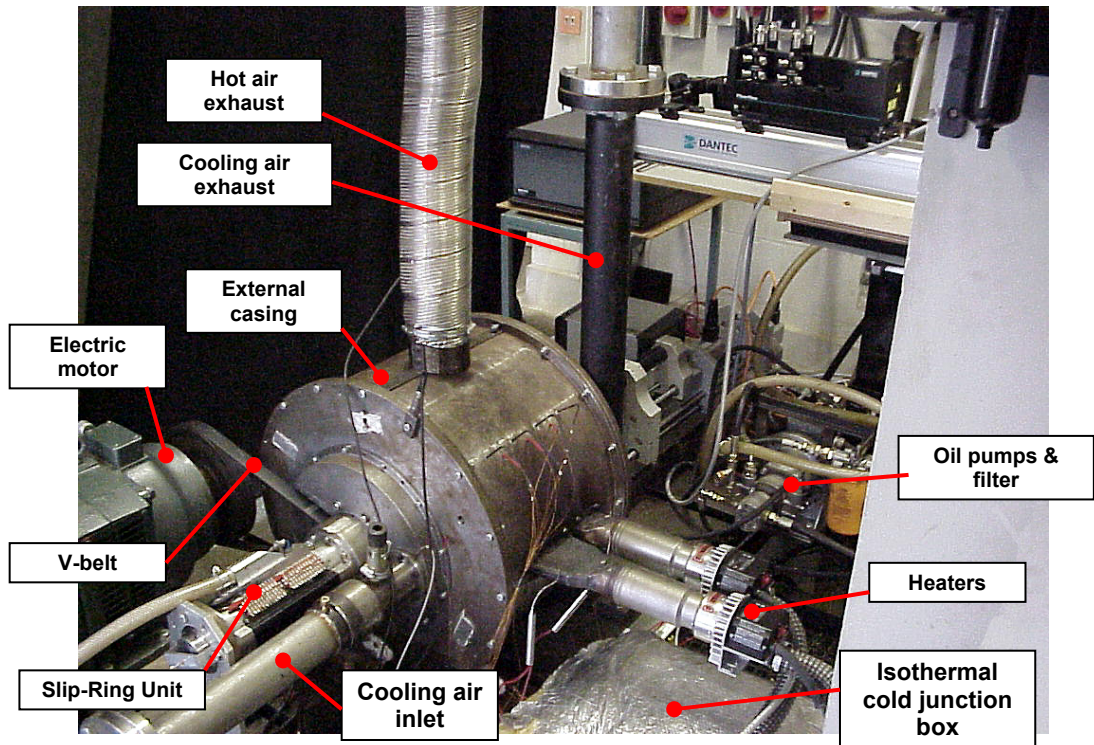


Fig 3. 10 External view of the Multiple Cavity Rig, Build 3

Fig 3.10 provides a general view of the rig. Fig 3.11 and 3.12 summarise the air supply and cooling layout of the rig.

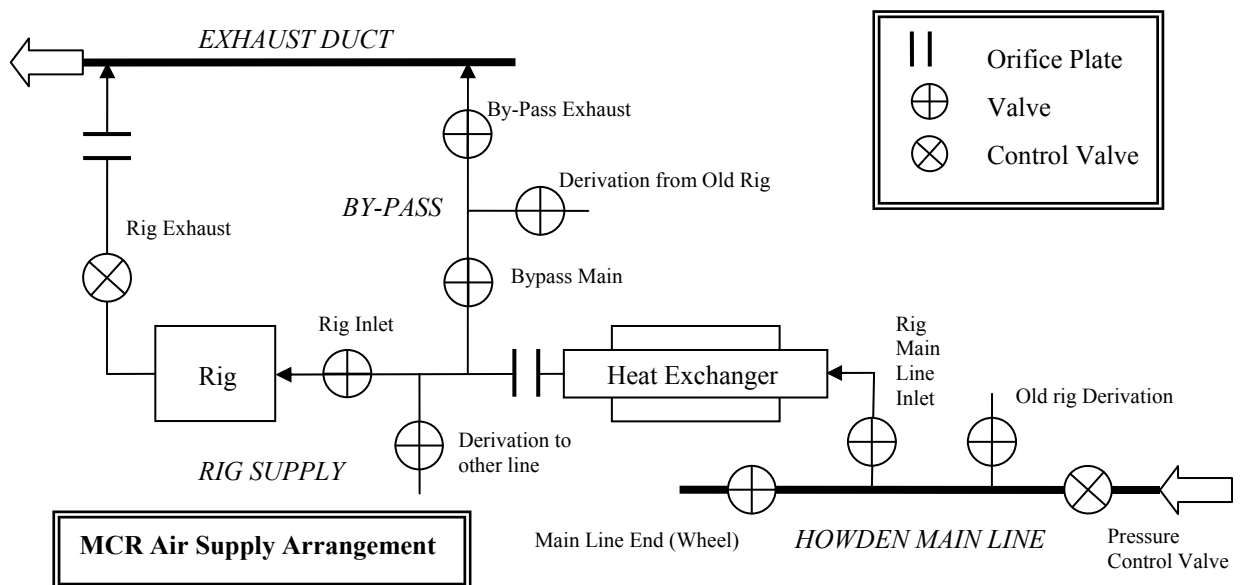


Fig 3. 11 Air supply diagram of the Multiple Cavity Rig, Build 3

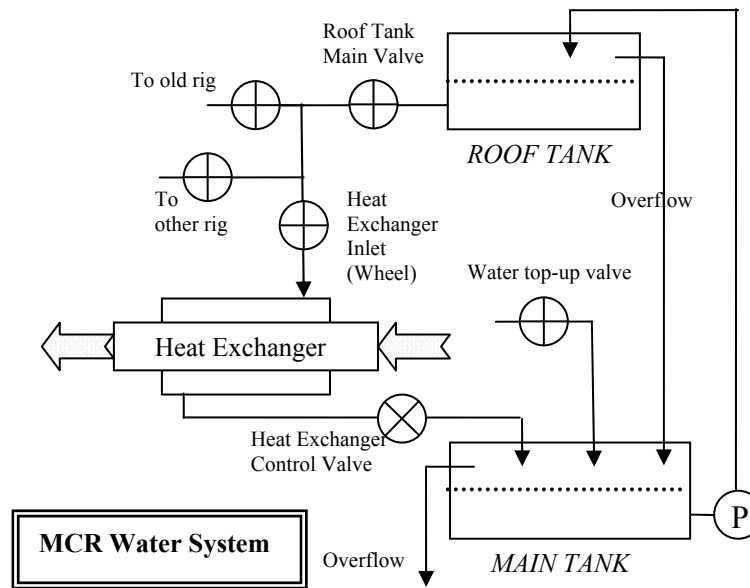


Fig 3. 12 Water cooling system diagram of the Multiple Cavity Rig, Build 3

3.3 Temperature measurement

Fig 3.13 shows the internals of the Multiple Cavity Rig and its points of instrumentation.

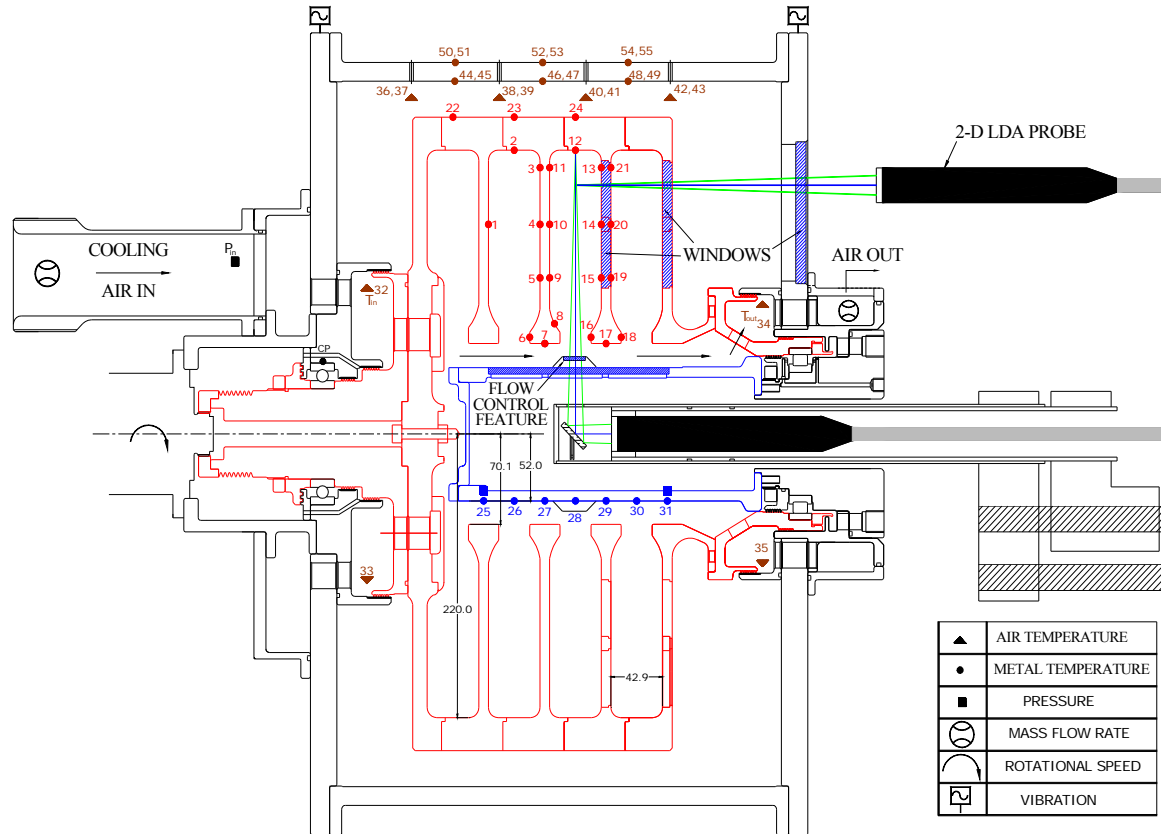


Fig 3. 13 Instrumentation of the Multiple Cavity Rig, Build 3

Temperature measurement can be distinguished between air-temperature measurements and surface temperature measurements. All temperatures were measured using calibrated 0.25 mm diameter wire K-type (NiCr-NiAl) thermocouples.

The air temperature thermocouples used for the inlet, exhaust and rotor casing airspace temperatures were placed inside a stainless steel tube protruding from the flanges or casing surfaces and holding the thermocouple bead at a distance of 5 mm from them.

Surface temperature thermocouples were cemented inside narrow channels in order to reduce thermal disturbances and titanium straps were welded on top of the channels to ensure that the leads could not be centrifuged out in the rotor parts as described in Alexiou (2000). The wires leading out from the beads in the rotor assembly were taken out tangentially in a 6 mm slot before bifurcating

into a radial channel in order to reduce the measurement errors due to heat conduction along the thermocouple wires in the radial direction. The thermocouple beads were punched into their slots so that they were flush with the surface and in intimate contact with the surrounding material in order to reduce thermal disturbances.

Referring to the numbering of the thermocouples in Fig 3.13, the thermocouple instrumentation of the Multiple Cavity Rig comprises an inlet cooling air temperature measurement comprising two diametrically opposed air temperature thermocouples located in the upstream part of the casing (thermocouples 32 and 33). Similarly, the cooling air exit temperature is measured by the air thermocouples 34 and 35. The rotor thermocouples relating to the heat transfer of disc 2 were located on the disc's surface temperature (thermocouples 3, 4, 5 and 9, 10, 11). The thermocouples related to the heat transfer of disc 3 were the surface thermocouples 13, 14, 15 on the upstream face of disc 3 and thermocouples 19, 20, 21 on the downstream face of disc 3. The shroud heat transfer of cavity 2 was obtained via the measurement of surface thermocouples 23 and 2, while the surface thermocouples 24 and 12 provided the data necessary to derive the heat transfer characteristics of the shroud in cavity 3. More details on the radial and relative circumferential locations of the rotor thermocouples are provided in table 3.1 below.

| Rotor thermocouple No. | Radial position (mm) | Circumferential position (°) |
|------------------------|----------------------|------------------------------|
| 1 | 162.5 | 90 |
| 2 | 220.0 | 135 |
| 3 | 206.5 | 225 |
| 4 | 162.5 | 45 |
| 5 | 121.0 | 0 |
| 6 | 75.0 | 180 |
| 7 | 70.1 | 0 |
| 8 | 85.5 | 90 |
| 9 | 121.0 | 270 |
| 10 | 162.5 | 315 |
| 11 | 206.5 | 135 |
| 12 | 220.0 | 90 |
| 13 | 206.5 | 135 |
| 14 | 162.5 | 315 |
| 15 | 121.0 | 45 |
| 16 | 75.0 | 225 |
| 17 | 70.1 | 45 |
| 18 | 75.0 | 225 |
| 19 | 121.0 | 180 |
| 20 | 162.5 | 0 |
| 21 | 206.5 | 270 |
| 22 | 245.65 | 270 |
| 23 | 245.65 | 315 |
| 24 | 246.45 | 180 |

Table 3. 1 Location of the rotating instrumentation (thermocouple numbers according to Fig. 3.13)

All the rotor thermocouple leads were led through instrumentation holes in the periphery of the rotor down to a hole in the upstream rotor drive flange to a Wendon 48 way Slip Ring Unit (SRU) linked to the end of it. The thermocouples and junction boxes had previously been made for Build 1 as detailed in Alexiou (2000). The calibration of this batch of thermocouple had previously been determined against a platinum resistance thermocouple in a thermostatically controlled water bath. A voltage characteristic of this batch was determined by a least squares quadratic curve in Alexiou (2000). A K-type thermocouple from the same batch as the rotating thermocouples was measuring the cold junction temperature at the leads connection of the slip ring unit. The non-rotating thermocouples had also previously been calibrated similarly to the rotating ones, and all leads were connected into an isothermal cold junction box. The temperature in the connection box was measured by a pair of RS components LM35CZ precision temperature sensors. The voltage signals from the slip ring unit thermocouples and from the isothermal junction box were measured by a Solartron Orion data logger with a sensitivity of $\pm 1 \mu\text{V}$, corresponding to $\pm 0.025^\circ\text{C}$ for the thermocouples used. The uncertainty level associated with the thermocouples measurement, calibration and cold junction compensation will be explained in Chapter 4. The Orion channels list is provided in Appendix 3.1.

A Q-BASIC program was used to control remotely the data logger and convert and record the temperatures from the calibration equations and the cold junction signals.

Fig 3.14 shows the layout of the instrumentation of the Multiple Cavity Rig. The thermocouples and Orion data logger arrangement can be seen in the left part of the diagram.

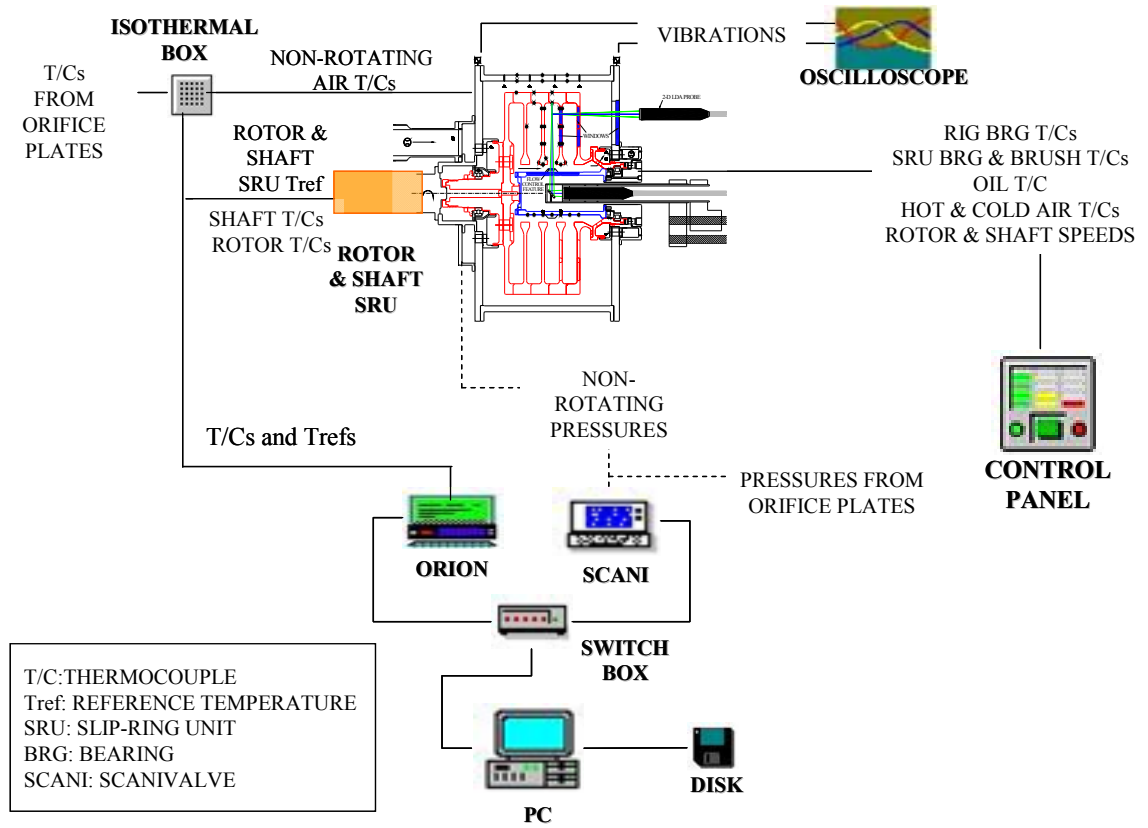


Fig 3. 14 Instrumentation layout of the Multiple Cavity Rig, Build 3

3.4 Pressure measurement

Static air pressures were measured by a Scanivalve system controlled remotely by the same computer used for the temperature data logging. The layout of the pressure measuring equipment is featured in Fig 3.14. The static pressures measured included the orifice plates, inlet and outlet cooling air pressure and shaft pressures. A detailed list of the logged channels can be found in appendix 3.2.

The Scanivalve pressure channels were calibrated against a dead-weight tester as per Alexiou (2000) and displayed an accuracy of 0.1%.

3.5 Air velocity measurement

Air flow velocity measurements within the Multiple Cavity Rig were performed by Laser Doppler Anemometry (LDA). This method of measurement allows a high spatial and temporal resolution. It requires no calibration as the output signal is a function of velocity, and is able to measure reverse flows with a suitable frequency shifting technique.

The main elements of an LDA system consist of:

- a continuous wave laser
- a beam splitter
- a frequency shift device
- transmitting optics (mirrors, fibre optic etc.)
- receiving optics
- a photomultiplier
- a signal processor
- a computer to remotely control the processor and log the data

LDA systems allow the determination of a velocity component from moving particles in a fluid. The intersection of a moving particle (be it dust naturally present in the air or a purposely introduced tracking particle), commonly referred to as a seeding particle, into a coherent source of light (laser beam) will scatter light around from reflection or refraction with the seeding particle. Due to the movement of the particle relative to the beam direction, the scattered light will show an apparent frequency shift compared to its source, also known as Doppler shift. The frequency shift is in direct relation to the relative displacement speed of the seeding particle's component of velocity along the laser beam direction. The scattered light signal therefore needs to be detected and compared to the source signal, so that the component of velocity can be determined.

Splitting the source beam into two separate beams directed so that the beams cross each other at a known angle allows the detection of velocities along the two beam directions, thus allowing a plane of measurement to be defined.

The known crossing angle will permit the definition of the direction of the velocity component measured from the addition of the two Doppler shift vectors. The resulting vector will run along the line perpendicular to the bisection of the crossing beams belonging to the plane of intersection. This represents the working principles of a one-dimensional differential Doppler system, shown in Fig 3.15.

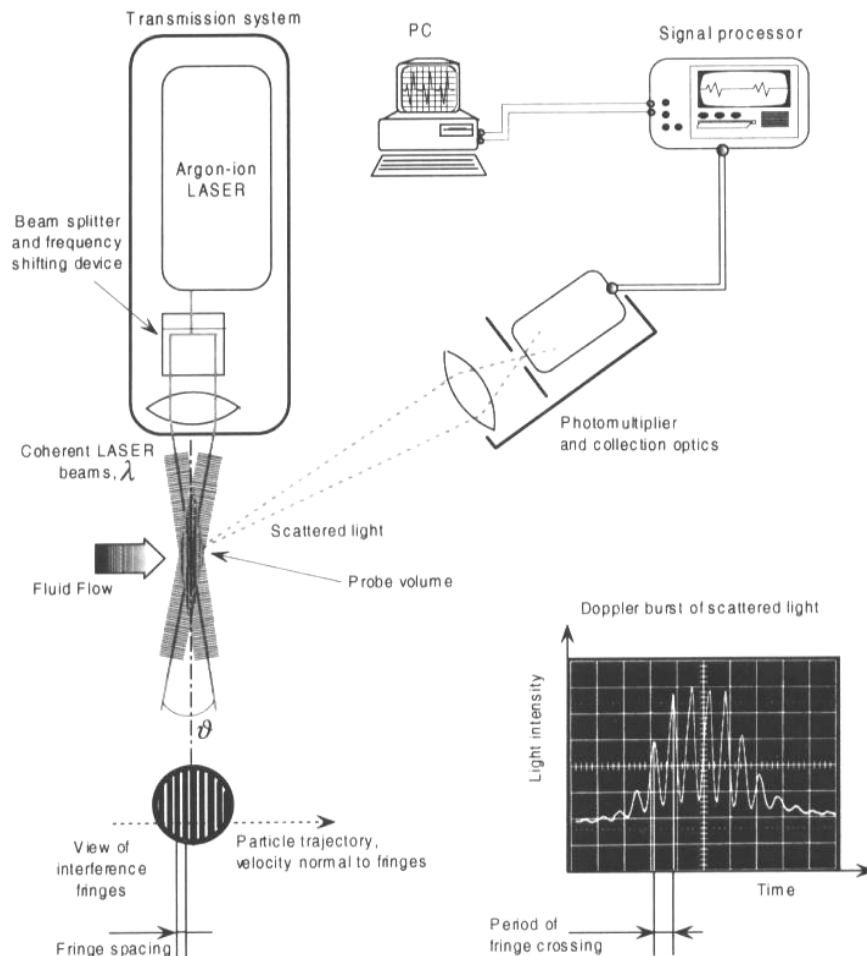


Fig 3. 15 Working principle schematics of a one-dimensional LDA system (from Begg 2003)

The two split beams of equal intensity cross into a focal point of high light intensity. This intersection region is referred to as the measurement volume, or probe volume. The laser beams cross sectional intensity pattern is in a bell shape with the maximum light intensity in the centre of the beam. This Gaussian mode of emission is also known as the TEM 00 transverse laser mode.

The high light intensity zone of the probe volume will therefore show an ellipsoidal shape as shown in Fig 3.16.

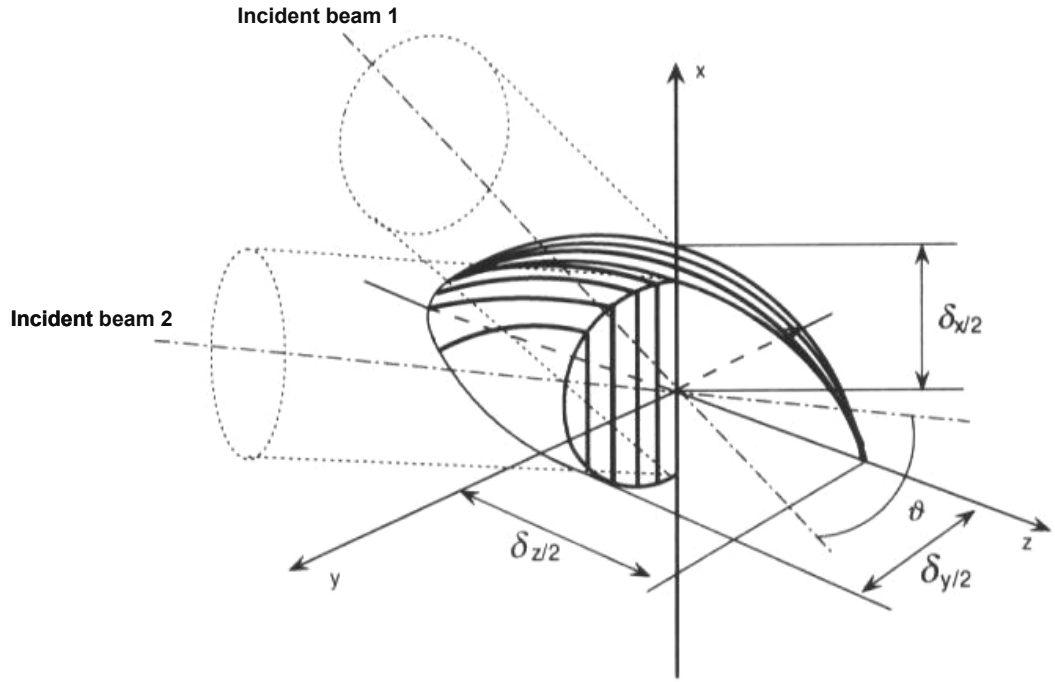


Fig 3. 16 Probe volume construction and dimensions (from Begg 2003)

The dimensions of the probe volume shown in Fig 3.13 can be expressed as:

-Height
$$\delta_x = \frac{4 \times F \times \lambda}{\pi \times E \times d_g \times \cos\left(\frac{\theta}{2}\right)}$$
 Eq. 3.1

-Width
$$\delta_y = \frac{4 \times F \times \lambda}{\pi \times E \times d_g}$$
 Eq. 3.2

-Length
$$\delta_z = \frac{4 \times F \times \lambda}{\pi \times E \times d_g \times \sin\left(\frac{\theta}{2}\right)}$$
 Eq. 3.3

Where F is the focal length of the incident beams
 λ is the wavelength of the emitted laser light
 E is the beam expansion factor
 d_g is the beam Gaussian diameter
 and θ is the angle of incidence of the beams

Considering the “interference fringe model” commonly found in the literature, at the point crossing the probe volume the two coherent light waveforms meet and add to each other when in phase or cancel each other when in opposite phase. The resulting interferences produce a series of equally spaced constructive and destructive interference fringes which materialise as bright and dark stripes. Fig 3.17 represents a simplified interference model of interfering wave fronts.

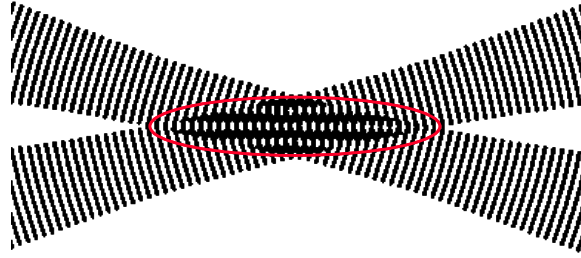


Fig 3. 17 Interference fringe model (Dantec 2004)

The fringe spacing is a function of the angle of incidence of the beams and their wavelength. It is expressed as:

$$\delta_f = \frac{\lambda}{2 \times \sin\left(\frac{\theta}{2}\right)} \quad \text{Eq. 3.4}$$

The interference fringes in the probe volume are formed parallel to the bisector of the beams and in a plane perpendicular to the plane of the beams intersection. The number of fringes in the probe volume is calculated as:

$$N_f = \frac{8 \times F \times \tan\left(\frac{\theta}{2}\right)}{\pi \times E \times d_g} \quad \text{Eq. 3.5}$$

The fringes in the probe volume offer a zone of high contrast compared to the light intensity of the incident beams. A seeding particle passing through the fringes will therefore scatter light at a level that can be detected and filtered out from other scattering not occurring in the probe volume.

A collection lens pointed at the probe volume is used to transmit the scattered light of seeding particles to a Photo Multiplier Tube (PMT) that will convert the fluctuating light intensity into an electric signal.

Seeding particles scatter light by refraction or reflection in all directions. However the highest light intensity of scattering is observed in the direction away from the laser source. This is referred to as forward scatter mode and is illustrated by the polar plot in Fig 3.18 for a particle size equal to the wavelength of the laser source.

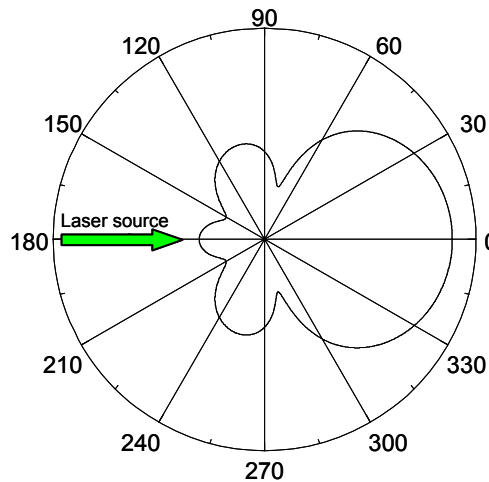


Fig 3. 18 Polar plot of light scattering from a particle of size equal to the wavelength of the laser source with the light intensity shown on a logarithmic scale (Dantec 2004)

The positioning of the receiving optics would therefore preferably be in the forward scatter mode (i.e. pointing towards the laser source) in order to improve the signal to noise ratio (SNR) of the measured scattered light through the probe volume. However, for practical reasons, it is often more convenient to locate the receiving optics pointing from the same direction as the laser sources as this configuration offers the convenience of having only one optical access required on the experimental set-up. Even though the scattering light intensity in back scatter configuration is less than in forward scatter, the signal to noise ratio of the scattered light from the probe volume is still strong enough for practical signal measurement and is widely used in commercial systems. As a seeding particle passes through the probe volume fringes, the scattered light from the particles will be seen to fluctuate from the succession of constructive and destructive wavefront interferences.

Due to the Gaussian transverse power distribution of the incident beams, the overall power intensity will be higher by the centre fringes of the probe volume.

The detected scattered light intensity signal of the seeding particle passing through the probe volume will typically look like the signal shown in Fig 3.19 below, with an oscillating pattern whose period corresponds to the travel time between a light and a dark fringe. The overall light intensity varies in a bell shape as the particle progresses through the fringes.

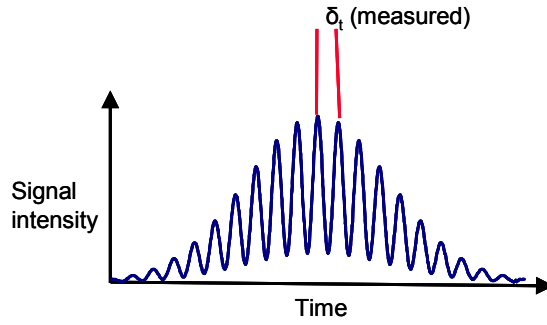


Fig 3. 19 Typical Doppler burst signal of scattered light (Dantec 2004)

Knowing the distance between the fringes, δ_f , and the time, δ_t , it took the seeding particle to cross them, the speed of the particle in the component normal to the fringes, U , can be deduced as:

$$U = \frac{\delta_f}{\delta_t} \quad \text{Eq. 3.6}$$

The frequency at which the light is scattered is equal to the Doppler frequency of a dual beam anemometer (Albrecht et al. 2003) and is expressed as:

$$f_d = \frac{U}{\delta_f} = \frac{2 \times U \times \sin\left(\frac{\theta}{2}\right)}{\lambda} \quad \text{Eq. 3.7}$$

Modern signal processors determine the Doppler frequency from the photo multiplier signal using correlation or Fast Fourier Transform (FFT) algorithms. Some high pass and low pass filtering is also applied to differentiate the randomly arriving frequency modulations of a seeding particle passing through the probe volume, named “burst” from the general optical and electronic noise collected by the PMTs. Signal filtering is also applied to discard bursts produced by multiple particles passing simultaneously into the probe volume.

The velocity component calculation being in direct relation to an induced frequency of passage through equally spaced fringes, it is not possible to distinguish whether the passage occurred in one direction or the other of the velocity component. This directional ambiguity can be eliminated by introducing a frequency shift to one of the incident beams. An acoustic-optical modulator, a “Bragg cell”, is commonly found in modern LDA systems in the transmitting optics in order to add a known frequency shift to one of the incident beams. The frequency modulation of one beam compared to the other produces variations in the interference patterns in the probe volume. As the beams are unequally shifted, the fringes will appear to be scanning across the probe volume at a constant velocity. Knowing the direction of the frequency shift, a particle moving in the direction of the shift at a certain speed will produce a burst signal of lower frequency than the frequency that would have been produced in a non-shifted probe volume. Conversely, a particle going against the fringes will produce a higher frequency burst. Additionally, a stationary particle will now be able to be detected as the burst frequency produced will correspond to the frequency shift.

Seeding particles introduced into the fluid of which velocity is to be measured must be sufficiently small in order to accurately follow the flow velocity field, yet be sufficiently large in order to scatter enough light to produce Doppler bursts strong enough to give an acceptable signal to noise ratio. Various materials and forms can be considered as suitable seeding particles, such as oxide powders or atomised oil. The type of seeding and size of seeding particles will depend on the type of flow to be studied, its environment, and the optical set-up of the LDA system used. Typically, atomised oil particles of diameter size in the order of $1\mu\text{m}$ are used in airflow measurements. The fidelity of motion of a flow tracer particle in turbulent flow can be quantified by its Stokes number (Sk) as explained in Tropea et al. (2007). It is defined as the ratio between the characteristic particle response time, τ_p , and the characteristic particle flow time scale τ_f . For Stokes numbers $Sk = 1$ the particles will follow the flow closely.

Considering a particle material density $\rho_p = 870 \text{ kg.m}^{-3}$ (quenching oil, RS 184-7939), a particle diameter produced by the seeder of $d_p = 1\mu\text{m}$, into air of

properties $\mu_f = 15 \times 10^{-6}$ Pa.s and $\rho_f = 1.2$ kg.m⁻³, the particle characteristic response time is given in Albrecht et al. (2003) as:

$$\tau_p = \frac{\rho_p \cdot d_p^2}{18\mu_f} \quad \text{Eq. 3.8}$$

The cut-off dynamic frequency at which the particles still follow the velocity oscillation of the flow, allowing for 1% slip, can be calculated as (Albrecht et al. 2003):

$$f_c = \frac{0.0227}{\tau_p} \quad \text{Eq. 3.9}$$

An expression of the Stokes number as a characteristic non-dimensional frequency of the particle response was shown in Melling (1997) from the solution of Hjelmfelt and Mockros (1966) as:

$$Sk_c = \left(\frac{\rho_f \cdot \omega_c}{\mu_f} \right)^{1/2} \cdot d_p \quad \text{Eq. 3.10}$$

Where $\omega_c = 2\pi f_c$

In the case of the presented LDA experiments, the cut-off frequency is estimated as $f_c = 70.5$ kHz for a 1 μ m seeding particle. This corresponds to Stoke numbers of the seeding of $Sk < 0.19$ at flow frequencies up to f_c ; which is considered as acceptable in the current experiment.

When selecting the optical set-up and seeding Begg (2003) recommends a fringe spacing about four times bigger than the mean particle diameter as best practice. The density of seeding is also important in order to increase the rate of burst sample occurrence, but too much seeding would also be increasing the likelihood of rejected data due to Doppler bursts from simultaneous particles within the probe volume. Longer seeding duration, especially in dense atomisation can also cause the optical access windows in the experimental rig to become dirty faster, adding optical glare in the emission and collection optical path, hence lowering the signal to noise ratio.

A two velocity component LDA system is easily obtained by adding another dual beam system and burst signal processing channel to the original system so that the second pair of beams' probe volume coincides with the first probe volume. By having the second pair of beams orthogonally positioned to the first one, a direct measurement of planar velocity components is possible. Distinguishing

the Doppler signals one component from another is achieved by using different wavelengths (colours) for each pair of beams. A colour separator after the collection optics filters the scattered light into two different optical components which in turn become converted into electrical signals by a PMT and processing unit per velocity component.

Ease of use and money savings can be provided from a dual frequency laser source such as the Stabilite 2017 laser from Spectra-Physics that was used on the Multiple Cavity Rig's LDA system. The described LDA equipment and 'in-shaft' coaxial traverse system have been used since Build 2 of the Multiple Cavity Rig and were previously mentioned in Alexiou (2001) and Alexiou (2002). The water cooled Argon-Ion laser was emitting in the green colour mode (514.5 nm wavelength) and blue (488 nm) at a power output of up to 5 Watts.

The transmitting and receiving equipment's configuration schematic is shown in Fig 3.20.

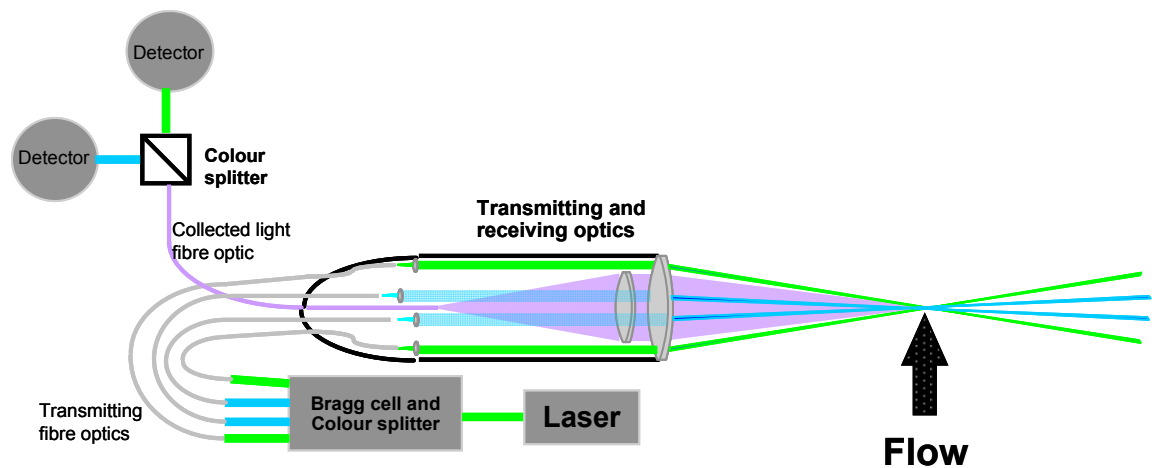


Fig 3. 20 Typical layout of a 2-D backscatter LDA system (adapted from Dantec 2004)

The laser source is emitted into a DANTEC Fiberflow 2D system. The laser source is split and a frequency shift of 40 MHz added by a Bragg cell. A colour separator outputs a pair of non-shifted and shifted beams for each colour frequency of the laser source. The beam pairs are then transmitted via a 10 m long fibre optic cable to an integrated back-scatter two dimensional probe.

The total output at the fibres was about 30% of the supplied power at the laser source. The 27 mm diameter Fiberflow probe allowed the transmission of the measurement beams in a factory aligned sealed casing. The front optic was

modified to provide a 250 mm focal length to the probe. This allowed a sufficient reach of the probe volume deep inside the rig's cavities. The 27 mm external diameter of the probe permitted its insertion inside the Multiple Cavity Rig's shaft as will be explained below. The probe volume characteristics are shown below in Table 3.2 together with other relevant LDA parameters.

| Parameter | Value |
|--------------------------------|-------------------------------------|
| Transmitting lens focal length | 250 mm |
| Beam spacing | 15 mm |
| Frequencies | |
| Tangential velocity (green) | 514.5 nm |
| Axial velocity (blue) | 488 nm |
| Laser beam diameter | 1.35 mm |
| Beam half angle | 1.718 ° |
| Expansion factor | 1 |
| Probe volume diameter | 0.121 mm (for $\lambda = 514.5$ nm) |
| Probe volume length | 4.046 mm (for $\lambda = 514.5$ nm) |
| Number of fringes | 14 |
| Fringe spacing | 8.579 μm |
| Frequency shift | 40 MHz |

Table 3. 2 LDA parameters

The 15 mm beam spacing and 250 mm focal length produced a length of the probe volume of about 4 mm. The relatively large length of the probe volume had the consequence of reducing the ability to measure near walls. The resulting fringe spacing of about 8 μm also meant that multiple measurement occurrences could be more frequent due to the 8 to 1 ratio of fringe spacing to seeding particle size of 1 μm provided by the TSI 9306A six jet atomiser seeding RS components RS184-7939 quenching oil into the rig.

Therefore, the seeding density had to be kept to a minimum for acceptable validated data rate. In order to reduce glare and optical noise while measuring, the internal surfaces of the rig were painted matt black and the optical access

windows were made out of BK7 optical grade glass treated with an anti-reflecting coating for the 480-520 nm wavelength range. The scattered light signals were sent to the photomultipliers from the probe via a fibre optic and colour separator.

The photomultipliers' signals were fed to a DANTEC BSA70 Burst Spectrum Analyser which processed the Doppler signals and managed the power supplies to the PMT and Bragg cell. The signal processor features automatic surge protection of the photomultipliers, a hardware Fast Fourier Transform algorithm capable of handling and validating Doppler bursts at rates of up to 100 kHz. A Baumer BHG06-25W360 encoder driving a Texas Instrument 74HCT240W logic transceiver mounted at the end of the slip ring unit of Build 3 as shown in Figure 3.21. The encoder driver, which connection diagram is shown in Appendix 3.3, provided a TTL level synchronisation input to the LDA signal processor. Although the encoder could provide 360 pulses per revolution, only the reset pulse was used as is normal practice in rotating machinery where the rotational speed can be considered steady.

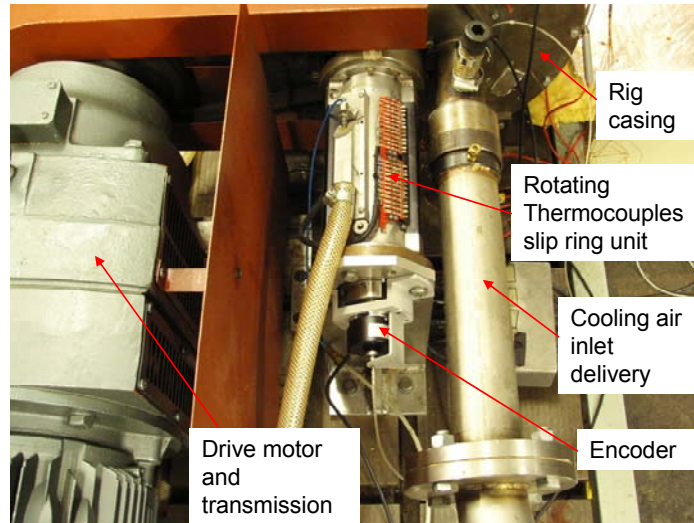


Fig 3. 21 View of the shaft encoder mount and slip ring unit

The BSA Flow software (v3.0) provided the user interface and data collection facility of the LDA system. It features an online Doppler monitoring window and indications of data rates and validation percentages. Controls over the different filtering and photomultipliers voltages were provided and the number of samples

to be acquired could be set as an end condition of measurement. A range of data calculations such as velocity moments, RMS and intervals of confidence could be applied as the samples were collected or in post-processing. Additional calculations such as spectral analysis were also included in the package. Traverse systems could be controlled automatically, allowing an easier and precise locating of the probe volume in a user-defined experimental spatial grid.

Two different traverse systems were used for the LDA measurements on the Multiple Cavity Rig depending on the type of cavity velocity component to be measured. For the measurements of axial and tangential components, the 2D LDA probe was mounted along the rig's centre line with the traverse system already developed for Build 2 (Alexiou 2001) represented in Fig 3.22.

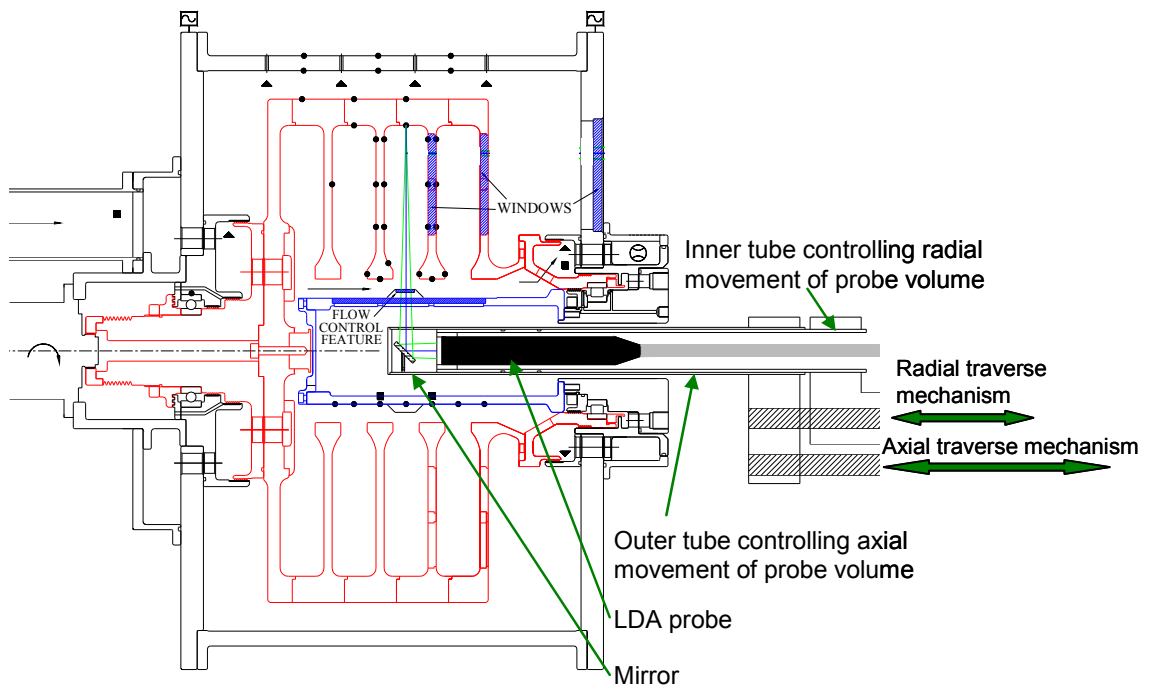


Fig 3. 22 Schematics of LDA traverse system for measurements of V_ϕ and V_z velocity components

A 45° mirror in line with the probe reflected the beams up inside the rig's cavities and allowed collection of the scattered light back to the probe. A longitudinal traverse system driving two co-axial tubes provided an accurate positioning of the probe volume within the rig. The inside tube supported the

probe while the outside tube held the mirror on its end. Moving the inside tube along the centre line relative to the outside tube modified the distance of the probe to the mirror, which resulted in the probe volume changing position accordingly in the vertical axis, thus changing the radial location. Moving both tubes together in the longitudinal axis resulted in the displacement of the probe volume across the cavities. The displacement of the tubes was controlled by stepper motors which were directly commanded from the LDA software, allowing the positioning of the points of measurement from a pre-determined positions grid. Figure 3.23 shows an outside view of the LDA system configuration for the measurements of tangential and axial velocities.

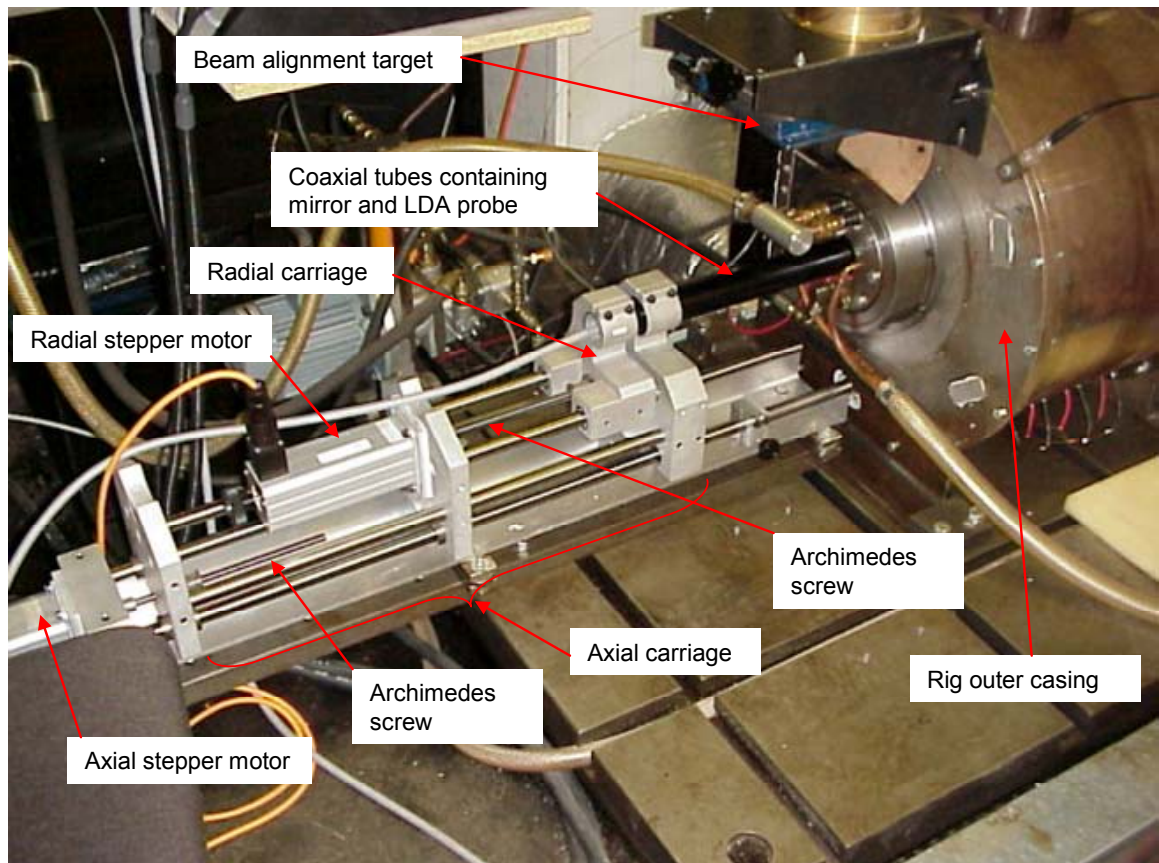


Fig 3. 23 External view of the traverse system for measurements of V_θ and V_z velocity components

The angular alignment of the probe was ensured by pulling out the traverse system so that the beams would shoot onto a horizontal external target referenced to the geometry of the rig, shown in Figure 3.23. The measuring areas allowed by this probe configuration are shown in Fig 3.24.

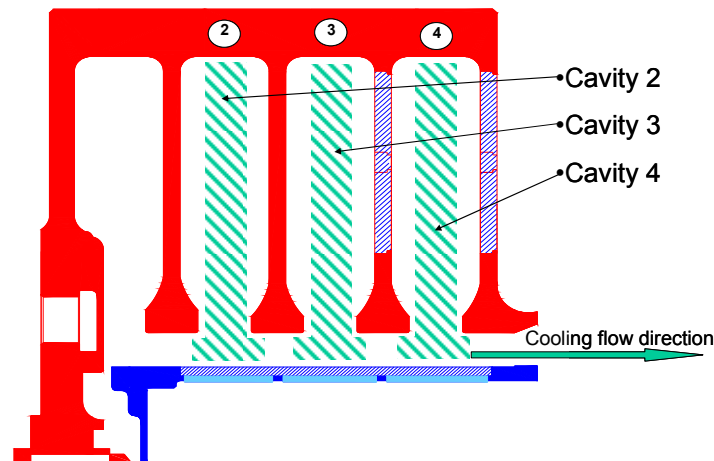


Fig 3. 24 Measurement zones for LDA V_z, V_ϕ in the Multiple Cavity Rig, Build 3

Due to the presence of the cobs, there was a limit on how close to the discs the measurements could be made. Had the optical access through the discs not been feasible in this rig, a useful modification on the through shaft LDA configuration would have been to allow the tube mirror angle to be varied in order to get the probe volume nearer to the disc wall without interference from the protruding cobs. This option would have however provided only valid tangential velocities as the axial velocity measurement would have been difficult to resolve with a tilted frame of reference, especially given the low radial and axial measured velocity values described in Chapter 6.

For the reported series of experiments, the measurement zone across the cavities were in the range $0.345 < z/s < 0.671$. Measurements across the whole radius of the cavities from the shaft surface to the shroud were possible, with the limitation of proximity to the surfaces only due to the excessive glare from the relatively long probe volume (~ 4 mm) getting in proximity with the surfaces. In these experiments, the closest points of measurement from the shaft, cob or shroud were set at least 3 mm away from the surfaces, giving a measurement range within the cavities of $0.264 < r/b < 0.955$. Similarly, measurements under the cobs were possible but limited to a range $0.264 < r/b < 0.305$.

The accuracy of positioning was considered as ± 0.5 mm in the longitudinal and radial axis.

Measurements of the tangential and radial velocity components were achieved by positioning the LDA probe off axis so that the beams would shoot across the discs' windows as shown in Fig 3.25. Optical access through the discs had already been integrated into the manufacturing of Build 3 (Alexiou 2002).

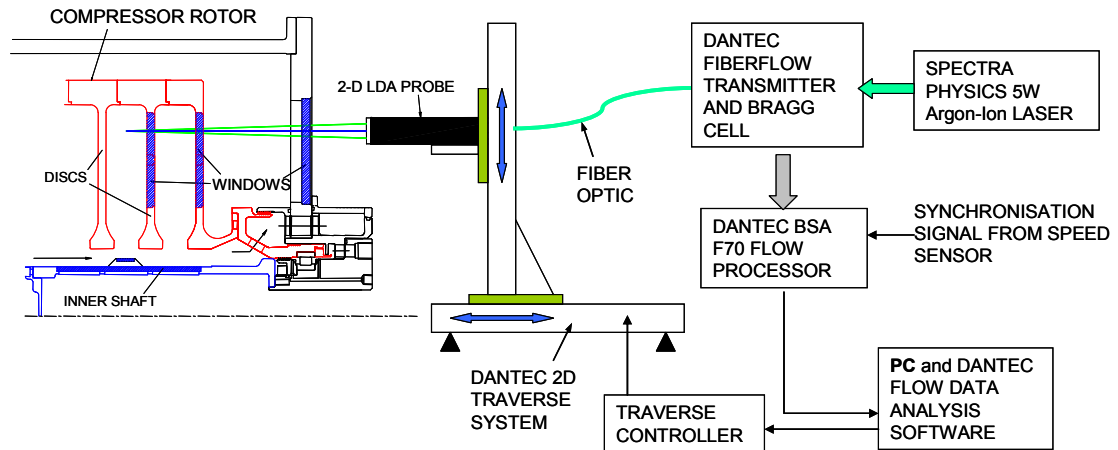


Fig 3. 25 Schematics of LDA traverse system for measurements of V_θ and V_r velocity components

A DANTEC 2D traverse system, implemented by the author, allowing 600 mm travel on both axes and an adjustable probe mount were used for the positioning of the probe volume as shown in Fig 3.26.

The volume taken by the traverse system and its location according to the rig required the re-location of the laser and ancillaries to a different bench pictured in Fig 3.27.

Position referencing of the probe volume according to the rig was achieved by the insertion of a locating target disc instead of the rig casing optical access window as shown in Fig 3.28.

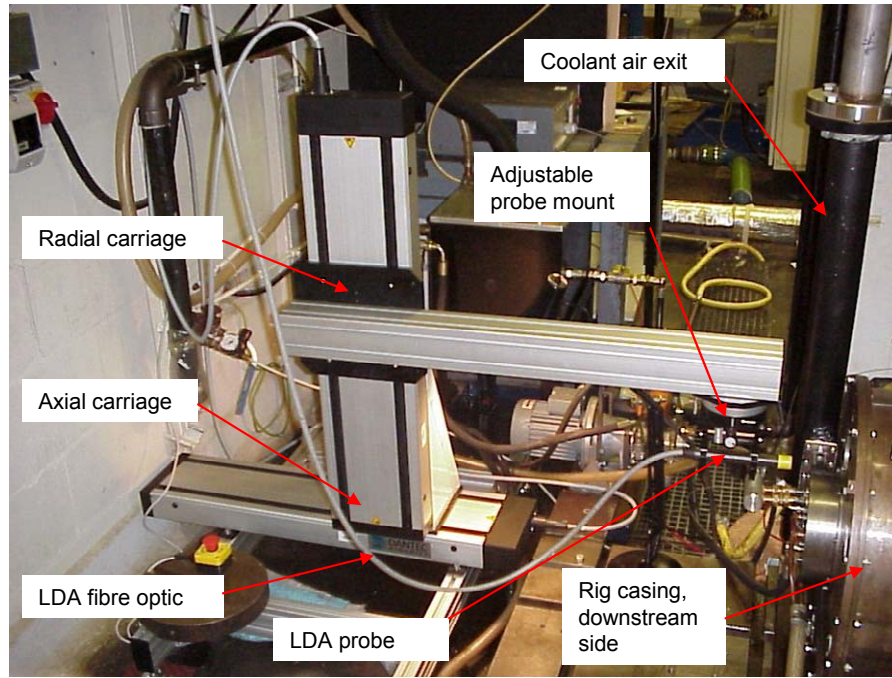


Fig 3. 26 External view of the traverse system for measurements of V_ϕ and V_r velocity components

The angular referencing of the target disc was maintained by a common punch mark alignment on the casing and the target. Longitudinal alignment of the probe was checked by actuating the traverse system along its full longitudinal displacement range and monitoring that the beam pairs would not drift from their horizontal and vertical respective planes represented as lines in the reference target. The accuracy of positioning of the probe volume was considered as ± 0.5 mm in the longitudinal and radial axis. The measurement zones inside cavities 3 and 4 were limited by the discs windows' radial locations and by a minimum clearance of the length of the probe volume from the discs' surfaces. The measurement zones in cavities 3 and 4 were determined as $0.067 \leq z/s \leq 0.964$ and $0.565 \leq r/b \leq 0.925$ as shown in Fig 3.29.

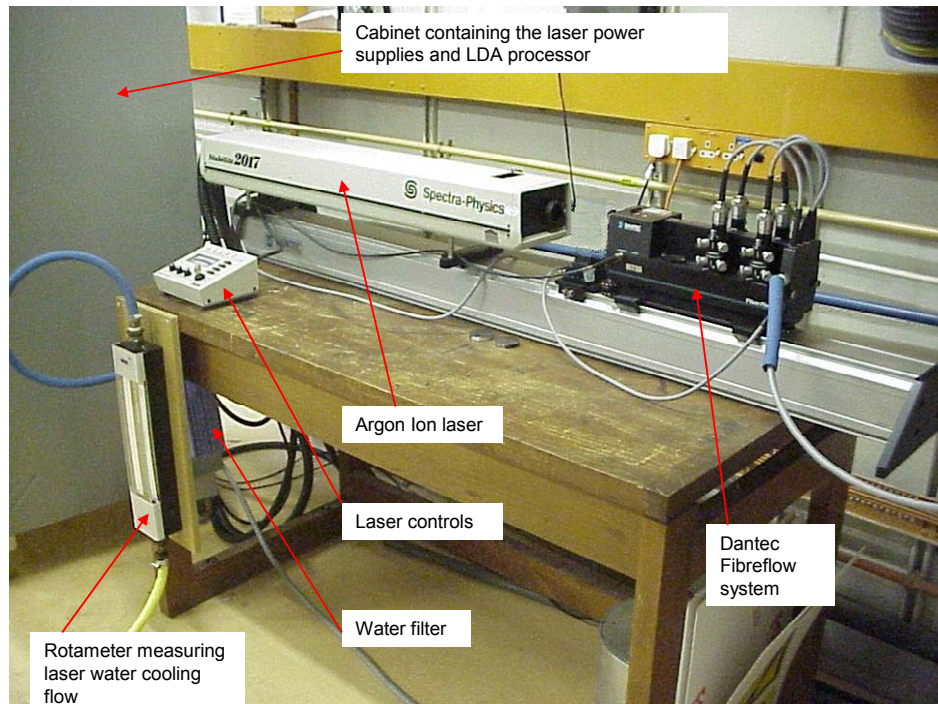


Fig 3. 27 View of the laser bench as installed for the measurements of V_ϕ and V_r velocity components

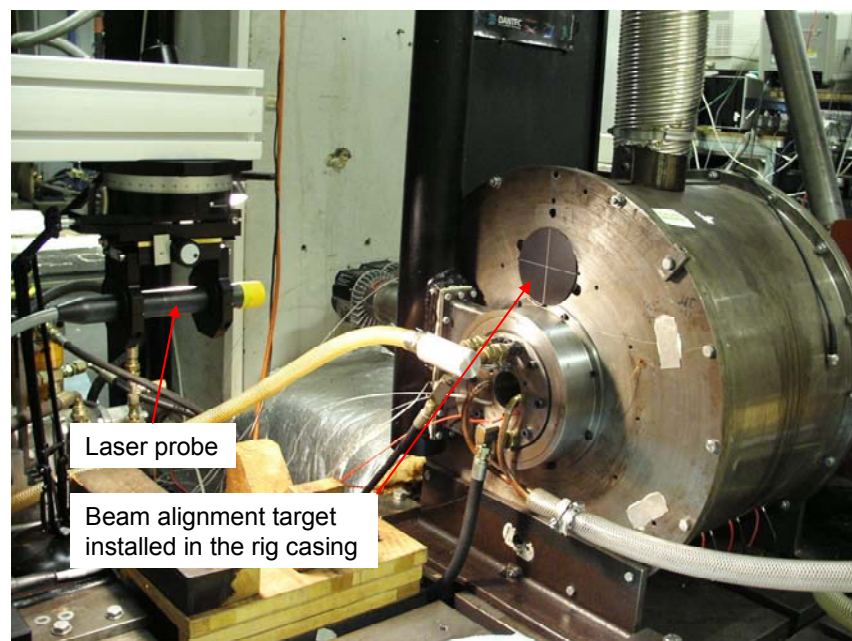


Fig 3. 28 View of the beam alignment target for the measurements of V_ϕ and V_r velocity components

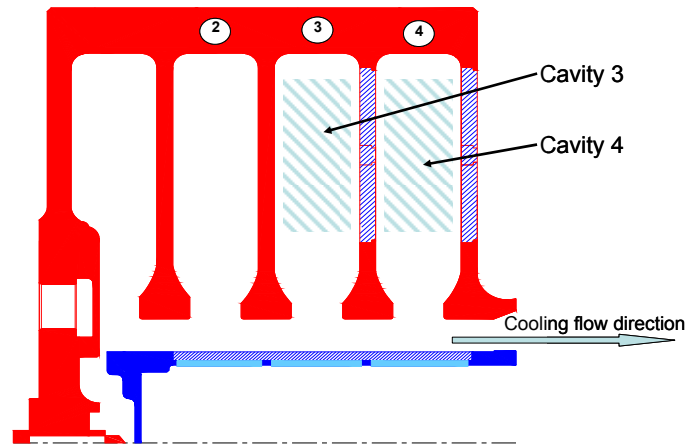


Fig 3. 29 Measurement zones for LDA V_r, V_ϕ in the Multiple Cavity Rig, Build 3

Before each experimental run, the traverse system was aligned and the probe volume position referenced. All optics were thoroughly cleaned with an optical grade lint free chamois pad soaked in methylated spirit. The internal surfaces of the optical access windows required the shaft to be dismantled in order to allow access for cleaning. The LDA set-up routine is described in Appendix 3.4 for the case of the longitudinal traverse configuration measuring the axial and tangential velocity components. On a weekly basis, the laser source output was checked and the mirrors and transmitting fibre optics re-aligned as required.

4. Experimental procedure and data processing

4.1 Experimental procedure

4.1.1 Pre-start

As mentioned in Chapter 3, the rig's LDA system, including the optical access windows, was cleaned, tuned and referenced prior to an experimental session (typically the day before). While the shaft was removed for cleaning of the windows, the rig's downstream roller bearing was inspected together with its pressurised lubrication system, and topped-up as necessary (Shell Tellus 22 oil). The air supply cooling water was circulated between the roof tank (5 m³ capacity) and the underground main water tank (10 m³ capacity) in order to ensure the water temperature was uniform. This required the circulating pump to be run for several hours, typically one full afternoon, especially if the rig had not been run for several weeks and the outdoor temperature was very different from the ambient temperature underground of the test cell. The pre-start operations on a day of running would typically consist in powering up the laser first according to the procedure in Appendix 4.1, so that the laser source temperature would have time to stabilise. The rig's air cooling and compressor were prepared following the procedure described in Appendix 4.2.

The rig air supply line operating positions were checked and set to isolate the rig through a by-pass line used at first when operating the compressor to clean-up the pipes of the air system for a few minutes before the rig would be pressurised as described in Appendix 4.3. All power supplies, control computers, data loggers and electrical ancillaries were powered-up and relevant control software loaded-up. When ready to start, the lab ventilation, shop air cooling to the slip ring unit, and Howden cooling air compressor would be started-up as described in the pre-start checklist in Appendix 4.4. The LDA atomiser air supply was provided by a standalone Hydrovane compressor. The atomiser was pressurised in standby non-atomising mode.

4.1.2 Start up and running

With the rig control software monitoring the temperatures, supply pressures, rotational speed and flow rates, the rig would be first pressurised to a minimum pressure by closing the air supply bypass and adjusting the compressor pressure control valve according to the procedure described in appendix 4.5 until the desired value of Re_z for the experiment has been achieved. The oil lubrication pump would then be switched on as there would no longer be a risk of oil leaks inside the rig through the labyrinths seals as the rig's rotor was now pressurised. The rig could then be set to rotate at the required speed to achieve the target value of Re_ϕ . The shroud heaters would then be switched on. The full starting procedure is described in Appendix 4.6.

While the rig heated up, the air supply temperature would be monitored and adjusted to remain constant by varying the water supply rate on the heat exchanger. The first thermal steady-state would typically be achieved after 45 minutes to 1 hour (30 to 45 minutes for subsequent conditions) and was confirmed by monitoring a change in surface temperatures of less than 0.2 °C in a period of 5 minutes. The rig shut down procedure is described in Appendix 4.7.

4.1.3 Data acquisition

Once the thermal steady-state of the desired experimental condition was achieved, the data acquisition could be initiated. The pressure logging consisted of running the acquisition program that operated remotely the scanivalve unit (scani.bas). Three scans of the pressure channels were performed and the pressure results for each scan were recorded on the computer disk as a text file. The temperature and transducer signals were recorded from the rig monitoring program (orion.bas) previously mentioned. This was driving the Orion data logger remotely and would record 100 separate scans (each taking 1.3 seconds) with a delay of 2.5 seconds between scans. The logged data was written into a test file to be used for post-processing. The standard deviation from the steady-state temperature logging was found to be always less than 0.2 °C, and typically less than 0.1°C across all the experiments.

The velocity measurements through the shaft in the V_ϕ, V_z velocity components were performed on typically 37 different points in the rig as represented in Fig 4.1.

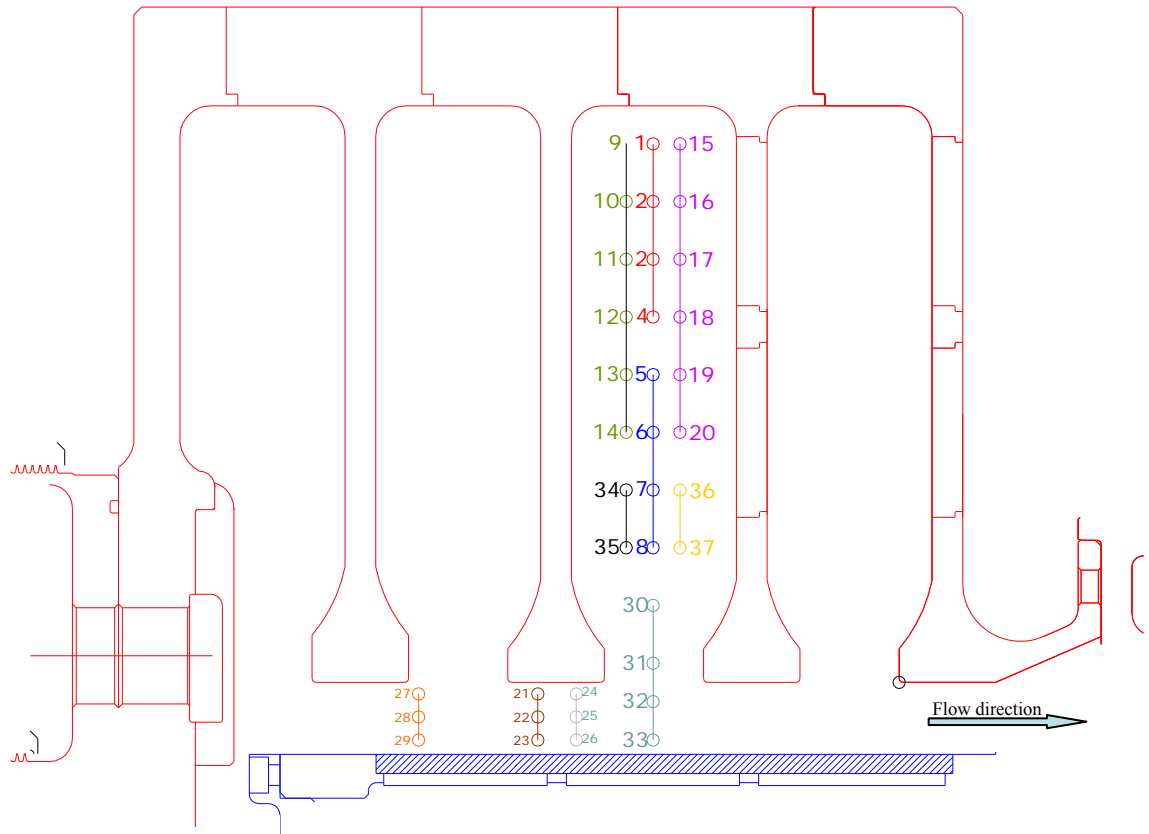


Fig 4. 1 LDA measurement points for V_ϕ, V_z configuration

4.2 Heat transfer data processing

Following is a description of the equations employed for the heat transfer related calculated parameters and the way they were derived from dedicated software.

4.2.1 Data processing method

Experimental measurements described in Chapter 3 resulted in output files comprising steady state data averaged over 100 scans for the temperature and rotational speed data, and three scans for the pressure related data, together with standard deviations. Cooling air mass flow rates were calculated from the orifice plates data on the rig and appropriate leakage through the rig deduced according to the method described in Alexiou (2000).

From these sets of data, the rotational Reynolds number for the discs was defined as:

$$Re_{\phi} = \frac{\Omega \times b^2}{\nu} \quad \text{Eq. 4.1}$$

Where ν is evaluated for $T_{in} = \frac{T_{32} + T_{33}}{2}$

The axial Reynolds number was defined as:

$$Re_z = \frac{W \times d_h}{\nu} \quad \text{Eq. 4.2}$$

Where d_h is the hydraulic diameter of the cavities annulus inlet: $d_h = 2(a - r_s)$,

$W = \frac{\dot{m}}{\rho \times \pi(a^2 - r_s^2)}$ is the mean axial velocity of the cooling flow in the intra cavity annulus.

The Rossby number was expressed as:

$$Ro = \frac{W}{\Omega \times a} = \frac{Re_z \times b^2}{2 \times Re_{\phi} \times a^2 \times \left(1 - \frac{r_s}{a}\right)} \quad \text{Eq. 4.3}$$

The conduction solution method used to derive local disc's heat transfer using finite differences in order to solve the two-dimensional steady-state conduction equation in cylindrical coordinates below:

$$\frac{\partial^2 T}{\partial z^2} + \frac{\partial^2 T}{\partial r^2} + \frac{1}{r} \times \frac{\partial T}{\partial r} = 0 \quad \text{Eq. 4.4}$$

Where T is the temperature at radius r and axial coordinate z

A calculation domain grid was defined through the cross section of the disc limited by the surface thermocouples as shown in Fig 4.2.

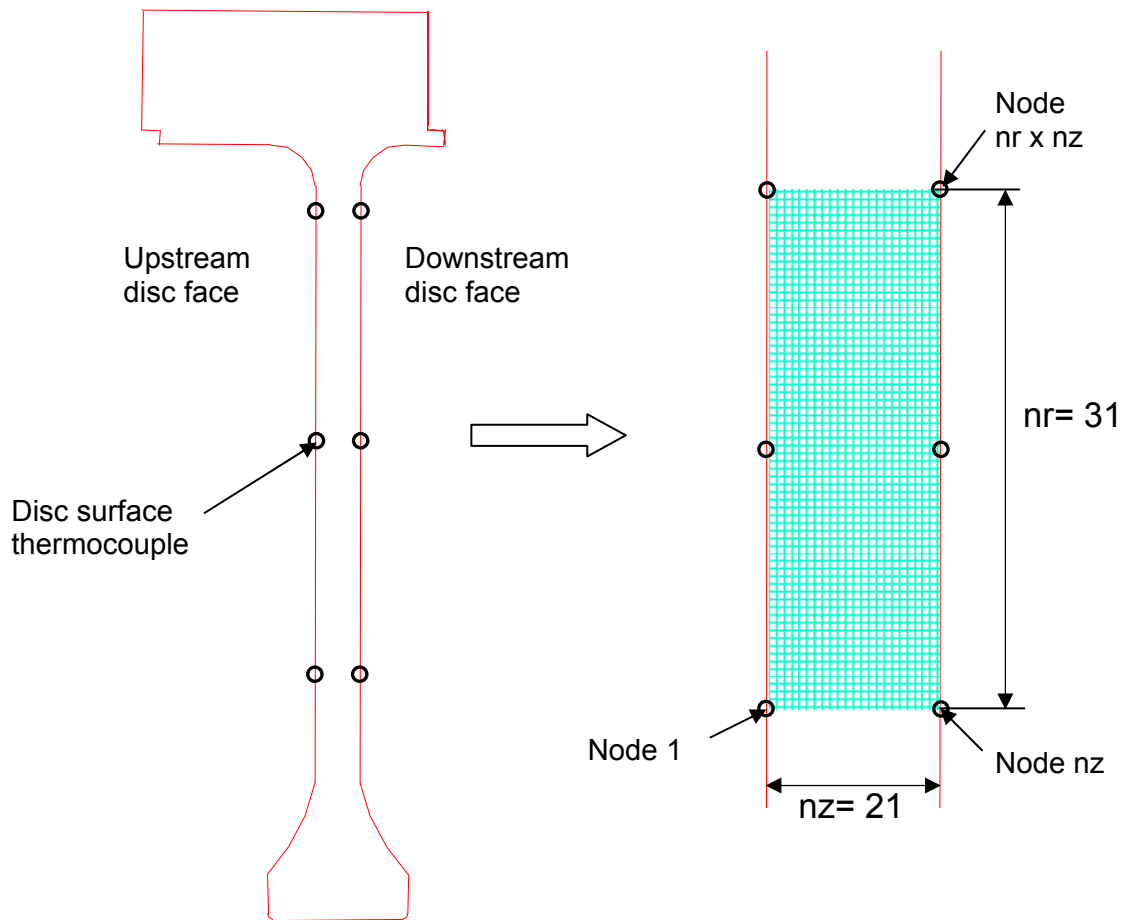


Fig 4. 2 Disc heat transfer grid

The cross section was divided into a 21 x 31 node matrix with the node numbering shown in Fig 4.2.

The two-dimensional steady-state conduction equation (Eq. 4.4) can be written in finite difference terms for a point node p within the calculation grid as:

$$A_n \cdot T_n + A_s \cdot T_s + A_p \cdot T_p + A_e \cdot T_e + A_w \cdot T_w = 0 \quad \text{Eq. 4.5}$$

Where: n, s, e, w, refer to the node relative to node p as defined in Fig 4.3

$$A_n = \left\{ \frac{1}{\delta r^2} + \frac{1}{2 \times r_{(p)} \times \delta r} \right\}$$

$$A_s = \left\{ \frac{1}{\delta r^2} - \frac{1}{2 \times r_{(p)} \times \delta r} \right\}$$

$$A_p = \left\{ \frac{-2}{\delta r^2} - \frac{2}{\delta z^2} \right\}$$

$$A_e = \left\{ \frac{1}{\delta z^2} \right\}$$

$$A_w = \left\{ \frac{1}{\delta z^2} \right\}$$

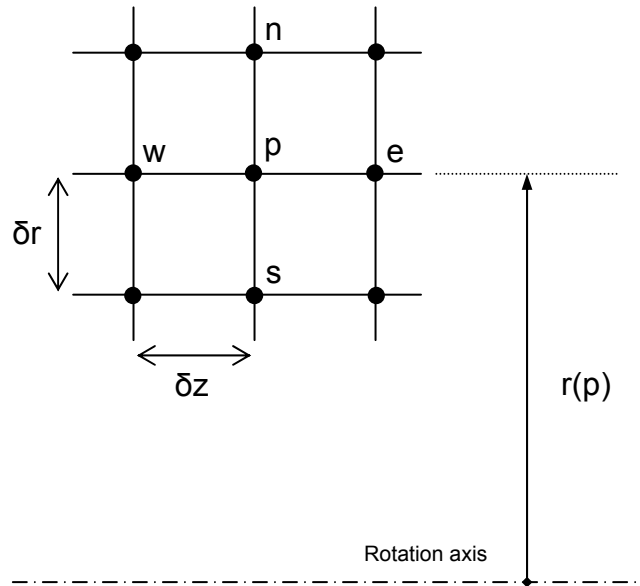


Fig 4. 3 Finite difference grid terminology

The boundary nodes on the periphery of the grid are directly defined by the surface temperature interpolated from the measured surface temperatures. The disc's surface temperatures were interpolated from the three surface thermocouples (e.g. between thermocouples 3, 4, and 5 for the upstream face of Disc 2, shown in Fig 3.13) using a third order polynomial curve fitting as explained in detail in Chapter 5. Inside disc temperatures at the inner and outer radii of the discs (e.g. between thermocouples 5 and 9 for the inner radius of disc 2) were derived by linear interpolation.

All the nodes temperature expressions put together formed a system of 21 x 31 simultaneous algebraic equations which were solved by Gaussian elimination. Once the internal temperatures in the grid were derived, the local total heat fluxes on the discs surfaces could be expressed by second order backward difference as:

$$q_T(j) = -k \left(\frac{-3T_j + 4T_{j+1} - T_{j+2}}{2\delta_z} \right) \quad \text{Eq 4.6}$$

Where k is the thermal conductivity of the disc material, δ_z is the axial distance between two nodes, T_j is the local disc temperature surface, T_{j+1} and T_{j+2} are respectively the next and second adjacent axial cell temperatures into the disc surface

A variation of the conduction solution method above (Eq. 4.4 to 4.6) was also employed to numerically derive the local total heat fluxes, based on the assumption of an adiabatic mid-plane in the disc thickness (following on Farthing et al. 1992b and Alexiou 2000 as explained in section 2.3.2). This was referred to as the fin solution and was solved using the same method as the two-dimensional solution above, only the adiabatic plane assumption was fulfilled by considering both faces of the disc to have a symmetrical temperature profile boundary condition. An average of the upstream and downstream local temperature profiles was therefore applied as boundary condition on both disc faces (Eq. 4.7) in order to fulfil the symmetrical temperature distribution and numerically resolve the fin solution of local heat fluxes.

$$T_{fin}(r) = \frac{T_{upstream}(r) + T_{downstream}(r)}{2} \quad \text{Eq. 4.7}$$

A radiation correction was then applied to the total calculated heat flux values for the 2D conduction solution and the fin conduction solution in order to deduce the convective local heat fluxes on the discs' surfaces such as:

$$q_{conv} = q_T - q_{rad} \quad \text{Eq. 4.8}$$

Where q_{rad} is the estimated radiative heat flux

The radiative heat flux calculation in Cavity 3 was based on the grey body method described in Long (1999), applied to the same number of annular surfaces as there were surface grid nodes. The remaining surfaces on the discs were modeled as bigger annular surfaces, as shown in Fig 4.4.

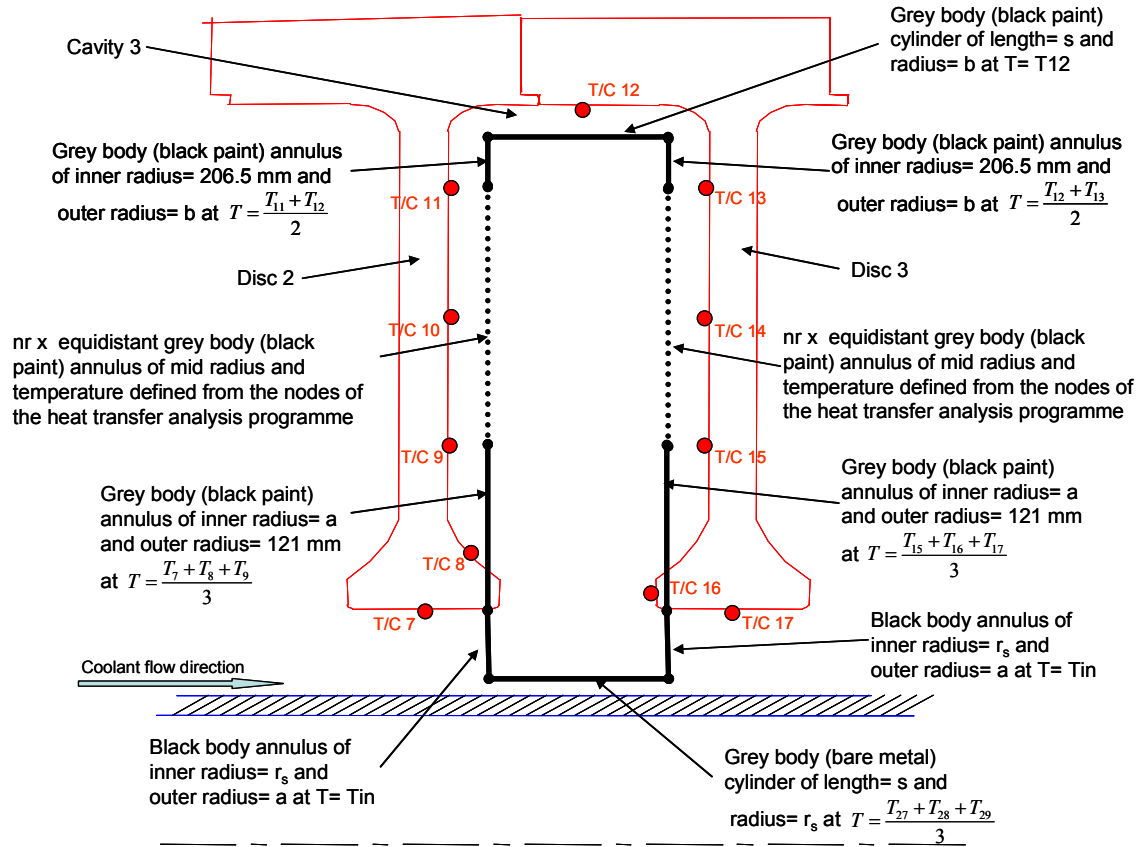


Fig 4. 4 Description of the radiation enclosure model of Cavity 3 (not to scale)

The view factors of the enclosure defined in Fig 4.4 were obtained from the catalogue of Howell (1982). The Stefan Boltzman constant was fixed at $\sigma=56.7 \times 10^{-9} \text{ W/m}^2\text{K}^4$, and the emissivity values used on the different sections of the cavity enclosure model were set as:

- Non-painted titanium: $\epsilon_1=0.2$ (from Alexiou 2000)
- Black-painted surfaces: $\epsilon_2=0.95$ (from Touloukian 1972)
- Annulus gap under discs: $\epsilon_3=1$ (black body)

The view factors were written in a square matrix form $F(i,j)$ of dimensions equal to the number of surfaces considered in the radiation model, where i denotes the index of the emitting surface in relation to the receiving surface j . "Self

viewing” surfaces, either flat or convex, $F(i,i)$, being not able to see parts of themselves, were therefore assigned a view factor $F(i,i)=0$. Shaft-to-disc annulus view factors were then defined from Rea (1975).

The shaft to shroud view factors could then be deduced using the summation rule (stating that the sum of the view factors in an enclosure equals to unity), hence:

$$F_{shaft,shroud} = 1 - \sum F_{shaft,all\ surfaces\ except\ shroud} \quad \text{Eq. 4.9}$$

After having defined the view factors from the shaft to all the other surfaces of the enclosure, the view factors from the enclosure surfaces to the shaft could be calculated from the reciprocity rule:

$$A_i \cdot F_{ij} = A_j \cdot F_{ji} \quad \text{Eq 4.10}$$

Where A_x is the area of the considered surface

The disc to disc view factors were then calculated from Minning (1979) and their counterparts also deduced by reciprocity. Then the summation rule was applied to obtain the discs to shroud view factors (and shroud to discs view factors by reciprocity). Finally, the summation rule was applied for the calculation of the shroud to shroud view factors.

From Long (1999) the radiation exchange between grey bodies in an enclosure is expressed as:

$$q_{rad,i} = \frac{E_{b,i} - J_i}{(1 - \varepsilon_i)/\varepsilon_i} = \sum_{j=1}^N F_{ij} (J_i - J_j) \quad \text{Eq. 4.11}$$

Where J_i is the radiosoty of the considered surface (W/m^2)

E_b is the surface black body emission (W/m^2)

$E_{b,i} = \sigma \cdot T_i^4$, with $\sigma = 56.7 \times 10^{-9} \text{ W}/(\text{m}^2 \cdot \text{K}^4)$

ε_i is the emissivity of the surface

N is the total number of surfaces of the radiation enclosure

The radiosities are determined from the second part of equation 4.11 for each surface which develops as (as an example for $i=1$ to 3 and $j=1$ to 3):

$$\begin{aligned} E_{b,1} &= \frac{1-\varepsilon_1}{\varepsilon_1} [0 + F_{12}(J_1 - J_2) + F_{13}(J_1 - J_3)] + J_1 \\ E_{b,2} &= \frac{1-\varepsilon_2}{\varepsilon_2} [F_{21}(J_2 - J_1) + 0 + F_{23}(J_2 - J_3)] + J_2 \\ E_{b,3} &= \frac{1-\varepsilon_3}{\varepsilon_3} [F_{31}(J_3 - J_1) + F_{32}(J_3 - J_2) + 0] + J_3 \end{aligned} \quad \text{Eq. 4.12}$$

Re-arranging into:

$$\begin{aligned} \left[\frac{1-\varepsilon_1}{\varepsilon_1} (0 + F_{12} + F_{13}) + 1 \right] \cdot J_1 + \left[-\frac{1-\varepsilon_1}{\varepsilon_1} \cdot F_{12} \right] \cdot J_2 + \left[-\frac{1-\varepsilon_1}{\varepsilon_1} \cdot F_{13} \right] \cdot J_3 &= E_{b,1} \\ \left[-\frac{1-\varepsilon_2}{\varepsilon_2} \cdot F_{21} \right] \cdot J_1 + \left[\frac{1-\varepsilon_2}{\varepsilon_2} (F_{21} + 0 + F_{23}) + 1 \right] \cdot J_2 + \left[-\frac{1-\varepsilon_2}{\varepsilon_2} \cdot F_{23} \right] \cdot J_3 &= E_{b,2} \\ \left[-\frac{1-\varepsilon_3}{\varepsilon_3} \cdot F_{31} \right] \cdot J_1 + \left[-\frac{1-\varepsilon_3}{\varepsilon_3} \cdot F_{32} \right] \cdot J_2 + \left[\frac{1-\varepsilon_3}{\varepsilon_3} (F_{31} + F_{32} + 0) + 1 \right] \cdot J_3 &= E_{b,3} \end{aligned} \quad \text{Eq. 4.13}$$

Which re-organised in a general matrix form is expressed as:

$$[A] \cdot (J) = E_b \quad \text{Eq. 4.14}$$

Where $[A]$ is the coefficient matrix function of F_{ij} and ε_i for each J_i in vector (J)

Equation 4.14 results in a system of N simultaneous equations that can be solved to find the unknown J_i .

Once the radiosities J_i are determined, the first part of equation 4.11 is used to determine the value of q_{rad} for each of the surfaces of the cavity.

The local Nusselt numbers are calculated at each radius location from the corrected heat flux in equation 4.8 as:

$$Nu_r = \frac{q_{conv,r} \cdot r}{k \cdot \Delta T} \quad \text{Eq. 4.15}$$

Where $\Delta T = T_{(r)} - T_{ref}$ with $T_{ref} = T_{in} + \frac{\Omega^2 (b^2 - a^2)}{2 C_p}$ the adiabatic wall fluid temperature estimation from Long et al (2003)

The average Nusselt numbers on the discs calculation zones were defined from the area weighted parameters $q_{conv,av}$ and ΔT_{av} as:

$$Nu_{av} = \frac{q_{conv,av} \cdot b}{k \cdot \Delta T_{av}} \quad \text{Eq. 4.16}$$

With

$$q_{conv,av} = \frac{2\pi}{A} \int_{r_i}^{r_o} q_{conv,r} \cdot r \cdot dr \quad \text{Eq. 4.17}$$

$$\Delta T_{av} = \frac{2\pi}{A} \int_{r_i}^{r_o} \Delta T \cdot r \cdot dr \quad \text{Eq. 4.18}$$

Where A is the area between the inner radius of the calculation grid r_i and outer radius r_o .

The integrals in equations 4.17 and 4.18 were numerically calculated using Simpson's rule.

A one-dimensional conduction solution equation was used to derive the heat flux through the shroud, using the measured temperatures on the inner and outer surfaces of the shroud, respectively $T_{sh,i}$ and $T_{sh,o}$.

The shroud total heat fluxes were calculated as:

$$q_{T,sh} = \frac{k(T_{sh,o} - T_{sh,i})}{b \cdot \ln(D/2b)} \quad \text{Eq. 4.19}$$

The shroud convective heat flux was then corrected for radiation contribution as above such as:

$$q_{conv,sh} = q_{T,sh} - q_{rad,sh} \quad \text{Eq. 4.20}$$

The shroud Nusselt number is defined as:

$$Nu_{sh} = \frac{q_{conv,sh} \cdot (s/2)}{(T_{sh,i} - T_{ref}) \cdot k} \quad \text{Eq. 4.21}$$

With

$$T_{ref} = T_{in} + \frac{\Omega^2 (b^2 - a^2)}{2Cp} \quad \text{Eq. 4.22}$$

The shroud Grashof number is defined as:

$$Gr_{sh} = \frac{\rho^2 \cdot \Omega^2 \cdot b \cdot \beta \cdot \Delta T_{sh} \left(\frac{s}{2} \right)^3}{\mu^2} \quad \text{Eq. 4.23}$$

Where $\beta \Delta T_{sh} = \frac{1}{T_{in}} (T_{sh} - T_{in})$

4.2.2 Processing software

As mentioned in chapter 3, the recorded temperature and pressure data were pre-processed in a Q-BASIC programme described in Alexiou (2000) which gave average values of rotational speed, pressure and temperature from the recording scans (100 scans for temperature and 3 scans for pressure), together with the associated standard deviations values for the rotational speed and temperatures. A listing of this programme is provided in Appendix 4.8.

The mass flow through the rig was corrected for leakage through the labyrinth seals according to Curzons (1991). From this raw data file, an example of which can be found in appendix 4.9, the lab atmospheric pressure, test identification and comments were manually added for each experimental condition and the output files were collated onto an excel worksheet as a list.

With all the experimental data in a file list, the post-processing programme could be run. This programme was written in Matlab and would take the raw data from the list file in MCRexcel.xls, process the relevant heat transfer and flow related calculations and then write the results back into a results worksheet

in MCRexcel.xls where they could be further post-processed into standard test reports by using a Visual Basic code and worksheet template described later. The overall architecture of the post-processing software is shown in Fig 4.5.

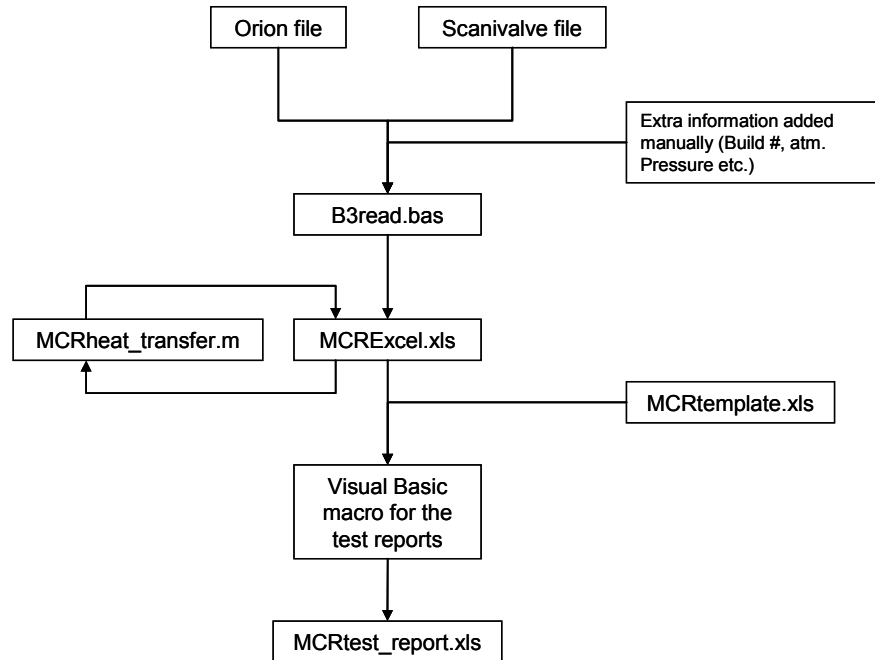


Fig 4. 5 Heat transfer post-processing software overall architecture

The main post processing programme named MCRheat_transfer.m allowed the calculation of the heat transfer parameters described in section 4.2.1, such as local heat fluxes and Nusselt numbers, Reynolds numbers etc. The calculated nominal values were bounded by calculated intervals of uncertainty on each parameters. Fig 4.6 shows a schematic of the structure of the MCRheat_transfer.m programme.

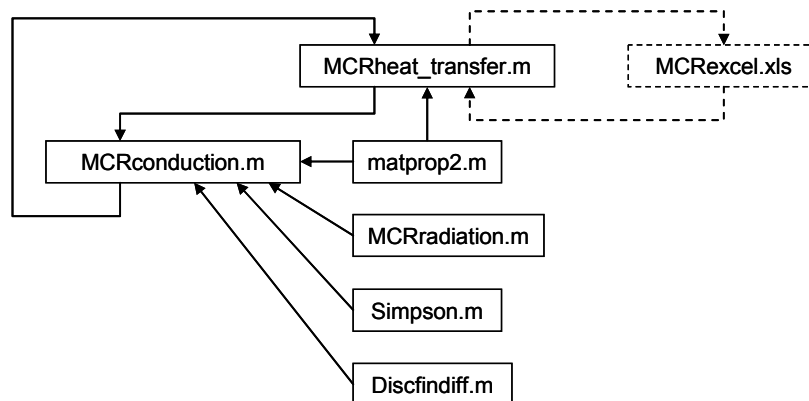


Fig 4. 6 Details of the subroutines of MCRheat_transfer.m

Following is a basic description of the written algorithm when performing a case of analysis (the reader is invited to refer to the programme listings in the appendix for further details and a complete list of the calculated parameters).

Firstly, running `MCRheat_transfer.m`, an initiation phase is performed where the cavity dimensions are declared and the grid for the conduction solution defined. The data labels for the results file are also declared. The raw data is read from the raw data worksheet in `MCRexcel.xls` and the tests are separated for each experimental case.

The experimental thermocouple data is then copied into a sub case matrix in the first row: this is the nominal data set for the considered experimental case. The following rows (2 to 101) are filled up with generated temperature variations from the uncertainty analysis on the experimental equipment described in detail in chapter 5. The uncertainty values for the experimental apparatus defined by the analysis are combined together with the calculated uncertainty of the measurement itself (from the 100 recording scans) and used to generate 100 sub cases around the nominal value with a normal distribution defined by the overall standard error deduced by the combined experimental and calculated uncertainties.

This “Monte Carlo” approach to the measured data allows the generation of a realistic set of calculations of uncertainties on the processed results of the experimental case directly, taking into account all experimental error sources, without introducing further uncertainty assumptions in the data processing stages. Standard uncertainties are derived from the results of the 100 generated sub cases for each calculated parameter and the final result of each processed datum is hence the nominal calculated value (first sub case) plus or minus the related uncertainty interval from the generated sub cases.

Once the nominal and randomly generated sub cases for the thermocouples and rotational speed values were generated, calculations for each sub cases are then initiated. The directly calculated values such as Ro , Re_ϕ , Re_z , etc. are obtained. The one-dimensional radial conduction solution for the shroud heat transfer parameters values were calculated from equations 4.18 to 4.21. Then, the relevant thermocouples values for the discs heat transfer finite difference calculations and the thermocouples radial locations are inputted into the `MCRconduction.m` subroutine (Appendix 4.11). The `MCRconduction.m`

subroutine allows the disc thermocouple temperature interpolation along the disc surfaces and discretises the interpolated temperatures for each of the outer nodes on the conduction solution grid. The number of thermocouples used in this interpolation and the degree of the polynomial curve fit used can be varied as discussed in chapter 5. Once the temperatures at the corners of the grid have been determined, the axial temperatures at the inner and outer radii of the grid are interpolated linearly. The temperature grid boundary conditions are now defined and the actual finite difference computations to define the temperature values within the grid can be initiated by invoking the `discfindiff.m` subroutine (Appendix 4.12). This subroutine solves the temperature grid from equation 4.5 and resolves the $n_z \times n_r$ system of equations by Gaussian pivot. As mentioned in 4.2.1, the fin conduction solutions use the same resolving method as above, the only difference being that the radially fitted temperatures are defined as the average of the temperature fits on both discs faces, applied to both surfaces. Thus creating a plane of symmetry by mirroring the temperature boundary conditions by the plane at mid-thickness of the disc.

Once the temperature for each grid node has been defined in the `discfindiff.m` subroutine, the total heat flux on the discs surfaces can be calculated by using the second order backward difference expression in equation 4.6. The radiative heat flux contribution is determined as explained in 4.2.1 by the `MCRradiation.m` subroutine which calculates the view factors, radiosities, and radiative heat fluxes according to the surfaces enclosure model in figure 4.4 (Appendix 4.13). It is worth mentioning again that the radiative heat fluxes are only calculated for the surfaces within Cavity 3. Therefore, only the surface heat flux for Cavity 3 upstream and Cavity 3 downstream are corrected for radiation. However, the computations showed that the contribution of radiative heat flux on the discs was typically less than 10% of the measured total heat flux on the disc's faces, and therefore were not deemed to significantly affect the overall results. With the heat fluxes defined and corrected as per equation 4.8, all the remaining results parameters such as Nu_{av} , $\beta \Delta T_{av}$, $T_{s,av}$, etc. can be calculated.

Back in the `MCRheat_transfer.m` programme, once the nominal and generated sub cases are processed, the output files are then organised by taking the nominal result value of each calculated parameter and by calculating the

associated standard deviation from the sub cases results. The 95% confidence semi-interval is then determined as:

$$U_{95} = \frac{s}{\sqrt{n}} \times 1.96 \quad \text{Eq. 4.24}$$

Where s is the standard deviation value of the considered parameter and n is the number of generated sub cases (100 in this instance).

Following these processes, the results data is labelled and copied into the MCRexcel.xls file in the format shown by the example in appendix 4.14 and in a separate list of non-dimensional and averaged values for each experimental case in appendix 6.2 and 6.3

An embedded Visual Basic routine into the MCRexcel.xls (listed in Appendix 4.15) file allows the creation of standard test report sheets from the template in MCRtemplate.xlt. For each experimental condition, a worksheet is created from the template model and named according to its test identification.

The template features pre-made plots and spaces for the experimental data results. The Visual Basic macro copies the calculated results of the tests into the right places in the test reports, hence filling the displayed values up and updating the plots with the related data of each experimental test. Examples of heat transfer test reports are shown in appendix 7.1. It should be noted that the data is provided as a nominal value and their 95% uncertainty interval, also reflected on the plots by the error bars.

The first page of the test report features the main operational and average heat transfer related parameters for the discs and shrouds.

The plot represents the measured disc and shroud thermocouple values and the fitted temperatures on the discs' grid nodes. Page 2 features plots of local Nu and heat flux h , along the disc's radius. Page 3 and 4 collect the numerical values related to the local temperatures and heat transfer on the discs faces.

4.3 Air velocity data processing

4.3.1 Data processing method

The post processing of the velocimetry data consisted mainly of the consideration of the measured data under two categories. The ensemble averaged post processing allowed the determination of averaged values over time of the validated bursts. Statistical data such as average velocity, RMS (Root Mean Square) and intervals of confidence were derived from this method. The time dependent post processing looks at the time sequence of the randomly arriving bursts of velocity values in order to characterise the cyclic properties of the flow through a spectral analysis.

The relevant post processing calculations for this study were readily integrated in the LDA post processing software and are briefly described below.

The mean value of each velocity component was defined (in the case of velocity component u):

$$\bar{u} = \sum_{i=0}^{N-1} \eta_i \cdot u_i \quad \text{Eq. 4.25}$$

Where N is the number of validated bursts collected on the Doppler channel u , and η_i is a statistical weighting factor.

The weighting factor η_i for the determination of ensemble averaged statistics was taken as the transit time weighted expression defined as:

$$\eta_i = \frac{t_i}{\sum_{j=0}^{N-1} t_j} \quad \text{Eq. 4.26}$$

Where t_i is the transit time of the i 'th particle crossing the measuring volume.

Transit time weighting ensures a bias free method of performing statistical averages on potentially statistically dependent samples, as will be explained in Chapter 5.

The transit time weighting bias correction avoids the necessity of having to use a dead time burst recording method that would involve the fastidious determination of the integral time scale of the flow in order to pace of the burst data recording by a period of two times the integral scale of the flow in order to ensure statistical independence between the samples. Moreover, the samples data rate would be largely reduced by the use of dead time recording, which would further add difficulties to the experiments.

The variance of the average velocity sample is defined as:

$$\sigma^2 = \sum_{i=0}^{N-1} \eta_i (u_i - \bar{u})^2 \quad \text{Eq. 4.27}$$

Steady turbulent flows can be treated as a time averaged flow on which is superimposed a turbulent time varying component. The RMS value of the velocity sample is defined as:

$$\sigma = \sqrt{\sigma^2} \quad \text{Eq. 4.28}$$

And the turbulence intensity of the flow is expressed as:

$$T_u = \frac{\sigma}{\bar{u}} \times 100\% \quad \text{Eq. 4.29}$$

An angular error between the horizontal marker of the rig beam target and the probe in the V_r/V_ϕ configuration was found after the measurements had been made. A precise orientation of the beams is important especially when one measured velocity component is considerably greater than another. This was the case with the radial velocity measurement, as explained later in Chapter 5. The angle declination was measured on the rig as $\alpha = -1.389^\circ \pm 0.0135^\circ$ with a precision machining spirit level on the rig according to the coordinate system in Fig 4.7.

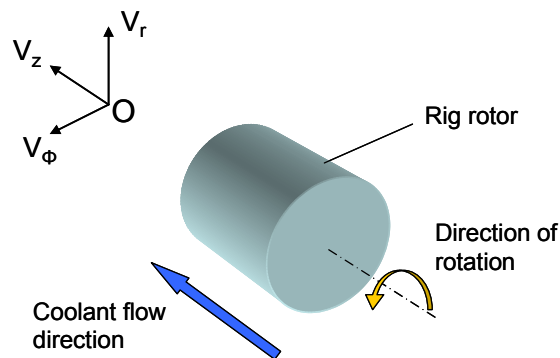


Fig 4. 7 Schematics of the coordinate system employed for the V_r, V_ϕ LDA measurements

The correction of the measured data from the probe angular error involved the application of a coordinate transform on the measured velocities. The two velocity channels samples had to be made coincident by filtering samples by arrival times to each other, considering two velocity samples would originate from the same particle if the arrival times would be within a time window set as the probe volume diameter divided by the highest velocity present in the flow as recommended in DANTEC (2004).

The rotation coordinate transform matrix applied to the sampled velocities data was expressed as:

$$\begin{pmatrix} x' \\ y' \\ z' \end{pmatrix} = \begin{bmatrix} \cos \alpha & \sin \alpha & 0 \\ -\sin \alpha & \cos \alpha & 0 \\ 0 & 0 & 1 \end{bmatrix} \cdot \begin{pmatrix} x \\ y \\ z \end{pmatrix} \quad \text{Eq. 4.30}$$

With $\alpha = -1.389^\circ$ and $z=0$ as the measurements were two-dimensional.

A spectral analysis was performed in order to highlight multiple flow cores in cavities as previously observed by Farthing et al. (1992a) and Owen and Powell (2004). The tangential component of velocity within the cavities was considered. DANTEC's post processing software integrates a Fast Fourier Transform functionality based on spectral analysis algorithms in order to resolve the time based sampled velocities into the frequency domain. The spectral analysis solver turns the samples discrete velocities into a waveform signal that in turn decomposes into a series of frequencies by the use of Fast Fourier Transforms. The Fast Fourier Transform algorithm used requires input samples equally spaced in time in order to operate. Because the nature of the LDA sample collecting being inherently random, a conditioning of the measured data is required prior to running the Fourier analysis. A re-sampling technique called "sample and hold" is applied to the raw sampled data. This consists in dividing the time domain into equally spaced bins and assuming the sampled velocity in the bins to remain constant at the latest burst (true) sampled value until another burst is present within the time bin. This is illustrated in Fig. 4.8, showing the randomly arriving burst velocity samples and the generated re-sampled values over time.

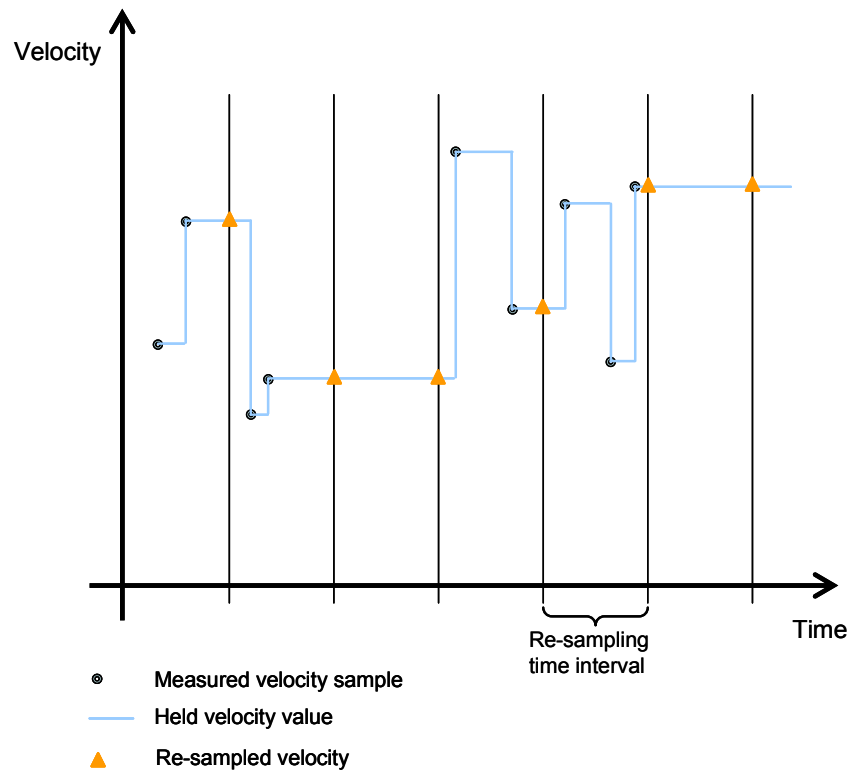


Fig 4. 8 Principles of the Sample and Hold re-sampling method for the LDA processing

It can be noticed from Fig. 4.8 that in the event of discontinuous measurements, re-sampled values will be generated at the last known value until new information about the flow is available. On the other hand, if several bursts are present within a time bin, only the last burst value within the time bin will be kept as the re-sampled value and information about the other bursts will be lost. The duration of the re-sampling interval is an important parameter to be adjusted when post processing the time series data as it will directly affect the rate of rejected data due to re-sampling. The re-sampling rate applied to the calculation process was determined as being twice the desired maximum frequency of the calculated spectrum, chosen as five times the mean data rate (Dantec 2005).

In the instance of flows measured through the shaft (V_z, V_ϕ configuration), the spectral analysis method described above is immediately applicable as the measurements are continuous.

However, when measuring through the passing rotor windows (V_r, V_ϕ configuration) the mechanical shuttering provided by the passing windows fools the re-sampling phase of the spectral analysis algorithm by filling up the non-measuring periods (in between windows) with the last measured values data at the re-sampling rate.

The FFT solver therefore picks up the windows passing frequency as dominant and internal filters damp the flow frequency out.

The flow could be re-composed from the measured data of each window added up over time by using the encoder time stamp of the rotor, but knowing that the cavity flow has a slippage from solid body rotation, the superimposed collected data per window would not be relevant as discontinuous in phase.

Moreover, following a discussion on this matter with a DANTEC representative (private conversation with R. Jariczewski 04/08/2005), it was realised that the spectrum analysis software requires a minimum of 1.1 to 1.2 times the flow cycle to be present in a given measurement period in order to be able to be detected and processed by the software. For instance, a window passage, when filtered into a measurement time period, can be for a rotational speed of 2700 rpm found to last about 0.5 ms resulting in a flow frequency of 2 kHz and above that can be processed. This is well beyond the minimal range of ~40 Hz to be expected at this specific operating condition.

Finally, considering a spectral analysis from a single window passage also would not be relevant due to the small amount of samples collected per window passage (between 1 and 15 typically).

Phasing of the collected samples in reference to the flow was attempted in order to improve the samples density over one flow resolution. Adaptive time stamping methods are required in order to re-organise the data collected over hundreds of cycles from the slipping rate of the flow measured on the time-averaged tangential velocity component. This method is considered to present the potential to provide possibly the best sample densities from measured data over a long period of time but could not be applied successfully within the scope and time constraints of the present project.

A practical solution to the spectral analysis from data collected through passing windows was suggested by Long (2004) who wrote a simple averaging programme in order to average the sample data per window passage into one value. Applied to a full LDA measurement, the data set obtained was a regularly sampled (one per window passage) smoothed data series over the whole test duration which was readily usable by the FFT software.

This method helped in obtaining a successful operation of the spectral analysis software in order to derive information on cavity flow structure.

4.3.2 Data processing software

As previously mentioned, most of the post processing related to the Build 2 and 3 LDA measurements was executed directly on the post processing modules of the DANTEC LDA software. The software allowed the production of ensemble averaged velocities and statistics for each measured point of each experiment. The data was then integrated into a specific Excel based test report template for uniformity of display and inclusion of the associated rig operation specific parameter. The non-dimensional speeds were plotted against the relevant rig's non-dimensional location and the 95% interval of confidence was displayed for each data point as an "error bar".

These test reports provided a rational and comparable set of data for archiving and data analysis. An example of an LDA test report can be found in Appendix 4.16.

The spectral analysis performed for several experimental conditions of Build 3 was executed by averaging the data as explained in 4.3.1 using `avenoco.bas` (reproduced in Appendix 4.17). The obtained data sets were imported back into the LDA software for frequency domain calculations. The resulting spectra were saved as numerical lists and plots, examples of which can be found in Appendix 4.18.

5. Data analysis

This chapter provides an estimation and discussion on the error and uncertainties associated with the measurement and analysis of the heat transfer and flow velocity data.

5.1 Heat transfer data analysis

5.1.1 Heat transfer measurements error

The error on the calculated heat transfer data was assessed by the use of a plain disc inverse fin calculation model, following a method described in the literature, notably Owen (1979), Alexiou (2000), Cooke (2007) and Patounas (2007). The inverse fin solution model employed used a modified algorithm from Patounas (2007) in order to generate the temperature profile on a plain disc that would return from a one-dimensional fin solution a fixed heat transfer coefficient, h_c , along the surface of a plain disc (thsol_MCR1.m in Appendix 5.1). The disc dimensions were taken as the disc's dimensions encountered on the Multiple Cavity Rig. The temperatures of the base and extremity of the modelled fin were inputted from typical experimental values, respectively $T_b = 372$ K and $T_e = 320$ K. The constant heat transfer coefficient, h_c , was given a value $h_c = 30$ W/(m².K). The generated temperature profile along the modelled plain disc can be seen in Figure 5.1.

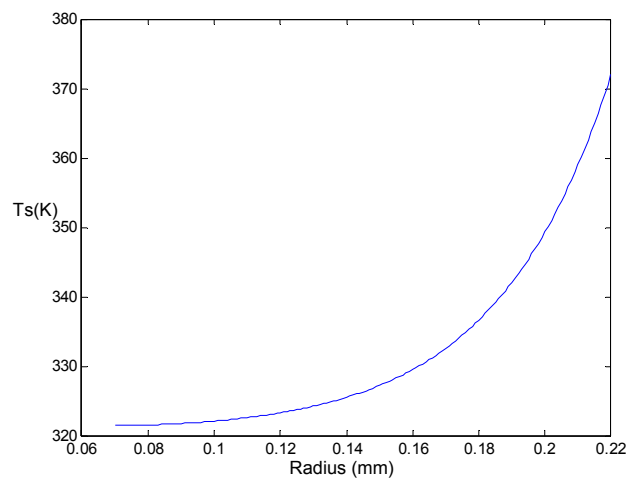


Fig 5. 1 Generated surface temperature profile of plain disc inverse fin solution model

The inverse fin model was checked by inputting the temperature profile back into a fin solution calculation along the length of a plain disc (finsol3_MCR1.m in Appendix 5.1). The temperature profile was also decomposed into discrete temperature values at known radii in order to represent the feed values from thermocouples in an experimental situation. The number and radius locations of the considered simulated thermocouples were chosen to be representative of thermocouples 7, 8, 9, 10, 11 and 12 related to Disc 2 in the Multiple Cavity Rig. Various curve fits were tested on the six simulated thermocouples and the heat transfer coefficient outputs compared in Figure 5.2.

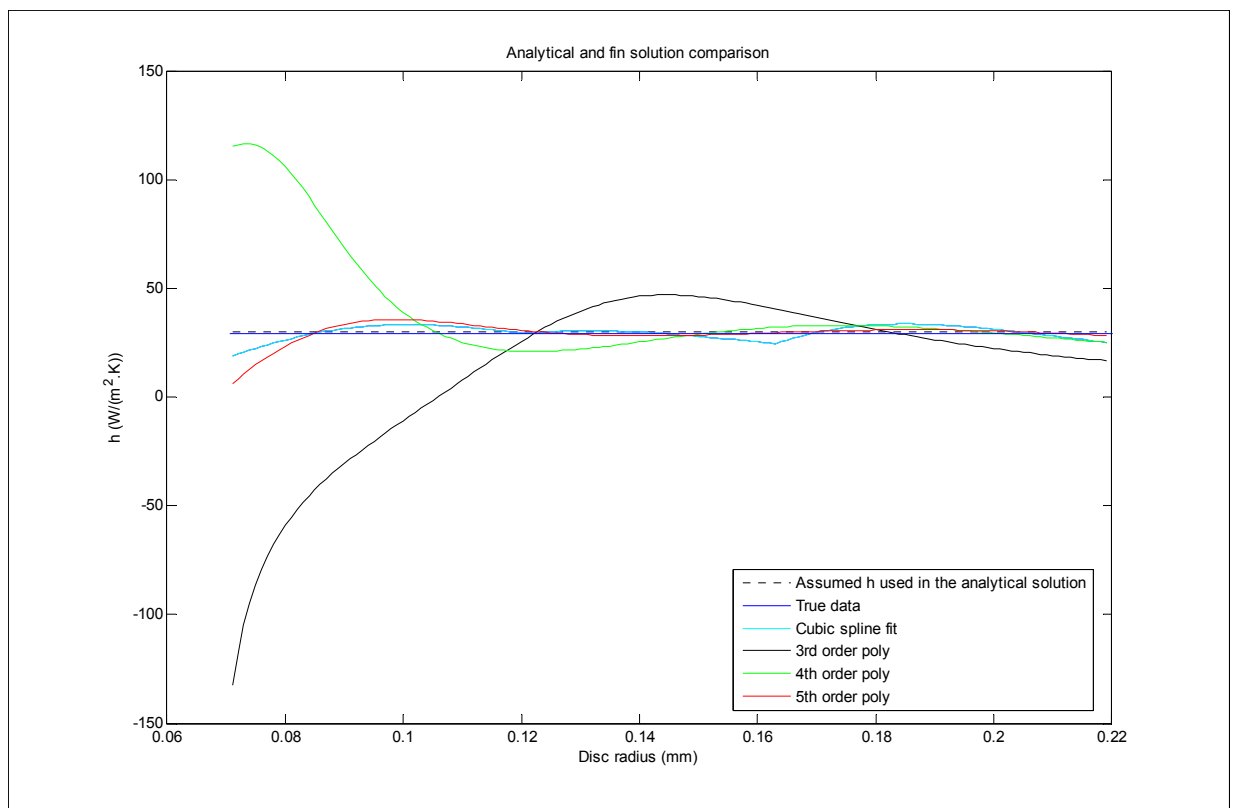


Fig 5. 2 Comparison of plain disc fin solution local heat transfer coefficient outputs from discrete temperature fits

From figure 5.2 it can be noticed that inputting the inverse fin solution 'true' temperature profile back into the fin solution results in the constant expected output values of $h_c = 30 \text{ W/(m}^2\text{.K)}$.

As already noticed by Alexiou (2000), Cooke (2007) and Patounas (2007), the highest order least square polynomial curve fit shows to return the best heat transfer coefficient results when solving from the modelled thermocouple data.

This can simply be explained by the minimisation of temperature bias at the N thermocouple locations resulting in the use of a $N-1$ order of polynomial which accounts for N pivoting points. Therefore, for the case of least square fitting of 6 thermocouples with a 5th order polynomial, each thermocouple point temperature value will be exact and the interpolated temperatures in between will be expected to be at their closest fit. Table 5.1 compares the resulting average heat transfer coefficient values h_{av} for each type of fitting considered in the present case.

| h_c Value $W/(m^2.K)$ | Analytical | | 3 rd order polynomial | | 4 th order polynomial | | 5 th order polynomial | | Cubic spline | |
|-------------------------------|-------------------------|---------------------|-------------------------------------|---------------------|-------------------------------------|---------------------|-------------------------------------|---------------------|-------------------------|---------------------|
| | h_{av} $W/(m^2.K)$ | % delta Vs h_c | h_{av} $W/(m^2.K)$ | % delta Vs h_c | h_{av} $W/(m^2.K)$ | % delta Vs h_c | h_{av} $W/(m^2.K)$ | % delta Vs h_c | h_{av} $W/(m^2.K)$ | % delta Vs h_c |
| 30 | 30.002 | 7×10^{-3} | 23.930 | 20.23 | 29.367 | 2.11 | 29.808 | 0.64 | 29.513 | 1.62 |

Table 5. 1 Comparison of average heat transfer coefficients from different temperature curve fittings on the inverse fin solution

The cubic spline fitting returned a close fit of results; however its broken aspect resulting in discontinuous rate variations causes local sudden bias that might not reflect a realistic spread of temperatures. Knowing that the local heat transfer calculations, either by one-dimensional fin analysis or 2D solution might see a small error of temperature fit largely amplified (see Owen 1979), to have sudden changes (however small) into a fit might induce distorted results of local heat transfer that could be misleading to an analyst.

From the previous plain disc fin solution model, a case whereby the number of simulated thermocouples would vary was considered. The thermocouple groups in Table 5.2 were temperature fitted with their highest order polynomial ($N_{T/C}-1$) and the resulting temperature profiles along the plain disc length was used to calculate the local heat transfer coefficients shown in Fig 5.3 (using finsol3_MCR3.m in Appendix 5.1).

| Number of T/Cs | Simulated rig thermocouples group | Temperature fit Polynomial order |
|----------------|-----------------------------------|----------------------------------|
| 3 | 9 – 10 – 11 | 2 nd |
| 4 | 8 – 9 – 10 – 11 | 3 rd |
| 5 | 7 – 8 – 9 – 10 – 11 | 4 th |
| 6 | 7 – 8 – 9 – 10 – 11 – 12 | 5 th |

Table 5. 2 Cases of varying thermocouple numbers in the plain disc fin solution model

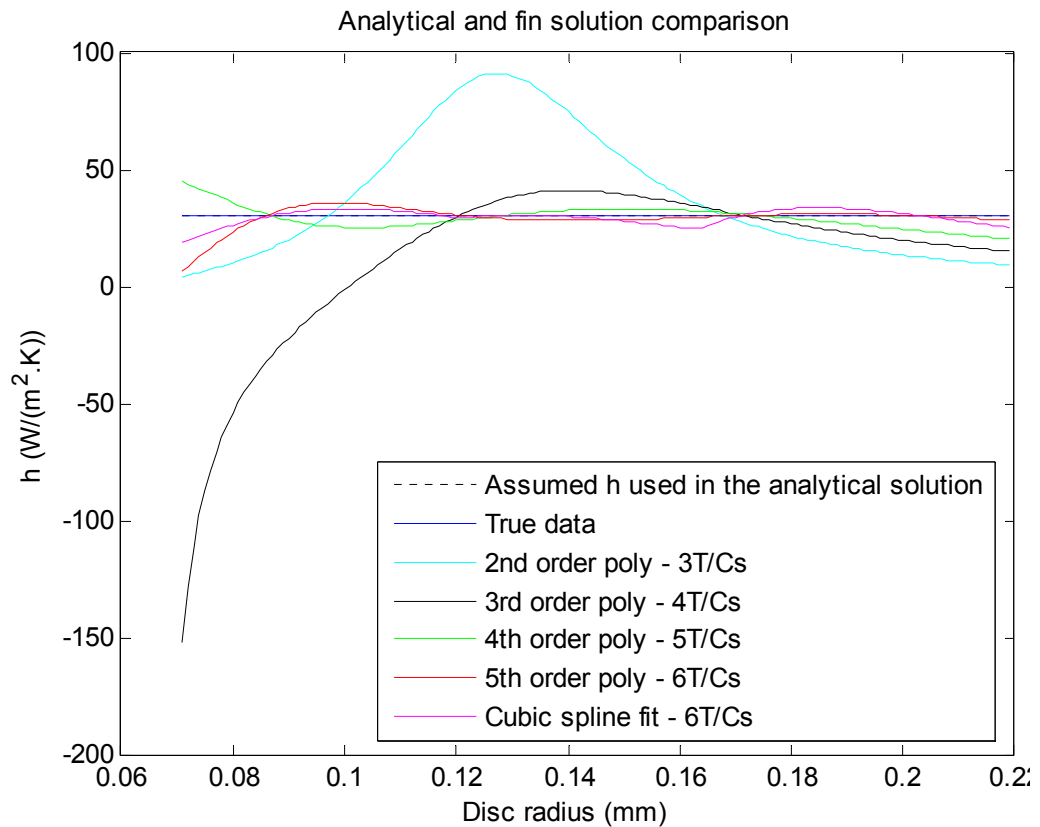


Fig 5. 3 Local heat transfer coefficients on the plain disc fins solution model with varying number of thermocouples

The local heat transfer coefficients shown in Fig 5.3 show better results for the cases including the most thermocouples. When considering cases with less thermocouples involved in the definition of the temperature profile along the disc length, the temperature errors on the extrapolated temperatures outside of the considered disc portion defined by the thermocouple group are amplified and therefore the heat transfer calculations might locally become unstable.

This was seen for the cases with 3 and 4 thermocouples. The average heat transfer coefficient values obtained are compared in Table 5.3.

| h _c | Analytical | | 3 T/C | | 4 T/C | | 5 T/C | | 6 T/C | |
|----------------|--------------------------------|------------------------------|------------------------------|------------------------------|------------------------------|------------------------------|------------------------------|------------------------------|------------------------------|------------------------------|
| | | | 2 nd order poly | | 3 rd order poly | | 4 th order poly | | 5 th order poly | |
| | Value W/(m ² .K) | hav W/(m ² .K) | % delta Vs h _c | hav W/(m ² .K) | % delta Vs h _c | hav W/(m ² .K) | % delta Vs h _c | hav W/(m ² .K) | % delta Vs h _c | hav W/(m ² .K) |
| 30 | 30.002 | 7 x 10 ⁻³ | 20.515 | 31.62 | 21.847 | 27.18 | 25.700 | 14.33 | 29.808 | 0.64 |

Table 5. 3 Comparison of average heat transfer coefficients from the plain disc fin model with varying number of thermocouples

The results in Table 5.3 above were obtained from simulated thermocouple temperatures analytically derived from the inverse fin solution previously described. A typical set of experimental temperatures (from run 2-1 in Build 3 of the Multiple Cavity Rig) was introduced into the plain disc model. The resulting local heat transfer coefficients were plotted in Figure 5.4 (using finsof_MCR4.m listed in Appendix 5.1).

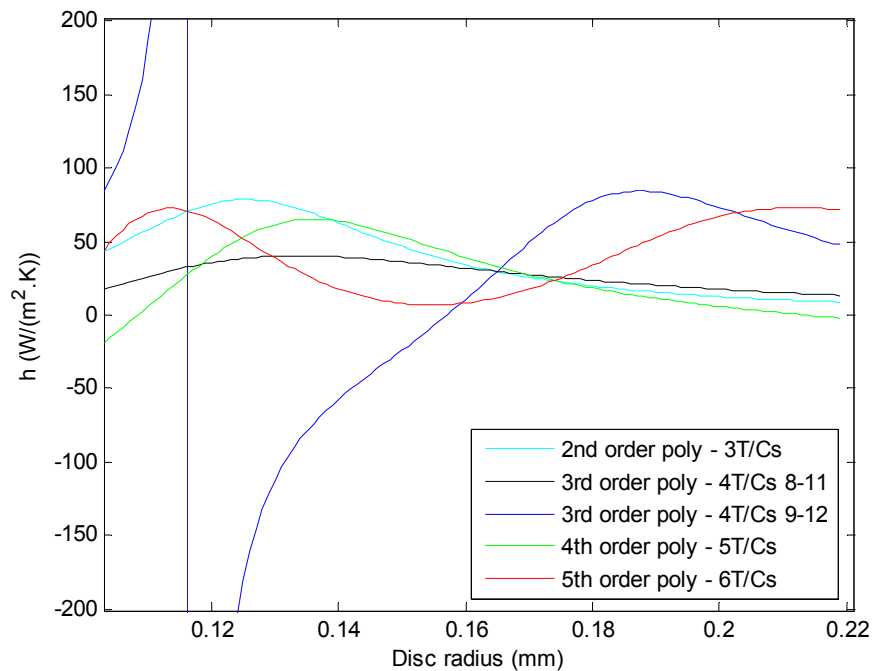


Fig 5. 4 Local heat transfer coefficients with varying number of thermocouples using an experimental temperature distribution

The results of real thermocouples temperatures in the plain disc model show a different behaviour than when inputting the idealised temperatures from the inverse fin solution.

When considering a greater number of thermocouples, the resulting local heat transfer coefficient profiles seem to become unstable on portions of the disc. Important facts to remember are that the experimental temperatures were obtained on a disc of varying dimensions along its radius, most certainly of varying heat transfer coefficient profile and that the temperature measurements could have been biased by experimental noise. A confirmation can be found when comparing the calculated local heat transfer coefficients considering the four-thermocouple set 4a: 8 – 9 – 10 – 11, and the four-thermocouple set 4b: 9 – 10 – 11 – 12. Set 4a is mostly located on a plain disc portion of Disc 2 in the Multiple Cavity Rig (minus a small portion of the disc cob by thermocouple 8), whereas set 4b is including the shroud portion of the disc. The sudden change of thickness and heat transfer characteristics by the disc rim might result in a measured temperature distribution that will cause discrepancies into the plain disc fin calculation model from bias in temperature fitting and also measurement noise as explained by Patounas (2007). It therefore appears that the use of 1D or 2D calculation models should be limited only to plain portions of discs for better accuracy of results. Patounas (2007) demonstrated the unsuitability of analytical 1D and 2D based local heat transfer calculations on a whole disc of variable geometry.

Cooke (2007) and Patounas (2007) used commercial codes such as Rolls Royce's SC03 (used as a solver for 2D heat conduction calculations from separately generated temperature profiles) and Fluent respectively in order to resolve the local heat transfer coefficients along a disc of realistic geometry. Even though an adaptive meshing and finer heat transfer properties definition could be obtained from these packages, the local heat transfer solving along the length of the disc still depended strongly on fitting accurately a radial temperature profile from few and sparse thermocouples data. These resolving methods can therefore still be error prone around the cobs and shrouds where sudden variations of geometries and temperatures would be encountered, which would not be possible to fit accurately as concluded by Cooke (2007).

Following this, and in order to minimise errors on the calculated results, the calculation zone on the Multiple Cavity Rig should therefore be limited to the

plain portions of the discs in order to avoid errors due to the sudden change of heat transfer properties and geometries as explained above.

The comparison of average heat transfer coefficient values along the disc obtained from the plain disc model using experimental temperatures plotted in Figure 5.4 is shown in Table 5.4.

| | | | | |
|---|--|---|--|---|
| 3 T/C 9 – 10 -11 2 nd order poly | 4 T/C a 8 – 9 – 10 – 11 3 rd order poly | 4 T/C b 9 – 10 – 11 – 12 4 th order poly | 5 T/C 7–8–9–10–11 5 th order poly | 6 T/C 7-8-9-10-11-12 5 th order poly |
| h_{av} W/(m ² .K) | h_{av} W/(m ² .K) | h_{av} W/(m ² .K) | h_{av} W/(m ² .K) | h_{av} W/(m ² .K) |
| 19.231 | 20.175 | 41.499 | 10.369 | 50.626 |

Table 5. 4 Comparison of average heat transfer coefficients from the plain disc fin model using experimental temperatures with varying number of modelled thermocouples

The results in Table 5.4 obtained in the cases of temperature fitting from thermocouples located in a plain portion of the disc surface (cases with 3 T/C and 4T/C a) produce close results of average heat transfer with each other. As discussed above, the accuracy of results is to be considered better only by limiting the calculation range of the disc surface to within the plain surface portion of the disc. Therefore the heat transfer calculations for the experiments on the Multiple Cavity Rig were limited to the discs cross sectional areas delimited by thermocouples 3 – 5 – 9 – 11 and 13 – 15 – 19 – 21 for Disc 2 and Disc 3 respectively.

A 4 thermocouple curve fit was employed in order to generate the experimental temperature profiles on the discs faces, using a third order polynomial as a pre-requisite to the heat transfer calculations on the discs following the method described in Chapter 4. This allowed for the best definition of temperatures along the disc portion considered, while minimising errors as discussed above.

An example of temperature curve fit obtained from various considerations of thermocouples inclusion on an experimental case can be seen in Figure 5.5, with a close up of the possible fits inputted into the analysis software shown in Figure 5.6.

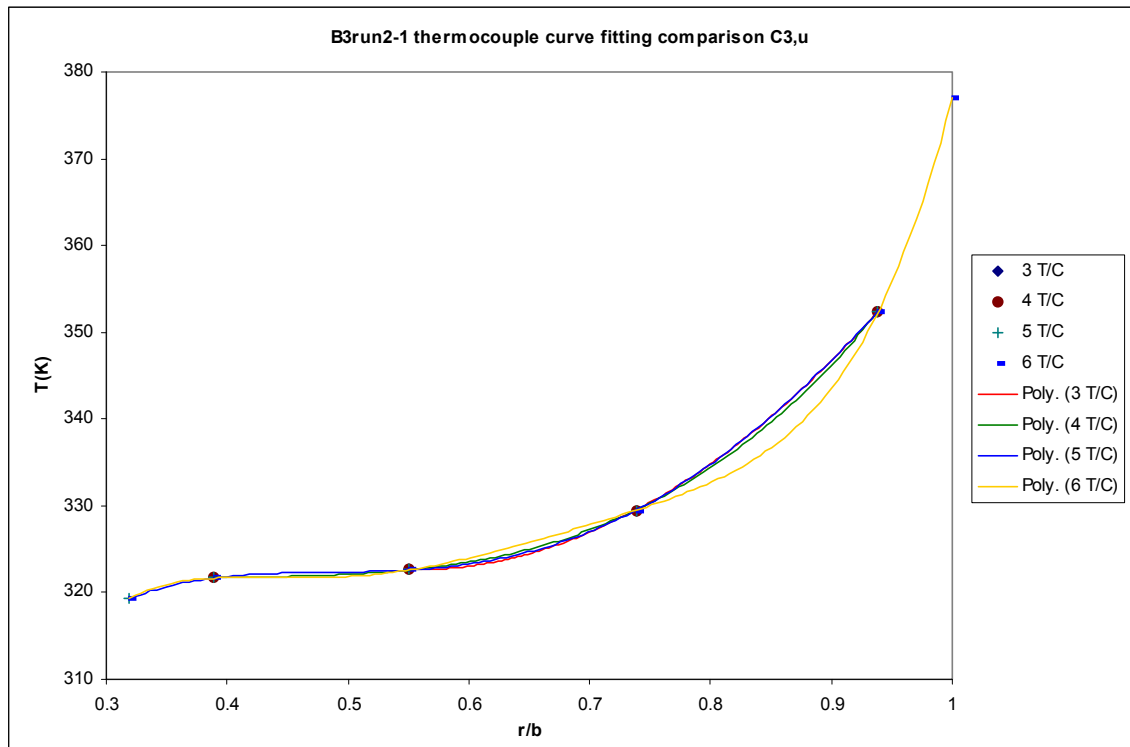


Fig 5. 5 Temperature fit comparison on Disc 2, downstream face from experimental temperature data

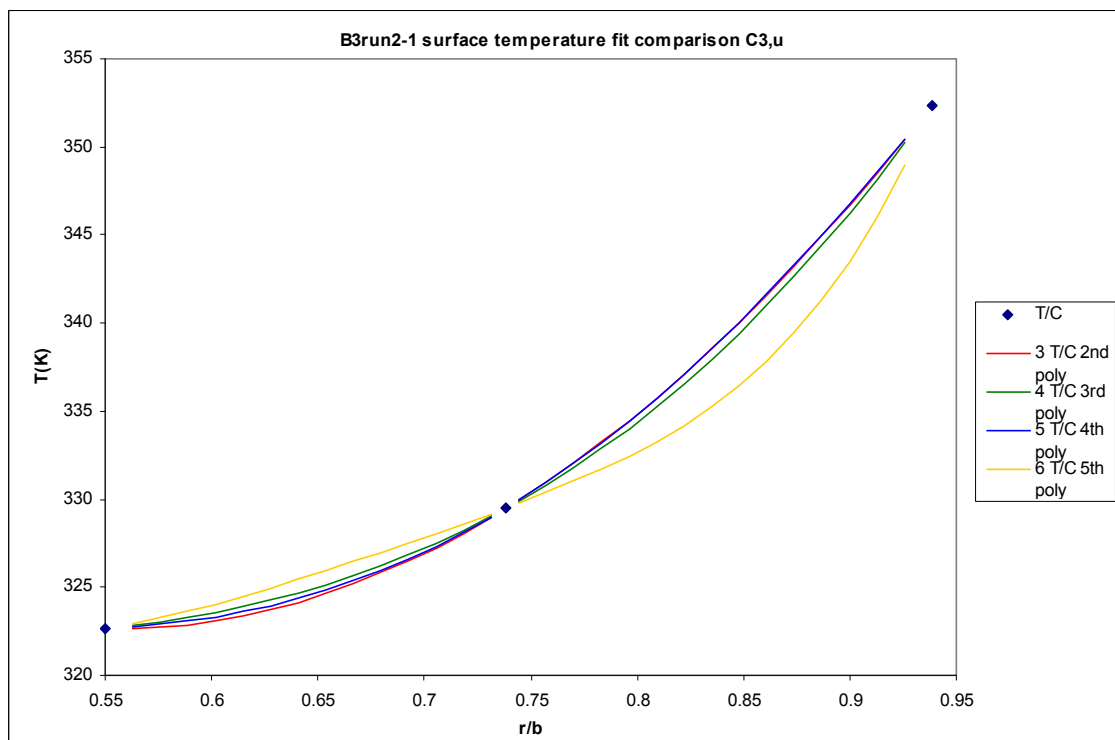


Fig 5. 6 Close up of temperature fits from experimental data on the analysis disc section

An error estimation of the temperature fit was performed on the average surface temperatures of Disc 2 for the fin calculation over the whole range of experiments performed on the Multiple Cavity Rig. The values of Deltas between the three thermocouple and 4 thermocouple fits were found to be within 0.04 % of each other. The derived sensitivity on the average heat transfer coefficient values were found to be typically within 7% of each other.

The effects of grid size on $\text{finD2}h_{av}$ were also assessed on the considered 4 T/C case. All the experimental cases were processed in turn with the original finite difference grid of 31 x 21 nodes. Then a 'quadrupled' grid size of 61 x 41 nodes was used and finally an 'octupled' grid of 121 x 41 nodes. The computing time per experimental case (143 in total to be processed) varied greatly between the original grid and the octupled grid by a factor of more than 50. The average relative error on all experimental cases between the finest grid average heat transfer coefficient and the original grid was found to be in the order of 4.2 %.

Therefore the 31 x 21 nodes grid was retained for having a reasonable computing time for an acceptable error on calculated heat transfer parameters. A summary of relative error and computation time against grid size is shown in Figure 5.7.

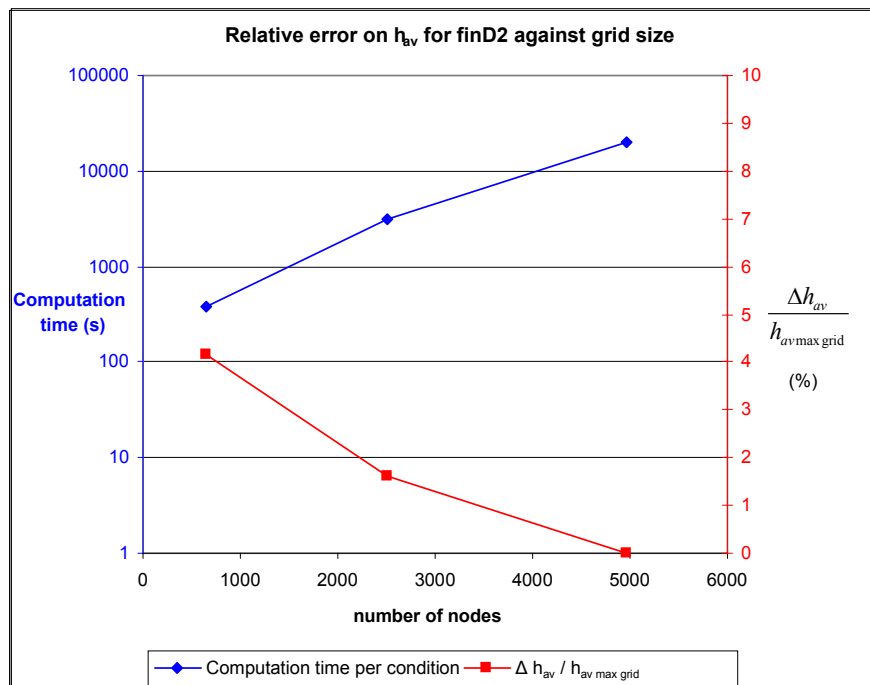


Fig 5. 7 Heat transfer relative error and computation time for different grid sizes against grid size

5.1.2 Heat transfer measurement uncertainties

The heat transfer calculation method used for the Multiple Cavity Rig can be very sensitive to small fluctuations of temperatures as seen in the previous section. The calculations error range due to bias in the way the thermocouple data was used into the finite difference processing was shown from defined, 'perfect' temperature inputs. The reality of experimental measurements however shows that the measured temperature data, even in a thermal steady state process, will show some fluctuations. These small experimental variations will influence the calculated results and small imperfections in temperature measurement could lead to discrepancies as reported by Owen (1979) and more recently by Cooke et al. (2006). The determination of the 'true' value of the measured temperature or calculated datum can only be determined by means of statistics.

This statistical process on a measured value will lead to a determined averaged 'nominal' value bounded within an uncertainty range representing the influence of the different sources of errors on the measuring process. Typical sources of uncertainty on measurements could be accounted for as:

- Noise in the recorded signal (e.g. random electrical interference)
- Sparseness of measurements (e.g. from physical fluctuations due to cyclic variations on flow)
- Resolution of recording instrumentation
- Calibration precision of instruments

Repeating an experimental measurement n number of times represents a statistical process of population n . The standard uncertainty, u , on the measurement is expressed as its Root Mean Square s , divided by the square root of the population:

$$u = \frac{s}{\sqrt{n}} \quad \text{Eq. 5.1}$$

In the case of data issued from past experience of measurement, calibration certificates, scarce number of measurements or knowledge of the noisy nature of the measurement, the standard uncertainty of the measurement can be estimated as the semi range of the standard spread of the measurement a , divided by $\sqrt{3}$.

$$u = \frac{a}{\sqrt{3}} \quad \text{Eq. 5.2}$$

The uncertainty in the latter case is said to be issued from a rectangular, or uniform distribution of measurements, in contrast to the former which is issued from a normal distribution.

The combined standard uncertainty in a case whereby the parameters' uncertainties add up to each other is expressed as the square root of the sum of the squares of each uncertainties.

$$u_{combined} = \sqrt{u_1^2 + u_2^2 + u_3^2 + \dots} \quad \text{Eq. 5.3}$$

More information on the expression of experimental uncertainties can be found in Bell (1999) and Moffat (1988).

The sources of uncertainties on the measured temperatures of the Multiple Cavity Rig are summarised in Table 5.5 below.

| Source of uncertainty | Value s ± °C | Probability distribution | Divisor | Standard uncertainty u ± °C | Source of data |
|---|--|---|----------------------------------|---|--|
| Thermocouple resolution (half interval) | 0.0125 | rectangular | $\sqrt{3}$ | 0.0072 | Orion data logger resolution specification |
| Cold junction resolution (half interval) | 0.0005 | rectangular | $\sqrt{3}$ | 0.0003 | Orion data logger spec. and Alexiou (2000) |
| Measurements spread (incl. noise, slip ring unit and cabling) | Calculated from measurements (100 scans) | Rectangular if signal is considered as noise dominated (Normal if considered as statistical process) | $\sqrt{3}$ $(\sqrt{100})$ | $\frac{s}{\sqrt{3}}$ $\left(\frac{s}{\sqrt{100}}\right)$ | Experimental data |
| Thermocouple calibration bath stability | | | | 0.04167 | Long et Childs (2007) |
| Calibration PRT precision | | | | 0.015 | Alexiou (2000) |
| Interpolation up to 100 °C | | | | 0.05 | Lancel (2002) |
| Repeatability | | | | 0.0415 | Lancel (2002) |

Table 5. 5 Sources of uncertainty on the measured temperatures of the Multiple Cavity Rig

Table 5.5 lists the standard error value of each source of uncertainty, the estimated standard uncertainty and specifies the origin of the estimation, from previous studies, materials specifications, or measured data. The Orion data logger used to measure the signals from the thermocouples had a resolution of $\pm 1 \mu\text{V}$, corresponding to $\pm 0.025 \text{ }^\circ\text{C}$. The cold junction reference sensor had a resolution of $\pm 10 \mu\text{V}$ corresponding to $\pm 0.001 \text{ }^\circ\text{C}$. The uncertainty values related to the thermocouples calibration process were obtained from different separate studies. The set-up and calibration of the thermocouples in the Multiple Cavity Rig followed the method described in Alexiou (2000). The calibration was performed in a water bath at constant temperature against a Platinum Resistance Thermometer.

The standard uncertainty on the stability of the water bath temperature was determined in Long et Childs (2007) as $\pm 0.04167 \text{ }^\circ\text{C}$. The Platinum Resistance Thermometer probe calibration uncertainty was estimated by Alexiou (2000) as $\pm 0.015 \text{ }^\circ\text{C}$. Lancel (2002) estimated the interpolation and repeatability on the calibration measurements respectively as $\pm 0.05 \text{ }^\circ\text{C}$ and $\pm 0.0415 \text{ }^\circ\text{C}$. The measured thermocouples temperature standard spread over 100 measurements during the experiment was representing the overall variation of the data due to experimental fluctuations, cabling noise and slip ring unit noise. Noise can be considered to have a rectangular probability distribution. Depending on the considered dominance of the temperature measurement variations or the noise in the fluctuations of the experimental thermocouples measurements, a choice of divisor is possible for the determination of the standard uncertainty (respectively $\sqrt{3}$ or \sqrt{n}). The experimental measurement spread over all the experiments was typically within a value of $0.2 \text{ }^\circ\text{C}$. Considering a normal distribution, an estimate of the standard uncertainty on the spread, u_A , would be:

$$u_A = \frac{s}{\sqrt{n}} = \frac{0.2}{\sqrt{100}} = 0.02 \quad \text{Eq. 5.4}$$

The overall combined standard uncertainty value on the thermocouples measurements would be:

$$u_{T/C,A} = \sqrt{0.0072^2 + 0.0003^2 + 0.04167^2 + 0.015^2 + 0.05^2 + 0.0415^2 + 0.02^2} = 0.08 \text{ }^\circ\text{C} \quad \text{Eq. 5.5}$$

The corresponding 95 % interval of confidence considering a normal distribution of the measured temperatures would therefore be estimated to be:

$$u_{95_{T/C,A}} = 1.96 \times u_{T/C,A} = 0.1568^{\circ}\text{C} \quad \text{Eq. 5.6}$$

Considering a rectangular distribution of the measured temperatures, the estimated standard uncertainty u_B becomes:

$$u_B = \frac{s}{\sqrt{3}} = \frac{0.2}{\sqrt{3}} = 0.11547^{\circ}\text{C} \quad \text{Eq. 5.7}$$

$$\text{And } u_{T/C,B} = 0.14^{\circ}\text{C} \quad \text{Eq. 5.8}$$

$$\text{And hence } u_{95_{T/C,B}} = 0.2744^{\circ}\text{C} \quad \text{Eq. 5.9}$$

However, considering that the rotating thermocouples on the rig use another thermocouple as the slip ring unit cold junction reference, the corrected standard uncertainties lead to:

For a normal distribution

$$u_{T/C,C} = \sqrt{(u_{T/C,A})^2 + (u_{T/C,B})^2} = \sqrt{2} \times 0.08 = 0.11^{\circ}\text{C} \quad \text{Eq. 5.10}$$

$$u_{95_{T/C,C}} = 0.2217^{\circ}\text{C} \quad \text{Eq. 5.11}$$

And for a rectangular distribution

$$u_{T/C,D} = \sqrt{(u_{T/C,B})^2 + (u_{T/C,B})^2} = \sqrt{2} \times 0.14 = 0.20^{\circ}\text{C} \quad \text{Eq. 5.12}$$

$$u_{95_{T/C,D}} = 0.3881^{\circ}\text{C} \quad \text{Eq. 5.13}$$

Previous studies on rotating temperature measurements by Lancel (2002), using a similar experimental set-up of data logger and slip ring unit, evaluated the 95% interval of confidence in the temperature measurements at a value of 0.294°C . It could be argued that the temperature measurement spread would follow a normal distribution and therefore the thermocouple uncertainty divisors to be used into the heat transfer analysis programme should follow the calculation method of $u_{T/C,A}$ and $u_{T/C,C}$. However, as the participation of noise into the recorded signals cannot be separated from the recorded data of the

physical temperature fluctuations, and in regard to the levels of uncertainty determined by Lancel (2002) in a similar range of temperature, it appears that a rectangular distribution of data should be considered for the determination of thermocouples measurement uncertainties in the Multiple Cavity Rig. Moreover, the measurement uncertainty value to be retained on a conservative basis should be the value associated with the rotating thermocouples, i.e. $u_{95_{T/C,D}} = 0.3881^{\circ}\text{C}$ (Eq. 5.13).

As described in Chapter 4, the estimated standard uncertainties and average values from the thermocouples temperatures and rotational speed recordings for each experiment were integrated into the heat transfer processing programme.

This generated a hundred and one sub-cases of processing based on the uncertainty spread in order to calculate the nominal results and their respective 95% interval of confidence.

Owen (1979), followed by Cooke (2007) previously studied gas turbine disc heat transfer experimental uncertainty using a calculation method based on a Monte-Carlo simulation. The heat transfer processing programme for the Multiple Cavity Rig operates on similar principles, but uses actual experimental data to be processed with an estimation of uncertainty values. The sensitivity of the results to small variations of surface temperatures was considered as predominant on the heat transfer results as reported by Owen (1979) and Cooke (2007). Therefore the variability of coolant pressure or hardware dimensions were not taken into account in this study.

The uncertainty values produced by the heat transfer processing programme therefore not only included the inputted fluctuating variables such as temperatures and rotational speed, but also any calculated parameter involving one or more of the inputs such as h , Nu , Re_z etc. A list of parameter outputs and their respective intervals of confidence can be seen in the processing examples in Appendix 7.1.

Figure 5.8 shows an example of results of local heat transfer results obtained from the two-dimensional solution method and the fin solution method, as previously described in Chapters 4 and 5, for the experimental conditions of $Re_\phi = 4.12 \times 10^6$, $Re_z = 1.16 \times 10^5$, $Ro = 0.54$ and $\beta\Delta T_{av} = 0.03$.

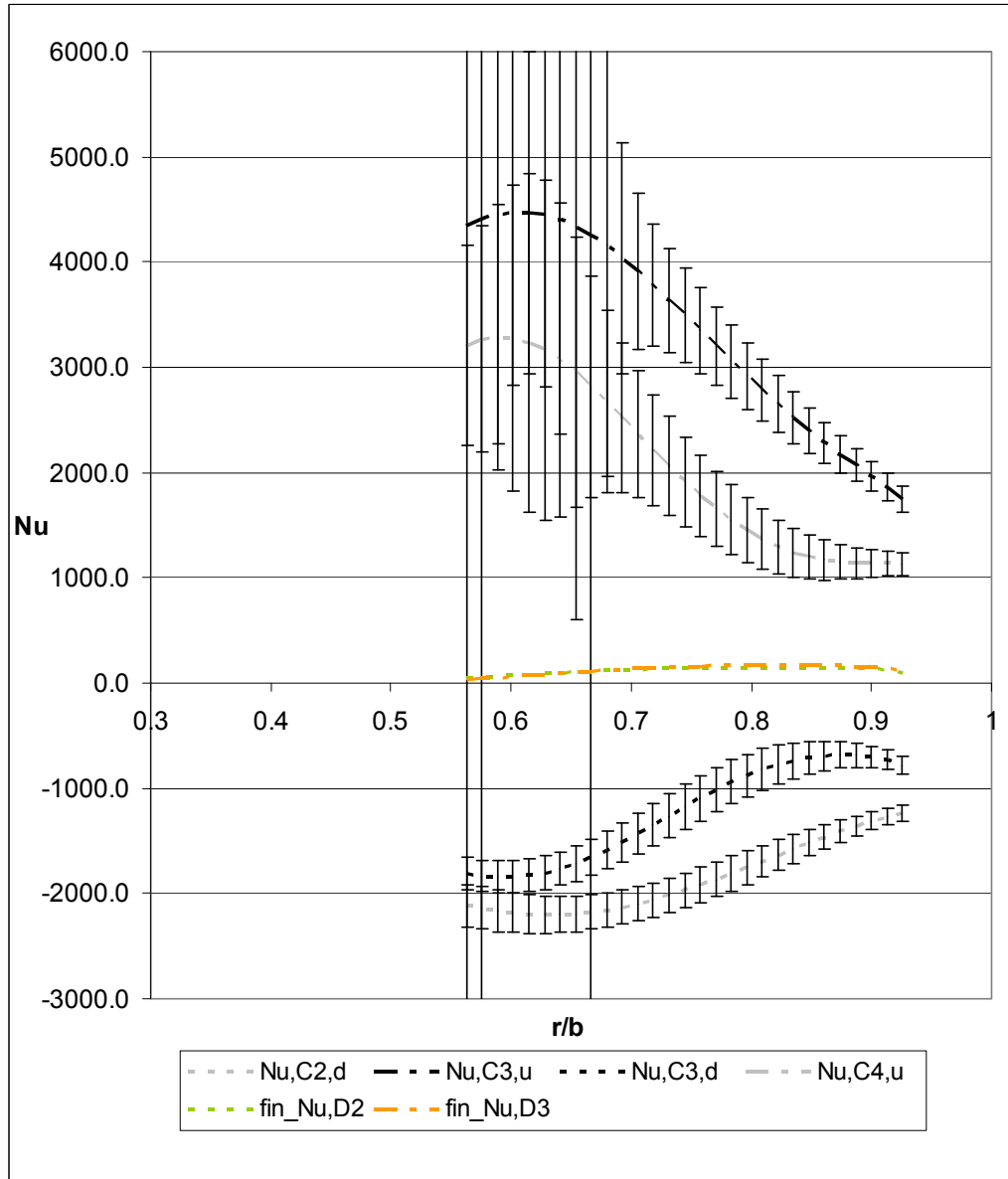


Fig 5. 8 Local Nusselt numbers against r/b for Build 3, run28-1, $Re_\phi = 4.12 \times 10^6$, $Re_z = 1.16 \times 10^5$, $Ro = 0.54$ and $\beta\Delta T_{av} = 0.03$

The plotted Nusselt numbers were determined on the flat surface portion of the discs between $0.55 \leq r/b \leq 0.94$ as previously described. The local Nusselt number values resulting from the two-dimensional solution are represented in Figure 5.8 with error bars indicating the calculated 95% interval of confidence.

The fin solution plots were not represented with error bars for clarity of display. Their 95% uncertainty intervals were typically in the order of $\pm 10\%$. The calculated average total heat flux from the two-dimensional solution for the downstream face of Disc 2 in this example was found to be $q_{T,av,D2,dn} = 3307 \text{ W/m}^2$, with a contribution of radiative heat flux of about 1.5%. Similarly, the calculated value for the upstream face of Disc 3 was $q_{T,av,D3,up} = 1665 \text{ W/m}^2$ with a radiative contribution of 1.2%. The overall contribution of radiative heat flux on the discs across all the experiments was typically found to be less than 10% of the measured total heat flux on the discs faces as already mentioned in section 4.2.2. The results from the two-dimensional solution show a lower overall level of heat transfer from Disc 3 (both upstream and downstream faces) compared to Disc 2. The differences between the upstream and downstream faces of both discs indicate large values of positive heat transfer on the upstream faces (heat flows out of the disc face to the cavity air) whereas the discs' downstream faces show negative values of Nusselt number that would suggest that heat is given into the disc from the cavity air. This described discs heat transfer behaviour does not fit the current understanding of cavity flow and heat transfer from its direction and magnitude in regards to the measured relatively low magnitudes of cavity air velocities previously described. The two-dimensional solution local heat transfer results could only illustrate further the inherent problems of results from this calculation method as explained above. Discrepancies on the calculated local heat transfer values from the two-dimensional solution were seen across the whole test range. Figure 5.9 shows another example of results for $Re_\phi = 0.49 \times 10^6$, $Re_z = 0.92 \times 10^5$, $Ro = 3.56$ and $\beta\Delta T_{av} = 0.09$. On this test, the local Nusselt number profiles on Disc 3 indicate both positive and negative heat transfer along the calculated discs surfaces. The variation of average Nusselt number for all tests carried out on Build 2 of the test rig (narrow annular gap, $0.4 \times 10^6 < Re_\phi < 5 \times 10^6$, $4.2 \times 10^4 < Re_z < 1.9 \times 10^5$, $0.29 < Ro < 8.6$) with Rossby number is shown in Figure 5.10 (2-D solution) and Figure 5.11 (fin solution). The behaviour of negative Nusselt numbers noted above is also applicable to average Nusselt numbers as shown in Figure 5.10. This inconsistency is not seen in the results of the fin solution in Figure 5.11.

In this chapter, the instrumentation design of the series of the Multiple Cavity Rigs using a finite number of surface thermocouples on the discs surfaces and relying on 2-D analysis to determine values of disc to air heat fluxes has been demonstrated as inadequate. However the results from the fin analysis are more in agreement with the current consensus in disc heater transfer even though the scale of the bias resulting from the limited number of thermocouples on the rig discs cannot be ascertained. Other methods of determining disc heat fluxes or improved temperature definition should be sought for future studies, as concluded by Cooke (2007) subsequently to the end of the presented experimental programme.

The presented heat transfer results in Chapter 7 therefore should be considered bearing in mind the limitations of the heat transfer calculation method implemented by design on the Multiple Cavity Rig.

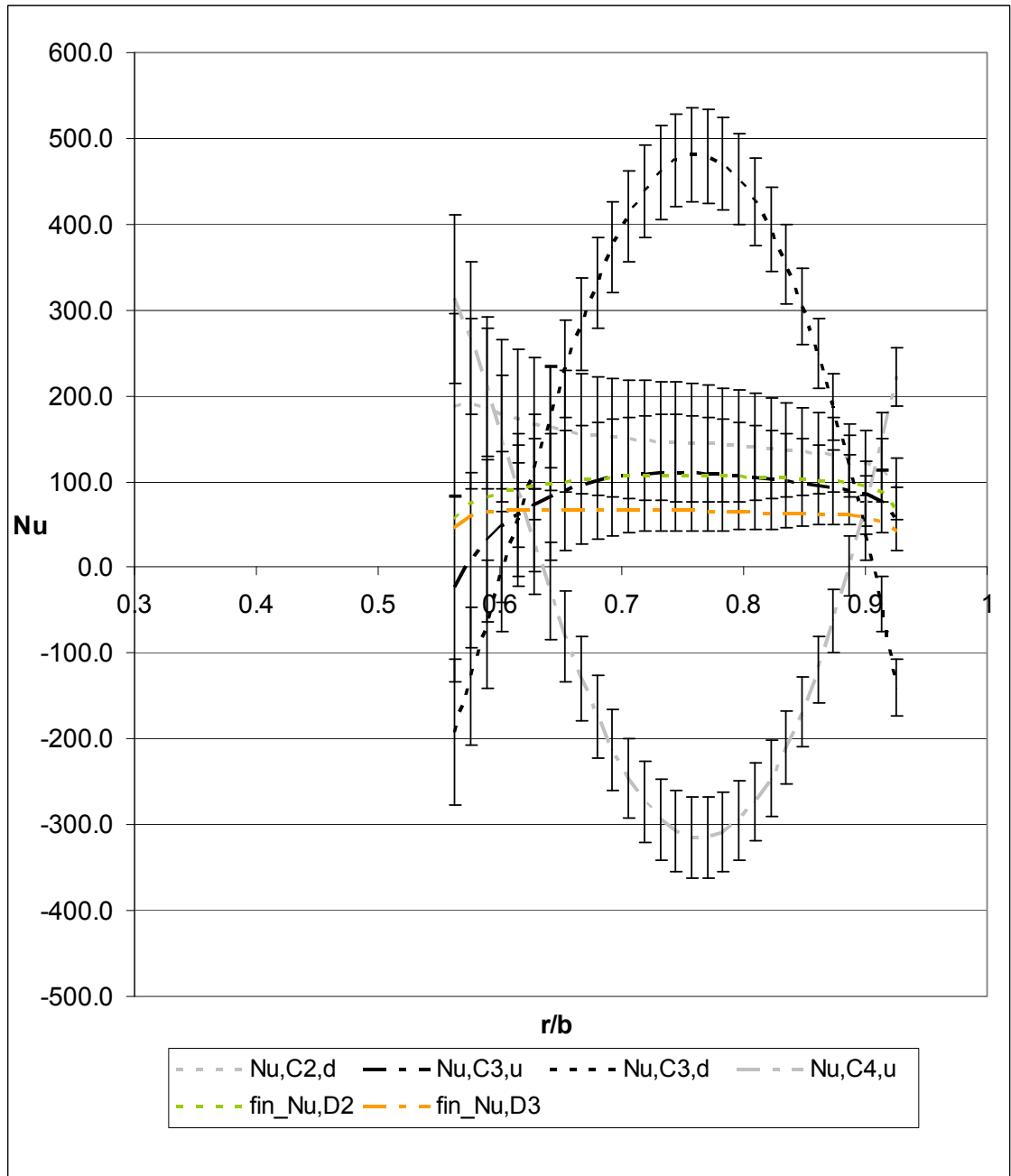


Fig 5. 9 Local Nusselt numbers against r/b for Build3, run32-1, $Re_\phi = 0.49 \times 10^6$, $Re_z = 0.92 \times 10^5$, $Ro = 3.56$ and $\beta\Delta T_{av} = 0.09$

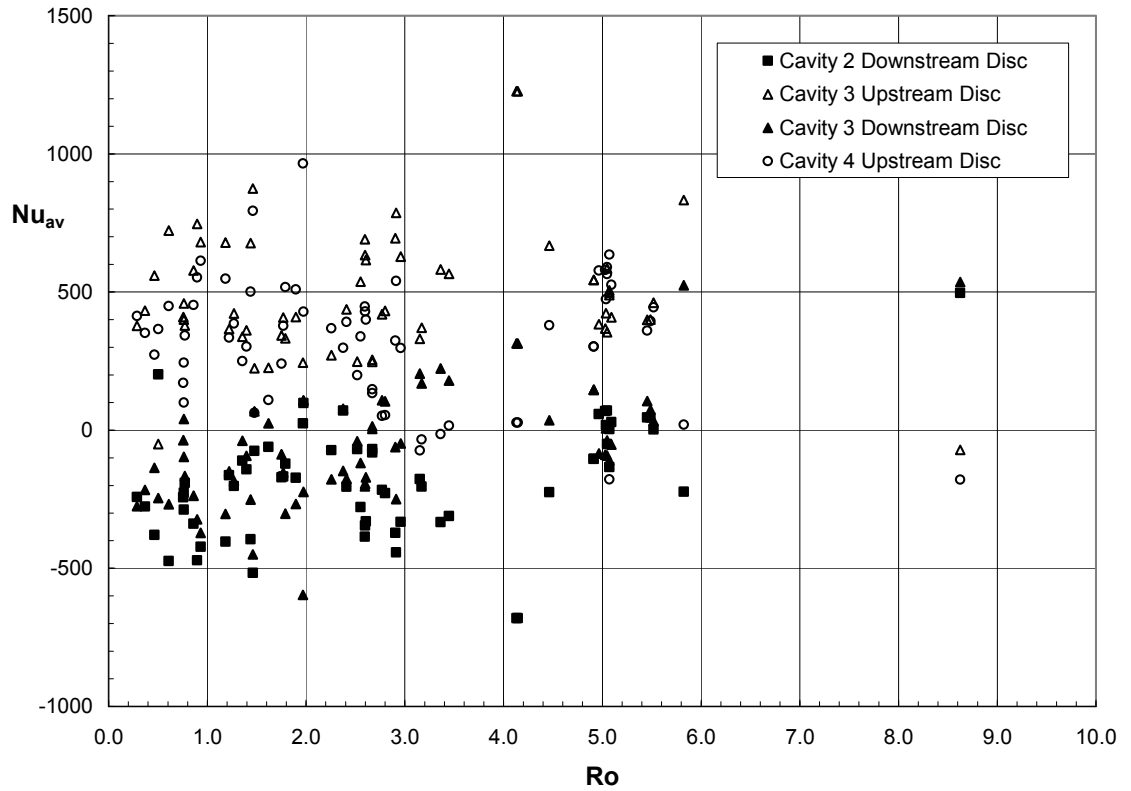


Fig 5. 10 2D solution results for Build 2 (narrow annular gap)
Variation of average Nusselt number with Rossby number

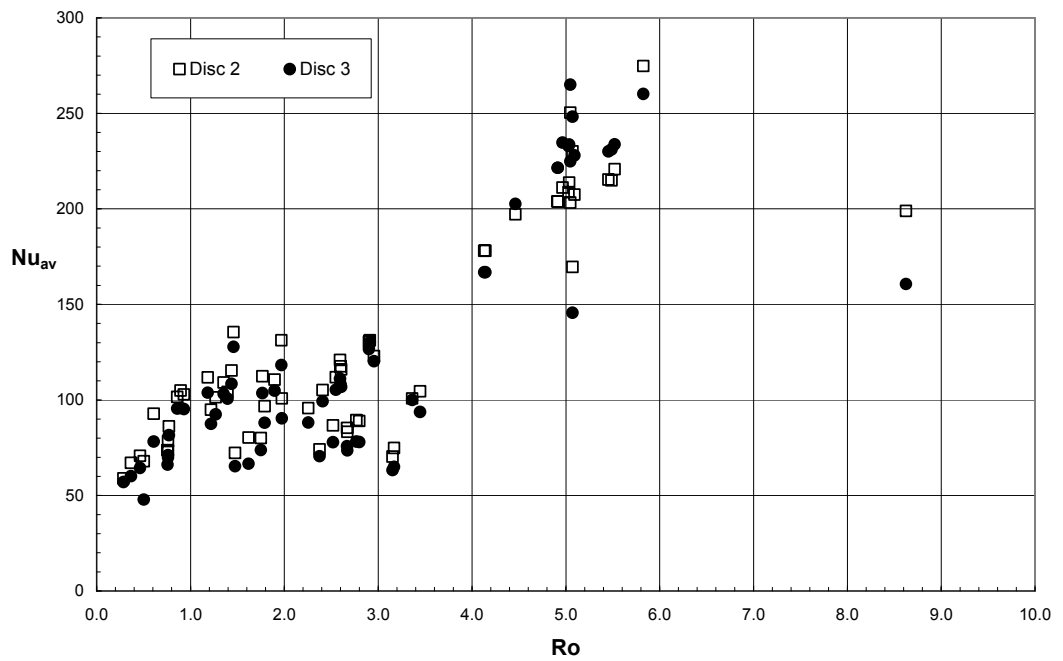


Fig 5. 11 Fin solution results for Build 2 (narrow annular gap)
Variation of average Nusselt number with Rossby number

5.2 Flow velocity measurement uncertainties

As explained in Chapter 4, Laser Doppler Anemometry is a direct measurement technique providing a direct reading of velocity without requiring prior calibration. However, in experimental situations, some bias can be introduced as a result of experimental set-up, flow conditions and data processing. Begg (2003) categorises the sources of uncertainty on the measurements into two groups: The systematic uncertainties introduced from the experiment, instrument and environment; and the statistical uncertainties arising from the estimation of flow quantities through a finite number of samples.

5.2.1 Systematic uncertainties

The principal systematic sources of uncertainties on the flow velocity measurements in the Multiple Cavity Rig were identified as:

- Signal noise
- Laser wavelength stability
- Beam crossing angle
- Seeding
- Velocity bias
- Velocity gradient broadening
- Probe alignment
- Finite transit time broadening
- Fringe bias

Following is a brief description of these sources of uncertainty and how they were addressed in the experimental set-up of the Multiple Cavity Rig.

Signal noise

The signal noise in the Laser Doppler Anemometry signals is considered as the main cause of data rejection in the burst signal processing unit, therefore impairing the validated data collection rate.

This in turn affects the velocity sample size able to be collected in a reasonable time. Signal noise is present electronically from the photomultipliers, amplifier circuits etc.

Noise also presents itself optically from reflections of scattered light onto surfaces, seeding particles and reflections on mirrors and windows. The optical noise was reduced in the test rig by having the discs and shrouds surfaces painted matt black to reduce surface reflections. The optical access windows are made of optical grade glass (BK7) widely used for Laser Doppler applications and their surfaces had an anti-reflective coating applied specifically in tune with the laser beams frequencies in order to minimise reflection losses. The windows and mirrors were also cleaned before each test. The laser power was kept at a working minimum (150 mW to 350 mW at the probe volume) in order to minimise the optical glare and saturation of the photomultipliers. The photomultipliers driving voltage and gain were adjusted in order to produce the best trade off between validation rate and data rate of the collected bursts signals. The signal to noise ratio threshold filter of the signal processor was also adjusted in order to achieve an optimum of this balance.

Seeding

The seeding in Laser Doppler Anemometry experiments is of prime importance in determining the accuracy of the flow measurements. As the measurements result from the tracking of seeding particles velocity, it is therefore of primordial importance that the seeding particles move at an equal velocity to the flow. Particle sizes and mass will affect their ability to follow the flow faithfully. Generally speaking, the smaller and lighter the particles, the better the accuracy. On the other hand, too small particles will increase the chances of multiple occurrences within the probe volume and therefore could be detrimental to the validated data rate of the measurement. Quenching oil was used for the flow measurements on the Multiple Cavity Rig. It was atomised at a size of approximately 1 μm . This is generally accepted as being small enough to faithfully follow the motion of air flow in experimental conditions.

No difference was noted in the measurements validation and data rates by varying the seeding atomisation rates from one jet to the maximum six jets, so one jet was used in order to reduce the fouling of the windows.

Laser wavelength stability

Laser Doppler Anemometry measurements assume a fixed light source frequency in order to derive burst frequencies. Therefore the wavelength stability of the laser source will directly affect the accuracy of the velocity measurements. Argon Ion lasers, as used on the present experiments, are considered as inherently stable in operation. Therefore the uncertainty associated with wavelength stability is considered as negligible.

Velocity bias

Velocity bias originates from the intermittent data collection nature of Laser Doppler Anemometry coming from passing seeding particles rather than from the air velocity itself. In regions of high flow velocity, the probability of a seeding particle to cross the measurement volume at a given time is higher than in a region of low velocity for a considered uniform particles concentration. Hence a disproportionate number of high velocity value samples are obtained when measuring in regions of turbulent flow, biasing the velocity measurement towards a higher value than the 'true' mean of the flow. This phenomenon is well known and a transit time weighted mean velocity flow estimator is generally accepted to give valid results in all conditions (Tropea 2002). It is expressed as:

$$\bar{u} = \frac{\sum (u_i \cdot \Delta t_i)}{\sum \Delta t_i} \quad \text{Eq. 5.14}$$

Where u_i is the individual velocity sample and Δt_i is its residence time in the measuring volume.

This corrected method of calculation was already implemented in the Dantec post processing software, therefore the uncertainty associated with velocity bias on the measurements was considered as negligible.

Velocity gradient broadening

When a fluid flow has a significant velocity gradient in the direction normal to the fringes of the measurement volume, the successive passage of particles at different positions into the probe volume will produce different velocity values according to their position in the velocity gradient field.

The measured velocity values will appear to fluctuate even though the flow might be perfectly steady. The extra variance to the measurements resulting from velocity gradient broadening is defined by:

$$V'_g = \left(\sigma_p \cdot \frac{\partial \bar{u}}{\partial y} \right)^2 \quad \text{Eq. 5.15}$$

Where $\frac{\partial \bar{u}}{\partial y}$ is the mean velocity gradient at the point of measurement and σ_p is the standard deviation of the distribution of particles in the direction parallel to the fringes in the measurement volume.

This variance needs to be subtracted from the measured value in order to obtain the true variance of the flow. In the Multiple Cavity Rig, velocity gradient broadening is unlikely to be a significant source of uncertainty in regions away from the solid surfaces.

Probe alignment

As mentioned in Chapter 4, the accuracy of positioning of the probe volume was considered as ± 0.5 mm in the axial and radial directions for both traverse configurations. The accuracy of orientation of the laser probe in respect to the rig was of particular importance as a small error in orientation of the orthogonal beams could be a significant source of uncertainty when one velocity component is greater than another. The angular locating of the laser beam alignment targets was set against a precision machining spirit level giving an accuracy of $\pm 0.015^\circ$. The uncertainty on the radial velocity measurements associated with probe alignment error is therefore:

$$u_{V_r, probe} = \pm V_\phi \cdot \cos(0.015^\circ) \quad \text{Eq. 5.16}$$

In the case of a velocity measurement at the largest tangential velocity measured (50 m/s), the corresponding uncertainty on the radial velocity measurement due to angular probe alignment was estimated as 0.014 m/s.

Transit time broadening

Transit time broadening occurs as a result of measurement of Doppler burst frequencies being obtained from a limited number of cycles, i.e. fringe passages. The fewer the fringes, the greater is the potential uncertainty in estimation of the frequency content of the signal.

The probe volume of the laser set-up used in this work featured 14 fringes, and therefore the uncertainty introduced considering the large number of samples usually taken is considered to be small.

Fringe bias

Fringe bias (or direction bias) occurs in flows with significant changes of direction, as in recirculation flows for instance, whereby the flow will change direction within the probe volume itself. Particles approaching the probe volume in such flows might arrive at a flat angle to the fringes, or reverse direction, and therefore might not have crossed a sufficient number of fringes for the burst signal processor to accept the sample as valid (thus affecting the data rate of measurement). This might affect the flow statistics, but the effects on the experimental measurements are considered to be marginal in the present case.

5.2.2 Statistical uncertainties

Statistical uncertainties on a Laser Doppler Anemometry measurement are a direct result of it being essentially an averaged technique of measurement based on irregular sample collection of instantaneous velocities over time. Sample sizes of 30 or more are recommended as a very minimum of validity or the calculation of a mean as the expression of the mean velocity \bar{u} in Equation 5.14. The uncertainty calculations on the velocity samples were carried out by the Dantec processing software while acquiring. The 95 % confidence interval on the mean velocity was expressed as:

$$\bar{u}_{95} = 1.96 \times \sqrt{\frac{\bar{u}^2}{N}} \quad \text{Eq. 5.17}$$

Where N is the number of collected samples, and \bar{u} is the mean of the sample velocities.

The 95 % confidence interval of the Root Mean Square of the collected samples was calculated as:

$$\sigma_{u_{95}} = 1.96 \times \sqrt{\frac{\bar{u}^2}{2 \cdot N}} \quad \text{Eq. 5.18}$$

Calculated flow measurement uncertainties were integrated in the results and shown on velocity measurement plots with “error bars” representing $\bar{u} \pm \bar{u}_{95}$ in the example of an average velocity plot.

6. Velocity measurements results

This chapter presents the velocity measurement results for Build 2 and Build 3 of the Multiple Cavity Rig. The main difference and measurement capability of Build 2 and Build 3 are recapped in Table 6.1 below.

All the measured velocity data cases were processed into detailed test reports for ease of archiving and analysis. The list of all experiments, with the velocity components measured when applicable and the operating conditions and main non-dimensional results can be found in Appendices 6.2 and 6.3. From these lists, the Reader can select a test of interest and consult the detailed results report in the attached CD-Rom following the instructions in Appendix 6.1. For convenience of reading, the reports of the specifically mentioned tests in this chapter are also included in Appendix 6.4.

| | Build 2 | Build 3 |
|---------------------------|------------------|---------------------------------------|
| Annular gap | 0.092 | 0.164 |
| Heat transfer measurement | Disc, shroud | Disc, shroud |
| LDA measurement | V_Φ , V_z | V_Φ , V_z and V_Φ , V_r |

Table 6. 1 Principal operational differences between Build 2 and Build 3 of the Multiple Cavity Rig

Table 6.2 summarises the range of experimental conditions performed for the velocity measurements on Build 2 and Build 3 of the Multiple Cavity Rig. By comparison, typical engine values for cruise operation may be taken as $Re_\Phi = 7.7 \times 10^6$ and $Re_z = 2.4 \times 10^5$ (Alexiou 2000). The experimental data reported here has been obtained over a range of values overlapping both real engine conditions and also those used by authors of previous research in this field as shown in Fig. 2.17.

| Parameter | | Build 2 LDA | Build 3 LDA |
|------------------------------------|----------------------------------|---|---|
| Rotor speed | Ω (rad/s) | $123 \leq \Omega \leq 306$ | $65 \leq \Omega \leq 570$ |
| Cooling air flow rate | \dot{m} (kg/s) | $0.184 \leq \dot{m} \leq 0.618$ | $0.2 \leq \dot{m} \leq 0.72$ |
| Inlet pressure | p_{in} (bar, abs) | $2.25 \leq p_{in} \leq 3$ | $2.3 \leq p_{in} \leq 3.1$ |
| Inlet temperature | T_{in} (K) | $311 \leq T_{in} \leq 319$ | $302 \leq T_{in} \leq 339$ |
| Disc average surface temperature | $T_{s,av}$ (K) | $328 \leq T_{s,av} \leq 339$ | $321 \leq T_{s,av} \leq 353$ |
| Shroud average surface temperature | T_{sh} (K) | $361 \leq T_{sh} \leq 376$ | $353 \leq T_{sh} \leq 385$ |
| Disc rotational Reynolds number | Re_{Φ} (Eq. 4.1) | $9.5 \times 10^5 \leq Re_{\Phi} \leq 2 \times 10^6$ | $4 \times 10^5 \leq Re_{\Phi} \leq 4.1 \times 10^6$ |
| Axial Reynolds number | Re_z (Eq. 4.2) | $4.8 \times 10^4 \leq Re_z \leq 1.5 \times 10^5$ | $4.7 \times 10^4 \leq Re_z \leq 1.8 \times 10^5$ |
| Rossby number | Ro (Eq. 4.3) | $1.27 \leq Ro \leq 5.1$ | $0.27 \leq Ro \leq 7.5$ |
| Disc buoyancy parameter | $\beta \Delta T_{av}$ | $0.04 \leq \beta \Delta T_{av} \leq 0.08$ | $0.02 \leq \beta \Delta T_{av} \leq 0.09$ |
| Shroud buoyancy parameter | $\beta \Delta T_{sh}$ (Eq. 4.20) | $0.15 \leq \beta \Delta T_{sh} \leq 0.19$ | $0.11 \leq \beta \Delta T_{sh} \leq 0.238$ |
| Shroud Grashof number | Gr_{sh} (Eq. 4.20) | $1.6 \times 10^8 \leq Gr_{sh} \leq 6.2 \times 10^8$ | $3.3 \times 10^7 \leq Gr_{sh} \leq 1.4 \times 10^9$ |

Table 6. 2 Range of LDA experimental conditions for Build 2 and Build 3 of the Multiple Cavity Rig

6.1 Axial and tangential velocity components

Figure 6.1 and Figure 6.2 show the measurements of tangential velocity, V_ϕ , in Build 2 and 3 of the Multiple Cavity Rig respectively, normalised with respect of the cavity rotation Ωr against the non dimensional radial location r/b . Owing to a difference in annular gap size, the velocity profiles along the non-dimensional radius from Build 2 and Build 3 differ, in that the tangential velocity profiles of Build 2 would tend to indicate a gradual acceleration to solid body rotation as the radius increases, while Build 3 would tend to indicate an over-spinning of the flow in the region between the shaft and the cavity entrance ($0.236 \leq r/b \leq 0.318$) with a gradual decrease to solid body rotation as progressing radially outwards in the cavity . More details will be provided later in this chapter.

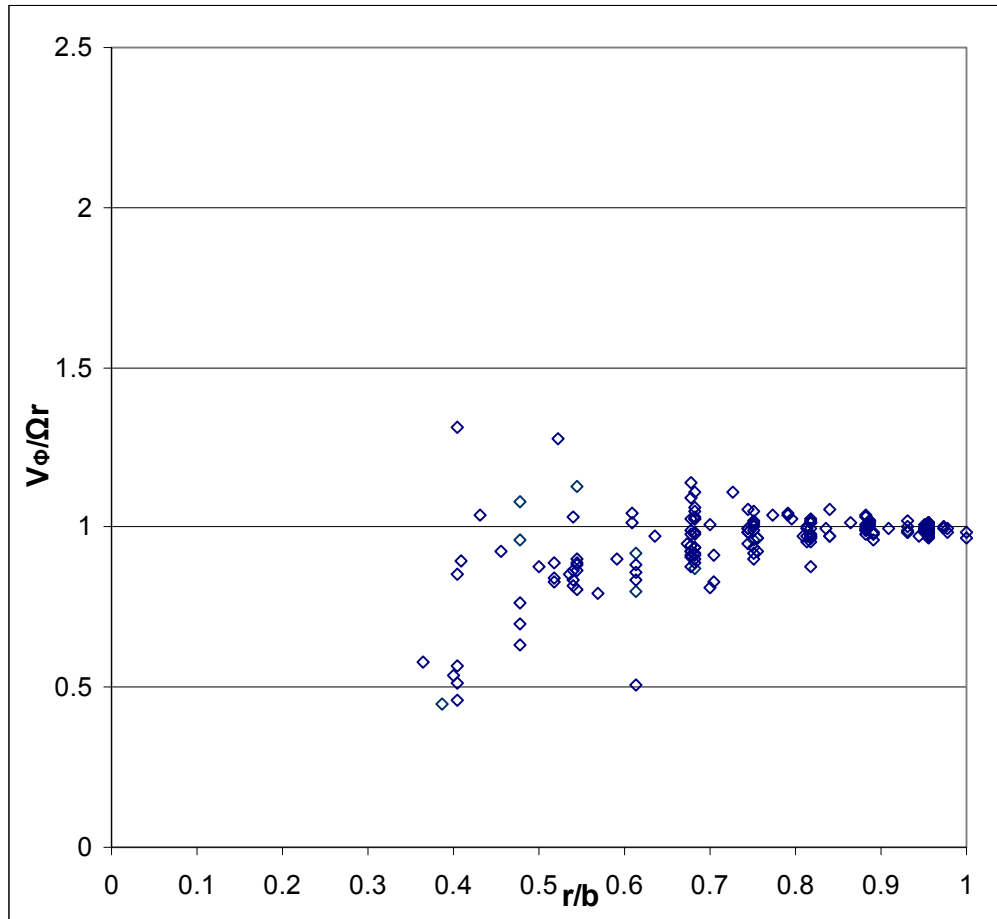


Fig 6. 1 Tangential velocity measurements for Build 2 (all data)

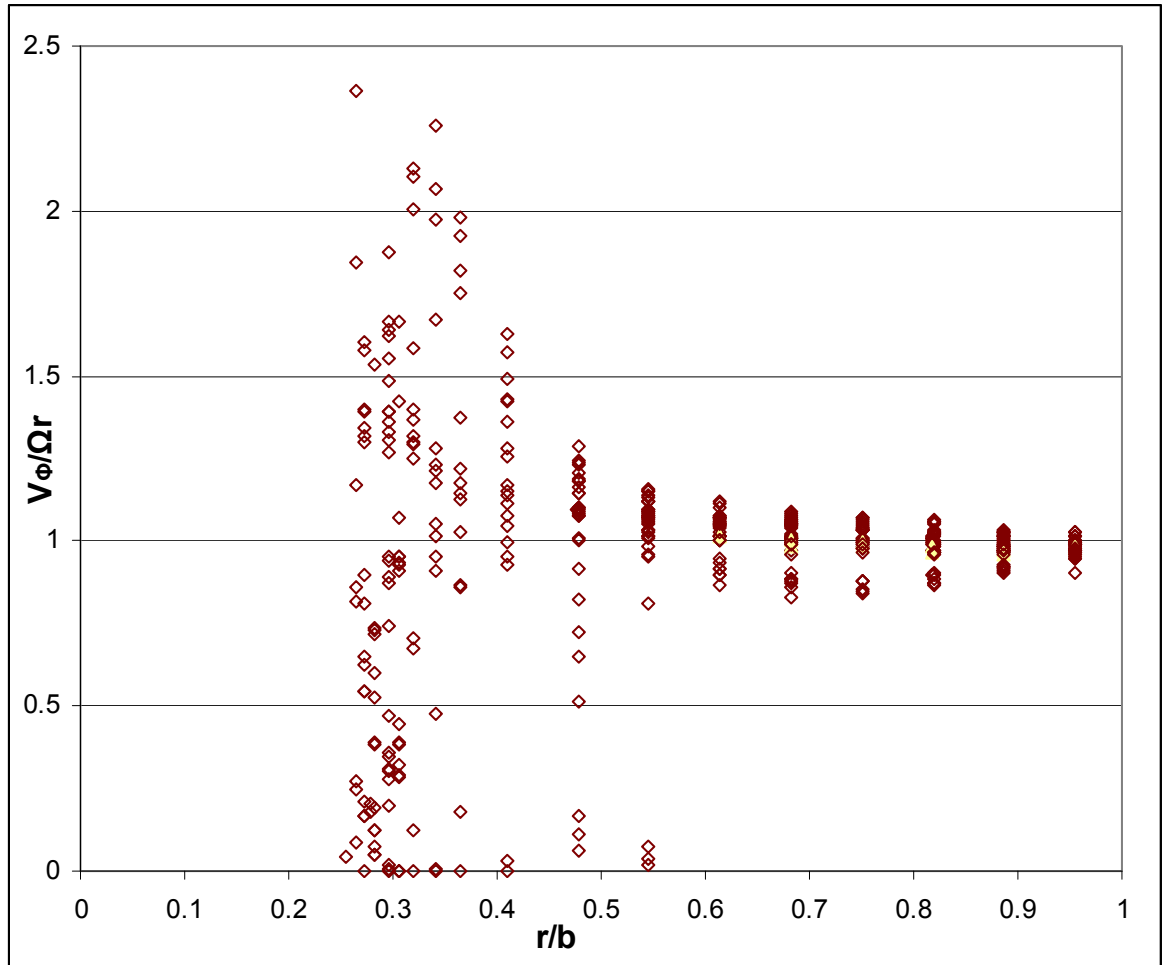


Fig 6. 2 Tangential velocity measurements for Build 3 (all data)

A typical plot of the measured variation of normalised tangential velocity in the Multiple Cavity Rig Build 3 against r/b is shown in Fig 6.3.

The velocity measurements were performed inside Cavity 3 and locations within the throughflow annular gap in axial regions by the cobs of Disc 2 and Disc 3.

The cavity measurement points were positioned along r/b on three different axial locations. The mid axial plane of measurement ($z/s = 0.508$) consisted of fifteen points in the radial range $0.264 \leq r/b \leq 0.955$. At the lower radii, the central measurement line at mid Cavity 3 comprised 6 points more closely spaced at a 5mm radius interval in the region between $0.264 \leq r/b \leq 0.364$ in order to improve the measurement definition in the lower part of the cavity. Measurements in the upstream ($z/s = 0.345$) and downstream ($z/s = 0.671$) parts of Cavity 3 comprised of six points each in the radial range $0.614 \leq r/b \leq 0.955$ for this experimental condition.

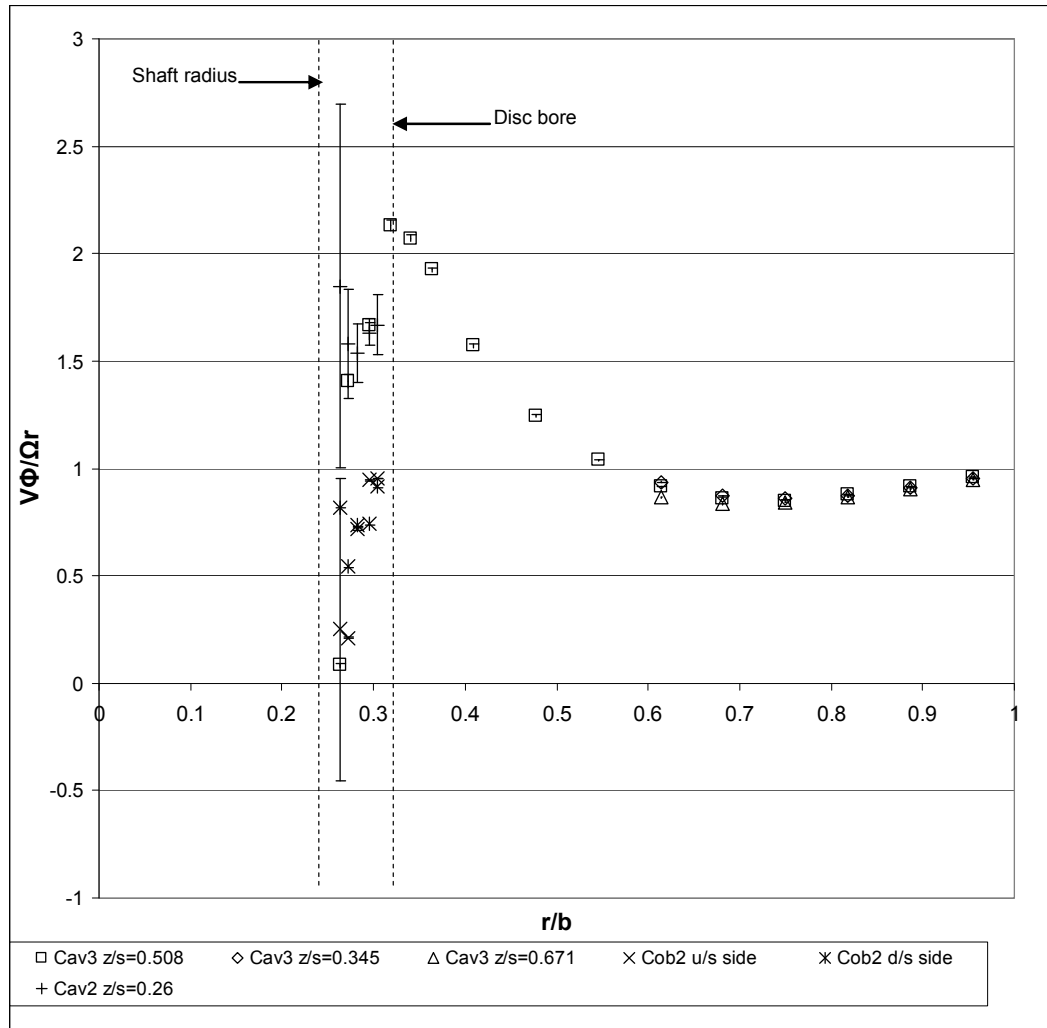


Fig 6. 3 Normalised tangential velocity plot for B3run32-1, $Re_\phi = 0.49 \times 10^6$, $Re_z = 0.92 \times 10^5$, $Ro = 3.56$, in Build 3 (wide annular gap)

The five measuring points under Cob 2 downstream side, Cob 3 upstream side, and the downstream corner of Cob 1 ($z/s = 0.26$ in Cavity 2), were close spaced together within the radius region representing the gap space of the central throughflow annulus (between the shaft, $r/b = 0.236$ and the disc bore, $r/b = 0.318$) in the range $0.264 \leq r/b \leq 0.305$.

On some other tests, the number of points and their locations could vary, therefore the measurement points locations shall be mentioned when appropriate in the results presented.

The velocity plots shown in Figure 6.3 were produced for an operating condition of $Re_\phi = 0.49 \times 10^6$, $Re_z = 0.92 \times 10^5$ and $Ro = 3.56$. The error bars associated with each velocity measurement represent the 95% uncertainty confidence interval values on the time averaged velocity samples as explained in Chapter 5. The

velocity measurements under Cob 2 show the tangential velocity varying from $V_{\phi}/\Omega r \approx 0.25$ close to the shaft surface to $V_{\phi}/\Omega r \approx 0.9$ by the bore of Disc 2.

For the measurements at the lower radii of the upstream part of Cavity 2 (by the axial location of the downstream face of Cob 2, $z/s = 0.26$), the tangential velocities measured show magnitudes of $V_{\phi}/\Omega r \geq 1.54$ along the measurement line, with peaks at the extremities of $V_{\phi}/\Omega r = 1.85$ at $r/b = 0.264$ and $V_{\phi}/\Omega r = 1.67$ at $r/b = 0.305$.

The tangential velocity profile at mid-Cavity 3 on the lower radii show a sharp tangential velocity increase from $V_{\phi}/\Omega r = 0.08$ by the shaft at $r/b = 0.264$, culminating at $V_{\phi}/\Omega r = 2.13$ at $r/b = 0.318$, which corresponds approximately to the discs bore radius.

It is believed that the overspinning of the central throughflow at the junction of a cavity is the result of the conversion of axial momentum being redirected tangentially as the flow penetrates within the inner section of the cavity. A radial inflow in the discs boundary layer at the inner radii of the cavity should occur in order to recirculate the flow being 'spun out' into the cavity but this could only be assumed as no velocity measurements were possible in this region of the cavity due to the limited optical access to the discs surfaces posed by the protruding discs cobs.

Progressing further outwards the cavity, the relative tangential velocities show a decrease with radius and tend to reach a value close to solid body rotation by $r/b \approx 0.6$. No significant differences of tangential velocities were found across the axial width of the cavity.

Variations in the instantaneous velocities of the measured flows can be represented in relation with the RMS values of the velocity samples at each position. Figure 6.4 shows the turbulence intensity (defined in Eq.4.25) related to the tangential velocity measurements shown in Figure 6.1 for mid Cavity 3. It can be related that the flow within the throughflow annular gap presents a level of turbulence indicating highly fluctuating tangential velocities. The levels of turbulence intensity sharply reduce up to a radius corresponding to the discs bore and then further reduces, albeit slowly, as progressing radially through the cavity.

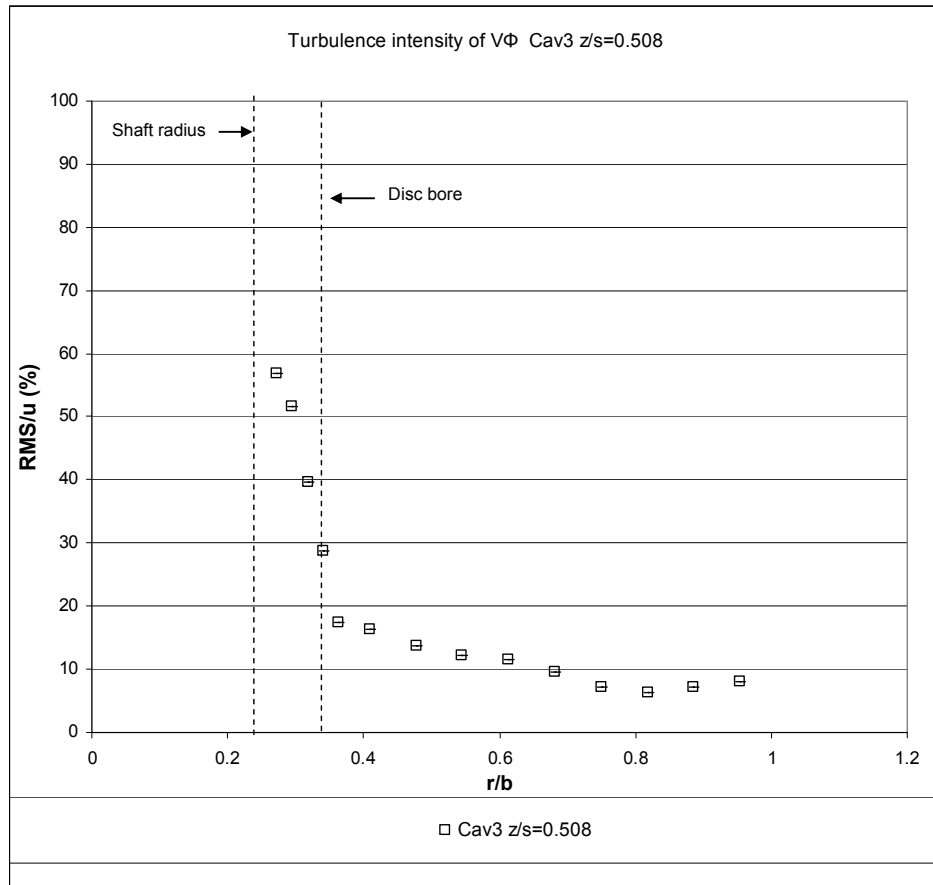


Fig 6. 4 Turbulence intensity of V_ϕ against r/b in mid Cavity 3 for B3run32-1, $Re_\phi = 0.49 \times 10^6$, $Re_z = 0.92 \times 10^5$ and $Ro = 3.56$

Figure 6.5 shows a contour plot interpretation of the measured normalised tangential velocities shown in Figure 6.3. The boundaries of this contour plot were attributed a non slip flow condition, $V_\phi/\Omega r = 1$, except on the (non-rotating) shaft surface where $V_\phi/\Omega r = 0$. The measurement point locations are materialised by squares on the plot.

The extent of the overspinning region by the entrance of Cavity 3 can clearly be seen from the plot, while a similar pattern can be guessed from the measurements at the entrance of Cavity 2. The swirling profile of the throughflow along the shaft annulus also clearly stabilises in the disc bore passage before being spun up again by interacting with Cavity 3.

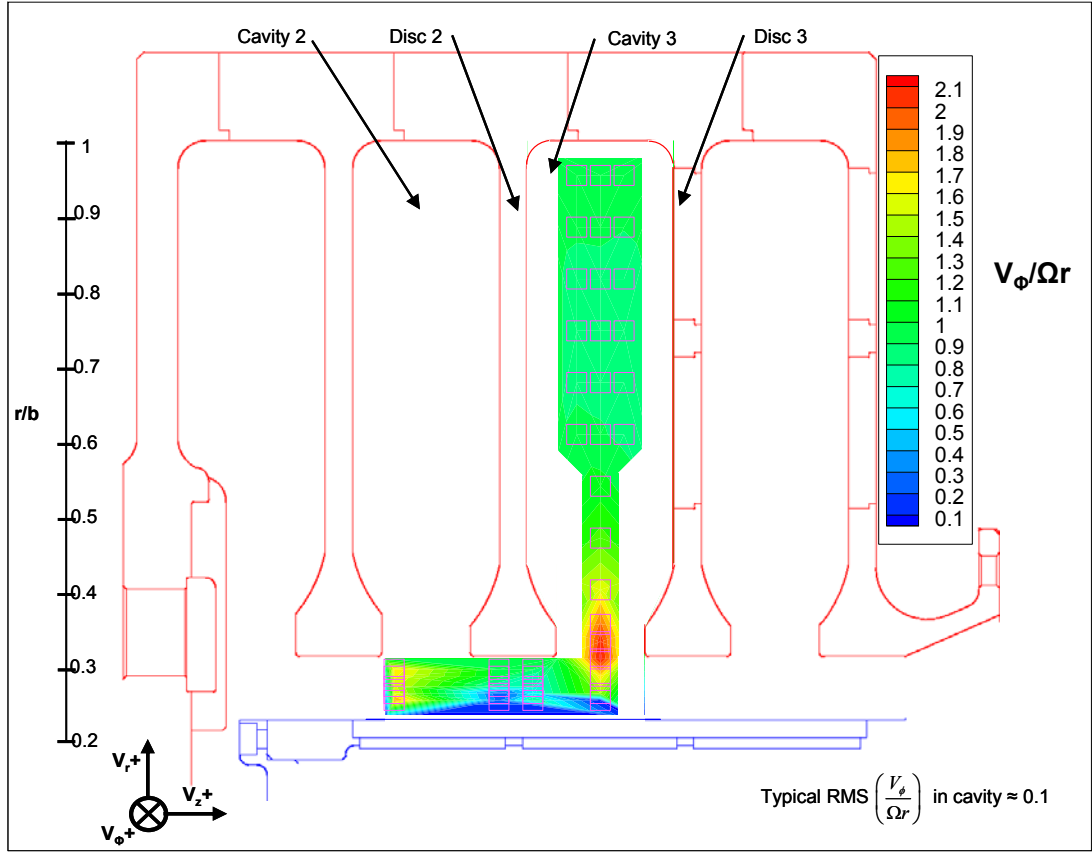


Fig 6. 5 Normalised tangential velocity contour plot for B3run32-1, $Re_\phi = 0.49 \times 10^6$, $Re_z = 0.92 \times 10^5$ and $Ro = 3.56$

The axial velocity measurements for this test are shown in Figure 6.6, with the error bars on the plot representing the 95% interval of confidence of the average measurements.

In the central annular throughflow region of the rig ($0.236 \leq r/b \leq 0.318$) the measured axial velocities are consistent with the bulk average velocity ($W = 17.45$ m/s in this instance) ranging within about $0.7 \leq V_z/W \leq 1.5$. In radial locations within Cavity 3 ($r/b \geq 0.318$), the axial velocities tend towards zero. The associated normalised RMS values on the averaged axial velocity measurements shown in Figure 6.7 indicate an increasing level of local velocity fluctuations for $0.236 \leq r/b \leq 0.318$, then a sharp decrease when entering Cavity 3, followed by a steady decrease as progressing radially through the cavity towards $r/b \approx 1$. The turbulence intensity (Eq. 4.25) levels achieved in the central throughflow reached values of the order of $RMS/\bar{u} = 53\%$ for $r/b = 0.38$ in the mid-plane of Cavity 3. The normalised axial velocity RMS values (RMS/W) were chosen to be plotted instead of the turbulence intensity in Figure 6.7 as the

excessively high values obtained for measured points where \bar{u} (Eq. 4.21) tended towards zero were not representative of the overall turbulence levels of the flow within the cavity.

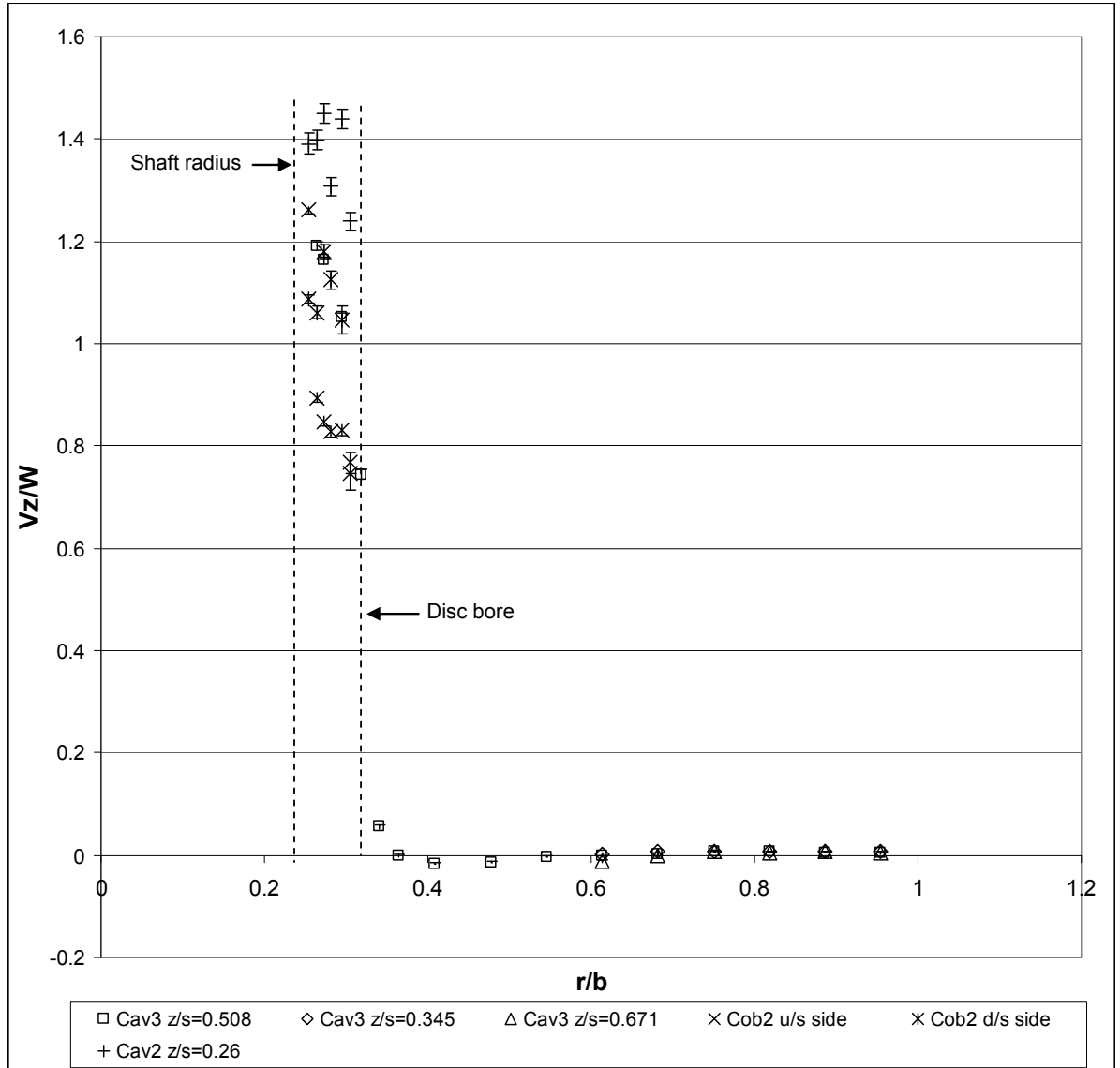


Fig 6. 6 Normalised axial velocity measurements for B3run32-1, $Re_\phi = 0.49 \times 10^6$, $Re_z = 0.92 \times 10^5$ and $Ro = 3.56$

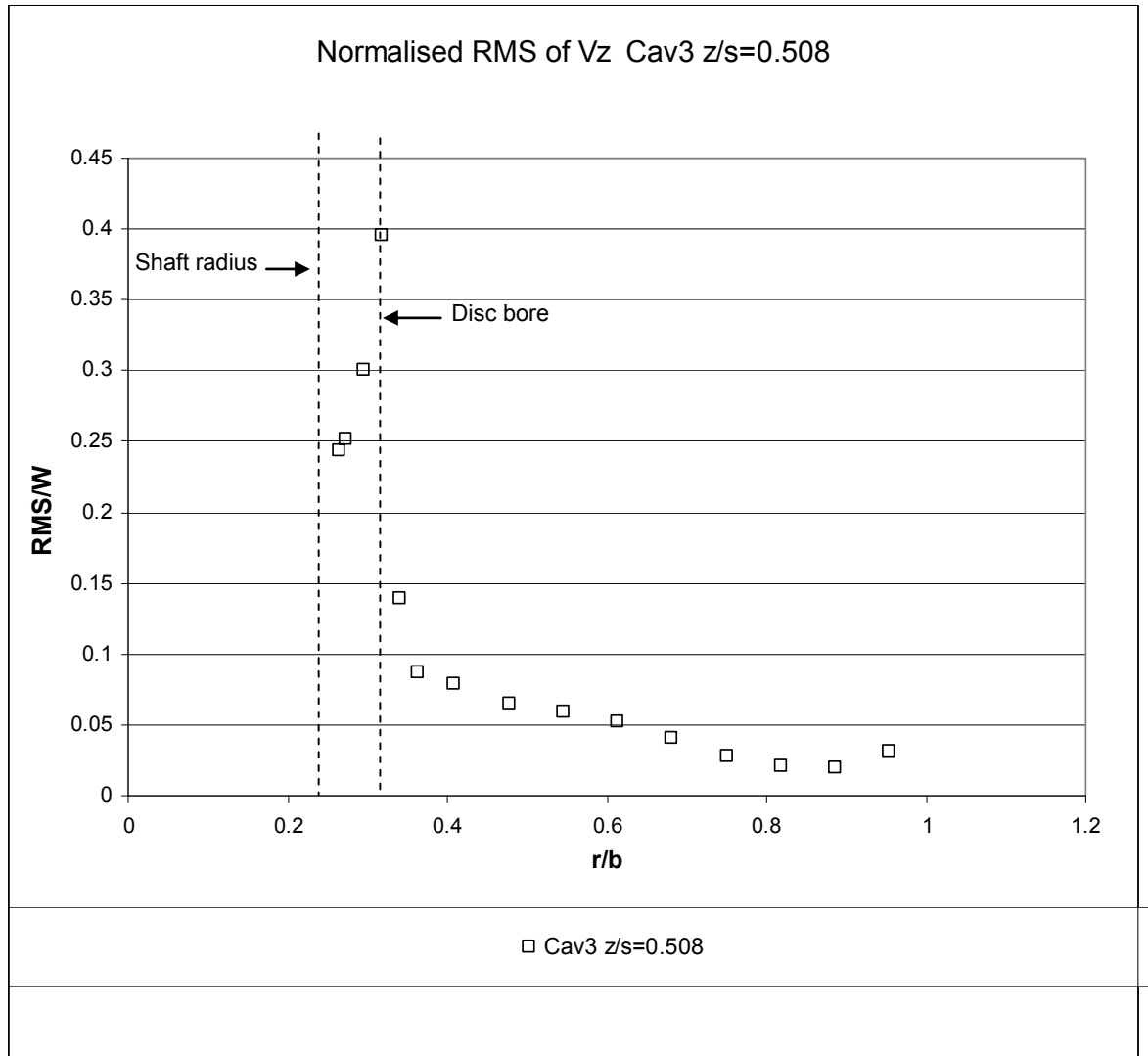


Fig 6. 7 Normalised RMS of V_z in mid-cavity 3 for B3run32-1, $Re_\phi = 0.49 \times 10^6$, $Re_z = 0.92 \times 10^5$ and $Ro = 3.56$

Figure 6.8 shows the contour plot of V_z/W in the measurement volume of the test rig for the considered test. The boundaries of this contour plot were attributed a non slip condition, $V_z=0$. A boundary condition had to be attributed on the upstream and downstream boundaries of the central annulus by the plotting software. Therefore the attention of the Reader is drawn to the display anomalies shown on the upstream and downstream regions due to the presence of a zero velocity boundary condition and is invited to bear this in mind when looking at the contour plot.

The interpretation of the plotted axial velocity measurements highlights an accelerated axial throughflow in the central annular gap within the cavities entrance regions (with $V_z/W > 1$) while the flow appears steadier while passing under Disc 2 ($V_z/W \approx 1$). This presents similarities with the tangential velocity measurements shown in Figure 6.3 which indicated a more stable flow under Disc 2.

The acceleration of the axial throughflow in the connecting regions of the cavities could result from an interaction with the cavity flow whereby the bulk of the throughflow might have to 'squeeze by' the interacting zone and therefore accelerate locally in the process as a result of conservation of mass.

The flow at the entrance of Cavity 3 appears to slow down from $V_z/W = 0.74$ at $r/b = 0.318$ (disc bore radius) to zero at $r/b = 0.364$. A clockwise recirculation motion also appears to be present in the remaining radially out region of Cavity 3, although of very low magnitudes.

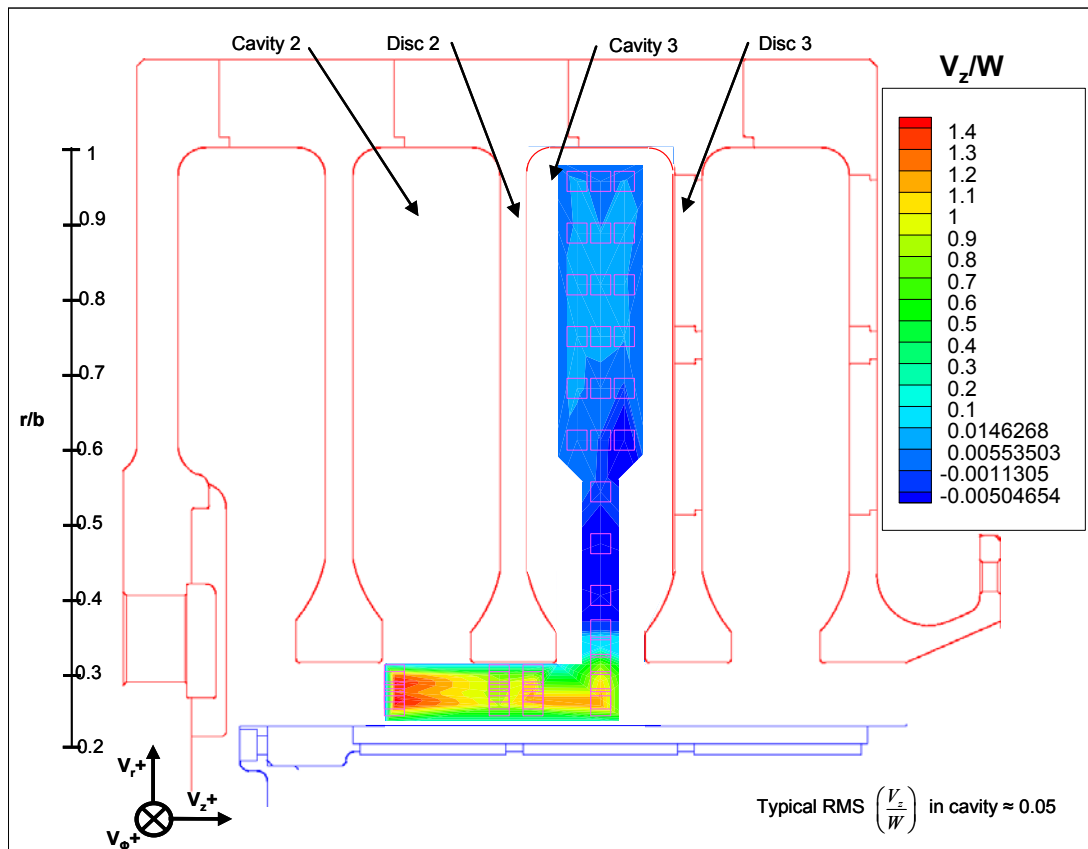


Fig 6. 8 Normalised axial velocity contour plot for B3run32-1, $Re_\phi = 0.49 \times 10^6$, $Re_z = 0.92 \times 10^5$ and $Ro = 3.56$

Figure 6.9 shows the normalised tangential velocity contour plot for an operating condition of $Re_\phi = 4.12 \times 10^6$, $Re_z = 1.16 \times 10^5$ and $Ro = 0.54$. This condition represents the other end of the spectrum of operating conditions of the rig with a flow condition expected to be rotationally dominated ($Ro = 0.54$), compared to the previously described condition in Figures 6.1 to 6.8 which is expected to be axially dominated ($Ro = 3.56$). The tangential velocity flow behaviour of the latter condition on Figure 6.9 is broadly similar to the previous condition, with an overspinning of the flow in the shaft annular gap region, reaching a maximum by the entrance of Cavity 3, then decreasing sharply as progressing radially through the cavity towards $r/b = 0.4$, then continuing to slow down progressively, reaching solid body rotation by the outer part of the cavity.

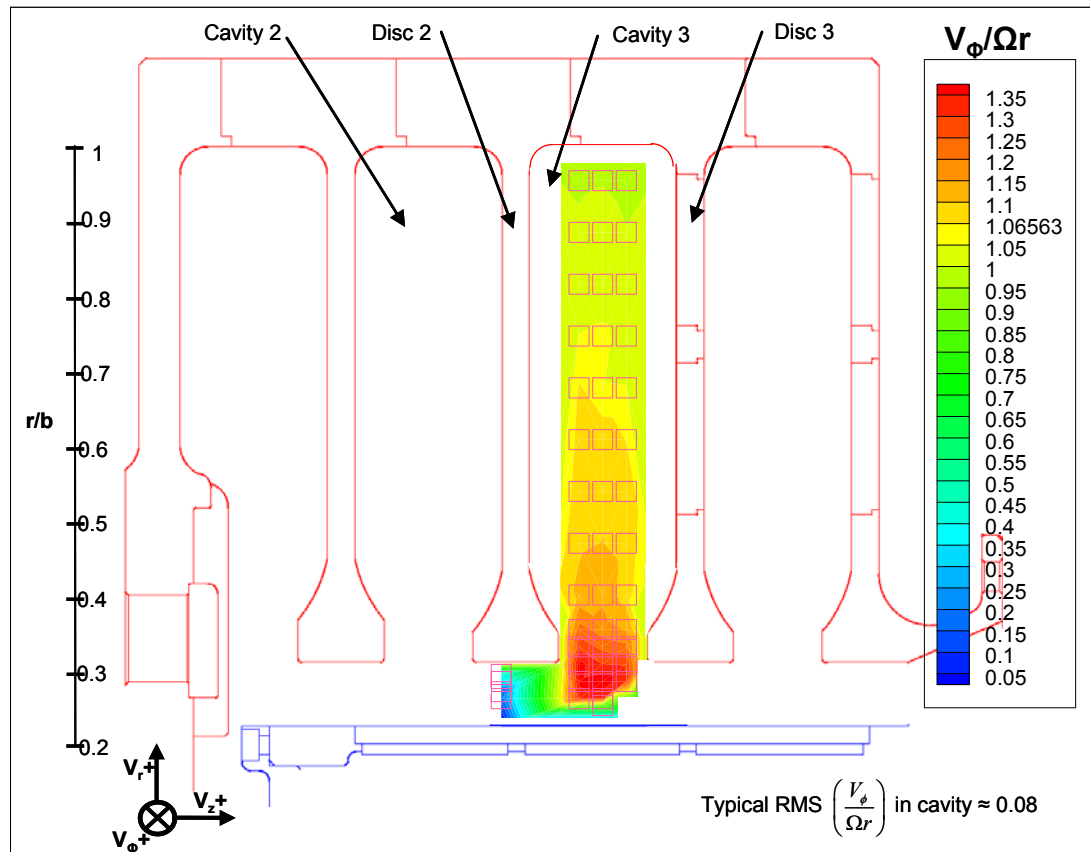


Fig 6. 9 Normalised tangential velocity contour plot for B3run28-1, $Re_\phi = 4.12 \times 10^6$, $Re_z = 1.16 \times 10^5$ and $Ro = 0.54$

The maximum overall level of swirl achieved was in the order of $V_\phi / \Omega r \approx 1.4$, which is less than achieved in the previous test for $Ro = 3.56$ ($V_\phi / \Omega r \approx 2.1$).

Tangential velocity measurements under the bore of Disc 2 returned values of $V_\phi/\Omega r = 0.17$ for $r/b = 0.27$ and $V_\phi/\Omega r = 0.38$ at $r/b = 0.30$.

Velocity measurements by the shaft proved more difficult to obtain at this operating condition, possibly due to an increase in vibrations from a much higher rotational speed of the rig (4552 rev/min against 666 rev/min for the previously described condition).

The corresponding contour plot of the normalised axial velocities is shown in Figure 6.10.

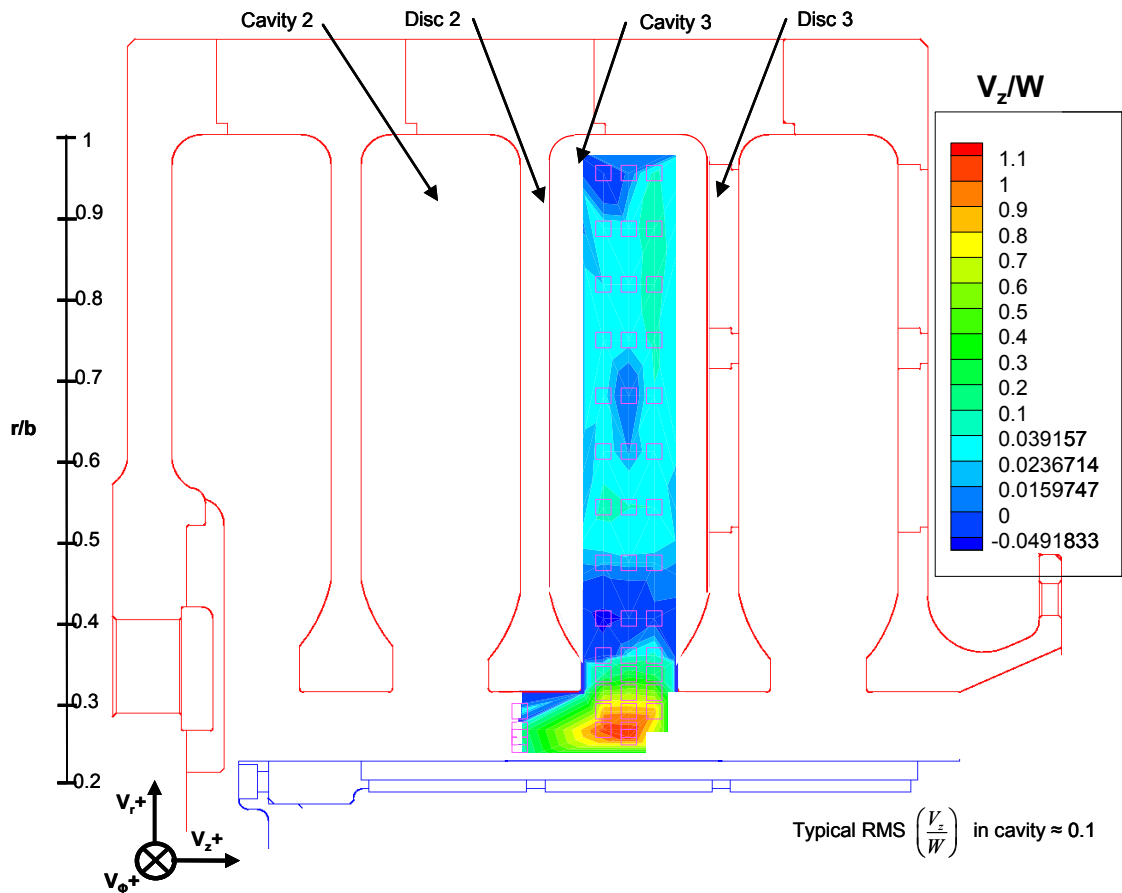


Fig 6. 10 Normalised axial velocity contour plot for B3run28-1, $Re_\phi = 4.12 \times 10^6$, $Re_z = 1.16 \times 10^5$ and $Ro = 0.54$

Axial velocity measurements under the bore of Disc 2 resulted in values of $V_z/W = 1.15$ at $r/b = 0.255$, $V_z/W = 1.18$ at $r/b = 0.264$, $V_z/W = 0.11$ at $r/b = 0.273$ and $V_z/W \approx 0$ for $0.282 \leq r/b \leq 0.305$. The contour plot generation unfortunately

failed to show these values on Figure 6.10 for the same reasons of boundary conditions as explained before for the case in Figure 6.8.

The axial flow at the entrance of Cavity 3, mid-plane, shows a decrease in velocity from $V_z/W=1.1$ at r/b , $V_z/W=0.59$ at $r/b=0.318$ (disc bore radius), to $V_z/W \approx 0$ for $r/b \geq 0.364$. The axial velocity measurements further inside the cavity appear to show a relatively quiescent region of flow with no noticeably clear recirculations present.

From the plots above, differences in flow patterns and magnitudes between the two previously described conditions (high throughflow – low rotation speed and low throughflow – high rotation speed respectively) can be observed. The influence on the tangential velocity within the mid axial plane of Cavity 3 with increasing Re_ϕ for a fixed axial Reynolds number of $Re_z \approx 1 \times 10^5$ is shown in Figure 6.11. The test conditions featured in Figure 6.11 are detailed in Table 6.3.

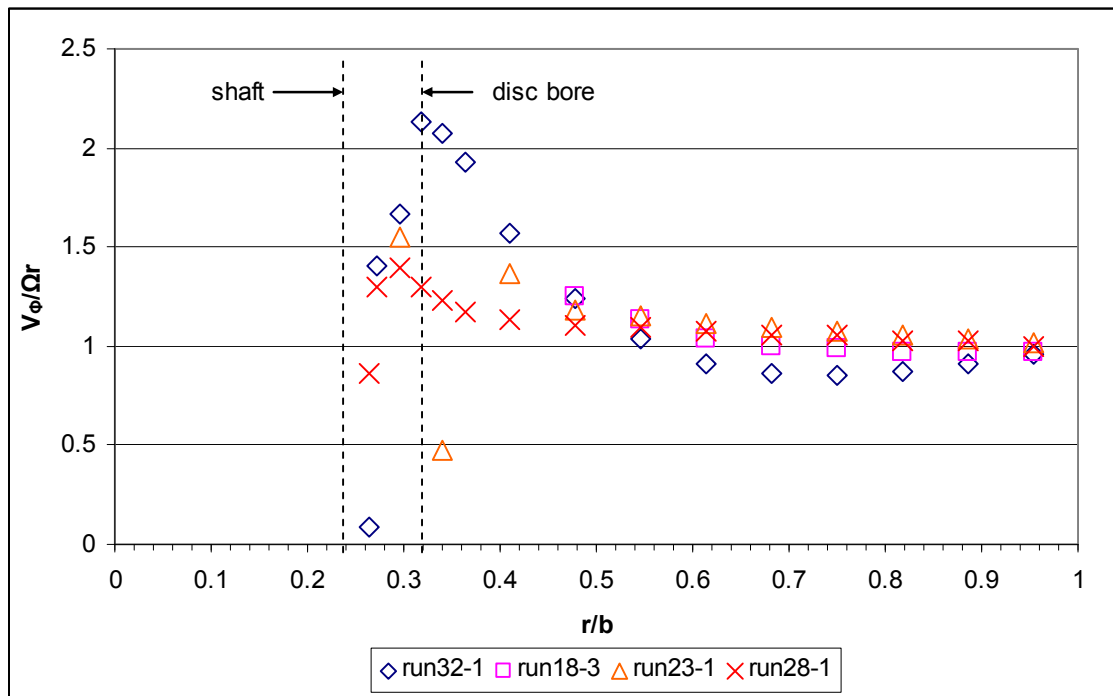


Fig 6. 11 Comparison of normalised tangential velocities in Build 3 at mid Cavity 3 with increasing Re_ϕ for fixed $Re_z \approx 10^5$

| Test | $Re_\phi/10^6$ | $Re_z/10^5$ | Ro | $\beta\Delta T_{av}$ | $\beta\Delta T_{sh,C3}$ | $Gr_{sh,C3}/10^7$ |
|---------|----------------|-------------|------|----------------------|-------------------------|-------------------|
| Run32-1 | 0.49 | 0.92 | 3.56 | 0.09 | 0.23 | 5.3 |
| Run18-3 | 1.03 | 1.01 | 1.87 | 0.05 | 0.18 | 17.1 |
| Run23-1 | 2.02 | 0.99 | 0.93 | 0.04 | 0.13 | 46.1 |
| Run28-1 | 4.12 | 1.16 | 0.54 | 0.03 | 0.11 | 137 |

Table 6. 3 Experimental conditions for Figure 6.9 (Build3)

For the lowest value of $Re_\phi = 0.49 \times 10^6$ ($Ro = 3.56$), the tangential velocity profile in the throughflow annulus possesses the sharpest increase in velocity, reaching a swirl value of $V_\phi/\Omega \approx 2.3$ by the radius corresponding to the disc bore, then decreases with increasing radial location for $r/b > 0.32$. A value just below solid body rotation is achieved at $r/b \approx 0.6$, before gradually reaching solid body rotation by $r/b = 1$. With an increase of Re_ϕ to 1×10^6 ($Ro = 1.87$), 2×10^6 ($Ro = 0.93$) and 4×10^6 ($Ro = 0.54$), the peak value of relative tangential velocity achieved by the entrance of Cavity 3 decreases, reading values of $V_\phi/\Omega \approx 1.3$ for $Re_\phi = 4 \times 10^6$. The tendency for decelerating swirling flows to reduce to values of normalised tangential velocity below solid body rotation in the cavity could indicate a recirculation flow in the tangential plane. The tangential velocity profiles for conditions with $Re_\phi \geq 2 \times 10^6$ show the decelerating tangential velocity values in the cavity above solid body rotation up to $r/b \approx 1$.

A similar trend can be observed when comparing tests at fixed rotational Reynolds number values and increasing Re_z (hence increasing Ro). Figure 6.12 shows a comparison between the tangential velocity profiles along the mid plane of Cavity 3 for operating conditions of increasing axial Reynolds number values $Re_z \approx 0.59 \times 10^5$, 1×10^5 and 1.7×10^5 at fixed $Re_\phi \approx 1.1 \times 10^6$. The details of the operating conditions of the tests plotted in Figure 6.12 are shown in Table 6.4.

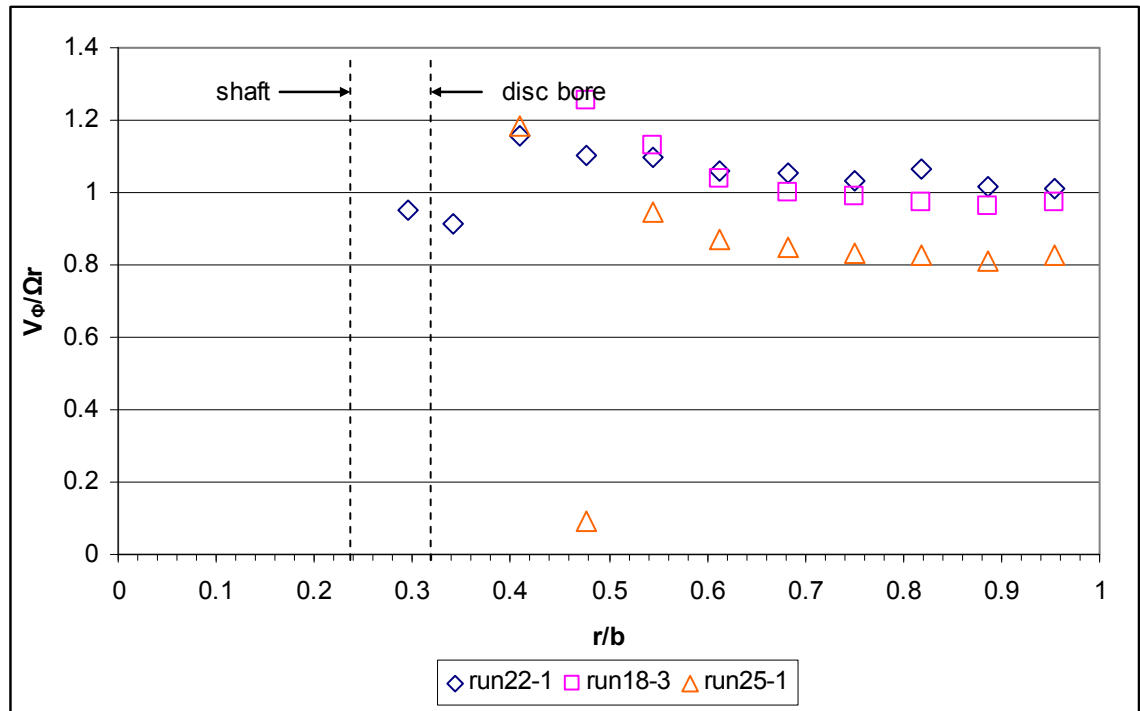


Fig 6. 12 Comparison of normalised tangential velocities in Build 3 at mid Cavity 3 with increasing Re_z for fixed $Re_\phi \approx 1.1 \times 10^6$

| Test | $Re_\phi / 10^6$ | $Re_z / 10^5$ | Ro | $\beta \Delta T_{av}$ | $\beta \Delta T_{sh,C3}$ | $Gr_{sh,C3} / 10^8$ |
|---------|------------------|---------------|------|-----------------------|--------------------------|---------------------|
| Run22-1 | 1.24 | 0.49 | 0.75 | 0.06 | 0.17 | 2.43 |
| Run18-3 | 1.03 | 1.01 | 1.87 | 0.05 | 0.18 | 1.71 |
| Run25-1 | 1.15 | 1.71 | 2.84 | 0.05 | 0.17 | 1.97 |

Table 6. 4 Experimental conditions for Figure 6.10 (Build3)

Although the velocity measurements shown in Figure 6.12 are incomplete at the lower radii ($r/b < 0.4$), the velocity profiles shown further inside the cavity ($r/b \geq 0.409$) would suggest a similar influence of the increase of Ro on the cavity velocity profiles. It can only be assumed that the influence of Ro on the peak swirl velocity in the inner region of the cavity ($r/b < 0.4$) will be similar to that observed in Figure 6.11.

The results shown in Figure 6.12 and 6.11 therefore suggest that the distribution of tangential velocity in the inner region of the cavity ($r/b < 0.5$) is governed by the Rossby number, with an increase in the magnitude of $V_\phi/\Omega r$ as the operating conditions become more dominated by the throughflow (large Ro).

It is interesting to notice some similarities with the observations in an isothermal cavity with throughflow by Farthing et al. (1992a) who described a similar increase of the relative tangential component to above solid body rotation in the central throughflow region of the cavity that was decreasing to solid body rotation as progressing radially out. They also observed an increase of the maximum relative swirl by the disc bore with increasing Rossby number.

Figure 6.13 compares the values of $V_\phi/\Omega r$ taken from Build 3 and Build 2 of the Multiple Cavity Rig. The principal geometrical difference between Build 2 and Build 3 of the Multiple Cavity Rig, as previously mentioned, lies in a shaft radius decrease between Build 2 and Build 3 from $r_s = 60$ mm to $r_s = 52$ mm. The corresponding non-dimensional annular gap sizes are $d_h/b = 0.164$ for Build 3 (wide annular gap) and $d_h/b = 0.092$ for Build 2 (narrow annular gap).

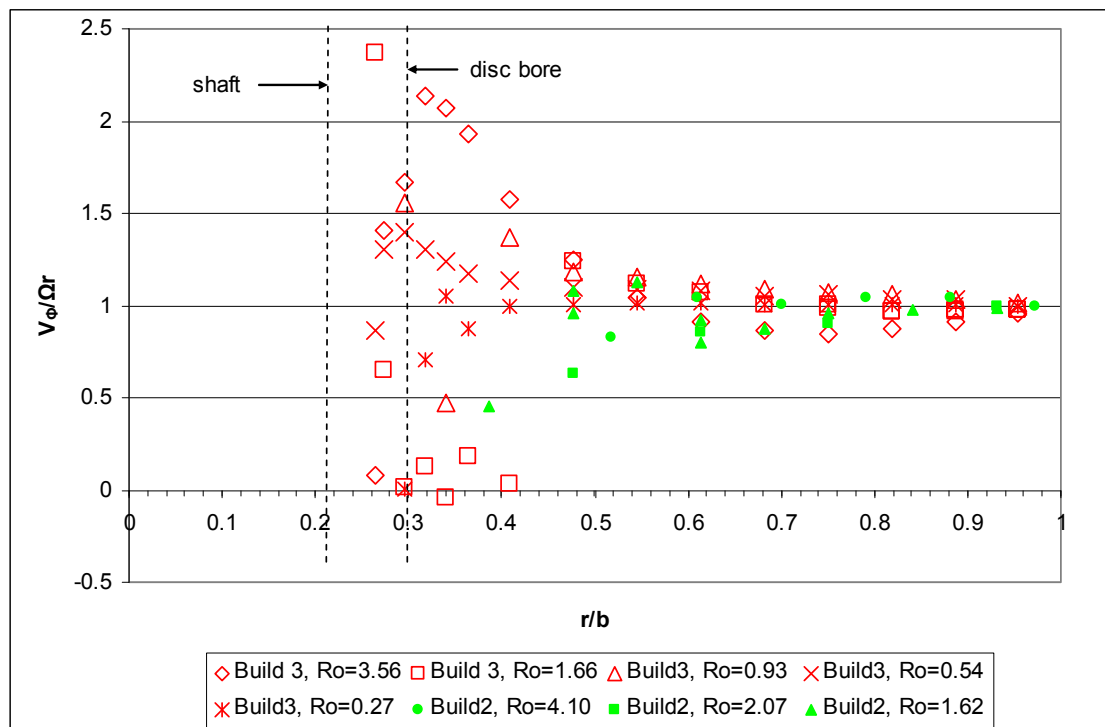


Fig 6. 13 Tangential velocity profiles at mid Cavity 3 for Build 2 and Build 3 with varying Ro

As also seen in Figure 6.1, the velocity measurements along the cavity radius in Build 2 shown in Fig. 6.13 suggest a gradual acceleration of the flow from the shaft region tending to $V_\phi/\Omega r = 1$ towards the outer radii of Cavity 3.

Some support to the tangential velocity observations on Build 2 can be found from the measurements of Owen and Powell (2004) who obtained similar cavity tangential velocity flow profiles behaviour on their rig with a narrower annular gap ($d_h/b = 0.026$) to Build 2 for a range of experimental conditions. It is worth noting that the low values of tangential velocity recorded for $Ro = 1.66$ between $0.26 \leq r/b \leq 0.42$ do not fit into the explication above and could be the result of a measurement fault or other phenomenon that cannot be explained by the author at present.

It would therefore appear that the tangential velocity profile characteristics in a rotating cavity with axial throughflow are principally governed by the influence of the throughflow interacting with the entrance of the cavity, that a variation in the throughflow dominance (variation in Ro) would affect the cavity tangential velocity profile. However the effects of the throughflow can be largely attenuated by reducing the annular gap ratio of the cavity, d_h/b , and also by reducing the cavity gap ratio, G , as previously demonstrated by Farthing et al (1992a).

Figures 6.14 and 6.15 show a comparison of tangential velocity profiles between Cavities 2, 3 and 4 in Build 3 and Build 2 respectively of the Multiple Cavity Rig.

The differences in measured tangential velocities shown for both builds between Cavities 2, 3 and 4 would suggest some evidence of variations of cavity tangential velocity profile behaviour as progressing through the rigs' cavities along the throughflow direction. From what was described above, the variations of the cavities tangential flow could perhaps suggest an increase of the influence of the throughflow interaction with the cavities as progressing through the rig, similarly to a local increase in Ro . This could suggest a cumulative influence of the exit flow conditions of one cavity onto the next, affecting the mechanism of vortex breakdown which in turn would affect the tangential distribution inside the cavities.

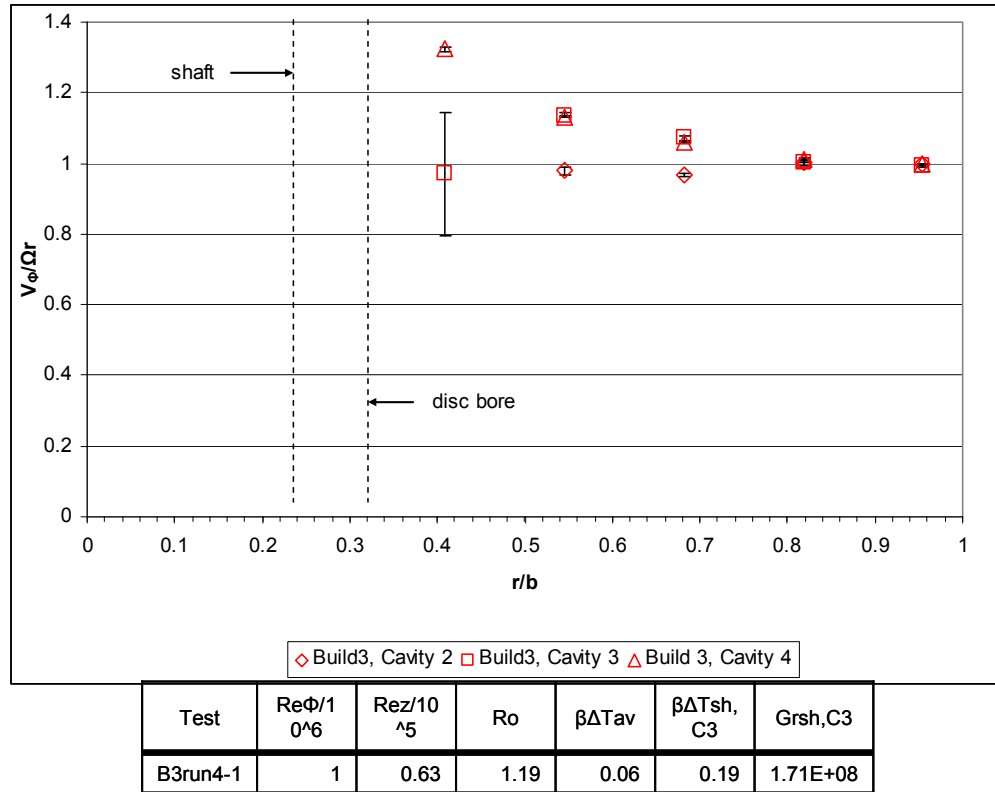


Fig 6. 14 Tangential velocity profile comparison between Cavities 2, 3 and 4 in Build 3

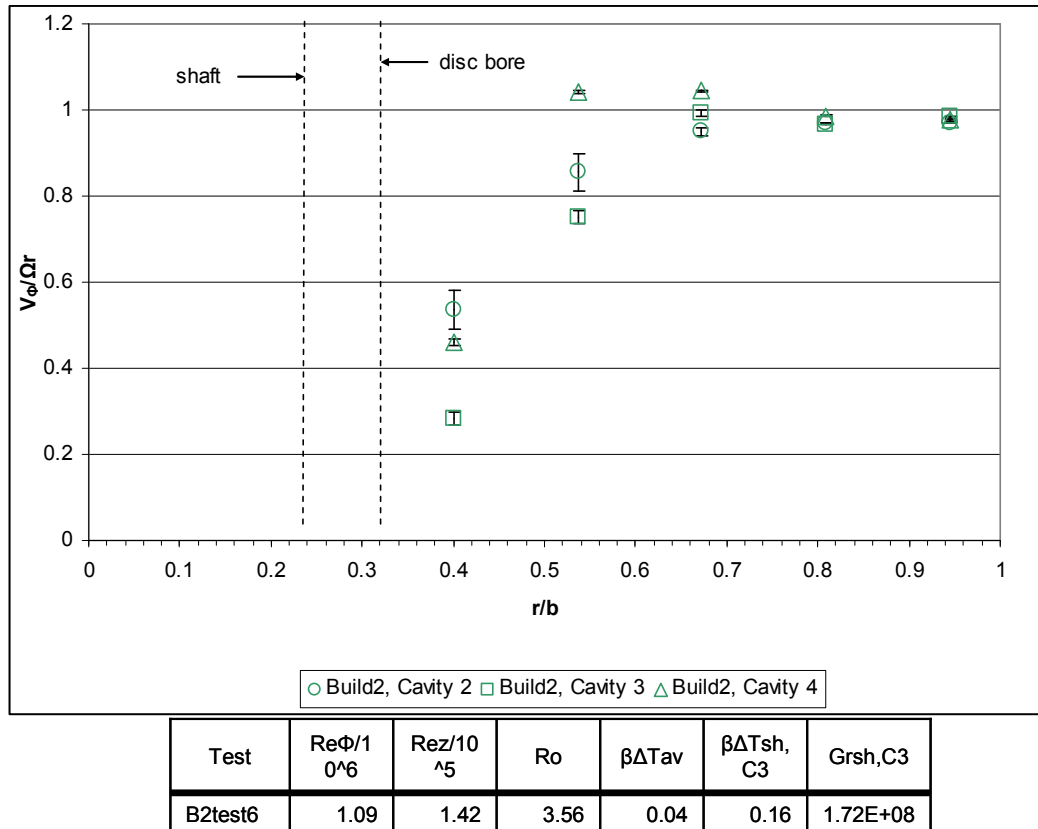


Fig 6. 15 Tangential velocity profile comparison between Cavities 2, 3 and 4 in Build 2

6.2 Radial velocity measurements

Simultaneous radial and tangential velocity measurements were performed in Build 3 of the Multiple Cavity Rig using the optical access of the passing windows of Disc 3 and Disc 4 as described in Chapter 3. The measurement regions in Cavity 3 and 4 were within $0.067 \leq z/s \leq 0.946$ and $0.565 \leq r/b \leq 0.925$. Figures 6.14, 6.15 and 6.16 show the time averaged radial velocity measurements at respectively $r/b = 0.924$, $r/b = 0.765$, and $r/b = 0.566$ across the cavities width for a test condition of $Re_\phi = 0.4 \times 10^6$, $Re_z = 1.57 \times 10^5$ and $Ro = 7.51$ (B3run 38-1).

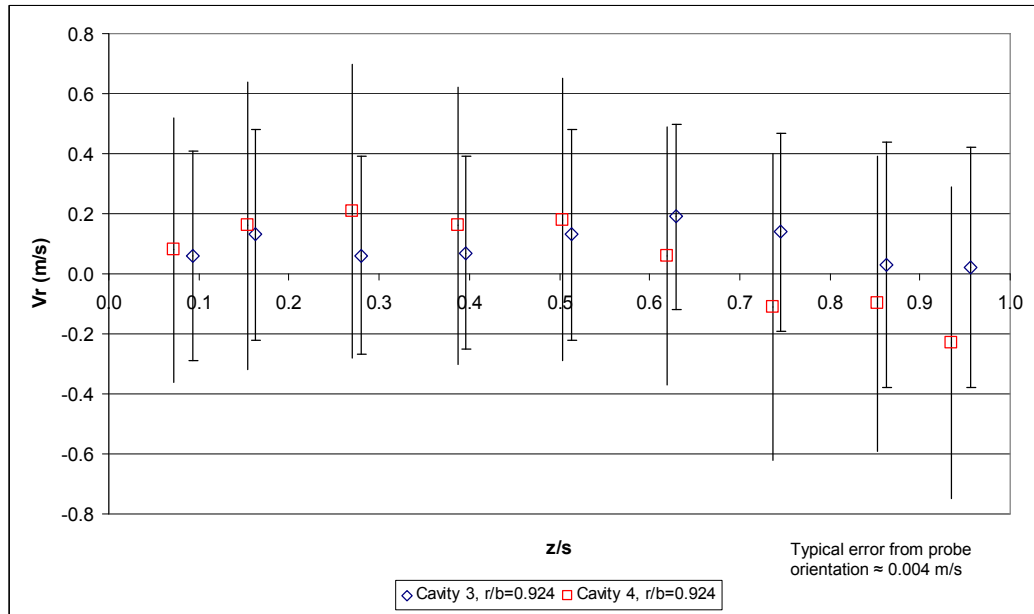


Fig 6. 16 Radial velocity measurements in Cavities 3 and 4 at $r/b = 0.924$ for $Re_\phi = 0.4 \times 10^6$, $Re_z = 1.57 \times 10^5$ and $Ro = 7.51$

The error bars on the velocity plot points of Figure 6.16, 6.17 and 6.18 represent the flow measurement spread, comprising the RMS of the measured velocity and the probe orientation error (Eq. 5.16). Positive values of radial velocity correspond to a cavity radial outflow while $z/s = 0$ corresponds to the upstream side disc wall of the considered cavity. The peak magnitude of the radial velocities in the present case can be seen from the plots to be in the order of $V_r \leq 0.25$ m/s at $r/b = 0.765$, which is about 0.7 % of the bulk average velocity ($W = 36.10$ m/s) and about 1.6% of the solid-body tangential velocity at $r = b$ ($\Omega b = 15.09$ m/s).

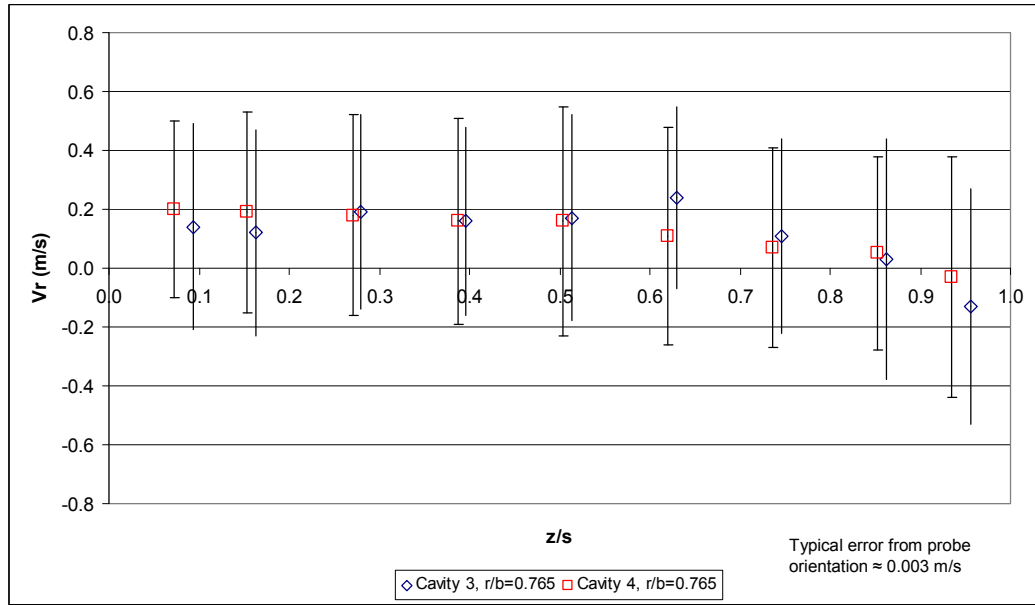


Fig 6. 17 Radial velocity measurements in Cavities 3 and 4 at $r/b = 0.765$ for $Re_\phi = 4 \times 10^6$, $Re_z = 1.57 \times 10^5$ and $Ro = 7.54$

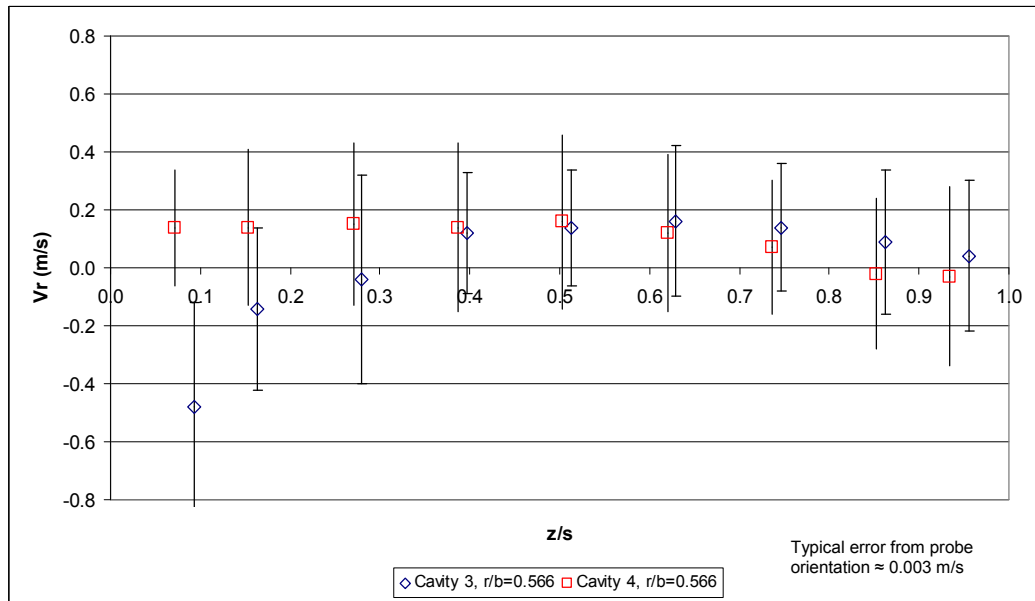


Fig 6. 18 Radial velocity measurements in Cavities 3 and 4 at $r/b = 0.566$ for Build 3, run38-1, $Re_\phi = 0.4 \times 10^6$, $Re_z = 1.57 \times 10^5$ and $Ro = 7.51$

The measured tangential velocities in this optical access configuration provided similar information than previously measured, i.e. velocities close to solid body rotation considering the location of the measuring region beyond $r/b \approx 0.6$. The measured tangential velocities were therefore deemed not necessary to be included in the following results for ease of reading.

The magnitude of the RMS values of the sampled radial velocities exceeded the magnitude of the measured radial velocities by a factor of two typically. This level of fluctuations in the radial velocities could be the result of circulation flows within the cavity as characterised by Farthing et al. (1992a), or turbulence resulting from the interaction with the central throughflow. The magnitude of the RMS is seen to increase as Ro decreases, probably suggesting vortex breakdown occurring as the flow conditions become rotationally dominated.

The magnitude of the error due to probe orientation represents about 1% of the magnitude of the RMS of the flow measurement. Typical values are mentioned included in Fig 6.16 to 6.18.

Contour plots of the local measured radial velocities in Cavity 3 and 4 of Build 3 of the Multiple Cavity Rig were generated for easier interpretation of the results. The considered experimental conditions are detailed in Table 6.5, the first condition of which was previously described in Figures 6.16 to 6.18.

| Condition | Test | $Re_\phi/10^6$ | $Re_z/10^5$ | Ro | $\beta\Delta T_{av}$ | $\beta\Delta T_{sh,C3}$ | $Gr_{sh,C3}/10^7$ |
|-------------|---------|----------------|-------------|------|----------------------|-------------------------|-------------------|
| Condition 1 | run38-1 | 0.40 | 1.57 | 7.51 | 0.067 | 0.22 | 3.3 |
| Condition 2 | Run40-1 | 0.39 | 0.61 | 2.98 | 0.107 | 0.24 | 3.4 |
| Condition 3 | Run44-1 | 1.99 | 1.65 | 1.58 | 0.004 | 0.11 | 37 |
| Condition 4 | Run39-1 | 1.93 | 0.53 | 0.53 | 0.001 | 0.08 | 25 |

Table 6. 5 Radial velocity measurements experimental conditions (Build3)

Figure 6.19 shows the radial velocity contour plot for Build 3, run38-1, $Re_\phi=0.4 \times 10^6$, $Re_z=1.57 \times 10^5$, $Ro=7.51$. The measurements points are shown as squares on the plot.

The radial velocity profiles in Cavity 3 show a central outflow region of peak magnitude $V_r \approx 0.2$ m/s originating from the inner section of the cavity reaching up to $r/b \approx 0.9$. On the upstream side of Cavity 3 (about $z/s = 0.1$), a relatively strong radial inflow of velocity $V_r \approx (-0.2)$ m/s is showing from $r/b = 0.57$, extending up to $r/b = 0.68$. At higher radii, the flow appears relatively quiescent with radial velocities decreasing to close to zero. The downstream side of Cavity 3 (by $z/s = 0.95$) shows a clear radial inflow region along the radial length of the measurement area, with a magnitude $V_r \approx (-0.1)$ m/s.

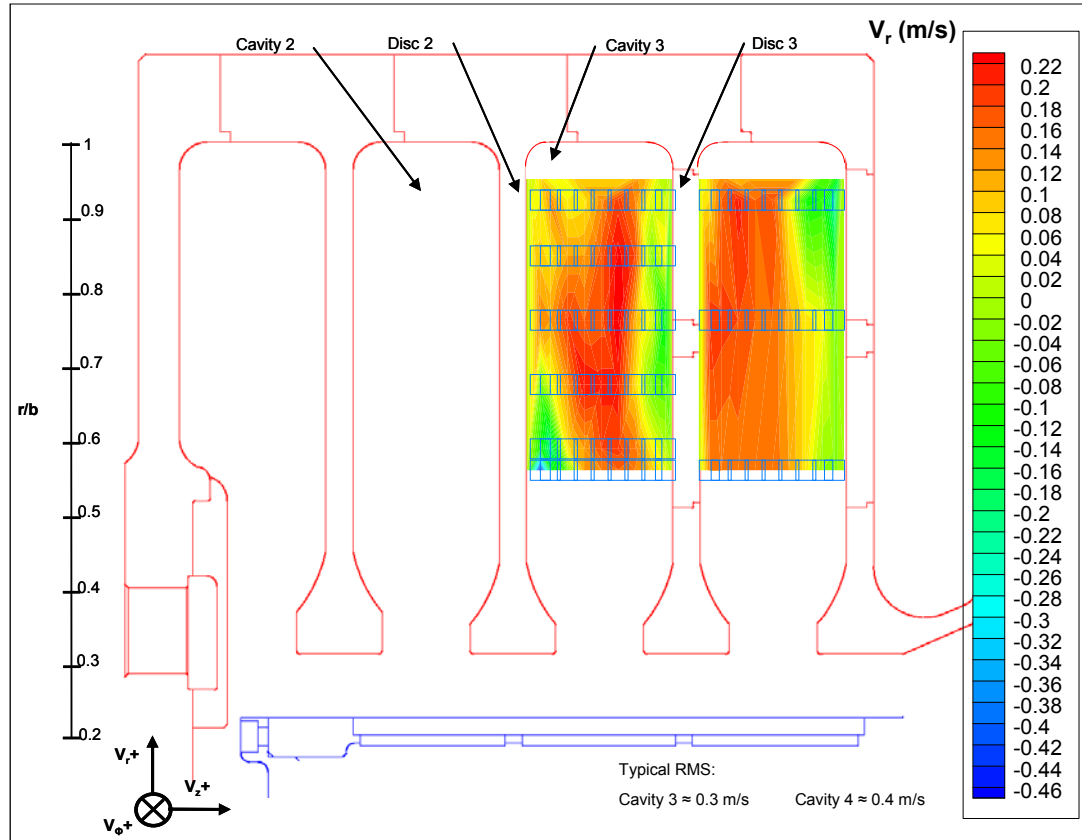


Fig 6. 19 Radial velocity contour plot for Build 3 run38-1, $Re_\phi = 0.4 \times 10^6$, $Re_z = 1.57 \times 10^5$ and $Ro = 7.51$

The measurements in Cavity 4 indicate a different radial flow pattern to Cavity 3, with a clockwise recirculation zone delimited by a radial outflow of $V_r \geq 0.15$ in the upstream axial portion of Cavity 4, between $0 \leq z/s \leq 0.6$, with the radial inflow region along the whole cavity measurement radii in the axial portion $0.6 \leq z/s \leq 1$, showing a magnitude of about $V_r = (-0.1)$ m/s.

The radial velocity contours shown in Figure 6.20 for run40-1, $Re_\phi = 0.39 \times 10^6$, $Re_z = 0.61 \times 10^5$ and $Ro = 2.98$, show some clear similarities in their patterns when compared to the previous condition.

The difference between the condition in Figure 6.19 and the condition in Figure 6.20 lies in the axial throughflow being lower in the latter case, with an axial Reynolds number value of $Re_z = 0.6 \times 10^5$ while the former case had a value nearly three times as much.

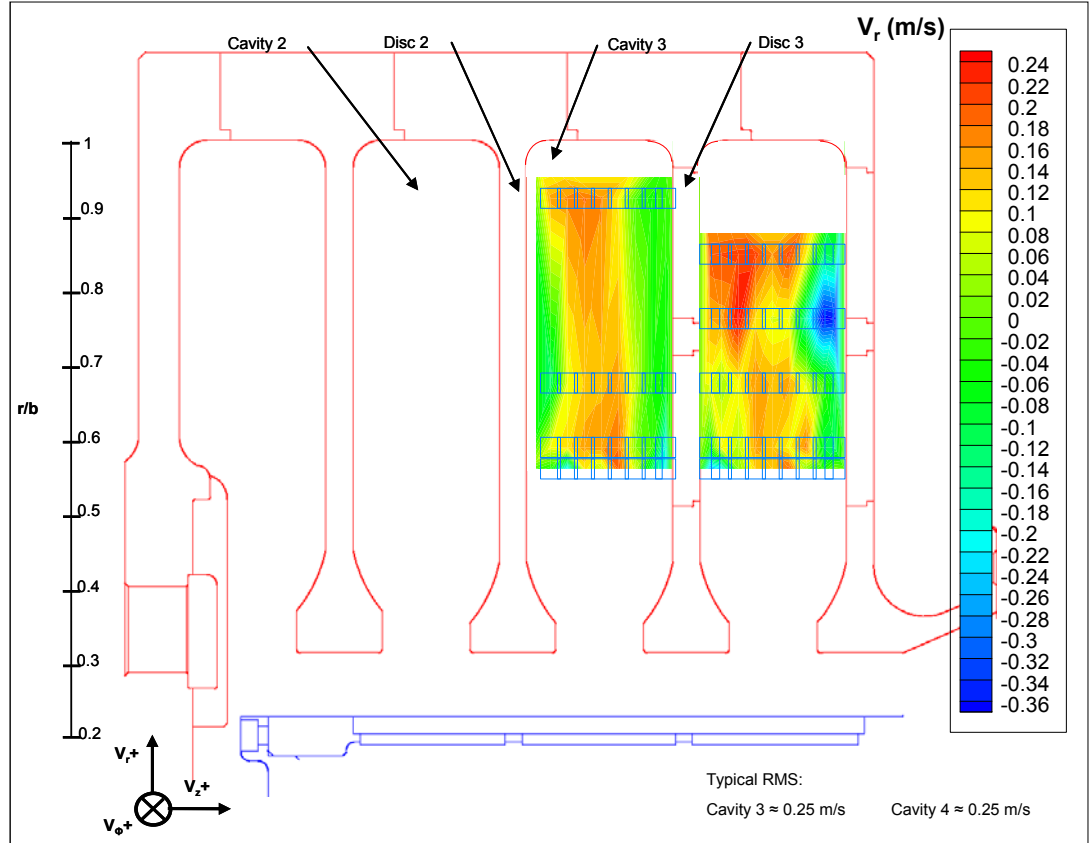


Fig 6. 20 Radial velocity contour plot for B3run40-1, $Re_\phi = 0.39 \times 10^6$, $Re_z = 0.61 \times 10^5$ and $Ro = 2.98$

The magnitude of the radial velocity component in both cavities in the condition featured in Figure 6.20 remains similar to the previous case, if somewhat slightly increased with a maximum of $V_r \approx 0.24$ m/s and a minimum of $V_r \approx (-0.36)$ m/s. Cavity 3 shows a radial outflow region located in mid cavity between $0.396 \leq z/s \leq 0.629$ of peak magnitude $V_r \approx 0.15$ m/s at $r/b = 0.68$. For $W = 13.31$ m/s, this corresponds to about 1.1 % of the throughflow velocity and about 1.1% of the solid-body tangential velocity at $r = b$ ($\Omega b = 14.03$ m/s). Radial inflow regions can be observed on the upstream and downstream side walls of Cavity 3, with magnitudes $\geq (-0.15)$ m/s by the inner radial part of the measurement region. The radial velocity contours in Cavity 4 present a similar recirculation flow pattern to run38-1 ($Re_\phi = 0.4 \times 10^6$, $Re_z = 1.57 \times 10^5$, $Ro = 7.51$). An outflow region is present in the upstream part of the cavity between $0 \leq z/s \leq 0.62$ with flow velocities reaching values above $V_r \approx 0.2$ m/s. The downstream part of the cavity features a radial inflow region of magnitude in excess of $V_r \approx (-0.2)$ m/s.

Comparing run38-1 (Fig.6.19) and run40-1 (Fig.6.20), it would appear that a reduction in axial Reynolds number affected the radial flows slightly in making the circulating flow regions somewhat weaker.

The radial velocity contour plots for run44-1, $Re_\phi = 1.99 \times 10^6$, $Re_z = 1.65 \times 10^5$ and $Ro = 1.58$ are shown in Figure 6.21.

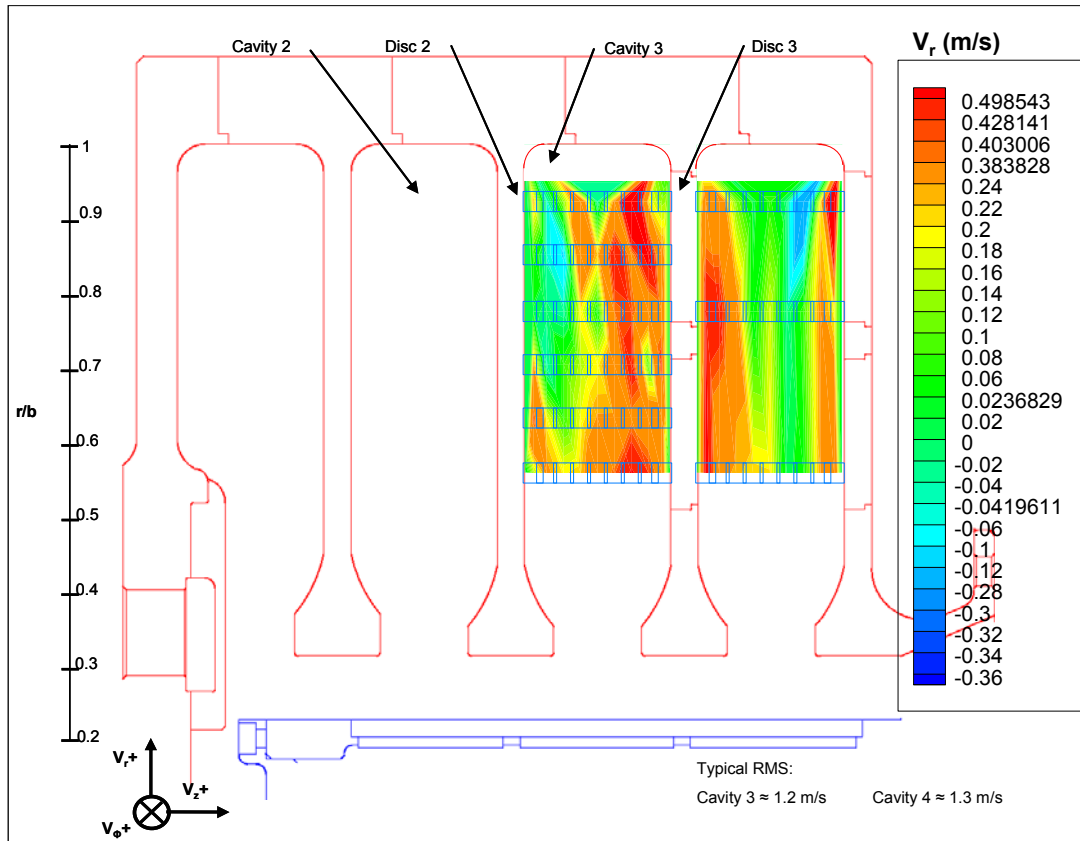


Fig 6. 21 Radial velocity contour plot for Build3, run44-1, $Re_\phi = 1.99 \times 10^6$, $Re_z = 1.65 \times 10^5$ and $Ro = 1.58$

Run38-1 (in Figure 6.19) and run44-1 (Fig. 6.21) were operated at similar values of $Re_z \approx 1.6 \times 10^5$, but with a five-fold increase in rotational Reynolds number ($Re_\phi = 0.4 \times 10^6$ to $Re_\phi = 1.99 \times 10^6$ for run38-1 and run44-1 respectively). This resulted in a near doubling in radial velocity magnitudes from peak values of $V_r \approx 0.22$ m/s (run38-1) to $V_r \approx 0.5$ m/s (run44-1). Radial velocity patterns in Cavities 3 and 4 appear radically different from the patterns encountered in Conditions 1 and 2. However, the time averaged radial velocity results presented in Figure 6.19 could be misleading at first sight. A second look would suggest the evidence of axi-symmetric vortex breakdown (possibly of type IIa from the classification of Farthing et al. (1992a)) entering into both cavities. The

signs of direct throughflow impingement along the downstream disc walls of Cavities 3 and 4 can be seen from a radial outflow region of velocity $V_r \approx 0.4$ m/s occupying most of the cavities' radial lengths.

These could effectively offset or superimpose onto the radial inflow regions previously described on the downstream sides of Cavities 3 and 4.

The maximum peak measured radial velocity magnitude at $r/b=0.78$ in Cavity 3 was 0.42 m/s. This corresponds to 1.3 % of the bulk average throughflow velocity ($W= 32.3$ m/s) and about 0.7% of the solid-body tangential velocity at $r=b$ ($\Omega b=63.94$ m/s). The radial flow pattern of Cavity 3 for this condition could therefore be described as a central region of radial outflow flanked by regions of radial inflows in the upstream and downstream sides of the cavity, with a downstream throughflow direct impingement resulting from vortex breakdown superimposing onto the cavity flow. Similarly in Cavity 4, the already described radial outflow in the upstream part of the cavity remains, with the vortex breakdown impingement flow superimposed on the downstream inflow region.

Figure 6.20 shows the radial velocity contours for Build 3, Run39-1, $Re_\phi= 1.93 \times 10^6$, $Re_z= 0.53 \times 10^5$ and $Ro= 0.53$, operated with a similar rotational Reynolds number than run44-1 ($Re_\phi= 1.93 \times 10^6$), but with a lower axial throughflow ($Re_z= 0.53 \times 10^5$) equivalent to run40-1. The associated Rossby number with this condition was $Ro= 0.53$. The magnitude of measured peak radial velocity in Cavity 3 at $r/b=0.765$ reached a value of 0.9 m/s, equivalent to about 7.4 % of the axial bulk velocity ($W= 12.1$ m/s), and about 1.2% of the solid-body tangential velocity at $r=b$ ($\Omega b=72.42$ m/s).

The radial velocity profile of Cavity 3 suggests a similar central outflow region as observed on the previous conditions, with some relatively strong inflow regions (of magnitude $V_r \approx (-1)$ m/s) along the lengths of the upstream and downstream discs. The radial measurements in Cavity 4 indicate a clear outflow region in the part of the cavity between approximately $0 \leq z/s \leq 0.5$ along the cavity radius, and an inflow region in the downstream part of the cavity.

Considering the inflow regions in Cavity 3 measured adjacent to the shroud (at $r/b=0.924$) on figures 6.19 to 6.22, it can be noticed that as the Rossby number of the experimental conditions decreases, the magnitude of the inflow increases. This behaviour is summarised in Table 6.6 below and would be expected from buoyancy driven flow in this part of the cavity.

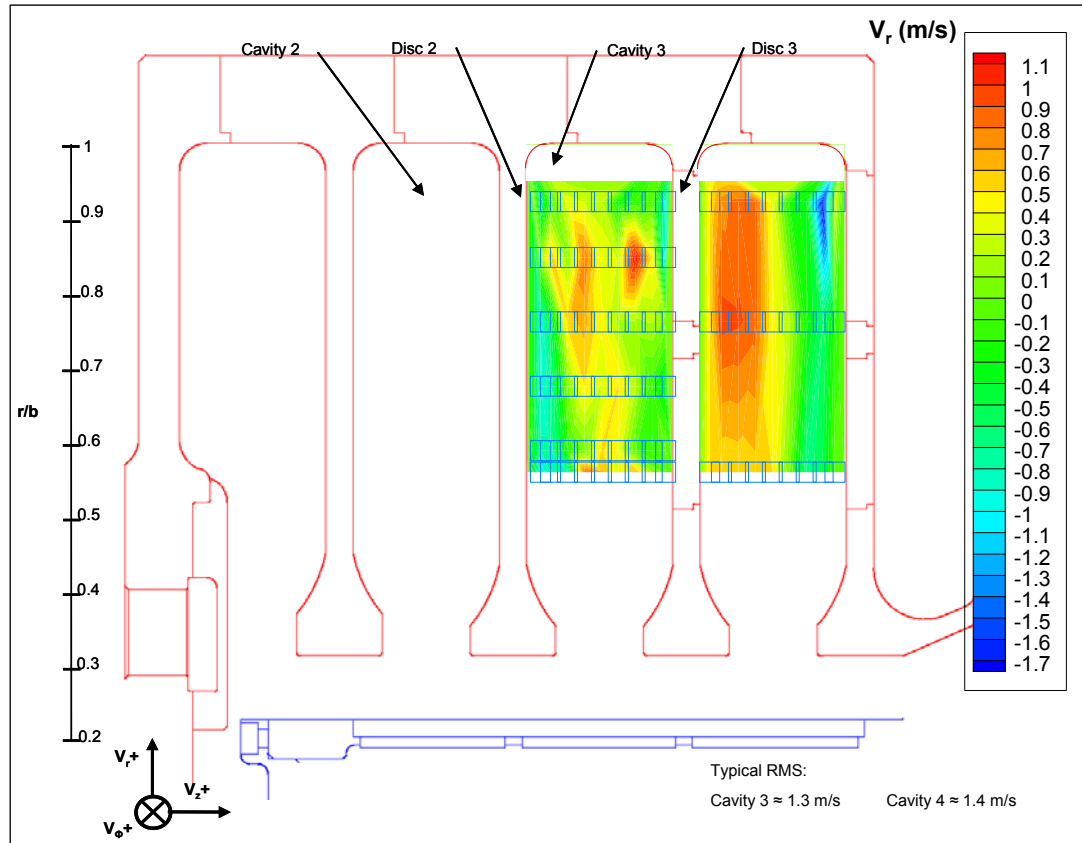


Fig 6. 22 Radial velocity contour plot for Build 3, run39-1, $Re_\phi = 1.93 \times 10^6$, $Re_z = 0.53 \times 10^5$ and $Ro = 0.53$

| Test (Build 3) | $Re_\phi/10^6$ | $Re_z/10^5$ | Ro | Cav3 peak radial inflow by shroud (m/s) |
|----------------|----------------|-------------|------|---|
| Run38-1 | 0.40 | 1.57 | 7.51 | -0.02 |
| Run40-1 | 0.39 | 0.61 | 2.98 | 0.05 |
| Run44-1 | 1.99 | 1.65 | 1.58 | 0.05 |
| Run39-1 | 1.93 | 0.53 | 0.53 | 0.89 |

Table 6. 6 Measured peak radial inflow in Build 3, Cavity 3, at $r/b=0.924$ for different experimental conditions (a negative value indicates an outflow velocity)

In the outer cavity region (at about $r/b=0.75$) , the magnitudes of the radial circulation flows appear to be related to the compound influence of the cavity rotation and the throughflow, with the velocity magnitudes generally increasing as Re_ϕ increases, but also being impeded by an increasing Re_z . The measured peak radial velocity values in Cavity 3 previously mentioned in the test descriptions are summarised in Table 6.7.

| Test (Build 3) | $Re_\phi / 10^6$ | $Re_z / 10^5$ | Ro | Mid-Cav3 peak V_r (m/s) | Meas. at (r/b) | Ω (rad/s) | W (m/s) | $V_r/\Omega b$ | V_r/W |
|-------------------|------------------|---------------|------|---------------------------------|----------------------|---------------------|--------------|-----------------------|-----------------------|
| Run38-1 | 0.40 | 1.57 | 7.51 | 0.24 | 0.765 | 68.6 | 36.1 | 1.59×10^{-2} | 6.65×10^{-3} |
| Run40-1 | 0.39 | 0.61 | 2.98 | 0.15 | 0.68 | 63.8 | 13.3 | 1.07×10^{-2} | 1.12×10^{-2} |
| Run44-1 | 1.99 | 1.65 | 1.58 | 0.42 | 0.78 | 290.7 | 32.3 | 6.57×10^{-3} | 1.30×10^{-2} |
| Run39-1 | 1.93 | 0.53 | 0.53 | 0.9 | 0.765 | 329.2 | 12.1 | 1.24×10^{-2} | 7.44×10^{-2} |

Table 6. 7 Cavity radial relative velocity magnitudes

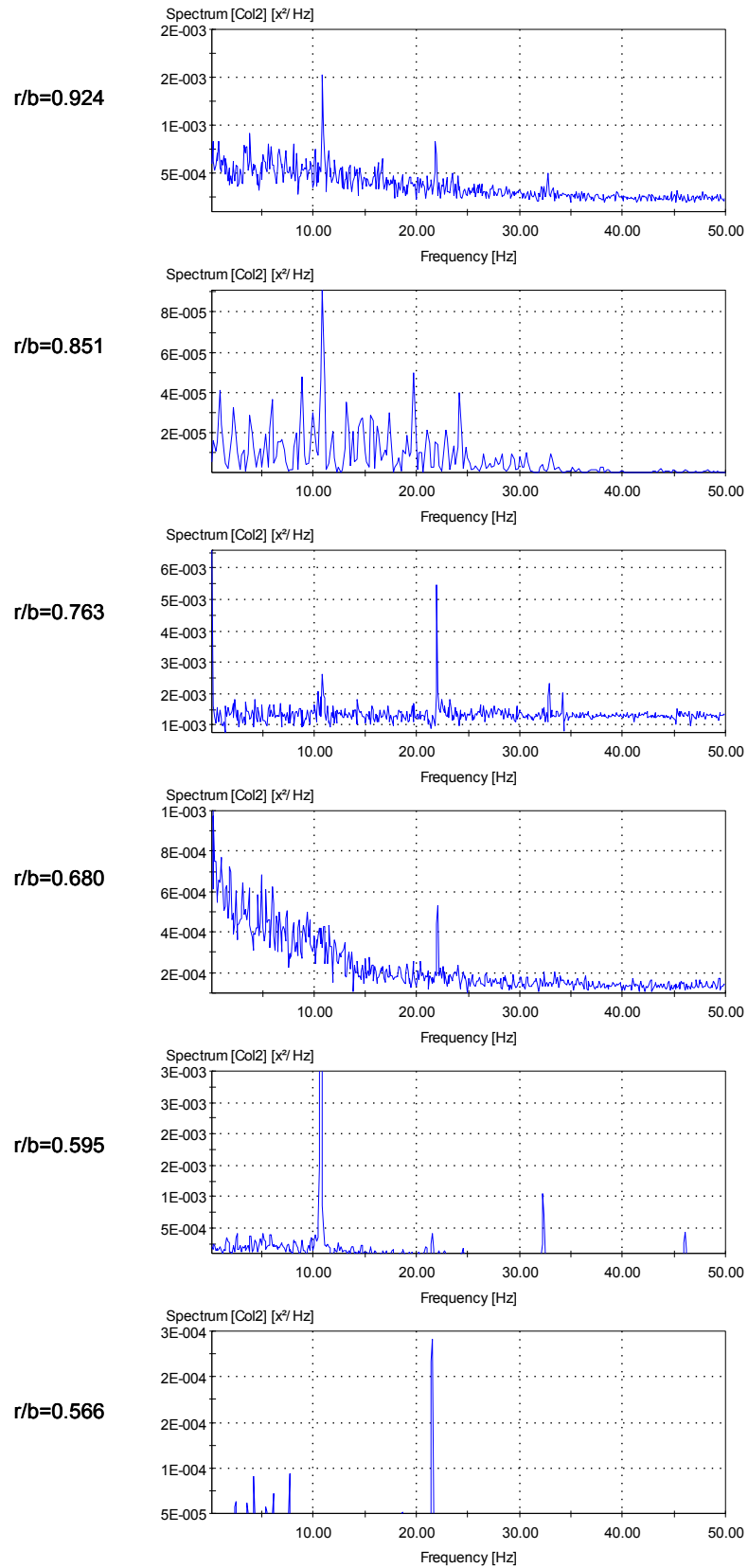
6.3 Spectral analysis of the tangential velocity component

A frequency spectrum analysis was carried out on the tangential velocity component measured data from the experimental configuration measuring simultaneous radial and tangential velocity components in Build 3 of the Multiple Cavity Rig. The experimental conditions were the same as previously described in Table 6.5. The frequency spectrum analysis operates in determining the natural frequencies in the succession of the collected velocity samples values over the measurement time. Following the frequency spectra analysis of Owen and Powell (2004), a Fast Fourier Transform algorithm was employed in the determination of the flow frequency characteristics as described in Chapter 3. The results were displayed in a power spectrum diagram representing the relative dominance of a frequency over the spectrum of analysis.

Figure 6.23 shows a series of power spectra diagrams for the measurement points located mid Cavity 3 ($z/s = 0.5$) at different radii.

The experimental condition of the results shown in Figure 6.23 was $Re_\phi = 0.4 \times 10^6$, $Re_z = 1.57 \times 10^5$ and $Ro = 7.51$. The rotational speed of the cavity was 655 rev/min; corresponding to a cavity rotational frequency of 10.92 Hz. Looking at the dominant spikes at the different radii plots, a broad interpretation on the recirculation pattern in view of the current understanding of heated cavity flow behaviour from Farthing et al. (1992a) would indicate the presence of a pair of contra rotating vortices (when a spike is about at the frequency of the cavity rotation assuming the tangential flow is near solid body rotation) between $0.765 \leq r/b \leq 0.924$, blending into two pairs of contra rotating vortices between $0.566 \leq r/b \leq 0.765$. Another interpretation could suggest a pair of vortices at higher radii (in the region $0.851 \leq r/b \leq 0.924$) and a pair at lower radii ($0.56 \leq r/b \leq 0.595$) intersecting each other in between (region $0.68 \leq r/b \leq 0.763$), resulting in the doubling of frequency seen. The presence of secondary spikes in the power spectrum diagrams at dominant frequencies of roughly multiples of the cavity rotational frequency could be explained from the simulations of Owen et al. (2006) and Sun et al. (2006), observing the ever changing nature over time of the cavity vortex structure in a heated cavity with axial throughflow. The laser

Doppler measurements resulting in the spectrum values shown in Figure 6.23 were, by nature of the measuring technique, taken over a certain length of time, sometimes in tens of seconds, therefore the information provided by the spectra determined for each location might be incomplete and moreover the relation between each location might also be non-representative of the flow as measured at different times from each other. The re-sampling method employed for the spectral analysis of velocimetry data taken across the rotor windows was also inducing a reduced time density of the data (as discussed in Chapter 4) which would undermine the accuracy of the analysis greatly as aliasing of the re-sampled data would be very likely to occur during the Fourier analysis. Bearing this in mind, the frequency spectra produced over the experimental conditions showed some broadly similar characteristics to as described above and can be seen in Appendix 6.5.



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Fig 6. 23 Tangential velocity power spectra on mid-plane of Cavity 3 for Build 3, run38-1, $Re_\phi = 0.4 \times 10^6$, $Re_z = 1.57 \times 10^5$ and $Ro = 7.51$

7. Heat transfer results

This chapter presents the heat transfer measurement results for Build 2 and Build 3 of the Multiple Cavity Rig. The main difference and measurement capability of Build 2 and Build 3 are recapped in Table 7.1 below.

All the measured heat transfer data cases were processed into detailed test reports for ease of archiving and analysis. The list of all experiments, operating conditions and main non-dimensional results can be found in Appendices 6.2 and 6.3. From this list, the Reader can select a test of interest and consult the detailed results report in the attached CD-Rom following the instructions in Appendix 6.1. For convenience of reading, the reports of the specifically mentioned tests in this chapter are also included in Appendix 7.1.

| | Build 2 | Build 3 |
|---------------------------|------------------|---------------------------------------|
| Annular gap | 0.092 | 0.164 |
| Heat transfer measurement | Disc, shroud | Disc, shroud |
| LDA measurement | V_ϕ , V_z | V_ϕ , V_z and V_ϕ , V_r |

Table 7. 1 Principal operational differences between Build 2 and Build 3 of the Multiple Cavity Rig

Table 7.2 summarises the range of experimental conditions performed for the velocity measurements on Build 2 and Build 3 of the Multiple Cavity Rig. By comparison, typical engine values for cruise operation may be taken as $Re_\phi = 7.7 \times 10^6$ and $Re_z = 2.4 \times 10^5$ (Alexiou 2000). The experimental data reported here has been obtained over a range of values overlapping both real engine conditions and also those used by authors of previous research in this field as shown in Fig. 2.17.

| Parameter | | Build 2 heat transfer | Build 3 heat transfer |
|------------------------------------|----------------------------------|---|---|
| Rotor speed | Ω (rad/s) | $57 \leq \Omega \leq 615$ | $65 \leq \Omega \leq 608$ |
| Cooling air flow rate | \dot{m} (kg/s) | $0.164 \leq \dot{m} \leq 0.8$ | $0.16 \leq \dot{m} \leq 0.75$ |
| Inlet pressure | p_{in} (bar, abs) | $2.2 \leq p_{in} \leq 3.16$ | $2.3 \leq p_{in} \leq 3.7$ |
| Inlet temperature | T_{in} (K) | $309 \leq T_{in} \leq 324$ | $302 \leq T_{in} \leq 339$ |
| Disc average surface temperature | $T_{s,av}$ (K) | $328 \leq T_{s,av} \leq 351$ | $321 \leq T_{s,av} \leq 353$ |
| Shroud average surface temperature | T_{sh} (K) | $359 \leq T_{sh} \leq 407$ | $353 \leq T_{sh} \leq 386$ |
| Disc rotational Reynolds number | Re_{ϕ} (Eq. 4.1) | $4 \times 10^5 \leq Re_{\phi} \leq 5 \times 10^6$ | $4.7 \times 10^5 \leq Re_{\phi} \leq 5 \times 10^6$ |
| Axial Reynolds number | Re_z (Eq. 4.2) | $4.2 \times 10^4 \leq Re_z \leq 1.9 \times 10^5$ | $4.3 \times 10^4 \leq Re_z \leq 2 \times 10^5$ |
| Rossby number | Ro (Eq. 4.3) | $0.29 \leq Ro \leq 8.6$ | $0.53 \leq Ro \leq 7.51$ |
| Disc buoyancy parameter | $\beta \Delta T_{av}$ | $0.04 \leq \beta \Delta T_{av} \leq 0.13$ | $0.017 \leq \beta \Delta T_{av} \leq 0.09$ |
| Shroud buoyancy parameter | $\beta \Delta T_{sh}$ (Eq. 4.20) | $0.14 \leq \beta \Delta T_{sh} \leq 0.302$ | $0.086 \leq \beta \Delta T_{sh} \leq 0.238$ |
| Shroud Grashof number | Gr_{sh} (Eq. 4.20) | $3.8 \times 10^7 \leq Gr_{sh} \leq 4.5 \times 10^9$ | $4 \times 10^7 \leq Gr_{sh} \leq 3.1 \times 10^9$ |

Table 7. 2 Range of heat transfer experimental conditions for Build 2 and Build 3 of the Multiple Cavity Rig

7.1 Disc heat transfer

Figures 7.1 and 7.2 show examples of discs temperature measurements in Build 3 of the Multiple Cavity Rig plotted against the non-dimensional radius (r/b) for operating conditions of $Re_\phi = 4.12 \times 10^6$, $Re_z = 1.16 \times 10^5$, $Ro = 0.54$, $\beta \Delta T_{av} = 0.03$ and $Re_\phi = 0.49 \times 10^6$, $Re_z = 0.92 \times 10^5$, $Ro = 3.56$, $\beta \Delta T_{av} = 0.09$ respectively. The thermocouples on the discs and shroud are shown as symbols on the plots, with their respective measurement uncertainty (calculated from the method in 5.1.2) are displayed as “error bars”. A profile of the disc shape is included above the horizontal axis for reference. The fitted temperature profiles used for the disc heat transfer calculations (section 4.2) are displayed as dotted lines. From Figs 7.1 and 7.2 it can be seen that the stability of thermocouples measurement is good and that there is a relatively small temperature difference across the axial thickness of the discs.

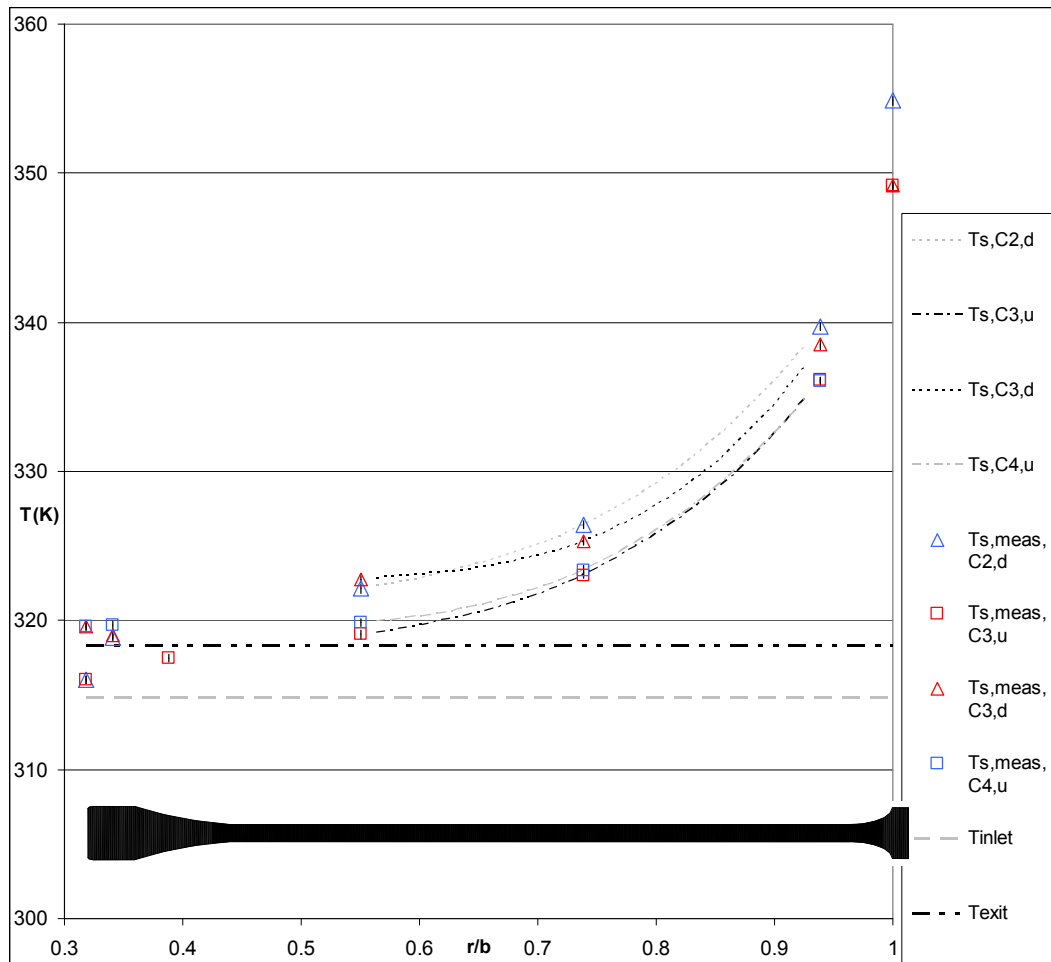


Fig 7. 1 Disc temperature measurements in Build 3 for run28-1, $Re_\phi = 4.12 \times 10^6$, $Re_z = 1.16 \times 10^5$, $Ro = 0.54$ and $\beta \Delta T_{av} = 0.03$

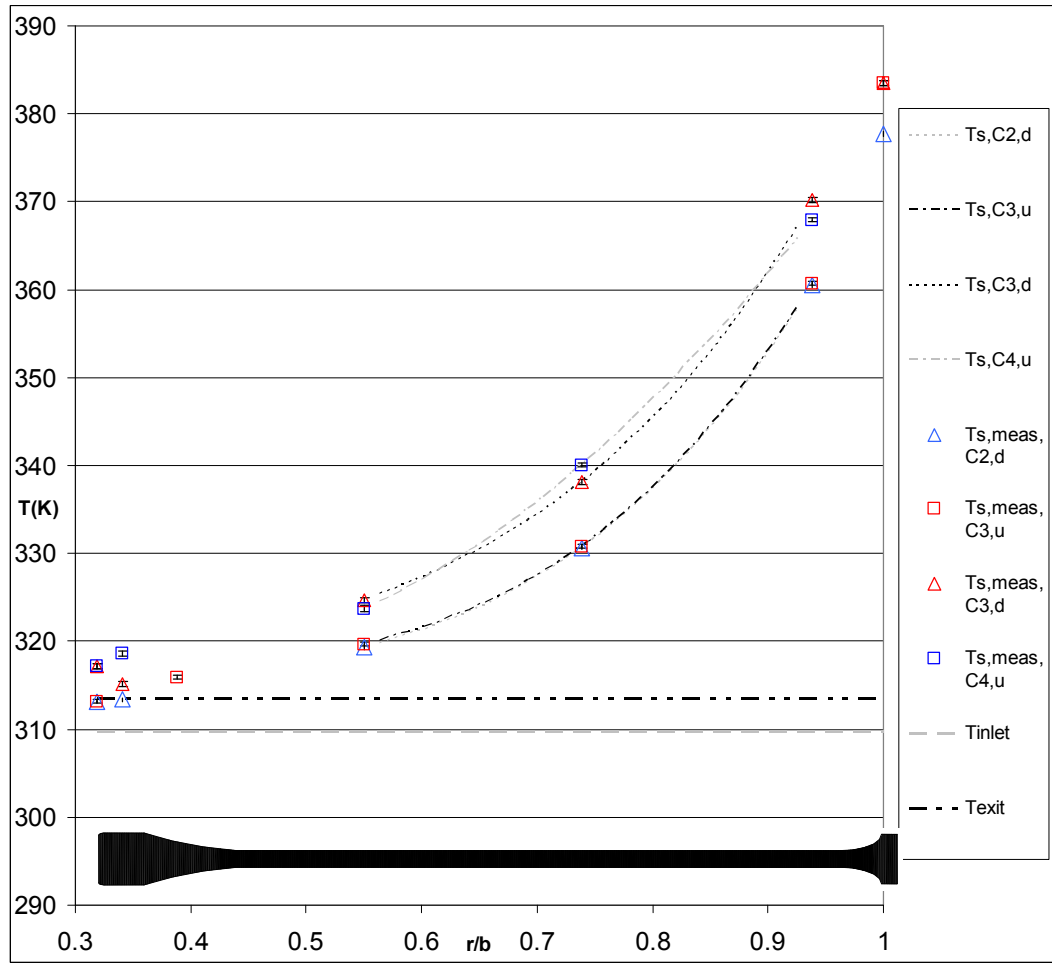


Fig 7. 2 Disc temperature measurements in Build 3 for run32-1, $Re_\phi = 0.49 \times 10^6$, $Re_z = 0.92 \times 10^5$, $Ro = 3.56$ and $\beta \Delta T_{av} = 0.09$

As explained in Chapter 5, the heat transfer results from the fin solution appear in better relation with a heat transfer behaviour involving heat conducted out of the discs (although no differentiation from the discs surfaces can be made due to the adiabatic mid-plane assumption) and with magnitude levels more in relation with relatively low cavity air velocities as illustrated in Figure 7.3 for Build 2 and Build 3 when compared to correlations for forced convection flow over a flat plate (from Long, 1999). A comparison with correlations for a free disc rotating in quiescent air from Dorfman (1963) is also included in Fig 7.4.

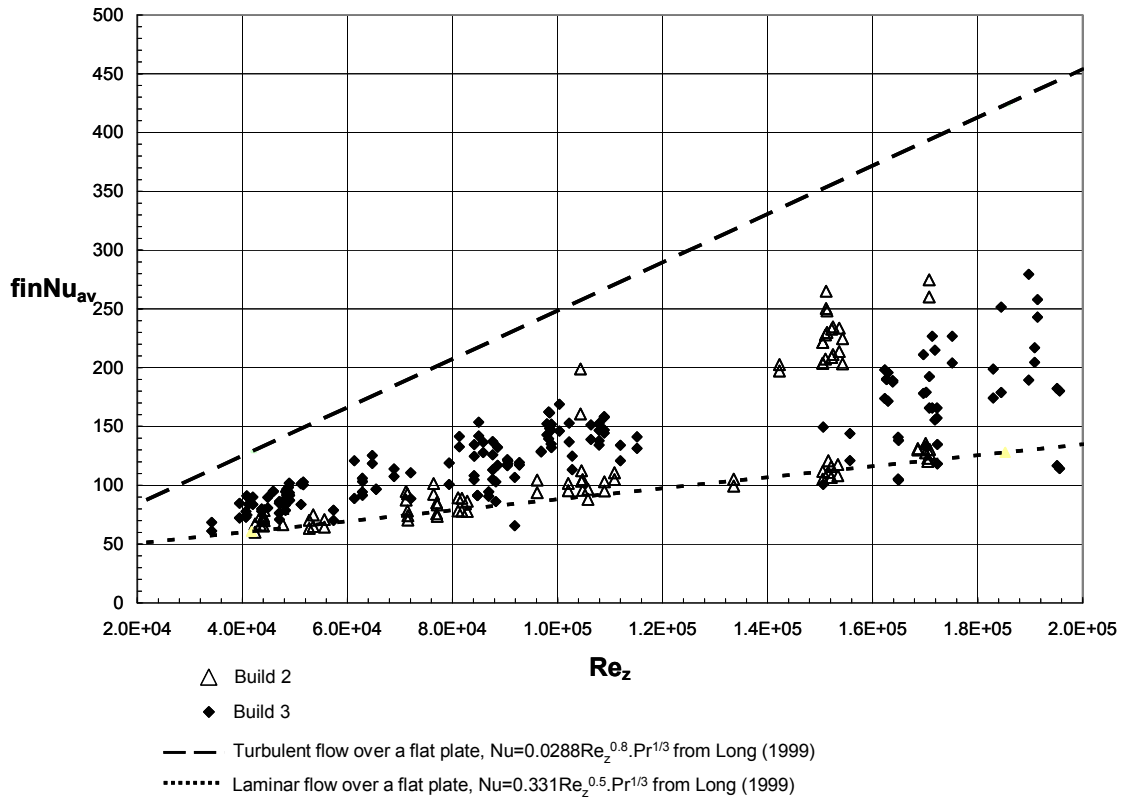


Fig 7. 3 Average disc Nusselt number from the fin solution for Build 2 and Build 3 (all data) compared with correlations of forced convection over a flat plate

An overall check of the estimated values of cavity heat transfer from the fin solution was performed by comparing the measured temperature rise of the cooling air across the rig against a calculated value of temperature rise resulting from the contribution of discs and shrouds heat transfer. The rig's energy balance was expressed on the principle that the total convective heat transfer from the disc and shroud surfaces is equal to the change of enthalpy of the cooling air, neglecting viscous dissipation and heat transfer to/from the shaft:

$$\dot{m} \cdot Cp (T_{exit} - T_{inlet}) = n_{sh} \cdot Q_{sh} + n_{df} \cdot Q_{df} \quad \text{Eq.7.1}$$

hence

$$\Delta T = (T_{exit} - T_{inlet}) = \frac{\pi \cdot k}{\dot{m} \cdot Cp} \left[\left\{ 4 \cdot b \cdot n_{sh} (T_{sh} - T_{in}) \cdot Nu_{sh} \right\} + \left\{ \frac{(b^2 - a^2)}{b} n_{df} (T_{s,av} - T_{in}) \cdot Nu_{av} \right\} \right] \quad \text{Eq.7.2}$$

Where n_{sh} is the number of shroud surfaces in the rig (4), n_{df} is the number of disc faces (8), Nu_{sh} is the mean of the calculated shroud Nusselt numbers (Eq.

4.18) in Cavities 2 and 3, Nu_{av} is the mean of the discs average Nusselt numbers (defined in Eq. 4.13) and $T_{s,av}$ is the mean value of the average disc surface temperatures for Disc 2 and 3.

The temperature differences between the measured and calculated values are plotted in Figure 7.5 for different values of Ro .

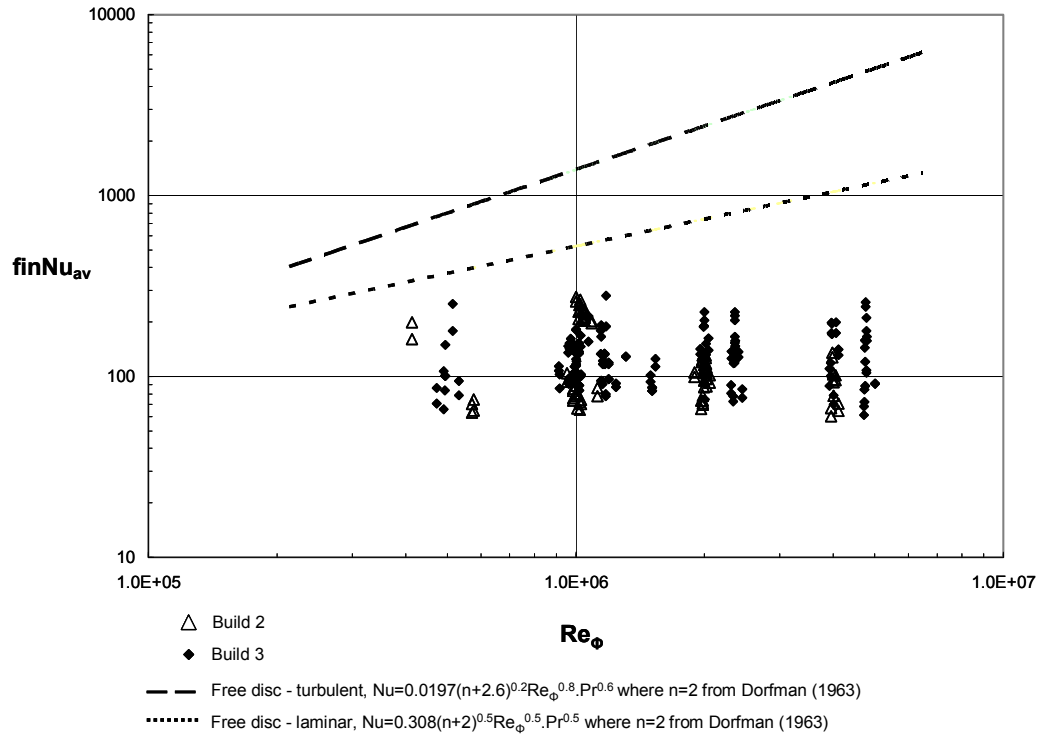


Fig 7. 4 Average disc Nusselt number from the fin solution for Build 2 and Build 3 (all data) compared with correlations for a free disc

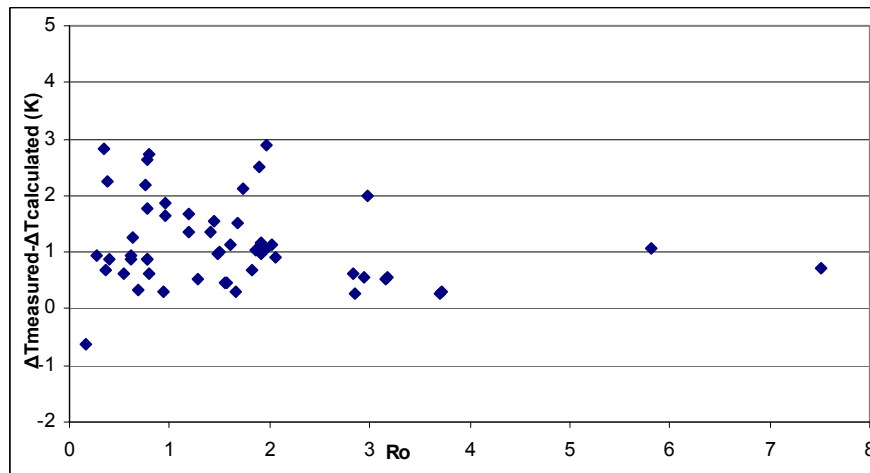


Fig 7. 5 Differences of measured rig temperature rise and calculated temperature rise with Rossby number

Figure 7.5 indicates that the calculated values of temperature difference across the rig are generally less than measured (as not taking into account all of the rig's surfaces participating into the energy balance) and are acceptable with the measured values, giving some level of confidence on the averaged Nusselt numbers of the fin solution to give an acceptably accurate indication of the overall heat transfer behaviour. The heat transfer results to follow will therefore be expressed from the results of the fin solution method and as average values as Discussed in Chapter 5.

The variation of average Nusselt number with axial Reynolds number for Build 2 (narrow annular gap) and for various values of rotational Reynolds number ($Re_\phi = 0.5 \times 10^6, 10^6, 2 \times 10^6$ and 4×10^6), is shown in Figure 7.6. Average values of Nusselt number are shown for both Disc 2 and Disc 3 and there appears to be little significant difference between the two discs. There also appears to be a clear trend – increasing the axial Reynolds number increases the average Nusselt number.

For the larger values of axial Reynolds number ($Re_z > 10^5$) and the lower values of rotational Reynolds number ($Re_\phi < 10^6$), i.e. the largest values of Rossby number, this increase is more noticeable. There is some evidence, at $Re_z < 10^5$, to suggest that increasing the rotational Reynolds number increases the average Nusselt number and the results for $Re_z > 10^5$ suggest the opposite is true as increasing Re_ϕ actually decreases the Nusselt number. This is consistent with the findings of Bohn et al. (1994), who also noted a similar attenuating effect due to the Coriolis acceleration.

A corresponding plot, but for Build3 (wide annular gap) is shown in Figure 7.7. The overall impression is that for the same value of Re_z and Re_ϕ , the average Nusselt number from the disc surface is increased by increasing the size of the annular gap between the shaft and disc bore.

The behaviour shown in Figure 7.6 for Build 2, where at the larger values of Re_z increasing Re_ϕ actually decreases the Nusselt number does not seem to occur with the wide annular gap. This and the higher overall level of heat transfer are caused by the difference in geometry. Equal values of Re_z imply that the average velocity of the throughflow is lower in the wider annular gap.

Consequently there will be a greater tendency for the flow to penetrate the cavity and interact with the disc surfaces. If this is the case, then a greater sensitivity to the rotational Reynolds number may be expected to be shown in Figure 7.7 (Build 3) compared to Figure 7.6 (Build 2). It is possible that this is reflected in the data but the results are far from conclusive. It is worth reminding that the LDA measurements in Chapter 6 made in this test rig show a significant difference in the distribution of tangential velocities between Build 2 and Build 3.

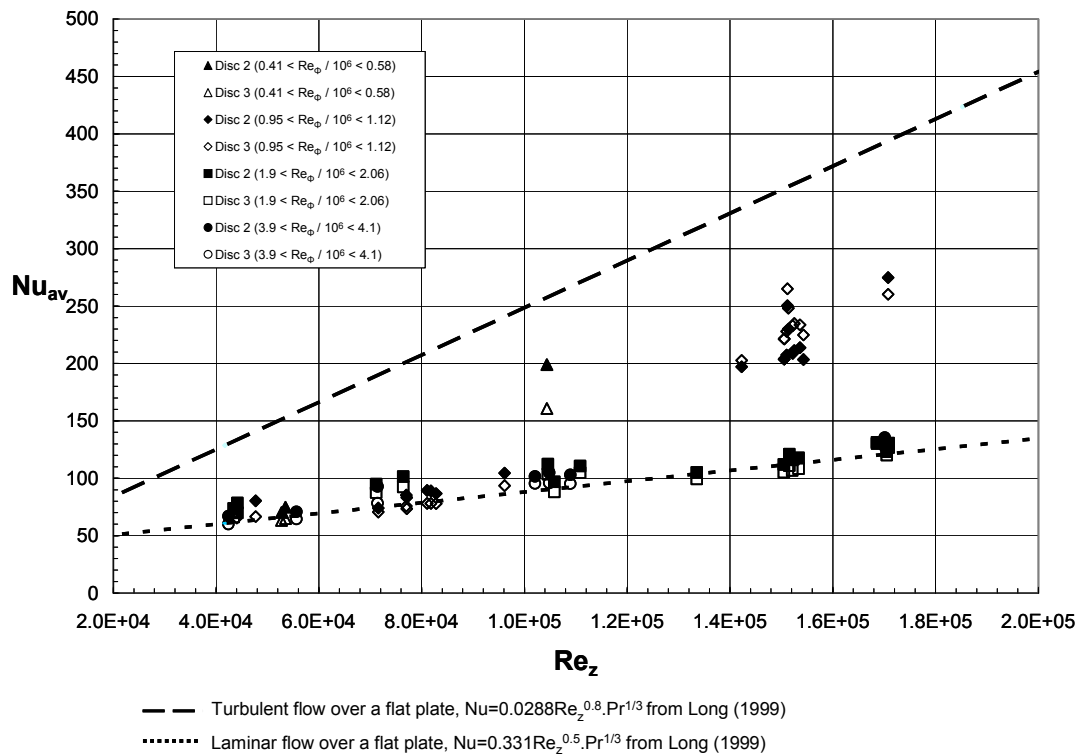


Fig 7. 6 Build 2, variation of Nu_{av} with Re_z for various values of Re_ϕ

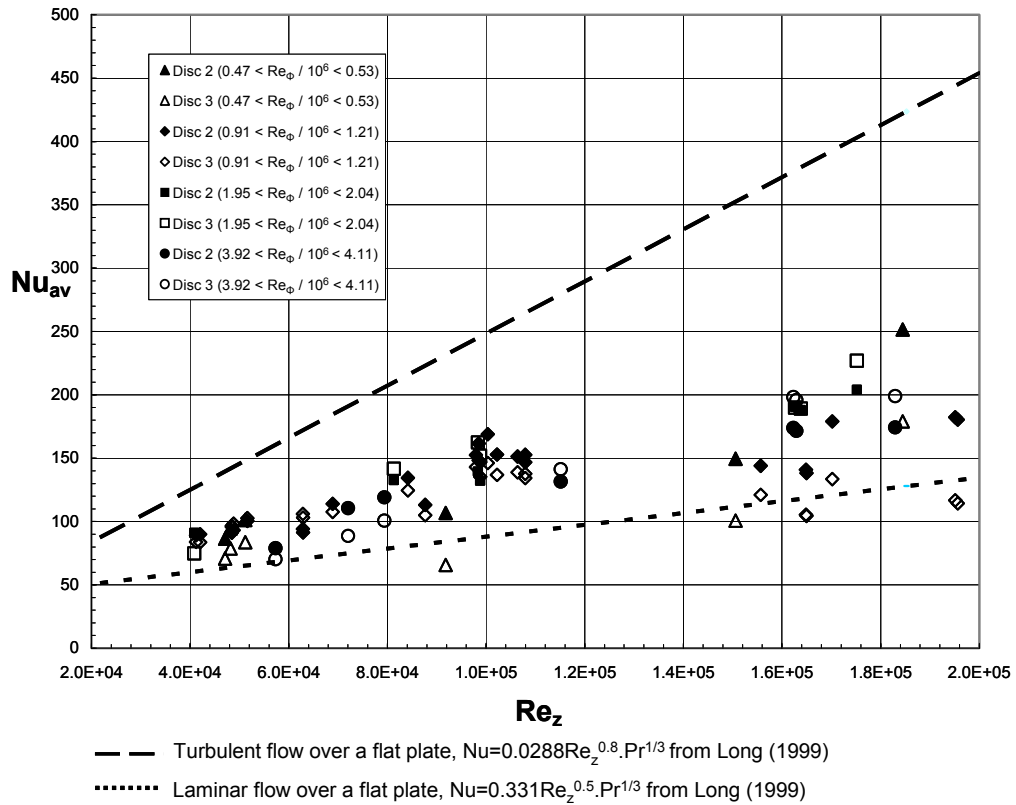


Fig 7. 7 Build 3, variation of Nu_{av} with Re_z for various values of Re_ϕ

Figure 7.8 shows a plot of average Nusselt numbers against Rossby number for Build 3, Disc 2 and Disc 3. The heat transfer in Disc 3 tends to be greater than Disc 2 for operating conditions of $Ro \leq 2$, while the trend seems reversed for values of $Ro \geq 2$. This could be attributed to differences in flow structures affecting the discs faces that vary with operating conditions.

From the preceding discussions, it is apparent that a simple plot of average Nusselt number against axial Reynolds number does not adequately represent trends in the heat transfer data. A number of attempts have been made at correlating the average Nusselt numbers and the most promising of these is shown in Figure 7.10 for Build 2 and Figure 7.9 for Build 3. In both cases the parameter $Nu_{av} / Re_\phi^{1/2}$ is plotted against the Rossby number. It is worth noting here that a previously used parameter, the buoyancy number $(Ro / (\beta \Delta T_{av}))^{0.5}$ – see Alexiou 2000), had also been used in the past. This differs from the Rossby number through the inclusion of the dimensionless temperature $\beta \Delta T_{av}$ in the

buoyancy number. The Rossby number has been chosen for use here as this was found to give a small reduction in the scatter of the data. The parameter $\beta\Delta T_{av}$ was varied in the presented experiments ($0.04 < \beta\Delta T_{av} < 0.13$ in Build 2 and $0.017 < \beta\Delta T_{av} < 0.09$ in Build 3) but not in any systematic sense. Further work is therefore required to establish the relevance of this parameter in characterising the disc heat transfer results.

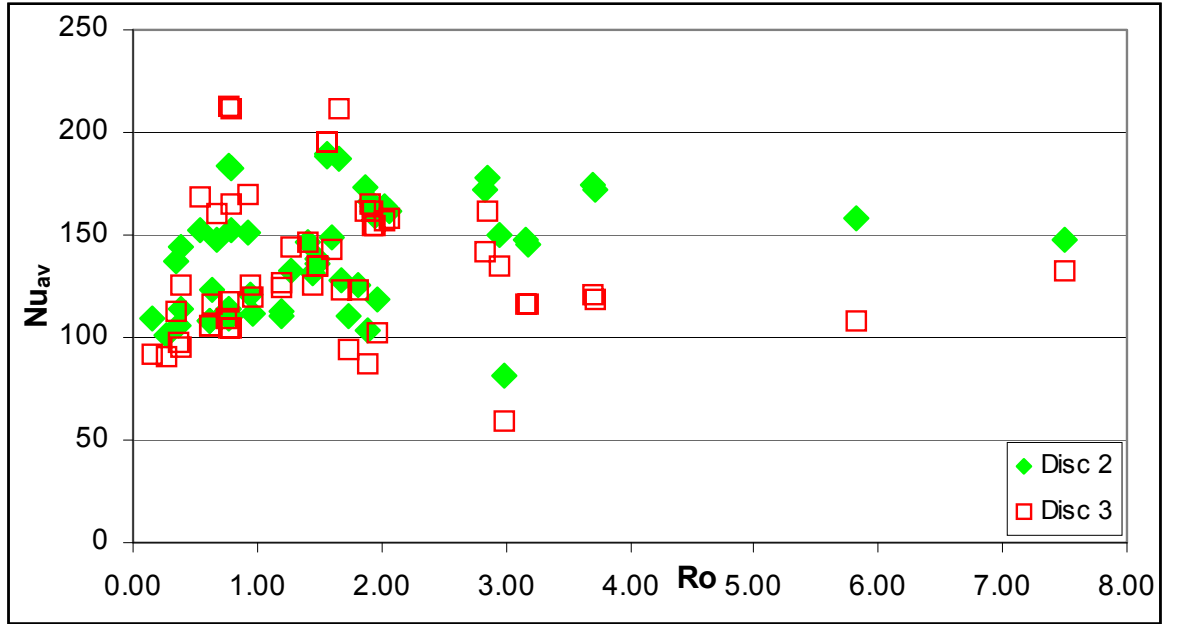


Fig 7. 8 Variations of Nu_{av} in Build 3 for Disc 2 and 3 with Ro

Correlation attempts on the average Nusselt numbers of Build 3 resulted in a good data collapse for $Nu_{av}/Re_{\phi}^{0.5}$ against Ro as shown in Figure 7.9.

The average Nusselt number values for both discs showed to follow the relationship below for values of $Ro \leq 2$ but appeared more scattered at higher values of Ro .

$$\frac{Nu_{av}}{Re_{\phi}^{0.5}} \approx 0.1 \times Ro^{0.5} \quad \text{Eq. 7.3}$$

From Eq. 7.3, since $Nu_{av} / Re_{\phi}^{0.5} \propto Ro^{0.5}$, then $Nu_{av} \propto Re_z^{0.5}$ suggesting laminar forced convection.

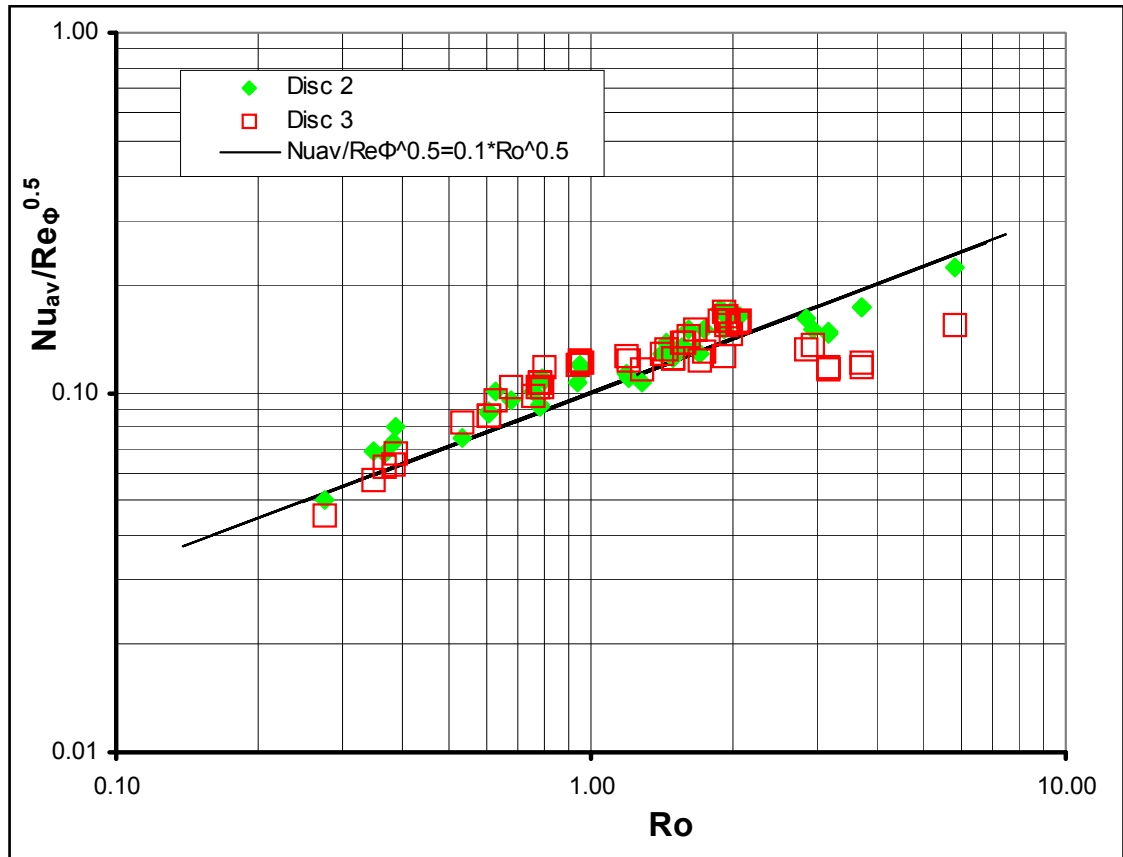


Fig 7. 9 Variations of $Nu_{av}/Re_{\phi}^{0.5}$ against Ro for Disc 2 and 3 in Build 3 with Ro

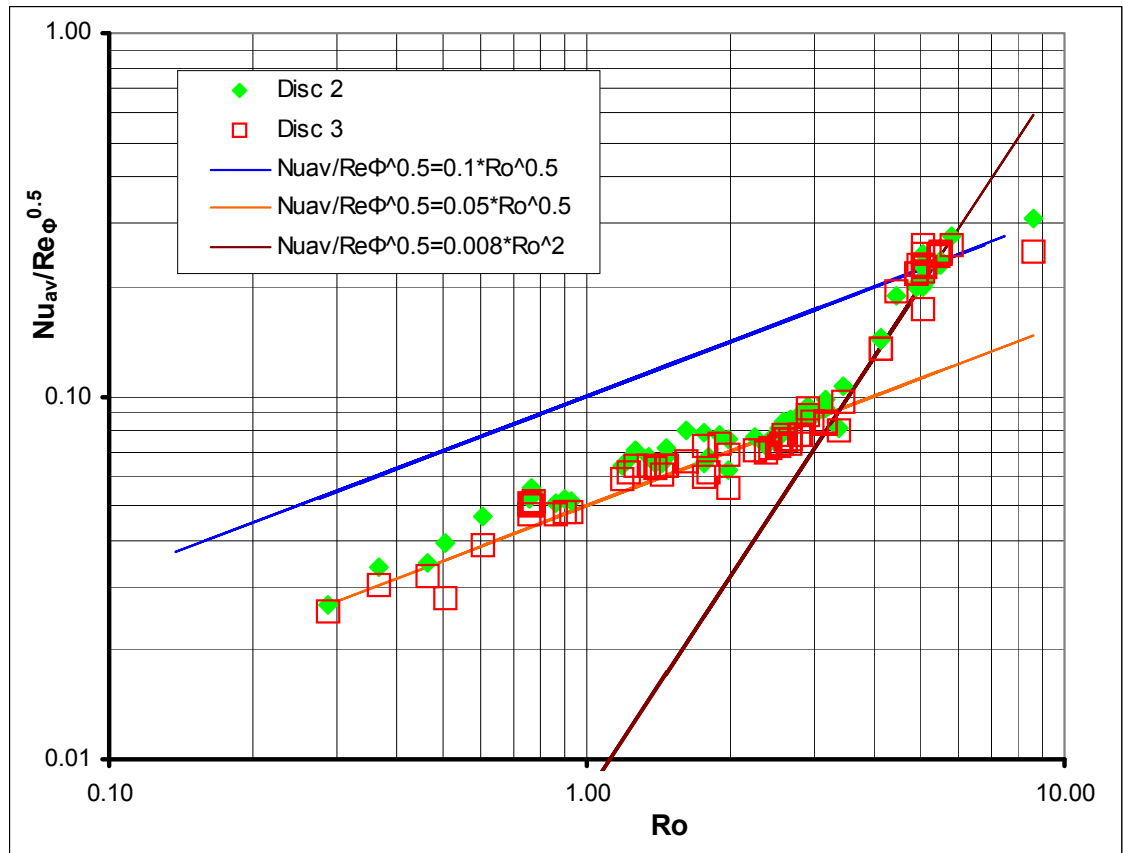


Fig 7. 10 Variations of $Nu_{av}/Re_{\phi}^{0.5}$ against Ro for Disc 2 and 3 in Build 2 with Ro

On the average Nusselt number plot for Build 2 (Fig 7.10), there appears to be two different 'regimes': 1) $0.3 < Ro < 3$, and 2) $Ro > 3$. In the first of these, since $Nu_{av} / Re_{\phi}^{0.5} \propto Ro^{0.5}$, then $Nu_{av} \propto Re_z^{0.5}$ suggesting laminar forced convection as for Build 3, following a relationship close to:

$$\frac{Nu_{av}}{Re_{\phi}^{0.5}} \approx 0.05 \times Ro^{0.5} \quad \text{Eq. 7.4}$$

In the second 'regime', for $Ro > 3$, an increase in Nu_{av} can be observed, following a relationship close to:

$$\frac{Nu_{av}}{Re_{\phi}^{0.5}} \approx 0.08 \times Ro^2 \quad \text{Eq. 7.5}$$

For Build 2, since $Nu_{av} / Re_{\phi}^{0.5} \propto Ro^2$, then $Nu_{av} \propto Re_z \cdot Re_{\phi}^{0.5}$. This is consistent with a flow which is mainly affected by the axial throughflow rate at large values of Rossby number and this sensitivity diminishes as the Rossby number (and inertial effects) is reduced. The differing influence of the rotational Reynolds number previously noted on Figure 7.7 is also evident in these two relationships.

Differences in magnitudes of Nu_{av} for Build 2 and 3 could be explained from the observations of Alexiou et al. (2000) in Build 1 of the Multiple Cavity Rig, which central shaft could be co-rotated or contra-rotated relative to the rotor direction. Co-rotating operation resulted in greater heat transfer values through the rig than contra-rotating conditions. Considering the tangential velocity profile of the throughflow under the discs of Build 2 and 3, Build 3, with its wider annular gap would present a tangential velocity shear profile lower than the narrow annular gap at same flow and rotational conditions. Therefore, making a parallel with the co-rotating condition increase of Nu_{av} on Build 1, it could be justified that Build 3's average Nusselt number values should be greater than Build 2 at similar conditions due to the lower tangential velocity shear profile of the throughflow by the shaft affecting the influence of the axial throughflow on the flow inside the cavity.

It is appropriate here to comment on similarities and differences between the heat transfer results from various test rigs with axial throughflow. The earlier

work on single cavities ($G = 0.13$, $a/b = 0.1$ and an unheated shroud) by Farthing et al. (1992b), showed the heat transfer from the discs to be consistent with free convection. Long (1994) found that the disc heat transfer ($G = 0.13$ and 0.36 , $a/b = 0.1$, heated and unheated shroud) was influenced by both forced and free convection effects. The results presented here seem to contradict these earlier findings. There is little evidence to suggest that the disc heat transfer is affected by buoyancy driven flow and is, in fact, mostly influenced by forced convection effects. This can be reconciled by the following. It is clear that the shroud has an influence on the flow in the cavity. For an unheated shroud and with a narrow cavity and small inlet radius ratio (as in the work of Farthing et al. 1992b), heating the discs creates buoyancy forces which destabilise the throughflow, and the narrow gap suppresses forced convection effects. For a wider gap ratio (as in the work of Long 1994), forced convection does affect the flow in the cavity. In both these cases, the maximum disc surface temperature was comparable to the shroud temperature, with values of the buoyancy parameter based on the average disc surface temperature of $\beta\Delta T_{av} \sim 0.25$. So, even in the wider gap ratio cavity, buoyancy driven effects occur towards the outer radius of the discs. The disc heating pattern on the Multiple Cavity Rig is quite different to that on the earlier single cavities. The maximum disc temperature is around 25K less than the shroud temperature and typical values of the buoyancy parameter are in the range: $0.02 < \beta\Delta T_{av} < 0.1$. The gap ratio at $G \sim 0.2$ is also significantly larger than the earlier single cavity work that so clearly indicated free convection on the discs. Taking into consideration all these factors, and the presence of a shaft at a relatively large radius ratio, it is not surprising that there is little evidence of buoyancy driven heat transfer behaviour on the disc surfaces of the Multiple Cavity Rig. It is however suggested that it may well occur. With improved spatial resolution of the surface temperature measurements, and more confidence in the heat transfer data, it may be possible to notice this behaviour towards the outer radius of the cavities in the future.

7.2 Shroud heat transfer

A parallel investigation on the shroud heat transfer in the series of Multiple Cavity Rigs was performed by Long and Childs (2007) from experimental data derived from the current work. A one dimensional radial conduction method was used to determine the heat transfer characteristics of the shroud, which is believed to provide better accuracy of results than from the disc heat transfer methods because of the lack of bias introduced by surface temperature fitting. Therefore in order to help understand the heat transfer in rotating cavities with axial throughflow, Long and Childs' observations ought to be described below. It was seen that the heat transfer from the shroud was governed by rotationally induced free convection and depended only on the shroud Grashof number. Figure 7.11 illustrates this for the results of Build 2, showing independence of the results on axial Reynolds number.

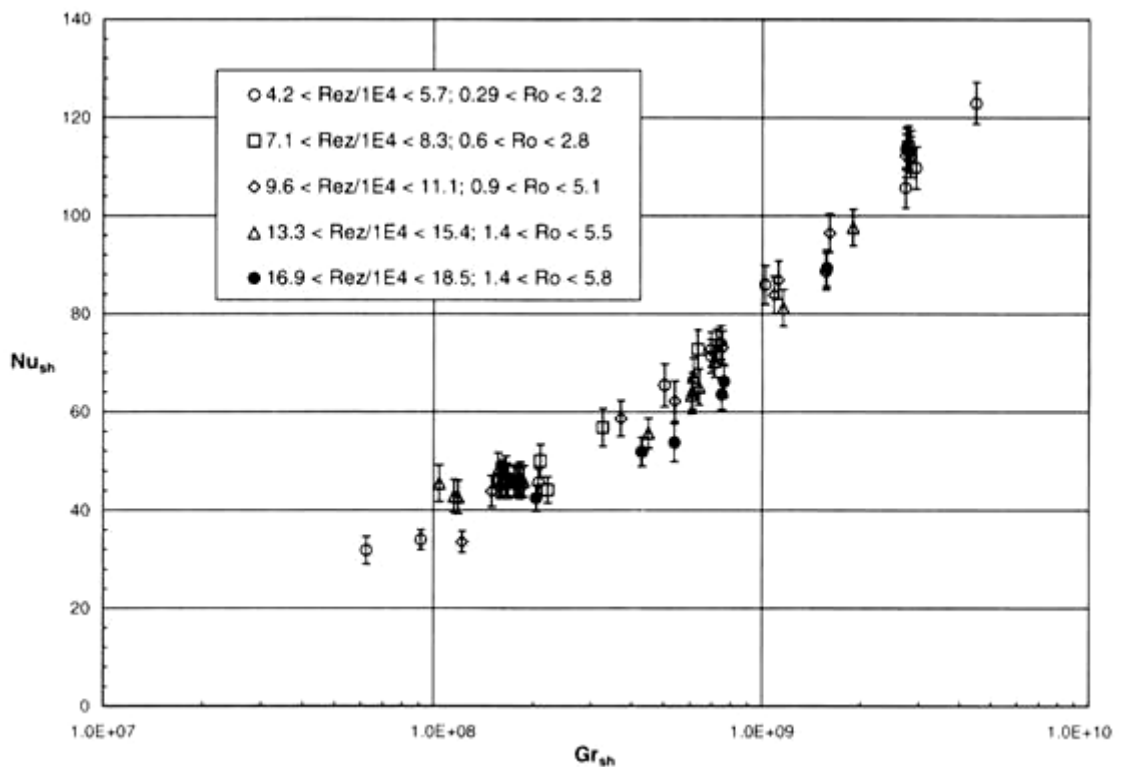


Fig 7. 11 Variation of shroud Nusselt number with Grashof number in Cavity 2 for Build 2 (narrow annular gap) and various values of Re_z (Long and Childs, 2007).

There were also indications that the shroud heat transfer results could be correlated on the characteristics similar to Rayleigh-Bénard convection in a stationary enclosure. Higher shroud heat transfer values were found in Build 3 of the Multiple Cavity Rig compared to Build 2, as was noticed from the discs heat transfer results, also suggesting a greater influence of the throughflow on the cavity heat transfer when the annular gap is increased. Figure 7.12 shows the shroud Nusselt number results for Build 2 (narrow annular gap). The most acceptable fit of the data for Nu_{sh} of Build 2 was determined as:

$$Nu_{sh} = 0.216(Gr_{sh} \cdot Pr)^{1/4} + 0.0494(Gr_{sh} \cdot Pr)^{1/3} \quad \text{Eq. 7.5}$$

Figure 7.13 shows the shroud Nusselt number results for Build 3 (wide annular gap). The data collapsed following:

$$Nu_{sh} = 0.25(Gr_{sh} \cdot Pr)^{1/4} + 0.057(Gr_{sh} \cdot Pr)^{1/3} \quad \text{Eq. 7.6}$$

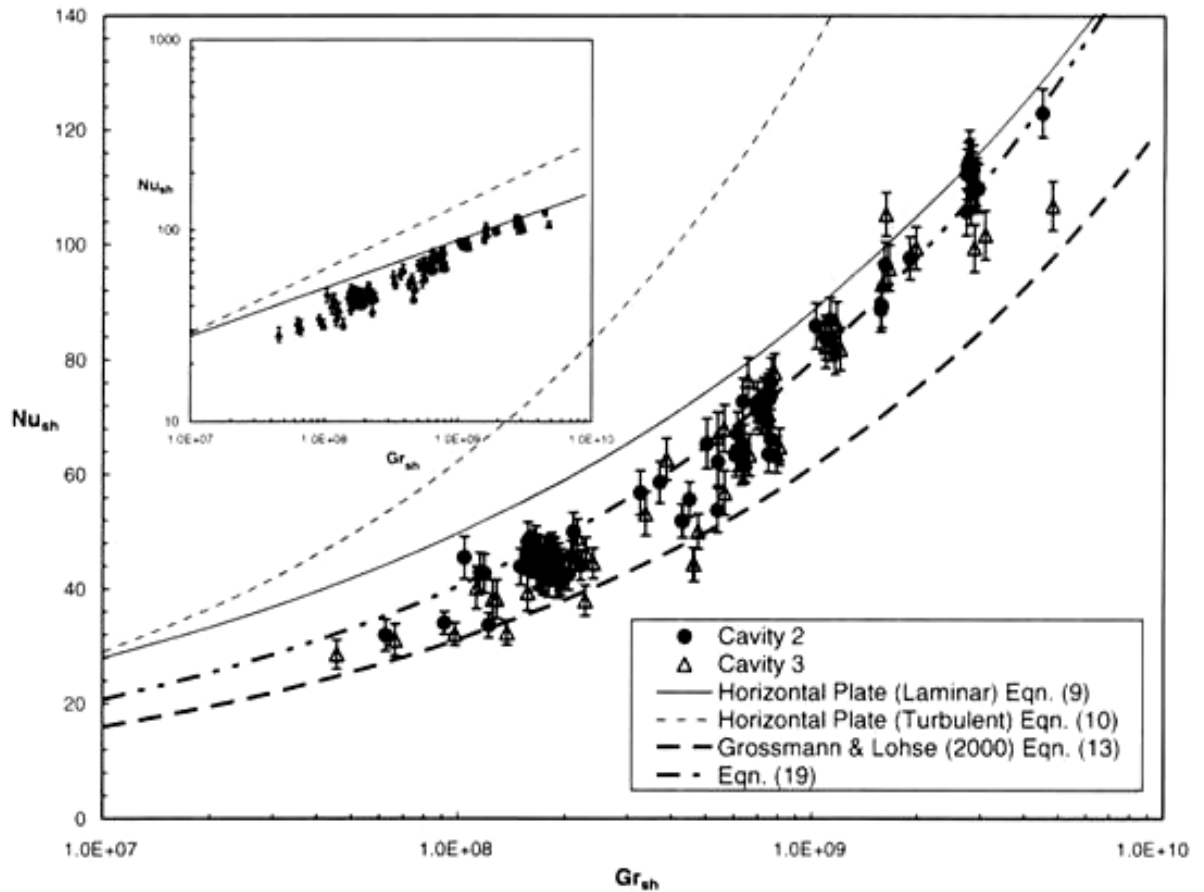


Fig 7. 12 Variation of shroud Nusselt number with Grashof number in both Cavity 2 and Cavity 3 for Build 2 (narrow annular gap); $4 \times 10^4 < Re_z < 1.9 \times 10^5$; $0.29 < Ro < 5.8$. Insert is same data plotted on a logarithmic scale (Long and Childs, 2007)

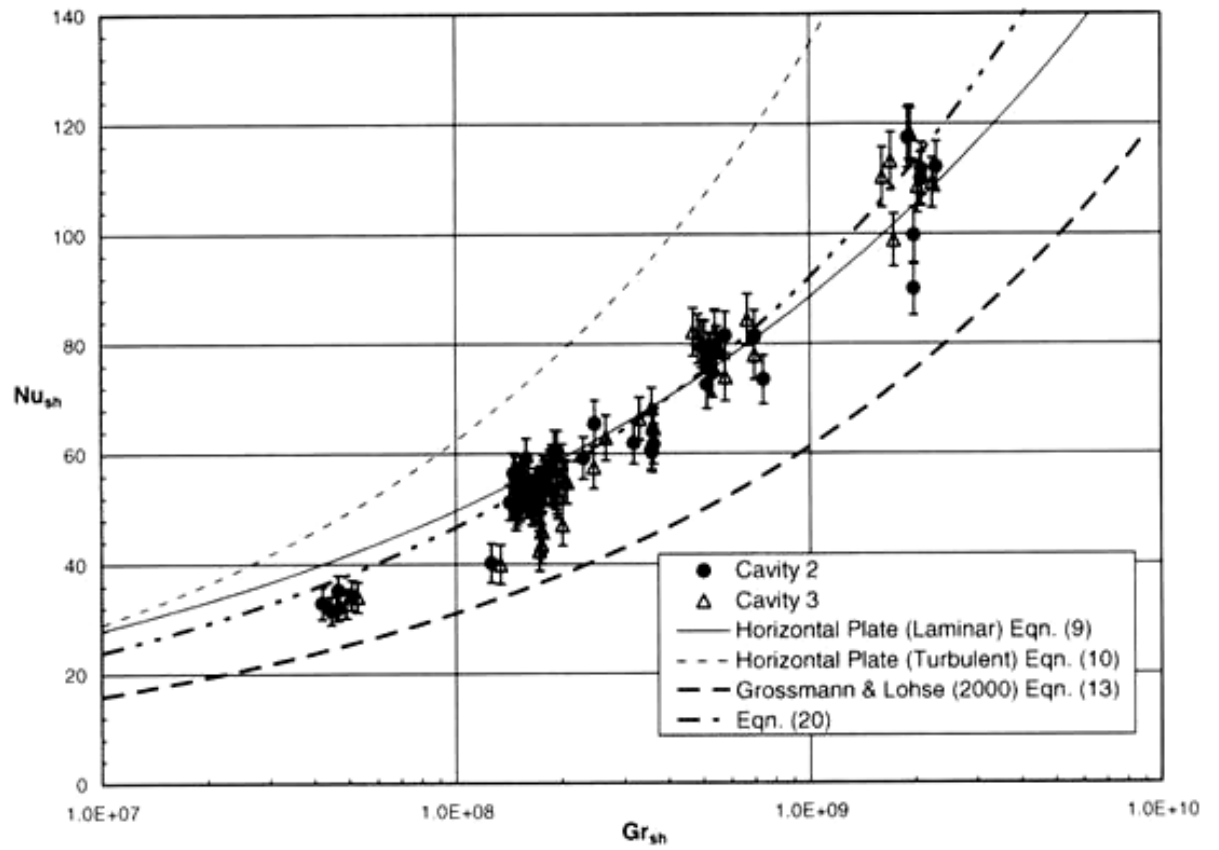


Fig 7. 13 Variation of shroud Nusselt number with Grashof number in both Cavity 2 and Cavity 3 for Build 3 (wide annular gap); $4.1 \times 10^4 < Re_z < 2.0 \times 10^5$; $0.27 < Ro < 5.8$ (Long and Childs, 2007).

7.3 Effect of the Flow Control Feature on cavity heat transfer

The heat transfer results of tests with the Flow Control Feature described in Chapter 3 were plotted for each position: position 1 being located under Disc 2, position 2 is at mid Cavity 3 and position 3 is located under Disc 3.

Figures 7.14 and 7.15 show $Nu_{av}/Re\phi^{0.5}$ against Ro for tests with and without Flow Control Feature for Disc 2 and Disc 3 respectively.

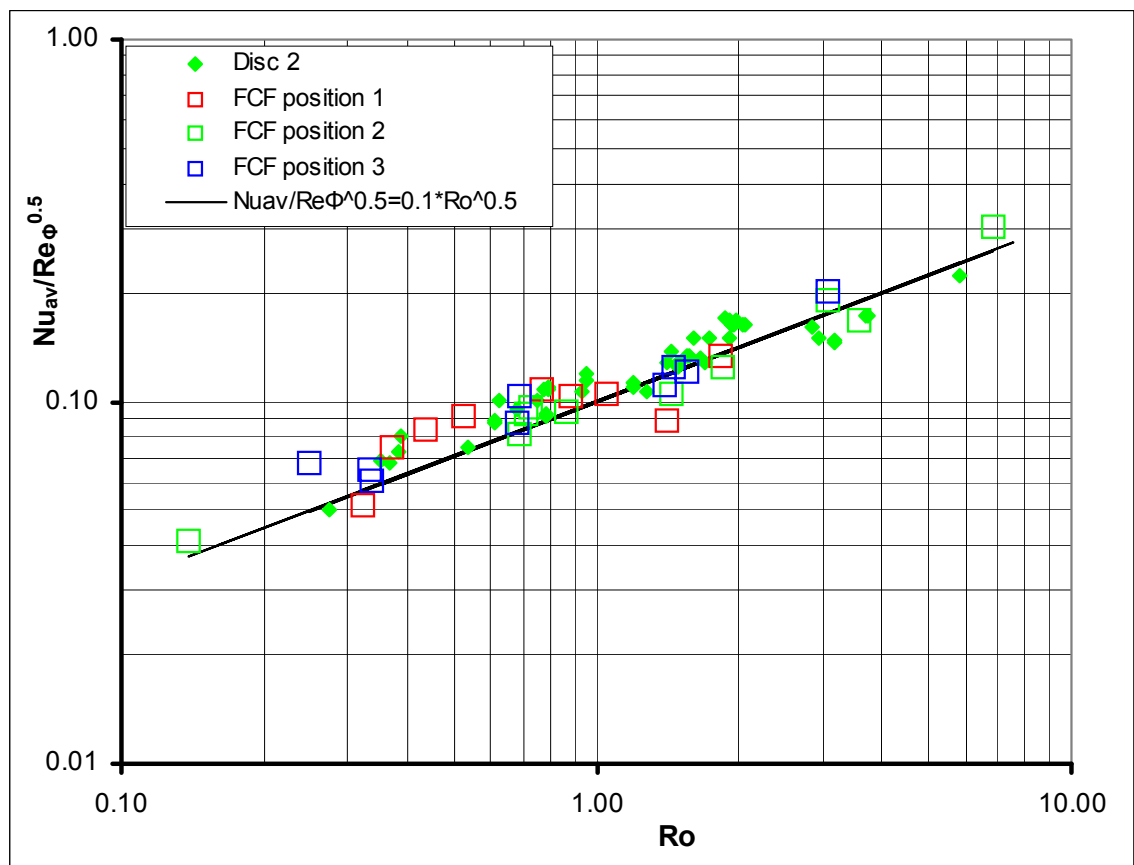


Fig 7. 14 Variations of Disc 2 $Nu_{av}/Re\phi^{0.5}$ against Ro for tests with and without Flow Control Feature against Ro

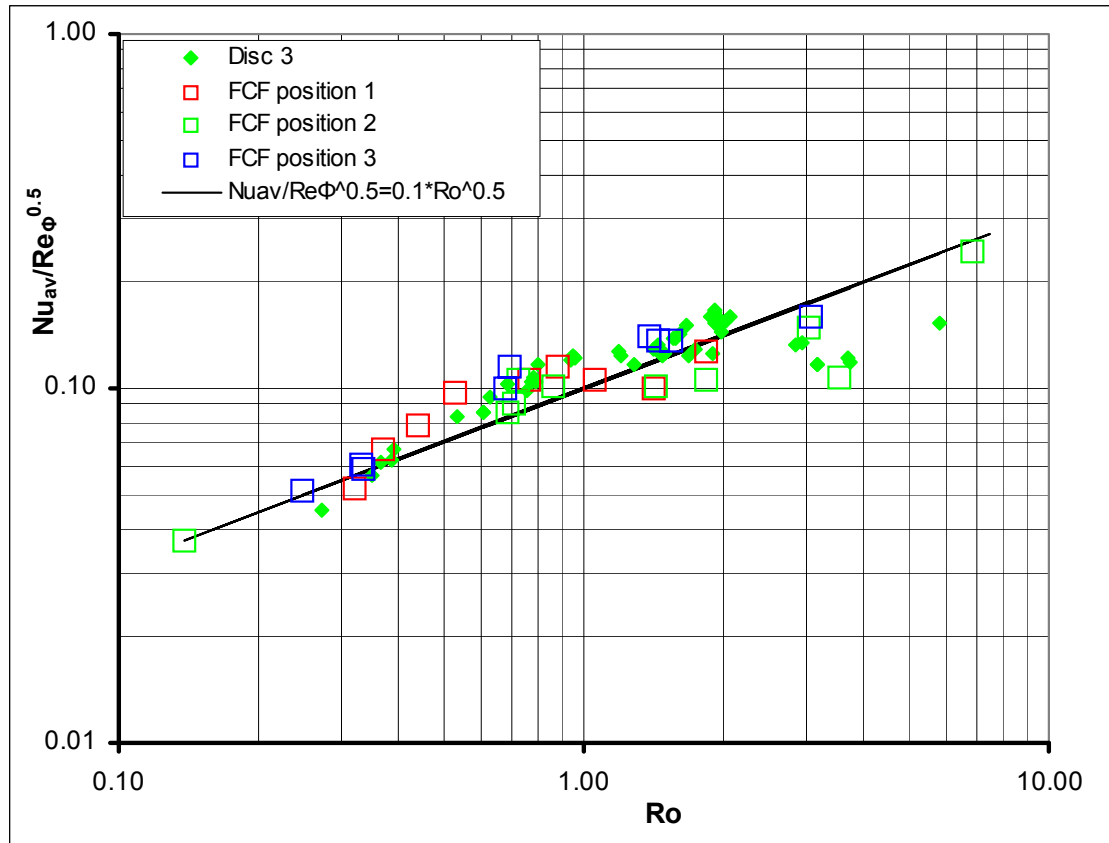


Fig 7. 15 Variations of Disc 3 $Nu_{av}/Re_{\phi}^{0.5}$ against Ro for tests with and without Flow Control Feature against Ro

Taking into account the overall level of uncertainty on the discs heat transfer results, it would appear from Figures 7.14 and 7.15 that the inclusion of the Flow Control Feature on the shaft (in any position) would not have affected the disc heat transfer in a significant or systematic way. The comparison of shroud heat transfer measurements with and without Flow Control Feature for Shroud 2 and 3 in Figures 7.16 and 7.17 respectively would also indicate that the Flow Control Feature did not affect the cavity flow enough to significantly modify the heat transfer characteristics of the shroud.

There probably is an insufficient amount of data to provide any conclusive remarks at this stage on the effect of this Flow Control Feature on the cavity heat transfer and probably more investigations would be required (possibly also with different geometries of Flow Control Features) in order to offer further conclusions on this matter.

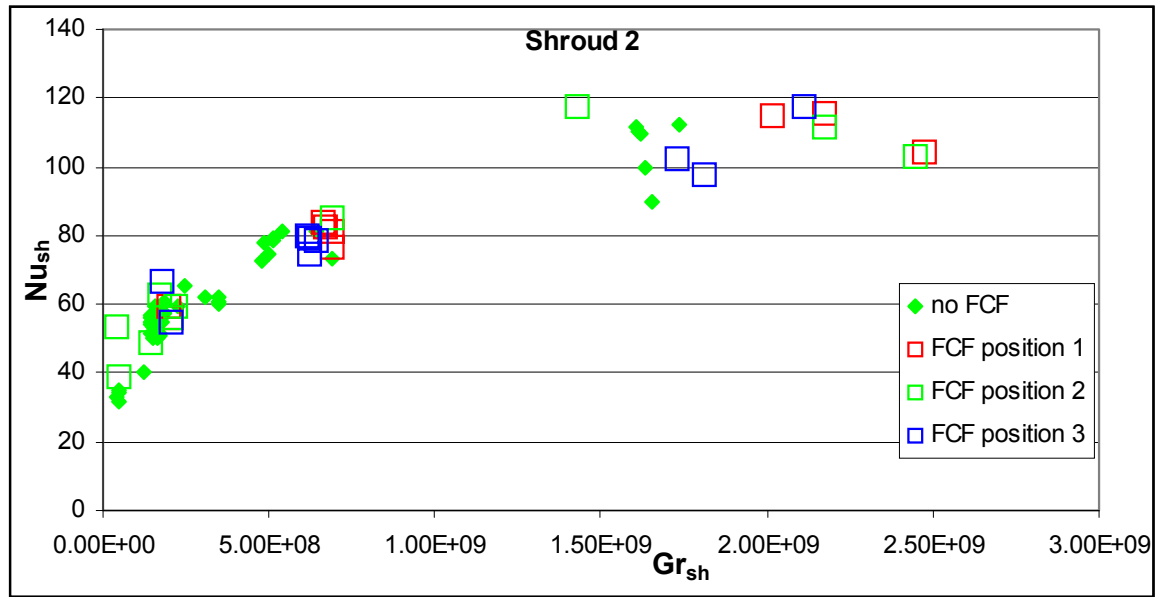


Fig 7. 16 Variations of Shroud 2 Nu_{sh} against Gr_{sh} for tests with and without Flow Control Feature

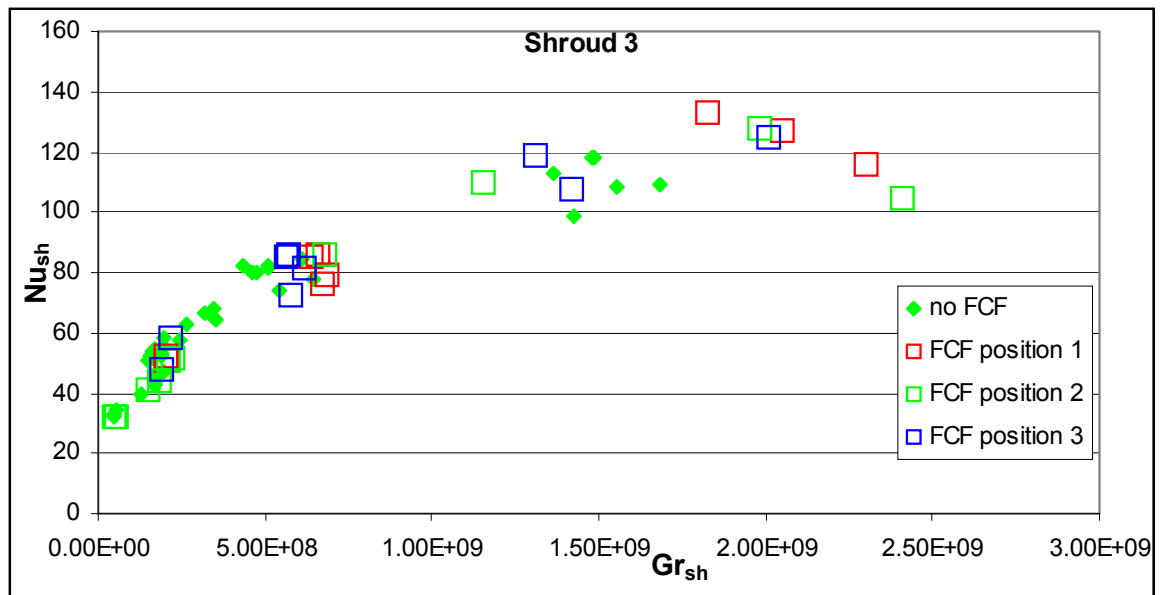


Fig 7. 17 Variations of Shroud 3 Nu_{sh} against Gr_{sh} for tests with and without Flow Control Feature

8. Conclusions

8.1 Experimental apparatus and methodology

This thesis has presented measurements of heat transfer and velocity flow in a heated multiple cavity test rig with axial throughflow.

The experimental test rig was previously designed to be a 70% full scale replica of the internal air system within the high pressure compressor of a gas turbine aero-engine. The rig comprised an encased rotor section reproducing four internal cavities from a typical compressor disc-shroud stack and a central shaft. The cavities had an internal outer diameter $b = 220$ mm and the axial spacing between discs was $s = 42.9$ mm, giving a gap ratio $G = s/b = 0.195$. The discs bore radius on the principal studied rig (Build 3 of the Multiple Cavity Rig) was $r_s = 52$ mm. In the course of this study, comparisons were made with previous results on a former version of the Multiple Cavity Rig, namely Build 2, which featured similar cavity dimensions but with a bigger shaft diameter of value $r_s = 60$ mm. Build 2's major geometrical difference therefore lay in a smaller annular gap ratio $d_h/b = 0.092$ compared to Build 3 ($d_h/b = 0.164$). In the wider annular gap of Build 3 a Flow Control Feature ring of trapezoidal cross section and outer diameter 60 mm could be sleeved around the shaft at three determined axial positions: under Disc 2, at mid Cavity 3 and under Disc 3. The simulated cooling throughflow of air was provided by a single screw compressor at rates up to 1.1 kg/s at 4 bar absolute and 200 °C. The related throughflow operating capacity of the rig was up to $Re_z \leq 1.8 \times 10^5$. Rotation of the rotor assembly was provided by an electrical motor via a belt drive at speeds up to 570 rad/s, corresponding to a maximum operation rotational condition of $Re_\phi \leq 4.1 \times 10^6$. The range of experimental steady-state conditions were $4.7 \times 10^4 \leq Re_z \leq 1.8 \times 10^5$, $4.0 \times 10^5 \leq Re_\phi \leq 4.1 \times 10^6$; giving a corresponding range of Rossby number $0.27 \leq Ro \leq 7.5$.

Heating of the outer rotor surface was provided by hot air impingement from two 10 kW hot air blowers delivering 0.01 kg/s of air at 800 K. This heating configuration provided a realistic, increasing with radius, temperature distribution in the shrouds and discs.

This is representative of the heat generated by the air mainstream in the high pressure compressor stages of a gas turbine. Mass flow and pressure across the rig were measured by orifice plates and pressure tapings linked to a pressure logging unit. Embedded thermocouples provided axi-symmetrical surface measurement temperatures on the surfaces of Shrouds 2 and 3 as well as Discs 2 and 3. The rotating thermocouples were led to a data logger via a slip ring unit and a thermally insulated cold junction box. Additional thermocouples provided information on throughflow air and shaft surface temperatures.

Two-dimensional laser velocimetry was employed for the throughflow and cavity airflow measurements. Two different laser probe mountings were possible, each providing access to different measurement zones and different velocity components. With the laser probe installed in a coaxially translating probe and mirror assembly within the shaft, measurements of axial and tangential velocities were possible in all cavities and the central annular gap regions within the bores of Discs 2 and 3. Optical access to these regions was provided by a longitudinal window on the shaft surface. With the laser probe shooting across windows installed on Discs 3 and 4, measurements of tangential and radial velocity components were made possible in Cavities 3 and 4 for $0.566 \leq r/b \leq 0.924$. Processing of the velocimetry data was executed using the processing software provided with the measuring equipment. The results were collated onto test reports for archiving and analysis, with calculated uncertainty intervals on the results included.

The data from the temperature and pressure measurements were processed into operational parameters and heat transfer local and average values from a purpose written processing piece of software that would handle the raw data, process it and turn the results into lists and standard test reports for each experimental conditions for the purpose of archiving and further analysis. The processing software included one-dimensional shroud heat transfer calculations, two-dimensional disc heat transfer and an adiabatic plane assumption fin solution on Discs 2 and 3. Heat fluxes on the surfaces of Cavity 3 were corrected for radiation by an integrated grey body enclosure radiation model.

A separate error analysis highlighted the sensitivity of the disc heat transfer solutions to the quality of interpolation of the temperature profiles along the discs surfaces from a limited number of thermocouples. This allowed the selection of the appropriate fitting technique to be implemented into the processing software in order to minimise bias on the results. This study also highlighted the sensitivity of the disc heat transfer calculations to the number of thermocouples involved in the surface temperature profiles definition and to the number of nodes of the finite difference grid. A comprehensive uncertainty analysis quantified the influence of different uncertainty sources onto the experimental temperature measurements. The compounded uncertainties were integrated into the processing software as an input into a Monte-Carlo analysis coupled onto the results calculation algorithm in order to provide the resulting values as nominal values bounded by an uncertainty interval for each calculated parameters of each experimental test.

8.2 Experimental results

8.2.1 Flow velocity measurements

Tangential velocity measurements in Build 3 of the Multiple Cavity Rig showed a sharply increasing profile between the shaft and a radius close to the disc bore, at the entrance of the cavity, where it reached a maximum. Then the tangential velocity within the cavity would decrease gradually until reaching solid body rotation at about $r/b \approx 0.6$. The tangential velocities measured in the annular gap showed a quasi linear velocity profile from close to zero by the shaft, tending to solid body rotation by the disc bore. The tangential velocity did not appear to vary significantly across the width of a cavity, although measurements could not be made inside the discs boundary layers. The magnitude of the tangential velocity peak by the cavity entrance appeared related to the dominance of the axial throughflow, with either a decrease of Re_ϕ or an increase in Re_z (hence increase in Ro in both cases) increasing the peak tangential velocity value in the inner regions of a cavity ($r/b < 0.5$). Comparing the tangential velocities of Build 3 to Build 2, which featured a smaller axial gap, showed attenuated velocity profiles compared to Build 3, suggesting a lesser influence of the throughflow on the cavity tangential velocities resulting from the smaller annular gap (as also reported by Owen and Powell 2004), similarly to the attenuating effect reported by Farthing et al. (1992a) when reducing the cavity width. Tangential velocity profiles between cavities in Builds 2 and 3 highlighted more acute velocity profiles as progressing from one cavity to the next. This could be attributed to a compounding influence on the exit flow conditions as progressing through the rig, affecting the cavity flows and vortex breakdown into what would appear to be an increase in throughflow dominance. Axial velocity measurements of the annular gap flow showed a relatively homogeneous flow, although highly fluctuating, under the discs bore being constricted (and accelerated) under the interacting region with the cavity entrance. Contra rotating circulations from the cavity entrance interaction with the throughflow to the outer cavity regions were highlighted, with axial velocity magnitudes of these circulations found to be less than $1/10^{\text{th}}$ of the axial throughflow bulk velocity.

Time averaged radial velocity measurements in the outer regions ($r/b > 0.55$) of Cavities 3 and 4 in Build 3 of the Multiple Cavity Rig were performed in the experimental range $0.39 \times 10^6 \leq Re_\phi \leq 1.99 \times 10^6$, $0.53 \times 10^5 \leq Re_z \leq 1.65 \times 10^5$, $0.53 \leq Ro \leq 7.51$. High levels of fluctuation in the measured velocities (higher to the average values by a factor of two typically) suggested an unsteady or circulatory nature of the flow in the cavities. The radial average velocity circulating flow patterns showed to be different for each cavity and similar across the experimental range, consistent with buoyancy driven flows. The direct inclusion of vortex breakdown into the cavity flow was also recorded on an experiment at $Ro \approx 1.5$. The cavity radial velocities suggested a dominant influence with rotational Reynolds number. Spectral analysis of the tangential velocity measurements across the discs windows showed evidence of periodicity in the flow structure, linked to the current understanding of the flow structure in a heated rotating cavity with axial throughflow.

8.2.2 Heat transfer measurements

The disc heat transfer data were analysed from an adiabatic plane assumption solution. Average disc heat transfer values on Build 3 of the Multiple Cavity Rig showed a good collapse with $Ro.Re_\phi$ and were correlated as:

$$\frac{Nu_{av}}{Re_\phi^{0.5}} \approx 0.1 \times Ro^{0.5} \quad \text{Eq. 8.1}$$

The average disc heat transfer data of Build 2 showed two different regimes, correlated for $Ro \leq 3$ by:

$$\frac{Nu_{av}}{Re_\phi^{0.5}} \approx 0.05 \cdot Ro^{0.5} \quad \text{Eq. 8.2}$$

For $Ro > 3$ an increase in average Nusselt number values was observed, consistent with a flow which is mainly affected by the axial throughflow at large values of rossby numbers, with this sensitivity decreasing as the Rossby number decreases and the influence of the rotational Reynolds increases.

The effects of the Flow Control Feature on the cavity heat transfer were assessed and appeared not to affect the cavity flow enough to significantly modify the cavity heat transfer.

This newly reported data which has been obtained at near engine representative conditions extends the understanding of the unsteady flow structures that exist in the rotating cavities commonly found in gas-turbine power plants. In particular, the velocimetry and temperature data provide improved quantification of the complex and often buoyancy driven flows that determine the heat transfer characteristics within multiple cavity engine architecture. This data is necessary for the validation of thermo-fluid models used to predict components temperature and life as part of the engine design process.

8.3 Recommendations for future work

8.3.1 *Flow velocity measurements*

Important difficulties in taking Laser Doppler Anemometry measurements inside the Multiple Cavity Rig were found in the relatively small validated data rates obtained. The relatively important probe volume due to a narrow crossing angle of the beams involves relatively wide spaced fringes and also fewer of them; hence all working towards a configuration more prone to surface proximity glaring, multiple occurrence sample rejection and poor signal to noise ratio. Improvements on the data measurements density and on the ability to better measure in the central annular gap region could easily be obtained on the current laser system by changing the probe front optics for a reduced focal length. This would induce a bigger crossing angle and therefore a smaller probe volume, more fringes etc. The immediate drawback would be reduced measurement volume range ability. If considering a focal length of 100 mm for instance (compared to 250 mm used in the present experiments), the measurement region range in the configuration with the laser probe installed within the shaft would not allow to measure deep within the cavity radius. On the other hand, the present measurements would indicate relatively quiescent flows in the outer part of cavities beyond $r/b \approx 0.5$, which might be of a lesser interest for further measurements compared to the complex circulatory nature of the flows encountered at the interaction of the annular gap and the cavity entrances and the onset characteristics of vortex breakdown. A more detailed study of these flow regions would therefore be of interest with the benefit of improved spatial and temporal flow definition potential brought by a smaller probe volume. With higher temporal densities also, spectral analysis would increase in confidence of accuracy and perhaps a cellular nature of the swirling throughflow akin to Taylor-Couette type of cells within the central vortex could be identified as the explanation for the very high values of fluctuations on the current time averaged measured flows in this region.

Particle Image Velocimetry could be suitably employed for measurements through the discs windows, with illumination from the shaft optical access.

This would provide a better planar definition than single point measurement and could capture more instantaneous flow characteristics. The difficulties of reconstructing a full cavity flow from discrete window passage measurements would however remain.

The effect of variations of annular gap on the cavity flow could easily be studied by the relatively simple adjunction of varying thickness sleeves with a provision for optical access included onto the existing shaft.

8.3.2 Heat transfer measurements

The presented work has shown that the instrumentation and method used on the Multiple Cavity Rig for the calculation of surface heat transfer is not adequate for the current design. More thermocouples on the disc faces could improve the surface temperature fitting definition but will not remove the fact that the heat transfer solutions are inherently error prone. Statistical error analyses (notably by Cooke 2007) have now demonstrated the extent of bias that such methods could incur. The task remains for future experiments to determine a better type of instrumentation and calculation suitable for inclusion within the intricate confines of a rotating cavity, allowing precise measurement of heat transfer with a good spatial definition, reliability of use and minimal re-calibration requirements. Thin film heat flux sensors, thermochromic paint and thermal imaging have been put forward as possible techniques to allow improved heat transfer measurements within rotating cavities, but each have their pros and cons and none are standing out so far as the best answer to this issue.

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Appendix 1.1: Reproduction of Long et al. (2007)



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Flow measurements inside a heated multiple rotating cavity with axial throughflow

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Abstract

This paper discusses experimental results from a multiple cavity test rig representative of a high pressure compressor internal air system. Measurements of the axial, tangential and radial velocity components are presented. These were made using a two component, laser doppler anemometry (LDA) system for a range of non-dimensional parameters representative of engine conditions (Re_ϕ up to 4×10^6 and Re_z up to 1.8×10^5). Tests were carried out for two different sizes of annular gap between the (non-rotating) drive shaft and the disc bores.

The axial and radial velocities inside the cavities are virtually zero. The size of the annular gap between disc bore and shaft has a significant effect on the radial distribution of tangential velocity. For the narrow annular gap ($d_h/b = 0.092$), there is an increase of non-dimensional tangential velocity $V_\phi/\Omega r$ with radial location from $V_\phi/\Omega r < 1$ at the lower radii to solid body rotation $V_\phi/\Omega r = 1$ further into the cavity. For the wider annular gap ($d_h/b = 0.164$), there is a decrease from $V_\phi/\Omega r > 1$ at the lower radii to solid body rotation further into the cavity. An analysis of the frequency spectrum obtained from the tangential velocity measurements is consistent with a flow structure in the r - ϕ plane consisting of pairs of contra rotating vortices.

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Keywords: Rotating cavity flow; Gas turbine internal air system; Axial throughflow; LDA measurements

1. Introduction

Contemporary gas turbine aircraft engines operate at the limits of technology. Indeed, the *compressor exit* temperature on a modern civil engine exceeds the *turbine entry* temperature of engines 50 years ago. The combination of high temperatures and high rotational speeds creates significant levels of stress. This can lead to deflections in components and cause a loss in efficiency, or worse, compromise safety. Such engines would not be able to operate without an internal air system. This provides ventilation (mostly cooling, but sometimes components are heated), sealing and purging air to the discs, blades, bearing chambers and various cavities.

A schematic diagram of a typical high pressure (H.P.) compressor internal air system is shown in Fig. 1. Air, extracted from stages of the intermediate pressure compressor and destined for the intermediate pressure turbine blades and seals, flows through the annular passage between the drive shaft and H.P. compressor disc bores. On its route, this throughflow may be heated by both convection and viscous dissipation. Since it is bled from the compressor where work has been done to raise its pressure, use of this air represents a parasitic loss to the main cycle. In fact, the internal air system as a whole may use 20% of the mainstream air flow and cost up to 5% of the specific fuel consumption in a modern turbofan engine (Rolls-Royce, 2005).

This paper is the first of two reporting on experimental measurements of flow and heat transfer in H.P. compressor internal air systems. The work reported here focuses on laser Doppler anemometer (LDA) measurements of the

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Nomenclature

| | | | |
|----------------------------|--|------------------------------------|---|
| a, b | inner and outer radii of a disc | W | bulk mean axial velocity of the throughflow |
| d_h | hydraulic diameter | $\beta = 1/T$ | volume expansion coefficient |
| $G = sb$ | gap ratio | ΔT | temperature difference |
| \dot{m} | mass flow rate of the throughflow | μ | dynamic viscosity |
| p | pressure | ν | kinematic viscosity |
| r, z, ϕ | radial, axial and tangential coordinates | ρ | density |
| r_s | shaft radius | Ω | rotational speed of the cavity |
| $Re_z = Wd_h/\nu$ | axial Reynolds number | <i>Subscripts and superscripts</i> | |
| $Re_\phi = \Omega b^2/\nu$ | rotational Reynolds number | av | radially weighted average |
| $Ro = W/\Omega a$ | Rossby number | in | value at inlet |
| s | axial width of cavity | s | denotes a value at the surface |
| t | time | sh | pertaining to the shroud |
| T | temperature | | |
| V_r, V_z, V_ϕ | radial, axial and tangential velocity components | | |

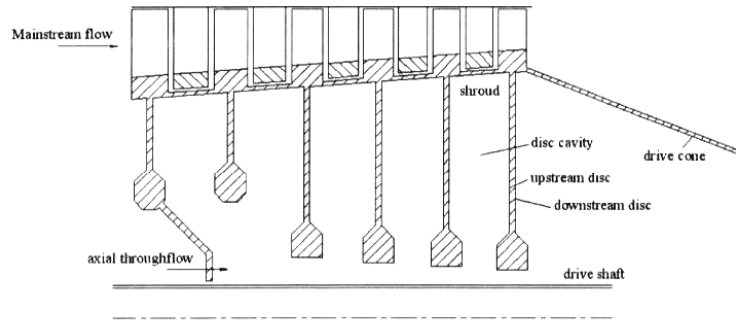


Fig. 1. A schematic diagram of a gas turbine high pressure compressor internal air system.

flow field inside the annular cavities of a test rig which is representative of the design and non-dimensional operating conditions of modern H.P. axial compressors. A background to the flow behaviour including a review of previous relevant work is presented in Section 2. The rig is described in Section 3. The LDA measurements of flow velocities are described in Section 4, and the conclusions to this study are presented in Section 5.

2. Background to the flow behaviour and review of previous work

Previous work, see for example Farthing et al. (1992), on rotating cavities with axial throughflow has mostly concentrated on an idealised single rotating cavity. A simplified model of the compressor cavities depicted in Fig. 1 is shown in Fig. 2. Two discs of outer radius, b , and inner radius, a , are separated by an axial gap, s . The rotational speed of the cavity is Ω and the bulk average velocity of the axial throughflow is W . For a fluid of kinematic viscos-

ity ν , the rotational and axial Reynolds numbers are defined as:

$$Re_\phi = \Omega b^2/\nu \quad (1)$$

$$Re_z = Wd_h/\nu \quad (2)$$

where d_h is the hydraulic diameter of the inlet (for a cavity with an inner shaft of radius, r_s , $d_h = 2(a - r_s)$, for one without an inner shaft, $d_h = 2a$). A further non-dimensional parameter, the Rossby number, Ro links the effects of both rotation and the inertia of the throughflow. This is defined as the ratio of the mean velocity of the throughflow to the tangential velocity at the bore radius:

$$Ro = W/\Omega a \quad (3)$$

This can also be expressed in terms of the rotational and axial Reynolds numbers as:

$$Ro = \left\{ \frac{b^2 Re_z}{2a(a - r_s) Re_\phi} \right\} \quad (4)$$

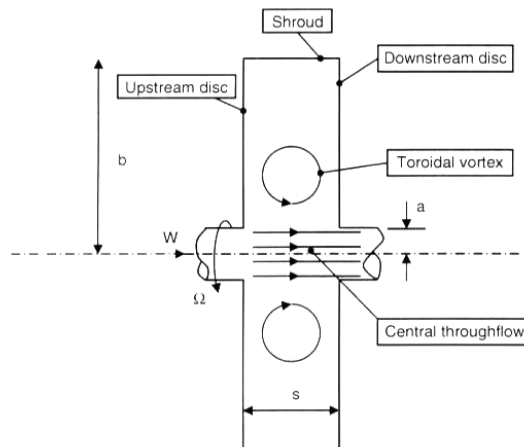


Fig. 2. A schematic diagram in the r - z plane of the isothermal flow structure in a rotating cavity with axial throughflow.

Laser illuminated flow visualisation and LDA was used by Farthing et al. to study the flow structure in unheated (or isothermal) and heated cavities with $a/b \approx 0.1$. The principal parameters to affect the resulting flow are the Rossby number, Ro , and the gap ratio, $G = s/b$. For no rotation, $Ro = \infty$, the throughflow generates one or more (depending on gap ratio) axisymmetric toroidal vortices. Rotation has the effect of suppressing the toroidal vortex and destabilising the central throughflow, creating a change in the behaviour of the central jet. This is characterised by a number of regimes of axisymmetric and non-axisymmetric vortex breakdown. For turbulent ($Re_z > 2000$) flow and for a constant gap ratio four separate regimes of vortex breakdown were identified as the Rossby number was decreased from around 100 to a value of less than 1. These were given the following names, in order of decreasing Rossby number: Mode 1a, Mode 2a, Mode 1b and Mode 2b. For a gap ratio of $G = 0.533$, the respective boundaries of these regimes occur at $Ro = 20, 2.6$ and 1.5 . A full description of these regimes of vortex breakdown can be found in Owen and Rogers (1995). In brief, Mode 1 regimes are associated with a non-axisymmetric behaviour of the central throughflow; Mode 2 behaviour is associated with axisymmetric behaviour. Decreasing the gap ratio suppresses the formation of Mode 1a behaviour.

When heated, significantly more of the throughflow appears to penetrate further into the outer part of the cavity. Most observations of the heated flow structure were made with a cavity ($G = 0.124$ and 0.267) having a surface temperature distribution that decreased with radius. A schematic diagram, in the r - ϕ plane, of the heated flow structure is shown in Fig. 3. Flow enters the cavity through a radial arm, bifurcates near the outer radius and forms a region of cyclonic flow (with the same sense of rotation as the cavity) and anticyclonic flow (having the opposite sense of rotation). It was not possible to obtain such clear visual

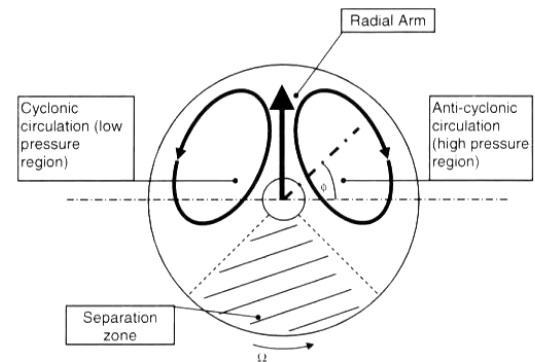


Fig. 3. A schematic diagram in the r - ϕ plane of the heated flow structure in a rotating cavity with axial throughflow.

or photographic evidence of the flow structure in a cavity with a surface temperature distribution that increased with radius. The overall impression was that significantly more of the central throughflow penetrated the cavity, the higher velocities leading to an ill defined flow structure especially in the region adjacent to the peripheral shroud. These differences may be qualitatively explained from consideration of thermal stratification in a rotating flow field. A surface-to-fluid temperature difference that decreases with radius results in stable stratification. For a surface-to-fluid temperature difference that increases with radius there is unstable stratification, resulting in increased radial mixing. This suggests that heating the cavity creates buoyancy forces that act to destabilise the central throughflow. The resulting flow structure may or may not be dominated by buoyancy forces – it depends on cavity geometry as well as physical parameters such as the axial and rotational Reynolds numbers.

The role of buoyancy forces is confirmed in the various studies carried out to measure heat transfer in rotating cavities with axial throughflow. A more detailed review can be found in Alexiou (2000). In brief, measurements appear to indicate a flow that can be either dominated by free convection, by forced convection or a combination of these. The heat transfer behaviour depends not only on the values of rotational and axial Reynolds numbers but also on geometric parameters such as gap ratio, inlet radius ratio, the presence of a shaft, its radius and sense of rotation relative to the cavity surfaces. It is obvious from this list of many variables that to carry out a comprehensive study of their effects on the flow presents a time consuming and daunting task.

Owen and Powell (2004) made velocity and heat transfer measurements in a single cavity research rig with an annular inlet, $a/b = 0.4$, $s/b = 0.2$ with just the downstream disc heated. Tests were carried out for $4 \times 10^5 < Re_\phi < 3.2 \times 10^6$ and $1.4 \times 10^3 < Re_z < 5 \times 10^4$. The time-average LDA measurements of the tangential velocity show that near to solid body rotation occurs ($0.96 < V_\phi/\Omega r < 0.99$ for the range of dimensionless radii $0.67 < r/b < 0.97$) in a cavity where the

downstream disc is heated to 75 K above the inlet air temperature. The radial velocity was found to be approximately two orders of magnitude smaller than the absolute value of tangential velocity. The time-average tangential velocity was also found to increase above $V_{\phi}/\Omega r = 1$ when the temperature difference between the disc surface and the inlet air was reduced to below approximately 40 K. A spectral analysis of the velocity measurements revealed behaviour that is consistent with one, two or three pairs of cyclonic and anticyclonic vortices in the flow field. In general, (although there are exceptions) for a constant axial Reynolds number, increasing the rotational Reynolds number reduces the number of cyclone–anticyclone vortex pairs. The heat transfer measurements support the existence of two different regimes: a buoyancy induced regime at large rotational speeds and small axial flow, and a throughflow dominated regime at the smaller values of rotational speed and larger values of axial throughflow.

Various numerical studies have been carried out: Long and Tucker (1994), Long et al. (1997), Tucker and Long (1995), Tucker and Long (1996), Wong (2002), Sun et al. (2004) and Tian et al. (2004). In general, the results from these CFD studies are qualitatively similar to experimental observations and give acceptable agreement with heat transfer and LDA measurements. However, massive computational resources are required for CFD simulations of this flow. Owen et al. (2006) report on using a cluster of 16 Linux PCs on a 3-D, time-dependent model with 2.6 million grid nodes. This required one week of CPU time. So, the accuracy of much of the numerical work is expected to be constrained by relatively coarse grid sizes and also inappropriate turbulence models. The recent 3D numerical study by Tian et al. (2004) supports the earlier qualitative flow visualisation work of Farthing et al. The isothermal flow structure is seen to be axisymmetric and stable, rotation decreases the influence of the central toroidal vortex. For a heated cavity, the flow may become unstable due to the influence of rotationally induced buoyancy. An instability that develops close to the shroud was seen to affect the rest of the cavity as the Grashof number was increased. Heat transfer data are also presented and these appear to be consistent with previous experimental measurements and show the different effects of the two regimes of forced and free convective heat transfer.

Johnson et al. (2004) developed a stability analysis and applied this to the case of rotating cavity flow with axial throughflow. This shows that for $Ro < 0.1$ the flow in the cavity may be stabilised by a density gradient that increases with increasing radius, but not for $Ro > 1$. For intermediate values of Ro ($0.1 < Ro < 1$) achieving stability involves a more complex relationship of velocity and temperature profiles.

3. Experimental apparatus

The LDA measurements discussed in this paper were carried out on a multiple cavity test rig. As shown in

Fig. 4, the rotor has three internal discs forming four cylindrical cavities and the disc bores are of identical inner radius. To allow optical access along the centreline, the central shaft is stationary. The titanium 318 rotor is bolted together at the periphery, each cylindrical cavity has an inner radius ratio of $a/b = 0.318$, and a gap ratio of $s/b = 0.195$ (dimensions are given in non-dimensional form at the request of the industrial sponsors). The rotor is independently driven by an A.C. motor up to 10,000 rev/min. As in a high pressure compressor, a steel ‘drive’ shaft runs through the centre of the rig. Two different sizes of shaft were investigated. These will be referred to as the narrow annular gap, with $d_h/b = 0.092$, and the wide annular gap with $d_h/b = 0.164$.

The rotor assembly is housed within a steel pressure casing and supported by a high precision grease lubricated ball bearing on the upstream side and an oil lubricated cylindrical roller bearing on the downstream side. Since the rig is pressurised, a number of labyrinth seals are used in critical locations to reduce the leakage flow. These seals were designed to industrial design codes; the fins are on the rotating part, the static radial clearance is 0.1 mm and the stationary components are coated with an abrasible material. Full details are available in Alexiou (2000).

Air is supplied to the rig by a single stage 275 kW Howden compressor which can deliver up to 1.1 kg/s of air at 4 bar (absolute) and 200 °C. Before entry to the rig, the air is cooled to 30–55 °C. To eliminate condensation, the actual value of this temperature is varied depending on the measured relative humidity of the air and its supply pressure. The air flow rate is measured by a single orifice plate at inlet and by another at exit to the rig. These orifice plates are designed to BS 1042, and this particular configuration allows separate determination of the inlet mass flow and by inference (and also calculation) the leakage through the labyrinth seals. Air enters the rig itself through a circumferential array of six 25 mm diameter holes. On the downstream side of this is a thin perforated plate designed to create a uniform velocity profile at inlet to the rig.

The outer surface of the rotor assembly is heated by impingement of hot air obtained from two 10 kW Leister 5000 8D5 heaters. These each supply 0.01 kg/s of air at up to 800 K. Using this system it was possible to obtain a maximum rotor surface temperature of 195 °C at 10,000 rev/min. As in an actual high pressure compressor, the discs are heated by conduction from the outer surface of the rotor. Although the level of surface temperature is lower, this method of heating the test rig achieves similar temperature distributions on the discs and similar values of the surface to air temperature difference to that found in an engine.

The surface temperatures of the rotor assembly and relevant air temperatures are measured using calibrated 0.25 mm wire diameter K-type (NiCr–NiAl) thermocouples. All the rotor instrumentation is led out to a 48-way, Wendon slip ring unit driven from the upstream side of the rig. The signals from all the temperature instrumentation are recorded by a Solartron Orion data logger, with

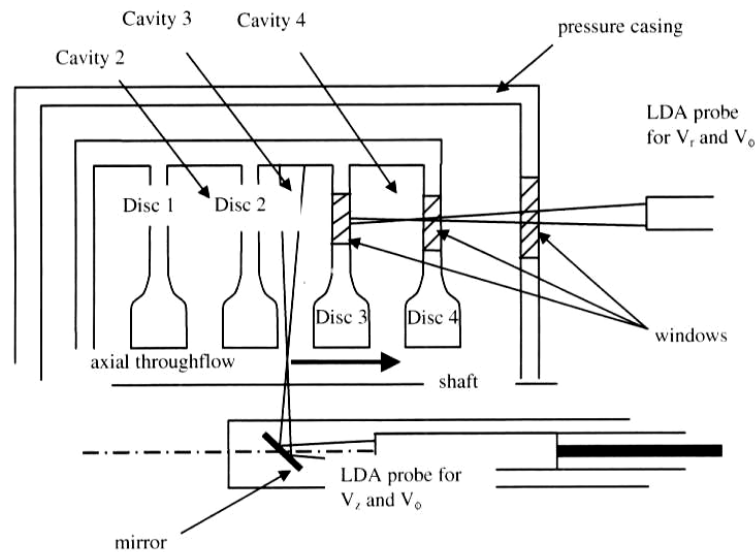


Fig. 4. A schematic diagram of the multiple cavity rig.

a sensitivity of $\pm 1 \mu\text{V}$ corresponding to $\pm 0.025^\circ\text{C}$ for the thermocouples used. However, the overall uncertainty in temperature measurement, which includes the effects of measurement and calibration uncertainty, is around an order of magnitude greater than this figure.

Velocity measurements were carried out using a Dantec, two component, LDA system. The discs and inner peripheral surfaces were painted black to reduce reflections and all the windows had an anti-reflection coating. The most important parameters of the LDA system are given in Table 1. The flow was seeded from the upstream part of the shaft with atomised quenching oil (RS 184-7939) using a TSI 9306 six jet oil droplet atomiser, the diameter of the seeding particles is expected to be approximately $1 \mu\text{m}$.

Two different configurations of optical access were used for the LDA measurements, depending on which compo-

nents of velocity were being measured. For the simultaneous measurement of axial and tangential components, the LDA probe was mounted on a traverse aligned with the centreline of the test rig. This allowed optical access into each cavity through a window in the central shaft made from 3 mm thick BK 7, Grade A optical quality glass, 20 mm wide and 150 mm long. Fig. 4 shows part of the traverse system used to move the LDA probe volume in the radial and axial directions. This tube is inserted into the downstream end of the rig inside the hollow non-rotating central drive shaft. The LDA probe, containing transmitting and receiving optics, is carried within the inner tube indicated in Fig. 4, the mirror, which directs the beams into the cavity space is carried inside the outer tube. Both inner and outer tubes can be independently positioned in the axial direction by stepper motors. Radial displacement of the probe volume is obtained by moving the inner tube (carrying the LDA probe) and axial displacement by moving the whole system. For the simultaneous measurement of radial and tangential components, the LDA probe was mounted off-axis in another traverse system and optical access into Cavities 3 and 4 obtained through eight 50 mm diameter windows in the outer casing and in the disc surfaces (four equally spaced at $r/b = 0.638$ and the other four at $r/b = 0.838$). This arrangement is also indicated in Fig. 4.

4. LDA velocity measurements

4.1. Experimental procedure and data analysis

The data from LDA tests discussed in this paper were acquired under thermal steady state conditions. This was

Table 1
LDA system parameters

| Parameter | Value |
|--------------------------------|--------------------------------|
| Transmitting lens focal length | 250 mm |
| Beam spacing | 15 mm |
| <i>Wavelengths</i> | |
| Tangential velocity (green) | 514.5 nm |
| Axial velocity (blue) | 488 nm |
| Laser beam diameter | 1.35 mm |
| Beam half angle | 1.718° |
| Expansion factor | 1 |
| Probe volume diameter | 0.121 mm (514.5 nm wavelength) |
| Probe volume length | 4.046 mm (514.5 nm wavelength) |
| Number of fringes | 14 |
| Fringe spacing | 8.579 μm |
| Frequency shift | 40 MHz |

confirmed by monitoring a change in surface temperature of less than 0.2 °C in a period of 5 min. The variables investigated, their range, the corresponding range of the relevant non-dimensional groups, together with typical values from an engine, are given in Table 2.

For all the measurements presented here, the mean value of velocity is estimated from a minimum sample size of 100, although usually the sample size was 5000. Also shown in some graphs is the root-mean-square (RMS) variation in the sample. It should be emphasised that this is not an 'error' bar, even though it is presented in a similar form. The RMS variation in the velocity measurement represents true physical fluctuations in the measured velocity in addition to various sources of experimental uncertainty.

Uncertainties in LDA measurements can arise from several sources:

- signal noise
- seeding
- velocity bias
- velocity gradient broadening
- probe alignment
- transit time broadening
- fringe bias.

It is perhaps worthwhile to briefly explain how these have been addressed and their effect on the measurements. Signal noise is reduced in the test rig by painting all the interior surfaces matt black. The windows are made from a high quality glass (BK7) that is recommended for LDA work. The windows and mirrors were cleaned regularly, usually before a test. The trade off between validation and data rate was investigated and the signal to noise ratio of the signal processor adjusted to achieve an optimum of this balance. The laser power was maintained at the lowest working value (usually around 150–350 mW), to reduce the effects of saturating the photomultipliers.

Quenching oil was used for the seeding medium and this can be atomised to a mean particle size of approximately 1 µm. This is generally accepted as being small enough to faithfully follow the motion of air flows with or without rotation. No difference was noted in the validation and data rates of using either one jet or all six jets in the atomiser, so one jet was used to reduce the fouling of the window.

Velocity bias originates from the fact that LDA signals are obtained from intermittent data. So a disproportionate number of signals are obtained when the velocity is high, since more of these particles pass through the probe volume in a given time. This results in a value of mean velocity that is biased towards a higher value than the 'true' mean. There are well known data processing algorithms to correct for this and these are implemented in the Dantec software and so no further correction was considered necessary.

Velocity gradient broadening occurs in regions where there is a significant velocity gradient in a direction normal to the fringes of the probe volume. This is unlikely to be a significant source of uncertainty in regions away from the solid surfaces. For the measurements of tangential velocity inside the cavity, at the highest rotational speed, there will be a velocity difference of about 2 m/s across the probe volume and this is less than 3% of the tangential velocity.

Transit time broadening occurs as a result of a measurement of Doppler burst frequency being obtained from a finite number of fringes – the fewer the fringes the greater is the potential uncertainty. The LDA apparatus uses 14 fringes and the uncertainty introduced by this considering the large number of samples usually taken is considered to be small.

Fringe bias occurs in flows with significant changes of direction, e.g., in highly recirculating flows, where the flow will change direction inside the probe volume itself. This causes the measurement to be rejected as insufficient fringes have been traversed. This will affect the flow statistics, but

Table 2
Range of experimental conditions

| Parameter | Symbol (units) | Narrow annular gap $d_h/b = 0.092$ | Wide annular gap $d_h/b = 0.164$ | Uncertainty/[] ^a |
|------------------------------------|--|--|--|------------------------------|
| Rotor speed | Ω (rad/s) | $123 \leq \Omega \leq 306$ | $65 \leq \Omega \leq 570$ | ± 1 rad/s |
| Cooling air flow rate | \dot{m} (kg/s) | $0.184 \leq \dot{m} \leq 0.618$ | $0.2 \leq \dot{m} \leq 0.72$ | $\pm 3\%$ |
| Inlet pressure | p_{in} (bar, abs.) | $2.25 \leq p_{in} \leq 3.0$ | $2.3 \leq p_{in} \leq 3.1$ | $\pm 0.1\%$ |
| Inlet temperature | T_{in} (K) | $311 \leq T_{in} \leq 319$ | $302 \leq T_{in} \leq 339$ | ± 0.2 K |
| Disc average surface temperature | $T_{s,av}$ (K) | $328 \leq T_{s,av} \leq 339$ | $321 \leq T_{s,av} \leq 353$ | ± 0.4 K |
| Shroud average surface temperature | T_{sh} (K) | $361 \leq T_{sh} \leq 376$ | $353 \leq T_{sh} \leq 385$ | ± 0.4 K |
| Disc rotational Reynolds number | Re_ϕ , Eq. (1) | $9.5 \times 10^5 \leq Re_\phi \leq 2 \times 10^6$ | $4.0 \times 10^5 \leq Re_\phi \leq 4.1 \times 10^6$ | $[8 \times 10^6]$ |
| Axial Reynolds number | Re_z , Eq. (2) | $4.8 \times 10^4 \leq Re_z \leq 1.5 \times 10^5$ | $4.7 \times 10^4 \leq Re_z \leq 1.8 \times 10^5$ | $[2 \times 10^5]$ |
| Rossby number | Ro , Eq. (3) | $1.27 \leq Ro \leq 5.1$ | $0.27 \leq Ro \leq 7.5$ | $[1.5]$ |
| Shroud buoyancy parameter | $\beta \Delta T_{sh} = \beta(T_{sh} - T_{in})$ | $0.15 \leq \beta \Delta T_{sh} \leq 0.19$ | $0.11 \leq \beta \Delta T_{sh} \leq 0.238$ | $[0.35]$ |
| Disc buoyancy parameter | $\beta \Delta T_{av} = \beta(T_{s,av} - T_{in})$ | $0.04 \leq \beta \Delta T_{av} \leq 0.08$ | $0.02 \leq \beta \Delta T_{av} \leq 0.09$ | $[0.15]$ |
| Grashof number | $Re_\phi^2(\beta \Delta T)_{sh}$ | $1.7 \times 10^{11} \leq Re_\phi^2(\beta \Delta T)_{sh} \leq 6 \times 10^{11}$ | $1.8 \times 10^{10} \leq Re_\phi^2(\beta \Delta T)_{sh} \leq 2.5 \times 10^{12}$ | $[2 \times 10^{13}]$ |

^a Values indicated in square brackets '[]' are typical of engine conditions.

its effect on the measurements in this particular flow is considered to be marginal.

The accuracy of the location of the probe with respect to the test rig and maintaining that accuracy over successive tests can be a significant source of uncertainty when one velocity component is considerably greater than another. This is the case in the measurements of radial velocity and as will be explained later in this section, great care was taken to achieve accurate alignment.

4.2. Axial and tangential velocity components

Fig. 5 shows the variation of tangential velocity, V_ϕ , with non-dimensional radial location, r/b , for a wide annular gap ($d_h/b = 0.164$) test with $Re_\phi = 0.49 \times 10^6$, $Re_z = 0.92 \times 10^5$ and $Ro = 3.57$. The velocity measurements are mostly made inside Cavity 3 (see Fig. 4), with others made directly under the bore of Disc 2 (3 mm from the upstream face and 3 mm from the downstream face) and a further set of measurements made at the entrance to Cavity 2 (approximately 3 mm downstream of the bore of Disc 1). The measurements at $z/s = 0.33$ are made in the same plane as are the measurements at $z/s = 0.5$ and 0.67 and the axial width of the cavity at $r/b = 0.9$ is used as a reference value for s . The outer radius of the shaft is located at $r/b = 0.236$ and the bore of the discs at $r/b = 0.318$. The velocity measurements show that the tangential velocity is below disc speed in the annular gap between shaft and disc bores, varying from around 1 m/s (corresponding to $V_\phi/\Omega r \approx 0.25$) just above the shaft to 5 m/s ($V_\phi/\Omega r = 0.9$), just radially inward of the bore of Disc 2. Within the cavity itself, the flow is spun up to above disc speed in the small radial dis-

tance between the shaft and the equivalent radius to the disc bore and a maximum tangential velocity of 12 m/s ($V_\phi/\Omega r = 2.2$) is obtained inside Cavity 3 at $r/b = 0.34$. A value of $V_\phi/\Omega r > 1$ suggests that radial inflow should occur in the disc boundary layers. However, due to the shape of the disc cobs it was not possible to measure velocities close to the disc surface. Further into the cavity, the tangential velocity reduces and tends toward disc speed or solid body rotation ($V_\phi/\Omega r = 1$). There appears to be little significant difference between the tangential velocities measured at the different planes inside Cavity 3. The RMS values are much larger in the inlet region than in the cavity itself indicating the presence of widely fluctuating velocities and a region of mixing between the cavity flow and the axial throughflow. It is interesting to compare the velocity measurements shown in Fig. 5 with those presented by Farthing et al., for *isothermal flow*. These show similar behaviour: an increase in the normalised tangential velocity to above solid body rotation in the inlet region and a decrease to solid body rotation further into the cavity.

The radial variation of the *axial* component of velocity for the test shown in Fig. 5 (with $Re_\phi = 0.49 \times 10^6$, $Re_z = 0.92 \times 10^5$, $Ro = 3.57$) is presented in Fig. 6. For $r/b > 0.318$, which is inside the cavity, the axial velocity is virtually zero irrespective of the axial location of the measurement plane. For $0.236 < r/b < 0.318$, which is in the inlet region, the values of axial velocity are comparable with the bulk average velocity ($W = 17.45$ m/s). As was noted for the tangential component of velocity, the RMS values are much larger in the inlet region indicating large local fluctuations in velocity, considerable turbulence and mixing between the throughflow and the cavity flow. An

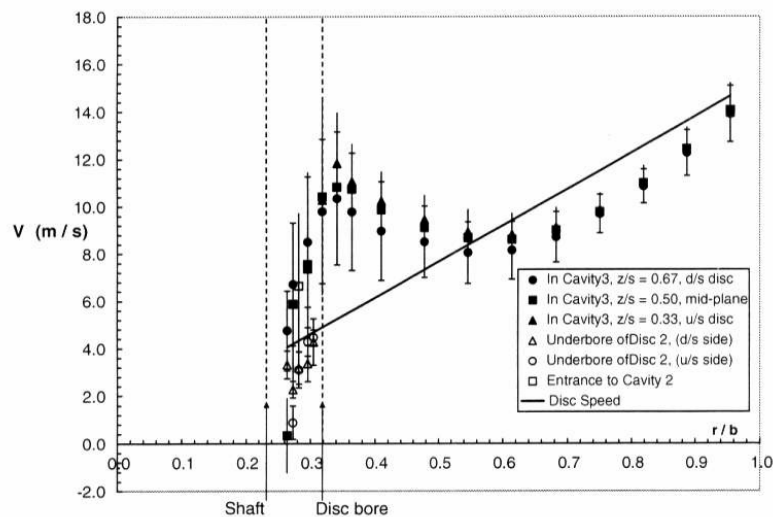


Fig. 5. Radial variation of tangential velocity at different measurement planes: $d_h/b = 0.164$, $Re_\phi = 0.49 \times 10^6$, $Re_z = 0.92 \times 10^5$, $\beta\Delta T_{av} = 0.09$, $\beta\Delta T_{sh} = 0.23$ and $Ro = 3.57$. Bars show RMS variation in velocity.

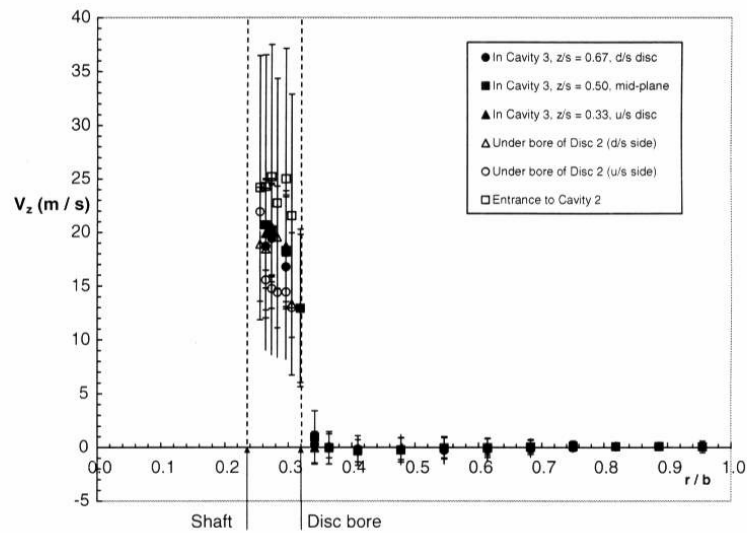


Fig. 6. Radial variation of axial velocity at different measurement planes: $d_h/b = 0.164$, $Re_\phi = 0.49 \times 10^6$, $Re_z = 0.92 \times 10^5$, $\beta\Delta T_{av} = 0.09$, $\beta\Delta T_{sh} = 0.23$ and $Ro = 3.57$. Bars show RMS variation in velocity.

estimate of the turbulence intensity can be made directly from the RMS values. For the data shown here, a value of $\pm 0.4 W$ is representative of the flow in the region between the shaft and the disc bore. For the flow in the outer part of the cavity, a value of $\pm 0.05\Omega r$ is representative of the data at $r/b = 0.89$.

Fig. 7 shows the variation of non dimensional tangential velocity $V_\phi/\Omega r$ with radius ratio, r/b , for a number of tests at $Re_z \approx 10^5$ and different values of rotational Reynolds number Re_ϕ with the wide annular gap ($d_h/b = 0.164$). The measurements are made in the mid-plane, $z/s = 0.5$, of Cavity 3. At the lowest value of Re_ϕ ($Re_\phi = 4.9 \times 10^5$), the normalised tangential velocity increases from just above zero near to the stationary shaft and reaches a maximum value of $V_\phi/\Omega r = 2.3$ at $r/b = 0.35$ which approximately corresponds to the radial location of the disc bores. Further into the cavity, $r/b > 0.35$, $V_\phi/\Omega r$ decreases and a value just below solid body rotation occurs at $r/b = 0.6$. The effect of increasing the rotational Reynolds number is seen by comparing these results with those for $Re_\phi = 10^6$, 2×10^6 and 4.1×10^6 . An increase in Re_ϕ (that is, a decrease in Ro) reduces the peak dimensionless tangential velocity in the region between the shaft and the disc bore. Further into the cavity there is little difference between the values of $V_\phi/\Omega r$.

A similar trend occurs when the axial Reynolds number is varied and the rotational Reynolds number held constant. This is illustrated in Fig. 8 which shows the variation of $V_\phi/\Omega r$ with r/b for tests with $Re_\phi \approx 10^6$ and $Re_z = 4.9 \times 10^4$, 9.7×10^4 and 1.7×10^5 . Decreasing the Rossby number by decreasing the axial Reynolds number reduces

the peak dimensionless tangential velocity in the region between the shaft and the disc bore.

The results shown in Figs. 7 and 8 suggest that the distribution of tangential velocity close to the shaft ($r/b < 0.5$) is controlled by the Rossby number (Eq. (4)) since this is a ratio of axial to rotational Reynolds numbers. A larger value of Rossby number implies an increase in the relative strength of the axial to tangential velocities which is expected to have more of an influence on the flow inside the cavity. This is also consistent with the findings of Farthing et al. who found that increasing the Rossby number increased the value of maximum normalised tangential velocity near the disc bore.

It is possible to use the results shown in Figs. 7 and 8 to comment on the role of buoyancy in this flow. For a buoyancy induced flow, the relative surface to freestream velocity outside the boundary layers is expected to be close to zero. Although the velocity inside the boundary layers will depend on the value of the Grashof number. The Grashof number may be thought of as the product Re_ϕ and $\beta\Delta T$. The sensitivity to Re_ϕ , as shown in Fig. 7, would therefore tend to suggest that the flow outside the boundary layers is governed by buoyancy forces. However, the results shown in Fig. 8 would appear to contradict this as such a flow regime would not be expected to show a variation with a Reynolds number based on the axial flow. The evidence presented by Long and Childs (2007) clearly demonstrates that the heat transfer from the peripheral shroud of this test rig is governed by the Grashof number. These conflicting observations may be resolved in the following way. As noted above, in the inner part of the cavity (say $r/b < 0.6$)

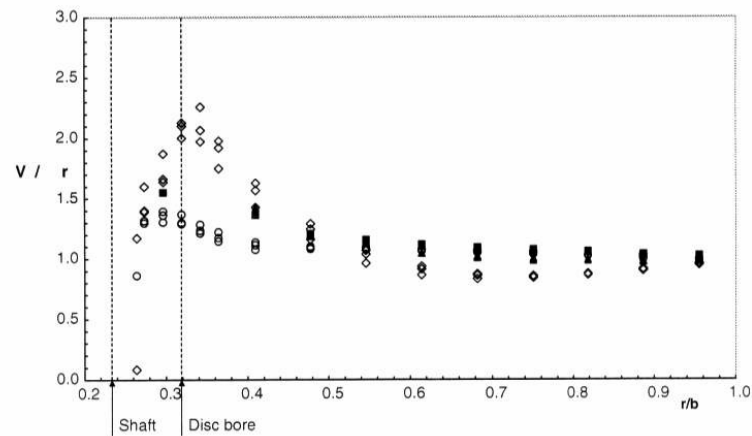


Fig. 7. Radial variation of normalised tangential velocity, $V_\phi/\Omega r$, with rotational Reynolds number for tests with the wide annular gap ($d_0/b = 0.164$) and $Re_z \approx 10^5$.

| | $Re_\phi/10^6$ | $Re_z/10^5$ | Ro | $\beta\Delta T_{av}$ | $\beta\Delta T_{sh}$ |
|---|----------------|-------------|------|----------------------|----------------------|
| ◇ | 0.49 | 0.92 | 3.57 | 0.09 | 0.23 |
| ▲ | 1.31 | 0.97 | 1.41 | 0.05 | 0.16 |
| ■ | 2.0 | 0.99 | 0.94 | 0.04 | 0.14 |
| ○ | 4.1 | 1.15 | 0.53 | 0.03 | 0.12 |

the flow is controlled by the ratio of axial to rotational Reynolds numbers as expressed by the Rossby number, Ro . In the outer part of the cavity ($r/b > 0.6$) solid body

rotation occurs for all values of Ro tested. Since the relative velocity between surface and freestream is close to zero, buoyancy forces will be significant. Hence the heat transfer

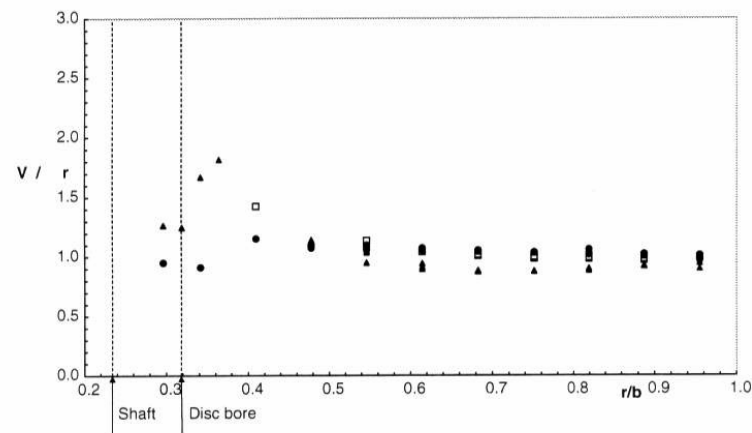


Fig. 8. Radial variation of normalised tangential velocity, $V_\phi/\Omega r$, with axial Reynolds number for tests with the wide annular gap ($d_0/b = 0.164$) and $Re_\phi \approx 10^6$.

| | $Re_\phi/10^6$ | $Re_z/10^5$ | Ro | $\beta\Delta T_{av}$ | $\beta\Delta T_{sh}$ |
|---|----------------|-------------|------|----------------------|----------------------|
| ● | 1.24 | 0.49 | 0.75 | 0.06 | 0.17 |
| □ | 1.31 | 0.97 | 1.41 | 0.05 | 0.16 |
| ▲ | 1.14 | 1.75 | 2.84 | 0.05 | 0.16 |

from the shroud can show a behaviour that is consistent with free convection and at the same time the flow structure in the inner part of the cavity can show a behaviour that is consistent with forced convection.

The dependence noted above, of the dimensionless tangential velocity on the Rossby number, is qualified in Fig. 9. This shows measured values of $V_\phi/\Omega r$ with r/b for a number of tests with the wide annular gap ($d_h/b = 0.164$) and also for the narrow annular gap ($d_h/b = 0.092$) between the disc bore and shaft. For $d_h/b = 0.164$, the tests were carried out at $Re_\phi = 0.49 \times 10^6$, 2×10^6 and 4×10^6 with $Re_z = 0.6 \times 10^5$, 10^5 and 1.8×10^5 . For $d_h/b = 0.092$, the tests were carried out at $Re_\phi = 10^6$, 1.5×10^6 and 2×10^6 with $Re_z = 0.8 \times 10^5$, 10^5 and 1.5×10^5 . For the wide annular gap, the comments made above can be observed – increasing the Rossby number increases the peak value of $V_\phi/\Omega r$ in the inlet region. Further into the cavity, the Rossby number has little influence on the values of $V_\phi/\Omega r$ and solid body rotation occurs. There certainly appears to be a marked difference between the two sets of results for the different annular gaps between shaft and disc bore. For the narrow annular gap ($d_h/b = 0.092$) the values of $V_\phi/\Omega r$ are seen to be lower than solid body rotation and increase to $V_\phi/\Omega r \approx 1$, further into the cavity. Since the annular gap between shaft and disc bore differs by a factor of almost two, it is not unreasonable to expect that this does have an influence on the flow inside the cavity. In this case reducing the annular gap between the shaft and disc bore attenuates the influence of the throughflow on the tangential velocity for $r/b < 0.6$.

Some support for this can be found in the measurements of Owen and Powell (2004) in a rig with an annular gap of $d_h/b = 0.026$ ($r_s = 145.2$ mm, $a = 150$ mm, $b = 371$ mm, $s = 75.2$ mm). These also show the behaviour noted here

in the narrow gap data. The detailed measurements of Pincombe (1983) display two different types of behaviour (free and forced vortex flow). For $G = 0.53$, these differences were attributed to a change from Mode 1a to Mode 2a (non-axisymmetric to axisymmetric as reported by Farthing et al.) vortex breakdown. This could be of direct relevance to the present measurements where it is suggested that the geometry of the cavity affects the mechanism of vortex breakdown which in turn affects the tangential velocity distribution inside the cavity.

4.3. Radial velocity component

Simultaneous measurements of the radial and tangential components of velocity were obtained from the test rig with the wider annular gap between shaft and disc bore ($d_h/b = 0.164$). Optical access was gained through windows in Discs 3 and 4 and the stationary outer casing. Since the tangential component of velocity can be two orders of magnitude greater than the radial component, any misalignment of the LDA beams would result in a portion of the tangential velocity biasing the measurement of radial velocity. The angle of the plane of the pair of beams used to measure the radial component of velocity was measured with respect to a vertical line passing through the centreline of the test rig. This angle was measured using a precision spirit level and the uncertainty in this measurement is estimated to be $\pm 0.015^\circ$. For the largest value of tangential velocity measured (50 m/s), this is equivalent to introducing an uncertainty of 0.014 m/s in the measured value of radial velocity. Typically several thousand simultaneous measurements of radial and tangential velocities were made at each location in the test rig. These locations covered the inside of Cavities 3 and 4, over the range $(0.067 \leq z/l \leq 0.946)$ and $(0.565 \leq r/b \leq 0.925)$.

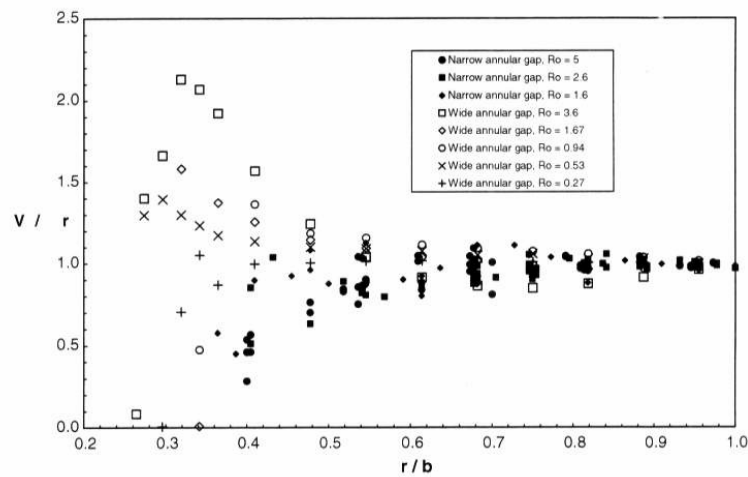


Fig. 9. Radial variation of normalised tangential velocity, $V_\phi/\Omega r$, with Rossby number – comparison between results for $d_h/b = 0.164$ and $d_h/b = 0.092$.

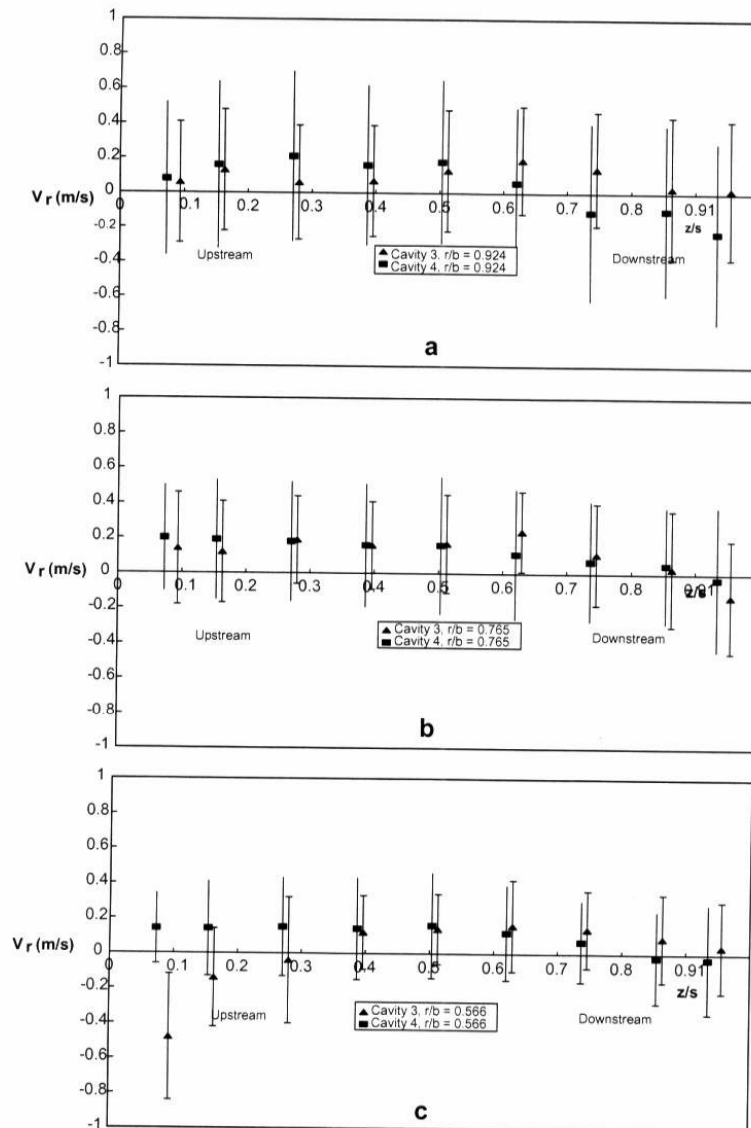


Fig. 10. Axial and radial variations of radial velocity for: $d_h/b = 0.164$, $Re_\phi = 0.4 \times 10^6$, $Re_z = 1.57 \times 10^5$, $\beta\Delta T_{av} = 0.07$, $\beta\Delta T_{sh} = 0.20$ and $Ro = 7.54$. Bars show RMS variation in velocity. (a) $r/b = 0.924$; (b) $r/b = 0.765$; (c) $r/b = 0.566$.

Fig. 10 shows the variation of the average values (each point is obtained from an average of all the measured instantaneous values at each radial and axial location) of radial velocity component with radial and axial locations in both Cavity 3 and Cavity 4. The flow conditions are $Re_\phi = 4 \times 10^5$, $Re_z = 1.57 \times 10^5$, $Ro = 7.54$, which is the largest value of Rossby number tested. A positive value of radial velocity indicates a radial outflow and a negative value, radial inflow. As can be seen, the magnitude of the

velocities is generally low, <0.3 m/s, which is less than 1% of the bulk average axial velocity and about 2% of the tangential velocity at $r = b$. For these conditions, the uncertainty in radial velocity due to uncertainty in probe angle is estimated to be less than 0.01 m/s and the size of the data points is chosen to reflect this. Also shown are the RMS values obtained for each measurement. The magnitude of all the RMS values exceeds the magnitude of the measured value of radial velocity. This is in part due to the

periodic variations in radial velocity referred to above and in part due to turbulent fluctuations in the radial velocity.

In Cavity 3 the measurements show evidence of a circulation generated by the throughflow. This is seen by radial outflow at $r/b = 0.566$ (Fig. 10c) in the central region of Cavity 3 and radial inflow close to the upstream disc surface. At $r/b = 0.765$ (Fig. 10b), there appears to be outflow close to the upstream disc surface and in the central region, with inflow adjacent to the downstream disc. This reversal could be a secondary contra-rotating circulation generated by the circulation formed at the inner radius. At the outer radius (Fig. 10a) of Cavity 3 the magnitude of radial velocity is reduced and what motion there is appears to be restricted to the central part ($0.15 < z/s < 0.8$) of the cavity. In the main there appears to be no significant difference between the measurements in Cavities 3 and 4. One exception is at $r/b = 0.566$, where the measurements in Cavity 4 show radial outflow adjacent to the upstream disc. It is quite likely that exit conditions do influence the flow in the cavity (see Tucker and Long (1996), for example), so this difference may be due to a difference in exit geometry. The axial length of the passage between disc bore and shaft is considerably shorter at the exit of Cavity 3 compared with the exit of Cavity 4. The flow leaving Cavity 4 also does not experience a further cavity downstream and this too may affect the flow in the inner region of that cavity. Away from the disc surfaces, in the centre of each cavity, there is little variation of radial velocity with radius. The value of bulk average velocity for this test is $W = 36.2$ m/s using a value of $V_r = \pm 0.2$ m/s as being characteristic of the results shown here gives a non-dimensional velocity of $V_r/W = \pm 0.005$. Although the value of Rossby number tested here ($Ro = 7.54$) is significantly lower than the data of Farthing et al., their data for $G = 0.267$ and $Ro = 80$ give $V_r/W \approx \pm 0.01$. It should be noted that application of mass continuity would imply that at any radial location the outflow should balance the inflow. This does not appear to be the case when looking at the individual data points. However, when the range of the RMS values is taken into consideration, then continuity does appear to be satisfied. Although not shown here, the values of radial velocity obtained from other test conditions ($Re_\phi = 1.9 \times 10^6$, $Re_z = 0.52 \times 10^5$; $Re_\phi = 0.39 \times 10^6$, $Re_z = 0.6 \times 10^5$ and $Re_\phi = 2 \times 10^6$, $Re_z = 1.65 \times 10^5$) show similar behaviour to those in Fig. 10. Decreasing the axial Reynolds number has very little effect on the magnitude of V_r whereas increasing the rotational Reynolds number appears to increase the magnitude of V_r . The magnitude of the radial velocity component remains about two orders of magnitude less than the tangential component.

4.4. Spectral analysis of the tangential component of velocity

As was noted in Section 2, the current model for the flow in a heated rotating cavity with axial throughflow comprises pairs of counter-rotating vortices in the r - ϕ plane (Fig. 3). There is direct experimental evidence of this (see

Farthing et al.) and also indirect evidence from the frequency spectrum obtained from velocity measurements (see Owen and Powell). It is therefore of some interest to carry out a similar frequency-spectrum analysis of the present data to see if this is consistent with this model of the heated flow structure.

This was achieved by obtaining a fast Fourier transform of a set of measured values of tangential velocity. Each measured value was obtained by averaging the appropriate number of instantaneous measurements over each pass of one of the windows. Fig. 11 shows the frequency spectrum obtained from the tangential velocity measurements for $Re_\phi = 4 \times 10^5$, $Re_z = 16 \times 10^4$, $Ro = 7.4$ in the mid-plane ($z/s = 0.5$) of Cavity 3 at a radial location of $r/b = 0.765$. There is a clear dominant spike in the spectrum at 22 Hz. The rotational speed of the cavity in this test is 655 rev/min, so the spike occurs at just over twice the rotational speed. It is possible to explain this behaviour in the context of the current model of the heated flow structure. For a uniform flow rotating at a constant speed of $V_\phi = \Omega r$, there is no cyclic variation of velocity in the absence of any vortex pairs. So this would produce a spectrum without any clearly defined peaks. The presence of white noise in this flow, generated by turbulence for example, would be shown as a background of closely spaced peaks and this is also evident in Fig. 11 either side of the main peak at 22 Hz. The velocity measurements from a single pair of contra-rotating vortices in a flow rotating at $V_\phi = \Omega r$ would produce, if the measurement point were located towards the outer part of the vortex structure, a velocity signal that increased above Ωr as the cyclonic vortex passed over the measurement point and then decreased below Ωr as the anti-cyclonic vortex passed over the measurement point. Each rotation of the whole flow would produce a signal with the frequency of approximately the rotational speed of the flow. For two vortex pairs, the dominant frequency would be twice the rotational speed; three pairs, three times etc. Naturally this simple explanation assumes the vortices are of identical shape and size and have identical magnitudes of rotational speed. A real flow with variations in these variables will create differences in the spectrum which at this stage are difficult to interpret. It would therefore

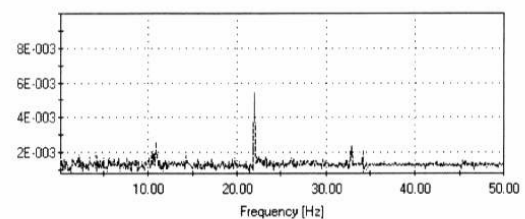


Fig. 11. Frequency Spectrum obtained from tangential velocity measurements: $d_0/b = 0.164$, $Re_\phi = 0.4 \times 10^6$, $Re_z = 1.57 \times 10^5$, $\beta\Delta T_{av} = 0.07$, $\beta\Delta T_{sh} = 0.20$ and $Ro = 7.4$ in the mid-plane ($z/s = 0.5$) of Cavity 3 at a radial location of $r/b = 0.765$ (rotational speed of cavity = 10.92 Hz).

appear that the spectrum shown in Fig. 11 is consistent with two pairs of contra-rotating vortices.

Although not shown here analysis of the frequency spectra obtained at other axial and radial locations and other values of rotational and axial Reynolds numbers is broadly in agreement with these observations. In some cases there is not one single dominant spike in the spectra but two or even three. These occur at frequencies that are harmonics of a fundamental. Recent experimental work and CFD simulation (Bohn et al., 2006 and Owen et al., 2006) has shown that it is possible for the cellular structure to change with time: one pair of vortices can evolve into two pairs of vortices and then into three. So, it may be possible that the appearance of multiple spikes in the frequency spectra indicates this behaviour. An alternative explanation is that the flow behaviour is not adequately represented by a single harmonic. In principle it should be possible to examine the spectral data to see if and when a spike at one frequency changes to a multiple of that frequency, indicating a transition from say two to three pairs of vortices. However, in order to build up a frequency spectrum tens of thousands of data points are required. To investigate how this changes over time would require data sets much larger than those collected from this experiment.

5. Conclusions

This paper has presented measurements of axial, radial and tangential velocities carried out in a heated multiple cavity test rig with axial throughflow. This test rig is designed to be a 70% full scale replica of a high pressure compressor internal air system. The main interest is the flow in the cavities formed between the adjacent heated disc surfaces and the interaction of this flow with the central throughflow of cooling air. Two different variants of the test rig have been investigated, these differ by the size of the annular gap between the central shaft and the disc bores. For the narrow annular gap $d_h/b = 0.092$ and for the wide annular gap $d_h/b = 0.164$.

Measurements were made inside the rig for a range of non-dimensional conditions that are representative of those found inside an engine (values of rotational Reynolds number up to $Re_\phi = 4 \times 10^6$ and axial Reynolds numbers up to $Re_z = 1.8 \times 10^5$). However, the Grashof number based on the shroud conditions ($Re_\phi^2 (\beta \Delta T)_{sh}$) is at least an order of magnitude lower. Measurements of the tangential and axial velocities inside the cavities were made using a two component LDA system, with optical access being provided through the shaft. Additional measurements of the tangential and radial components of velocity were obtained in for the wide annular gap with optical access being gained via windows in the outer casing and the disc surfaces.

The main conclusions from this work are summarised below:

Although it was not possible to make measurements inside the disc boundary layers, the tangential velocity does not appear to vary across the axial width of a cavity. The

axial velocities inside the cavity are close to zero. For the wider annular gap between shaft and disc bore ($d_h/b = 0.164$), the maximum value of $V_\phi/\Omega r$ occurs close to the inner radius of the cavity and its magnitude increases with increasing Rossby number. The non-dimensional tangential velocity decreases from this maximum value to solid body rotation, $V_\phi/\Omega r = 1$, as $r/b > 0.6$. For $d_h/b = 0.092$, the non-dimensional tangential velocities do not show a maximum until solid body rotation is achieved near the outer radius. This difference in radial variation of tangential velocity is attributed to a difference in the influence of the central throughflow on the flow inside the rotating cavities; and a narrow annular gap attenuates this interaction. It appears likely that the behaviour of the jet and hence the mode of vortex breakdown is affected by this change in geometry.

The values of radial velocity are typically two orders of magnitude lower than the tangential component and comparable to the relative tangential velocity. It is difficult to interpret the overall behaviour of the flow from these measurements because the flow is unsteady and an average is misleading. However there is evidence to suggest the existence of circulating flow within the cavities. There also appears to be a difference in the flow structure between Cavity 3 and Cavity 4 which could be attributed to a difference in exit conditions. An analysis of the frequency spectrum obtained from the tangential velocity measurements shows clear evidence of periodicity in the flow structure. This is linked to the existence of pairs of contra-rotating vortices which is consistent with the current understanding of the heated flow structure.

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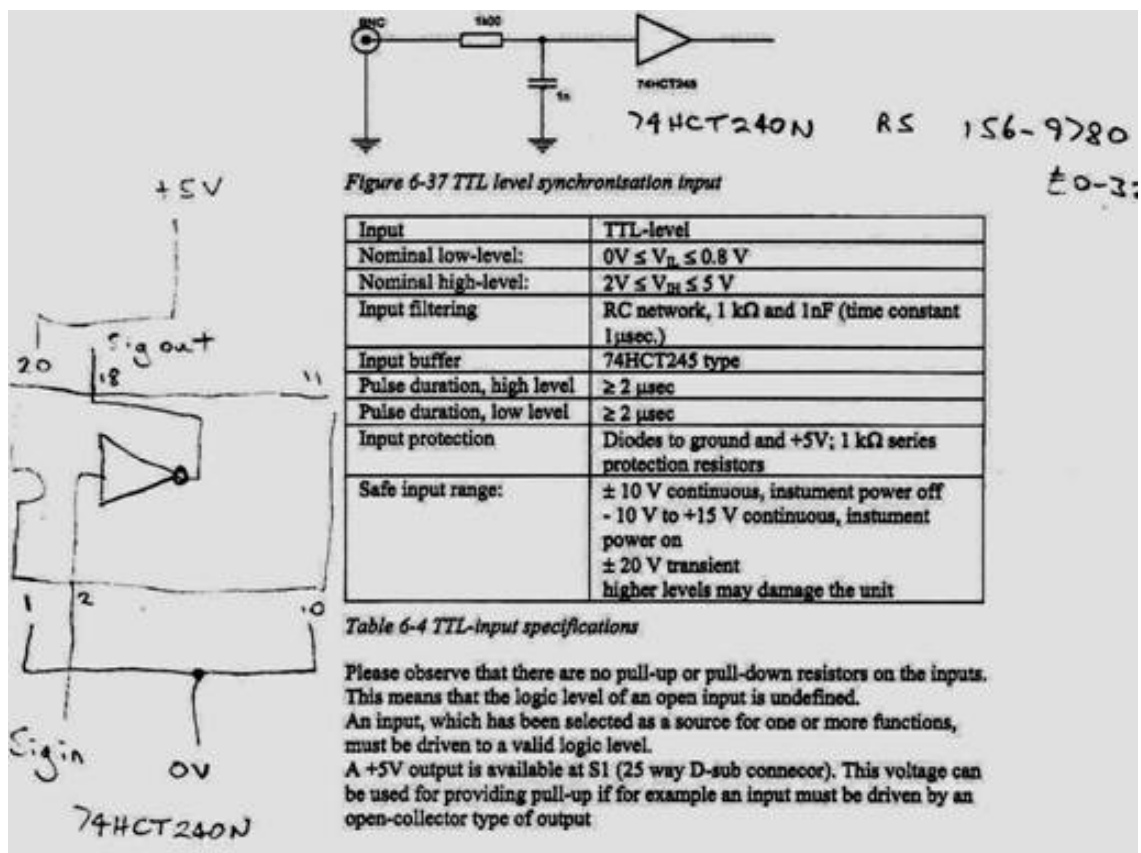
Appendix 3.1: ORION data logger channels connections

| Thermocouple or Data logger channel number | Location | Thermocouple or Data logger channel number | Location |
|--|---------------|--|------------------------|
| 1 | Rotor | 38 | Annulus (Air) |
| 2 | Rotor | 39 | Annulus (Air) |
| 3 | Rotor | 40 | Annulus (Air) |
| 4 | Rotor | 41 | Annulus (Air) |
| 5 | Rotor | 42 | Annulus (Air) |
| 6 | Rotor | 43 | Annulus (Air) |
| 7 | Rotor | 44 | Casing (inner surface) |
| 8 | Rotor | 45 | Casing (inner surface) |
| 9 | Rotor | 46 | Casing (inner surface) |
| 10 | Rotor | 47 | Casing (inner surface) |
| 11 | Rotor | 48 | Casing (inner surface) |
| 12 | Rotor | 49 | Casing (inner surface) |
| 13 | Rotor | 50 | Casing (outer surface) |
| 14 | Rotor | 51 | Casing (outer surface) |
| 15 | Rotor | 52 | Casing (outer surface) |
| 16 | Rotor | 53 | Casing (outer surface) |
| 17 | Rotor | 54 | Casing (outer surface) |
| 18 | Rotor | 55 | Casing (outer surface) |
| 19 | Rotor | 56 | Orifice (inlet) |
| 20 | Rotor | 57 | Orifice (outlet) |
| 21 | Rotor | 58 | Heater 1 |
| 22 | Rotor | 59 | Heater 2 |
| 23 | Rotor | 60 | Rotor SRU |
| 24 | Rotor | 61 | Ambient |
| 25 | Shaft | 62 | Sensor 1 |
| 26 | Shaft | 63 | Sensor 2 |
| 27 | Shaft | 64 | Rotor speed (rev/min) |
| 28 | Shaft | 65 | Pressure Trsdcr 1 (V) |
| 29 | Shaft | 66 | Pressure Trsdcr 2 (V) |
| 30 | Shaft | 67 | Pressure Trsdcr 3 (V) |
| 31 | Shaft | 68 | Mass flow rate (kg/s) |
| 32 | Inlet (Air) | 69 | Re_{ϕ} |
| 33 | Inlet (Air) | 70 | Re_z |
| 34 | Outlet (Air) | 71 | Ro |
| 35 | Outlet (Air) | | |
| 36 | Annulus (Air) | | |
| 37 | Annulus (Air) | | |

Appendix 3.2: Scanivalve data logger channels connections

| Channel number | Location |
|----------------|------------------------|
| 0 | Reference |
| 3 | Air IN O/P upstream |
| 4 | Air IN O/P downstream |
| 5 | Air OUT O/P upstream |
| 6 | Air OUT O/P downstream |
| 10 | Shaft upstream |
| 11 | Shaft downstream |
| 12 | Inlet pressure |
| 13 | Outlet pressure |
| 15 | Heater 1 |
| 16 | Heater 2 |

Appendix 3.3: Encoder driver connections



Annotated connections diagram from Dantec (2004)

Appendix 3.4: LDA traverse alignment checklist

TRAVERSE SET-UP:

Start laser at low power (0.25W) according to Procedure.

Bring control computer (+ signal and control cables) closer for convenience

Connect and set-up control boxes + computer

Run BSA Flow Software on control computer

New project, F/P BSA application, connect (check connection with on-line LED on processor)

The laser probe is fitted in the traverse tube from the back and slotted in the support ring 1cm out to allow cleaning.

Blow surface of probe optics with dry air canister. !!Hold can UP to avoid blowing liquid.

Clean optic by applying special cloth. Press uniformly and very gently, move in steady and slow motion out. Turn pad on clean area and start again. REMARK: There is 1 scratch in the optic that was already there.

Align optic with tube surface then lock with circumferential screws.

Unlock upper stepper motor to allow access of probe tube in its holding support. Align probe tube with surface of back bracket (rough axial position). Re-screw stepper motor.

Optics cleaning:

Optical surfaces are coated with a non-reflective film that is really thin and fragile. DO NOT touch with hands directly, or scratch in any way.

Blow surface with dry air canister (keep canister up to avoid blowing liquid)

Take optical wash pad and scrape lightly with light pressure in a slow steady motion

Turn pad over and do again

For access and cleaning of disc windows inside the rig, a pad support is employed. The cleaning is monitored visually through the opening of the external casing window (removed).

Mirror cleaning (NOTE: Mirror Extremely Fragile!!):

One of the last thing to be installed. An extremely clean mirror is essential for measurement, scratches are easy to obtain. Same method of cleaning as optics is required, with even more care (sit at table to avoid accidental drops).

Remove little fixating screws.

Without touching it, insert mirror in tube (peg for orientation), and screw the mirror in tube.

Insert mirror tube a bit in shaft after having pulled traverse at max back.

Insert mirror tube in front bracket and align with peg and back surface of bracket.

Orientation of the beams:

The pairs of beams need to be referenced perpendicularly to the geometry of the rig.

Pull back totally the traverse system, cover mirror with piece of paper to avoid accidental contamination.

Install special alignment support on casing.

Sit on rig's ground frame, Goggles ON.

Open probe beams shutters.

Look (with goggles) to target and turn mirror tube to make coincide laser dots and perpendicular lines (precision: as much as it gets).

Fix the mirror tube in support.

Axial and Radial position calibration:

Launch BSA F/P Software.

In properties window (for LDA1 or LDA2), adjust High Voltage at low value first and gradually increase not to overload the photomultiplier.

Set Data collection Type to Continuous instead of burst when calibrating the traverse system.

In Project Explorer, add histogram + 2D plot + list (raw data) objects

Right-click main window to access system monitor, allowing setting the signals according to the beam position for the traverse set-up.

Remove the traverse blocker before operating the traverse

Radial location: Adjust traverse radial position to get maximum signal on the reference plane. If Anode Current indicator gets >1, reduce High Voltage on LDA settings.

The validation indicators get to ~100% when the surface is reached. The radial location is therefore known. Redo with another radial reference to confirm radial calibration.

Axial location: Move along X from surface, reading OK of radial position, until one of the beam pair gets crossed (getting near). Once the first signal is lost, try to get exact location of when second signal is lost (axial location found). Redo with another reference plane to confirm axial calibration.

When the axial and radial locations are calibrated, check repeatability of positioning on different cavity features from design dimensions of the rig.

The stepper motors positions value indicator can be zeroed and a blocker can be put on the axial traverse at max X position. **(Remember to remove the traverse blocker if traverse manipulations are to be done later)**

Re-calibration/Check:

Put traverse in position+ remove blocker

Get control computer closer to rig (convenience)

Log in Bungalow (not to forget the capital B)

Run BSA, open B3setup.Ida

Connect to processor (right-click processor)

Connect to traverse

Right-click traverse system to open traverse dialog box (allows manual operation of stepper motors)

Switch transmitter to through position + open fibre shutters

Open AutoCad drawing

Run system monitor to check whereabouts the beams are

Re-twitch to adjust origin position + reset axis (precision $\pm 0.5\text{mm}$)

Check X position accuracy by reaching next cob

Optimise Y position + move on shroud to check

On exit save changes

Turn off computer + control boxes

Shut fibres

Transmitter half-way

Appendix 4.1: Laser operation checklist

LASER OPERATION:

LASER STARTUP:

- WATER: ON (Open water valve to black mark). ⚠️ Check Water Filter ~6Months
- POWER MAINS: Isolator ON
- DOOR: Locked
- LASER POWER: Minimum, Power controls in Current mode set at 15-20 Amps
- LASER SHUTTER: SHUT, Diaphragm Minimum
- PLASMA: Key switch ON (15s wait until laser emits)
- FIBRE SHUTTERS: 4xShut
- TRANSMITTER: Half-way position, Beam way CLEAR
- LASER SHUTTER: OPEN
- WARM UP: Power increased to setting (W), log start time, Warm-up time 30 min
- COMPUTER: ON, Check connection signal cables (Ethernet in LDA slot, COM in top COM plug on computer)
- PROCESSOR: ON, (Ready lamp ON)
- TRAVERSE: Position OK, traverse controller ON
- BSA F/P: Connected (Processor On-line)

LASER SWITCH OFF:

- COMPUTER: OFF
- PROCESSOR: OFF
- TRAVERSE: OFF
- LASER POWER: Minimum
- FIBRE SHUTTERS: 4*shut
- TRANSMITTER: Half-way position
- LASER SHUTTER: Shut
- PLASMA: OFF
- DUST COVER: On transmitter
- COOLING: Wait for cooling (check with water pipes)
- POWER MAINS: Isolator OFF
- WATER: OFF
- LOG: Functioning duration and power used

Laser power change:

- ⚠️ : When changing power of laser, turn fibres OFF and allow 10 Minutes stabilisation

Appendix 4.2: Air supply checklist and air cooling diagram

HOWDEN and Water Recirculation:

Single stage, screw type compressor driven by a 275kW electric motor via a 3.95:1 gearbox. Can supply up to 1.1kg/s of air at 4 bar absolute and 200°C.

When the Howden hasn't been run for ~3 or more weeks, the water temperature between the roof and underground tanks needs to be circulated to avoid too important discrepancies of temperature that would produce some irregularities of the cooling (mixing difficult from the baffles inside underground tank especially, resulting in discrepancies of temperatures during run as the cooling flow uniformises). To avoid, turn on recirculation pump for 3-4 hours the afternoon before running tests.

WATER RECIRCULATION:

●**VALVES**: 1.Roof Tank Main Valve, 2.Heat Exchanger Inlet FULL OPEN, 3.Flow Control valves OPEN TO MARK

●**PUMP**: ON (A or B)

SAFETY AIR SUPPLY VALVE:

●**AIR SUPPLY**: ON

●**OPERATIVE**: CHECK

HOWDEN START-UP:

●**HOWDEN WATER COOLING PUMP**: ON (on control panel), CHECK Water Pressure (Downstairs) – 5Psi

●**MAIN POWER ISOLATOR**: ON (Downstairs), Howden double-doors semi locked

●**OIL PUMP**: ON (control panel), CHECK Oil pressure 12-14 Psi

●**PRESSURE CONTROL VALVE**: Check OFF-LOAD

●**CONTROL PANEL RESET**: All Lights ON

●**HOWDEN START**: Switch ON, Check started

DURING RUN: LOGBOOK

AT START: Time of Start + Finish
Initials of operators

EACH HOUR: Pressure + Temperature

HOWDEN SHUT DOWN:

●**PRESSURE CONTROL VALVE**: Turn hand wheel OFF-LOAD

●**HOWDEN STOP**: Switch OFF

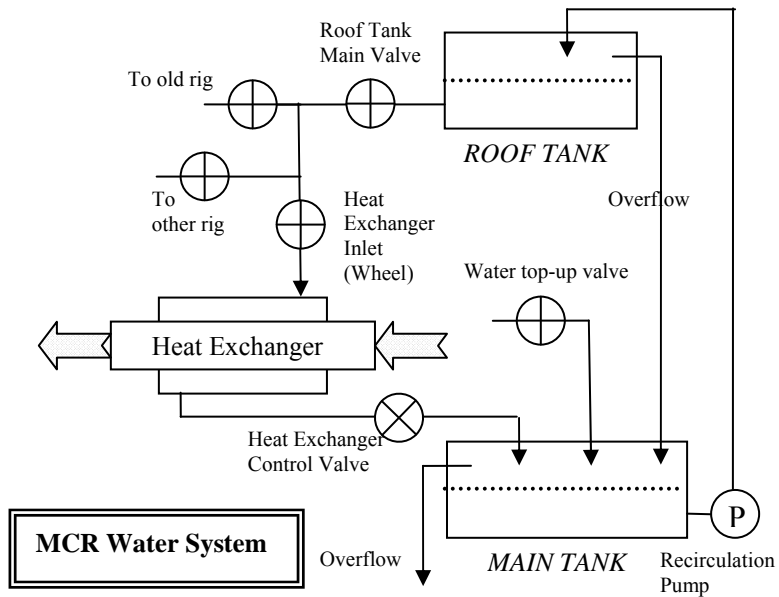
●**COOLING**: Leave Oil Pump, Water Recirculation, Water cooling pumps, ON, WAIT AN HOUR

●**OIL PUMP**: OFF

- HOWDEN WATER COOLING PUMP: OFF
- CONTROL PANEL: Switch OFF
- MAIN POWER ISOLATOR: OFF, Leave doors open
- WATER RECIRCULATION VALVES: 1.Flow Control, 2.Heat Exchanger Inlet, 3.Roof Tank Main Valve FULL SHUT

RIG AIR WATER COOLING SYSTEM:

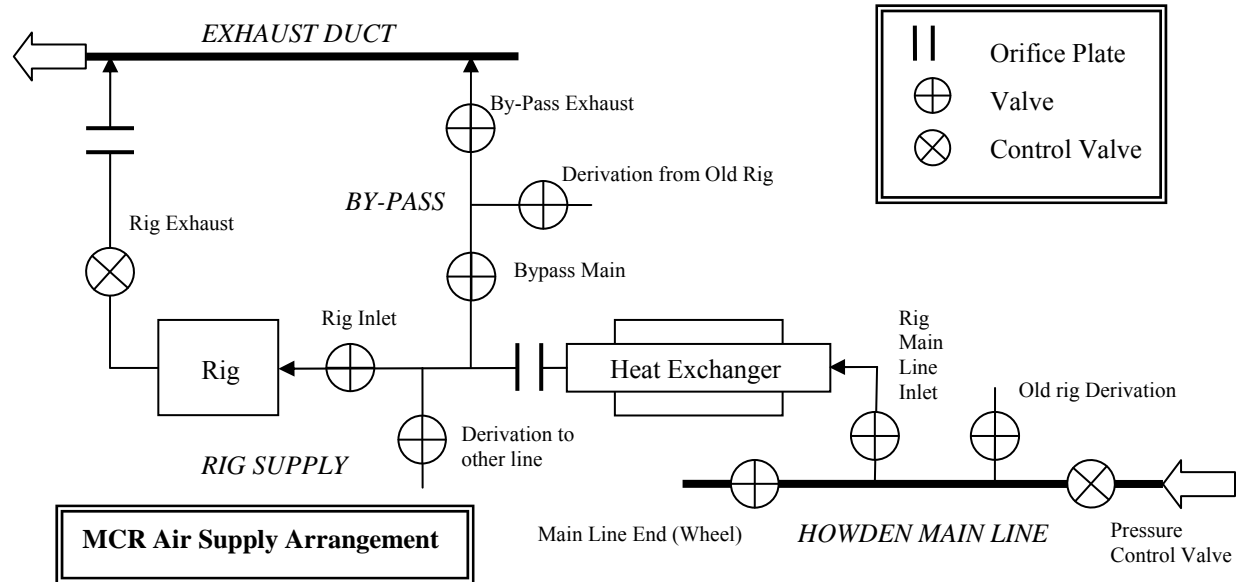
ARRANGEMENT DIAGRAM:



Appendix 4.3: Rig air supply checklist

RIG AIR SUPPLY:

ARRANGEMENT DIAGRAM:



PRE-RUN LINE CHECK:

HOWDEN MAIN LINE:

- OLD RIG DERIVATION: Always OFF
- RIG MAIN LINE INLET: Always ON
- MAIN LINE END (Wheel): Always OFF

RIG SUPPLY:

- DERIVATION TO OTHER LINE: Always OFF
- RIG INLET: OFF
- RIG EXHAUST: Full OPEN

BYPASS:

- BY-PASS MAIN: ON
- DERIVATION FROM OLD RIG: Always OFF
- BY-PASS EXHAUST: ON

HOWDEN SAFETY VALVE:

- AIR SUPPLY: ON
- OPERATION: CHECK

Appendix 4.4: Rig pre-start checklist

MULTI-CAVITY RIG

PRE START CHECK LIST

- 1 Water valves (roof tank / heat exchanger) - water pump (re-circulating)
- 2 Compressed air valves (sru, window cleaning)
- 3 Water for laser, laser 3-phase on, plasma on, set laser power
- 4 Air inlet valves (howden, by-pass, rig)
- 5 Oil tank valve & oil level
- 6 Power supplies, isolators, panel switches & pcs (laser, orion-scani) on
- 7 Signal processor, traverse controller on
- 8 Lab ventilation, main fans, oil tank centrifuge, hot air extractor valve
- 9 Howden water pump, howden isolator, howden oil pump → reset → start howden
- 10 Hydrovane on, atomiser pressurised (min 25 psi above rig working pressure)

Appendix 4.5: Air supply condition setting checklist

AIR SUPPLY RUN CONDITIONS SETTING:

- RIG BY-PASS: Double Check OPEN (Pre-Run checklist)
- START HOWDEN: (Howden checklist)
- PRESSURE CONTROL VALVE: Apply 10Psi, wait 10 minutes
- RIG EXHAUST VALVE: Double Check FULL OPEN
- RIG INLET VALVE: OPEN
- RIG BY-PASS: SHUT
- CONDENSATION: Check
- FLOW CONDITIONS SETTING: Increase pressure from Pressure Control Valve, Modify opening of Rig Exhaust Valve, until desired value of Rez obtained.

Appendix 4.6: Rig start checklist

MULTI-CAVITY RIG

STARTING PROCEDURE

- 1 Adjust water flow rate to achieve the required cooling air temperature (1st through by-pass and then in the rig)
- 2 Using downstream control valve & howden bypass valve pre-set internal rig pressure (min 10 psi)
- 3 Switch on - oil pump & tasc motor
- 4 Set rotor speed and rig pressure as required
- 5 Start blower and switch on heaters
- 6 Monitor vibration levels & fill in log sheet
- 7 Wait for thermal steady state

Appendix 4.7: Rig shut down checklist

MULTI-CAVITY RIG

SHUT DOWN PROCEDURE

- 1 Switch off heaters
- 2 Reduce rotor speed to 2000 rpm
- 3 Fully open air flow control valve (min 15 psi)
- 4 Let rotor rotate for 15 min after switching off heaters to aid cooling
- 5 Turn off tasc motor
- 6 When rotor speed falls below 500 rpm stop oil pump
- 7 When rotor has completely stopped shut down howden
- 8 Switch off blower
- 9 Reverse all pre-start conditions

Appendix 4.8: Read.bas listing

```

REM ****PROGRAM MCRread.bas***
REM written by A. Alexiou
REM program last modified 30/04/05 by N. Miché
REM This program reads the data files from the ORION and
SCANIVALVE
REM and writes them to files so that they can be used for further
processing
REM This program requires the modified scanivalve with added
parameters
REM data from build 2 or build 3 is automatically recognised and
processed
DECLARE SUB labyrinth (pup!, pdn!, tup!, r1!, cl!, lk!)
REM SUB "labyrinth" calculates the leakage through the various labyrinth
seals
REM in the rig according to Curzons (1991) RR TDR 90294
DECLARE SUB orifice (no, pin!, pout!, cin!, d1!, d!, mnl!)
REM SUB "orifice" calculates the mass flow rate in & out of the rig
REM according to BS 1042 for orifice plates
np = 71    'number of parameters supplied by the data logger program
sn = 100    'number of data logger scans
nc = 8      'number of pressures measured by Scanivalve
nsc = 3     'number of Scanivalve scans
DIM scan(sn)
DIM tm(sn)
DIM ChNo(np)
DIM frnm(sn)
DIM dlpr(3, sn)
DIM total(np), row$(28), aval(np), sq(np), valsd(np), header$(20)
DIM PR(nc, nsc), prtot(nc), avpr(nc), ptot(3), pav(3)
DIM value(71, sn)
REM pr?(psi for calculations, Pa for display and reading), temp?(C),
dor?(m), Dpipe?(m), rd?(m)
REM Read Orifice & Pipe Diameter
REM *****
READ dor1, Dpipe1, dor2, Dpipe2
DATA 0.055,0.075,0.055,0.075
REM Read Seal Radius and No of Fins
REM *****
READ rd1, cl1, rd2, cl2, rd3, cl3, rd4, cl4, rd5, cl5
DATA
0.1265,0.00005,0.0515,0.00005,0.108,0.0001,0.051,0.0001,0.029,0.000
1
CLS
REM Output File
REM *****
fileout$ = "G:\MCRread\test8.dat"
OPEN fileout$ FOR OUTPUT AS #4
REM INPUT FILES
REM *****
orion$ = "G:\MCRread\or36-1.dat"
scani$ = "G:\MCRread\testsc36.dat"
OPEN orion$ FOR INPUT AS #2
OPEN scani$ FOR INPUT AS #3
REM *****
REM Read Data from Orion
REM *****
c = 273.15
FOR k = 1 TO sn
LINE INPUT #2, ScanNo$
LINE INPUT #2, Times$
scan(k) = VAL(MID$(ScanNo$, 9))
tm(k) = VAL(MID$(Times$, 6, 6))
FOR j = 1 TO 64
INPUT #2, ChNo(j), value(j, k)
NEXT j
FOR j = 65 TO 67
INPUT #2, ChNo(j), value(j, k), dlpr(j - 64, k)
NEXT j
INPUT #2, ChNo(j), value(j, k), frnm(k)
FOR j = 69 TO 71
INPUT #2, ChNo(j), value(j, k)
NEXT j
LINE INPUT #2, emptyline$
NEXT k

ntms = sn
REM *** transient printing ***
'FOR k = 1 TO ntms
'PRINT #4, USING "#####.###"; tm(k);

'FOR j = 1 TO np - 1
'PRINT #4, USING "#####.###"; value(j, k);
'NEXT j
'PRINT #4, USING "#####.###"; value(np, k);
'PRINT #4, USING "###.###"; dlpr(1, k); dlpr(2, k); dlpr(3, k)
'NEXT k

REM *****
REM Calculate Average Values & Standard Deviation
REM *****
FOR k = 1 TO ntms
FOR i = 1 TO np
total(i) = total(i) + value(i, k)
NEXT i
NEXT k
REM ***** delete
FOR k = 1 TO ntms
FOR i = 1 TO 3
ptot(i) = ptot(i) + dlpr(i, k)
NEXT i
NEXT k
REM *****
FOR i = 1 TO np
aval(i) = total(i) / ntms
NEXT i
REM ***** delete
FOR i = 1 TO 3
pav(i) = ptot(i) / ntms
NEXT i
REM *****
FOR k = 1 TO ntms
FOR i = 1 TO np
sq(i) = sq(i) + (value(i, k) - aval(i)) ^ 2
NEXT i
NEXT k
FOR i = 1 TO np
valsd(i) = (sq(i) / (ntms - 1)) ^ .5
PRINT #4, i;
PRINT #4, USING "#####.###"; aval(i); valsd(i)
NEXT i
REM ***** delete
'FOR i = 1 TO 3
'PRINT #4, USING "###.###"; pav(i)
'PRINT USING "###.###"; pav(i)
'NEXT i
REM *****
REM Read Data from Scanivalve
REM *****
FOR i = 1 TO 20
LINE INPUT #3, header$(i)
NEXT i
REM buildn = VAL(MID$(header$(1), 1)) 'rig build number
REM prec = VAL(MID$(header$(6), 8)) 'weather station recorded
pressure in mbar
buildn = VAL(header$(1)) 'rig build number
prec = VAL(header$(6)) 'weather station recorded pressure in mbar
nn = 1
DO WHILE NOT EOF(3)
FOR i = 1 TO 13
LINE INPUT #3, row$(i)
NEXT i
PR(1, nn) = VAL(MID$(row$(2), 45, 11)) 'inlet orifice upstream
PR(2, nn) = VAL(MID$(row$(3), 9, 11)) 'inlet orifice downstream
PR(3, nn) = VAL(MID$(row$(3), 21, 11)) 'outlet orifice upstream
PR(4, nn) = VAL(MID$(row$(3), 33, 11)) 'outlet orifice downstream
PR(5, nn) = VAL(MID$(row$(5), 9, 11)) 'inlet pressure
PR(6, nn) = VAL(MID$(row$(5), 21, 11)) 'outlet pressure
REM *** Currently there is NO pressure tapping at the rig OUTLET !!!!***

```

```

PR(7, nn) = VAL(MID$(row$(4), 33, 11)) 'upstream cavity pressure
PR(8, nn) = VAL(MID$(row$(4), 45, 11)) 'downstream cavity pressure
nn = nn + 1
LOOP
isc = nn - 1
IF isc > nsc OR isc < nsc THEN
PRINT "Number of Scanivalve Scans is not Correct"
END IF
FOR k = 1 TO nsc
  FOR i = 1 TO 8
    prtot(i) = prtot(i) + PR(i, k)
  NEXT i
NEXT k
FOR i = 1 TO 8
  avpr(i) = prtot(i) / nsc
  PRINT #4, USING "###.###"; avpr(i)
NEXT i
REM Atmospheric pressure
REM -----
g = 9.80665 'm/s^2
aws = 252.6 'altitude of weather station in ft
afl = 242.5 'altitude of fluids lab in ft
rhoa = (prec * 100) / (287 * (aval(61) + 273.15))
patm = -rhoa * g * (afl - aws) * .3048 + (prec * 100) 'patm in Pa
REM Orifice Plate Temperatures
REM -----
temp1 = aval(56)
temp2 = aval(57)
REM Mass Flow Rate in & Out of the Rig
REM -----
CALL orifice(1, avpr(1), avpr(2), temp1, dor1, Dpipe1, min)
IF avpr(3) > avpr(4) THEN
CALL orifice(2, avpr(3), avpr(4), temp2, dor2, Dpipe2, mout)
END IF
REM Inlet & Outlet Air Temperatures in C
REM -----
tin = (aval(32) + aval(33)) / 2
tout = (aval(34) + aval(35)) / 2
REM Find (1/rho)
REM -----
inrho = 287 * (tin + c) / (avpr(5) * 6896.55 + patm)
REM find pressure drop thru the casing holes
(Dp=(m/(CD*A))^2*(1/(2*rho)))
REM -----
prdrop = ((min / .001508) ^ 2) * (.5 * inrho) * .000145
pupin = avpr(5) - prdrop 'pressure upstream of lab seals at rotor inlet
pcav = avpr(7) 'Cavity Pressure
pgauge = 0 'gauge
CALL labyrinth(pupin, pgauge, tin, rd1, cl1, lk1)
CALL labyrinth(pupin, pgauge, tin, rd2, cl2, lk2)
CALL labyrinth(avpr(6), pgauge, tout, rd3, cl3, lk3)
CALL labyrinth(avpr(6), pgauge, tout, rd4, cl4, lk4)
CALL labyrinth(pcav, pgauge, aval(31), rd5, cl5, lk5)
leak = lk1 + lk2 + lk3 + lk4 + lk5
diff = mout + leak - min
m1 = (min - (lk1 + lk2))
m2 = mout + lk3 + lk4 + lk5
PRINT USING "###.###"; m1; m2; (lk1 + lk2)
PRINT USING "###.###"; diff
PRINT USING "###.###"; lk1; lk2; lk3; lk4; lk5
PRINT #4, USING "###.###"; m1
PRINT #4, USING "###.###"; m2
PRINT #4, USING "###.###"; lk1 + lk2
PRINT #4, "-----"
REM CALCULATE REZ and REF
REM -----
IF buildn = 3 THEN
REM data for build 3
rb = .22
ra = .0701
rs = .052
END IF
IF buildn = 2 THEN
REM data for build 2
rb = .22
ra = .0701
rs = .06
END IF
PI = 3.141592654#

```

```

speed = aval(64) * PI / 30
te = tin + c
mi = (1.46E-06 * te ^ 1.5) / (110.33 + te)
pcv = avpr(7) * 6894.8 + patm
rho = pcv / (287 * te)
ni = mi / rho
Ref = speed * rb ^ 2 / (ni * 1000000!)
v = m1 / (rho * (ra ^ 2 - rs ^ 2) * PI)
PRINT pcv, rho, v, speed
'Rez = 2 * (ra - rs) * v / (ni * 100000!)
Rez = 2 * m1 / (PI * (ra + rs) * mi * 100000!)
Ro = v / (speed * ra)
PRINT #4, buildn
PRINT #4, header$(2)
PRINT #4, header$(3)
PRINT #4, header$(4)
PRINT #4, header$(5)
PRINT #4, patm
PRINT #4, header$(7)
PRINT #4, header$(8)
PRINT #4, header$(9)
PRINT #4, header$(10)
PRINT #4, "****_***_****"
PRINT
PRINT "Rotational Speed = "; USING "#####"; aval(64);
PRINT " rev/min"
PRINT "Mass Flow Rate In = "; USING "#.###"; m1;
PRINT " kg/s"
PRINT "Cavity Pressure = "; USING "#.###"; pcv / 100000!;
PRINT " bar"
PRINT "Inlet Temperature = "; USING "###.##"; te;
PRINT " K"
PRINT "Max Hot Air Temperature = "; USING "###.##"; (aval(40) +
aval(41)) / 2 + c;
PRINT " K"
PRINT "Axial Reynolds No = "; USING "###.###"; Rez;
PRINT " E5"
PRINT "Rotational Reynolds No = "; USING "###.###"; Ref;
PRINT " E6"
PRINT "Rossby No = "; USING "###.###"; Ro
PRINT "Max Buoyancy = "; USING "###.###"; (aval(12) - tin) / te
PRINT "Axial Velocity = "; USING "###.###"; v
PRINT "Tangential Velocity = "; USING "###.###"; speed * .0701
PRINT
PRINT USING "#####"; avpr(1); pav(1)
PRINT USING "#####"; avpr(2); pav(2)
PRINT USING "#####"; avpr(7); pav(3)
PRINT
PRINT aval(65), aval(66), aval(67)
CLOSE #2
CLOSE #3
CLOSE #1
CLOSE #4
END
SUB labyrinth (pup, pdn, tup, r1, cl, lk)
PI = 3.141592654#

ptc = .002
ht = .002
nf = 7
REM patm = 14.7 'check from barometer
Pu = pup + patm / 6894.8
pd = pdn + patm / 6894.8
PR = pd / Pu
Tt = tup + 273.15
area = PI * ((r1 + cl) ^ 2 - r1 ^ 2) * 1550
gam = 1.4
lam = 1.2
r = 2.984 '(lbf/lb)^2/K - Gas Constant
k = (1 - EXP(-.054 * (ptc / cl))) * (nf - 1) + 1
'X1 = 1.1844 - .0242 * LOG(GAM)
X1 = 1.1808637#
X2 = .2491 - .1562 * ((gam - 1) ^ .937)
X3 = .588 + .3046 * ((gam - 1) ^ 1.153)
X4 = .719 + .1229 * ((gam - 1) ^ 4.178)
X5 = .0218 + .002317 * ((gam - 1) ^ .725)
a = (X1 - X2 * EXP(-X3 * (k - 1) ^ X4)) / k ^ X5
N = k / a
b = SQR(1 + 2 * gam * (1 + gam) * N)

```

```

F1 = 1 - (.0553 * (SIN((3 * PI) / (2 + (ptc / ht))) ^ 2))
F2 = 1
Cd = .92 * F1 * F2
Qc = Cd * SQR((b - 1) / (r * N * (b + 1)))
alpha = (1 - lam * a) / Qc ^ 2
beta = lam * a / 2
delta = ((1 - PR) * (PR * (b + 1) + b - 3) * Cd ^ 2) / (N * r * (b - 1))
q = SQR((SQR(beta ^ 2 + alpha * delta) - beta) / alpha)
lk = area * Pu * q / SQR(Ti) * .454
END SUB
SUB orifice (no, pin, pout, cin, d1, d, mn)
REM   pin (psi), pout (psi), cin (C), d1 (m), D (m)
REM   Flange taps
        PI = 3.141592654#
        k = 1.4
REM   patm = 14.7   'check from barometer
REM   k=isentropic index (cp/cv)=1.4 for air
REM   mu=dynamic viscosity
REM   Converting psi to Pa & gauge pressure to absolute
        p1 = pin * 6894.8 + patm
        p2 = pout * 6894.8 + patm
REM   Converting t1 to Kelvin
        t1 = cin + 273
REM   p1=upstream orifice pressure
REM   p2=downstream orifice pressure
REM   t1=upstream fluid temperature
REM   d1=orifice plate diameter
REM   D=pipe diameter
        b = d1 / d
        a = PI * (d ^ 2) / 4!
        E = (1! - (b ^ 4)) ^ (-.5)
        dp = p1 - p2
        ex = 1! - ((.41 + (.35 * (b ^ 4))) * (dp / (k * p1)))
        rho = p1 / (287! * t1)
REM   b=diameter ratio
REM   A=pipe diameter
REM   E=velocity approach factor
REM   dp=pressure differential
REM   ex=expansion factor
REM   rho=density (p=rho*R*T) (ideal gas equation)

```

```

REM   mu calculated from Sutherlands law
        mu = (1.46E-06 * (t1 ^ 1.5)) / (t1 + 110.33)
REM   Guess a value for the pipe velocity u (m/s)
        u = 1!
REM   m=mass flow rate (kg/s)
REM   Red=pipe Reynolds number.
REM   C=discharge coefficient
REM   l1=0.0254/D, l2d=0.0254/D for flange tappings.
        l1 = .0254 / d
        l2d = .0254 / d
REM   mn=recalculated mass flow rate.
REM   tol=tolerance for convergence
        TOL = .00001
REM   Iterative loop for mass flow rate
        i = 0
TARGET1:
        IF i > 0 THEN u = mn / (rho * a)
        m = rho * u * a
        Red = rho * u * d / mu
        x = .5959 + (.0312 * (b ^ 2.1)) - (.184 * (b ^ 8))
        z = .0029 * (b ^ 2.5) * ((1000000! / Red) ^ .75)
        y = (.09 * l1 * (b ^ 4) * (1! / (1 - (b ^ 4)))) - (.0337 * l2d * (b
^ 3))
        c = x + y + z
        mn = c * E * ex * (PI / 4!) * (d1 ^ 2) * ((2! * ABS(dp) * rho) ^
.5)
        i = i + 1
        v = ABS(mn - m)
        IF v > TOL THEN GOTO TARGET1
REM   Check for limits of use of BS1042:1.1. See section 7.3.1
        IF d1 < (.0125) THEN PRINT "WARNING:d1<12.5E-3"
        IF (d < .05 OR d > .76) THEN PRINT "WARNING:D<0.05 OR
D>0.76 for orifice"; no;
        IF (b < .2 OR b > .75) THEN PRINT "WARNING:b<0.2 OR b>0.75
for orifice"; no
        bdre = 1260000! * (b ^ 2) * d1
        IF (Red < bdre OR Red > 1E+08) THEN PRINT "WARNING:RED<";
bdre; "or RED>1E8 for orifice"; no; Red
        IF ((p2 / p1) < .75) THEN PRINT "WARNING: p2 / p1 < .75 for
orifice"; no; "(="; p2 / p1; ")"
END SUB

```

Appendix 4.9: Example of raw data file from READ.bas

File B3R2-1.DAT

| | | | | | |
|----|-----------|---------|----|-------------|---------|
| 1 | 50.36618 | 0.14552 | 48 | 136.35988 | 0.33489 |
| 2 | 95.02438 | 0.24085 | 49 | 88.78379 | 0.41649 |
| 3 | 80.08801 | 0.16213 | 50 | 120.53569 | 0.37170 |
| 4 | 57.25042 | 0.16239 | 51 | 77.12339 | 0.37712 |
| 5 | 50.08289 | 0.15133 | 52 | 140.26653 | 0.32937 |
| 6 | 47.21490 | 0.12951 | 53 | 86.97269 | 0.39985 |
| 7 | 46.16839 | 0.13801 | 54 | 137.02914 | 0.33352 |
| 8 | 48.51921 | 0.14983 | 55 | 88.23521 | 0.41377 |
| 9 | 49.45892 | 0.16272 | 56 | 42.33789 | 0.10563 |
| 10 | 56.35631 | 0.22404 | 57 | 43.21550 | 0.12943 |
| 11 | 79.20781 | 0.18610 | 58 | 404.23419 | 0.24925 |
| 12 | 103.84522 | 0.22606 | 59 | 393.02402 | 0.24024 |
| 13 | 87.45613 | 0.23742 | 60 | 36.85699 | 0.11593 |
| 14 | 60.61670 | 0.23733 | 61 | 26.32000 | 0.20099 |
| 15 | 52.56359 | 0.15676 | 62 | 25.98370 | 0.02299 |
| 16 | 47.75462 | 0.13398 | 63 | 25.61559 | 0.03688 |
| 17 | 48.85879 | 0.14269 | 64 | 1417.51001 | 0.90448 |
| 18 | 50.31999 | 0.15326 | 65 | 0.03346 | 0.00003 |
| 19 | 52.18708 | 0.15876 | 66 | 0.02994 | 0.00003 |
| 20 | 62.47570 | 0.19171 | 67 | 0.01362 | 0.00001 |
| 21 | 85.96541 | 0.20870 | 68 | 0.60030 | 0.00171 |
| 22 | 98.32199 | 0.21945 | 69 | 0.98050 | 0.00219 |
| 23 | 106.19989 | 0.20109 | 70 | 1.48630 | 0.00691 |
| 24 | 114.06989 | 0.22333 | 71 | 5.16950 | 0.02096 |
| 25 | 43.33789 | 0.13578 | | 29.675 | |
| 26 | 43.52032 | 0.13759 | | 26.658 | |
| 27 | 43.71591 | 0.13931 | | 4.118 | |
| 28 | 43.92071 | 0.14623 | | -2.763 | |
| 29 | 44.15640 | 0.14358 | | 26.578 | |
| 30 | 44.58379 | 0.14504 | | 0.006 | |
| 31 | 44.67130 | 0.15036 | | 20.510 | |
| 32 | 43.39379 | 0.12572 | | 20.460 | |
| 33 | 44.05550 | 0.13180 | | 0.610 | |
| 34 | 45.35569 | 0.14248 | | 0.550 | |
| 35 | 45.94619 | 0.15837 | | 0.015 | |
| 36 | 88.81811 | 0.29171 | | ---- | |
| 37 | 112.20763 | 0.60255 | | 3 | |
| 38 | 103.95870 | 0.51640 | | B3run2-1 | |
| 39 | 205.20535 | 0.61638 | | 3S----- | |
| 40 | 142.92012 | 0.43456 | | 10/10/2002 | |
| 41 | 219.11324 | 0.92805 | | 000 | |
| 42 | 118.75108 | 0.52458 | | 14.64081 | |
| 43 | 165.40338 | 1.02676 | | run OK | |
| 44 | 121.21970 | 0.36338 | | 0 | |
| 45 | 77.32111 | 0.37520 | | - | |
| 46 | 141.33212 | 0.32820 | | - | |
| 47 | 87.55138 | 0.40145 | | ***-----*** | |

Appendix 4.10: Listing of MCRheat_transfer.m

```
%program to process the Multiple Cavity Rig heat transfer data
%requires MATLAB7, MCRconduction.m, discindiff.m, simpson.m,
matprop2.m,
%MCRradiation.m, reas.m, Minning.m
%Data from and to mcrcexcel.xls
%by N.Miche
%last modified 27/03/08
%
%initialisation
%-----
echo off;
close all;
clear all;
global drd nd dr dz nr cti nz tc rsp Cp r rdb rcl kair ctk...
c_Cp c_k c_mu c_rho c_alfa c_beta c_musu c_krel r_l r_l m rotc phi
nc rdb rsh rs axd qrad;

%load raw data file N:numerical data matrix T:text data matrix
[NUM,TEXT]=xlsread('mcrcexcel.xls','rawdata');
endtest=NUM(1,2); %reads the number of tests to process

%constants
ctk=273.15; %Celsius to Kelvin

%MESH
%Parameters of disc meshed area
%-----
ro=0.121; %radius of inside thermocouple
rl=0.0855; %length of radius distance between inner and outer disc
thermocouples
zl=0.008; %disc thickness
% axdtc=[0,zl] %relative axial positions of discs thermocouples
%nd=int8(3) %number of thermocouples on each disc side
nd=3;
%nr=int8(31) %number of radial nodes
nr=31;
%nz=int8(21) %number of axial nodes
nz=21;
m=nr*nz;
rotc=206.5e-3; %radius of outside thermocouple
phi=pi/2;

%Dimensions of mesh elements
%-----
dr=rl/(nr-1);
dz=zl/(nz-1);

%Radial position of all the nodes
%-----
for j=1:nr
    for i=1:nz
        r(i,j)=ro+dz*(i-1)*cos(phi)+dr*(j-1)*sin(phi)
        r(i,j)=ro+dr*(j-1); %node radius for calculation
    end
    rcl(j)=r(1,j); %node radii for results display
end

%Results parameters matrice templates
param1_t=cell(21,6);
param1_t(:,1)={'Build','ExpCode','Rez','Ro','MassFlow(kg/s)','Bo*','Tinlet(K)','Tcav2(K)',...

'DTmeas(K)','BDTav','Ø','Ts,C2,d,av(K)','Ts,C3,d,av(K)','finTs,D2,av(K)',...
'Nu,C2,d,av',...

'Nu,C3,d,av','finNu,D2,av','Tsh,C2(K)','Nush,C2','Grsh,C2','Theta2;');
param1_t(3:21,3)={'±'};

param1_t(:,5)={'ExpDate','Run','Refi','RotSpeed(rpm)','Pcav(abs Pa)','Pr','Texit(K)','Tcav3(K)',...

'DTcalc(K)','finBDTav','finØ','Ts,C3,u,av(K)','Ts,C4,u,av(K)','finTs,D3,av(K)',...
'Nu,C3,u,av',...
'Nu,C4,u,av','finNu,D3,av','Tsh,C3(K)','Nush,C3','Grsh,C3','Theta3;');
param1_t(3:21,7)={'±'};

param2_t=cell(38,50);
param2_t(1,:)=['radius(m)','non-dim radius','shroud/cob C2','+/-',
'shroud/cob C3','+/-','T/C C2,d (K)','+/-',...
'T/C C3,u (K)','+/-','T/C C3,d (K)','+/-','T/C C4,u (K)','+/-','Ts,C2,d (K)',...
'Ts,C3,u (K)','+/-','Ts,C3,d (K)','+/-',...
'Ts,C4,u (K)','+/-','finTs,D2 (K)','+/-','finTs,D3 (K)','+/-','Nu,C2,d','+/-',
'Nu,C3,u','+/-','Nu,C3,d','+/-','Nu,C4,u','+/-',...
'finNu,D2','+/-','finNu,D3','+/-','h,C2,d','+/-','h,C3,u','+/-','h,C3,d','+/-',
'h,C4,u','+/-','finh,D2','+/-','finh,D3','+/-'];
param2_t(2:5,1)={0.0701,0.0750,0.0855,0.1210}; %thermocouple radii
param2_t(20,1)={0.1625};
param2_t(36:38,1)={0.2065,0.220,0.24565};

param3=cell(endtest+1,88);
param3(1,:)=['Build','ExpCode','Run','ExpDate','LDA code','FCF pos','Lab atm press (Pa)','Comments','Refi','+/-','Rez','+/-','BDTav','+/-',
'finBDTav','+/-','Ro','+/-','Bo*','+/-','Grsh,C2','+/-',...
'Grsh,C3','+/-','Pr','+/-','RotSpeed(rpm)','+/-','MassFlow(kg/s)','+/-',
'Ø','+/-','finØ','+/-','Tcav2(K)','+/-','Tcav3(K)','+/-','Tinlet(K)','+/-',...

'Texit(K)','+/-','DTmeas(K)','+/-','DTcalc(K)','+/-','Tsh,C2(K)','+/-',
'Tsh,C3(K)','+/-','Pcav(abs Pa)','+/-','Ts,C2,d,av(K)','+/-',...
'Ts,C3,u,av(K)','+/-','Ts,C3,d,av(K)','+/-','Ts,C4,u,av(K)','+/-',
'finTs,D2,av(K)','+/-','finTs,D3,av(K)','+/-','Nu,C2,d,av','+/-',...
'Nu,C3,u,av','+/-','Nu,C3,d,av','+/-','Nu,C4,u,av','+/-','Nush,C2','+/-',
'Nush,C3','+/-','finNu,D2,av','+/-','finNu,D3,av','+/-','Tsh,C2,out(K)','+/-',...
'Tsh,C3,out(K)','+/-','k_air(Tin)','+/-','V (m/s)','+/-'];

param4_t=cell(4,2);
param4_t(:,1)={'LDA code','FCF pos','Lab Patm (Pa)','Comments;'};

%initialise results arrays
param1=cell(endtest,1);
param2=cell(endtest,1);
results=cell(40*endtest,50);

%ERROR ESTIMATION
%ALL PRESSURES ARE ASSUMED CONSTANT
%ALL LENGTHS ARE ASSUMED CONSTANT
% index raw data numerical and text matrix per test
Traw=cell(endtest,1);
Nraw=cell(endtest,1);
for i=1:endtest
    Traw(i)=TEXT(2+(94*(i-1)):1+(94*i),:);
    Nraw(i)=NUM(2+(94*(i-1)):1+(94*i),:);
    Nraw(i)(1:63,3)=Nraw(i)(1:63,3)+ctk; %convert temperatures to Kelvin
end

%Thermocouples error intervals and uncertainties
for i=1:endtest
    for j=1:63
        %u1(j)=Nraw(i)(j,4)/sqrt(100); %measurement standard uncertainty on 100 measurements
        u1(j)=Nraw(i)(j,4)/sqrt(3); %measurement standard uncertainty considering rectangular distribution
        u2=0.0072; %thermocouple resolution standard
        uncertainty
        u3=0.0003; %cold junction standard uncertainty
        u4=0.04167; %calibration water bath stability standard
        uncertainty
        u5=0.015; %calibration PRT precision standard
        uncertainty
        u6=0.05; %Interpolation up to 100C standard
        uncertainty
        u7=0.0415; %Repeatability standard uncertainty
        Ucomb_TC(i)(j)=sqrt(u1(j)^2+u2^2+u3^2+u4^2+u5^2+u6^2+u7^2);
        %thermocouple combined standard uncertainty
        U_TC(i)(j)=Ucomb_TC(i)(j)*1.96; %T/C 95% uncertainty
        sig_TC(i)(j)=Ucomb_TC(i)(j)*sqrt(100); %thermocouple standard deviation from 100 measurements
    end
    sig_TC(i)(64)=Nraw(i)(64,4); %standard deviation for rpm
    % U_TC(i)(64)=sig_TC(i)(64)/(sqrt(100))*1.96; %95% uncertainty for rpm from 100 measurements
end

%calculation for each test
%-----
tic
for nrun=1:endtest; %counter nrun: run number
    nrun
    %Rig parameters
    %testing which rig build
    if Nraw(nrun)(84,1)==3
        %data for build 3
        rsh=0.220; %shroud radius
        rdb=70.1e-3; %disc bore radius
        rs=52e-3; %shaft radius
    end
    if Nraw(nrun)(84,1)==2
        %data for build 2
        rsh=0.220;
        rdb=70.1e-3;
        rs=0.06;
    end
    rout=0.24565; %rotor outer radius
    axd=42.9e-3; %axial distance between discs
    aan=pi*(rdb^2-rs^2); %annulus area between disc bore and shaft
    cti=7.72; %conductivity of titanium 318 (W/mK)
    drd=[0.121,0.1625,0.2065]; %radii of the three outer radius thermocouples for each side of the discs in m

    %discs measured temperature distribution
    %-----
    %Discs measured temperature
    raw_meastemp=[Nraw(nrun)(5,3),Nraw(nrun)(4,3),Nraw(nrun)(3,3);...
        Nraw(nrun)(9,3),Nraw(nrun)(10,3),Nraw(nrun)(11,3);...
        Nraw(nrun)(15,3),Nraw(nrun)(14,3),Nraw(nrun)(13,3);...
        Nraw(nrun)(19,3),Nraw(nrun)(20,3),Nraw(nrun)(21,3)];

    raw_fintemp=[mean(raw_meastemp(1:2,1)),mean(raw_meastemp(1:2,2)),...
        mean(raw_meastemp(1:2,3))];

    mean(raw_meastemp(3:4,1)),mean(raw_meastemp(3:4,2)),mean(raw_meastemp(3:4,3))];
```

```

%standard deviation on the discs measured temperatures
raw_sig_TC=[sig_TC{nrun}(5),sig_TC{nrun}(4),sig_TC{nrun}(3);...
sig_TC{nrun}(9),sig_TC{nrun}(10),sig_TC{nrun}(11);...
sig_TC{nrun}(15),sig_TC{nrun}(14),sig_TC{nrun}(13);...
sig_TC{nrun}(19),sig_TC{nrun}(20),sig_TC{nrun}(21)];
raw_sig_fintemp=[sqrt(raw_sig_TC(1,1)^2+raw_sig_TC(2,1)^2),...

sqrt(raw_sig_TC(1,2)^2+raw_sig_TC(2,2)^2),sqrt(raw_sig_TC(1,3)^2+raw_sig_TC(2,3)^2);...

sqrt(raw_sig_TC(3,1)^2+raw_sig_TC(4,1)^2),sqrt(raw_sig_TC(3,2)^2+raw_sig_TC(4,2)^2),...
sqrt(raw_sig_TC(3,3)^2+raw_sig_TC(4,3)^2)];

%discs measured temperature cases generation
% first row is the nominal measured value, following rows are
% the generated temperatures including the normally distributed error
of the
% thermocouple around its nominal value
meastemp_d2up(1,:)=raw_meastemp(1,:);
meastemp_d2dn(1,:)=raw_meastemp(2,:);
meastemp_d3up(1,:)=raw_meastemp(3,:);
meastemp_d3dn(1,:)=raw_meastemp(4,:);
meastemp_find2(1,:)=raw_fintemp(1,:);
meastemp_find3(1,:)=raw_fintemp(2,:);
sig_meastemp_d2up(1,:)=raw_sig_TC(1,:);
sig_meastemp_d2dn(1,:)=raw_sig_TC(2,:);
sig_meastemp_d3up(1,:)=raw_sig_TC(3,:);
sig_meastemp_d3dn(1,:)=raw_sig_TC(4,:);
sig_meastemp_find2(1,:)=raw_sig_fintemp(1,:);
sig_meastemp_find3(1,:)=raw_sig_fintemp(2,:);

for i=2:101
    for j=1:3

meastemp_d2up(i,j)=meastemp_d2up(1,j)+sig_meastemp_d2up(1,j)*randn;

meastemp_d2dn(i,j)=meastemp_d2dn(1,j)+sig_meastemp_d2dn(1,j)*randn;

meastemp_d3up(i,j)=meastemp_d3up(1,j)+sig_meastemp_d3up(1,j)*randn;

meastemp_d3dn(i,j)=meastemp_d3dn(1,j)+sig_meastemp_d3dn(1,j)*randn;

meastemp_find2(i,j)=meastemp_find2(1,j)+sig_meastemp_find2(1,j)*randn;

meastemp_find3(i,j)=meastemp_find3(1,j)+sig_meastemp_find3(1,j)*randn;
    end
end

%other fluctuating parameters
rsp(1)=Nraw(nrun)(64,3)*pi/30; %nominal value of rotational speed (rad/s)
sig_rsp=Nraw(nrun)(64,4)*pi/30;
tc(1)=(Nraw(nrun)(32,3)+Nraw(nrun)(33,3))/2; %nominal value of inlet temp
sig_tc=sqrt(U_TC(nrun)(32)^2+U_TC(nrun)(33)^2)/1.96*sqrt(100);
te(1)=(Nraw(nrun)(34,3)+Nraw(nrun)(35,3))/2; %nominal value of exit temp
sig_te=sqrt(U_TC(nrun)(34)^2+U_TC(nrun)(35)^2)/1.96*sqrt(100);
TC23(1)=Nraw(nrun)(23,3); %T/C 23 (rim cav2)
sig_TC23=sig_TC(nrun)(23);
TC2(1)=Nraw(nrun)(2,3); %T/C 2 (shroud cav2)
sig_TC2=sig_TC(nrun)(2);
TC24(1)=Nraw(nrun)(24,3); %T/C 24 (rim cav3)
sig_TC24=sig_TC(nrun)(24);
TC12(1)=Nraw(nrun)(12,3); %T/C 12 (shroud cav3)
sig_TC12=sig_TC(nrun)(12);
TC26(1)=Nraw(nrun)(26,3); %T/C 26 (shaft temp under cav2)
sig_TC26=sig_TC(nrun)(26);
TC28(1)=Nraw(nrun)(28,3); %T/C 28 (shaft temp under cav3)
sig_TC28=sig_TC(nrun)(28);
TC27(1)=Nraw(nrun)(27,3); %T/C 27 (shaft under cob2)
sig_TC27=sig_TC(nrun)(27);
TC29(1)=Nraw(nrun)(29,3); %T/C 29 (shaft under cob3)
sig_TC29=sig_TC(nrun)(29);
TC8(1)=Nraw(nrun)(8,3); %T/C 8 (cob2 downstream face)
sig_TC8=sig_TC(nrun)(8);
TC16(1)=Nraw(nrun)(16,3); %T/C 16 (cob3 upstream face)
sig_TC16=sig_TC(nrun)(16);
TC6(1)=Nraw(nrun)(6,3); %T/C 6 (cob2 upstream face)
sig_TC6=sig_TC(nrun)(6);
TC7(1)=Nraw(nrun)(7,3); %T/C 7 (bore 2)
sig_TC7=sig_TC(nrun)(7);
TC17(1)=Nraw(nrun)(17,3); %T/C 17 (bore 3)
sig_TC17=sig_TC(nrun)(17);
TC18(1)=Nraw(nrun)(18,3); %T/C 18 (cob 3 downstream face)
sig_TC18=sig_TC(nrun)(18);

for i=2:101
    rsp(i)=rsp(1)+sig_rsp*randn; %normally distributed
    tc(i)=tc(1)+sig_tc*randn;
    te(i)=tc(1)+sig_tc*randn;
    TC23(i)=TC23(1)+sig_TC23*randn;
    TC2(i)=TC2(1)+sig_TC2*randn;
    TC24(i)=TC24(1)+sig_TC24*randn;
    TC12(i)=TC12(1)+sig_TC12*randn;
    TC26(i)=TC26(1)+sig_TC26*randn;
    TC28(i)=TC28(1)+sig_TC28*randn;
    TC27(i)=TC27(1)+sig_TC27*randn;
    TC29(i)=TC29(1)+sig_TC29*randn;
    TC8(i)=TC8(1)+sig_TC8*randn;
    TC16(i)=TC16(1)+sig_TC16*randn;
    TC6(i)=TC6(1)+sig_TC6*randn;
    TC7(i)=TC7(1)+sig_TC7*randn;
    TC17(i)=TC17(1)+sig_TC17*randn;
    TC18(i)=TC18(1)+sig_TC18*randn;
end

Tfluct.TC23=TC23; %Multiple Array Tfluct comprising temperatures
Tfluct.TC2=TC2; %to be used for the radiation correction
Tfluct.TC24=TC24;
Tfluct.TC12=TC12;
Tfluct.TC26=TC26;
Tfluct.TC28=TC28;
Tfluct.TC27=TC27;
Tfluct.TC29=TC29;
Tfluct.TC8=TC8;
Tfluct.TC16=TC16;
Tfluct.TC7=TC7;
Tfluct.TC17=TC17;

%%
%%
%%CALCULATIONS FOR EACH TEMPERATURE CASE

for nc=1:101 %nc: temperature case counter

%OPERATIONAL PARAMETERS
expdate=Traw(nrun){87,1}; %date of experiment
expcode=Traw(nrun){86,1}; %experiment code
runid=Traw(nrun){85,1}; %experiment name
patm=Nraw(nrun)(89,1)*6894.757; %Atm press in lab (Pa)
pc=Nraw(nrun)(78,1)*6894.757+patm; %abs cavity pressure from upstream tap in shaft (Pa)
fi=Nraw(nrun)(80,1); %mass flow IN (kg/s)

[Cp,kair,visc,rho,alfa,beta]=matprop2(tc(nc),pc); %air properties for tc, pc

n_kair(nc)=kair;
V(nc)=fi/(rho*aan); %flow velocity in annulus
Rez(nc)=2*rho*V(nc)*(rdb-rs)/visc; %Rez
Ro(nc)=V(nc)/(rsp(nc)*rdb); %Ro
Rephi(nc)=(rho*rsp(nc)*rsh^2)/visc; %Ref
Pr(nc)=visc*Cp/kair; %Pr

%SHROUD HEAT TRANSFER (1-D radial conduction)
ta2(nc)=tc(nc)+(rsp(nc)^2*(rsh^2-rdb^2))/(2*Cp); %relative total temperature at rsh in cavity2
ca2(nc)=0.741e-4*ta2(nc)+4.01e-3; %conductivity of air for cavity2
da2(nc)=pc/(ta2(nc)^287); %density in cavity2
va2(nc)=1.46e-6*ta2(nc)^1.5/(ta2(nc)+110); %viscosity in cavity2
shq2(nc)=cti*(TC23(nc)-TC2(nc))/(rsh*log(rout/rsh)); %shroud heat flux in cavity2
shnu2(nc)=shq2(nc)*(axd/2)/(ca2(nc)*(TC2(nc)-ta2(nc)));
%shroud nusselt number in cavity2
shbd2(nc)=(TC2(nc)-ta2(nc))/ta2(nc); %shroud bDT, cavity2

gr2(nc)=(da2(nc)^2*rsp(nc)^2*rsh*shbd2(nc)*(axd/2)^3)/(va2(nc)^2);
%shroud grashof number in cavity2
bostarsh2(nc)=sqrt(shbd2(nc))/Ro(nc); %shroud Bo* in cavity2

%ta3(nc)=TC28(nc)+(rsp(nc)^2*(rsh^2-rdb^2))/(2*Cp);
%ta3(nc)=ta2(nc); %relative total temperature at rsh in cavity3
ca3(nc)=0.741e-4*ta3(nc)+4.01e-3; %conductivity of air for cavity3
da3(nc)=pc/(ta3(nc)^287); %density in cavity3
va3(nc)=1.46e-6*ta3(nc)^1.5/(ta3(nc)+110); %viscosity in cavity3
shq3(nc)=cti*(TC24(nc)-TC12(nc))/(rsh*log(rout/rsh)); %shroud heat flux in cavity3
shnu3(nc)=shq3(nc)*(axd/2)/(ca3(nc)*(TC12(nc)-ta3(nc)));
%shroud nusselt number in cavity3
shbd3(nc)=(TC12(nc)-ta3(nc))/ta3(nc); %shroud bDT, cavity3

gr3(nc)=(da3(nc)^2*rsp(nc)^2*rsh*shbd3(nc)*(axd/2)^3)/(va3(nc)^2);
%shroud grashof number in cavity3
bostarsh3(nc)=sqrt(shbd3(nc))/Ro(nc); %shroud Bo* in cavity3

%calculations for Tcav2
if gr2(nc)*Pr(nc) > 1e7
    shroud2C=0.14;
    shroud2n=1/3;
else
    shroud2C=0.54;
    shroud2n=1/4;
end

```

```

TshC2(nc)=TC2(nc);
Tcav2(nc)=TshC2(nc)-(TshC2(nc)-
tc(nc))/(shroud2C*(gr2(nc)*Pr(nc))^shroud2n/shnu2(nc))^(1/(shroud2n+1)
);
Theta2(nc)=(TshC2(nc)-tc(nc))/(TshC2(nc)-Tcav2(nc));

%calculations for Tcav3
if gr3(nc)*Pr(nc) > 1e7
shroud3C=0.14;
shroud3n=1/3;
else
shroud3C=0.54;
shroud3n=1/4;
end
TshC3(nc)=TC12(nc);
Tcav3(nc)=TshC3(nc)-(TshC3(nc)-
tc(nc))/(shroud3C*(gr3(nc)*Pr(nc))^shroud3n/shnu3(nc))^(1/(shroud3n+1)
);
Theta3(nc)=(TshC3(nc)-tc(nc))/(TshC3(nc)-Tcav3(nc));

%DISCS CONDUCTION
%building meastemp matrix for discs surfaces
meastemp(1,:)=meastemp_d2up(nc,:);
meastemp(2,:)=meastemp_d2dn(nc,:);
meastemp(3,:)=meastemp_d3up(nc,:);
meastemp(4,:)=meastemp_d3dn(nc,:);
meastemp(5,:)=meastemp_find2(nc,:);
meastemp(6,:)=meastemp_find2(nc,:);
meastemp(7,:)=meastemp_find3(nc,:);
meastemp(8,:)=meastemp_find3(nc,:);
%
%Building temperature matrix for radial temp fit and thermocouples
location
% NOT used for fin calculations

fittemp(1,:)= [TC7(nc),TC6(nc),meastemp(1,1),meastemp(1,2),meastemp(
1,3),TC2(nc)]; %d2_up

fittemp(2,:)= [TC7(nc),TC8(nc),meastemp(2,1),meastemp(2,2),meastemp(
2,3),TC12(nc)]; %d2_dn

fittemp(3,:)= [TC17(nc),TC16(nc),meastemp(3,1),meastemp(3,2),measte
mp(3,3),TC12(nc)]; %d3_up

fittemp(4,:)= [TC17(nc),TC18(nc),meastemp(4,1),meastemp(4,2),measte
mp(4,3),meastemp(4,3)]; %d3_dn shroud T/C n/a
fitradius(1,:)= [0.0701,0.0750,0.1210,0.1625,0.2065,0.2200];
%d2_up
fitradius(2,:)= [0.0701,0.0855,0.1210,0.1625,0.2065,0.2200];
%d2_dn
fitradius(3,:)= [0.0701,0.0750,0.1210,0.1625,0.2065,0.2200];
%d3_up
fitradius(4,:)= [0.0701,0.0750,0.1210,0.1625,0.2065,0.2065];
%d3_dn (5 thermocouples max on rig)
%
%call MCR conduction subroutine

[trdial,q,grad,DT,Nur,Nu_av,BDT_av,q_av,DT_av]=MCRconduction(me
astemp,Tfluct,fittemp,fitradius);
%
%
i=nc; %temperature case index
trdial_d2up(i,:)=trdial(1,:);
trdial_d2dn(i,:)=trdial(2,:);
trdial_d3up(i,:)=trdial(3,:);
trdial_d3dn(i,:)=trdial(4,:);
trdial_find2(i,:)=trdial(5,:);
trdial_find3(i,:)=trdial(7,:);
q_d2up(i,:)=q(1,:);
q_d2dn(i,:)=q(2,:);
q_d3up(i,:)=q(3,:);
q_d3dn(i,:)=q(4,:);
q_find2(i,:)=q(5,:);
q_find3(i,:)=q(7,:);
%
qrad_d2up(i,:)=qrad(1,:);
qrad_d2dn(i,:)=qrad(2,:);
%
qrad_d3up(i,:)=qrad(3,:);
qrad_d3dn(i,:)=qrad(4,:);
%
qrad_find2(i,:)=qrad(5,:);
%
qrad_find3(i,:)=qrad(7,:);
Nur_d2up(i,:)=Nur(1,:);
Nur_d2dn(i,:)=Nur(2,:);
Nur_d3up(i,:)=Nur(3,:);
Nur_d3dn(i,:)=Nur(4,:);
Nur_find2(i,:)=Nur(5,:);
Nur_find3(i,:)=Nur(7,:);
q_av_d2up(i)=q_av(1);
q_av_d2dn(i)=q_av(2);
q_av_d3up(i)=q_av(3);
q_av_d3dn(i)=q_av(4);
q_av_find2(i)=q_av(5);
q_av_find3(i)=q_av(7);
DT_av_d2up(i)=DT_av(1);
DT_av_d2dn(i)=DT_av(2);
DT_av_d3up(i)=DT_av(3);
DT_av_d3dn(i)=DT_av(4);
DT_av_find2(i)=DT_av(5);
DT_av_find3(i)=DT_av(7);
Nu_av_d2up(i)=Nu_av(1);
Nu_av_d2dn(i)=Nu_av(2);
Nu_av_d3up(i)=Nu_av(3);

Nu_av_d3dn(i)=Nu_av(4);
Nu_av_find2(i)=Nu_av(5);
Nu_av_find3(i)=Nu_av(7);
BDT_av_d2up(i)=BDT_av(1);
BDT_av_d2dn(i)=BDT_av(2);
BDT_av_d3up(i)=BDT_av(3);
BDT_av_d3dn(i)=BDT_av(4);
BDT_av_find2(i)=BDT_av(5);
BDT_av_find3(i)=BDT_av(7);

%Ts_av
TsC2dav(nc)=(BDT_av_d2up(nc)*tc(nc))+tc(nc);
TsC3uav(nc)=(BDT_av_d2dn(nc)*tc(nc))+tc(nc);
TsC3dav(nc)=(BDT_av_d3up(nc)*tc(nc))+tc(nc);
TsC4uav(nc)=(BDT_av_d3dn(nc)*tc(nc))+tc(nc);
finTsD2av(nc)=(BDT_av_find2(nc)*tc(nc))+tc(nc);
finTsD3av(nc)=(BDT_av_find3(nc)*tc(nc))+tc(nc);

%DT measured
DTmeas(nc)=te(nc)-tc(nc);

%DT calculated
DTcalc(nc)=pi*rsh*kair/(fl*Cp)*(4*4*((TshC3(nc)+TshC2(nc))/2-
tc(nc))*(shnu2(nc)+shnu3(nc))/2+...
8*((finTsD2av(nc)+finTsD3av(nc))/2-
tc(nc))*(Nu_av_find2(nc)+Nu_av_find3(nc))/2);

%BDTavav
BDTavav(nc)=(BDT_av_d2up(nc)+BDT_av_d2dn(nc)+BDT_av_d3up(nc)
+BDT_av_d3dn(nc))/4;
finBDTavav(nc)=(BDT_av_find2(nc)+BDT_av_find3(nc))/2;

%Bo*
Bostar(nc)=sqrt(BDTavav(nc))/Ro(nc);

%O stroke
Ostroke(nc)=4*Theta3(nc)/(Rez(nc)*(Theta3(nc)-1)*Pr(nc)^2*(rdb-
rs)*(TshC3(nc)-tc(nc)))...
*(shnu3(nc)*(TshC3(nc)-tc(nc))^4*rsh)...
+(Nu_av_d2dn(nc)*(TsC3uav(nc)-tc(nc))/rsh*(rsh^2-rdb^2))...
+(Nu_av_d3up(nc)*(TsC3dav(nc)-tc(nc))/rsh*(rsh^2-rdb^2));
finOstroke(nc)=4*Theta3(nc)/(Rez(nc)*(Theta3(nc)-1)*Pr(nc)^2*(rdb-
rs)*(TshC3(nc)-tc(nc)))...
*(shnu3(nc)*(TshC3(nc)-tc(nc))^4*rsh)...
+(Nu_av_find2(nc)*(finTsD2av(nc)-tc(nc))/rsh*(rsh^2-rdb^2))...
+(Nu_av_find3(nc)*(finTsD3av(nc)-tc(nc))/rsh*(rsh^2-rdb^2));

end
%%
%%
%%
%DISCS RESULTS

%q_av
q_C2dav=q_av_d2up(1);
sd_q_C2dav=std(q_av_d2up(2:101));
q_C3uav=q_av_d2dn(1);
sd_q_C3uav=std(q_av_d2dn(2:101));
q_C3dav=q_av_d3up(1);
sd_q_C3dav=std(q_av_d3up(2:101));
q_C4uav=q_av_d3dn(1);
sd_q_C4uav=std(q_av_d3dn(2:101));
q_finD2av=q_av_find2(1);
sd_q_finD2av=std(q_av_find2(2:101));
q_finD3av=q_av_find3(1);
sd_q_finD3av=std(q_av_find3(2:101));

%DT_av
DT_C2dav=DT_av_d2up(1);
sd_DT_C2dav=std(DT_av_d2up(2:101));
DT_C3uav=DT_av_d2dn(1);
sd_DT_C3uav=std(DT_av_d2dn(2:101));
DT_C3dav=DT_av_d3up(1);
sd_DT_C3dav=std(DT_av_d3up(2:101));
DT_C4uav=DT_av_d3dn(1);
sd_DT_C4uav=std(DT_av_d3dn(2:101));
DT_finD2av=DT_av_find2(1);
sd_DT_finD2av=std(DT_av_find2(2:101));
DT_finD3av=DT_av_find3(1);
sd_DT_finD3av=std(DT_av_find3(2:101));

%BDTav
BDTC2dav=BDT_av_d2up(1);
sd_BDTC2dav=std(BDT_av_d2up(2:101));
BDTC3uav=BDT_av_d2dn(1);
sd_BDTC3uav=std(BDT_av_d2dn(2:101));
BDTC3dav=BDT_av_d3up(1);
sd_BDTC3dav=std(BDT_av_d3up(2:101));
BDTC4uav=BDT_av_d3dn(1);
sd_BDTC4uav=std(BDT_av_d3dn(2:101));
finBDTD2av=BDT_av_find2(1);
sd_finBDTD2av=std(BDT_av_find2(2:101));
finBDTD3av=BDT_av_find3(1);
sd_finBDTD3av=std(BDT_av_find3(2:101));

%Nuav
n_NuC2dav=Nu_av_d2up(1);
sd_NuC2dav=std(Nu_av_d2up(2:101));
n_NuC3uav=Nu_av_d2dn(1);
sd_NuC3uav=std(Nu_av_d2dn(2:101));
n_NuC3dav=Nu_av_d3up(1);

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sd_NuC3dav=std(Nu_av_d3up(2:101));
n_NuC4uav=Nu_av_d3dn(1);
sd_NuC4uav=std(Nu_av_d3dn(2:101));
n_finNuD2av=Nu_av_find2(1);
sd_finNuD2av=std(Nu_av_find2(2:101));
n_finNuD3av=Nu_av_find3(1);
sd_finNuD3av=std(Nu_av_find3(2:101));

%interpolated values
for i=1:nr
%
    %n_Ts, sd_Ts
    n_Ts_d2up(i)=tradiat_d2up(1,i);
    sd_Ts_d2up(i)=std(tradiat_d2up(2:101,i));
    n_Ts_d2dn(i)=tradiat_d2dn(1,i);
    sd_Ts_d2dn(i)=std(tradiat_d2dn(2:101,i));
    n_Ts_d3up(i)=tradiat_d3up(1,i);
    sd_Ts_d3up(i)=std(tradiat_d3up(2:101,i));
    n_Ts_d3dn(i)=tradiat_d3dn(1,i);
    sd_Ts_d3dn(i)=std(tradiat_d3dn(2:101,i));
    n_Ts_find2(i)=tradiat_find2(1,i);
    sd_Ts_find2(i)=std(tradiat_find2(2:101,i));
    n_Ts_find3(i)=tradiat_find3(1,i);
    sd_Ts_find3(i)=std(tradiat_find3(2:101,i));

    %n_Nu, sd_Nu
    n_Nu_d2up(i)=Nur_d2up(1,i);
    sd_Nu_d2up(i)=std(Nur_d2up(2:101,i));
    n_Nu_d2dn(i)=Nur_d2dn(1,i);
    sd_Nu_d2dn(i)=std(Nur_d2dn(2:101,i));
    n_Nu_d3up(i)=Nur_d3up(1,i);
    sd_Nu_d3up(i)=std(Nur_d3up(2:101,i));
    n_Nu_d3dn(i)=Nur_d3dn(1,i);
    sd_Nu_d3dn(i)=std(Nur_d3dn(2:101,i));
    n_Nu_find2(i)=Nur_find2(1,i);
    sd_Nu_find2(i)=std(Nur_find2(2:101,i));
    n_Nu_find3(i)=Nur_find3(1,i);
    sd_Nu_find3(i)=std(Nur_find3(2:101,i));

    %h
    for j=1:101
        [c_Cp, c_k, c_mu, c_rho, c_a,
c_b]=matprop2(tradiat_d2up(j,i),pc);
        h_d2up(j,i)=c_k*Nur_d2up(j,i)/rcl(i);
        [c_Cp, c_k, c_mu, c_rho, c_a,
c_b]=matprop2(tradiat_d2dn(j,i),pc);
        h_d2dn(j,i)=c_k*Nur_d2dn(j,i)/rcl(i);
        [c_Cp, c_k, c_mu, c_rho, c_a,
c_b]=matprop2(tradiat_d3up(j,i),pc);
        h_d3up(j,i)=c_k*Nur_d3up(j,i)/rcl(i);
        [c_Cp, c_k, c_mu, c_rho, c_a,
c_b]=matprop2(tradiat_d3dn(j,i),pc);
        h_d3dn(j,i)=c_k*Nur_d3dn(j,i)/rcl(i);
        [c_Cp, c_k, c_mu, c_rho, c_a,
c_b]=matprop2(tradiat_find2(j,i),pc);
        h_find2(j,i)=c_k*Nur_find2(j,i)/rcl(i);
        [c_Cp, c_k, c_mu, c_rho, c_a,
c_b]=matprop2(tradiat_find3(j,i),pc);
        h_find3(j,i)=c_k*Nur_find3(j,i)/rcl(i);
    end
    n_h_d2up(i)=h_d2up(1,i);
    sd_h_d2up(i)=std(h_d2up(2:101,i));
    n_h_d2dn(i)=h_d2dn(1,i);
    sd_h_d2dn(i)=std(h_d2dn(2:101,i));
    n_h_d3up(i)=h_d3up(1,i);
    sd_h_d3up(i)=std(h_d3up(2:101,i));
    n_h_d3dn(i)=h_d3dn(1,i);
    sd_h_d3dn(i)=std(h_d3dn(2:101,i));
    n_h_find2(i)=h_find2(1,i);
    sd_h_find2(i)=std(h_find2(2:101,i));
    n_h_find3(i)=h_find3(1,i);
    sd_h_find3(i)=std(h_find3(2:101,i));

end

%CALCULATED OPERATIONAL PARAMETERS NOMINAL VALUES
AND S.D.
n_V=V(1);
sd_V=std(V(2:101));
n_tc=tc(1);
sd_tc=std(tc(2:101));
n_te=te(1);
sd_te=std(te(2:101));
n_Rez=Rez(1); % n_ nominal value
sd_Rez=std(Rez(2:101)); % sd_ Standard deviation
n_Rephi=Rephi(1);
sd_Rephi=std(Rephi(2:101));
n_Ro=Ro(1);
sd_Ro=std(Ro(2:101));
n_Pr=Pr(1);
sd_Pr=std(Pr(2:101));
n_shnu2=shnu2(1);
sd_shnu2=std(shnu2(2:101));
n_shbd2=shbd2(1);
sd_shbd2=std(shbd2(2:101));
n_gr2=gr2(1);
sd_gr2=std(gr2(2:101));
n_bostarsh2=bostarsh2(1);
sd_bostarsh2=std(bostarsh2(2:101));
n_shnu3=shnu3(1);
sd_shnu3=std(shnu3(2:101));

n_shbd3=shbd3(1);
sd_shbd3=std(shbd3(2:101));
n_gr3=gr3(1);
sd_gr3=std(gr3(2:101));
n_bostarsh3=bostarsh3(1);
sd_bostarsh3=std(bostarsh3(2:101));
n_TshC2=TshC2(1);
sd_TshC2=std(TshC2(2:101));
n_Tcav2=Tcav2(1);
sd_Tcav2=std(Tcav2(2:101));
n_Theta2=Theta2(1);
sd_Theta2=std(Theta2(2:101));
n_TshC3=TshC3(1);
sd_TshC3=std(TshC3(2:101));
n_Tcav3=Tcav3(1);
sd_Tcav3=std(Tcav3(2:101));
n_Theta3=Theta3(1);
sd_Theta3=std(Theta3(2:101));
n_Ostroke=Ostroke(1);
sd_Ostroke=std(Ostroke(2:101));
n_finOstroke=finOstroke(1);
sd_finOstroke=std(finOstroke(2:101));
n_DTmeas=DTmeas(1);
sd_DTmeas=std(DTmeas(2:101));
n_DTcalc=DTcalc(1);
sd_DTcalc=std(DTcalc(2:101));
n_TsC2dav=TsC2dav(1);
sd_TsC2dav=std(TsC2dav(2:101));
n_TsC3uav=TsC3uav(1);
sd_TsC3uav=std(TsC3uav(2:101));
n_TsC3dav=TsC3dav(1);
sd_TsC3dav=std(TsC3dav(2:101));
n_TsC4uav=TsC4uav(1);
sd_TsC4uav=std(TsC4uav(2:101));
n_finTsD2av=finTsD2av(1);
sd_finTsD2av=std(finTsD2av(2:101));
n_finTsD3av=finTsD3av(1);
sd_finTsD3av=std(finTsD3av(2:101));
n_BDTavav=BDTavav(1);
sd_BDTavav=std(BDTavav(2:101));
n_finBDTavav=finBDTavav(1);
sd_finBDTavav=std(finBDTavav(2:101));
n_Bostar=Bostar(1);
sd_Bostar=std(Bostar(2:101));

%fill results arrays
%param1
param1{nrn}=param1_t; %labels from param1 template
param1{nrn}{1,2}=Nraw{nrn}(84,1); %Build
param1{nrn}{1,6}=expdate; %Date of experiment
param1{nrn}{2,2}=expcode; %Exp code
param1{nrn}{2,6}=runid; %Run

param1{nrn}{3,2}=n_Rez; %Rez
param1{nrn}{3,4}=sd_Rez/sqrt(100)*1.96; %u_Rez 95%
confidence standard uncertainty
param1{nrn}{3,6}=n_Rephi; %Ref
param1{nrn}{3,8}=sd_Rephi/sqrt(100)*1.96; %u_Rephi
param1{nrn}{4,2}=n_Ro; %Ro
param1{nrn}{4,4}=sd_Ro/sqrt(100)*1.96; %u_Ro
param1{nrn}{4,6}=rsp(1)*30/pi; %rot speed
param1{nrn}{4,8}=sig_rsp/sqrt(100)*1.96; %u_rsp
param1{nrn}{5,2}=fl; %Mass flow
%param1{nrn}{5,4}= %u_fl
param1{nrn}{5,6}=pc; %Pcav
%param1{nrn}{5,8}= %u_pc
param1{nrn}{6,2}=n_Bostar; %Bo*
param1{nrn}{6,4}=sd_Bostar/sqrt(100)*1.96; %u_Bo*
param1{nrn}{6,6}=n_Pr; %Pr
param1{nrn}{6,8}=sd_Pr/sqrt(100)*1.96; %u_Pr
param1{nrn}{7,2}=tc(1); %Tinlet
param1{nrn}{7,4}=sig_tc/sqrt(100)*1.96; %u_tc
param1{nrn}{7,6}=te(1); %Texit
param1{nrn}{7,8}=sig_te/sqrt(100)*1.96; %u_te
param1{nrn}{8,2}=n_Tcav2; %Tcav2
param1{nrn}{8,4}=sd_Tcav2/sqrt(100)*1.96; %u_Tcav2
param1{nrn}{8,6}=n_Tcav3; %Tcav3
param1{nrn}{8,8}=sd_Tcav3/sqrt(100)*1.96; %u_Tcav3
param1{nrn}{9,2}=n_DTmeas; %DTmeas
param1{nrn}{9,4}=sd_DTmeas/sqrt(100)*1.96; %u_DTmeas
param1{nrn}{9,6}=n_DTcalc; %DTcalc
param1{nrn}{9,8}=sd_DTcalc/sqrt(100)*1.96; %u_DTcalc
param1{nrn}{10,2}=n_BDTavav; %BDTavav
param1{nrn}{10,4}=sd_BDTavav/sqrt(100)*1.96; %u_BDTavav
param1{nrn}{10,6}=n_finBDTavav; %finBDTav
param1{nrn}{10,8}=sd_finBDTavav/sqrt(100)*1.96; %u_finBDTav
param1{nrn}{11,2}=n_Ostroke; %O
param1{nrn}{11,4}=sd_Ostroke/sqrt(100)*1.96; %u_Ostroke
param1{nrn}{11,6}=n_finOstroke; %finO
param1{nrn}{11,8}=sd_finOstroke/sqrt(100)*1.96; %u_finOstroke
param1{nrn}{12,2}=n_TsC2dav; %Ts.C2,d.av
param1{nrn}{12,4}=sd_TsC2dav/sqrt(100)*1.96; %u_TsC2dav
param1{nrn}{12,6}=n_TsC3uav; %Ts.C3,u.av
param1{nrn}{12,8}=sd_TsC3uav/sqrt(100)*1.96; %u_TsC3uav
param1{nrn}{13,2}=n_TsC3dav; %Ts.C3,d.av
param1{nrn}{13,4}=sd_TsC3dav/sqrt(100)*1.96; %u_TsC3dav
param1{nrn}{13,6}=n_TsC4uav; %Ts.C4,u.av
param1{nrn}{13,8}=sd_TsC4uav/sqrt(100)*1.96; %u_TsC4uav
param1{nrn}{14,2}=n_finTsD2av; %finTs.D2.av
param1{nrn}{14,4}=sd_finTsD2av/sqrt(100)*1.96; %u_finTsD2av
param1{nrn}{14,6}=n_finTsD3av; %finTs.D3.av

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param1{nrn}{14,8}=sd_finTsD3av/sqrt(100)*1.96;    %u_finTsD3av
param1{nrn}{15,2}=n_NuC2dav;                    %Nu,C2,d,av
param1{nrn}{15,4}=sd_NuC2dav/sqrt(100)*1.96;    %u_NuC2dav
param1{nrn}{15,6}=n_NuC3uav;                    %Nu,C3,u,av
param1{nrn}{15,8}=sd_NuC3uav/sqrt(100)*1.96;    %u_NuC3uav
param1{nrn}{16,2}=n_NuC3dav;                    %Nu,C3,d,av
param1{nrn}{16,4}=sd_NuC3dav/sqrt(100)*1.96;    %u_NuC3dav
param1{nrn}{16,6}=n_NuC4uav;                    %Nu,C4,u,av
param1{nrn}{16,8}=sd_NuC4uav/sqrt(100)*1.96;    %u_NuC4uav
param1{nrn}{17,2}=n_finNuD2av;                  %finNu,D2,av
param1{nrn}{17,4}=sd_finNuD2av/sqrt(100)*1.96;
%u_finNuD2av
param1{nrn}{17,6}=n_finNuD3av;                  %finNu,D3,av
param1{nrn}{17,8}=sd_finNuD3av/sqrt(100)*1.96;
%u_finNuD3av
param1{nrn}{18,2}=n_TshC2;                      %Tsh,C2
param1{nrn}{18,4}=sd_TshC2/sqrt(100)*1.96;      %u_TshC2
param1{nrn}{18,6}=n_TshC3;                      %Tsh,C3
param1{nrn}{18,8}=sd_TshC3/sqrt(100)*1.96;      %u_TshC3
param1{nrn}{19,2}=n_shnu2;                      %Nush,C2
param1{nrn}{19,4}=sd_shnu2/sqrt(100)*1.96;      %u_shnu2
param1{nrn}{19,6}=n_shnu3;                      %Nush,C3
param1{nrn}{19,8}=sd_shnu3/sqrt(100)*1.96;      %u_shnu3
param1{nrn}{20,2}=n_gr2;                        %Grsh,C2
param1{nrn}{20,4}=sd_gr2/sqrt(100)*1.96;        %u_gr2
param1{nrn}{20,6}=n_gr3;                        %Grsh,C3
param1{nrn}{20,8}=sd_gr3/sqrt(100)*1.96;        %u_gr3
param1{nrn}{21,2}=n_Theta2;                    %Theta2
param1{nrn}{21,4}=sd_Theta2/sqrt(100)*1.96;    %u_Theta2
param1{nrn}{21,6}=n_Theta3;                    %Theta3
param1{nrn}{21,8}=sd_Theta3/sqrt(100)*1.96;    %u_Theta3

%fill interpolated results array average and standard deviation
%param2
param2{nrn}=param2_t;                          %labels
for i=1:14
    param2{nrn}{i+5,1}=rcl(i+1);              %node radii
end
for i=1:15
    param2{nrn}{i+20,1}=rcl(i+15);
end
for i=2:37
    param2{nrn}{i,2}=param2{nrn}{i,1}/rsh;    %non-dim radius
end
param2{nrn}{2,3}=Nraw{nrn}(7,3);              %T/C 7
param2{nrn}{2,4}=U_TC{nrn}(7);                %u T/C 7
param2{nrn}{3,3}=Nraw{nrn}(2,3);              %T/C 2
param2{nrn}{3,4}=U_TC{nrn}(2);                %u T/C 2
param2{nrn}{3,8,3}=Nraw{nrn}(23,3);          %T/C 23
param2{nrn}{3,8,4}=U_TC{nrn}(23);            %u T/C 23
param2{nrn}{2,5}=Nraw{nrn}(17,3);            %T/C 17
param2{nrn}{2,6}=U_TC{nrn}(17);              %u T/C 17
param2{nrn}{3,7,5}=Nraw{nrn}(12,3);          %T/C 12
param2{nrn}{3,7,6}=U_TC{nrn}(12);            %u T/C 12
param2{nrn}{3,8,5}=Nraw{nrn}(24,3);          %T/C 24
param2{nrn}{3,8,6}=U_TC{nrn}(24);            %u T/C 24
param2{nrn}{3,7}=Nraw{nrn}(6,3);              %T/C 6
param2{nrn}{3,8}=U_TC{nrn}(6);               %u T/C 6
param2{nrn}{5,7}=meastemp_d2up(1,1);          %T/C 5
param2{nrn}{5,8}=std(meastemp_d2up(2:101,1))/sqrt(100)*1.96;
%u T/C 5
param2{nrn}{20,7}=meastemp_d2up(1,2);         %T/C 4
param2{nrn}{20,8}=std(meastemp_d2up(2:101,2))/sqrt(100)*1.96;
%u T/C 5
param2{nrn}{36,7}=meastemp_d2up(1,3);         %T/C 3
param2{nrn}{36,8}=std(meastemp_d2up(2:101,3))/sqrt(100)*1.96;
%u T/C 3
param2{nrn}{4,9}=Nraw{nrn}(8,3);              %T/C 8
param2{nrn}{4,10}=U_TC{nrn}(8);               %u T/C 8
param2{nrn}{5,9}=meastemp_d2dn(1,1);          %T/C 9
param2{nrn}{5,10}=std(meastemp_d2dn(2:101,1))/sqrt(100)*1.96;
%u T/C 9
param2{nrn}{20,9}=meastemp_d2dn(1,2);          %T/C 10
param2{nrn}{20,10}=std(meastemp_d2dn(2:101,2))/sqrt(100)*1.96;
%u T/C 10
param2{nrn}{36,9}=meastemp_d2dn(1,3);          %T/C 11
param2{nrn}{36,10}=std(meastemp_d2dn(2:101,3))/sqrt(100)*1.96;
%u T/C 11
param2{nrn}{3,11}=Nraw{nrn}(16,3);            %T/C 16
param2{nrn}{3,12}=U_TC{nrn}(16);              %u T/C 16
param2{nrn}{5,11}=meastemp_d3up(1,1);          %T/C 15
param2{nrn}{5,12}=std(meastemp_d3up(2:101,1))/sqrt(100)*1.96;
%u T/C 15
param2{nrn}{20,11}=meastemp_d3up(1,2);         %T/C 14
param2{nrn}{20,12}=std(meastemp_d3up(2:101,2))/sqrt(100)*1.96;
%u T/C 14
param2{nrn}{36,11}=meastemp_d3up(1,3);         %T/C 13
param2{nrn}{36,12}=std(meastemp_d3up(2:101,3))/sqrt(100)*1.96;
%u T/C 13
param2{nrn}{3,13}=Nraw{nrn}(18,3);            %T/C 18
param2{nrn}{3,14}=U_TC{nrn}(18);              %u T/C 18
param2{nrn}{5,13}=meastemp_d3dn(1,1);          %T/C 19
param2{nrn}{5,14}=std(meastemp_d3dn(2:101,1))/sqrt(100)*1.96;
%u T/C 19
param2{nrn}{20,13}=meastemp_d3dn(1,2);         %T/C 20
param2{nrn}{20,14}=std(meastemp_d3dn(2:101,2))/sqrt(100)*1.96;
%u T/C 20
param2{nrn}{36,13}=meastemp_d3dn(1,3);         %T/C 21
param2{nrn}{36,14}=std(meastemp_d3dn(2:101,3))/sqrt(100)*1.96;
%u T/C 21

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for i=1:14
    param2{nrn}{i+5,15}=n_Ts_d2up(i+1);      %interpolated line6:19
    param2{nrn}{i+5,16}=sd_Ts_d2up(i+1)/sqrt(100)*1.96;
    param2{nrn}{i+5,17}=n_Ts_d2dn(i+1);
    param2{nrn}{i+5,18}=sd_Ts_d2dn(i+1)/sqrt(100)*1.96;
    param2{nrn}{i+5,19}=n_Ts_d3up(i+1);
    param2{nrn}{i+5,20}=sd_Ts_d3up(i+1)/sqrt(100)*1.96;
    param2{nrn}{i+5,21}=n_Ts_d3dn(i+1);
    param2{nrn}{i+5,22}=sd_Ts_d3dn(i+1)/sqrt(100)*1.96;
    param2{nrn}{i+5,23}=n_Ts_find2(i+1);
    param2{nrn}{i+5,24}=sd_Ts_find2(i+1)/sqrt(100)*1.96;
    param2{nrn}{i+5,25}=n_Ts_find3(i+1);
    param2{nrn}{i+5,26}=sd_Ts_find3(i+1)/sqrt(100)*1.96;
    param2{nrn}{i+5,27}=n_Nu_d2up(i+1);
    param2{nrn}{i+5,28}=sd_Nu_d2up(i+1)/sqrt(100)*1.96;
    param2{nrn}{i+5,29}=n_Nu_d2dn(i+1);
    param2{nrn}{i+5,30}=sd_Nu_d2dn(i+1)/sqrt(100)*1.96;
    param2{nrn}{i+5,31}=n_Nu_d3up(i+1);
    param2{nrn}{i+5,32}=sd_Nu_d3up(i+1)/sqrt(100)*1.96;
    param2{nrn}{i+5,33}=n_Nu_d3dn(i+1);
    param2{nrn}{i+5,34}=sd_Nu_d3dn(i+1)/sqrt(100)*1.96;
    param2{nrn}{i+5,35}=n_Nu_find2(i+1);
    param2{nrn}{i+5,36}=sd_Nu_find2(i+1)/sqrt(100)*1.96;
    param2{nrn}{i+5,37}=n_Nu_find3(i+1);
    param2{nrn}{i+5,38}=sd_Nu_find3(i+1)/sqrt(100)*1.96;
    param2{nrn}{i+5,39}=n_h_d2up(i+1);
    param2{nrn}{i+5,40}=sd_h_d2up(i+1)/sqrt(100)*1.96;
    param2{nrn}{i+5,41}=n_h_d2dn(i+1);
    param2{nrn}{i+5,42}=sd_h_d2dn(i+1)/sqrt(100)*1.96;
    param2{nrn}{i+5,43}=n_h_d3up(i+1);
    param2{nrn}{i+5,44}=sd_h_d3up(i+1)/sqrt(100)*1.96;
    param2{nrn}{i+5,45}=n_h_d3dn(i+1);
    param2{nrn}{i+5,46}=sd_h_d3dn(i+1)/sqrt(100)*1.96;
    param2{nrn}{i+5,47}=n_h_find2(i+1);
    param2{nrn}{i+5,48}=sd_h_find2(i+1)/sqrt(100)*1.96;
    param2{nrn}{i+5,49}=n_h_find3(i+1);
    param2{nrn}{i+5,50}=sd_h_find3(i+1)/sqrt(100)*1.96;
end
for i=1:15
    param2{nrn}{i+20,15}=n_Ts_d2up(i+15);    %interpolated line21:35
    param2{nrn}{i+20,16}=sd_Ts_d2up(i+15)/sqrt(100)*1.96;
    param2{nrn}{i+20,17}=n_Ts_d2dn(i+15);
    param2{nrn}{i+20,18}=sd_Ts_d2dn(i+15)/sqrt(100)*1.96;
    param2{nrn}{i+20,19}=n_Ts_d3up(i+15);
    param2{nrn}{i+20,20}=sd_Ts_d3up(i+15)/sqrt(100)*1.96;
    param2{nrn}{i+20,21}=n_Ts_d3dn(i+15);
    param2{nrn}{i+20,22}=sd_Ts_d3dn(i+15)/sqrt(100)*1.96;
    param2{nrn}{i+20,23}=n_Ts_find2(i+15);
    param2{nrn}{i+20,24}=sd_Ts_find2(i+15)/sqrt(100)*1.96;
    param2{nrn}{i+20,25}=n_Ts_find3(i+15);
    param2{nrn}{i+20,26}=sd_Ts_find3(i+15)/sqrt(100)*1.96;
    param2{nrn}{i+20,27}=n_Nu_d2up(i+15);
    param2{nrn}{i+20,28}=sd_Nu_d2up(i+15)/sqrt(100)*1.96;
    param2{nrn}{i+20,29}=n_Nu_d2dn(i+15);
    param2{nrn}{i+20,30}=sd_Nu_d2dn(i+15)/sqrt(100)*1.96;
    param2{nrn}{i+20,31}=n_Nu_d3up(i+15);
    param2{nrn}{i+20,32}=sd_Nu_d3up(i+15)/sqrt(100)*1.96;
    param2{nrn}{i+20,33}=n_Nu_d3dn(i+15);
    param2{nrn}{i+20,34}=sd_Nu_d3dn(i+15)/sqrt(100)*1.96;
    param2{nrn}{i+20,35}=n_Nu_find2(i+15);
    param2{nrn}{i+20,36}=sd_Nu_find2(i+15)/sqrt(100)*1.96;
    param2{nrn}{i+20,37}=n_Nu_find3(i+15);
    param2{nrn}{i+20,38}=sd_Nu_find3(i+15)/sqrt(100)*1.96;
    param2{nrn}{i+20,39}=n_h_d2up(i+15);
    param2{nrn}{i+20,40}=sd_h_d2up(i+15)/sqrt(100)*1.96;
    param2{nrn}{i+20,41}=n_h_d2dn(i+15);
    param2{nrn}{i+20,42}=sd_h_d2dn(i+15)/sqrt(100)*1.96;
    param2{nrn}{i+20,43}=n_h_d3up(i+15);
    param2{nrn}{i+20,44}=sd_h_d3up(i+15)/sqrt(100)*1.96;
    param2{nrn}{i+20,45}=n_h_d3dn(i+15);
    param2{nrn}{i+20,46}=sd_h_d3dn(i+15)/sqrt(100)*1.96;
    param2{nrn}{i+20,47}=n_h_find2(i+15);
    param2{nrn}{i+20,48}=sd_h_find2(i+15)/sqrt(100)*1.96;
    param2{nrn}{i+20,49}=n_h_find3(i+15);
    param2{nrn}{i+20,50}=sd_h_find3(i+15)/sqrt(100)*1.96;
end

%non-dim line matrix param3
%param3{1,1:84}=param3_t(1,1:84)
param3{nrn+1,1}=Nraw{nrn}(84,1);              %Build
param3{nrn+1,2}=expcode;                      %Exp code
param3{nrn+1,3}=runid;                        %Run
param3{nrn+1,4}=expdate;                      %Date of experiment
param3{nrn+1,5}=Nraw{nrn}(88,1);              %LDA code
param3{nrn+1,6}=Nraw{nrn}(91,1);              %FCF position
param3{nrn+1,7}=Nraw{nrn}(89,1)*6894.757;    %Lab atm press (Pa)
param3{nrn+1,8}=Traw{nrn}(90,1);              %Comments of experiment
param3{nrn+1,9}=n_Rephi;                      %Ref
param3{nrn+1,10}=sd_Rephi/sqrt(100)*1.96;
param3{nrn+1,11}=n_Rez;                      %Rez
param3{nrn+1,12}=sd_Rez/sqrt(100)*1.96;
param3{nrn+1,13}=n_BDTavav;                  %BDTavav
param3{nrn+1,14}=sd_BDTavav/sqrt(100)*1.96;
param3{nrn+1,15}=n_finBDTavav;               %finBDTavav
param3{nrn+1,16}=sd_finBDTavav/sqrt(100)*1.96;
param3{nrn+1,17}=n_Ro;                      %Ro
param3{nrn+1,18}=sd_Ro/sqrt(100)*1.96;
param3{nrn+1,19}=n_Bostar;                   %Bo*
param3{nrn+1,20}=sd_Bostar/sqrt(100)*1.96;
param3{nrn+1,21}=n_gr2;                      %Grsh,C2
param3{nrn+1,22}=sd_gr2/sqrt(100)*1.96;

```

```

param3{nrn+1,23}=n_gr3;          %Grsh,C3
param3{nrn+1,24}=sd_gr3/sqrt(100)*1.96;
param3{nrn+1,25}=n_Pr;          %Pr
param3{nrn+1,26}=sd_Pr/sqrt(100)*1.96;
param3{nrn+1,27}=rsp(1)*30/pi;  %rot speed (rpm)
param3{nrn+1,28}=sig_rsp/sqrt(100)*1.96*30/pi;
param3{nrn+1,29}=fl;           %Mass flow
%param3{nrn+1,30}=
param3{nrn+1,31}=n_Ostroke;      %Ø
param3{nrn+1,32}=sd_Ostroke/sqrt(100)*1.96;
param3{nrn+1,33}=n_finOstroke;   %finØ
param3{nrn+1,34}=sd_finOstroke/sqrt(100)*1.96;
param3{nrn+1,35}=n_Tcav2;        %Tcav2
param3{nrn+1,36}=sd_Tcav2/sqrt(100)*1.96;
param3{nrn+1,37}=n_Tcav3;        %Tcav3
param3{nrn+1,38}=sd_Tcav3/sqrt(100)*1.96;
param3{nrn+1,39}=n_tc;          %Tinlet
param3{nrn+1,40}=sd_tc/sqrt(100)*1.96;
param3{nrn+1,41}=n_te;          %Texit
param3{nrn+1,42}=sd_te/sqrt(100)*1.96;
param3{nrn+1,43}=n_DTmeas;       %DTmeas
param3{nrn+1,44}=sd_DTmeas/sqrt(100)*1.96;
param3{nrn+1,45}=n_DTcalc;       %DTcalc
param3{nrn+1,46}=sd_DTcalc/sqrt(100)*1.96;
param3{nrn+1,47}=n_TshC2;        %Tsh,C2
param3{nrn+1,48}=sd_TshC2/sqrt(100)*1.96;
param3{nrn+1,49}=n_TshC3;        %Tsh,C3
param3{nrn+1,50}=sd_TshC3/sqrt(100)*1.96;
param3{nrn+1,51}=pc;            %Pcav
%param3{nrn+1,52}=
param3{nrn+1,53}=n_TsC2dav;       %Ts,C2,d,av
param3{nrn+1,54}=sd_TsC2dav/sqrt(100)*1.96;
param3{nrn+1,55}=n_TsC3uav;      %Ts,C3,u,av
param3{nrn+1,56}=sd_TsC3uav/sqrt(100)*1.96;
param3{nrn+1,57}=n_TsC3dav;      %Ts,C3,d,av
param3{nrn+1,58}=sd_TsC3dav/sqrt(100)*1.96;
param3{nrn+1,59}=n_TsC4uav;      %Ts,C4,u,av
param3{nrn+1,60}=sd_TsC4uav/sqrt(100)*1.96;
param3{nrn+1,61}=n_finTsD2av;     %finTs,D2,av
param3{nrn+1,62}=sd_finTsD2av/sqrt(100)*1.96;
param3{nrn+1,63}=n_finTsD3av;     %finTs,D3,av
param3{nrn+1,64}=sd_finTsD3av/sqrt(100)*1.96;
param3{nrn+1,65}=n_NuC2dav;       %Nu,C2,d,av
param3{nrn+1,66}=sd_NuC2dav/sqrt(100)*1.96;
param3{nrn+1,67}=n_NuC3uav;       %Nu,C3,u,av
param3{nrn+1,68}=sd_NuC3uav/sqrt(100)*1.96;
param3{nrn+1,69}=n_NuC3dav;       %Nu,C3,d,av
param3{nrn+1,70}=sd_NuC3dav/sqrt(100)*1.96;
param3{nrn+1,71}=n_NuC4uav;       %Nu,C4,u,av
param3{nrn+1,72}=sd_NuC4uav/sqrt(100)*1.96;
param3{nrn+1,73}=n_shnu3;         %Nush,C2

```

```

param3{nrn+1,74}=sd_shnu2/sqrt(100)*1.96;
param3{nrn+1,75}=n_shnu3;        %Nush,C3
param3{nrn+1,76}=sd_shnu3/sqrt(100)*1.96;
param3{nrn+1,77}=n_finNuD2av;    %finNu,D2,av
param3{nrn+1,78}=sd_finNuD2av/sqrt(100)*1.96;
param3{nrn+1,79}=n_finNuD3av;    %finNu,D3,av
param3{nrn+1,80}=sd_finNuD3av/sqrt(100)*1.96;
param3{nrn+1,81}=Nraw{nrn}(23,3); %T/C 23
param3{nrn+1,82}=U_TC{nrn}(23);  %T/C 24
param3{nrn+1,83}=Nraw{nrn}(24,3); %T/C 24
param3{nrn+1,84}=U_TC{nrn}(24);  %T/C 24
param3{nrn+1,85}=n_kair(1);       %k(Tinlet)
param3{nrn+1,86}=std(n_kair(2:101))/sqrt(100)*1.96;
param3{nrn+1,87}=n_V;             %Flow annulus velocity (m/s)
param3{nrn+1,88}=sd_V/sqrt(100)*1.96;

%param4
param4{nrn}=param4_t;
param4{nrn}{1,2}=Nraw{nrn}(88,1); %LDA code
param4{nrn}{2,2}=Nraw{nrn}(91,1); %FCF position
param4{nrn}{3,2}=Nraw{nrn}(89,1)*6894.757; %Lab atm press (Pa)
param4{nrn}{4,2}=Traw{nrn}(90,1); %Comments of

experiment
toc
end

%Build results matrix
for i=1:endtest
    for j=1:21
        for k=1:8
            results{j+(i-1)*40,k}=param1{i,j,k};
        end
    end
end
for j=1:38
    for k=10:59
        results{j+(i-1)*40,k}=param2{i,j,k-9};
    end
end
for j=23:26
    for k=1:2
        results{j+(i-1)*40,k}=param4{i,j-22,k};
    end
end
end
end

%copy result matrix worksheet
xlswrite('MCRExcel.xls','results','results')

%copy to non-dim worksheet
xlswrite('MCRExcel.xls',param3,'non-dims')

```

Appendix 4.11: Listing of MCRconduction.m

```

function[tr radial,q,grad,DT,Nur,Nu_av,BDT_av,q_av,DT_av]=MCRconduc
tion(meastemp,Tfluct,fitemp,fitradius)
%subroutine calculating the conduction solution of the Multiple Cavity
Rig's discs surfaces
%by N.Miche
%last modified 22/03/08
%requires
%
%input: meastemp
%output: radial,q,grad,DT,Nur,Nu_av,BDT_av
global drd nd dr zl dz nr cti nz tc rsp Cp r rdb rcl kair ro rl zl m rotc phi nc
rdb rsh rs axd grad;
%radial temperature curve fitting
%-----
%
%fitting on 3 thermocouples per disc face: 2nd order polynomial
%for i=1:4
% fitradial=polyfit(drd,meastemp(i,1:nd),2);
% radial(i,:)=polyval(fitradial,drd(1):drd(nd));
%end
%
%fitting on 4 thermocouples per disc face: 3rd order poly, cob and disc
%for i=1:4
% fitradial=polyfit(fitradius(i,2:5),fitemp(i,2:5),3);
% radial(i,:)=polyval(fitradial,drd(1):drd(nd));
%end
%
%fitting on 5 thermocouples per disc face, bore,cob and disc, 4th order
%poly
%for i=1:4
% fitradial=polyfit(fitradius(i,1:5),fitemp(i,1:5),4);
% radial(i,:)=polyval(fitradial,drd(1):drd(nd));
%end
%
%fitting on 6 thermocouples per disc face, bore, cob, disc, shroud, 5th
%order polynomial (d3_dn fitted on 5 thermocouples)
%for i=1:3
% fitradial=polyfit(fitradius(i,:),fitemp(i,:),5);
% radial(i,:)=polyval(fitradial,drd(1):drd(nd));
%end
%fitradial=polyfit(fitradius(4,1:5),fitemp(4,1:5),5); %d3_dn has only 5
thermocouples max
%radial(4,:)=polyval(fitradial,drd(1):drd(nd));
%
%for i=5:8 %fits on 3 thermocouple per face for fin disks
%fitradial=polyfit(drd,meastemp(i,1:nd),2); %2nd order polynomial
%apply fitted temperatures at nodes
%radial(i,:)=polyval(fitradial,drd(1):drd(nd)); %interpolation
between the three thermocouples on discs faces
% if i== 2 | 4 | 6 | 8 %if i=2 or 4 or 6 or 8
% axq(i/2,j)=(cti/ztl)*(radial(i-1,j)-radial(i,j)) %1-D axial
% heat flux calculation
%end
radial(5,:)=(radial(1,:)+radial(2,:))/2; %average temperature profile for
finTs,D2,u
radial(6,:)=radial(5,:); %fin,Ts,D2,d
radial(7,:)=(radial(3,:)+radial(4,:))/2; %fin,Ts,D3,u
radial(8,:)=radial(7,:); %fin,Ts,D3,d
%temperatures at the inner and outer radii by linear interpolation
%-----
for j=2:2:8
infitaxial=polyfit([0,zl],[meastemp((j-1):j,1)],1); %linear
interpolation
outfitaxial=polyfit([0,zl],[meastemp((j-1):j,3)],1);
taxial_inner(j/2,:)=polyval(infitaxial,0:dz:zl);
taxial_outer(j/2,:)=polyval(outfitaxial,0:dz:zl);
end
%1-D axial heat flux calculation
%-----
for i=2:2:8
for j=1:nr
axq(i/2,j)=(cti/ztl)*(radial(i-1,j)-radial(i,j));
end
end
%FINITE DIFFERENCE
[TDisc2]=discfindiff(radial(1:2,:),[taxial_inner(1,:);taxial_outer(1,:)]);
%conditions for disc2
[TDisc3]=discfindiff(radial(3:4,:),[taxial_inner(2,:);taxial_outer(2,:)]);
%conditions for disc3
[Tfindisc2]=discfindiff(radial(5:6,:),[taxial_inner(3,:);taxial_outer(3,:)]);
%conditions for findisc2
[Tfindisc3]=discfindiff(radial(7:8,:),[taxial_inner(4,:);taxial_outer(4,:)]);
%conditions for findisc3
%total heat flux using backward difference formula
T=[TDisc2;TDisc3;Tfindisc2;Tfindisc3];
for i=2:2:8
for j=1:nr
qt((i-1),j)=cti*(-3*T((i/2),(j-1)*nz+1))+4*T((i/2),(j-1)*nz+2))-T((i/2),(j-1)*nz+3))/(2*dz);
qt(i,j)=cti*(-3*T((i/2),(j*nz))+4*T((i/2),(j*nz-1))-T((i/2),(j*nz-2)))/(2*dz);
end
end
%RADIATION HEAT FLUX
%Temperatures for the radiation calculations
Trad(1)=(Tfluct.TC27(nc)+Tfluct.TC28(nc)+Tfluct.TC29(nc))/3; %Tsurf1
(shaft)
Trad(2)=tc(nc); %Tsurf2
Trad(3)=(Tfluct.TC8(nc)+meastemp(2,1)+Tfluct.TC7(nc))/3;
%Tsurf3
for i=4:nr+3
Trad(i)=tr radial(2,i-3); %Interpolated surf temp upstream
end
Trad(nr+4)=(meastemp(2,nd)+Tfluct.TC12(nc))/2; %Tsurf35
Trad(nr+5)=Tfluct.TC12(nc); %Tsurf36 (shroud)
Trad(nr+6)=(Tfluct.TC12(nc)+meastemp(3,nd))/2; %Tsurf37
for i=nr+7:2*nr+6
Trad(i)=tr radial(3,(2*nr+7)-i); %Intepolated surf temp downstream
end
Trad(2*nr+7)=(meastemp(3,1)+Tfluct.TC16(nc)+Tfluct.TC17(nc))/3;
%Tsurf69
Trad(2*nr+8)=tc(nc); %Tsurf70
%grad=zeros(8,nr);
[grad]=MCRRadiation(Trad);
%corrected heat flux
q(1,:)=qt(1,:); %D2up
q(2,:)=qt(2,1:nr)-grad(4:nr+3); %D2dn corrected between T/C9 and
T/C11
for i=1:nr
q(3,i)=qt(3,i)-grad(2*nr+7-i); %D3up corrected between T/C15 and
T/C13
end
q(4,:)=qt(4,:); %D3dn
q(5:8,:)=qt(5:8,:); %finD2 and finD3
%LOCAL NON-DIMENSIONAL NUMBERS
for i=2:2:8
for j=1:nr
Tref(j)=tc(nc)+rsp(nc)^2*(r(1,j)^2-rdb^2)/(2*Cp); %adiabatic wall
fluid temperature
DT((i-1),j)=T((i/2),(j-1)*nz+1))-Tref(j); %surface to fluid Delta T
(upstream disc face)
DT(i,j)=T((i/2),(j*nz))-Tref(j); % (downstream)
Nur((i-1),j)=q((i-1),j)*rcl(j)/(kair*DT((i-1),j)); %local Nu
Nur(i,j)=q(i,j)*rcl(j)/(kair*DT(i,j));
end
end
%average nusselt number and average BetaDeltaT
for i=1:8
for j=1:nr-2
y1(j)=q(i,j+1)*r(1,j+1);
y2(j)=DT(i,j+1)*r(1,j+1);
end
q_av(i)=simpson(y1,nr-2,dr);
DT_av(i)=simpson(y2,nr-2,dr);
Nu_av(i)=(q_av(i)*rotc)/(DT_av(i)*kair);
BDT_av(i)=(1/tc(nc))*(2/(rotc^2-ro^2))*DT_av(i);
End

```

Function matprop2.m

```

function [c,Cp,c_krel,c_musu,c_rho,c_alfa,c_beta] = matprop2(T,P)
% function [Cp, k, mu, rho, alfa, beta] = matprop2(T,P)
% return material properties value for given temperature
% 10/8/99
% L.S.Wong, TFMRC, University of Sussex
%modified by N. Miché 05/05/2005
b0=0.1034009e4;
b1=-0.2848870;
b2=0.7816818e-3;
b3=-0.4970786e-6;
b4=0.1077024e-9;
c0=-2.276501e-3;
c1=1.2598485e-4;
c2=-1.4815235e-7;
c3=1.73550646e-10;
c4=-1.066657e-13;
c5=2.47663035e-17;
d0=-9.8601e-7;
d1=9.080125e-8;
d2=-1.17635575e-10;
d3=1.2349703e-13;
d4=-5.7971299e-17;
R=287; % Specific gas constant

```

Appendix 4.12: Listing of discfindiff.m

```
function [T]=discfindiff(Temp radial,Temp axial)
%Finite difference for a disc in the multiple cavity rig
%-----
%----- AT=B to be solved-----
%A: temperature dependant coefficient matrix
%T: node temperature vector
%B: vectore containing the temperature independant boundary condition
information
%clear all
global nr nz dr dz r;
%dnr=int16(nr)
%dnz=int16(nz)
%nr=31
%nz=21
%dr=0.00285
%dz=0.0004
%load tempaxial1
%load tempradial1
%load r1
%Build matrix A and B
A=zeros(nr*nz); %A: empty matrix (nr*nz)X(nr*nz)
B=zeros(1,nr*nz); %B: empty matrix 1X(nr*nz)
T=zeros(1,nr*nz);
for j=1:nr
% if j==nr
% l=nz-1
% else
% l=nz
% end
for i=1:nz
k=(j-1)*nz+i;
if i==1 %assign left wall boundary temperature
A(k,k)=1;
B(k)=Tempradial(1,j); %upstream disc radial temperature
elseif i==nz %assign right wall boundary temperature
A(k,k)=1;
B(k)=Tempradial(2,j); %downstream disc radial temperature
elseif j==1 & i==(1 | nz) %assign inner radius boundary
temperature
A(k,k)=1;
B(k)=Tempaxial(1,i); %inner radius temperature
elseif j==nr & i==(1 | nz) %assign outer radius boundary
temperature
A(k,k)=1;
B(k)=Tempaxial(2,i); %outer radius temperature
else %assign coefficients for inside boundaries point
A(k,k+nz)=(1/dr^2+1/(2*r(i,j)*dr)); %north node
A(k,k-nz)=(1/dr^2-1/(2*r(i,j)*dr)); %south node
A(k,k)=(-2/dr^2-2/dz^2); %point node
A(k,k+1)=1/dz^2; %east node
A(k,k-1)=1/dz^2; %west node
end
end
end

%solve system of simultaneous linear equations AT=B
%T'=minres(A,B') %minimal residual method (iterative)
%T=B/A
T=A\B'; %Gaussian pivot
```

T=T';

Function simpson.m

```
function [area]=simpson(y,n,h)
% SEE ICASGT1 REPORT
% y: function
% n: number of points (must be odd)
% h: strip width =(b-a)/m where m=n-1: number of strips
% a & b: lower and upper limits of integration
%
area=0;
for i=1:2:n-2
y0=y(i);
y1=y(i+1);
y2=y(i+2);
area=area+((h/3)*(y0+(4*y1)+y2));
end
```

Function minning.m

```
function [VFdtd]=minning(ri,ro,rc,rho1,rho2,s)
%Calculates the View Factors between coaxial annular discs separated
by a solid cylinder
%based on Mining (1978) Eq. (9)
%
%The input values are [ri, ro] disc radius (A2)
% [rho1, rho2] differential area radius (dA1)
% [rc] cylinder outer radius
% [s] disc spacing
% A. Alexiou (28/11/00)

step=(rho2-rho1)/1000;
r=rho1:step:rho2;
omg=acos(rc./r)+acos(rc/ro);
phi=acos(rc./r)+acos(rc/ri);

f = r .* ((omg - phi) ./ (2*pi) + (ro.^2 - r.^2 - s.^2) ./...
(pi * sqrt((ro.^2 + r.^2 + s.^2).^2 - (4 * r.^2 * ro.^2))).*...
atan(sqrt((ro.^2 + r.^2 + s.^2 + (2.*r*ro))./...
(ro.^2 + r.^2 + s.^2 - (2.*r*ro))) .* tan(omg/2)) - (ri.^2 - r.^2 - s.^2) ./...
(pi*sqrt((ri.^2 + r.^2 + s.^2).^2 - (4.*r.^2 * ri.^2))).*...
atan(sqrt((ri.^2 + r.^2 + s.^2 + (2.*r*ri))./...
(ri.^2 + r.^2 + s.^2 - (2.*r*ri))) .* tan(phi/2))). ./...

area=trapz(r,f);

VFdtd=2*area/(rho2^2 - rho1^2);
```

Appendix 4.13: Listing of MCRradiation.m

```

function[grad]=MCRradiation(Trad)
%Funtion to calculate the black and grey body radiation of the Multiple
Cavity Rig
%in cavity 3. Assumes Isothermal surfaces.
%requires, MATLAB7, MCRheat_transfer.m main program, Rea.m
subroutine,
%minning.m subroutine
global rs rdb rcl dr nr rsh axd

F=zeros(2*nr+8);

%radiation surfaces nodes RADII
%-----
rrd(1)=rs;
rrd(2)=rdb;
rrd(3)=rcl(1)-dr;
for j=1:nr
    rrd(3+j)=rcl(j)+dr;
end
rrd(4+nr)=rsh;

%AREA calculation
%-----
A(1)=2*pi*rrd(1)*axd;      %shaft
A(2)=pi*(rrd(2)^2-rrd(1)^2); %Disc2,downstream
A(3)=pi*(rrd(3)^2-rrd(2)^2);
for i=1:nr
    A(3+i)=pi*(rrd(3+i)^2-rrd(2+i)^2);
end
A(nr+4)=pi*(rrd(nr+4)^2-rrd(nr+3)^2);
A(nr+5)=2*pi*rrd(nr+4)*axd; %shroud
for i=1:(nr+3)
    A(nr+5+i)=A(nr+5-i); %Disc3,upstream
end

%VIEW FACTORS
%-----
%Flat or convex surfaces: F(i,i)=0
F(1,1)=0;
F(2:nr+4,2:nr+4)=0;
F(nr+6:2*nr+8,nr+6:2*nr+8)=0;

%Shaft-Disc View Factors using Rea (1975) Eq.(2)
F(1,2)=reas(axd,rrd(2),rrd(1)); %shaft to Disc2,downstream
for j=3:nr+4
    F(1,j)=reas(axd,rrd(j),rrd(1))-reas(axd,rrd(j-1),rrd(1));
end
for i=1:nr+3
    F(1,nr+5+i)=F(1,nr+5-i); %shaft to Disc3,upstream
end
F(1,nr+5)=1-sum(F(1,:)); %shaft to shroud by summation rule:
sum(F)=1

%Disc-Shaft View Factors from Reciprocity r/ship: A(i).F(i,j)=A(j).F(j,i)
for i=2:2*nr+8
    F(i,1)=A(1).*F(1,i)./A(i);
end

%UpDisc-DwnDisc View Factors from Minning (1979) Eq (9)
for i=1:nr+3
    for j=1:nr+3
        F(i+1,2*nr+9-j)=minning(rrd(j),rrd(j+1),rrd(1),rrd(i),rrd(i+1),axd);
    end
end

%DwnDisc-UpDisc View Factors (by reciprocity)
for i=0:nr+2
    for j=0:nr+2
        F(2*nr+8-j,2+i)=A(2+i).*F(2+i,2*nr+8-j)./A(2*nr+8-j);
    end
end

%Disc-Shroud View Factors (by summation rule)
for i=0:nr+2
    F(2+i,nr+5)=1-sum(F(2+i,:));
    F(nr+6+i,nr+5)=1-sum(F(nr+6+i,:));
end

%Shroud-Disc View Factors (by reciprocity)
for i=0:nr+2
    F(nr+5,2+i)=A(2+i).*F(2+i,nr+5)./A(nr+5);
    F(nr+5,nr+6+i)=A(nr+6+i).*F(nr+6+i,nr+5)./A(nr+5);
end

%Shroud-Shroud View Factor (by summation)
F(nr+5,nr+5)=1-sum(F(nr+5,:));

%TEMPERATURES on radiation surfaces
%-----
%Use of array Trad, defined in MCRconduction.m

%RADIATIVE HEAT FLUX
%-----
sigma=56.7e-9; %Stefan-Boltzman constant (W/m^2K^4)

%Black bodies
%qradb(0)=0
for i=1:2*nr+8
    qj=0;
    for j=1:2*nr+8
        %qradb(i)=sig.*(Trad(i).^4-(F(i,j)*Trad(j).^4))
        qj=qj+sigma.*F(i,j).*A(j).*(Trad(i).^4-Trad(j).^4); % (From C.Long, p250)
    end
    qradb(i)=qj;
end

%Grey bodies
e1=0.2; %emissivity value for shaft (not painted)
e2=0.95; %emissivity value for rotor surfaces (painted black)
e3=1; %emissivity value for cooling annulus gap (black body)
Eps(1)=e1; %shaft
Eps(2)=e3; %upstream gap
Eps(3:2*nr+7)=e2; %rotor
Eps(2*nr+8)=e3; %downstream gap
Phi=zeros(1,2*nr+8);
for i=1:2*nr+8
    Phi(i)=(1-Eps(i))/Eps(i);
end

Eb=sigma*Trad.^4;

%solve [B]*(J)=(Eb) for finding (J)
B=zeros(2*nr+8);
for i=1:2*nr+8
    for j=1:2*nr+8
        B(i,j)=Phi(i)*F(i,j);
    end
    B(i,i)=Phi(i)*(sum(F(i,:))-F(i,i))+1;
end
J=inv(B)*Eb';

grad=(Eb-J')./Phi;

```

Function Reas.m

```

function [VFctd]=reas(ls,R,rs)
%Calculates the cylinder-disc View Factors
%based on Rea (1975) Eq. (2)
%The input values are [ls] cylinder length
% [R] Disc outer radius
% [rs] cylinder outer radius
% A. Alexiou (28/11/00)

VFctd=(acos((ls^2 - R^2 + rs^2) / (ls^2 + R^2 - rs^2)) - (rs / (2*ls)) * ...
        (((ls^2 + R^2 + rs^2)^2 / rs^4) - (4 * (R / rs)^2)^0.5) * ...
        acos(rs * (ls^2 - R^2 + rs^2) / (R * (ls^2 + R^2 - rs^2))) + ...
        ((ls^2 - R^2 + rs^2) / rs^2) * asin(rs / R) - ...
        (pi * (ls^2 + R^2 - rs^2) / (2 * rs^2)))) / (2*pi);

```

Appendix 4.14: Example of raw results output from the heat transfer processing software

```

Build:                               3                               ExpDate:       10/10/2002
ExpCode:    3S-----              Run:           B3run2-1
Re:         165239.0376 ±         127.4045483 Refi:       994725.8213 ±         1747.92541
Ro:         3.168312702 ±         0.003119535 RotSpeed(rpm): 1417.51001 ±         0.018564517
MassFlow(kg/s): 0.61 ±              Pcav(abs Pa):       242356.2933 ±
Bo*:        0.074591357 ±         0.00073897 Pr:         0.70541207 ±         5.15656E-05
Tinlet(K):   316.874645 ±         0.3006544 Textt(K):     318.80094 ±         0.325609864
Tcav2(K):    323.4873662 ±         1.486228649 Tcav3(K):    335.2485335 ±         1.322226369
DTmeas(K):   1.926295 ±         0.427102029 DTcalc(K):    1.916532512 ±         0.049249949
BDTav:       0.055851275 ±         0.001014786 finBDTav:   0.055851273 ±         0.001014786
Ø:           0.309718793 ±         0.056327455 finØ:       0.163084407 ±         0.035363792
Ts,C2,d,av(K): 332.6420843 ±         0.169165908 Ts,C3,u,av(K): 331.8430508 ±         0.221672842
Ts,C3,d,av(K): 336.3923706 ±         0.22221217 Ts,C4,u,av(K): 337.4124858 ±         0.202030464
finTs,D2,av(K): 332.2425669 ±         0.146426366 finTs,D3,av(K): 336.9024278 ±         0.149754908
Nu,C2,d,av:  -223.3319762 ±         118.0043467 Nu,C3,u,av:   590.4948835 ±         140.6639831
Nu,C3,d,av:   520.243974 ±         115.9289847 Nu,C4,u,av:  -252.2868387 ±         105.5465394
finNu,D2,av:  147.983547 ±         12.74509059 finNu,D3,av:  116.8387643 ±         11.81800361
Tsh,C2(K):    368.17438 ±         0.320184871 Tsh,C3(K):     376.99522 ±         0.304000388
Nush,C2:      54.52348377 ±         2.295260988 Nush,C3:      42.50712749 ±         1.681067738
Grsh,C2:     145431656.3 ±         1829398.146 Grsh,C3:     170672181.1 ±         1971165.54
Theta2:       1.147978589 ±         0.042482679 Theta3:     1.440128068 ±         0.049213152

LDA code:                0
FCF pos:                 0
Lab Patm (Pa):    100944.8272
Comments:      run OK

```

| radius(m) | non-dim radius | shroud/cob C2 +/- | shroud/cob C3 +/- | T/C C2,d (K) +/- | T/C C3,u (K) +/- | T/C C3,d (K) +/- | T/C C4,u (K) +/- |
|-----------|----------------|-------------------|-------------------|------------------|------------------|------------------|------------------|
| 0.0701 | 0.318636364 | 319.31839 | 0.219873993 | 320.3649 | 0.213149577 | 320.90462 | 0.216658614 |
| 0.075 | 0.340909091 | | | | | | |
| 0.0855 | 0.388636364 | | | | | | |
| 0.121 | 0.55 | | | 323.23289 | 0.216698825 | 322.60892 | 0.223286466 |
| 0.12385 | 0.562954545 | | | | | | |
| 0.1267 | 0.575909091 | | | | | | |
| 0.12955 | 0.588863636 | | | | | | |
| 0.1324 | 0.601818182 | | | | | | |
| 0.13525 | 0.614772727 | | | | | | |
| 0.1381 | 0.627727273 | | | | | | |
| 0.14095 | 0.640681818 | | | | | | |
| 0.1438 | 0.653636364 | | | | | | |
| 0.14665 | 0.666590909 | | | | | | |
| 0.1495 | 0.679545455 | | | | | | |
| 0.15235 | 0.6925 | | | | | | |
| 0.1552 | 0.705454545 | | | | | | |
| 0.15805 | 0.718409091 | | | | | | |
| 0.1609 | 0.731363636 | | | | | | |
| 0.1625 | 0.738636364 | | | 330.40042 | 0.228654131 | 329.50631 | 0.30930212 |
| 0.16375 | 0.744318182 | | | | | 333.7667 | 0.296427294 |
| 0.1666 | 0.757272727 | | | | | | |
| 0.16945 | 0.770227273 | | | | | | |
| 0.1723 | 0.783181818 | | | | | | |
| 0.17515 | 0.796136364 | | | | | | |
| 0.178 | 0.809090909 | | | | | | |
| 0.18085 | 0.822045455 | | | | | | |
| 0.1837 | 0.835 | | | | | | |
| 0.18655 | 0.847954545 | | | | | | |
| 0.1894 | 0.860909091 | | | | | | |
| 0.19225 | 0.873863636 | | | | | | |
| 0.1951 | 0.886818182 | | | | | | |
| 0.19795 | 0.899772727 | | | | | | |
| 0.2008 | 0.912727273 | | | | | | |
| 0.20365 | 0.925681818 | | | 353.23801 | 0.254401911 | 352.35781 | 0.272102752 |
| 0.2065 | 0.938636364 | | | | | 360.60613 | 0.286195571 |
| 0.22 | 1 | 368.17438 | 0.313427287 | | | | |
| 0.24565 | | 379.34989 | 0.275201251 | 376.99522 | 0.298987984 | | |
| | | | | 387.21989 | 0.296349139 | | |
| | | | | | | 359.11541 | 0.264547647 |

Appendix 4.15: Listing of the Visual Basic routine arranging the heat transfer results data into test reports from the layout template MCRtemplate.xlt

```

Public Sub MCR_test_reports()
'Macro processing the raw data into test reports from a template
'Created by N. Miché on 06/06/05
'Last modified by N. Miché on 06/06/2005

Dim i As Integer
Dim wbd As Workbook 'workbook containing the raw data (present
workbook)
Dim wbt As Workbook 'template workbook
Dim wslabel(1 To 250) As String 'worksheet labels
Dim n As Integer
Dim rng As Range
Dim j As Integer
Dim k As Integer

'open template file
Workbooks.Open Filename:="C:\Documents and
Settings\nicolas\Desktop\Matlab_conduction\MCRtemplate.xlt",
editable:=True
'initialisation
Set wbd = Workbooks("mcrcexcel.xls")
Set wbt = Workbooks("MCRtemplate.xlt")
n = wbd.Sheets("rawdata").Cells(1, 2).Value

'program core

'For i = 1 To 25
' j = i
'For i = 26 To 50
' j = i - 25
'For i = 51 To 75
' j = i - 50
'For i = 76 To 100
' j = i - 75
'For i = 101 To 125
' j = i - 100
For i = 126 To n
j = i - 125

'create test report sheet
'copy template sheet at end of book
wbt.Activate
wbt.Worksheets("template").Copy
after:=wbt.Worksheets(Worksheets.Count)
'read worksheet label
wslabel(i) = wbd.Sheets("results").Cells(2 + (i - 1) * 40, 6).Value
'rename added worksheet with label
wbt.Sheets(j + 1).Name = wslabel(i)

'COPY TEST RESULTS TO TEMPLATE
wbt.Sheets(j + 1).Activate

'PARAM1
'build
wbd.Worksheets("results").Cells((i - 1) * 40 + 1, 2).Copy
wbt.Sheets(j + 1).Cells(2, 3).PasteSpecial Paste:=xlPasteValues,
Operation:=xlNone, SkipBlanks:=
:=False, Transpose:=False

'exp code
wbd.Worksheets("results").Cells((i - 1) * 40 + 2, 2).Copy
wbt.Sheets(j + 1).Activate
Range("C3:F3").Select
ActiveSheet.Paste

'exp date
wbd.Worksheets("results").Cells((i - 1) * 40 + 1, 6).Copy
wbt.Sheets(j + 1).Activate
Range("J2:M2").Select
ActiveSheet.Paste

'run
wbd.Worksheets("results").Cells((i - 1) * 40 + 2, 6).Copy
wbt.Sheets(j + 1).Activate
Range("J3:M3").Select
ActiveSheet.Paste

'Rez
wbd.Worksheets("results").Cells((i - 1) * 40 + 3, 2).Copy
wbt.Sheets(j + 1).Activate
Range("C5:D5").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 3, 4).Copy
wbt.Sheets(j + 1).Activate
Range("F5:G5").Select
ActiveSheet.Paste

'Ref
wbd.Worksheets("results").Cells((i - 1) * 40 + 3, 6).Copy
wbt.Sheets(j + 1).Activate
Range("J5:K5").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 3, 8).Copy
wbt.Sheets(j + 1).Activate
Range("C6:D6").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 4, 2).Copy
wbt.Sheets(j + 1).Activate
Range("F6:G6").Select
ActiveSheet.Paste

'Rot speed
wbd.Worksheets("results").Cells((i - 1) * 40 + 4, 6).Copy
wbt.Sheets(j + 1).Activate
Range("J6:K6").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 4, 8).Copy
wbt.Sheets(j + 1).Activate
Range("M6:N6").Select
ActiveSheet.Paste

'Mflow
wbd.Worksheets("results").Cells((i - 1) * 40 + 5, 2).Copy
wbt.Sheets(j + 1).Activate
Range("C7:D7").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 5, 4).Copy
wbt.Sheets(j + 1).Activate
Range("F7:G7").Select
ActiveSheet.Paste

'Pcav
wbd.Worksheets("results").Cells((i - 1) * 40 + 5, 6).Copy
wbt.Sheets(j + 1).Activate
Range("J7:K7").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 5, 8).Copy
wbt.Sheets(j + 1).Activate
Range("M7:N7").Select
ActiveSheet.Paste

'Bo*
wbd.Worksheets("results").Cells((i - 1) * 40 + 6, 2).Copy
wbt.Sheets(j + 1).Activate
Range("C8:D8").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 6, 4).Copy
wbt.Sheets(j + 1).Activate
Range("F8:G8").Select
ActiveSheet.Paste

'Pr
wbd.Worksheets("results").Cells((i - 1) * 40 + 6, 6).Copy
wbt.Sheets(j + 1).Activate
Range("J8:K8").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 6, 8).Copy
wbt.Sheets(j + 1).Activate
Range("M8:N8").Select
ActiveSheet.Paste

'Tinlet
wbd.Worksheets("results").Cells((i - 1) * 40 + 7, 2).Copy
wbt.Sheets(j + 1).Activate
Range("C9:D9").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 7, 4).Copy
wbt.Sheets(j + 1).Activate
Range("F9:G9").Select
ActiveSheet.Paste

'Texit
wbd.Worksheets("results").Cells((i - 1) * 40 + 7, 6).Copy
wbt.Sheets(j + 1).Activate
Range("J9:K9").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 7, 8).Copy
wbt.Sheets(j + 1).Activate
Range("M9:N9").Select
ActiveSheet.Paste

'Tcav2
wbd.Worksheets("results").Cells((i - 1) * 40 + 8, 2).Copy
wbt.Sheets(j + 1).Activate
Range("C10:D10").Select
ActiveSheet.Paste

```



```
wbd.Worksheets("results").Cells((i - 1) * 40 + 8, 4).Copy  
wbt.Sheets(j + 1).Activate  
Range("F10:G10").Select  
ActiveSheet.Paste
```

```
Tcav3
wbd.Worksheets("results").Cells((i - 1) * 40 + 8, 6).Copy
wbt.Sheets(j + 1).Activate
Range("J10:K10").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 8, 6).Copy
wbt.Sheets(j + 1).Activate
Range("M10:N10").Select
ActiveSheet.Paste
```

```
'DTmeads
wbd.Worksheets("results").Cells((i - 1) * 40 + 9, 2).Copy
wbt.Sheets(j + 1).Activate
Range("C11:D11").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 9, 4).Copy
wbt.Sheets(j + 1).Activate
Range("F11:G11").Select
ActiveSheet.Paste
```

```

'DTcalc
wb.Worksheets("results").Cells((i - 1) * 40 + 9, 6).Copy
wb.Sheets(j + 1).Activate
Range("J11:K11").Select
ActiveSheet.Paste
wb.Worksheets("results").Cells((i - 1) * 40 + 9, 8).Copy
wb.Sheets(j + 1).Activate
Range("M11:N11").Select
ActiveSheet.Paste

```

```

'BDTav
wbd.Worksheets("results").Cells((i - 1) * 40 + 10, 2).Copy
wbt.Sheets(j + 1).Activate
Range("C12:D12").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 10, 4).Copy
wbt.Sheets(j + 1).Activate
Range("F12:G12").Select
ActiveSheet.Paste

```

```
finBDTav  
wbd.Worksheets("results").Cells((i - 1) * 40 + 10, 6).Copy  
wb1.Sheets(j + 1).Activate  
Range("J12:K12").Select  
ActiveSheet.Paste  
wbd.Worksheets("results").Cells((i - 1) * 40 + 10, 8).Copy  
wb1.Sheets(j + 1).Activate  
Range("M12:N12").Select  
ActiveSheet.Paste
```

```
'Ostroke  
wb.Worksheets("results").Cells((i - 1) * 40 + 11, 2).Copy  
wb.Sheets(j + 1).Activate  
Range("C13:D13").Select  
ActiveSheet.Paste  
wb.Worksheets("results").Cells((i - 1) * 40 + 11, 4).Copy  
wb.Sheets(j + 1).Activate  
Range("F13:G13").Select  
ActiveSheet.Paste
```

```
finOStroke  
wbd.Worksheets("results").Cells((i - 1) * 40 + 11, 6).Copy  
wbt.Sheets(j + 1).Activate  
Range("J13:K13").Select  
ActiveSheet.Paste  
wbd.Worksheets("results").Cells((i - 1) * 40 + 11, 8).Copy  
wbt.Sheets(j + 1).Activate  
Range("M13:N13").Select  
ActiveSheet.Paste
```

```
TsC2dav
wbd.Worksheets("results").Cells((i - 1) * 40 + 12, 2).Copy
wbt.Sheets(j + 1).Activate
Range("C14:D14").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 12, 4).Copy
wbt.Sheets(j + 1).Activate
Range("F14:G14").Select
ActiveSheet.Paste
```

```
TsC3uav
wbd.Worksheets("results").Cells((i - 1) * 40 + 12, 6).Copy
wbt.Sheets(j + 1).Activate
Range("J14:K14").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 12, 8).Copy
wbt.Sheets(j + 1).Activate
Range("M14:N14").Select
ActiveSheet.Paste
```

```
"TsC3dav  
wbd.Worksheets("results").Cells((i - 1) * 40 + 13, 2).Copy  
wbt.Sheets(j + 1).Activate  
Range("C15:D15").Select  
ActiveSheet.Paste  
wbd.Worksheets("results").Cells((i - 1) * 40 + 13, 4).Copy
```

```
wbt.Sheets(j + 1).Activate  
Range("F15:G15").Select  
ActiveSheet.Paste
```

```
TsC4uav  
wbd.Worksheets("results").Cells((i - 1) * 40 + 13, 6).Copy  
wb1.Sheets(j + 1).Activate  
Range("J15:K15").Select  
ActiveSheet.Paste  
wbd.Worksheets("results").Cells((i - 1) * 40 + 13, 8).Copy  
wb1.Sheets(j + 1).Activate  
Range("M15:N15").Select  
ActiveSheet.Paste
```

```

FinTsD2av
wbd.Worksheets("results").Cells((i - 1) * 40 + 14, 2).Copy
wbt.Sheets(j + 1).Activate
Range("C16:D16").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 14, 4).Copy
wbt.Sheets(j + 1).Activate
Range("F16:G16").Select
ActiveSheet.Paste

```

```

finTsD3av
wbd.Worksheets("results").Cells((i - 1) * 40 + 14, 6).Copy
wbt.Sheets(j + 1).Activate
Range("J16:K16").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 14, 8).Copy
wbt.Sheets(j + 1).Activate
Range("M16:N16").Select
ActiveSheet.Paste

```

```
Nuc2dav
wbd.Worksheets("results").Cells((i - 1) * 40 + 15, 2).Copy
wbt.Sheets(j + 1).Activate
Range("C17:D17").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 15, 4).Copy
wbt.Sheets(j + 1).Activate
Range("F17:G17").Select
ActiveSheet.Paste
```

```
NuC3uav  
wbd.Worksheets("results").Cells((i - 1) * 40 + 15, 6).Copy  
wb1.Sheets(j + 1).Activate  
Range("J17:K17").Select  
ActiveSheet.Paste  
wbd.Worksheets("results").Cells((i - 1) * 40 + 15, 8).Copy  
wb1.Sheets(j + 1).Activate  
Range("M17:N17").Select  
ActiveSheet.Paste
```

```
NuC3dav
wbd.Worksheets("results").Cells((i - 1) * 40 + 16, 2).Copy
wbt.Sheets(j + 1).Activate
Range("C18:D18").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 16, 4).Copy
wbt.Sheets(j + 1).Activate
Range("F18:G18").Select
ActiveSheet.Paste
```

```
NuC4uav
wbd.Worksheets("results").Cells((i - 1) * 40 + 6, 8).Copy
wbt.Sheets(j + 1).Activate
Range("J18:K18").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 16, 8).Copy
wbt.Sheets(j + 1).Activate
Range("M18:N18").Select
ActiveSheet.Paste
```

```
finNuD2av
wbd.Worksheets("results").Cells((i - 1) * 40 + 17, 2).Copy
wbt.Sheets(j + 1).Activate
Range("C19:D19").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 17, 4).Copy
wbt.Sheets(j + 1).Activate
Range("F19:G19").Select
ActiveSheet.Paste
```

```
finNuD3av
wbd.Worksheets("results").Cells((i - 1) * 40 + 17, 6).Copy
wbt.Sheets(j + 1).Activate
Range("J19:K19").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 17, 8).Copy
wbt.Sheets(j + 1).Activate
Range("M19:N19").Select
ActiveSheet.Paste
```

```
TshC2
wbd.Worksheets("results").Cells((i - 1) * 40 + 18, 2).Copy
wbt.Sheets(j + 1).Activate
Range("C20:D20").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 18, 4).Copy
wbt.Sheets(j + 1).Activate
```

```

Range("F20:G20").Select
ActiveSheet.Paste

'TshC3
wbd.Worksheets("results").Cells((i - 1) * 40 + 18, 6).Copy
wbt.Sheets(j + 1).Activate
Range("J20:K20").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 18, 8).Copy
wbt.Sheets(j + 1).Activate
Range("M20:N20").Select
ActiveSheet.Paste

'NushC2
wbd.Worksheets("results").Cells((i - 1) * 40 + 19, 2).Copy
wbt.Sheets(j + 1).Activate
Range("C21:D21").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 19, 4).Copy
wbt.Sheets(j + 1).Activate
Range("F21:G21").Select
ActiveSheet.Paste

'NushC3
wbd.Worksheets("results").Cells((i - 1) * 40 + 19, 6).Copy
wbt.Sheets(j + 1).Activate
Range("J21:K21").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 19, 8).Copy
wbt.Sheets(j + 1).Activate
Range("M21:N21").Select
ActiveSheet.Paste

'GrshC2
wbd.Worksheets("results").Cells((i - 1) * 40 + 20, 2).Copy
wbt.Sheets(j + 1).Activate
Range("C22:D22").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 20, 4).Copy
wbt.Sheets(j + 1).Activate
Range("F22:G22").Select
ActiveSheet.Paste

'GrshC3
wbd.Worksheets("results").Cells((i - 1) * 40 + 20, 6).Copy
wbt.Sheets(j + 1).Activate
Range("J22:K22").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 20, 8).Copy
wbt.Sheets(j + 1).Activate
Range("M22:N22").Select
ActiveSheet.Paste

'Theta 2
wbd.Worksheets("results").Cells((i - 1) * 40 + 21, 2).Copy
wbt.Sheets(j + 1).Activate
Range("C23:D23").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 21, 4).Copy
wbt.Sheets(j + 1).Activate
Range("F23:G23").Select
ActiveSheet.Paste

'Theta 3
wbd.Worksheets("results").Cells((i - 1) * 40 + 21, 6).Copy
wbt.Sheets(j + 1).Activate
Range("J23:K23").Select
ActiveSheet.Paste
wbd.Worksheets("results").Cells((i - 1) * 40 + 21, 8).Copy
wbt.Sheets(j + 1).Activate
Range("M23:N23").Select
ActiveSheet.Paste

'PARAM2

'page 3 report

wbd.Worksheets("results").Activate
With ActiveSheet
Set rng = .Range(Cells(((i - 1) * 40 + 2), 12), Cells(((i - 1) * 40 + 2),
13))
rng.Copy
wbt.Sheets(j + 1).Range("C58").PasteSpecial
Paste:=xlPasteValues, Operation:=xlNone, SkipBlanks _
:=False, Transpose:=False
End With

wbd.Worksheets("results").Activate
With ActiveSheet
Set rng = .Range(Cells(((i - 1) * 40 + 37), 12), Cells(((i - 1) * 40 +
38), 13))
rng.Copy
wbt.Sheets(j + 1).Range("C93").PasteSpecial
Paste:=xlPasteValues, Operation:=xlNone, SkipBlanks _
:=False, Transpose:=False
End With

wbd.Worksheets("results").Cells((i - 1) * 40 + 2, 14).Copy
wbt.Sheets(j + 1).Activate
Range("E58:F58").Select
ActiveSheet.Paste

wbd.Worksheets("results").Cells((i - 1) * 40 + 37, 14).Copy
wbt.Sheets(j + 1).Activate
Range("E93:F93").Select
ActiveSheet.Paste

wbd.Worksheets("results").Cells((i - 1) * 40 + 38, 14).Copy
wbt.Sheets(j + 1).Activate
Range("E94:F94").Select
ActiveSheet.Paste

wbd.Worksheets("results").Cells((i - 1) * 40 + 2, 15).Copy
wbt.Sheets(j + 1).Cells(58, 7).PasteSpecial Paste:=xlPasteValues,
Operation:=xlNone, SkipBlanks _
:=False, Transpose:=False

wbd.Worksheets("results").Cells((i - 1) * 40 + 37, 15).Copy
wbt.Sheets(j + 1).Cells(93, 7).PasteSpecial Paste:=xlPasteValues,
Operation:=xlNone, SkipBlanks _
:=False, Transpose:=False

wbd.Worksheets("results").Cells((i - 1) * 40 + 38, 15).Copy
wbt.Sheets(j + 1).Cells(94, 7).PasteSpecial Paste:=xlPasteValues,
Operation:=xlNone, SkipBlanks _
:=False, Transpose:=False

wbd.Worksheets("results").Activate
With ActiveSheet
Set rng = .Range(Cells(((i - 1) * 40 + 3), 16), Cells(((i - 1) * 40 + 3),
17))
rng.Copy
wbt.Sheets(j + 1).Range("H59").PasteSpecial
Paste:=xlPasteValues, Operation:=xlNone, SkipBlanks _
:=False, Transpose:=False
End With

wbd.Worksheets("results").Activate
With ActiveSheet
Set rng = .Range(Cells(((i - 1) * 40 + 5), 16), Cells(((i - 1) * 40 + 5),
19))
rng.Copy
wbt.Sheets(j + 1).Range("H61").PasteSpecial
Paste:=xlPasteValues, Operation:=xlNone, SkipBlanks _
:=False, Transpose:=False
End With

wbd.Worksheets("results").Activate
With ActiveSheet
Set rng = .Range(Cells(((i - 1) * 40 + 20), 16), Cells(((i - 1) * 40 +
20), 19))
rng.Copy
wbt.Sheets(j + 1).Range("H76").PasteSpecial
Paste:=xlPasteValues, Operation:=xlNone, SkipBlanks _
:=False, Transpose:=False
End With

wbd.Worksheets("results").Activate
With ActiveSheet
Set rng = .Range(Cells(((i - 1) * 40 + 36), 16), Cells(((i - 1) * 40 +
36), 19))
rng.Copy
wbt.Sheets(j + 1).Range("H92").PasteSpecial
Paste:=xlPasteValues, Operation:=xlNone, SkipBlanks _
:=False, Transpose:=False
End With

wbd.Worksheets("results").Activate
With ActiveSheet
Set rng = .Range(Cells(((i - 1) * 40 + 4), 18), Cells(((i - 1) * 40 + 4),
19))
rng.Copy
wbt.Sheets(j + 1).Range("J60").PasteSpecial Paste:=xlPasteValues,
Operation:=xlNone, SkipBlanks _
:=False, Transpose:=False
End With

wbd.Worksheets("results").Cells((i - 1) * 40 + 3, 20).Copy
wbt.Sheets(j + 1).Activate
Range("L59:M59").Select
ActiveSheet.Paste

wbd.Worksheets("results").Cells((i - 1) * 40 + 5, 20).Copy
wbt.Sheets(j + 1).Activate
Range("L61:M61").Select
ActiveSheet.Paste

wbd.Worksheets("results").Cells((i - 1) * 40 + 20, 20).Copy
wbt.Sheets(j + 1).Activate
Range("L76:M76").Select
ActiveSheet.Paste

wbd.Worksheets("results").Cells((i - 1) * 40 + 36, 20).Copy
wbt.Sheets(j + 1).Activate
Range("L92:M92").Select
ActiveSheet.Paste

wbd.Worksheets("results").Activate

```

```

With ActiveSheet
Set rng = .Range(Cells(((i - 1) * 40 + 3), 21), Cells(((i - 1) * 40 + 3), 23))
    rng.Copy
    wbt.Sheets(j + 1).Range("N59").PasteSpecial
Paste:=xlPasteValues, Operation:=xlNone, SkipBlanks _
:=False, Transpose:=False
End With

    wbd.Worksheets("results").Activate
    With ActiveSheet
    Set rng = .Range(Cells(((i - 1) * 40 + 5), 21), Cells(((i - 1) * 40 + 5), 23))
        rng.Copy
        wbt.Sheets(j + 1).Range("N61").PasteSpecial
Paste:=xlPasteValues, Operation:=xlNone, SkipBlanks _
:=False, Transpose:=False
End With

        wbd.Worksheets("results").Activate
        With ActiveSheet
        Set rng = .Range(Cells(((i - 1) * 40 + 20), 21), Cells(((i - 1) * 40 + 20), 23))
            rng.Copy
            wbt.Sheets(j + 1).Range("N76").PasteSpecial
Paste:=xlPasteValues, Operation:=xlNone, SkipBlanks _
:=False, Transpose:=False
End With

            wbd.Worksheets("results").Activate
            With ActiveSheet
            Set rng = .Range(Cells(((i - 1) * 40 + 36), 21), Cells(((i - 1) * 40 + 36), 23))
                rng.Copy
                wbt.Sheets(j + 1).Range("N92").PasteSpecial
Paste:=xlPasteValues, Operation:=xlNone, SkipBlanks _
:=False, Transpose:=False
End With

                wbd.Worksheets("results").Activate
                With ActiveSheet
                Set rng = .Range(Cells(((i - 1) * 40 + 6), 24), Cells(((i - 1) * 40 + 6), 24))
                    rng.Copy
                    wbt.Sheets(j + 1).Range("Q62").PasteSpecial
Paste:=xlPasteValues, Operation:=xlNone, SkipBlanks _
:=False, Transpose:=False
End With

                    wbd.Worksheets("results").Activate
                    With ActiveSheet
                    Set rng = .Range(Cells(((i - 1) * 40 + 21), 24), Cells(((i - 1) * 40 + 21), 24))
                        rng.Copy
                        wbt.Sheets(j + 1).Range("Q77").PasteSpecial
Paste:=xlPasteValues, Operation:=xlNone, SkipBlanks _
:=False, Transpose:=False
End With

'page4 report

        wbd.Worksheets("results").Activate
        With ActiveSheet
        Set rng = .Range(Cells(((i - 1) * 40 + 6), 36), Cells(((i - 1) * 40 + 6), 37))
            rng.Copy
            wbt.Sheets(j + 1).Range("C103").PasteSpecial
Paste:=xlPasteValues, Operation:=xlNone, SkipBlanks _
:=False, Transpose:=False
End With

            wbd.Worksheets("results").Activate
            With ActiveSheet
            Set rng = .Range(Cells(((i - 1) * 40 + 21), 36), Cells(((i - 1) * 40 + 21), 37))
                rng.Copy
                wbt.Sheets(j + 1).Range("C118").PasteSpecial
Paste:=xlPasteValues, Operation:=xlNone, SkipBlanks _
:=False, Transpose:=False
End With

                wbd.Worksheets("results").Cells((i - 1) * 40 + 6, 38).Copy
                wbt.Sheets(j + 1).Activate
                Range("E103:F103").Select
                ActiveSheet.Paste

                wbd.Worksheets("results").Cells((i - 1) * 40 + 7, 38).Copy
                wbt.Sheets(j + 1).Activate
                Range("E104:F104").Select
                ActiveSheet.Paste

                wbd.Worksheets("results").Cells((i - 1) * 40 + 8, 38).Copy
                wbt.Sheets(j + 1).Activate
                Range("E105:F105").Select
                ActiveSheet.Paste

                wbd.Worksheets("results").Cells((i - 1) * 40 + 9, 38).Copy
                wbt.Sheets(j + 1).Activate
                Range("E106:F106").Select
                ActiveSheet.Paste

```

wbd.Worksheets("results").Cells((i - 1) * 40 + 10, 38).Copy
wbt.Sheets(j + 1).Activate
Range("E107:F107").Select
ActiveSheet.Paste

wbd.Worksheets("results").Cells((i - 1) * 40 + 11, 38).Copy
wbt.Sheets(j + 1).Activate
Range("E108:F108").Select
ActiveSheet.Paste

wbd.Worksheets("results").Cells((i - 1) * 40 + 12, 38).Copy
wbt.Sheets(j + 1).Activate
Range("E109:F109").Select
ActiveSheet.Paste

wbd.Worksheets("results").Cells((i - 1) * 40 + 13, 38).Copy
wbt.Sheets(j + 1).Activate
Range("E110:F110").Select
ActiveSheet.Paste

wbd.Worksheets("results").Cells((i - 1) * 40 + 14, 38).Copy
wbt.Sheets(j + 1).Activate
Range("E111:F111").Select
ActiveSheet.Paste

wbd.Worksheets("results").Cells((i - 1) * 40 + 15, 38).Copy
wbt.Sheets(j + 1).Activate
Range("E112:F112").Select
ActiveSheet.Paste

wbd.Worksheets("results").Cells((i - 1) * 40 + 16, 38).Copy
wbt.Sheets(j + 1).Activate
Range("E113:F113").Select
ActiveSheet.Paste

wbd.Worksheets("results").Cells((i - 1) * 40 + 17, 38).Copy
wbt.Sheets(j + 1).Activate
Range("E114:F114").Select
ActiveSheet.Paste

wbd.Worksheets("results").Cells((i - 1) * 40 + 18, 38).Copy
wbt.Sheets(j + 1).Activate
Range("E115:F115").Select
ActiveSheet.Paste

wbd.Worksheets("results").Cells((i - 1) * 40 + 19, 38).Copy
wbt.Sheets(j + 1).Activate
Range("E116:F116").Select
ActiveSheet.Paste

wbd.Worksheets("results").Cells((i - 1) * 40 + 21, 38).Copy
wbt.Sheets(j + 1).Activate
Range("E118:F118").Select
ActiveSheet.Paste

wbd.Worksheets("results").Cells((i - 1) * 40 + 22, 38).Copy
wbt.Sheets(j + 1).Activate
Range("E119:F119").Select
ActiveSheet.Paste

wbd.Worksheets("results").Cells((i - 1) * 40 + 23, 38).Copy
wbt.Sheets(j + 1).Activate
Range("E120:F120").Select
ActiveSheet.Paste

wbd.Worksheets("results").Cells((i - 1) * 40 + 24, 38).Copy
wbt.Sheets(j + 1).Activate
Range("E121:F121").Select
ActiveSheet.Paste

wbd.Worksheets("results").Cells((i - 1) * 40 + 25, 38).Copy
wbt.Sheets(j + 1).Activate
Range("E122:F122").Select
ActiveSheet.Paste

wbd.Worksheets("results").Cells((i - 1) * 40 + 26, 38).Copy
wbt.Sheets(j + 1).Activate
Range("E123:F123").Select
ActiveSheet.Paste

wbd.Worksheets("results").Cells((i - 1) * 40 + 27, 38).Copy
wbt.Sheets(j + 1).Activate
Range("E124:F124").Select
ActiveSheet.Paste

wbd.Worksheets("results").Cells((i - 1) * 40 + 28, 38).Copy
wbt.Sheets(j + 1).Activate
Range("E125:F125").Select
ActiveSheet.Paste

wbd.Worksheets("results").Cells((i - 1) * 40 + 29, 38).Copy
wbt.Sheets(j + 1).Activate
Range("E126:F126").Select
ActiveSheet.Paste

wbd.Worksheets("results").Cells((i - 1) * 40 + 30, 38).Copy
wbt.Sheets(j + 1).Activate
Range("E127:F127").Select
ActiveSheet.Paste

43))

59)

```

        Set rng = .Range(Cells(((i - 1) * 40 + 21), 45), Cells(((i - 1) * 40 +
35), 59))
        rng.Copy
        wbt.Sheets(j + 1).Range("N118").PasteSpecial
        Paste:=xlPasteValues, Operation:=xlNone, SkipBlanks _
            :=False, Transpose:=False
        End With
    Next i

```

```

'Create results workbook
wbt.SaveAs Filename:="C:\Documents and
Settings\nicolas\Desktop\Matlab_conduction\MCR_HT-reports6.xls",
    FileFormat:=xlNormal, Password:="", WriteResPassword:="",
    ReadOnlyRecommended:=False, CreateBackup:=False

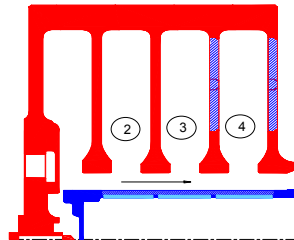
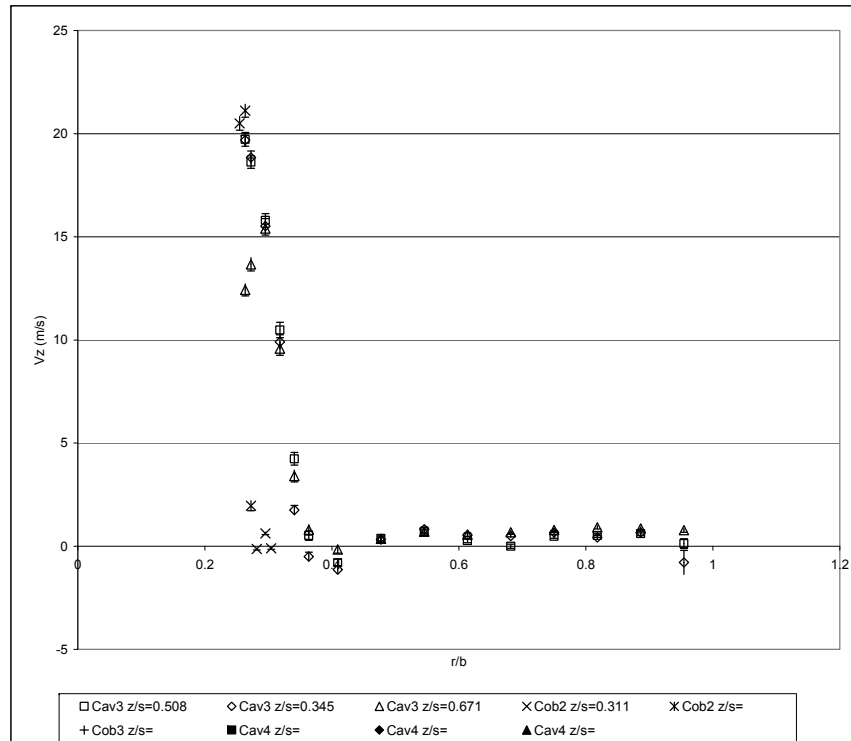
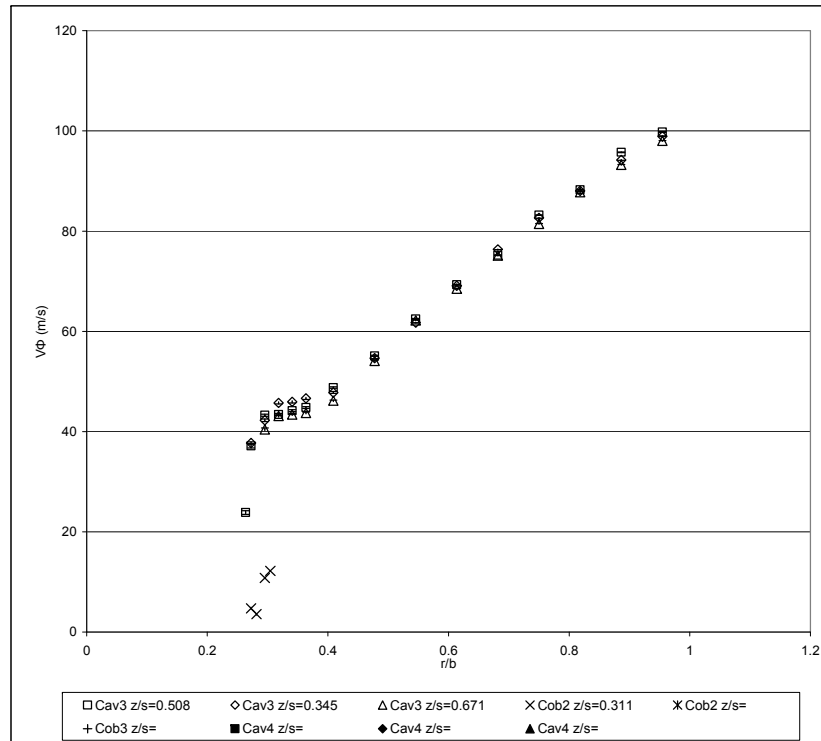
```

```
End Sub
```

Appendix 4.16: Example of LDA test report

LDA measurement**Build:** 3**LDA test # :** B3run28-1**Heat Transfer run:** B3run28-1**LDA code:** 101**Re Φ :** 4.12E+06**Rez:** 1.16E+05**Ro:** 0.54**rpm:** 4552.2**Date:** 12/11/2003**raw LDA file:** \MCRLDA\Build3\B3run28-1.lda**LDA list file:** \MCRLDA\Build3\B3run28-1.xls**Perfd by:** NM**LDA1:** V Φ **LDA2:** Vz**W (m/s):** 17.88**Printed:** 16/09/2008**Comments:** run OK**Traverse ref:**

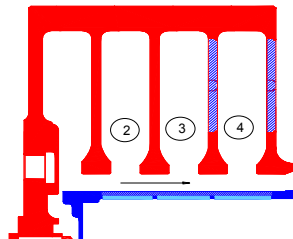
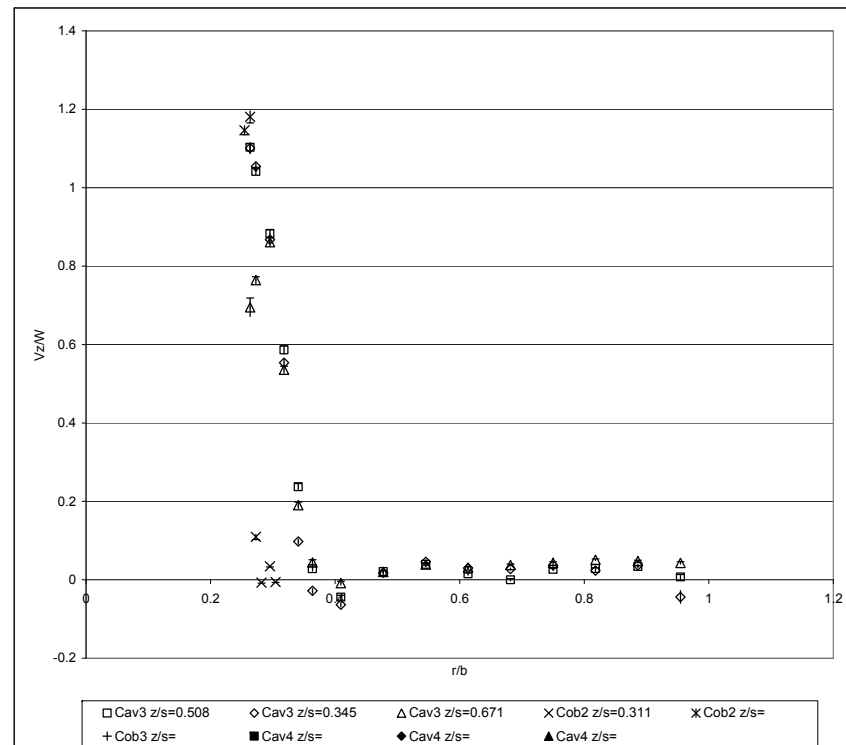
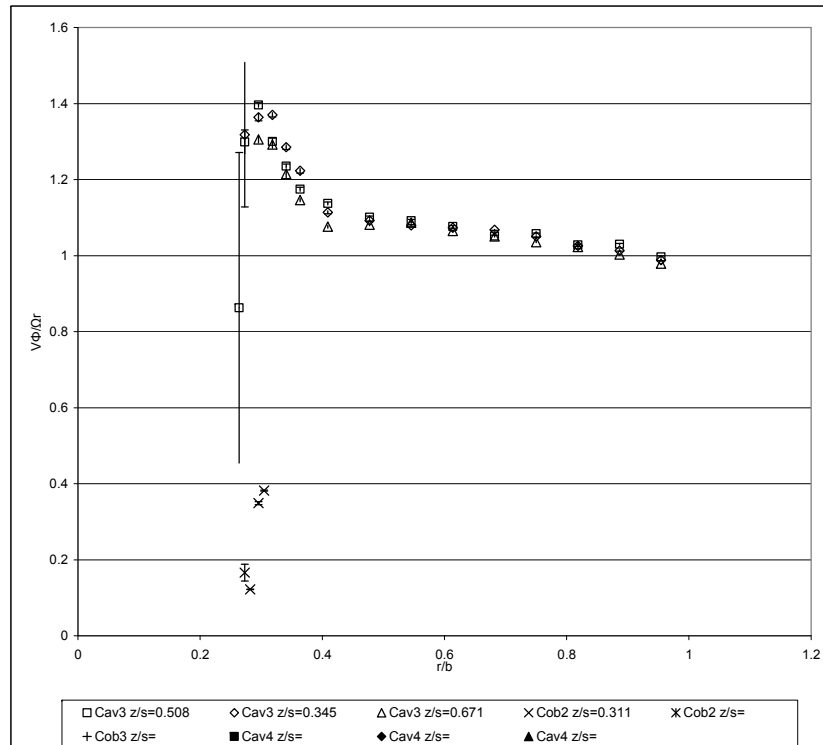
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LDA measurement**Build:** 3**LDA code:** 101**Re_φ:** 4.12E+06**rpm:** 4552.2**Date:** 12/11/2003**Perfd by:** NM

Printed: 16/09/2008

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LDA1: Vφ**Comments:** run OK**LDA test # :** B3run28-1**Re_z:** 1.16E+05**raw LDA file:** \MCRLDA\Build3\B3run28-1.lda**LDA2:** Vz**Traverse corresp:** 0**Heat Transfer run:** B3run28-1**Ro:** 0.54**LDA list file:** \MCRLDA\Build3\B3run28-1.xls**W (m/s):** 17.88

| X [mm] | corrected d X | Y [mm] | corrected Y / Radius (mm) | Count{1} | Count{2} | Data Rate{1} [#/s] | Data Rate{2} [#/s] | Validation n(1) [%] | Validation (2) [%] | LDA1 Mean [m/s] | LDA2 Mean [m/s] | LDA1 RMS [m/s] | LDA2 RMS [m/s] | LDA1 MeanConf [m/s] | LDA2 MeanConf [m/s] | LDA1 RMSConf [m/s] | LDA2 RMSConf [m/s] | Cavity | r/lb | z/s | Ωr (m/s) | V ₀ /Ωr | V ₀ rms/Ωr | V _z rms/W | V _z W/W | | |
|--------|------------------|--------|------------------------------|----------|----------|-----------------------|-----------------------|------------------------|-----------------------|-----------------------|-----------------------|----------------------|----------------------|---------------------------|---------------------------|--------------------------|--------------------------|--------|-------|-------|----------|--------------------|-----------------------|----------------------|--------------------|------|----------|
| | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| 64 | | 140 | 210 | 5000 | 5000 | 10221.0 | 3203.8 | | | 99.79 | 0.13 | 9.62 | 4.21 | 0.27 | 0.12 | 0.19 | 0.08 | 3 | 0.955 | 0.508 | 100.11 | 1.00 | 2.66E-03 | 0.10 | 0.01 | 0.24 | 6.53E-03 |
| 64 | | 125 | 195 | 5000 | 5000 | 12325.4 | 4253.1 | | | 95.76 | 0.61 | 6.38 | 1.65 | 0.18 | 0.05 | 0.13 | 0.03 | 3 | 0.886 | 0.508 | 92.96 | 1.03 | 1.90E-03 | 0.07 | 0.03 | 0.09 | 2.57E-03 |
| 64 | | 110 | 180 | 5000 | 5000 | 13403.7 | 5287.8 | | | 88.24 | 0.51 | 4.52 | 1.51 | 0.13 | 0.04 | 0.09 | 0.03 | 3 | 0.818 | 0.508 | 85.81 | 1.03 | 1.46E-03 | 0.05 | 0.03 | 0.08 | 2.34E-03 |
| 64 | | 95 | 165 | 5000 | 5000 | 13316.7 | 5092.9 | | | 83.22 | 0.48 | 4.29 | 1.65 | 0.12 | 0.05 | 0.08 | 0.03 | 3 | 0.750 | 0.508 | 78.66 | 1.06 | 1.51E-03 | 0.05 | 0.03 | 0.09 | 2.56E-03 |
| 64 | | 80 | 150 | 5000 | 4941 | 28626.4 | 82.4 | | | 75.20 | 0.00 | 5.08 | 1.78 | 0.14 | 0.05 | 0.10 | 0.04 | 3 | 0.682 | 0.508 | 71.51 | 1.05 | 1.97E-03 | 0.07 | 0.00 | 0.10 | 2.77E-03 |
| 64 | | 65 | 135 | 5000 | 5000 | 28517.4 | 234.1 | | | 69.30 | 0.26 | 5.45 | 1.76 | 0.15 | 0.05 | 0.11 | 0.03 | 3 | 0.614 | 0.508 | 64.36 | 1.08 | 2.35E-03 | 0.08 | 0.01 | 0.10 | 2.72E-03 |
| 64 | | 50 | 120 | 5000 | 5000 | 12005.9 | 596.4 | | | 62.47 | 0.71 | 6.98 | 2.02 | 0.19 | 0.06 | 0.14 | 0.04 | 3 | 0.545 | 0.508 | 57.20 | 1.09 | 3.38E-03 | 0.12 | 0.04 | 0.11 | 3.13E-03 |
| 64 | | 35 | 105 | 5000 | 5000 | 396.0 | 1118.5 | | | 55.12 | 0.38 | 5.09 | 2.21 | 0.14 | 0.06 | 0.10 | 0.04 | 3 | 0.477 | 0.508 | 50.05 | 1.10 | 2.82E-03 | 0.10 | 0.02 | 0.12 | 3.43E-03 |
| 71 | | 140 | 210 | 5000 | 5000 | 3371.3 | 7047.3 | | | 98.93 | 0.79 | 5.22 | 10.37 | 0.14 | 0.29 | 0.10 | 0.20 | 3 | 0.955 | 0.345 | 100.11 | 0.99 | 1.44E-03 | 0.05 | -0.04 | 0.58 | 1.61E-02 |
| 71 | | 125 | 195 | 5000 | 5000 | 3076.8 | 6422.7 | | | 94.19 | 0.66 | 4.98 | 1.66 | 0.14 | 0.05 | 0.10 | 0.03 | 3 | 0.886 | 0.345 | 92.96 | 1.01 | 1.48E-03 | 0.05 | 0.04 | 0.09 | 2.57E-03 |
| 71 | | 110 | 180 | 5000 | 5000 | 2985.2 | 5562.2 | | | 88.02 | 0.42 | 5.04 | 1.69 | 0.14 | 0.05 | 0.10 | 0.03 | 3 | 0.818 | 0.345 | 85.81 | 1.03 | 1.63E-03 | 0.06 | 0.02 | 0.09 | 2.62E-03 |
| 71 | | 95 | 165 | 5000 | 5000 | 3270.3 | 7008.6 | | | 82.64 | 0.66 | 4.27 | 1.79 | 0.12 | 0.05 | 0.08 | 0.04 | 3 | 0.750 | 0.345 | 78.66 | 1.05 | 1.50E-03 | 0.05 | 0.04 | 0.10 | 2.77E-03 |
| 71 | | 80 | 150 | 5000 | 5000 | 3670.7 | 8791.2 | | | 76.35 | 0.49 | 3.34 | 1.56 | 0.09 | 0.04 | 0.07 | 0.03 | 3 | 0.682 | 0.345 | 71.51 | 1.07 | 1.30E-03 | 0.05 | 0.03 | 0.09 | 2.42E-03 |
| 71 | | 65 | 135 | 5000 | 5000 | 3320.9 | 8138.1 | | | 69.08 | 0.54 | 2.91 | 1.74 | 0.08 | 0.05 | 0.06 | 0.03 | 3 | 0.614 | 0.345 | 64.36 | 1.07 | 1.25E-03 | 0.05 | 0.03 | 0.10 | 2.70E-03 |
| 57 | | 140 | 210 | 5000 | 5000 | 1064.4 | 1723.1 | | | 98.02 | 0.77 | 5.25 | 1.92 | 0.15 | 0.05 | 0.10 | 0 | | | | | | | | | | |

Appendix 4.17: Listing of avenoco.bas

```

100 REM Program to average tangential velocity
    measurements over each pass of a window
101 REM Christopher Long 8th December 2004
102 REM Modified on 8th July to read from new files
    (corrected for probe misalignment)
103 REM Also does batch processing of up to 100
    different files
104 REM Modified to read from non-coincident data
    files 23rd August 2005
200 DIM at(500), lda1(500), lda2(500), pt(10), sp(10)
205 DIM xval4(500), xval6(500)
210 DIM infile$(100), outfile$(100), xprobe(100),
    yprobe(100)

211 infile$(1) = "E:\SPECTRA\R44\R44_5.TXT"
212 infile$(2) = "E:\SPECTRA\R44\R44_14.TXT"
213 infile$(3) = "E:\SPECTRA\R44\R44_23.TXT"
214 infile$(4) = "E:\SPECTRA\R44\R44_32.TXT"
215 infile$(5) = "E:\SPECTRA\R44\R44_41.TXT"
216 infile$(6) = "E:\SPECTRA\R44\R44_50.TXT"
REM 217 infile$(7) = "D:\CAL\MCAV\VR\Run38-
1\007.TXT"
REM 218 infile$(8) = "D:\CAL\MCAV\VR\Run38-
1\008.TXT"
REM 219 infile$(9) = "D:\CAL\MCAV\VR\Run38-
1\009.TXT"
REM 220 infile$(10) = "D:\CAL\MCAV\VR\Run38-
1\00a.TXT"
REM 221 infile$(11) = "D:\CAL\MCAV\VR\Run38-
1\00b.TXT"

251 outfile$(1) = "E:\SPECTRA\R44\P44_5.TXT"
252 outfile$(2) = "E:\SPECTRA\R44\P44_14.TXT"
253 outfile$(3) = "E:\SPECTRA\R44\P44_23.TXT"
254 outfile$(4) = "E:\SPECTRA\R44\P44_32.TXT"
255 outfile$(5) = "E:\SPECTRA\R44\P44_41.TXT"
256 outfile$(6) = "E:\SPECTRA\R44\P44_50.TXT"
REM 257 outfile$(7) = "D:\CAL\MCAV\VR\Run38-
1\007.DAT"
REM 258 outfile$(8) = "D:\CAL\MCAV\VR\Run38-
1\008.DAT"
REM 259 outfile$(9) = "D:\CAL\MCAV\VR\Run38-
1\009.DAT"
REM 260 outfile$(10) = "D:\CAL\MCAV\VR\Run38-
1\00a.DAT"
REM 261 outfile$(11) = "D:\CAL\MCAV\VR\Run38-
1\00b.DAT"

270 nfiles = 6

1300 PRINT "Enter the rotational speed for this series
of tests"
1310 INPUT revmin
1305 REM tau is the time period between window
    passes [in ms]
1320 tau = (60! / (revmin * 4!)) * 1000!

1345 REM **** NOTE ****
1346 REM Program requires correct filepath name
1347 FOR ifile = 1 TO nfiles
1350 OPEN infile$(ifile) FOR INPUT AS #1
1360 OPEN outfile$(ifile) FOR OUTPUT AS #2
1370 PRINT #2, "Time (ms)", " ", "Vphi(m / s)", " ",
    "Npts"
1400 LINE INPUT #1, line1$
3400 a$ = before$ + after$
3500 x4 = VAL(a$)
3999 RETURN

4000 REM Get sixth variable (tangential velocity)
4050 c$ = " "
4100 blen = pt(6) - sp(5) - 1

1410 LINE INPUT #1, line2$
1420 LINE INPUT #1, line3$
1430 LINE INPUT #1, line4$
1440 LINE INPUT #1, line5$

1500 kline = 0: irec = 0: fileline = 0

1502 IF EOF(1) THEN
1503 CLOSE 1: GOTO 1995
1504 END IF

1505 fileline = fileline + 1
1506 irec = irec + 1
1510 LINE INPUT #1, in$
1550 GOSUB 2000
1560 GOSUB 3000
1570 GOSUB 4000

1590 xval4(irec) = x4
1595 xval6(irec) = x6
1600 IF fileline <> 1 THEN
1605 tdiff = xval4(irec) - xval4(irec - 1)
1609 END IF
1620 IF tdiff > tau / 4! THEN
1625 Npts = irec - 1
1630 GOSUB 5000
1632 PRINT #2, tave / 1000, CHR$(9), vphiave,
    CHR$(9), Npts
1635 irec = 1
1660 END IF

1990 GOTO 1502
1995 CLOSE #2: NEXT ifile
1999 STOP

2000 REM Does character handling bit
2001 lenin = LEN(in$)
2100 ipt = 0: isp = 0

2150 FOR i = 1 TO 10
2170 pt(i) = 0
2180 sp(i) = 0
2190 NEXT i

2200 FOR i = 1 TO lenin
2300 IF MID$(in$, i, 1) = "." THEN
2450 ipt = ipt + 1: pt(ipt) = i
2570 END IF
2600 IF MID$(in$, i, 1) = CHR$(9) THEN
2620 isp = isp + 1: sp(isp) = i
2650 END IF

2700 NEXT i

2800 REM PRINT pt(1), pt(2), pt(3), pt(4), pt(5), pt(6)
2950 REM PRINT sp(1), sp(2), sp(3), sp(4), sp(5),
    sp(6)
2999 RETURN

3000 REM Get fourth variable (arrival time 2)
3050 blen = pt(4) - sp(3) - 1
3100 before$ = MID$(in$, sp(3) + 1, blen)
3200 after$ = MID$(in$, pt(4), 4)
3300 a$ = " "

```

```

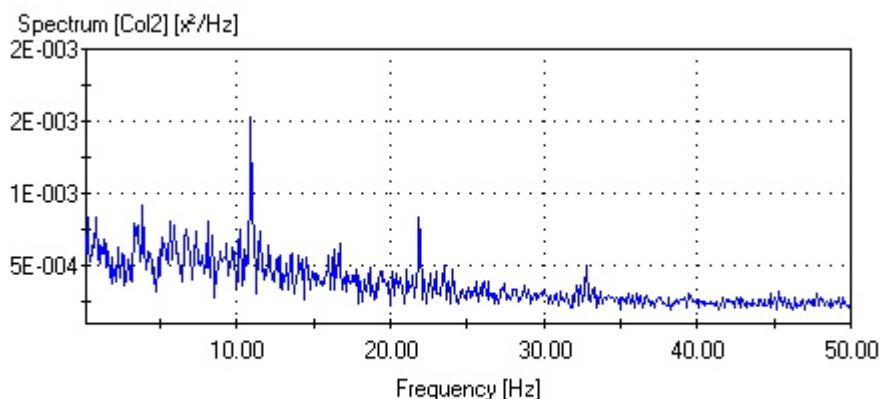
4200 before$ = MID$(in$, sp(5) + 1, blen)
4300 after$ = MID$(in$, pt(6), 3)
4400 c$ = before$ + after$
4500 x6 = VAL(c$)
4999 RETURN
5000 REM Do averaging on npts of data, transfer last point into new data array
5005 sumt = 0!: sumvphi = 0!
5010 FOR idata = 1 TO Npts
5020 sumt = sumt + xval4(idata)
5026 sumvphi = sumvphi + xval6(idata)
5030 tave = sumt / Npts: vphiave = sumvphi / Npts
5050 NEXT idata
5070 xval4(1) = xval4(Npts + 1)
5072 xval6(1) = xval6(Npts + 1)
5999 RETURN

9001 RETURN

```

Appendix 4.18: Example of tangential velocity power spectrum plot and output file

Plot



Project: SPECT_R38_pos05 - **Pos:** 0.00;0.00;0.00 - **Date/Time:** #N/A

List

| | | | | | | |
|-------------------------|----|------|--------|-----|-------|--------|
| DXEX v1 | 36 | 3.52 | 0.0008 | 78 | 7.62 | 0.0005 |
| Untitled | 37 | 3.61 | 0.0008 | 79 | 7.71 | 0.0006 |
| No date/time available. | 38 | 3.71 | 0.0005 | 80 | 7.81 | 0.0006 |
| 0.00 mm;0.00 mm;0.00 mm | 39 | 3.81 | 0.0006 | 81 | 7.91 | 0.0004 |
| "Row#" "Frequency [Hz]" | 40 | 3.91 | 0.0009 | 82 | 8.01 | 0.0005 |
| "Spectrum [Col3]" | 41 | 4.00 | 0.0006 | 83 | 8.11 | 0.0008 |
| [x2/Hz]" | 42 | 4.10 | 0.0005 | 84 | 8.20 | 0.0005 |
| 1 0.10 0.001 | 43 | 4.20 | 0.0006 | 85 | 8.30 | 0.0004 |
| 2 0.20 0.0006 | 44 | 4.30 | 0.0005 | 86 | 8.40 | 0.0007 |
| 3 0.29 0.0008 | 45 | 4.39 | 0.0006 | 87 | 8.50 | 0.0004 |
| 4 0.39 0.0006 | 46 | 4.49 | 0.0005 | 88 | 8.59 | 0.0003 |
| 5 0.49 0.0005 | 47 | 4.59 | 0.0004 | 89 | 8.69 | 0.0005 |
| 6 0.59 0.0006 | 48 | 4.69 | 0.0005 | 90 | 8.79 | 0.0005 |
| 7 0.68 0.0007 | 49 | 4.79 | 0.0003 | 91 | 8.89 | 0.0006 |
| 8 0.78 0.0006 | 50 | 4.88 | 0.0005 | 92 | 8.98 | 0.0005 |
| 9 0.88 0.0008 | 51 | 4.98 | 0.0006 | 93 | 9.08 | 0.0005 |
| 10 0.98 0.0006 | 52 | 5.08 | 0.0004 | 94 | 9.18 | 0.0006 |
| 11 1.07 0.0005 | 53 | 5.18 | 0.0007 | 95 | 9.28 | 0.0005 |
| 12 1.17 0.0006 | 54 | 5.27 | 0.0006 | 96 | 9.38 | 0.0007 |
| 13 1.27 0.0006 | 55 | 5.37 | 0.0006 | 97 | 9.47 | 0.0004 |
| 14 1.37 0.0007 | 56 | 5.47 | 0.0006 | 98 | 9.57 | 0.0005 |
| 15 1.46 0.0005 | 57 | 5.57 | 0.0005 | 99 | 9.67 | 0.0005 |
| 16 1.56 0.0007 | 58 | 5.66 | 0.0008 | 100 | 9.77 | 0.0006 |
| 17 1.66 0.0006 | 59 | 5.76 | 0.0006 | 101 | 9.86 | 0.0006 |
| 18 1.76 0.0004 | 60 | 5.86 | 0.0006 | 102 | 9.96 | 0.0003 |
| 19 1.86 0.0006 | 61 | 5.96 | 0.0008 | 103 | 10.06 | 0.0004 |
| 20 1.95 0.0004 | 62 | 6.05 | 0.0006 | 104 | 10.16 | 0.0006 |
| 21 2.05 0.0005 | 63 | 6.15 | 0.0006 | 105 | 10.25 | 0.0007 |
| 22 2.15 0.0004 | 64 | 6.25 | 0.0005 | 106 | 10.35 | 0.0004 |
| 23 2.25 0.0006 | 65 | 6.35 | 0.0005 | 107 | 10.45 | 0.0006 |
| 24 2.34 0.0006 | 66 | 6.45 | 0.0004 | 108 | 10.55 | 0.0004 |
| 25 2.44 0.0004 | 67 | 6.54 | 0.0006 | 109 | 10.64 | 0.0006 |
| 26 2.54 0.0006 | 68 | 6.64 | 0.0007 | 110 | 10.74 | 0.0005 |
| 27 2.64 0.0006 | 69 | 6.74 | 0.0007 | 111 | 10.84 | 0.001 |
| 28 2.73 0.0004 | 70 | 6.84 | 0.0007 | 112 | 10.94 | 0.002 |
| 29 2.83 0.0004 | 71 | 6.93 | 0.0006 | 113 | 11.04 | 0.0009 |
| 30 2.93 0.0006 | 72 | 7.03 | 0.0006 | 114 | 11.13 | 0.0007 |
| 31 3.03 0.0005 | 73 | 7.13 | 0.0004 | 115 | 11.23 | 0.0005 |
| 32 3.13 0.0004 | 74 | 7.23 | 0.0006 | 116 | 11.33 | 0.0003 |
| 33 3.22 0.0004 | 75 | 7.32 | 0.0007 | 117 | 11.43 | 0.0006 |
| 34 3.32 0.0008 | 76 | 7.42 | 0.0007 | 118 | 11.52 | 0.0007 |
| 35 3.42 0.0007 | 77 | 7.52 | 0.0005 | 119 | 11.62 | 0.0005 |

| | | | | | | | | |
|-----|-------|--------|-----|-------|--------|-----|-------|--------|
| 120 | 11.72 | 0.0006 | 196 | 19.14 | 0.0003 | 272 | 26.56 | 0.0003 |
| 121 | 11.82 | 0.0004 | 197 | 19.24 | 0.0004 | 273 | 26.66 | 0.0003 |
| 122 | 11.91 | 0.0005 | 198 | 19.34 | 0.0005 | 274 | 26.76 | 0.0002 |
| 123 | 12.01 | 0.0005 | 199 | 19.43 | 0.0005 | 275 | 26.86 | 0.0003 |
| 124 | 12.11 | 0.0006 | 200 | 19.53 | 0.0004 | 276 | 26.95 | 0.0003 |
| 125 | 12.21 | 0.0005 | 201 | 19.63 | 0.0004 | 277 | 27.05 | 0.0002 |
| 126 | 12.30 | 0.0004 | 202 | 19.73 | 0.0004 | 278 | 27.15 | 0.0003 |
| 127 | 12.40 | 0.0004 | 203 | 19.82 | 0.0004 | 279 | 27.25 | 0.0003 |
| 128 | 12.50 | 0.0004 | 204 | 19.92 | 0.0004 | 280 | 27.34 | 0.0003 |
| 129 | 12.60 | 0.0005 | 205 | 20.02 | 0.0002 | 281 | 27.44 | 0.0004 |
| 130 | 12.70 | 0.0006 | 206 | 20.12 | 0.0005 | 282 | 27.54 | 0.0003 |
| 131 | 12.79 | 0.0004 | 207 | 20.21 | 0.0004 | 283 | 27.64 | 0.0003 |
| 132 | 12.89 | 0.0006 | 208 | 20.31 | 0.0003 | 284 | 27.73 | 0.0003 |
| 133 | 12.99 | 0.0003 | 209 | 20.41 | 0.0004 | 285 | 27.83 | 0.0003 |
| 134 | 13.09 | 0.0004 | 210 | 20.51 | 0.0003 | 286 | 27.93 | 0.0003 |
| 135 | 13.18 | 0.0004 | 211 | 20.61 | 0.0004 | 287 | 28.03 | 0.0003 |
| 136 | 13.28 | 0.0005 | 212 | 20.70 | 0.0004 | 288 | 28.13 | 0.0003 |
| 137 | 13.38 | 0.0003 | 213 | 20.80 | 0.0004 | 289 | 28.22 | 0.0003 |
| 138 | 13.48 | 0.0006 | 214 | 20.90 | 0.0002 | 290 | 28.32 | 0.0003 |
| 139 | 13.57 | 0.0006 | 215 | 21.00 | 0.0004 | 291 | 28.42 | 0.0003 |
| 140 | 13.67 | 0.0004 | 216 | 21.09 | 0.0005 | 292 | 28.52 | 0.0003 |
| 141 | 13.77 | 0.0003 | 217 | 21.19 | 0.0003 | 293 | 28.61 | 0.0003 |
| 142 | 13.87 | 0.0004 | 218 | 21.29 | 0.0003 | 294 | 28.71 | 0.0004 |
| 143 | 13.96 | 0.0005 | 219 | 21.39 | 0.0004 | 295 | 28.81 | 0.0004 |
| 144 | 14.06 | 0.0006 | 220 | 21.48 | 0.0004 | 296 | 28.91 | 0.0003 |
| 145 | 14.16 | 0.0005 | 221 | 21.58 | 0.0003 | 297 | 29.00 | 0.0003 |
| 146 | 14.26 | 0.0005 | 222 | 21.68 | 0.0004 | 298 | 29.10 | 0.0003 |
| 147 | 14.36 | 0.0003 | 223 | 21.78 | 0.0005 | 299 | 29.20 | 0.0002 |
| 148 | 14.45 | 0.0004 | 224 | 21.88 | 0.0008 | 300 | 29.30 | 0.0003 |
| 149 | 14.55 | 0.0006 | 225 | 21.97 | 0.0007 | 301 | 29.39 | 0.0003 |
| 150 | 14.65 | 0.0005 | 226 | 22.07 | 0.0003 | 302 | 29.49 | 0.0003 |
| 151 | 14.75 | 0.0003 | 227 | 22.17 | 0.0004 | 303 | 29.59 | 0.0003 |
| 152 | 14.84 | 0.0004 | 228 | 22.27 | 0.0004 | 304 | 29.69 | 0.0003 |
| 153 | 14.94 | 0.0005 | 229 | 22.36 | 0.0002 | 305 | 29.79 | 0.0003 |
| 154 | 15.04 | 0.0004 | 230 | 22.46 | 0.0004 | 306 | 29.88 | 0.0003 |
| 155 | 15.14 | 0.0004 | 231 | 22.56 | 0.0004 | 307 | 29.98 | 0.0003 |
| 156 | 15.23 | 0.0004 | 232 | 22.66 | 0.0004 | 308 | 30.08 | 0.0003 |
| 157 | 15.33 | 0.0004 | 233 | 22.75 | 0.0003 | 309 | 30.18 | 0.0003 |
| 158 | 15.43 | 0.0004 | 234 | 22.85 | 0.0004 | 310 | 30.27 | 0.0003 |
| 159 | 15.53 | 0.0004 | 235 | 22.95 | 0.0005 | 311 | 30.37 | 0.0002 |
| 160 | 15.63 | 0.0004 | 236 | 23.05 | 0.0003 | 312 | 30.47 | 0.0003 |
| 161 | 15.72 | 0.0004 | 237 | 23.14 | 0.0003 | 313 | 30.57 | 0.0002 |
| 162 | 15.82 | 0.0005 | 238 | 23.24 | 0.0003 | 314 | 30.66 | 0.0003 |
| 163 | 15.92 | 0.0005 | 239 | 23.34 | 0.0004 | 315 | 30.76 | 0.0003 |
| 164 | 16.02 | 0.0006 | 240 | 23.44 | 0.0004 | 316 | 30.86 | 0.0003 |
| 165 | 16.11 | 0.0003 | 241 | 23.54 | 0.0005 | 317 | 30.96 | 0.0002 |
| 166 | 16.21 | 0.0005 | 242 | 23.63 | 0.0003 | 318 | 31.05 | 0.0003 |
| 167 | 16.31 | 0.0006 | 243 | 23.73 | 0.0004 | 319 | 31.15 | 0.0003 |
| 168 | 16.41 | 0.0003 | 244 | 23.83 | 0.0004 | 320 | 31.25 | 0.0003 |
| 169 | 16.50 | 0.0005 | 245 | 23.93 | 0.0002 | 321 | 31.35 | 0.0003 |
| 170 | 16.60 | 0.0005 | 246 | 24.02 | 0.0005 | 322 | 31.45 | 0.0003 |
| 171 | 16.70 | 0.0007 | 247 | 24.12 | 0.0003 | 323 | 31.54 | 0.0003 |
| 172 | 16.80 | 0.0005 | 248 | 24.22 | 0.0004 | 324 | 31.64 | 0.0003 |
| 173 | 16.89 | 0.0003 | 249 | 24.32 | 0.0003 | 325 | 31.74 | 0.0002 |
| 174 | 16.99 | 0.0004 | 250 | 24.41 | 0.0002 | 326 | 31.84 | 0.0002 |
| 175 | 17.09 | 0.0005 | 251 | 24.51 | 0.0003 | 327 | 31.93 | 0.0003 |
| 176 | 17.19 | 0.0003 | 252 | 24.61 | 0.0003 | 328 | 32.03 | 0.0003 |
| 177 | 17.29 | 0.0004 | 253 | 24.71 | 0.0003 | 329 | 32.13 | 0.0004 |
| 178 | 17.38 | 0.0004 | 254 | 24.80 | 0.0003 | 330 | 32.23 | 0.0003 |
| 179 | 17.48 | 0.0004 | 255 | 24.90 | 0.0003 | 331 | 32.32 | 0.0002 |
| 180 | 17.58 | 0.0004 | 256 | 25.00 | 0.0003 | 332 | 32.42 | 0.0004 |
| 181 | 17.68 | 0.0004 | 257 | 25.10 | 0.0004 | 333 | 32.52 | 0.0003 |
| 182 | 17.77 | 0.0005 | 258 | 25.20 | 0.0003 | 334 | 32.62 | 0.0003 |
| 183 | 17.87 | 0.0002 | 259 | 25.29 | 0.0003 | 335 | 32.71 | 0.0003 |
| 184 | 17.97 | 0.0004 | 260 | 25.39 | 0.0003 | 336 | 32.81 | 0.0005 |
| 185 | 18.07 | 0.0004 | 261 | 25.49 | 0.0003 | 337 | 32.91 | 0.0003 |
| 186 | 18.16 | 0.0003 | 262 | 25.59 | 0.0003 | 338 | 33.01 | 0.0003 |
| 187 | 18.26 | 0.0004 | 263 | 25.68 | 0.0004 | 339 | 33.11 | 0.0003 |
| 188 | 18.36 | 0.0004 | 264 | 25.78 | 0.0003 | 340 | 33.20 | 0.0003 |
| 189 | 18.46 | 0.0003 | 265 | 25.88 | 0.0003 | 341 | 33.30 | 0.0004 |
| 190 | 18.55 | 0.0004 | 266 | 25.98 | 0.0004 | 342 | 33.40 | 0.0002 |
| 191 | 18.65 | 0.0005 | 267 | 26.07 | 0.0003 | 343 | 33.50 | 0.0003 |
| 192 | 18.75 | 0.0003 | 268 | 26.17 | 0.0004 | 344 | 33.59 | 0.0002 |
| 193 | 18.85 | 0.0004 | 269 | 26.27 | 0.0003 | 345 | 33.69 | 0.0003 |
| 194 | 18.95 | 0.0003 | 270 | 26.37 | 0.0004 | 346 | 33.79 | 0.0002 |
| 195 | 19.04 | 0.0004 | 271 | 26.46 | 0.0003 | 347 | 33.89 | 0.0003 |

| | | | | | | | | |
|-----|-------|--------|-----|-------|--------|-----|-------|--------|
| 348 | 33.98 | 0.0003 | 424 | 41.41 | 0.0002 | 500 | 48.83 | 0.0002 |
| 349 | 34.08 | 0.0003 | 425 | 41.50 | 0.0002 | 501 | 48.93 | 0.0002 |
| 350 | 34.18 | 0.0003 | 426 | 41.60 | 0.0003 | 502 | 49.02 | 0.0003 |
| 351 | 34.28 | 0.0003 | 427 | 41.70 | 0.0003 | 503 | 49.12 | 0.0002 |
| 352 | 34.38 | 0.0003 | 428 | 41.80 | 0.0002 | 504 | 49.22 | 0.0003 |
| 353 | 34.47 | 0.0003 | 429 | 41.89 | 0.0003 | 505 | 49.32 | 0.0002 |
| 354 | 34.57 | 0.0003 | 430 | 41.99 | 0.0003 | 506 | 49.41 | 0.0002 |
| 355 | 34.67 | 0.0003 | 431 | 42.09 | 0.0003 | 507 | 49.51 | 0.0003 |
| 356 | 34.77 | 0.0003 | 432 | 42.19 | 0.0002 | 508 | 49.61 | 0.0002 |
| 357 | 34.86 | 0.0003 | 433 | 42.29 | 0.0003 | 509 | 49.71 | 0.0002 |
| 358 | 34.96 | 0.0002 | 434 | 42.38 | 0.0002 | 510 | 49.80 | 0.0002 |
| 359 | 35.06 | 0.0003 | 435 | 42.48 | 0.0002 | 511 | 49.90 | 0.0002 |
| 360 | 35.16 | 0.0003 | 436 | 42.58 | 0.0003 | 512 | 50.00 | #N/A |
| 361 | 35.25 | 0.0002 | 437 | 42.68 | 0.0003 | | | |
| 362 | 35.35 | 0.0003 | 438 | 42.77 | 0.0002 | | | |
| 363 | 35.45 | 0.0002 | 439 | 42.87 | 0.0003 | | | |
| 364 | 35.55 | 0.0003 | 440 | 42.97 | 0.0003 | | | |
| 365 | 35.64 | 0.0002 | 441 | 43.07 | 0.0002 | | | |
| 366 | 35.74 | 0.0003 | 442 | 43.16 | 0.0002 | | | |
| 367 | 35.84 | 0.0003 | 443 | 43.26 | 0.0003 | | | |
| 368 | 35.94 | 0.0002 | 444 | 43.36 | 0.0002 | | | |
| 369 | 36.04 | 0.0003 | 445 | 43.46 | 0.0002 | | | |
| 370 | 36.13 | 0.0003 | 446 | 43.55 | 0.0003 | | | |
| 371 | 36.23 | 0.0002 | 447 | 43.65 | 0.0002 | | | |
| 372 | 36.33 | 0.0003 | 448 | 43.75 | 0.0002 | | | |
| 373 | 36.43 | 0.0002 | 449 | 43.85 | 0.0002 | | | |
| 374 | 36.52 | 0.0002 | 450 | 43.95 | 0.0002 | | | |
| 375 | 36.62 | 0.0003 | 451 | 44.04 | 0.0003 | | | |
| 376 | 36.72 | 0.0003 | 452 | 44.14 | 0.0003 | | | |
| 377 | 36.82 | 0.0003 | 453 | 44.24 | 0.0002 | | | |
| 378 | 36.91 | 0.0003 | 454 | 44.34 | 0.0003 | | | |
| 379 | 37.01 | 0.0002 | 455 | 44.43 | 0.0002 | | | |
| 380 | 37.11 | 0.0003 | 456 | 44.53 | 0.0002 | | | |
| 381 | 37.21 | 0.0002 | 457 | 44.63 | 0.0002 | | | |
| 382 | 37.30 | 0.0003 | 458 | 44.73 | 0.0003 | | | |
| 383 | 37.40 | 0.0003 | 459 | 44.82 | 0.0002 | | | |
| 384 | 37.50 | 0.0003 | 460 | 44.92 | 0.0002 | | | |
| 385 | 37.60 | 0.0003 | 461 | 45.02 | 0.0003 | | | |
| 386 | 37.70 | 0.0002 | 462 | 45.12 | 0.0003 | | | |
| 387 | 37.79 | 0.0002 | 463 | 45.21 | 0.0002 | | | |
| 388 | 37.89 | 0.0002 | 464 | 45.31 | 0.0003 | | | |
| 389 | 37.99 | 0.0002 | 465 | 45.41 | 0.0003 | | | |
| 390 | 38.09 | 0.0002 | 466 | 45.51 | 0.0002 | | | |
| 391 | 38.18 | 0.0003 | 467 | 45.61 | 0.0002 | | | |
| 392 | 38.28 | 0.0002 | 468 | 45.70 | 0.0003 | | | |
| 393 | 38.38 | 0.0002 | 469 | 45.80 | 0.0002 | | | |
| 394 | 38.48 | 0.0003 | 470 | 45.90 | 0.0003 | | | |
| 395 | 38.57 | 0.0002 | 471 | 46.00 | 0.0003 | | | |
| 396 | 38.67 | 0.0003 | 472 | 46.09 | 0.0002 | | | |
| 397 | 38.77 | 0.0002 | 473 | 46.19 | 0.0002 | | | |
| 398 | 38.87 | 0.0003 | 474 | 46.29 | 0.0002 | | | |
| 399 | 38.96 | 0.0003 | 475 | 46.39 | 0.0003 | | | |
| 400 | 39.06 | 0.0003 | 476 | 46.48 | 0.0002 | | | |
| 401 | 39.16 | 0.0003 | 477 | 46.58 | 0.0002 | | | |
| 402 | 39.26 | 0.0003 | 478 | 46.68 | 0.0002 | | | |
| 403 | 39.36 | 0.0002 | 479 | 46.78 | 0.0003 | | | |
| 404 | 39.45 | 0.0003 | 480 | 46.88 | 0.0003 | | | |
| 405 | 39.55 | 0.0003 | 481 | 46.97 | 0.0002 | | | |
| 406 | 39.65 | 0.0002 | 482 | 47.07 | 0.0002 | | | |
| 407 | 39.75 | 0.0003 | 483 | 47.17 | 0.0002 | | | |
| 408 | 39.84 | 0.0002 | 484 | 47.27 | 0.0003 | | | |
| 409 | 39.94 | 0.0003 | 485 | 47.36 | 0.0002 | | | |
| 410 | 40.04 | 0.0002 | 486 | 47.46 | 0.0003 | | | |
| 411 | 40.14 | 0.0003 | 487 | 47.56 | 0.0002 | | | |
| 412 | 40.23 | 0.0002 | 488 | 47.66 | 0.0002 | | | |
| 413 | 40.33 | 0.0003 | 489 | 47.75 | 0.0003 | | | |
| 414 | 40.43 | 0.0002 | 490 | 47.85 | 0.0002 | | | |
| 415 | 40.53 | 0.0002 | 491 | 47.95 | 0.0002 | | | |
| 416 | 40.63 | 0.0002 | 492 | 48.05 | 0.0003 | | | |
| 417 | 40.72 | 0.0002 | 493 | 48.14 | 0.0003 | | | |
| 418 | 40.82 | 0.0003 | 494 | 48.24 | 0.0002 | | | |
| 419 | 40.92 | 0.0002 | 495 | 48.34 | 0.0002 | | | |
| 420 | 41.02 | 0.0002 | 496 | 48.44 | 0.0003 | | | |
| 421 | 41.11 | 0.0002 | 497 | 48.54 | 0.0002 | | | |
| 422 | 41.21 | 0.0002 | 498 | 48.63 | 0.0002 | | | |
| 423 | 41.31 | 0.0002 | 499 | 48.73 | 0.0003 | | | |

Appendix 5.1: Programme listings

thsol MCR1.m

```
%program THSOL
%analytical inverse fin solution for a fin with
constant heat transfer
%adapted from Patounas (2007)
```

```
%uses MCR related parameters
```

```
%input parameters
```

```
hc=30;
t=0.004;
K=7.72;
r1=0.22;
r2=0.07;
Tb=372;
Te=320;
dr=0.001;
```

```
%calculated parameters
```

```
bita=sqrt(hc/(t*K));
z1=bita*r1;
z2=bita*r2;
```

```
%calculate Bessel values
```

```
I0z1=besseli(0,z1);
I1z1=besseli(1,z1);
K0z1=besselk(0,z1);
K1z1=besselk(1,z1);
I0z2=besseli(0,z2);
I1z2=besseli(1,z2);
K0z2=besselk(0,z2);
K1z2=besselk(1,z2);
```

```
%calculate theta argument
```

```
arg1=K1z2/(I0z1*K1z2+I1z2*K0z1);
arg2=I1z2/(I0z1*K1z2+I1z2*K0z1);
```

```
%establish r grid
```

```
r=r2:dr:r1;
npoints=(r1-r2)/dr+1;
```

```
%loop for all points in the grid
```

```
for c=1:npoints
    %calculate theta using analytical solution
    z=bita*r(c);
    I0z=besseli(0,z);
    K0z=besselk(0,z);
    theta=arg1*I0z+arg2*K0z;
    %calculate T from theta
    T(c)=theta*(Tb-Te)+Te;
end
plot(r,T)
```

finso13 MCR3.m

```
%program finso13_MCR3
```

```
%m-file to apply to fit various curves
%and apply the fin solution on whole length of disc
```

```
%
```

```
%adapted from Patounas (2007)
```

```
%
```

```
%input is the measurements of several combinations of T/Cs and
%polynomial interpolation orders
```

```
%define basic parameters
```

```
%T has already been defined from ansol
dr=0.001; %grid increment 1mm
L=0.004; %half thickness of disc
K=7.72; %thermal conductivity titanium 318
r=0.07:dr:0.22;
npoints=(0.22-0.07)/0.001+1;
Te=320;
Tb=372;
```

```
%set-up thermocouple temperatures and locations
```

```
%6 T/C case
```

```
x6=r([1 11 52 94 138 151]);
y6=T([1 11 52 94 138 151]);
```

```
%5 T/C case
```

```
x5=r([1 11 52 94 138]);
y5=T([1 11 52 94 138]);
```

```
%4 T/C case
```

```
x4=r([1 11 52 94 138]);
y4=T([1 11 52 94 138]);
```

```
%3 T/C case
```

```
x3=r([52 94 138]);
y3=T([52 94 138]);
```

```
%use analytical solution
```

```
Tan=T;
```

finso13 MCR1.m

```
%program finso13_MCR1
```

```
%m-file to apply to fit various curves
%and apply the fin solution on whole length
of disc
```

```
%
```

```
%adapted from Patounas (2007)
```

```
%
```

```
%input is the measurements of several t/cs
```

```
%define basic parameters
```

```
%T has already been defined from ansol
dr=0.001;
L=0.004;
K=7.72;
r=0.07:dr:0.22;
npoints=(0.22-0.07)/0.001+1;
Te=320;
Tb=372;
```

```
%set-up thermocouple temperatures and
```

```
locations
```

```
%6 T/C case
```

```
x6=r([1 11 52 94 138 151]);
y6=T([1 11 52 94 138 151]);
```

```
%use analytical solution
```

```
Tan=T;
```

```
%fit third order polynomial on temperatures
```

```
pol3=polyfit(x6,y6,3);
Tpol3=polyval(pol3,r);
```

```
%fit 4th order polynomial on temperatures
```

```
pol4=polyfit(x6,y6,4);
Tpol4=polyval(pol4,r);
```

```
%fit 5th order polynomial on temperatures
```

```
pol5=polyfit(x6,y6,5);
Tpol5=polyval(pol5,r);
```

```
%fit cubic spline on temperatures
```

```
s=spline(x6,y6);
Tspline=ppval(s,r);
```

```
%calculation loop
```

```
for times=1:5
    if times==1;
        T=Tan;
    elseif times==2;
        T=Tpol3;
    elseif times==3;
        T=Tpol4;
    elseif times==4;
        T=Tpol5;
    elseif times==5;
        T=Tspline;
    end
end
```

```
%fit second order polynomial using T/Cs 9-10-11
```

```
pol2=polyfit(x3,y3,2);
Tpol2=polyval(pol2,r);
```

```
%fit third order polynomial using T/Cs 8-9-10-11
```

```
pol3=polyfit(x4,y4,3);
Tpol3=polyval(pol3,r);
```

```
%fit 4th order polynomial using T/Cs 7-8-9-10-11
```

```
pol4=polyfit(x5,y5,4);
Tpol4=polyval(pol4,r);
```

```
%fit 5th order polynomial using T/Cs 7-8-9-10-11-12
```

```
pol5=polyfit(x6,y6,5);
Tpol5=polyval(pol5,r);
```

```
%fit cubic spline on temperatures
```

```
s=spline(x6,y6);
Tspline=ppval(s,r);
```

```
%calculation loop
```

```
for times=1:6
    if times==1;
        T=Tan;
    elseif times==2;
        T=Tpol2;
    elseif times==3;
        T=Tpol3;
    elseif times==4;
        T=Tpol4;
    elseif times==5;
        T=Tpol5;
    elseif times==6;
        T=Tspline;
    end
    for c=1:(npoints-2)
        p=c+1;
        dev2=(Tt(p+1)-2*Tt(p)+Tt(p-1))/dr^2;
        rdev=(Tt(p+1)-Tt(p-1))/(2*r(p)*dr);
        h(c)=q(c)/((T(p)-Te));
    end
end
```

```
T=Tpol5;
elseif times==5;
    T=Tspline;
end
for c=1:(npoints-2)
    p=c+1;
    dev2=(T(p+1)-2*T(p)+T(p-1))/dr^2;
    rdev=(T(p+1)-T(p-1))/(2*r(p)*dr);
    q(c)=K*L*(dev2+rdev);
    h(c)=q(c)/((T(p)-Te));
end
%create reduced 149 points matrices
r149=r(2:150);
T149=T(2:150);
```

```
%calculate average values
```

```
qr=q./r149;
rDT=(T149-Te)./r149;
hav=(trapz(r149,qr)/trapz(r149,rDT));
```

```
%save output for each type of fitting
```

```
if times==1;
    han=h;
    hanav=hav;
elseif times==2;
    hpol3=h;
    hpol3av=hav;
elseif times==3;
    hpol4=h;
    hpol4av=hav;
elseif times==4;
    hpol5=h;
    hpol5av=hav;
elseif times==5;
    hspline=h;
    hsplineav=hav;
end
end
```

```
%plot the results
sampleh=hc*ones(1,149);
plot(r149,sampleh,'k')
hold on
plot(r149,han)
plot(r149,hspline,'c')
plot(r149,hpol3,'k')
plot(r149,hpol4,'g')
plot(r149,hpol5,'r')
legend('Assumed h used in the analytical
solution','True data','Cubic spline fit','3rd
order poly','4th order poly','5th order poly','4)
title('Analytical and fin solution comparison')
xlabel('Disc radius (mm)')
ylabel('h (W/(m^2.K))')
```

```

q(c)=K*L*(dev2+rdev);
h(c)=q(c)/((Tt(p)-Te));
end
%create reduced 149 points matrices
r149=r(2:150);
T149=Tt(2:150);

%calculate average values
qr=q.*r149;
rDT=(T149-Te).*r149;
hav=(trapz(r149,qr)/trapz(r149,rDT));

%save output for each type of fitting
if times==1;
    han=h;
    hanav=hav;
elseif times==2;
    hpol2=h;
    hpol2av=hav;
elseif times==3;
    hpol3=h;
    hpol3av=hav;
elseif times==4;
    hpol4=h;
    hpol4av=hav;
elseif times==5;

```

finsoL MCR4.m

```

%program finsoL_MCR4
%m-file to apply to fit various curves
%and apply the fin solution on whole length of plain disc
%
%adapted from Patounas (2007)
%
%input is the experimental temperature measurements of several
combinations of T/Cs and
%polynomial interpolation orders - using thermocouples values from
B3run2-1

```

```

%define basic parameters
%
dr=0.001; %grid increment 1mm
L=0.004; %half thickness of disc
K=7.72; %thermal conductivity titanium 318
r=0.07;dr:0.22;
nopoints=(0.22-0.07)/0.001+1;
Te=319.3;
Tb=377;

```

```

%set-up thermocouple temperatures and locations
%6 T/C case
x6=[0.070 0.080 0.121 0.163 0.207 0.22];
y6=[319.3 321.7 322.6 329.5 352.4 377.0];
%5 T/C case 7-11
x5=[0.070 0.080 0.121 0.163 0.207];
y5=[319.3 321.7 322.6 329.5 352.4];
%4 T/C case T/Cs 8-11
x4a=[0.080 0.121 0.163 0.207];
y4a=[321.7 322.6 329.5 352.4];
%4 T/C case T/Cs 9-12
x4b=[0.121 0.163 0.207 0.22];
y4b=[322.6 329.5 352.4 377.0];
%3 T/C case
x3=[0.121 0.163 0.207];
y3=[322.6 329.5 352.4];

```

```

%use analytical solution
%Tan=T;

```

```

%fit second order polynomial using T/Cs 9-10-11
pol2=polyfit(x3,y3,2);
Tpol2=polyval(pol2,r);

```

```

%fit third order polynomial using T/Cs 8-9-10-11
pol3a=polyfit(x4a,y4a,3);
Tpol3a=polyval(pol3a,r);

```

```

%fit third order polynomial using T/Cs 9-10-11-12
pol3b=polyfit(x4b,y4b,3);
Tpol3b=polyval(pol3b,r);

```

```

%fit 4th order polynomial using T/Cs 7-8-9-10-11
pol4=polyfit(x5,y5,4);
Tpol4=polyval(pol4,r);

```

```

%fit 5th order polynomial using T/Cs 7-8-9-10-11-12
pol5=polyfit(x6,y6,5);
Tpol5=polyval(pol5,r);

```

```

%fit cubic spline on temperatures
s=spline(x6,y6);
Tspline=ppval(s,r);

```

```

%calculation loop
for times=1:6
    if times==1;
        Tt=Tpol3b;
    elseif times==2;
        Tt=Tpol2;
    elseif times==3;

```

```

        hpol5=h;
        hpol5av=hav;
    elseif times==6;
        hspline=h;
        hsplineav=hav;
    end
end

```

```

%plot the results
sampleh=hc*ones(1,149);
plot(r149,sampleh,'k')
hold on
plot(r149,han)
plot(r149,hpol2,'c')
plot(r149,hpol3,'k')
plot(r149,hpol4,'g')
plot(r149,hpol5,'r')
%plot(r149,hspline,'m')
legend('Assumed h used in the analytical solution','True data','2nd order
poly - 3T/Cs','3rd order poly - 4T/Cs','4th order poly - 5T/Cs','5th order
poly - 6T/Cs',4)
title('Plain disc fin solution')
xlabel('Disc radius (mm)')
ylabel('h (W/(m^2.K))')

```

```

Tt=Tpol3a;
elseif times==4;
    Tt=Tpol4;
elseif times==5;
    Tt=Tpol5;
elseif times==6;
    Tt=Tspline;
end
for c=1:(nopoints-2)
    p=c+1;
    dev2=(Tt(p+1)-2*Tt(p)+Tt(p-1))/dr^2;
    rdev=(Tt(p+1)-Tt(p-1))/(2*r(p)*dr);
    q(c)=K*L*(dev2+rdev);
    h(c)=q(c)/((Tt(p)-Te));
end
%create reduced 149 points matrices
r149=r(2:150);
T149=Tt(2:150);

```

```

%calculate average values
qr=q.*r149;
rDT=(T149-Te).*r149;
hav=(trapz(r149,qr)/trapz(r149,rDT));

```

```

%save output for each type of fitting
if times==1;
    hpol3b=h;
    hpol3bav=hav;
elseif times==2;
    hpol2=h;
    hpol2av=hav;
elseif times==3;
    hpol3a=h;
    hpol3aav=hav;
elseif times==4;
    hpol4=h;
    hpol4av=hav;
elseif times==5;
    hpol5=h;
    hpol5av=hav;
elseif times==6;
    hspline=h;
    hsplineav=hav;
end
end

```

```

%plot the results
%sampleh=hc*ones(1,149);
%plot(r149,sampleh,'k')
%hold on
%plot(r149,han)
plot(r149,hpol2,'c')
hold on
plot(r149,hpol3a,'k')
plot(r149,hpol3b,'b')
plot(r149,hpol4,'g')
plot(r149,hpol5,'r')
%plot(r149,hspline,'m')
legend('2nd order poly - 3T/Cs','3rd order poly - 4T/Cs 8-11','3rd order
poly - 4T/Cs 9-12','4th order poly - 5T/Cs','5th order poly - 6T/Cs',4)
title('Comparison of different experimental configurations on a plain disc
fin solution')
xlabel('Disc radius (mm)')
ylabel('h (W/(m^2.K))')

```


Appendix 6.1: Instructions for the selection and access to the experimental data

The experimental data comes in a separate CD-Rom included with this thesis.

The list of all experimental conditions for Build 2 and Build 3 can be found in the file **MCR_non_dim_results.xls** or in Appendix 6.2 and 6.3. For each test the information provided in the columns are explained in the table below.

| | |
|---------------------------|---|
| Build | Which Build (2 or 3) |
| ExpCode | Not used |
| Run | Test identification |
| ExpDate | Date of testing |
| LDA code | 000: No LDA included, 100: LDA V_ϕ , 101: LDA V_ϕ and V_z , 110: LDA V_ϕ and V_r |
| FCF pos | 0: Flow Control Feature not installed, 1: FCF in upstream position, 2: FCF in mid-position, 3: FCF in downstream position |
| Lab atm press (Pa) | Measured atmospheric pressure |
| Comments | Note on test |
| Re_{fi} | Value of Re_ϕ for test |
| +/- | 95% confidence interval of calculated Re_ϕ |
| Re_z | Value of Re_z for test |
| +/- | 95% confidence interval of calculated Re_z |
| BDTav | Calculated value of $\beta\Delta T_{av}$ for test |
| +/- | 95% confidence interval of calculated $\beta\Delta T_{av}$ |
| finBDTav | Calculated value of $fin\beta\Delta T_{av}$ for test |
| +/- | 95% confidence interval of calculated $fin\beta\Delta T_{av}$ |
| Ro | Value of Ro for test |
| +/- | 95% confidence interval of calculated Ro |
| etc. | ... |

Having selected a test of interest from this list, the corresponding test reports can be found in the Directories **\Heat Transfer Test Reports** and **\LDA Test Reports**, the contents of which are described below. Please note that the test reports of the tests specifically mentioned in Chapter 6 and 7 are also included in Appendix 6.4 for LDA and Appendix 7.1 for heat transfer for convenience of reading.

CD-Rom contents:

**\Heat Transfer Test Reports\
MCR_HT-reports1.xls
MCR_HT-reports2.xls
MCR_HT-reports3.xls
MCR_HT-reports4.xls
MCR_HT-reports5.xls
MCR_HT-reports6.xls**

B3run2-1 to B3run16-1
B3run16-2 to B3run31-1
B3run32-1 to B3run35-5
B3run35-6 to B2run08-1
B2run08-2 to B2t13-a
B2t13-b to B2t19-d

\LDA Test Reports**B2_LDA_test-reports1.xls**

All Build 2 LDA data

B3_LDA_test-reports1.xls

B3run3-1 to B3run28-1

B3_LDA_test-reports2.xls

B3run28-2 to B3run45-1

\LDA Test Reports\SPECTRA

This contains the output files for the spectral analysis in Chapter 6 and plots. The positions are taken in the mid plane of Cavity 3 with non-dimensional radius included in the file name.

e.g: SPECT_R38_rb-0566_list.txt

output file for B3run38-1, $r/b=0.566$

SPECT_R38_rb-0566.jpg

plot image for B3run38-1,

 $r/b=0.566$

Appendix 6.2: Test conditions and main non-dimensional results for Build 2

| Build | Exp Code | Run | ExpDate | LDA code | FCF pos | Lab atm press (Pa) | Comments | Refi | +/- | Rez | +/- | BDTav | +/- | finBDTav | +/- | Ro | +/- |
|-------|----------|------------|------------|----------|---------|--------------------|----------|----------|---------|----------|---------|-------|---------|----------|---------|--------|---------|
| 2 | 2S---- | B2run02-1 | 6/30/2000 | 0 | 0 | 1.016E+05 | | 3.20E+04 | 4.8E+04 | 1.05E+05 | 2.2E+02 | 0.086 | 4.4E-03 | 0.086 | 4.4E-03 | 112.08 | 1.9E+01 |
| 2 | 2S---- | B2run02-2 | 6/30/2000 | 0 | 0 | 1.016E+05 | | 4.72E+04 | 5.2E+04 | 1.28E+05 | 8.2E+01 | 0.018 | 1.4E-03 | 0.018 | 1.4E-03 | 92.52 | 2.9E+01 |
| 2 | 2S---- | B2run02-3 | 6/30/2000 | 0 | 0 | 1.016E+05 | | 3.16E+04 | 6.5E+04 | 1.04E+05 | 2.4E+02 | 0.081 | 3.9E-03 | 0.081 | 3.9E-03 | 112.47 | 1.3E+01 |
| 2 | 2S---- | B2run03-1 | 07/04/2000 | 0 | 0 | 1.007E+05 | | 2.59E+04 | 3.4E+04 | 1.58E+05 | 1.5E+02 | 0.077 | 3.5E-03 | 0.077 | 3.5E-03 | 208.98 | 1.3E+02 |
| 2 | 2S---- | B2run04-1 | 7/19/2000 | 0 | 0 | 1.022E+05 | | 2.62E+04 | 5.0E+04 | 1.26E+05 | 1.8E+02 | 0.077 | 2.7E-03 | 0.077 | 2.7E-03 | 165.17 | 4.1E+01 |
| 2 | 2S---- | B2run04-2 | 7/19/2000 | 0 | 0 | 1.022E+05 | | 1.56E+04 | 2.8E+04 | 1.28E+05 | 1.8E+02 | 0.085 | 2.7E-03 | 0.085 | 2.7E-03 | 280.80 | 5.8E+01 |
| 2 | 2S---- | B2run04-3 | 7/19/2000 | 0 | 0 | 1.022E+05 | | 4.03E+04 | 7.8E+04 | 8.42E+04 | 2.2E+02 | 0.070 | 3.7E-03 | 0.070 | 3.7E-03 | 71.41 | 1.3E+02 |
| 2 | 2S---- | B2run04-4 | 7/19/2000 | 0 | 0 | 1.022E+05 | | 2.57E+04 | 4.9E+04 | 8.44E+04 | 1.9E+02 | 0.073 | 3.6E-03 | 0.073 | 3.6E-03 | 112.27 | 1.9E+01 |
| 2 | 2S---- | B2run04-7 | 7/19/2000 | 0 | 0 | 1.022E+05 | | 2.52E+04 | 4.9E+04 | 8.53E+04 | 1.6E+02 | 0.075 | 3.0E-03 | 0.075 | 3.0E-03 | 115.66 | 4.9E+01 |
| 2 | 2S---- | B2run06-1 | 10/03/2000 | 0 | 0 | 1.020E+05 | | 4.15E+05 | 5.9E+02 | 8.49E+04 | 4.9E+01 | 0.070 | 7.9E-04 | 0.070 | 7.9E-04 | 7.00 | 6.2E-03 |
| 2 | 2S---- | B2run06-2 | 10/03/2000 | 0 | 0 | 1.020E+05 | | 7.05E+05 | 9.3E+02 | 8.49E+04 | 4.7E+01 | 0.086 | 7.8E-04 | 0.086 | 7.8E-04 | 4.12 | 3.2E-03 |
| 2 | 2S---- | B2run08-1 | 10/05/2000 | 0 | 0 | 1.017E+05 | | 1.03E+06 | 1.5E+03 | 5.92E+04 | 3.7E+01 | 0.095 | 8.6E-04 | 0.095 | 8.6E-04 | 1.96 | 1.6E-03 |
| 2 | 2S---- | B2run08-2 | 10/05/2000 | 0 | 0 | 1.017E+05 | | 2.00E+06 | 3.0E+03 | 5.89E+04 | 3.9E+01 | 0.070 | 8.9E-04 | 0.070 | 8.9E-04 | 1.01 | 8.7E-04 |
| 2 | 2S---- | B2run08-3 | 10/05/2000 | 0 | 0 | 1.017E+05 | | 4.02E+06 | 5.0E+03 | 5.91E+04 | 3.2E+01 | 0.061 | 7.3E-04 | 0.061 | 7.3E-04 | 0.50 | 3.5E-04 |
| 2 | 2S---- | B2run08-4 | 10/05/2000 | 0 | 0 | 1.017E+05 | | 4.01E+06 | 5.6E+03 | 8.61E+04 | 5.2E+01 | 0.056 | 8.1E-04 | 0.056 | 8.1E-04 | 0.73 | 5.8E-04 |
| 2 | 2S---- | B2run08-5 | 10/05/2000 | 0 | 0 | 1.017E+05 | | 3.05E+06 | 4.0E+03 | 8.65E+04 | 4.8E+01 | 0.056 | 7.4E-04 | 0.056 | 7.4E-04 | 0.97 | 7.4E-04 |
| 2 | 2S---- | B2run08-6 | 10/05/2000 | 0 | 0 | 1.017E+05 | | 2.03E+06 | 2.4E+03 | 8.58E+04 | 4.5E+01 | 0.062 | 7.3E-04 | 0.062 | 7.3E-04 | 1.45 | 1.0E-03 |
| 2 | 2S---- | B2run08-7 | 10/05/2000 | 0 | 0 | 1.017E+05 | | 2.03E+06 | 2.4E+03 | 8.68E+04 | 4.3E+01 | 0.048 | 7.0E-04 | 0.048 | 7.0E-04 | 1.46 | 1.0E-03 |
| 2 | 2S---- | B2run09-1 | 10/09/2000 | 0 | 0 | 9.881E+04 | | 2.02E+06 | 3.0E+03 | 1.43E+05 | 9.3E+01 | 0.060 | 9.1E-04 | 0.060 | 9.1E-04 | 2.43 | 2.1E-03 |
| 2 | 2S---- | B2run09-2 | 10/09/2000 | 0 | 0 | 9.881E+04 | | 3.99E+06 | 6.8E+03 | 1.43E+05 | 1.1E+02 | 0.046 | 9.8E-04 | 0.046 | 9.8E-04 | 1.22 | 1.2E-03 |
| 2 | 2S---- | B2run09-3 | 10/09/2000 | 0 | 0 | 9.881E+04 | | 2.97E+06 | 4.4E+03 | 1.43E+05 | 9.2E+01 | 0.052 | 8.3E-04 | 0.052 | 8.3E-04 | 1.64 | 1.4E-03 |
| 2 | 2S---- | B2run09-5a | 10/09/2000 | 0 | 0 | 9.881E+04 | | 1.01E+06 | 1.6E+03 | 1.43E+05 | 9.6E+01 | 0.055 | 9.1E-04 | 0.054504 | 9.1E-04 | 4.87 | 4.3E-03 |
| 2 | 2S---- | B2run09-5b | 10/09/2000 | 0 | 0 | 9.881E+04 | | 1.98E+06 | 3.0E+03 | 1.43E+05 | 9.4E+01 | 0.062 | 8.8E-04 | 0.062393 | 8.8E-04 | 2.47 | 2.1E-03 |

| Build | Exp Code | Run | ExpDate | LDA code | FCF pos | Lab atm press (Pa) | Comments | Refi | +/- | Rez | +/- | BDTav | +/- | finBDTav | +/- | Ro | +/- |
|-------|----------|-----------|------------|----------|---------|--------------------|----------|----------|---------|----------|---------|-------|---------|----------|---------|------|---------|
| 2 | 2S---- | B2run09-6 | 10/09/2000 | 0 | 0 | 9.881E+04 | | 1.99E+06 | 2.9E+03 | 1.41E+05 | 8.8E+01 | 0.044 | 8.0E-04 | 0.04407 | 8.0E-04 | 2.43 | 2.0E-03 |
| 2 | 2S---- | B2run12-1 | 10/24/2000 | 0 | 0 | 1.021E+05 | | 2.00E+06 | 3.0E+03 | 9.15E+04 | 6.0E+01 | 0.063 | 9.0E-04 | 0.062996 | 9.0E-04 | 1.56 | 1.3E-03 |
| 2 | 2S---- | B2run12-2 | 10/24/2000 | 0 | 0 | 1.021E+05 | | 4.01E+06 | 5.6E+03 | 9.04E+04 | 5.5E+01 | 0.057 | 8.4E-04 | 0.057225 | 8.4E-04 | 0.77 | 6.2E-04 |
| 2 | 2S---- | B2t05 | 08/04/2001 | 101 | 0 | 1.017E+05 | | 1.56E+06 | 2.1E+03 | 8.16E+04 | 4.7E+01 | 0.057 | 7.6E-04 | 0.057178 | 7.6E-04 | 1.79 | 1.4E-03 |
| 2 | 2S---- | B2t06 | 08/07/2001 | 101 | 0 | 1.007E+05 | | 1.09E+06 | 1.4E+03 | 1.14E+05 | 6.5E+01 | 0.041 | 7.4E-04 | 0.041394 | 7.4E-04 | 3.56 | 2.6E-03 |
| 2 | 2S---- | B2t07-a | 8/13/2001 | 101 | 0 | 1.016E+05 | | 1.05E+06 | 1.4E+03 | 1.23E+05 | 6.9E+01 | 0.049 | 7.5E-04 | 0.048657 | 7.5E-04 | 3.99 | 3.1E-03 |
| 2 | 2S---- | B2t07-b | 8/13/2001 | 101 | 0 | 1.016E+05 | | 2.02E+06 | 3.0E+03 | 1.23E+05 | 7.7E+01 | 0.051 | 8.5E-04 | 0.050863 | 8.5E-04 | 2.07 | 1.7E-03 |
| 2 | 2S---- | B2t09a | 09/03/2001 | 0 | 0 | 1.014E+05 | | 8.70E+05 | 1.2E+03 | 1.12E+05 | 6.4E+01 | 0.040 | 7.8E-04 | 0.039554 | 7.8E-04 | 4.39 | 3.4E-03 |
| 2 | 2S---- | B2t09b | 09/03/2001 | 0 | 0 | 1.014E+05 | | 8.80E+05 | 1.2E+03 | 1.12E+05 | 6.9E+01 | 0.044 | 8.0E-04 | 0.043624 | 8.0E-04 | 4.37 | 3.5E-03 |
| 2 | 2S---- | B2t09-c | 09/03/2001 | 0 | 0 | 1.014E+05 | | 8.87E+05 | 1.1E+03 | 1.13E+05 | 6.2E+01 | 0.044 | 7.5E-04 | 0.044162 | 7.5E-04 | 4.35 | 3.1E-03 |
| 2 | 2S---- | B2t10 | 09/05/2001 | 101 | 0 | 1.023E+05 | | 1.50E+06 | 1.6E+03 | 6.40E+04 | 2.9E+01 | 0.062 | 6.5E-04 | 0.061683 | 6.5E-04 | 1.45 | 9.0E-04 |
| 2 | 2S---- | B2t11 | 09/06/2001 | 101 | 0 | 1.018E+05 | | 9.56E+05 | 1.9E+03 | 7.60E+04 | 6.3E+01 | 0.062 | 1.1E-03 | 0.061815 | 1.1E-03 | 2.72 | 3.0E-03 |
| 2 | 2S---- | B2t12 | 09/06/2001 | 100 | 0 | 1.018E+05 | | 1.13E+06 | 1.7E+03 | 6.63E+04 | 4.3E+01 | 0.067 | 8.3E-04 | 0.066697 | 8.3E-04 | 2.01 | 1.7E-03 |
| 2 | 2s---- | B2t13-a | 10/17/2001 | 0 | 0 | 1.015E+05 | | 1.03E+06 | 6.5E+03 | 1.23E+05 | 3.3E+02 | 0.041 | 3.6E-04 | 0.040952 | 3.6E-04 | 4.10 | 1.5E-02 |
| 2 | 2S---- | B2t13-b | 10/17/2001 | 100 | 0 | 1.015E+05 | | 1.05E+06 | 1.6E+03 | 1.25E+05 | 8.1E+01 | 0.047 | 8.6E-04 | 0.047233 | 8.6E-04 | 4.10 | 3.5E-03 |
| 2 | 2S---- | B2t14 | 10/19/2001 | 101 | 0 | 1.008E+05 | | 1.02E+06 | 1.7E+03 | 1.23E+05 | 8.6E+01 | 0.048 | 9.5E-04 | 0.047568 | 9.5E-04 | 4.13 | 4.0E-03 |
| 2 | 2S---- | B2t15-a | 10/23/2001 | 0 | 0 | 1.012E+05 | | 1.05E+06 | 1.6E+03 | 1.26E+05 | 7.8E+01 | 0.049 | 8.5E-04 | 0.048812 | 8.5E-04 | 4.10 | 3.6E-03 |
| 2 | 2S---- | B2t15-b | 10/23/2001 | 101 | 0 | 1.012E+05 | | 2.00E+06 | 3.0E+03 | 1.24E+05 | 8.0E+01 | 0.050 | 8.4E-04 | 0.049629 | 8.4E-04 | 2.12 | 1.8E-03 |
| 2 | 2S---- | B2t16-a | 11/20/2001 | 101 | 0 | 1.032E+05 | | 1.02E+06 | 3.3E+03 | 1.23E+05 | 1.3E+02 | 0.044 | 1.4E-03 | 0.044454 | 1.4E-03 | 4.12 | 1.0E-02 |
| 2 | 2S---- | B2t16-b | 11/20/2001 | 101 | 0 | 1.032E+05 | | 1.04E+06 | 1.4E+03 | 1.24E+05 | 7.4E+01 | 0.049 | 7.8E-04 | 0.049024 | 7.8E-04 | 4.08 | 3.2E-03 |
| 2 | 2S---- | B2t16-c | 11/20/2001 | 0 | 0 | 1.032E+05 | | 1.05E+06 | 2.3E+03 | 1.24E+05 | 1.2E+02 | 0.049 | 1.2E-03 | 0.048797 | 1.2E-03 | 4.03 | 5.0E-03 |
| 2 | 2S---- | B2t17-a | 11/21/2001 | 101 | 0 | 1.031E+05 | | 2.00E+06 | 2.9E+03 | 1.24E+05 | 7.7E+01 | 0.049 | 8.6E-04 | 0.049293 | 8.6E-04 | 2.11 | 1.7E-03 |
| 2 | 2S---- | B2t17-b | 11/21/2001 | 101 | 0 | 1.031E+05 | | 2.02E+06 | 2.7E+03 | 1.25E+05 | 7.2E+01 | 0.051 | 8.0E-04 | 0.050684 | 8.0E-04 | 2.12 | 1.6E-03 |
| 2 | 2S---- | B2t17-c | 11/21/2001 | 0 | 0 | 1.022E+05 | | 2.00E+06 | 2.3E+03 | 1.25E+05 | 6.1E+01 | 0.050 | 6.7E-04 | 0.050421 | 6.7E-04 | 2.13 | 1.4E-03 |
| 2 | 2S---- | B2t18-a | 11/22/2001 | 101 | 0 | 1.022E+05 | | 9.89E+05 | 1.3E+03 | 6.22E+04 | 3.6E+01 | 0.068 | 8.1E-04 | 0.067906 | 8.1E-04 | 2.15 | 1.7E-03 |
| 2 | 2S---- | B2t18-b | 11/22/2001 | 101 | 0 | 1.022E+05 | | 9.89E+05 | 1.3E+03 | 6.23E+04 | 3.5E+01 | 0.069 | 7.4E-04 | 0.068677 | 7.4E-04 | 2.15 | 1.7E-03 |
| 2 | 2S---- | B2t18-c | 11/22/2001 | 101 | 0 | 1.022E+05 | | 2.06E+06 | 2.7E+03 | 6.17E+04 | 3.5E+01 | 0.054 | 8.1E-04 | 0.054443 | 8.1E-04 | 1.02 | 7.7E-04 |

| Build | Exp Code | Run | ExpDate | LDA code | FCF pos | Lab atm press (Pa) | Comments | Refi | +/- | Rez | +/- | BDTav | +/- | finBDTav | +/- | Ro | +/- |
|-------|----------|---------|------------|----------|---------|--------------------|----------|----------|---------|----------|---------|-------|---------|----------|---------|------|---------|
| 2 | 2S---- | B2t18-d | 11/22/2001 | 0 | 0 | 1.022E+05 | | 2.06E+06 | 2.9E+03 | 6.21E+04 | 3.8E+01 | 0.055 | 8.1E-04 | 0.054533 | 8.1E-04 | 1.03 | 8.2E-04 |
| 2 | 2S---- | B2t19-a | 11/23/2001 | 101 | 0 | 1.030E+05 | | 1.00E+06 | 1.4E+03 | 6.55E+04 | 3.9E+01 | 0.067 | 8.0E-04 | 0.067237 | 8.0E-04 | 2.23 | 1.7E-03 |
| 2 | 2S---- | B2t19-b | 11/23/2001 | 101 | 0 | 1.030E+05 | | 1.00E+06 | 1.5E+03 | 6.60E+04 | 4.2E+01 | 0.067 | 8.5E-04 | 0.067338 | 8.5E-04 | 2.25 | 1.9E-03 |
| 2 | 2S---- | B2t19-c | 11/23/2001 | 0 | 0 | 1.030E+05 | | 2.03E+06 | 2.2E+03 | 6.47E+04 | 3.1E+01 | 0.055 | 6.5E-04 | 0.055157 | 6.5E-04 | 1.09 | 6.8E-04 |
| 2 | 2S---- | B2t19-d | 11/23/2001 | 0 | 0 | 1.030E+05 | | 2.03E+06 | 2.9E+03 | 6.51E+04 | 4.0E+01 | 0.055 | 8.1E-04 | 0.055017 | 8.1E-04 | 1.10 | 8.8E-04 |

| Build | Exp Code | Run | Bo* | +/- | Grsh,C2 | +/- | Grsh,C3 | +/- | Pr | +/- | Rot Speed (rpm) | +/- | Mass Flow (kg/s) | Ø | +/- | finØ | +/- |
|-------|----------|------------|-------|---------|----------|---------|----------|---------|-------|---------|-----------------|---------|------------------|--------|---------|--------|---------|
| 2 | 2S---- | B2run02-1 | 0.003 | 3.8E-03 | 1.30E+05 | 1.8E+06 | 1.40E+05 | 6.5E+06 | 0.705 | 1.5E-04 | 51.3 | 7.7E+01 | 0.418 | -0.008 | 1.1E-01 | -0.033 | 1.7E-02 |
| 2 | 2S---- | B2run02-2 | 0.001 | 1.7E-03 | 6.81E+04 | 8.0E+05 | 7.21E+04 | 1.0E+06 | 0.705 | 4.3E-05 | 65.9 | 6.9E+01 | 0.503 | -0.024 | 4.1E-01 | -0.025 | 2.8E-01 |
| 2 | 2S---- | B2run02-3 | 0.003 | 5.3E-03 | 1.13E+05 | 3.8E+06 | 1.24E+05 | 6.7E+06 | 0.704 | 1.7E-04 | 50.9 | 9.9E+01 | 0.415 | -0.011 | 7.4E-02 | -0.032 | 2.0E-02 |
| 2 | 2S---- | B2run03-1 | 0.001 | 1.7E-03 | 8.45E+04 | 9.7E+05 | 9.06E+04 | 1.9E+06 | 0.704 | 7.1E-05 | 31.4 | 4.7E+01 | 0.634 | -0.011 | 1.3E-01 | -0.020 | 1.4E-02 |
| 2 | 2S---- | B2run04-1 | 0.002 | 3.2E-03 | 7.27E+04 | 1.9E+06 | 7.96E+04 | 2.3E+06 | 0.705 | 9.8E-05 | 29.9 | 5.8E+01 | 0.499 | -0.013 | 8.8E-01 | -0.024 | 1.4E-01 |
| 2 | 2S---- | B2run04-2 | 0.001 | 1.9E-03 | 3.07E+04 | 6.7E+05 | 3.31E+04 | 1.6E+06 | 0.705 | 9.2E-05 | 17.7 | 3.4E+01 | 0.504 | -0.010 | 2.8E-02 | -0.021 | 1.1E-02 |
| 2 | 2S---- | B2run04-3 | 0.004 | 7.3E-03 | 1.58E+05 | 4.2E+06 | 1.69E+05 | 7.3E+06 | 0.705 | 1.8E-04 | 45.1 | 8.8E+01 | 0.333 | -0.016 | 4.9E-01 | -0.042 | 3.9E-02 |
| 2 | 2S---- | B2run04-4 | 0.002 | 4.7E-03 | 6.76E+04 | 2.1E+06 | 7.32E+04 | 3.0E+06 | 0.705 | 1.5E-04 | 28.7 | 5.6E+01 | 0.333 | -0.019 | 2.0E-01 | -0.036 | 1.0E-01 |
| 2 | 2S---- | B2run04-7 | 0.002 | 4.7E-03 | 6.64E+04 | 2.3E+06 | 7.19E+04 | 3.5E+06 | 0.705 | 1.3E-04 | 28.2 | 5.5E+01 | 0.337 | -0.019 | 6.5E-02 | -0.036 | 1.7E-02 |
| 2 | 2S---- | B2run06-1 | 0.038 | 2.4E-04 | 3.84E+07 | 2.5E+05 | 4.55E+07 | 2.9E+05 | 0.706 | 3.8E-05 | 539.5 | 2.2E-01 | 0.332 | 0.633 | 8.9E-02 | 0.506 | 8.0E-02 |
| 2 | 2S---- | B2run06-2 | 0.071 | 3.8E-04 | 1.22E+08 | 7.9E+05 | 1.37E+08 | 8.3E+05 | 0.706 | 3.7E-05 | 921.6 | 1.9E-01 | 0.333 | 0.658 | 4.1E-02 | 0.384 | 1.7E-02 |
| 2 | 2S---- | B2run08-1 | 0.157 | 8.4E-04 | 2.19E+08 | 1.8E+06 | 2.38E+08 | 1.9E+06 | 0.706 | 4.2E-05 | 1239.7 | 1.7E-01 | 0.232 | 0.447 | 1.2E-01 | 0.745 | 5.8E-02 |
| 2 | 2S---- | B2run08-2 | 0.262 | 1.9E-03 | 6.96E+08 | 6.1E+06 | 7.31E+08 | 6.3E+06 | 0.706 | 4.4E-05 | 2402.5 | 1.4E-01 | 0.231 | 1.546 | 1.5E-01 | 1.475 | 8.0E-02 |
| 2 | 2S---- | B2run08-3 | 0.491 | 3.3E-03 | 2.32E+09 | 2.0E+07 | 2.41E+09 | 2.3E+07 | 0.705 | 3.6E-05 | 4914.2 | 2.1E-01 | 0.233 | 2.241 | 2.3E-01 | 1.952 | 1.5E-01 |
| 2 | 2S---- | B2run08-4 | 0.322 | 2.6E-03 | 2.32E+09 | 2.1E+07 | 2.38E+09 | 2.1E+07 | 0.705 | 4.1E-05 | 5177.6 | 2.6E-01 | 0.340 | 1.554 | 1.3E-01 | 1.392 | 9.0E-02 |
| 2 | 2S---- | B2run08-5 | 0.244 | 1.8E-03 | 1.44E+09 | 1.1E+07 | 1.48E+09 | 1.2E+07 | 0.705 | 3.8E-05 | 3945.4 | 3.6E-01 | 0.342 | 1.400 | 1.1E-01 | 1.270 | 6.8E-02 |
| 2 | 2S---- | B2run08-6 | 0.172 | 1.1E-03 | 7.19E+08 | 5.6E+06 | 7.44E+08 | 4.9E+06 | 0.705 | 3.6E-05 | 2623.6 | 4.0E-01 | 0.339 | 1.224 | 1.0E-01 | 1.124 | 5.7E-02 |
| 2 | 2S---- | B2run08-7 | 0.149 | 1.2E-03 | 5.10E+08 | 4.8E+06 | 5.31E+08 | 4.6E+06 | 0.705 | 3.4E-05 | 2621.5 | 5.7E-01 | 0.343 | 0.781 | 1.5E-01 | 0.937 | 1.2E-01 |
| 2 | 2S---- | B2run09-1 | 0.101 | 8.6E-04 | 7.27E+08 | 6.8E+06 | 7.71E+08 | 7.7E+06 | 0.706 | 4.3E-05 | 2595.9 | 2.9E-01 | 0.560 | 0.500 | 8.3E-02 | 0.410 | 4.7E-02 |
| 2 | 2S---- | B2run09-2 | 0.176 | 2.0E-03 | 2.24E+09 | 2.6E+07 | 2.27E+09 | 2.8E+07 | 0.706 | 4.9E-05 | 5143.9 | 2.2E-01 | 0.560 | 1.080 | 2.3E-01 | 1.004 | 2.0E-01 |
| 2 | 2S---- | B2run09-3 | 0.138 | 1.2E-03 | 1.42E+09 | 1.3E+07 | 1.43E+09 | 1.3E+07 | 0.706 | 4.3E-05 | 3824.4 | 2.0E-01 | 0.560 | 0.485 | 6.0E-02 | 0.722 | 3.8E-02 |
| 2 | 2S---- | B2run09-5a | 0.048 | 4.5E-04 | 2.02E+08 | 1.7E+06 | 2.26E+08 | 1.9E+06 | 0.706 | 4.5E-05 | 1288.0 | 2.6E-01 | 0.561 | 0.400 | 3.8E-02 | 0.262 | 2.0E-02 |
| 2 | 2S---- | B2run09-5b | 0.101 | 8.0E-04 | 7.18E+08 | 6.0E+06 | 7.59E+08 | 6.5E+06 | 0.706 | 4.4E-05 | 2538.2 | 1.3E-01 | 0.560 | 0.472 | 3.7E-02 | 0.393 | 1.4E-02 |
| 2 | 2S---- | B2run09-6 | 0.086 | 8.6E-04 | 5.10E+08 | 5.1E+06 | 5.35E+08 | 5.1E+06 | 0.705 | 4.2E-05 | 2585.7 | 3.3E-01 | 0.557 | 0.421 | 4.1E-02 | 0.371 | 2.6E-02 |
| 2 | 2S---- | B2run12-1 | 0.161 | 1.3E-03 | 7.16E+08 | 6.8E+06 | 7.33E+08 | 6.7E+06 | 0.706 | 4.2E-05 | 2409.4 | 3.3E-01 | 0.355 | 1.000 | 1.5E-01 | 1.030 | 8.4E-02 |
| 2 | 2S---- | B2run12-2 | 0.310 | 2.5E-03 | 2.38E+09 | 2.1E+07 | 2.43E+09 | 2.0E+07 | 0.706 | 3.9E-05 | 4813.9 | 4.0E-01 | 0.352 | 1.361 | 8.4E-02 | 1.278 | 5.6E-02 |
| 2 | 2S---- | B2t05 | 0.134 | 9.9E-04 | 3.57E+08 | 3.3E+06 | 3.72E+08 | 3.4E+06 | 0.705 | 3.9E-05 | 2257.3 | 8.9E-02 | 0.321 | 0.917 | 9.8E-01 | 1.021 | 8.4E-01 |
| 2 | 2S---- | B2t06 | 0.057 | 5.5E-04 | 1.58E+08 | 1.5E+06 | 1.69E+08 | 1.4E+06 | 0.706 | 3.7E-05 | 1655.2 | 2.6E-01 | 0.446 | 0.563 | 9.8E-02 | 0.495 | 7.3E-02 |
| 2 | 2S---- | B2t07-a | 0.055 | 4.7E-04 | 1.80E+08 | 1.5E+06 | 1.92E+08 | 1.6E+06 | 0.705 | 3.8E-05 | 1514.9 | 2.6E-01 | 0.483 | 0.482 | 6.3E-02 | 0.436 | 3.4E-02 |

| Build | Exp Code | Run | Bo* | +/- | Grsh,C2 | +/- | Grsh,C3 | +/- | Pr | +/- | Rot Speed (rpm) | +/- | Mass Flow (kg/s) | Ø | +/- | finØ | +/- |
|-------|----------|---------|-------|---------|----------|---------|----------|---------|-------|---------|-----------------|---------|------------------|-------|---------|-------|---------|
| 2 | 2S---- | B2t07-b | 0.109 | 9.9E-04 | 5.73E+08 | 5.5E+06 | 6.03E+08 | 5.9E+06 | 0.705 | 4.2E-05 | 2924.8 | 2.8E-01 | 0.483 | 0.562 | 5.8E-02 | 0.490 | 2.8E-02 |
| 2 | 2S---- | B2t09a | 0.045 | 4.8E-04 | 1.03E+08 | 1.1E+06 | 1.11E+08 | 1.1E+06 | 0.705 | 4.0E-05 | 1353.0 | 1.4E-01 | 0.442 | 0.521 | 6.5E+00 | 0.543 | 1.0E+01 |
| 2 | 2S---- | B2t09b | 0.048 | 4.8E-04 | 1.14E+08 | 9.9E+05 | 1.24E+08 | 1.1E+06 | 0.705 | 4.2E-05 | 1353.2 | 1.9E-01 | 0.443 | 0.425 | 8.0E-02 | 0.443 | 4.2E-02 |
| 2 | 2S---- | B2t09-c | 0.048 | 4.4E-04 | 1.17E+08 | 9.8E+05 | 1.27E+08 | 1.0E+06 | 0.705 | 3.7E-05 | 1359.0 | 0.0E+00 | 0.444 | 0.423 | 6.5E-02 | 0.433 | 4.1E-02 |
| 2 | 2S---- | B2t10 | 0.171 | 1.0E-03 | 3.18E+08 | 2.4E+06 | 3.30E+08 | 2.5E+06 | 0.706 | 3.0E-05 | 1745.6 | 1.5E-01 | 0.250 | 1.049 | 2.5E-01 | 0.888 | 2.6E-01 |
| 2 | 2S---- | B2t11 | 0.092 | 9.1E-04 | 1.48E+08 | 2.0E+06 | 1.55E+08 | 1.8E+06 | 0.706 | 5.5E-05 | 1384.9 | 6.2E-02 | 0.298 | 0.870 | 2.5E-01 | 0.554 | 2.2E-01 |
| 2 | 2S---- | B2t12 | 0.128 | 9.0E-04 | 2.07E+08 | 1.9E+06 | 2.13E+08 | 1.9E+06 | 0.706 | 4.2E-05 | 1524.7 | 9.4E-02 | 0.259 | 0.924 | 1.9E-01 | 0.897 | 1.3E-01 |
| 2 | 2s---- | B2t13-a | 0.049 | 2.6E-03 | 1.55E+08 | 6.1E+06 | 1.66E+08 | 6.4E+06 | 0.705 | 1.9E-04 | 1487.8 | 8.3E-02 | 0.488 | 0.470 | 5.1E-01 | 0.475 | 2.1E+00 |
| 2 | 2S---- | B2t13-b | 0.053 | 5.3E-04 | 1.76E+08 | 1.7E+06 | 1.87E+08 | 1.5E+06 | 0.705 | 4.3E-05 | 1484.5 | 2.8E-01 | 0.493 | 0.377 | 4.7E-02 | 0.387 | 2.4E-02 |
| 2 | 2S---- | B2t14 | 0.053 | 5.8E-04 | 1.66E+08 | 1.7E+06 | 1.77E+08 | 1.8E+06 | 0.705 | 4.7E-05 | 1469.8 | 4.2E-01 | 0.484 | 0.336 | 5.3E-02 | 0.408 | 2.9E-02 |
| 2 | 2S---- | B2t15-a | 0.054 | 5.1E-04 | 1.80E+08 | 1.7E+06 | 1.91E+08 | 1.6E+06 | 0.706 | 4.2E-05 | 1485.3 | 3.8E-01 | 0.494 | 0.298 | 4.4E-02 | 0.390 | 2.1E-02 |
| 2 | 2S---- | B2t15-b | 0.105 | 9.8E-04 | 5.68E+08 | 5.7E+06 | 5.91E+08 | 5.7E+06 | 0.705 | 4.4E-05 | 2876.6 | 4.3E-01 | 0.491 | 0.542 | 6.7E-02 | 0.480 | 3.1E-02 |
| 2 | 2S---- | B2t16-a | 0.051 | 9.0E-04 | 1.63E+08 | 3.0E+06 | 1.74E+08 | 2.8E+06 | 0.705 | 7.3E-05 | 1476.0 | 2.9E+00 | 0.486 | 0.396 | 2.4E-01 | 0.471 | 1.8E-01 |
| 2 | 2S---- | B2t16-b | 0.054 | 4.7E-04 | 1.79E+08 | 1.5E+06 | 1.90E+08 | 1.6E+06 | 0.706 | 4.0E-05 | 1482.4 | 2.4E-01 | 0.486 | 0.327 | 4.9E-02 | 0.413 | 2.8E-02 |
| 2 | 2S---- | B2t16-c | 0.055 | 7.5E-04 | 1.84E+08 | 2.4E+06 | 1.96E+08 | 2.4E+06 | 0.705 | 6.3E-05 | 1503.1 | 5.0E-02 | 0.488 | 0.334 | 4.9E-02 | 0.410 | 2.9E-02 |
| 2 | 2S---- | B2t17-a | 0.105 | 1.0E-03 | 5.72E+08 | 5.7E+06 | 5.96E+08 | 6.0E+06 | 0.705 | 4.3E-05 | 2874.9 | 2.8E-01 | 0.489 | 0.576 | 7.9E-02 | 0.493 | 5.2E-02 |
| 2 | 2S---- | B2t17-b | 0.106 | 9.2E-04 | 5.99E+08 | 5.3E+06 | 6.22E+08 | 5.1E+06 | 0.705 | 3.9E-05 | 2871.1 | 9.6E-02 | 0.493 | 0.560 | 5.1E-02 | 0.491 | 2.3E-02 |
| 2 | 2S---- | B2t17-c | 0.105 | 7.7E-04 | 5.85E+08 | 4.4E+06 | 6.08E+08 | 4.7E+06 | 0.705 | 3.4E-05 | 2866.0 | 0.0E+00 | 0.493 | 0.555 | 6.6E-02 | 0.483 | 3.0E-02 |
| 2 | 2S---- | B2t18-a | 0.121 | 8.1E-04 | 1.56E+08 | 1.3E+06 | 1.65E+08 | 1.4E+06 | 0.706 | 3.7E-05 | 1270.7 | 3.0E-01 | 0.241 | 0.893 | 2.5E-01 | 0.832 | 1.7E-01 |
| 2 | 2S---- | B2t18-b | 0.122 | 7.4E-04 | 1.57E+08 | 1.3E+06 | 1.65E+08 | 1.3E+06 | 0.706 | 3.5E-05 | 1274.6 | 2.3E-01 | 0.242 | 0.899 | 1.3E-01 | 0.800 | 7.1E-02 |
| 2 | 2S---- | B2t18-c | 0.228 | 1.9E-03 | 5.99E+08 | 5.0E+06 | 6.18E+08 | 5.6E+06 | 0.706 | 3.7E-05 | 2671.3 | 1.5E-01 | 0.240 | 1.828 | 2.3E-01 | 1.714 | 2.1E-01 |
| 2 | 2S---- | B2t18-d | 0.227 | 1.8E-03 | 5.97E+08 | 5.6E+06 | 6.17E+08 | 5.9E+06 | 0.706 | 3.9E-05 | 2674.6 | 2.8E-01 | 0.242 | 1.734 | 6.0E-01 | 1.632 | 5.6E-01 |
| 2 | 2S---- | B2t19-a | 0.116 | 7.8E-04 | 1.65E+08 | 1.4E+06 | 1.73E+08 | 1.4E+06 | 0.706 | 3.8E-05 | 1292.9 | 5.1E-02 | 0.254 | 1.022 | 1.6E-01 | 0.720 | 1.5E-01 |
| 2 | 2S---- | B2t19-b | 0.115 | 8.2E-04 | 1.64E+08 | 1.5E+06 | 1.72E+08 | 1.4E+06 | 0.706 | 4.0E-05 | 1293.0 | 3.9E-02 | 0.256 | 1.011 | 1.5E-01 | 0.703 | 6.5E-02 |
| 2 | 2S---- | B2t19-c | 0.216 | 1.4E-03 | 5.95E+08 | 4.7E+06 | 6.15E+08 | 4.6E+06 | 0.706 | 3.0E-05 | 2622.5 | 2.1E-01 | 0.251 | 1.724 | 2.3E-01 | 1.611 | 1.4E-01 |
| 2 | 2S---- | B2t19-d | 0.214 | 1.7E-03 | 5.91E+08 | 5.6E+06 | 6.10E+08 | 5.6E+06 | 0.706 | 3.9E-05 | 2622.1 | 4.7E-02 | 0.253 | 1.702 | 2.0E-01 | 1.585 | 1.6E-01 |

| Build | Exp Code | Run | Tcav2 (K) | +/- | Tcav3 (K) | +/- | Tinlet (K) | +/- | Texit (K) | +/- | Dtmeas (K) | +/- | Dtcalc (K) | +/- | Tsh,C2 (K) | +/- | Tsh,C3 (K) | +/- |
|-------|----------|------------|-----------|---------|-----------|---------|------------|---------|-----------|---------|------------|---------|------------|---------|------------|---------|------------|---------|
| 2 | 2S---- | B2run02-2 | 301.81 | 9.2E+00 | 458.42 | 3.5E+01 | 317.04 | 2.7E-01 | 320.42 | 2.4E-01 | 3.38 | 3.8E-01 | -0.71 | 1.4E+00 | 327.53 | 5.9E-01 | 328.15 | 2.6E+00 |
| 2 | 2S---- | B2run02-3 | 315.63 | 4.6E+01 | 840.36 | 1.1E+02 | 323.06 | 9.9E-01 | 369.58 | 1.0E+00 | 46.53 | 1.4E+00 | 1.46 | 5.5E+00 | 362.81 | 2.7E+00 | 366.44 | 1.2E+01 |
| 2 | 2S---- | B2run03-1 | 187.55 | 4.7E+01 | 922.32 | 1.6E+02 | 324.73 | 4.2E-01 | 355.12 | 3.9E-01 | 30.40 | 5.9E-01 | -0.38 | 1.4E+00 | 369.13 | 2.6E+00 | 372.33 | 1.2E+01 |
| 2 | 2S---- | B2run04-1 | 281.49 | 4.0E+01 | 846.67 | 1.2E+02 | 318.15 | 6.0E-01 | 355.50 | 5.8E-01 | 37.35 | 7.8E-01 | 0.68 | 1.6E+00 | 354.76 | 2.3E+00 | 358.20 | 8.4E+00 |
| 2 | 2S---- | B2run04-2 | 345.35 | 6.6E+01 | 1049.07 | 1.3E+02 | 317.41 | 5.7E-01 | 351.90 | 5.4E-01 | 34.49 | 8.1E-01 | -0.35 | 1.5E+01 | 360.89 | 2.6E+00 | 364.40 | 9.9E+00 |
| 2 | 2S---- | B2run04-3 | 201.63 | 6.3E+01 | 739.68 | 1.4E+02 | 318.90 | 1.1E+00 | 359.20 | 1.2E+00 | 40.30 | 1.6E+00 | 1.55 | 2.2E+00 | 352.43 | 2.6E+00 | 354.81 | 9.2E+00 |
| 2 | 2S---- | B2run04-4 | 295.48 | 4.2E+01 | 838.52 | 1.0E+02 | 318.18 | 9.4E-01 | 356.97 | 1.0E+00 | 38.80 | 1.4E+00 | 1.13 | 2.2E+00 | 353.51 | 2.3E+00 | 356.41 | 9.7E+00 |
| 2 | 2S---- | B2run04-7 | 292.57 | 5.8E+01 | 852.65 | 1.0E+02 | 318.92 | 7.8E-01 | 357.28 | 8.4E-01 | 38.36 | 1.1E+00 | 0.96 | 2.1E+00 | 355.06 | 2.5E+00 | 358.07 | 8.8E+00 |
| 2 | 2S---- | B2run06-1 | 310.51 | 1.1E+00 | 339.82 | 1.4E+00 | 314.28 | 2.4E-01 | 320.70 | 2.2E-01 | 6.42 | 3.4E-01 | 4.66 | 6.2E-02 | 390.40 | 1.6E-01 | 404.48 | 1.7E-01 |
| 2 | 2S---- | B2run06-2 | 346.59 | 1.0E+00 | 353.96 | 1.0E+00 | 315.73 | 2.3E-01 | 320.80 | 2.4E-01 | 5.07 | 3.6E-01 | 4.66 | 6.4E-02 | 399.95 | 1.7E-01 | 410.63 | 1.7E-01 |
| 2 | 2S---- | B2run08-1 | 338.74 | 1.3E+00 | 341.42 | 1.2E+00 | 315.35 | 2.6E-01 | 322.09 | 2.4E-01 | 6.74 | 3.8E-01 | 6.02 | 1.3E-01 | 386.50 | 2.8E-01 | 392.45 | 2.6E-01 |
| 2 | 2S---- | B2run08-2 | 332.29 | 5.9E-01 | 333.02 | 5.3E-01 | 315.92 | 2.7E-01 | 323.71 | 2.3E-01 | 7.79 | 3.5E-01 | 7.80 | 8.0E-02 | 378.06 | 1.8E-01 | 381.12 | 1.6E-01 |
| 2 | 2S---- | B2run08-3 | 331.83 | 6.4E-01 | 334.00 | 8.1E-01 | 317.52 | 2.2E-01 | 328.94 | 2.2E-01 | 11.42 | 3.2E-01 | 10.15 | 1.3E-01 | 376.74 | 2.4E-01 | 378.80 | 3.1E-01 |
| 2 | 2S---- | B2run08-4 | 333.91 | 5.7E-01 | 334.97 | 5.5E-01 | 318.64 | 2.5E-01 | 327.42 | 2.7E-01 | 8.78 | 3.5E-01 | 7.11 | 6.3E-02 | 379.31 | 2.1E-01 | 380.90 | 2.0E-01 |
| 2 | 2S---- | B2run08-5 | 333.72 | 5.8E-01 | 334.65 | 4.9E-01 | 318.95 | 2.4E-01 | 325.96 | 2.3E-01 | 7.00 | 3.4E-01 | 6.25 | 6.3E-02 | 378.98 | 1.8E-01 | 380.58 | 1.9E-01 |
| 2 | 2S---- | B2run08-6 | 335.87 | 6.2E-01 | 334.64 | 6.1E-01 | 318.62 | 2.2E-01 | 324.03 | 2.5E-01 | 5.40 | 3.2E-01 | 5.50 | 6.2E-02 | 382.05 | 1.8E-01 | 384.15 | 1.7E-01 |
| 2 | 2S---- | B2run08-7 | 332.17 | 8.6E-01 | 330.74 | 7.4E-01 | 318.63 | 2.1E-01 | 322.06 | 2.1E-01 | 3.43 | 3.1E-01 | 3.43 | 7.1E-02 | 364.14 | 2.5E-01 | 365.97 | 2.0E-01 |
| 2 | 2S---- | B2run09-1 | 335.44 | 1.1E+00 | 338.04 | 1.3E+00 | 314.58 | 2.7E-01 | 317.12 | 2.9E-01 | 2.54 | 4.1E-01 | 3.02 | 6.6E-02 | 378.49 | 3.3E-01 | 382.26 | 3.7E-01 |
| 2 | 2S---- | B2run09-2 | 329.78 | 8.8E-01 | 329.46 | 1.2E+00 | 315.82 | 3.1E-01 | 320.89 | 3.1E-01 | 5.07 | 4.5E-01 | 4.26 | 6.6E-02 | 374.62 | 2.8E-01 | 375.30 | 3.8E-01 |
| 2 | 2S---- | B2run09-3 | 333.28 | 6.0E-01 | 331.98 | 5.4E-01 | 315.66 | 2.7E-01 | 319.25 | 2.7E-01 | 3.59 | 3.6E-01 | 3.66 | 4.7E-02 | 377.14 | 2.1E-01 | 377.32 | 1.9E-01 |
| 2 | 2S---- | B2run09-5a | 338.61 | 7.7E-01 | 346.71 | 1.3E+00 | 315.43 | 2.8E-01 | 317.72 | 2.6E-01 | 2.29 | 3.8E-01 | 2.67 | 4.5E-02 | 384.73 | 1.5E-01 | 392.64 | 2.5E-01 |
| 2 | 2S---- | B2run09-5b | 338.10 | 6.2E-01 | 340.06 | 5.4E-01 | 315.47 | 2.7E-01 | 318.10 | 2.8E-01 | 2.64 | 3.9E-01 | 3.02 | 3.8E-02 | 381.19 | 1.8E-01 | 384.87 | 1.5E-01 |
| 2 | 2S---- | B2run09-6 | 335.01 | 5.8E-01 | 334.90 | 6.7E-01 | 317.59 | 2.6E-01 | 319.10 | 2.7E-01 | 1.51 | 3.5E-01 | 1.98 | 3.6E-02 | 364.68 | 1.5E-01 | 366.91 | 1.8E-01 |
| 2 | 2S---- | B2run12-1 | 327.64 | 9.2E-01 | 326.98 | 6.5E-01 | 311.07 | 2.7E-01 | 315.87 | 2.7E-01 | 4.80 | 3.8E-01 | 5.08 | 6.9E-02 | 373.83 | 2.9E-01 | 375.22 | 2.1E-01 |
| 2 | 2S---- | B2run12-2 | 327.60 | 5.5E-01 | 329.05 | 4.8E-01 | 312.57 | 2.5E-01 | 320.75 | 2.4E-01 | 8.18 | 3.7E-01 | 6.66 | 7.1E-02 | 372.34 | 1.9E-01 | 373.51 | 1.8E-01 |
| 2 | 2S---- | B2t05 | 330.56 | 9.8E-01 | 329.56 | 1.1E+00 | 316.73 | 2.4E-01 | 319.97 | 2.5E-01 | 3.24 | 3.6E-01 | 4.00 | 8.1E-02 | 369.02 | 2.3E-01 | 371.24 | 2.7E-01 |

| Build | Exp Code | Run | Tcav2 (K) | +/- | Tcav3 (K) | +/- | Tinlet (K) | +/- | Texit (K) | +/- | Dtmeas (K) | +/- | Dtcalc (K) | +/- | Tsh,C2 (K) | +/- | Tsh,C3 (K) | +/- |
|-------|----------|---------|-----------|---------|-----------|---------|------------|---------|-----------|---------|------------|---------|------------|---------|------------|---------|------------|---------|
| 2 | 2S---- | B2t06 | 326.08 | 1.0E+00 | 329.06 | 1.1E+00 | 314.76 | 2.3E-01 | 315.71 | 2.4E-01 | 0.95 | 3.5E-01 | 2.28 | 4.9E-02 | 360.98 | 1.9E-01 | 364.00 | 1.9E-01 |
| 2 | 2S---- | B2t07-a | 332.41 | 8.4E-01 | 336.00 | 8.2E-01 | 317.25 | 2.4E-01 | 318.77 | 2.3E-01 | 1.52 | 3.5E-01 | 2.64 | 3.7E-02 | 374.40 | 1.6E-01 | 377.96 | 1.8E-01 |
| 2 | 2S---- | B2t07-b | 331.64 | 6.2E-01 | 334.94 | 6.1E-01 | 317.33 | 2.6E-01 | 319.37 | 2.5E-01 | 2.05 | 3.7E-01 | 2.77 | 4.3E-02 | 368.94 | 1.8E-01 | 371.52 | 1.6E-01 |
| 2 | 2S---- | B2t09a | 326.84 | 1.7E+00 | 332.12 | 2.1E+00 | 318.77 | 2.4E-01 | 319.82 | 2.3E-01 | 1.05 | 3.2E-01 | 2.37 | 7.6E-02 | 366.26 | 2.7E-01 | 370.32 | 3.6E-01 |
| 2 | 2S---- | B2t09b | 329.16 | 8.4E-01 | 334.50 | 8.6E-01 | 317.49 | 2.6E-01 | 318.78 | 2.6E-01 | 1.29 | 3.9E-01 | 2.47 | 4.3E-02 | 368.91 | 1.5E-01 | 373.20 | 1.6E-01 |
| 2 | 2S---- | B2t09-c | 329.10 | 8.3E-01 | 334.39 | 8.0E-01 | 316.90 | 2.3E-01 | 318.23 | 2.2E-01 | 1.33 | 3.0E-01 | 2.48 | 4.1E-02 | 368.91 | 1.6E-01 | 373.23 | 1.6E-01 |
| 2 | 2S---- | B2t10 | 326.82 | 8.5E-01 | 329.40 | 8.0E-01 | 313.98 | 1.9E-01 | 318.24 | 2.2E-01 | 4.26 | 3.0E-01 | 4.42 | 9.9E-02 | 363.07 | 2.2E-01 | 364.78 | 1.8E-01 |
| 2 | 2S---- | B2t11 | 330.55 | 1.9E+00 | 335.17 | 1.7E+00 | 315.77 | 3.5E-01 | 318.70 | 3.1E-01 | 2.93 | 4.5E-01 | 3.53 | 1.2E-01 | 371.99 | 4.0E-01 | 374.72 | 3.3E-01 |
| 2 | 2S---- | B2t12 | 327.98 | 7.4E-01 | 329.37 | 8.3E-01 | 313.71 | 2.7E-01 | 317.90 | 2.7E-01 | 4.19 | 3.8E-01 | 4.48 | 8.7E-02 | 370.17 | 1.6E-01 | 371.73 | 1.7E-01 |
| 2 | 2s---- | B2t13-a | 330.95 | 2.6E+00 | 335.47 | 2.6E+00 | 319.67 | 1.1E+00 | 320.55 | 1.2E+00 | 0.88 | 1.5E+00 | 2.45 | 2.1E-01 | 371.45 | 5.3E-01 | 375.05 | 5.2E-01 |
| 2 | 2S---- | B2t13-b | 331.45 | 8.7E-01 | 336.14 | 8.6E-01 | 316.40 | 2.7E-01 | 317.72 | 3.0E-01 | 1.33 | 4.1E-01 | 2.50 | 4.4E-02 | 372.51 | 1.8E-01 | 376.06 | 1.7E-01 |
| 2 | 2S---- | B2t14 | 331.97 | 8.8E-01 | 336.46 | 7.8E-01 | 317.31 | 2.9E-01 | 318.75 | 2.5E-01 | 1.44 | 3.6E-01 | 2.54 | 3.9E-02 | 373.34 | 1.7E-01 | 377.16 | 1.5E-01 |
| 2 | 2S---- | B2t15-a | 331.82 | 8.1E-01 | 335.83 | 6.6E-01 | 315.96 | 2.6E-01 | 317.37 | 2.4E-01 | 1.41 | 3.5E-01 | 2.52 | 3.8E-02 | 372.95 | 1.6E-01 | 376.47 | 1.3E-01 |
| 2 | 2S---- | B2t15-b | 334.49 | 6.6E-01 | 336.08 | 6.8E-01 | 318.45 | 2.7E-01 | 320.55 | 2.7E-01 | 2.10 | 3.8E-01 | 2.68 | 5.0E-02 | 370.78 | 1.8E-01 | 372.77 | 1.7E-01 |
| 2 | 2S---- | B2t16-a | 330.61 | 2.5E+00 | 334.50 | 1.9E+00 | 317.81 | 4.5E-01 | 318.98 | 3.8E-01 | 1.17 | 6.3E-01 | 2.55 | 8.9E-02 | 372.33 | 5.5E-01 | 375.86 | 3.9E-01 |
| 2 | 2S---- | B2t16-b | 331.86 | 8.3E-01 | 335.75 | 8.0E-01 | 316.11 | 2.5E-01 | 317.55 | 2.5E-01 | 1.44 | 3.6E-01 | 2.63 | 4.1E-02 | 374.08 | 1.7E-01 | 377.62 | 1.5E-01 |
| 2 | 2S---- | B2t16-c | 332.92 | 8.2E-01 | 336.54 | 7.7E-01 | 316.75 | 3.9E-01 | 318.17 | 3.7E-01 | 1.42 | 5.8E-01 | 2.62 | 4.1E-02 | 374.83 | 1.6E-01 | 378.33 | 1.7E-01 |
| 2 | 2S---- | B2t17-a | 334.34 | 8.2E-01 | 336.14 | 9.6E-01 | 318.55 | 2.6E-01 | 320.64 | 2.6E-01 | 2.08 | 3.9E-01 | 2.74 | 5.6E-02 | 371.13 | 2.2E-01 | 373.26 | 2.7E-01 |
| 2 | 2S---- | B2t17-b | 333.30 | 6.4E-01 | 334.97 | 5.6E-01 | 317.09 | 2.4E-01 | 319.29 | 2.4E-01 | 2.20 | 3.4E-01 | 2.78 | 4.0E-02 | 370.76 | 1.6E-01 | 372.74 | 1.7E-01 |
| 2 | 2S---- | B2t17-c | 334.77 | 7.0E-01 | 336.32 | 6.3E-01 | 318.24 | 2.1E-01 | 320.42 | 2.5E-01 | 2.17 | 3.3E-01 | 2.76 | 3.8E-02 | 371.83 | 1.8E-01 | 373.90 | 1.8E-01 |
| 2 | 2S---- | B2t18-a | 323.92 | 1.1E+00 | 327.02 | 1.2E+00 | 310.64 | 2.4E-01 | 314.90 | 2.6E-01 | 4.26 | 3.3E-01 | 4.37 | 9.3E-02 | 365.19 | 2.0E-01 | 368.12 | 2.0E-01 |
| 2 | 2S---- | B2t18-b | 325.29 | 8.4E-01 | 328.38 | 8.6E-01 | 311.50 | 2.3E-01 | 315.82 | 2.4E-01 | 4.31 | 3.5E-01 | 4.34 | 8.1E-02 | 366.41 | 1.7E-01 | 369.29 | 1.7E-01 |
| 2 | 2S---- | B2t18-c | 324.60 | 6.7E-01 | 323.85 | 5.5E-01 | 312.43 | 2.3E-01 | 318.00 | 2.6E-01 | 5.56 | 3.6E-01 | 6.02 | 8.5E-02 | 362.91 | 1.7E-01 | 364.50 | 1.5E-01 |
| 2 | 2S---- | B2t18-d | 325.43 | 5.3E-01 | 324.94 | 6.7E-01 | 313.12 | 2.5E-01 | 318.70 | 2.4E-01 | 5.58 | 3.6E-01 | 5.95 | 8.9E-02 | 363.72 | 1.6E-01 | 365.38 | 1.9E-01 |
| 2 | 2S---- | B2t19-a | 324.89 | 9.2E-01 | 328.66 | 1.0E+00 | 310.77 | 2.4E-01 | 314.86 | 2.6E-01 | 4.09 | 3.9E-01 | 4.19 | 8.0E-02 | 366.49 | 1.8E-01 | 369.19 | 2.0E-01 |
| 2 | 2S---- | B2t19-b | 325.46 | 8.6E-01 | 329.29 | 7.8E-01 | 311.18 | 2.6E-01 | 315.28 | 2.2E-01 | 4.11 | 3.4E-01 | 4.14 | 8.6E-02 | 366.95 | 1.7E-01 | 369.68 | 1.6E-01 |
| 2 | 2S---- | B2t19-c | 323.53 | 6.7E-01 | 322.91 | 6.1E-01 | 311.15 | 1.9E-01 | 316.54 | 2.2E-01 | 5.39 | 3.0E-01 | 5.83 | 8.5E-02 | 362.59 | 1.9E-01 | 364.23 | 1.8E-01 |
| 2 | 2S---- | B2t19-d | 324.41 | 5.7E-01 | 323.72 | 6.2E-01 | 311.91 | 2.5E-01 | 317.31 | 2.3E-01 | 5.40 | 3.8E-01 | 5.77 | 7.3E-02 | 363.36 | 1.6E-01 | 364.94 | 1.8E-01 |

| Build | Exp Code | Run | Tcav2 (K) | +/- | Tcav3 (K) | +/- | Tinlet (K) | +/- | Texit (K) | +/- | Dtmeas (K) | +/- | Dtcalc (K) | +/- | Tsh,C2 (K) | +/- | Tsh,C3 (K) | +/- |
|-------|----------|------------|--------------|---------|--------------|---------|---------------|---------|--------------|---------|---------------|---------|---------------|---------|---------------|---------|---------------|---------|
| 2 | 2S---- | B2run02-2 | 301.81 | 9.2E+00 | 458.42 | 3.5E+01 | 317.04 | 2.7E-01 | 320.42 | 2.4E-01 | 3.38 | 3.8E-01 | -0.71 | 1.4E+00 | 327.53 | 5.9E-01 | 328.15 | 2.6E+00 |
| 2 | 2S---- | B2run02-3 | 315.63 | 4.6E+01 | 840.36 | 1.1E+02 | 323.06 | 9.9E-01 | 369.58 | 1.0E+00 | 46.53 | 1.4E+00 | 1.46 | 5.5E+00 | 362.81 | 2.7E+00 | 366.44 | 1.2E+01 |
| 2 | 2S---- | B2run03-1 | 187.55 | 4.7E+01 | 922.32 | 1.6E+02 | 324.73 | 4.2E-01 | 355.12 | 3.9E-01 | 30.40 | 5.9E-01 | -0.38 | 1.4E+00 | 369.13 | 2.6E+00 | 372.33 | 1.2E+01 |
| 2 | 2S---- | B2run04-1 | 281.49 | 4.0E+01 | 846.67 | 1.2E+02 | 318.15 | 6.0E-01 | 355.50 | 5.8E-01 | 37.35 | 7.8E-01 | 0.68 | 1.6E+00 | 354.76 | 2.3E+00 | 358.20 | 8.4E+00 |
| 2 | 2S---- | B2run04-2 | 345.35 | 6.6E+01 | 1049.07 | 1.3E+02 | 317.41 | 5.7E-01 | 351.90 | 5.4E-01 | 34.49 | 8.1E-01 | -0.35 | 1.5E+01 | 360.89 | 2.6E+00 | 364.40 | 9.9E+00 |
| 2 | 2S---- | B2run04-3 | 201.63 | 6.3E+01 | 739.68 | 1.4E+02 | 318.90 | 1.1E+00 | 359.20 | 1.2E+00 | 40.30 | 1.6E+00 | 1.55 | 2.2E+00 | 352.43 | 2.6E+00 | 354.81 | 9.2E+00 |
| 2 | 2S---- | B2run04-4 | 295.48 | 4.2E+01 | 838.52 | 1.0E+02 | 318.18 | 9.4E-01 | 356.97 | 1.0E+00 | 38.80 | 1.4E+00 | 1.13 | 2.2E+00 | 353.51 | 2.3E+00 | 356.41 | 9.7E+00 |
| 2 | 2S---- | B2run04-7 | 292.57 | 5.8E+01 | 852.65 | 1.0E+02 | 318.92 | 7.8E-01 | 357.28 | 8.4E-01 | 38.36 | 1.1E+00 | 0.96 | 2.1E+00 | 355.06 | 2.5E+00 | 358.07 | 8.8E+00 |
| 2 | 2S---- | B2run06-1 | 310.51 | 1.1E+00 | 339.82 | 1.4E+00 | 314.28 | 2.4E-01 | 320.70 | 2.2E-01 | 6.42 | 3.4E-01 | 4.66 | 6.2E-02 | 390.40 | 1.6E-01 | 404.48 | 1.7E-01 |
| 2 | 2S---- | B2run06-2 | 346.59 | 1.0E+00 | 353.96 | 1.0E+00 | 315.73 | 2.3E-01 | 320.80 | 2.4E-01 | 5.07 | 3.6E-01 | 4.66 | 6.4E-02 | 399.95 | 1.7E-01 | 410.63 | 1.7E-01 |
| 2 | 2S---- | B2run08-1 | 338.74 | 1.3E+00 | 341.42 | 1.2E+00 | 315.35 | 2.6E-01 | 322.09 | 2.4E-01 | 6.74 | 3.8E-01 | 6.02 | 1.3E-01 | 386.50 | 2.8E-01 | 392.45 | 2.6E-01 |
| 2 | 2S---- | B2run08-2 | 332.29 | 5.9E-01 | 333.02 | 5.3E-01 | 315.92 | 2.7E-01 | 323.71 | 2.3E-01 | 7.79 | 3.5E-01 | 7.80 | 8.0E-02 | 378.06 | 1.8E-01 | 381.12 | 1.6E-01 |
| 2 | 2S---- | B2run08-3 | 331.83 | 6.4E-01 | 334.00 | 8.1E-01 | 317.52 | 2.2E-01 | 328.94 | 2.2E-01 | 11.42 | 3.2E-01 | 10.15 | 1.3E-01 | 376.74 | 2.4E-01 | 378.80 | 3.1E-01 |
| 2 | 2S---- | B2run08-4 | 333.91 | 5.7E-01 | 334.97 | 5.5E-01 | 318.64 | 2.5E-01 | 327.42 | 2.7E-01 | 8.78 | 3.5E-01 | 7.11 | 6.3E-02 | 379.31 | 2.1E-01 | 380.90 | 2.0E-01 |
| 2 | 2S---- | B2run08-5 | 333.72 | 5.8E-01 | 334.65 | 4.9E-01 | 318.95 | 2.4E-01 | 325.96 | 2.3E-01 | 7.00 | 3.4E-01 | 6.25 | 6.3E-02 | 378.98 | 1.8E-01 | 380.58 | 1.9E-01 |
| 2 | 2S---- | B2run08-6 | 335.87 | 6.2E-01 | 334.64 | 6.1E-01 | 318.62 | 2.2E-01 | 324.03 | 2.5E-01 | 5.40 | 3.2E-01 | 5.50 | 6.2E-02 | 382.05 | 1.8E-01 | 384.15 | 1.7E-01 |
| 2 | 2S---- | B2run08-7 | 332.17 | 8.6E-01 | 330.74 | 7.4E-01 | 318.63 | 2.1E-01 | 322.06 | 2.1E-01 | 3.43 | 3.1E-01 | 3.43 | 7.1E-02 | 364.14 | 2.5E-01 | 365.97 | 2.0E-01 |
| 2 | 2S---- | B2run09-1 | 335.44 | 1.1E+00 | 338.04 | 1.3E+00 | 314.58 | 2.7E-01 | 317.12 | 2.9E-01 | 2.54 | 4.1E-01 | 3.02 | 6.6E-02 | 378.49 | 3.3E-01 | 382.26 | 3.7E-01 |
| 2 | 2S---- | B2run09-2 | 329.78 | 8.8E-01 | 329.46 | 1.2E+00 | 315.82 | 3.1E-01 | 320.89 | 3.1E-01 | 5.07 | 4.5E-01 | 4.26 | 6.6E-02 | 374.62 | 2.8E-01 | 375.30 | 3.8E-01 |
| 2 | 2S---- | B2run09-3 | 333.28 | 6.0E-01 | 331.98 | 5.4E-01 | 315.66 | 2.7E-01 | 319.25 | 2.7E-01 | 3.59 | 3.6E-01 | 3.66 | 4.7E-02 | 377.14 | 2.1E-01 | 377.32 | 1.9E-01 |
| 2 | 2S---- | B2run09-5a | 338.61 | 7.7E-01 | 346.71 | 1.3E+00 | 315.43 | 2.8E-01 | 317.72 | 2.6E-01 | 2.29 | 3.8E-01 | 2.67 | 4.5E-02 | 384.73 | 1.5E-01 | 392.64 | 2.5E-01 |
| 2 | 2S---- | B2run09-5b | 338.10 | 6.2E-01 | 340.06 | 5.4E-01 | 315.47 | 2.7E-01 | 318.10 | 2.8E-01 | 2.64 | 3.9E-01 | 3.02 | 3.8E-02 | 381.19 | 1.8E-01 | 384.87 | 1.5E-01 |
| 2 | 2S---- | B2run09-6 | 335.01 | 5.8E-01 | 334.90 | 6.7E-01 | 317.59 | 2.6E-01 | 319.10 | 2.7E-01 | 1.51 | 3.5E-01 | 1.98 | 3.6E-02 | 364.68 | 1.5E-01 | 366.91 | 1.8E-01 |
| 2 | 2S---- | B2run12-1 | 327.64 | 9.2E-01 | 326.98 | 6.5E-01 | 311.07 | 2.7E-01 | 315.87 | 2.7E-01 | 4.80 | 3.8E-01 | 5.08 | 6.9E-02 | 373.83 | 2.9E-01 | 375.22 | 2.1E-01 |
| 2 | 2S---- | B2run12-2 | 327.60 | 5.5E-01 | 329.05 | 4.8E-01 | 312.57 | 2.5E-01 | 320.75 | 2.4E-01 | 8.18 | 3.7E-01 | 6.66 | 7.1E-02 | 372.34 | 1.9E-01 | 373.51 | 1.8E-01 |
| 2 | 2S---- | B2t05 | 330.56 | 9.8E-01 | 329.56 | 1.1E+00 | 316.73 | 2.4E-01 | 319.97 | 2.5E-01 | 3.24 | 3.6E-01 | 4.00 | 8.1E-02 | 369.02 | 2.3E-01 | 371.24 | 2.7E-01 |

| Build | Exp Code | Run | Tcav2 (K) | +/- | Tcav3 (K) | +/- | Tinlet (K) | +/- | Texit (K) | +/- | Dtmeas (K) | +/- | Dtcalc (K) | +/- | Tsh,C2 (K) | +/- | Tsh,C3 (K) | +/- |
|-------|----------|---------|-----------|---------|-----------|---------|------------|---------|-----------|---------|------------|---------|------------|---------|------------|---------|------------|---------|
| 2 | 2S---- | B2t06 | 326.08 | 1.0E+00 | 329.06 | 1.1E+00 | 314.76 | 2.3E-01 | 315.71 | 2.4E-01 | 0.95 | 3.5E-01 | 2.28 | 4.9E-02 | 360.98 | 1.9E-01 | 364.00 | 1.9E-01 |
| 2 | 2S---- | B2t07-a | 332.41 | 8.4E-01 | 336.00 | 8.2E-01 | 317.25 | 2.4E-01 | 318.77 | 2.3E-01 | 1.52 | 3.5E-01 | 2.64 | 3.7E-02 | 374.40 | 1.6E-01 | 377.96 | 1.8E-01 |
| 2 | 2S---- | B2t07-b | 331.64 | 6.2E-01 | 334.94 | 6.1E-01 | 317.33 | 2.6E-01 | 319.37 | 2.5E-01 | 2.05 | 3.7E-01 | 2.77 | 4.3E-02 | 368.94 | 1.8E-01 | 371.52 | 1.6E-01 |
| 2 | 2S---- | B2t09a | 326.84 | 1.7E+00 | 332.12 | 2.1E+00 | 318.77 | 2.4E-01 | 319.82 | 2.3E-01 | 1.05 | 3.2E-01 | 2.37 | 7.6E-02 | 366.26 | 2.7E-01 | 370.32 | 3.6E-01 |
| 2 | 2S---- | B2t09b | 329.16 | 8.4E-01 | 334.50 | 8.6E-01 | 317.49 | 2.6E-01 | 318.78 | 2.6E-01 | 1.29 | 3.9E-01 | 2.47 | 4.3E-02 | 368.91 | 1.5E-01 | 373.20 | 1.6E-01 |
| 2 | 2S---- | B2t09-c | 329.10 | 8.3E-01 | 334.39 | 8.0E-01 | 316.90 | 2.3E-01 | 318.23 | 2.2E-01 | 1.33 | 3.0E-01 | 2.48 | 4.1E-02 | 368.91 | 1.6E-01 | 373.23 | 1.6E-01 |
| 2 | 2S---- | B2t10 | 326.82 | 8.5E-01 | 329.40 | 8.0E-01 | 313.98 | 1.9E-01 | 318.24 | 2.2E-01 | 4.26 | 3.0E-01 | 4.42 | 9.9E-02 | 363.07 | 2.2E-01 | 364.78 | 1.8E-01 |
| 2 | 2S---- | B2t11 | 330.55 | 1.9E+00 | 335.17 | 1.7E+00 | 315.77 | 3.5E-01 | 318.70 | 3.1E-01 | 2.93 | 4.5E-01 | 3.53 | 1.2E-01 | 371.99 | 4.0E-01 | 374.72 | 3.3E-01 |
| 2 | 2S---- | B2t12 | 327.98 | 7.4E-01 | 329.37 | 8.3E-01 | 313.71 | 2.7E-01 | 317.90 | 2.7E-01 | 4.19 | 3.8E-01 | 4.48 | 8.7E-02 | 370.17 | 1.6E-01 | 371.73 | 1.7E-01 |
| 2 | 2s---- | B2t13-a | 330.95 | 2.6E+00 | 335.47 | 2.6E+00 | 319.67 | 1.1E+00 | 320.55 | 1.2E+00 | 0.88 | 1.5E+00 | 2.45 | 2.1E-01 | 371.45 | 5.3E-01 | 375.05 | 5.2E-01 |
| 2 | 2S---- | B2t13-b | 331.45 | 8.7E-01 | 336.14 | 8.6E-01 | 316.40 | 2.7E-01 | 317.72 | 3.0E-01 | 1.33 | 4.1E-01 | 2.50 | 4.4E-02 | 372.51 | 1.8E-01 | 376.06 | 1.7E-01 |
| 2 | 2S---- | B2t14 | 331.97 | 8.8E-01 | 336.46 | 7.8E-01 | 317.31 | 2.9E-01 | 318.75 | 2.5E-01 | 1.44 | 3.6E-01 | 2.54 | 3.9E-02 | 373.34 | 1.7E-01 | 377.16 | 1.5E-01 |
| 2 | 2S---- | B2t15-a | 331.82 | 8.1E-01 | 335.83 | 6.6E-01 | 315.96 | 2.6E-01 | 317.37 | 2.4E-01 | 1.41 | 3.5E-01 | 2.52 | 3.8E-02 | 372.95 | 1.6E-01 | 376.47 | 1.3E-01 |
| 2 | 2S---- | B2t15-b | 334.49 | 6.6E-01 | 336.08 | 6.8E-01 | 318.45 | 2.7E-01 | 320.55 | 2.7E-01 | 2.10 | 3.8E-01 | 2.68 | 5.0E-02 | 370.78 | 1.8E-01 | 372.77 | 1.7E-01 |
| 2 | 2S---- | B2t16-a | 330.61 | 2.5E+00 | 334.50 | 1.9E+00 | 317.81 | 4.5E-01 | 318.98 | 3.8E-01 | 1.17 | 6.3E-01 | 2.55 | 8.9E-02 | 372.33 | 5.5E-01 | 375.86 | 3.9E-01 |
| 2 | 2S---- | B2t16-b | 331.86 | 8.3E-01 | 335.75 | 8.0E-01 | 316.11 | 2.5E-01 | 317.55 | 2.5E-01 | 1.44 | 3.6E-01 | 2.63 | 4.1E-02 | 374.08 | 1.7E-01 | 377.62 | 1.5E-01 |
| 2 | 2S---- | B2t16-c | 332.92 | 8.2E-01 | 336.54 | 7.7E-01 | 316.75 | 3.9E-01 | 318.17 | 3.7E-01 | 1.42 | 5.8E-01 | 2.62 | 4.1E-02 | 374.83 | 1.6E-01 | 378.33 | 1.7E-01 |
| 2 | 2S---- | B2t17-a | 334.34 | 8.2E-01 | 336.14 | 9.6E-01 | 318.55 | 2.6E-01 | 320.64 | 2.6E-01 | 2.08 | 3.9E-01 | 2.74 | 5.6E-02 | 371.13 | 2.2E-01 | 373.26 | 2.7E-01 |
| 2 | 2S---- | B2t17-b | 333.30 | 6.4E-01 | 334.97 | 5.6E-01 | 317.09 | 2.4E-01 | 319.29 | 2.4E-01 | 2.20 | 3.4E-01 | 2.78 | 4.0E-02 | 370.76 | 1.6E-01 | 372.74 | 1.7E-01 |
| 2 | 2S---- | B2t17-c | 334.77 | 7.0E-01 | 336.32 | 6.3E-01 | 318.24 | 2.1E-01 | 320.42 | 2.5E-01 | 2.17 | 3.3E-01 | 2.76 | 3.8E-02 | 371.83 | 1.8E-01 | 373.90 | 1.8E-01 |
| 2 | 2S---- | B2t18-a | 323.92 | 1.1E+00 | 327.02 | 1.2E+00 | 310.64 | 2.4E-01 | 314.90 | 2.6E-01 | 4.26 | 3.3E-01 | 4.37 | 9.3E-02 | 365.19 | 2.0E-01 | 368.12 | 2.0E-01 |
| 2 | 2S---- | B2t18-b | 325.29 | 8.4E-01 | 328.38 | 8.6E-01 | 311.50 | 2.3E-01 | 315.82 | 2.4E-01 | 4.31 | 3.5E-01 | 4.34 | 8.1E-02 | 366.41 | 1.7E-01 | 369.29 | 1.7E-01 |
| 2 | 2S---- | B2t18-c | 324.60 | 6.7E-01 | 323.85 | 5.5E-01 | 312.43 | 2.3E-01 | 318.00 | 2.6E-01 | 5.56 | 3.6E-01 | 6.02 | 8.5E-02 | 362.91 | 1.7E-01 | 364.50 | 1.5E-01 |
| 2 | 2S---- | B2t18-d | 325.43 | 5.3E-01 | 324.94 | 6.7E-01 | 313.12 | 2.5E-01 | 318.70 | 2.4E-01 | 5.58 | 3.6E-01 | 5.95 | 8.9E-02 | 363.72 | 1.6E-01 | 365.38 | 1.9E-01 |
| 2 | 2S---- | B2t19-a | 324.89 | 9.2E-01 | 328.66 | 1.0E+00 | 310.77 | 2.4E-01 | 314.86 | 2.6E-01 | 4.09 | 3.9E-01 | 4.19 | 8.0E-02 | 366.49 | 1.8E-01 | 369.19 | 2.0E-01 |
| 2 | 2S---- | B2t19-b | 325.46 | 8.6E-01 | 329.29 | 7.8E-01 | 311.18 | 2.6E-01 | 315.28 | 2.2E-01 | 4.11 | 3.4E-01 | 4.14 | 8.6E-02 | 366.95 | 1.7E-01 | 369.68 | 1.6E-01 |
| 2 | 2S---- | B2t19-c | 323.53 | 6.7E-01 | 322.91 | 6.1E-01 | 311.15 | 1.9E-01 | 316.54 | 2.2E-01 | 5.39 | 3.0E-01 | 5.83 | 8.5E-02 | 362.59 | 1.9E-01 | 364.23 | 1.8E-01 |
| 2 | 2S---- | B2t19-d | 324.41 | 5.7E-01 | 323.72 | 6.2E-01 | 311.91 | 2.5E-01 | 317.31 | 2.3E-01 | 5.40 | 3.8E-01 | 5.77 | 7.3E-02 | 363.36 | 1.6E-01 | 364.94 | 1.8E-01 |

| Build | Exp Code | Run | Pcav (abs Pa) | Ts,C2,d,av (K) | +/- | Ts,C3,u,av (K) | +/- | TS,C3,d,av (K) | +/- | Ts,C4,u,av (K) | +/- | finTs,D2,av (K) | +/- | finTs,D3,av (K) | +/- |
|-------|----------|------------|---------------|----------------|---------|----------------|---------|----------------|---------|----------------|---------|-----------------|---------|-----------------|---------|
| 2 | 2S---- | B2run02-2 | 2.479E+05 | 319.54 | 4.4E-01 | 325.69 | 8.3E-01 | 319.76 | 5.0E-01 | 325.97 | 9.7E-01 | 322.62 | 4.9E-01 | 322.87 | 5.5E-01 |
| 2 | 2S---- | B2run02-3 | 2.219E+05 | 333.10 | 6.3E-01 | 360.69 | 1.6E+00 | 335.65 | 6.5E-01 | 367.27 | 2.0E+00 | 346.89 | 8.7E-01 | 351.46 | 1.0E+00 |
| 2 | 2S---- | B2run03-1 | 2.974E+05 | 332.06 | 1.1E+00 | 365.30 | 3.0E+00 | 332.56 | 1.0E+00 | 368.95 | 3.0E+00 | 348.68 | 1.6E+00 | 350.75 | 1.5E+00 |
| 2 | 2S---- | B2run04-1 | 3.049E+05 | 328.15 | 5.5E-01 | 353.21 | 1.7E+00 | 330.50 | 5.6E-01 | 358.39 | 1.7E+00 | 340.68 | 8.7E-01 | 344.44 | 9.2E-01 |
| 2 | 2S---- | B2run04-2 | 3.050E+05 | 327.48 | 8.5E-01 | 358.75 | 1.8E+00 | 327.97 | 8.9E-01 | 362.87 | 2.2E+00 | 343.12 | 9.7E-01 | 345.42 | 1.3E+00 |
| 2 | 2S---- | B2run04-3 | 3.122E+05 | 328.36 | 5.0E-01 | 350.47 | 1.4E+00 | 330.56 | 5.1E-01 | 356.08 | 2.0E+00 | 339.41 | 7.7E-01 | 343.32 | 1.1E+00 |
| 2 | 2S---- | B2run04-4 | 3.109E+05 | 327.08 | 5.7E-01 | 352.36 | 1.8E+00 | 329.32 | 6.1E-01 | 357.30 | 1.8E+00 | 339.72 | 9.6E-01 | 343.31 | 9.3E-01 |
| 2 | 2S---- | B2run04-7 | 3.117E+05 | 328.35 | 6.1E-01 | 353.54 | 1.6E+00 | 330.68 | 5.2E-01 | 358.62 | 1.9E+00 | 340.94 | 8.7E-01 | 344.65 | 9.6E-01 |
| 2 | 2S---- | B2run06-1 | 2.618E+05 | 332.99 | 1.2E-01 | 333.58 | 1.1E-01 | 339.06 | 1.1E-01 | 340.05 | 1.2E-01 | 333.29 | 7.8E-02 | 339.56 | 8.4E-02 |
| 2 | 2S---- | B2run06-2 | 2.625E+05 | 340.58 | 1.2E-01 | 339.70 | 1.2E-01 | 345.10 | 1.3E-01 | 346.14 | 1.3E-01 | 340.14 | 8.6E-02 | 345.62 | 9.0E-02 |
| 2 | 2S---- | B2run08-1 | 2.851E+05 | 342.55 | 1.5E-01 | 342.60 | 2.0E-01 | 348.53 | 1.7E-01 | 347.28 | 2.2E-01 | 342.57 | 1.4E-01 | 347.90 | 1.4E-01 |
| 2 | 2S---- | B2run08-2 | 2.859E+05 | 336.85 | 1.2E-01 | 335.97 | 1.3E-01 | 339.87 | 1.2E-01 | 339.02 | 1.2E-01 | 336.41 | 9.0E-02 | 339.44 | 8.5E-02 |
| 2 | 2S---- | B2run08-3 | 2.835E+05 | 335.90 | 1.6E-01 | 334.15 | 1.6E-01 | 339.34 | 1.5E-01 | 338.15 | 1.3E-01 | 335.03 | 1.2E-01 | 338.75 | 9.5E-02 |
| 2 | 2S---- | B2run08-4 | 2.702E+05 | 335.99 | 1.3E-01 | 334.34 | 1.3E-01 | 338.24 | 1.4E-01 | 337.00 | 1.4E-01 | 335.17 | 1.0E-01 | 337.62 | 8.5E-02 |
| 2 | 2S---- | B2run08-5 | 2.700E+05 | 336.38 | 1.1E-01 | 334.92 | 1.1E-01 | 338.49 | 1.2E-01 | 337.33 | 1.2E-01 | 335.65 | 7.8E-02 | 337.91 | 8.5E-02 |
| 2 | 2S---- | B2run08-6 | 2.695E+05 | 337.59 | 1.4E-01 | 336.69 | 1.1E-01 | 340.08 | 1.0E-01 | 339.36 | 1.2E-01 | 337.14 | 8.5E-02 | 339.72 | 7.8E-02 |
| 2 | 2S---- | B2run08-7 | 2.695E+05 | 332.90 | 1.5E-01 | 332.49 | 1.3E-01 | 335.31 | 1.6E-01 | 334.40 | 1.5E-01 | 332.69 | 9.9E-02 | 334.85 | 1.1E-01 |
| 2 | 2S---- | B2run09-1 | 2.650E+05 | 333.03 | 1.7E-01 | 331.76 | 1.5E-01 | 334.57 | 1.7E-01 | 333.90 | 1.7E-01 | 332.40 | 1.0E-01 | 334.23 | 1.2E-01 |
| 2 | 2S---- | B2run09-2 | 2.666E+05 | 330.26 | 1.5E-01 | 328.77 | 1.4E-01 | 331.82 | 1.6E-01 | 330.55 | 1.5E-01 | 329.52 | 1.0E-01 | 331.18 | 1.1E-01 |
| 2 | 2S---- | B2run09-3 | 2.662E+05 | 331.16 | 1.4E-01 | 330.84 | 1.8E-01 | 333.93 | 1.7E-01 | 332.11 | 1.3E-01 | 331.00 | 1.3E-01 | 333.02 | 1.0E-01 |
| 2 | 2S---- | B2run09-5a | 2.673E+05 | 331.89 | 1.6E-01 | 330.90 | 1.4E-01 | 333.70 | 1.5E-01 | 334.01 | 1.7E-01 | 331.39 | 1.1E-01 | 333.85 | 1.1E-01 |
| 2 | 2S---- | B2run09-5b | 2.669E+05 | 334.79 | 1.3E-01 | 333.58 | 1.4E-01 | 336.46 | 1.2E-01 | 335.77 | 1.3E-01 | 334.19 | 9.8E-02 | 336.11 | 9.3E-02 |
| 2 | 2S---- | B2run09-6 | 2.663E+05 | 331.56 | 1.3E-01 | 330.45 | 1.2E-01 | 332.62 | 1.2E-01 | 331.70 | 1.1E-01 | 331.00 | 9.7E-02 | 332.16 | 9.0E-02 |
| 2 | 2S---- | B2run12-1 | 2.780E+05 | 329.99 | 1.7E-01 | 329.15 | 1.3E-01 | 332.31 | 1.5E-01 | 331.22 | 1.5E-01 | 329.57 | 1.1E-01 | 331.77 | 9.7E-02 |
| 2 | 2S---- | B2run12-2 | 2.809E+05 | 330.13 | 1.4E-01 | 328.61 | 1.2E-01 | 332.21 | 1.2E-01 | 330.86 | 1.4E-01 | 329.37 | 9.7E-02 | 331.54 | 9.0E-02 |
| 2 | 2S---- | B2t05 | 2.387E+05 | 333.91 | 1.7E-01 | 333.45 | 1.6E-01 | 336.42 | 1.4E-01 | 335.58 | 1.5E-01 | 333.68 | 1.2E-01 | 336.00 | 1.1E-01 |
| 2 | 2S---- | B2t06 | 2.254E+05 | 327.60 | 1.1E-01 | 326.97 | 1.2E-01 | 328.52 | 1.2E-01 | 328.05 | 1.1E-01 | 327.28 | 9.0E-02 | 328.29 | 8.5E-02 |

| Build | Exp Code | Run | Pcav (abs Pa) | Ts,C2,d,av (K) | +/- | Ts,C3,u,av (K) | +/- | TS,C3,d,av (K) | +/- | Ts,C4,u,av (K) | +/- | finTs,D2,av (K) | +/- | finTs,D3,av (K) | +/- |
|-------|----------|---------|---------------|----------------|---------|----------------|---------|----------------|---------|----------------|---------|-----------------|---------|-----------------|---------|
| 2 | 2S---- | B2t07-a | 2.400E+05 | 332.34 | 1.1E-01 | 331.89 | 1.3E-01 | 333.42 | 1.2E-01 | 333.08 | 1.1E-01 | 332.11 | 7.9E-02 | 333.25 | 7.7E-02 |
| 2 | 2S---- | B2t07-b | 2.395E+05 | 333.00 | 1.2E-01 | 332.11 | 1.2E-01 | 334.68 | 1.3E-01 | 334.07 | 1.3E-01 | 332.55 | 8.2E-02 | 334.38 | 9.6E-02 |
| 2 | 2S---- | B2t09a | 2.245E+05 | 330.83 | 1.4E-01 | 330.62 | 1.4E-01 | 332.24 | 1.4E-01 | 331.82 | 1.2E-01 | 330.73 | 1.0E-01 | 332.03 | 9.2E-02 |
| 2 | 2S---- | B2t09b | 2.253E+05 | 330.78 | 1.2E-01 | 330.63 | 1.1E-01 | 332.16 | 9.8E-02 | 331.79 | 1.2E-01 | 330.71 | 8.5E-02 | 331.97 | 7.4E-02 |
| 2 | 2S---- | B2t09-c | 2.253E+05 | 330.33 | 1.1E-01 | 330.18 | 1.1E-01 | 331.71 | 1.1E-01 | 331.37 | 1.1E-01 | 330.26 | 7.5E-02 | 331.54 | 7.6E-02 |
| 2 | 2S---- | B2t10 | 2.929E+05 | 332.45 | 1.6E-01 | 331.64 | 1.2E-01 | 334.92 | 1.3E-01 | 334.40 | 1.1E-01 | 332.04 | 9.9E-02 | 334.66 | 8.8E-02 |
| 2 | 2S---- | B2t11 | 2.370E+05 | 334.60 | 2.4E-01 | 333.59 | 1.9E-01 | 336.47 | 2.4E-01 | 336.49 | 2.3E-01 | 334.09 | 1.4E-01 | 336.48 | 1.8E-01 |
| 2 | 2S---- | B2t12 | 2.509E+05 | 333.56 | 1.3E-01 | 333.11 | 1.6E-01 | 336.17 | 1.4E-01 | 335.68 | 1.5E-01 | 333.33 | 1.1E-01 | 335.93 | 9.6E-02 |
| 2 | 2s---- | B2t13-a | 2.420E+05 | 332.40 | 4.7E-01 | 332.03 | 4.4E-01 | 333.58 | 5.0E-01 | 333.05 | 4.4E-01 | 332.21 | 3.4E-01 | 333.32 | 3.6E-01 |
| 2 | 2S---- | B2t13-b | 2.426E+05 | 330.96 | 1.4E-01 | 330.68 | 1.2E-01 | 332.10 | 1.2E-01 | 331.61 | 1.3E-01 | 330.82 | 9.5E-02 | 331.86 | 9.5E-02 |
| 2 | 2S---- | B2t14 | 2.395E+05 | 331.88 | 1.4E-01 | 331.75 | 1.2E-01 | 333.34 | 1.1E-01 | 332.64 | 1.2E-01 | 331.82 | 9.1E-02 | 332.99 | 8.2E-02 |
| 2 | 2S---- | B2t15-a | 2.425E+05 | 330.86 | 1.0E-01 | 330.81 | 1.3E-01 | 332.32 | 9.8E-02 | 331.55 | 1.0E-01 | 330.84 | 8.9E-02 | 331.93 | 6.5E-02 |
| 2 | 2S---- | B2t15-b | 2.423E+05 | 333.83 | 1.6E-01 | 332.90 | 1.4E-01 | 335.49 | 1.5E-01 | 334.78 | 1.3E-01 | 333.36 | 1.1E-01 | 335.14 | 1.1E-01 |
| 2 | 2S---- | B2t16-a | 2.407E+05 | 331.48 | 2.4E-01 | 331.29 | 2.2E-01 | 332.86 | 2.2E-01 | 332.14 | 2.5E-01 | 331.38 | 1.7E-01 | 332.50 | 1.8E-01 |
| 2 | 2S---- | B2t16-b | 2.408E+05 | 331.10 | 1.2E-01 | 331.05 | 1.1E-01 | 332.50 | 9.9E-02 | 331.77 | 1.1E-01 | 331.08 | 9.0E-02 | 332.14 | 7.5E-02 |
| 2 | 2S---- | B2t16-c | 2.417E+05 | 331.73 | 1.3E-01 | 331.63 | 1.3E-01 | 333.08 | 1.2E-01 | 332.38 | 1.3E-01 | 331.68 | 9.6E-02 | 332.73 | 8.6E-02 |
| 2 | 2S---- | B2t17-a | 2.428E+05 | 333.91 | 1.4E-01 | 332.80 | 1.4E-01 | 335.55 | 1.5E-01 | 334.77 | 1.4E-01 | 333.35 | 9.6E-02 | 335.16 | 1.0E-01 |
| 2 | 2S---- | B2t17-b | 2.437E+05 | 332.77 | 1.2E-01 | 331.75 | 1.4E-01 | 334.45 | 1.2E-01 | 333.69 | 1.2E-01 | 332.26 | 9.6E-02 | 334.07 | 8.4E-02 |
| 2 | 2S---- | B2t17-c | 2.433E+05 | 333.91 | 1.4E-01 | 332.85 | 1.4E-01 | 335.59 | 1.2E-01 | 334.81 | 1.3E-01 | 333.38 | 1.0E-01 | 335.20 | 9.3E-02 |
| 2 | 2S---- | B2t18-a | 2.595E+05 | 330.28 | 1.2E-01 | 329.89 | 1.3E-01 | 333.56 | 1.4E-01 | 333.22 | 1.3E-01 | 330.08 | 8.7E-02 | 333.39 | 9.8E-02 |
| 2 | 2S---- | B2t18-b | 2.601E+05 | 331.46 | 1.1E-01 | 331.03 | 1.3E-01 | 334.70 | 1.4E-01 | 334.41 | 1.4E-01 | 331.24 | 9.3E-02 | 334.55 | 1.0E-01 |
| 2 | 2S---- | B2t18-c | 2.603E+05 | 328.78 | 1.2E-01 | 327.98 | 1.2E-01 | 330.86 | 9.8E-02 | 330.15 | 1.1E-01 | 328.38 | 8.9E-02 | 330.50 | 7.6E-02 |
| 2 | 2S---- | B2t18-d | 2.604E+05 | 329.54 | 1.4E-01 | 328.74 | 1.3E-01 | 331.62 | 1.4E-01 | 330.89 | 1.2E-01 | 329.14 | 9.3E-02 | 331.26 | 9.8E-02 |
| 2 | 2S---- | B2t19-a | 2.592E+05 | 330.49 | 1.2E-01 | 329.70 | 1.3E-01 | 333.24 | 1.2E-01 | 333.21 | 1.1E-01 | 330.10 | 9.4E-02 | 333.22 | 8.5E-02 |
| 2 | 2S---- | B2t19-b | 2.592E+05 | 330.98 | 1.2E-01 | 330.15 | 1.3E-01 | 333.72 | 1.3E-01 | 333.68 | 1.4E-01 | 330.56 | 8.5E-02 | 333.70 | 9.2E-02 |
| 2 | 2S---- | B2t19-c | 2.591E+05 | 327.65 | 1.1E-01 | 326.80 | 1.1E-01 | 329.78 | 1.5E-01 | 329.03 | 1.2E-01 | 327.23 | 8.3E-02 | 329.40 | 9.4E-02 |
| 2 | 2S---- | B2t19-d | 2.597E+05 | 328.40 | 1.1E-01 | 327.52 | 1.3E-01 | 330.56 | 1.1E-01 | 329.79 | 1.3E-01 | 327.96 | 8.4E-02 | 330.18 | 8.9E-02 |

| Build | Exp Code | Run | Nu,C2,d,av +/- | | Nu,C3,u,av +/- | | Nu,C3,d,av +/- | | Nu,C4,u,av +/- | | Nush,C2 +/- | | Nush,C3 +/- | | finNu,D2,av +/- | | finNu,D3,av +/- | |
|----------|------------|-----|----------------|---------|----------------|---------|----------------|---------|----------------|--------|-------------|-----|-------------|---------|-----------------|---------|-----------------|---------|
| 2 2S---- | B2run02-2 | | 17905.7 | 2.9E+04 | -5187.7 | 1.0E+03 | 16564.5 | 3.7E+05 | -5.1E+03 | 2515.3 | 2.5E+01 | 26 | -229.4 | 9.0E+02 | 5.5 | 5.0E+02 | -56.4 | 2.9E+02 |
| 2 2S---- | B2run02-3 | | 19628.1 | 2.0E+05 | -5243.2 | 2.3E+02 | 18085.7 | 1.0E+04 | -5.1E+03 | 198.5 | 3.1E+02 | 61 | -240.2 | 6.9E+02 | 23.9 | 1.1E+02 | 40.8 | 1.2E+02 |
| 2 2S---- | B2run03-1 | | 32249.4 | 2.3E+05 | -5847.9 | 3.1E+02 | 33245.0 | 5.5E+04 | -5.8E+03 | 297.5 | 2.0E+02 | 50 | -238.2 | 1.9E+02 | 20.9 | 1.9E+02 | -2.3 | 1.4E+02 |
| 2 2S---- | B2run04-1 | | 18126.2 | 7.3E+03 | -5177.5 | 1.9E+02 | 16465.6 | 9.7E+03 | -5.0E+03 | 186.1 | 2.9E+02 | 59 | -246.9 | 3.0E+02 | 20.7 | 9.1E+01 | 32.6 | 1.1E+02 |
| 2 2S---- | B2run04-2 | | 22513.1 | 1.5E+04 | -5502.5 | 1.8E+02 | 24137.8 | 1.2E+04 | -5.6E+03 | 190.2 | 2.1E+02 | 48 | -247.8 | 2.4E+03 | 17.2 | 8.3E+01 | 12.2 | 8.0E+01 |
| 2 2S---- | B2run04-3 | | 16886.4 | 1.8E+04 | -5061.1 | 2.9E+02 | 15932.0 | 5.3E+03 | -4.9E+03 | 226.5 | 3.3E+02 | 65 | -253.5 | 2.4E+02 | 23.1 | 1.1E+02 | 48.9 | 1.3E+02 |
| 2 2S---- | B2run04-4 | | 20525.8 | 6.3E+04 | -5360.1 | 2.5E+02 | 18298.9 | 1.7E+04 | -5.1E+03 | 234.8 | 3.0E+02 | 211 | -252.1 | 1.8E+02 | 18.8 | 1.4E+02 | 32.9 | 1.2E+02 |
| 2 2S---- | B2run04-7 | | 19274.1 | 1.9E+04 | -5262.1 | 1.9E+02 | 17274.3 | 7.0E+03 | -5.0E+03 | 183.0 | 2.9E+02 | 82 | -251.5 | 1.4E+02 | 19.6 | 9.0E+01 | 28.9 | 8.9E+01 |
| 2 2S---- | B2run06-1 | | 441.7 | 6.7E+01 | 45.4 | 6.4E+01 | 500.5 | 4.7E+01 | -1.1E+02 | 43.1 | 4.5E+01 | 1 | 28.6 | 7.6E-01 | 205.5 | 7.3E+00 | 175.0 | 6.5E+00 |
| 2 2S---- | B2run06-2 | | -90.1 | 5.1E+01 | 498.1 | 5.6E+01 | 439.5 | 4.9E+01 | -9.7E+01 | 45.1 | 3.4E+01 | 1 | 32.3 | 7.3E-01 | 170.9 | 6.1E+00 | 154.2 | 5.3E+00 |
| 2 2S---- | B2run08-1 | | 106.5 | 6.3E+01 | 113.5 | 6.4E+01 | -179.3 | 6.1E+01 | 3.8E+02 | 66.5 | 4.4E+01 | 2 | 44.5 | 1.2E+00 | 92.4 | 9.2E+00 | 89.5 | 8.8E+00 |
| 2 2S---- | B2run08-2 | | -188.8 | 5.9E+01 | 477.6 | 6.4E+01 | -135.8 | 5.4E+01 | 3.8E+02 | 58.9 | 7.3E+01 | 1 | 74.8 | 1.0E+00 | 118.6 | 6.8E+00 | 111.1 | 5.1E+00 |
| 2 2S---- | B2run08-3 | | -571.3 | 8.3E+01 | 931.8 | 1.0E+02 | -281.6 | 6.6E+01 | 5.2E+02 | 74.5 | 1.1E+02 | 2 | 110.0 | 2.3E+00 | 120.4 | 1.1E+01 | 102.3 | 9.0E+00 |
| 2 2S---- | B2run08-4 | | -562.6 | 7.2E+01 | 947.1 | 8.9E+01 | -319.1 | 8.2E+01 | 6.1E+02 | 90.7 | 1.1E+02 | 2 | 110.9 | 1.6E+00 | 131.8 | 9.2E+00 | 122.3 | 7.6E+00 |
| 2 2S---- | B2run08-5 | | -469.2 | 6.9E+01 | 844.7 | 8.5E+01 | -284.0 | 6.3E+01 | 5.9E+02 | 70.4 | 9.7E+01 | 1 | 95.9 | 1.3E+00 | 137.1 | 9.8E+00 | 129.7 | 7.6E+00 |
| 2 2S---- | B2run08-6 | | -211.4 | 7.1E+01 | 541.4 | 7.8E+01 | -99.5 | 5.7E+01 | 3.8E+02 | 59.9 | 7.3E+01 | 1 | 77.7 | 1.1E+00 | 134.8 | 9.1E+00 | 126.3 | 6.9E+00 |
| 2 2S---- | B2run08-7 | | -88.8 | 1.0E+02 | 364.8 | 1.0E+02 | -276.3 | 9.5E+01 | 5.3E+02 | 105.2 | 6.2E+01 | 2 | 68.1 | 1.7E+00 | 116.7 | 1.1E+01 | 108.8 | 1.2E+01 |
| 2 2S---- | B2run09-1 | | -361.4 | 9.8E+01 | 730.6 | 1.1E+02 | -89.4 | 8.9E+01 | 4.0E+02 | 95.3 | 6.6E+01 | 2 | 64.8 | 2.2E+00 | 143.6 | 1.3E+01 | 139.0 | 1.3E+01 |
| 2 2S---- | B2run09-2 | | -593.4 | 1.0E+02 | 1056.7 | 1.3E+02 | -401.5 | 9.2E+01 | 7.8E+02 | 108.8 | 1.1E+02 | 3 | 115.7 | 3.5E+00 | 161.9 | 1.6E+01 | 154.9 | 1.6E+01 |
| 2 2S---- | B2run09-3 | | 7.7 | 9.5E+01 | 351.0 | 1.1E+02 | -575.0 | 8.7E+01 | 9.5E+02 | 109.1 | 8.9E+01 | 1 | 93.1 | 1.3E+00 | 154.7 | 1.4E+01 | 138.4 | 1.2E+01 |
| 2 2S---- | B2run09-5a | | -207.3 | 9.4E+01 | 775.2 | 1.1E+02 | 398.3 | 9.5E+01 | 1.1E+02 | 93.8 | 4.3E+01 | 1 | 38.0 | 1.4E+00 | 237.2 | 1.3E+01 | 234.3 | 1.1E+01 |
| 2 2S---- | B2run09-5b | | -320.8 | 6.9E+01 | 668.7 | 8.1E+01 | -85.8 | 6.1E+01 | 3.8E+02 | 63.2 | 6.4E+01 | 1 | 63.4 | 9.2E-01 | 136.9 | 9.6E+00 | 133.7 | 8.9E+00 |
| 2 2S---- | B2run09-6 | | -440.3 | 8.1E+01 | 808.1 | 9.8E+01 | -287.4 | 7.1E+01 | 6.1E+02 | 77.7 | 5.4E+01 | 1 | 56.8 | 1.4E+00 | 139.0 | 1.2E+01 | 139.5 | 9.9E+00 |
| 2 2S---- | B2run12-1 | | -196.8 | 8.5E+01 | 517.0 | 9.1E+01 | -236.7 | 7.8E+01 | 5.3E+02 | 89.2 | 7.4E+01 | 2 | 76.9 | 1.3E+00 | 132.8 | 9.0E+00 | 128.4 | 9.9E+00 |
| 2 2S---- | B2run12-2 | | -510.1 | 7.0E+01 | 873.9 | 8.6E+01 | -370.6 | 6.6E+01 | 6.7E+02 | 75.2 | 1.1E+02 | 2 | 110.1 | 1.4E+00 | 130.0 | 1.1E+01 | 121.7 | 1.0E+01 |
| 2 2S---- | B2t05 | | -75.7 | 1.0E+02 | 347.6 | 1.0E+02 | -189.2 | 7.3E+01 | 4.3E+02 | 79.4 | 5.9E+01 | 2 | 62.7 | 2.1E+00 | 116.1 | 1.1E+01 | 108.1 | 9.4E+00 |

| Build | Exp Code | Run | Nu,C2,d,av +/- | | Nu,C3,u,av +/- | | Nu,C3,d,av +/- | | Nu,C4,u,av +/- | | Nush,C2 +/- | | Nush,C3 +/- | | finNu,D2,av +/- | | finNu,D3,av +/- | |
|----------|----------|-----|----------------|---------|----------------|---------|----------------|---------|----------------|-------|-------------|---|-------------|---------|-----------------|---------|-----------------|---------|
| 2 2S---- | B2t06 | | -171.5 | 8.5E+01 | 605.9 | 9.8E+01 | -30.0 | 8.9E+01 | 4.5E+02 | 95.7 | 4.6E+01 | 2 | 43.6 | 1.6E+00 | 187.6 | 1.2E+01 | 195.1 | 1.1E+01 |
| 2 2S---- | B2t07-a | | -25.5 | 8.8E+01 | 462.2 | 9.9E+01 | 82.0 | 7.8E+01 | 3.7E+02 | 81.0 | 4.7E+01 | 1 | 43.9 | 1.1E+00 | 192.5 | 1.0E+01 | 208.4 | 8.9E+00 |
| 2 2S---- | B2t07-b | | -280.0 | 8.2E+01 | 602.2 | 9.6E+01 | -118.2 | 6.9E+01 | 3.9E+02 | 75.1 | 6.7E+01 | 1 | 62.3 | 1.3E+00 | 129.5 | 9.4E+00 | 121.2 | 9.3E+00 |
| 2 2S---- | B2t09a | | 79.5 | 1.2E+02 | 381.7 | 1.2E+02 | 18.9 | 1.0E+02 | 4.6E+02 | 99.8 | 4.6E+01 | 2 | 40.2 | 2.6E+00 | 205.8 | 1.7E+01 | 218.8 | 1.9E+01 |
| 2 2S---- | B2t09b | | 122.3 | 9.4E+01 | 328.1 | 9.8E+01 | 58.3 | 8.0E+01 | 4.1E+02 | 80.9 | 4.3E+01 | 1 | 38.2 | 1.1E+00 | 201.9 | 1.3E+01 | 216.8 | 9.8E+00 |
| 2 2S---- | B2t09-c | | 119.7 | 9.2E+01 | 329.9 | 9.9E+01 | 73.9 | 7.7E+01 | 3.9E+02 | 83.8 | 4.3E+01 | 1 | 38.2 | 9.9E-01 | 201.4 | 1.1E+01 | 215.7 | 9.8E+00 |
| 2 2S---- | B2t10 | | -224.4 | 8.1E+01 | 468.8 | 8.6E+01 | -78.3 | 5.7E+01 | 2.8E+02 | 59.3 | 5.7E+01 | 2 | 53.2 | 1.5E+00 | 100.0 | 8.8E+00 | 95.3 | 8.7E+00 |
| 2 2S---- | B2t11 | | -275.2 | 1.2E+02 | 566.7 | 1.4E+02 | 129.5 | 1.0E+02 | 1.0E+02 | 101.3 | 4.4E+01 | 3 | 39.3 | 2.1E+00 | 117.2 | 1.4E+01 | 108.0 | 1.4E+01 |
| 2 2S---- | B2t12 | | -61.3 | 7.4E+01 | 303.5 | 7.9E+01 | -54.3 | 7.2E+01 | 2.6E+02 | 76.3 | 5.0E+01 | 1 | 48.9 | 1.2E+00 | 104.0 | 8.6E+00 | 95.2 | 8.3E+00 |
| 2 2s---- | B2t13-a | | 4.9 | 1.1E+03 | 484.1 | 7.9E+02 | -18.9 | 6.1E+02 | 5.2E+02 | 710.0 | 4.8E+01 | 4 | 43.7 | 3.8E+00 | 216.4 | 2.5E+02 | 230.9 | 6.6E+01 |
| 2 2S---- | B2t13-b | | 60.2 | 9.0E+01 | 384.3 | 9.4E+01 | 17.3 | 7.7E+01 | 4.5E+02 | 82.0 | 4.6E+01 | 1 | 41.7 | 1.1E+00 | 198.5 | 1.1E+01 | 216.4 | 1.2E+01 |
| 2 2S---- | B2t14 | | 131.1 | 9.0E+01 | 303.5 | 9.4E+01 | -78.9 | 7.6E+01 | 5.5E+02 | 86.3 | 4.6E+01 | 1 | 41.8 | 1.0E+00 | 194.5 | 1.1E+01 | 212.7 | 9.2E+00 |
| 2 2S---- | B2t15-a | | 168.7 | 7.4E+01 | 258.6 | 7.5E+01 | -107.6 | 7.0E+01 | 5.8E+02 | 75.9 | 4.6E+01 | 1 | 42.2 | 8.6E-01 | 191.9 | 1.2E+01 | 210.9 | 8.6E+00 |
| 2 2S---- | B2t15-b | | -308.1 | 9.8E+01 | 641.6 | 1.1E+02 | -167.2 | 8.1E+01 | 4.4E+02 | 86.8 | 6.3E+01 | 1 | 62.0 | 1.4E+00 | 132.2 | 1.2E+01 | 120.5 | 1.0E+01 |
| 2 2S---- | B2t16-a | | 105.5 | 1.8E+02 | 353.1 | 1.9E+02 | -99.9 | 1.5E+02 | 5.9E+02 | 180.8 | 4.8E+01 | 4 | 44.2 | 2.5E+00 | 205.0 | 2.8E+01 | 221.8 | 2.8E+01 |
| 2 2S---- | B2t16-b | | 168.1 | 7.2E+01 | 262.6 | 7.3E+01 | -81.4 | 6.7E+01 | 5.6E+02 | 73.1 | 4.6E+01 | 1 | 42.9 | 1.0E+00 | 193.2 | 9.5E+00 | 214.5 | 9.0E+00 |
| 2 2S---- | B2t16-c | | 147.9 | 8.7E+01 | 283.6 | 9.1E+01 | -76.1 | 8.5E+01 | 5.5E+02 | 97.2 | 4.6E+01 | 1 | 43.1 | 9.8E-01 | 193.4 | 1.3E+01 | 213.7 | 9.4E+00 |
| 2 2S---- | B2t17-a | | -386.0 | 9.9E+01 | 742.7 | 1.2E+02 | -189.5 | 8.4E+01 | 4.7E+02 | 92.5 | 6.4E+01 | 2 | 62.5 | 1.9E+00 | 136.9 | 1.3E+01 | 124.2 | 1.1E+01 |
| 2 2S---- | B2t17-b | | -340.0 | 8.2E+01 | 682.3 | 9.8E+01 | -179.0 | 7.1E+01 | 4.6E+02 | 77.5 | 6.5E+01 | 1 | 63.5 | 1.1E+00 | 134.1 | 1.0E+01 | 121.6 | 9.8E+00 |
| 2 2S---- | B2t17-c | | -356.1 | 9.1E+01 | 704.1 | 1.1E+02 | -185.9 | 6.8E+01 | 4.7E+02 | 75.9 | 6.4E+01 | 1 | 62.6 | 1.3E+00 | 135.2 | 1.2E+01 | 123.2 | 9.3E+00 |
| 2 2S---- | B2t18-a | | -42.6 | 6.8E+01 | 281.9 | 7.2E+01 | -8.1 | 6.3E+01 | 2.0E+02 | 64.4 | 4.6E+01 | 2 | 43.7 | 1.5E+00 | 102.6 | 7.2E+00 | 92.7 | 8.0E+00 |
| 2 2S---- | B2t18-b | | -59.9 | 5.6E+01 | 295.3 | 6.0E+01 | 7.9 | 6.0E+01 | 1.8E+02 | 60.5 | 4.6E+01 | 1 | 43.2 | 1.1E+00 | 100.5 | 8.0E+00 | 90.2 | 6.3E+00 |
| 2 2S---- | B2t18-c | | -236.9 | 6.6E+01 | 539.5 | 7.5E+01 | -155.8 | 5.4E+01 | 4.1E+02 | 58.7 | 7.3E+01 | 2 | 76.4 | 1.3E+00 | 124.5 | 1.0E+01 | 115.3 | 7.9E+00 |
| 2 2S---- | B2t18-d | | -235.8 | 8.9E+01 | 536.7 | 9.6E+01 | -160.9 | 6.9E+01 | 4.2E+02 | 75.0 | 7.2E+01 | 1 | 75.4 | 1.5E+00 | 123.7 | 8.6E+00 | 114.5 | 1.0E+01 |
| 2 2S---- | B2t19-a | | -189.9 | 6.1E+01 | 445.5 | 6.7E+01 | 93.6 | 5.1E+01 | 1.1E+02 | 50.2 | 4.6E+01 | 1 | 42.6 | 1.3E+00 | 105.6 | 7.9E+00 | 94.2 | 6.6E+00 |
| 2 2S---- | B2t19-b | | -205.5 | 6.9E+01 | 461.5 | 7.7E+01 | 90.6 | 6.6E+01 | 1.1E+02 | 66.6 | 4.6E+01 | 1 | 42.3 | 1.0E+00 | 105.0 | 7.7E+00 | 94.0 | 7.4E+00 |
| 2 2S---- | B2t19-c | | -255.5 | 6.6E+01 | 563.8 | 7.5E+01 | -166.9 | 7.7E+01 | 4.3E+02 | 84.2 | 7.3E+01 | 2 | 75.9 | 1.4E+00 | 126.0 | 9.8E+00 | 117.0 | 8.8E+00 |
| 2 2S---- | B2t19-d | | -269.9 | 7.5E+01 | 581.2 | 8.6E+01 | -175.7 | 6.5E+01 | 4.4E+02 | 72.9 | 7.2E+01 | 1 | 75.6 | 1.4E+00 | 126.4 | 8.4E+00 | 116.5 | 8.1E+00 |

| Build | Exp Code | Run | Tsh,C2,out(K) +/- | | Tsh,C3,out(K) +/- | | k_air(Tin) | +/- | V (m/s) | +/- |
|-------|-----------|------------|-------------------|-----|-------------------|-------|------------|----------|---------|---------|
| | 2 2S----- | B2run02-2 | 329 | 0.3 | 3.2E+02 | 2.46 | 2.8E-02 | 1.98E-05 | 44.73 | 3.8E-02 |
| | 2 2S----- | B2run02-3 | 414 | 0.6 | 3.2E+02 | 10.35 | 2.8E-02 | 7.37E-05 | 42.00 | 1.3E-01 |
| | 2 2S----- | B2run03-1 | 405 | 0.7 | 3.3E+02 | 11.98 | 2.8E-02 | 3.11E-05 | 48.12 | 6.2E-02 |
| | 2 2S----- | B2run04-1 | 397 | 0.4 | 3.2E+02 | 9.09 | 2.8E-02 | 4.44E-05 | 36.19 | 6.8E-02 |
| | 2 2S----- | B2run04-2 | 398 | 0.4 | 3.2E+02 | 9.33 | 2.8E-02 | 4.25E-05 | 36.46 | 6.6E-02 |
| | 2 2S----- | B2run04-3 | 397 | 0.2 | 3.2E+02 | 9.08 | 2.8E-02 | 8.21E-05 | 23.65 | 8.2E-02 |
| | 2 2S----- | B2run04-4 | 397 | 0.4 | 3.2E+02 | 9.15 | 2.8E-02 | 7.00E-05 | 23.69 | 7.0E-02 |
| | 2 2S----- | B2run04-7 | 398 | 0.5 | 3.2E+02 | 9.20 | 2.8E-02 | 5.80E-05 | 23.97 | 5.9E-02 |
| | 2 2S----- | B2run06-1 | 404 | 0.2 | 4.1E+02 | 0.17 | 2.7E-02 | 1.78E-05 | 27.70 | 2.1E-02 |
| | 2 2S----- | B2run06-2 | 411 | 0.2 | 4.2E+02 | 0.17 | 2.7E-02 | 1.70E-05 | 27.85 | 2.0E-02 |
| | 2 2S----- | B2run08-1 | 399 | 0.3 | 4.1E+02 | 0.31 | 2.7E-02 | 1.94E-05 | 17.84 | 1.5E-02 |
| | 2 2S----- | B2run08-2 | 396 | 0.2 | 4.0E+02 | 0.16 | 2.7E-02 | 2.03E-05 | 17.75 | 1.5E-02 |
| | 2 2S----- | B2run08-3 | 402 | 0.3 | 4.0E+02 | 0.30 | 2.8E-02 | 1.66E-05 | 18.14 | 1.3E-02 |
| | 2 2S----- | B2run08-4 | 404 | 0.2 | 4.1E+02 | 0.21 | 2.8E-02 | 1.88E-05 | 27.88 | 2.2E-02 |
| | 2 2S----- | B2run08-5 | 401 | 0.2 | 4.0E+02 | 0.20 | 2.8E-02 | 1.75E-05 | 28.09 | 2.1E-02 |
| | 2 2S----- | B2run08-6 | 400 | 0.2 | 4.0E+02 | 0.17 | 2.8E-02 | 1.62E-05 | 27.86 | 1.9E-02 |
| | 2 2S----- | B2run08-7 | 375 | 0.2 | 3.8E+02 | 0.21 | 2.8E-02 | 1.54E-05 | 28.19 | 1.8E-02 |
| | 2 2S----- | B2run09-1 | 395 | 0.3 | 4.0E+02 | 0.38 | 2.7E-02 | 2.01E-05 | 46.21 | 4.0E-02 |
| | 2 2S----- | B2run09-2 | 399 | 0.3 | 4.0E+02 | 0.33 | 2.7E-02 | 2.27E-05 | 46.13 | 4.5E-02 |
| | 2 2S----- | B2run09-3 | 398 | 0.2 | 4.0E+02 | 0.16 | 2.7E-02 | 1.99E-05 | 46.17 | 3.9E-02 |
| | 2 2S----- | B2run09-5a | 396 | 0.2 | 4.0E+02 | 0.30 | 2.7E-02 | 2.07E-05 | 46.02 | 4.1E-02 |
| | 2 2S----- | B2run09-5b | 398 | 0.2 | 4.0E+02 | 0.17 | 2.7E-02 | 2.02E-05 | 46.02 | 4.0E-02 |
| | 2 2S----- | B2run09-6 | 375 | 0.2 | 3.8E+02 | 0.17 | 2.8E-02 | 1.94E-05 | 46.19 | 3.8E-02 |
| | 2 2S----- | B2run12-1 | 392 | 0.2 | 3.9E+02 | 0.20 | 2.7E-02 | 1.98E-05 | 27.62 | 2.4E-02 |
| | 2 2S----- | B2run12-2 | 397 | 0.2 | 4.0E+02 | 0.19 | 2.7E-02 | 1.85E-05 | 27.24 | 2.2E-02 |
| | 2 2S----- | B2t05 | 381 | 0.3 | 3.8E+02 | 0.27 | 2.7E-02 | 1.79E-05 | 29.62 | 2.3E-02 |

| Build | Exp Code | Run | Tsh,C2,out(K) +/- | | Tsh,C3,out(K) +/- | | k_air(Tin) | +/- | V (m/s) | +/- |
|-------|-----------|---------|-------------------|-----|-------------------|------|------------|----------|---------|---------|
| | 2 2S----- | B2t06 | 369 | 0.2 | 3.7E+02 | 0.19 | 2.7E-02 | 1.74E-05 | 43.30 | 3.2E-02 |
| | 2 2S----- | B2t07-a | 385 | 0.2 | 3.9E+02 | 0.18 | 2.8E-02 | 1.74E-05 | 44.38 | 3.3E-02 |
| | 2 2S----- | B2t07-b | 382 | 0.2 | 3.8E+02 | 0.17 | 2.8E-02 | 1.94E-05 | 44.50 | 3.7E-02 |
| | 2 2S----- | B2t09a | 375 | 0.3 | 3.8E+02 | 0.34 | 2.8E-02 | 1.80E-05 | 43.63 | 3.3E-02 |
| | 2 2S----- | B2t09b | 378 | 0.2 | 3.8E+02 | 0.16 | 2.8E-02 | 1.90E-05 | 43.39 | 3.5E-02 |
| | 2 2S----- | B2t09-c | 378 | 0.2 | 3.8E+02 | 0.16 | 2.7E-02 | 1.70E-05 | 43.41 | 3.1E-02 |
| | 2 2S----- | B2t10 | 374 | 0.2 | 3.8E+02 | 0.17 | 2.7E-02 | 1.40E-05 | 18.63 | 1.1E-02 |
| | 2 2S----- | B2t11 | 382 | 0.4 | 3.8E+02 | 0.32 | 2.7E-02 | 2.57E-05 | 27.60 | 3.0E-02 |
| | 2 2S----- | B2t12 | 381 | 0.2 | 3.8E+02 | 0.16 | 2.7E-02 | 1.99E-05 | 22.52 | 1.9E-02 |
| | 2 2s----- | B2t13-a | 381 | 0.5 | 3.8E+02 | 0.47 | 2.8E-02 | 8.51E-05 | 44.81 | 1.6E-01 |
| | 2 2S----- | B2t13-b | 383 | 0.2 | 3.9E+02 | 0.16 | 2.7E-02 | 2.00E-05 | 44.70 | 3.8E-02 |
| | 2 2S----- | B2t14 | 384 | 0.2 | 3.9E+02 | 0.16 | 2.8E-02 | 2.17E-05 | 44.58 | 4.1E-02 |
| | 2 2S----- | B2t15-a | 383 | 0.2 | 3.9E+02 | 0.16 | 2.7E-02 | 1.92E-05 | 44.75 | 3.7E-02 |
| | 2 2S----- | B2t15-b | 384 | 0.2 | 3.9E+02 | 0.16 | 2.8E-02 | 2.00E-05 | 44.86 | 3.8E-02 |
| | 2 2S----- | B2t16-a | 383 | 0.4 | 3.9E+02 | 0.37 | 2.8E-02 | 3.34E-05 | 44.62 | 6.3E-02 |
| | 2 2S----- | B2t16-b | 385 | 0.2 | 3.9E+02 | 0.16 | 2.7E-02 | 1.86E-05 | 44.36 | 3.5E-02 |
| | 2 2S----- | B2t16-c | 385 | 0.2 | 3.9E+02 | 0.16 | 2.7E-02 | 2.88E-05 | 44.45 | 5.5E-02 |
| | 2 2S----- | B2t17-a | 384 | 0.2 | 3.9E+02 | 0.26 | 2.8E-02 | 1.95E-05 | 44.60 | 3.7E-02 |
| | 2 2S----- | B2t17-b | 384 | 0.2 | 3.9E+02 | 0.17 | 2.8E-02 | 1.79E-05 | 44.59 | 3.4E-02 |
| | 2 2S----- | B2t17-c | 385 | 0.2 | 3.9E+02 | 0.17 | 2.8E-02 | 1.53E-05 | 44.83 | 2.9E-02 |
| | 2 2S----- | B2t18-a | 375 | 0.2 | 3.8E+02 | 0.20 | 2.7E-02 | 1.76E-05 | 20.06 | 1.5E-02 |
| | 2 2S----- | B2t18-b | 376 | 0.2 | 3.8E+02 | 0.17 | 2.7E-02 | 1.68E-05 | 20.15 | 1.5E-02 |
| | 2 2S----- | B2t18-c | 377 | 0.2 | 3.8E+02 | 0.17 | 2.7E-02 | 1.74E-05 | 20.03 | 1.5E-02 |
| | 2 2S----- | B2t18-d | 378 | 0.2 | 3.8E+02 | 0.17 | 2.7E-02 | 1.86E-05 | 20.23 | 1.6E-02 |
| | 2 2S----- | B2t19-a | 377 | 0.2 | 3.8E+02 | 0.19 | 2.7E-02 | 1.80E-05 | 21.17 | 1.7E-02 |
| | 2 2S----- | B2t19-b | 377 | 0.2 | 3.8E+02 | 0.17 | 2.7E-02 | 1.91E-05 | 21.37 | 1.8E-02 |
| | 2 2S----- | B2t19-c | 377 | 0.2 | 3.8E+02 | 0.17 | 2.7E-02 | 1.43E-05 | 20.95 | 1.3E-02 |
| | 2 2S----- | B2t19-d | 378 | 0.2 | 3.8E+02 | 0.16 | 2.7E-02 | 1.86E-05 | 21.13 | 1.7E-02 |

Appendix 6.3: Test conditions and main non-dimensional results for Build 3

| Build | Exp Code | Run | ExpDate | LDA code | FCF pos | Lab atm press (Pa) | Comments | Refi | +/- | Rez | +/- | BDTav | +/- | finBDTav | +/- | Ro | +/- |
|-------|----------|-----------|------------|----------|---------|--------------------|------------------------------------|----------|---------|----------|---------|-------|---------|----------|---------|------|---------|
| 3 | 3S---- | B3run2-1 | 10/10/2002 | 0 | 0 | 1.009E+05 | run OK | 9.95E+05 | 1.7E+03 | 1.65E+05 | 1.3E+02 | 0.056 | 1.0E-03 | 0.056 | 1.0E-03 | 3.17 | 3.1E-03 |
| 3 | 3S---- | B3run2-2 | 10/10/2002 | 0 | 0 | 1.009E+05 | run OK | 9.92E+05 | 1.3E+03 | 1.65E+05 | 9.4E+01 | 0.056 | 7.8E-04 | 0.056 | 7.8E-04 | 3.17 | 2.4E-03 |
| 3 | 3S---- | B3run3-1 | 10/21/2002 | 101 | 0 | 9.962E+04 | run OK | 1.01E+06 | 1.6E+03 | 1.56E+05 | 1.1E+02 | 0.054 | 9.2E-04 | 0.054 | 9.2E-04 | 2.94 | 2.7E-03 |
| 3 | 3S---- | B3run4-1 | 10/23/2002 | 101 | 0 | 1.006E+05 | run OK | 1.00E+06 | 2.0E+03 | 6.27E+04 | 5.5E+01 | 0.062 | 1.1E-03 | 0.062 | 1.1E-03 | 1.19 | 1.4E-03 |
| 3 | 3S---- | B3run4-2 | 10/23/2002 | 0 | 0 | 1.006E+05 | run OK | 1.00E+06 | 2.4E+03 | 6.30E+04 | 6.5E+01 | 0.063 | 1.4E-03 | 0.063 | 1.4E-03 | 1.20 | 1.6E-03 |
| 3 | 3S---- | B3run5-1 | 10/29/2002 | 101 | 0 | 1.012E+05 | run OK | 9.90E+05 | 1.8E+03 | 1.07E+05 | 8.2E+01 | 0.054 | 1.1E-03 | 0.054 | 1.1E-03 | 2.05 | 2.1E-03 |
| 3 | 3S---- | B3run5-3 | 10/29/2002 | 0 | 0 | 1.012E+05 | run OK | 1.02E+06 | 1.3E+03 | 1.08E+05 | 5.8E+01 | 0.059 | 6.9E-04 | 0.059 | 6.9E-04 | 2.03 | 1.4E-03 |
| 3 | 3S---- | B3run6-1 | 11/14/2002 | 101 | 0 | 9.806E+04 | run OK | 9.63E+05 | 1.5E+03 | 9.85E+04 | 6.3E+01 | 0.058 | 8.8E-04 | 0.058 | 8.8E-04 | 1.95 | 1.7E-03 |
| 3 | 3S---- | B3run6-2 | 11/14/2002 | 0 | 0 | 9.806E+04 | run OK | 9.76E+05 | 5.1E+03 | 9.79E+04 | 2.2E+02 | 0.054 | 3.1E-03 | 0.054 | 3.1E-03 | 1.91 | 5.6E-03 |
| 3 | 3S---- | B3run6-3 | 11/14/2002 | 0 | 0 | 9.806E+04 | run OK | 9.69E+05 | 1.4E+03 | 9.77E+04 | 6.2E+01 | 0.055 | 8.8E-04 | 0.055 | 8.8E-04 | 1.92 | 1.6E-03 |
| 3 | 3S---- | B3run7-1 | 11/28/2002 | 101 | 0 | 1.007E+05 | run OK | 1.03E+06 | 1.7E+03 | 5.14E+04 | 3.7E+01 | 0.068 | 1.2E-03 | 0.068 | 1.2E-03 | 0.95 | 9.2E-04 |
| 3 | 3S---- | B3run7-2 | 11/28/2002 | 0 | 0 | 1.007E+05 | run OK | 9.70E+05 | 2.2E+03 | 4.87E+04 | 4.7E+01 | 0.063 | 1.3E-03 | 0.063 | 1.3E-03 | 0.96 | 1.2E-03 |
| 3 | 3S---- | B3run7-4 | 11/28/2002 | 0 | 0 | 1.007E+05 | run OK | 5.34E+05 | 1.5E+03 | 4.83E+04 | 5.9E+01 | 0.074 | 1.7E-03 | 0.074 | 1.7E-03 | 1.73 | 2.8E-03 |
| 3 | 3S---- | B3run8-1 | 12/03/2002 | 101 | 0 | 1.011E+05 | run OK | 4.74E+05 | 8.3E+02 | 4.72E+04 | 3.4E+01 | 0.087 | 1.0E-03 | 0.087 | 1.0E-03 | 1.90 | 2.1E-03 |
| 3 | 3S---- | B3run9-1 | 12/05/2002 | 101 | 0 | 1.025E+05 | run OK | 4.96E+05 | 7.1E+02 | 1.51E+05 | 9.3E+01 | 0.066 | 8.3E-04 | 0.066 | 8.3E-04 | 5.81 | 4.7E-03 |
| 3 | 3S---- | B3run9-2 | 12/05/2002 | 0 | 0 | 1.025E+05 | run OK | 1.51E+06 | 2.2E+03 | 4.83E+04 | 3.0E+01 | 0.061 | 8.4E-04 | 0.061 | 8.4E-04 | 0.61 | 5.0E-04 |
| 3 | 3S---- | B3run9-3 | 12/05/2002 | 0 | 0 | 1.025E+05 | run OK | 1.51E+06 | 2.1E+03 | 4.83E+04 | 2.9E+01 | 0.060 | 7.9E-04 | 0.060 | 7.9E-04 | 0.61 | 4.8E-04 |
| 3 | 3S---- | B3run9-4 | 12/05/2002 | 0 | 0 | 1.025E+05 | run OK | 1.54E+06 | 2.4E+03 | 1.03E+05 | 7.0E+01 | 0.049 | 9.0E-04 | 0.049 | 9.0E-04 | 1.28 | 1.2E-03 |
| 3 | 3S---- | B3run12-1 | 3/18/2003 | 0 | 0 | 1.033E+05 | run OK | 9.15E+05 | 1.1E+03 | 6.93E+04 | 3.6E+01 | 0.064 | 7.3E-04 | 0.064 | 7.3E-04 | 1.44 | 1.0E-03 |
| 3 | 3S---- | B3run13-1 | 04/10/2003 | 0 | 0 | 1.010E+05 | not fully steady state measurement | 1.00E+06 | 1.8E+03 | 1.95E+05 | 1.5E+02 | 0.053 | 1.0E-03 | 0.053 | 1.0E-03 | 3.71 | 4.0E-03 |
| 3 | 3S---- | B3run13-2 | 04/10/2003 | 0 | 0 | 1.010E+05 | OK | 1.00E+06 | 2.5E+03 | 1.96E+05 | 2.1E+02 | 0.054 | 1.4E-03 | 0.054 | 1.4E-03 | 3.72 | 5.2E-03 |
| 3 | 3S---- | B3run13-3 | 04/10/2003 | 0 | 0 | 1.010E+05 | measurement OK | 3.93E+06 | 5.3E+03 | 7.22E+04 | 4.2E+01 | 0.044 | 7.7E-04 | 0.044 | 7.7E-04 | 0.35 | 2.7E-04 |

| Build | Exp Code | Run | ExpDate | LDA code | FCF pos | Lab atm press (Pa) | Comments | Refi | +/- | Rez | +/- | BDTav | +/- | finBDTav | +/- | Ro | +/- |
|-------|----------|-----------|------------|----------|---------|--------------------|---|----------|---------|----------|---------|-------|---------|----------|---------|------|---------|
| 3 | 3S---- | B3run15-1 | 4/15/2003 | 0 | 0 | 1.019E+05 | run OK | 1.00E+06 | 1.5E+03 | 8.44E+04 | 5.4E+01 | 0.056 | 8.5E-04 | 0.056 | 8.5E-04 | 1.60 | 1.5E-03 |
| 3 | 3S---- | B3run15-2 | 4/15/2003 | 0 | 0 | 1.019E+05 | run OK | 2.01E+06 | 2.7E+03 | 4.10E+04 | 2.3E+01 | 0.054 | 7.6E-04 | 0.054 | 7.6E-04 | 0.39 | 3.0E-04 |
| 3 | 3S---- | B3run16-1 | 4/16/2003 | 0 | 0 | 1.024E+05 | run OK | 1.96E+06 | 2.7E+03 | 8.18E+04 | 4.8E+01 | 0.038 | 7.3E-04 | 0.038 | 7.3E-04 | 0.80 | 6.3E-04 |
| 3 | 3S---- | B3run16-2 | 4/16/2003 | 0 | 0 | 1.024E+05 | run OK | 3.95E+06 | 5.3E+03 | 7.98E+04 | 4.6E+01 | 0.037 | 7.5E-04 | 0.037 | 7.5E-04 | 0.39 | 3.0E-04 |
| 3 | 3S---- | B3run17-1 | 4/24/2003 | 0 | 0 | 1.016E+05 | run OK | 2.00E+06 | 2.5E+03 | 1.64E+05 | 8.8E+01 | 0.035 | 6.8E-04 | 0.035 | 6.8E-04 | 1.57 | 1.1E-03 |
| 3 | 3S---- | B3run17-2 | 4/24/2003 | 0 | 0 | 1.016E+05 | run OK | 2.00E+06 | 2.8E+03 | 1.63E+05 | 1.0E+02 | 0.035 | 8.0E-04 | 0.035 | 8.0E-04 | 1.56 | 1.3E-03 |
| 3 | 3S---- | B3run17-3 | 4/24/2003 | 0 | 0 | 1.016E+05 | run OK | 3.98E+06 | 5.5E+03 | 1.63E+05 | 9.7E+01 | 0.027 | 7.3E-04 | 0.027 | 7.3E-04 | 0.78 | 6.1E-04 |
| 3 | 3S---- | B3run17-4 | 4/24/2003 | 0 | 0 | 1.016E+05 | run OK | 3.98E+06 | 5.4E+03 | 1.63E+05 | 9.5E+01 | 0.027 | 7.4E-04 | 0.027 | 7.4E-04 | 0.78 | 6.0E-04 |
| 3 | 3S---- | B3run17-5 | 4/24/2003 | 0 | 0 | 1.016E+05 | run OK | 1.02E+06 | 1.2E+03 | 4.21E+04 | 2.2E+01 | 0.069 | 7.0E-04 | 0.069 | 7.0E-04 | 0.79 | 5.5E-04 |
| 3 | 3S---- | B3run17-6 | 4/24/2003 | 0 | 0 | 1.016E+05 | run OK | 1.02E+06 | 1.5E+03 | 4.13E+04 | 2.5E+01 | 0.070 | 8.3E-04 | 0.070 | 8.3E-04 | 0.77 | 6.3E-04 |
| 3 | 3S---- | B3run18-1 | 6/21/2003 | 101 | 0 | 1.021E+05 | run OK | 4.96E+05 | 6.8E+02 | 5.13E+04 | 3.1E+01 | 0.081 | 8.4E-04 | 0.081 | 8.4E-04 | 1.97 | 1.5E-03 |
| 3 | 3S---- | B3run18-2 | 6/21/2003 | 101 | 0 | 1.021E+05 | run OK | 1.50E+06 | 2.0E+03 | 4.93E+04 | 2.8E+01 | 0.052 | 7.9E-04 | 0.052 | 7.9E-04 | 0.63 | 4.8E-04 |
| 3 | 3S---- | B3run18-3 | 6/21/2003 | 101 | 0 | 1.021E+05 | run OK | 1.03E+06 | 1.3E+03 | 1.01E+05 | 5.3E+01 | 0.050 | 6.8E-04 | 0.050 | 6.8E-04 | 1.87 | 1.3E-03 |
| 3 | 3S---- | B3run19-1 | 6/24/2003 | 101 | 0 | 1.021E+05 | run OK | 1.02E+06 | 1.9E+03 | 1.03E+05 | 8.0E+01 | 0.051 | 1.2E-03 | 0.051 | 1.2E-03 | 1.92 | 2.1E-03 |
| 3 | 3S---- | B3run20-1 | 6/30/2003 | 101 | 0 | 9.993E+04 | run OK | 1.16E+06 | 3.8E+03 | 9.06E+04 | 1.3E+02 | 0.050 | 1.8E-03 | 0.050 | 1.8E-03 | 1.49 | 2.8E-03 |
| 3 | 3S---- | B3run21-1 | 07/01/2003 | 101 | 0 | 9.993E+04 | run OK | 1.20E+06 | 1.6E+03 | 9.29E+04 | 5.4E+01 | 0.046 | 7.7E-04 | 0.046 | 7.7E-04 | 1.48 | 1.1E-03 |
| 3 | 3S---- | B3run22-1 | 07/07/2003 | 101 | 0 | 1.024E+05 | run OK | 1.24E+06 | 1.7E+03 | 4.91E+04 | 3.0E+01 | 0.061 | 8.1E-04 | 0.061 | 8.1E-04 | 0.75 | 6.1E-04 |
| 3 | 3S---- | B3run22-2 | 07/07/2003 | 101 | 0 | 1.024E+05 | run OK | 2.41E+06 | 3.4E+03 | 8.60E+04 | 5.2E+01 | 0.033 | 8.5E-04 | 0.033 | 8.5E-04 | 0.68 | 5.4E-04 |
| 3 | 3S---- | B3run23-1 | 07/09/2003 | 101 | 0 | 1.022E+05 | run OK | 2.02E+06 | 2.4E+03 | 9.91E+04 | 5.1E+01 | 0.035 | 6.8E-04 | 0.035 | 6.8E-04 | 0.93 | 6.2E-04 |
| 3 | 3S---- | B3run24-1 | 7/29/2003 | 101 | 0 | 1.022E+05 | run OK | 1.31E+06 | 1.9E+03 | 9.74E+04 | 5.9E+01 | 0.050 | 7.9E-04 | 0.050 | 7.9E-04 | 1.41 | 1.2E-03 |
| 3 | 3S---- | B3run24-2 | 7/29/2003 | 0 | 0 | 1.022E+05 | run OK | 9.94E+05 | 1.7E+03 | 8.79E+04 | 6.3E+01 | 0.047 | 9.9E-04 | 0.047 | 9.9E-04 | 1.69 | 1.6E-03 |
| 3 | 3S---- | B3run25-1 | 8/27/2003 | 101 | 0 | 1.015E+05 | run OK | 1.15E+06 | 1.4E+03 | 1.71E+05 | 8.7E+01 | 0.045 | 6.6E-04 | 0.045 | 6.6E-04 | 2.84 | 1.9E-03 |
| 3 | 3S---- | B3run26-1 | 10/17/2003 | 101 | 0 | 1.014E+05 | run OK LDA file is identified as run27-2 | 2.46E+06 | 2.9E+03 | 4.71E+04 | 2.3E+01 | 0.043 | 6.5E-04 | 0.043 | 6.5E-04 | 0.37 | 2.5E-04 |
| 3 | 3S---- | B3run27-1 | 10/31/2003 | 101 | 0 | 9.778E+04 | | 2.00E+06 | 2.7E+03 | 1.75E+05 | 1.0E+02 | 0.029 | 8.0E-04 | 0.029 | 8.0E-04 | 1.66 | 1.3E-03 |
| 3 | 3S---- | B3run28-1 | 11/12/2003 | 101 | 0 | 1.017E+05 | run OK | 4.12E+06 | 5.7E+03 | 1.16E+05 | 6.9E+01 | 0.027 | 7.7E-04 | 0.027 | 7.7E-04 | 0.54 | 4.4E-04 |
| 3 | 3S---- | B3run28-2 | 11/12/2003 | 101 | 0 | 1.017E+05 | run OK | 4.02E+06 | 5.7E+03 | 5.77E+04 | 3.5E+01 | 0.040 | 7.8E-04 | 0.040 | 7.8E-04 | 0.27 | 2.2E-04 |
| 3 | 3S---- | B3run30-1 | 12/10/2003 | 101 | 0 | 1.016E+05 | run OK | 4.07E+06 | 5.3E+03 | 1.84E+05 | 1.0E+02 | 0.021 | 7.2E-04 | 0.021 | 7.2E-04 | 0.86 | 6.3E-04 |

| Build | Exp Code | Run | ExpDate | LDA code | FCF pos | Lab atm press (Pa) | Comments | Refi | +/- | Rez | +/- | BDTav | +/- | finBDTav | +/- | Ro | +/- |
|-------|----------|------------|------------|----------|---------|--------------------|----------|----------|---------|----------|---------|-------|---------|----------|---------|------|---------|
| 3 | 3S---- | B3run31-1 | 12/12/2003 | 101 | 0 | 1.018E+05 | run OK | 5.17E+05 | 7.7E+02 | 1.85E+05 | 1.2E+02 | 0.050 | 8.2E-04 | 0.050 | 8.2E-04 | 6.82 | 5.9E-03 |
| 3 | 3S---- | B3run32-1 | 1/16/2004 | 101 | 0 | 9.956E+04 | run OK | 4.92E+05 | 6.9E+02 | 9.18E+04 | 5.5E+01 | 0.089 | 8.3E-04 | 0.089 | 8.3E-04 | 3.56 | 2.9E-03 |
| 3 | 3S---- | B3run33-1 | 04/07/2004 | 0 | 2 | 1.014E+05 | FCF | 9.18E+05 | 1.5E+03 | 8.85E+04 | 6.4E+01 | 0.067 | 9.3E-04 | 0.067 | 9.3E-04 | 1.84 | 1.8E-03 |
| 3 | 3S---- | B3run33-2 | 04/07/2004 | 0 | 2 | 1.014E+05 | FCF | 2.32E+06 | 3.0E+03 | 8.79E+04 | 4.9E+01 | 0.040 | 7.4E-04 | 0.040 | 7.4E-04 | 0.72 | 5.3E-04 |
| 3 | 3S---- | B3run33-3 | 04/07/2004 | 0 | 2 | 1.014E+05 | FCF | 1.18E+06 | 1.5E+03 | 4.38E+04 | 2.4E+01 | 0.067 | 7.4E-04 | 0.067 | 7.4E-04 | 0.71 | 5.1E-04 |
| 3 | 3S---- | B3run33-4 | 04/07/2004 | 0 | 2 | 1.014E+05 | FCF | 1.07E+06 | 1.5E+03 | 1.72E+05 | 1.0E+02 | 0.046 | 8.1E-04 | 0.046 | 8.1E-04 | 3.06 | 2.4E-03 |
| 3 | 3S---- | B3run33-5 | 04/07/2004 | 0 | 2 | 1.014E+05 | FCF | 4.81E+06 | 7.1E+03 | 1.73E+05 | 1.1E+02 | 0.026 | 8.5E-04 | 0.026 | 8.5E-04 | 0.69 | 5.8E-04 |
| 3 | 3S---- | B3run33-6 | 04/07/2004 | 0 | 2 | 1.014E+05 | FCF | 4.74E+06 | 6.0E+03 | 3.43E+04 | 1.9E+01 | 0.053 | 7.4E-04 | 0.053 | 7.4E-04 | 0.14 | 1.0E-04 |
| 3 | 3S---- | B3run33-7 | 04/07/2004 | 0 | 2 | 1.014E+05 | FCF | 1.16E+06 | 1.4E+03 | 8.71E+04 | 4.4E+01 | 0.060 | 6.9E-04 | 0.060 | 6.9E-04 | 1.43 | 9.6E-04 |
| 3 | 3S---- | B3run33-8 | 04/07/2004 | 0 | 2 | 1.014E+05 | FCF | 2.34E+06 | 3.0E+03 | 1.73E+05 | 9.6E+01 | 0.040 | 7.3E-04 | 0.040 | 7.3E-04 | 1.41 | 1.0E-03 |
| 3 | 3S---- | B3run33-9 | 04/07/2004 | 0 | 2 | 1.014E+05 | FCF | 2.31E+06 | 2.7E+03 | 4.49E+04 | 2.2E+01 | 0.052 | 6.7E-04 | 0.052 | 6.7E-04 | 0.37 | 2.4E-04 |
| 3 | 3S---- | B3run33-10 | 04/07/2004 | 0 | 2 | 1.014E+05 | FCF | 5.02E+06 | 5.7E+03 | 8.50E+04 | 4.1E+01 | 0.036 | 6.1E-04 | 0.036 | 6.1E-04 | 0.32 | 2.1E-04 |
| 3 | 3S---- | B3run34-1 | 4/13/2004 | 0 | 1 | 1.026E+05 | FCF | 1.17E+06 | 1.7E+03 | 1.13E+05 | 6.8E+01 | 0.050 | 8.0E-04 | 0.050 | 8.0E-04 | 1.83 | 1.5E-03 |
| 3 | 3S---- | B3run34-2 | 4/13/2004 | 0 | 1 | 1.026E+05 | FCF | 4.76E+06 | 5.7E+03 | 1.09E+05 | 5.7E+01 | 0.028 | 7.1E-04 | 0.028 | 7.1E-04 | 0.44 | 3.0E-04 |
| 3 | 3S---- | B3run34-3 | 4/13/2004 | 0 | 1 | 1.026E+05 | FCF | 2.36E+06 | 3.3E+03 | 6.52E+04 | 3.9E+01 | 0.039 | 8.1E-04 | 0.039 | 8.1E-04 | 0.53 | 4.2E-04 |
| 3 | 3S---- | B3run34-4 | 4/13/2004 | 0 | 1 | 1.026E+05 | FCF | 1.19E+06 | 1.5E+03 | 6.59E+04 | 3.4E+01 | 0.052 | 6.9E-04 | 0.052 | 6.9E-04 | 1.05 | 7.6E-04 |
| 3 | 3S---- | B3run34-5 | 4/13/2004 | 0 | 1 | 1.026E+05 | FCF | 2.37E+06 | 3.0E+03 | 1.09E+05 | 6.1E+01 | 0.034 | 7.0E-04 | 0.034 | 7.0E-04 | 0.88 | 6.4E-04 |
| 3 | 3S---- | B3run34-6 | 4/13/2004 | 0 | 1 | 1.026E+05 | FCF | 4.77E+06 | 5.0E+03 | 1.92E+05 | 8.6E+01 | 0.020 | 5.8E-04 | 0.020 | 5.8E-04 | 0.77 | 4.5E-04 |
| 3 | 3S---- | B3run34-7 | 4/13/2004 | 0 | 1 | 1.026E+05 | FCF | 4.76E+06 | 5.8E+03 | 6.17E+04 | 3.2E+01 | 0.034 | 7.0E-04 | 0.034 | 7.0E-04 | 0.25 | 1.7E-04 |
| 3 | 3S---- | B3run34-8 | 4/13/2004 | 0 | 1 | 1.026E+05 | FCF | 2.36E+06 | 2.9E+03 | 1.92E+05 | 1.0E+02 | 0.029 | 7.0E-04 | 0.029 | 7.0E-04 | 1.55 | 1.1E-03 |
| 3 | 3S---- | B3run34-9 | 4/13/2004 | 0 | 1 | 1.026E+05 | FCF | 1.18E+06 | 1.7E+03 | 1.90E+05 | 1.1E+02 | 0.037 | 7.8E-04 | 0.037 | 7.8E-04 | 3.08 | 2.5E-03 |
| 3 | 3S---- | B3run35-1 | 4/15/2004 | 0 | 3 | 1.017E+05 | FCF | 1.17E+06 | 1.4E+03 | 8.91E+04 | 4.7E+01 | 0.053 | 7.1E-04 | 0.053 | 7.1E-04 | 1.45 | 1.0E-03 |
| 3 | 3S---- | B3run35-2 | 4/15/2004 | 0 | 3 | 1.017E+05 | FCF | 2.36E+06 | 3.2E+03 | 1.72E+05 | 1.0E+02 | 0.027 | 7.6E-04 | 0.027 | 7.6E-04 | 1.39 | 1.1E-03 |
| 3 | 3S---- | B3run35-3 | 4/15/2004 | 0 | 3 | 1.017E+05 | FCF | 2.36E+06 | 2.8E+03 | 8.55E+04 | 4.4E+01 | 0.031 | 6.9E-04 | 0.031 | 6.9E-04 | 0.69 | 4.7E-04 |
| 3 | 3S---- | B3run35-4 | 4/15/2004 | 0 | 3 | 1.017E+05 | FCF | 4.79E+06 | 7.9E+03 | 1.70E+05 | 1.2E+02 | 0.017 | 8.9E-04 | 0.017 | 8.9E-04 | 0.68 | 6.9E-04 |
| 3 | 3S---- | B3run35-5 | 4/15/2004 | 0 | 3 | 1.017E+05 | FCF | 2.34E+06 | 2.8E+03 | 4.08E+04 | 2.1E+01 | 0.045 | 6.7E-04 | 0.045 | 6.7E-04 | 0.33 | 2.3E-04 |
| 3 | 3S---- | B3run35-6 | 4/15/2004 | 0 | 3 | 1.017E+05 | FCF | 4.79E+06 | 6.2E+03 | 8.43E+04 | 4.4E+01 | 0.024 | 6.9E-04 | 0.024 | 6.9E-04 | 0.34 | 2.7E-04 |

| Build | Exp Code | Run | ExpDate | LDA code | FCF pos | Lab atm press (Pa) | Comments | Refi | +/- | Rez | +/- | BDTav | +/- | finBDTav | +/- | Ro | +/- |
|-------|----------|-----------|------------|----------|---------|--------------------|------------------------|----------|---------|----------|---------|--------|---------|----------|---------|---------|---------|
| 3 | 3S---- | B3run35-7 | 4/15/2004 | 0 | 3 | 1.017E+05 | FCF | 1.15E+06 | 1.3E+03 | 4.62E+04 | 2.3E+01 | 0.058 | 6.6E-04 | 0.058 | 6.6E-04 | 0.77 | 5.0E-04 |
| 3 | 3S---- | B3run35-8 | 4/15/2004 | 0 | 3 | 1.017E+05 | FCF | 1.15E+06 | 1.6E+03 | 1.71E+05 | 1.0E+02 | 0.041 | 7.9E-04 | 0.041 | 7.9E-04 | 2.84 | 2.3E-03 |
| 3 | 3S---- | B3run35-9 | 4/15/2004 | 0 | 3 | 1.017E+05 | FCF | 4.75E+06 | 6.1E+03 | 3.95E+04 | 2.2E+01 | 0.029 | 7.1E-04 | 0.029 | 7.1E-04 | 0.16 | 1.2E-04 |
| 3 | 3S---- | B3run36-1 | 9/20/2004 | 110 | 0 | 1.012E+05 | run OK | 1.15E+06 | 1.6E+03 | 1.10E+05 | 6.7E+01 | 0.057 | 8.1E-04 | 0.057 | 8.1E-04 | 1.82 | 1.5E-03 |
| 3 | 3S---- | B3run37-1 | 9/24/2004 | 0 | 0 | 1.021E+05 | LDA Vr Vf unsuccessful | 1.88E+06 | 8.6E+03 | 5.47E+04 | 1.1E+02 | 0.001 | 3.0E-03 | 0.001 | 3.0E-03 | 0.56 | 1.5E-03 |
| 3 | 3S---- | B3run38-1 | 9/27/2004 | 110 | 0 | 1.024E+05 | Run OK | 3.99E+05 | 5.8E+02 | 1.57E+05 | 9.7E+01 | 0.067 | 8.3E-04 | 0.067 | 8.3E-04 | 7.51 | 6.2E-03 |
| 3 | 3S---- | B3run39-1 | 9/28/2004 | 110 | 0 | 1.021E+05 | run OK | 1.93E+06 | 2.5E+03 | 5.31E+04 | 2.9E+01 | -0.001 | 7.0E-04 | -0.001 | 7.0E-04 | 0.53 | 4.0E-04 |
| 3 | 3S---- | B3run40-1 | 10/05/2004 | 110 | 0 | 1.011E+05 | run OK | 3.88E+05 | 4.9E+02 | 6.07E+04 | 3.3E+01 | 0.107 | 7.9E-04 | 0.107 | 7.9E-04 | 2.98 | 2.2E-03 |
| 3 | 3S---- | B3run44-1 | 12/20/2004 | 0 | 0 | 1.022E+05 | run OK | 1.99E+06 | 1.1E+04 | 1.65E+05 | 3.8E+02 | 0.004 | 3.4E-03 | 0.004 | 3.4E-03 | 1.58 | 4.8E-03 |
| 3 | 3S---- | B3run44-2 | 12/20/2004 | 0 | 0 | 1.022E+05 | run OK | 1.97E+06 | 1.2E+04 | 1.64E+05 | 4.2E+02 | 0.012 | 3.6E-03 | 0.012 | 3.6E-03 | 1.59 | 5.4E-03 |
| 3 | 3S---- | B3run44-3 | 12/20/2004 | 110 | 0 | 1.022E+05 | run OK | 1.93E+06 | 2.7E+03 | 1.63E+05 | 1.0E+02 | 0.016 | 7.7E-04 | 0.016 | 7.7E-04 | 1.61 | 1.3E-03 |
| 3 | 3S---- | B3run45-1 | 12/22/2004 | 110 | 0 | 1.015E+05 | no rotation of rotor | 6.86E+02 | 1.2E+00 | 5.09E+04 | 3.8E+01 | -0.010 | 9.2E-04 | -0.010 | 9.2E-04 | 1415.40 | 1.4E+00 |

| Build | Exp Code | Run | Bo* | +/- | Grsh,C2 | +/- | Grsh,C3 | +/- | Pr | +/- | Rot Speed (rpm) | +/- | Mass Flow (kg/s) | Ø | +/- | finØ | +/- |
|-------|----------|-----------|-------|---------|----------|---------|----------|---------|-------|---------|-----------------|---------|------------------|--------|---------|-------|---------|
| 3 | 3S----- | B3run2-1 | 0.075 | 7.4E-04 | 1.45E+08 | 1.8E+06 | 1.71E+08 | 2.0E+06 | 0.705 | 5.2E-05 | 1417.5 | 1.8E-01 | 0.610 | 0.310 | 5.6E-02 | 0.163 | 3.5E-02 |
| 3 | 3S----- | B3run2-2 | 0.075 | 5.7E-04 | 1.45E+08 | 1.4E+06 | 1.70E+08 | 1.3E+06 | 0.705 | 3.9E-05 | 1415.3 | 1.4E-01 | 0.610 | 0.296 | 2.9E-02 | 0.162 | 1.3E-02 |
| 3 | 3S----- | B3run3-1 | 0.079 | 7.5E-04 | 1.50E+08 | 1.6E+06 | 1.73E+08 | 1.7E+06 | 0.705 | 4.7E-05 | 1480.7 | 1.0E-01 | 0.578 | 0.357 | 4.7E-02 | 0.187 | 1.9E-02 |
| 3 | 3S----- | B3run4-1 | 0.209 | 2.1E-03 | 1.62E+08 | 2.1E+06 | 1.71E+08 | 2.1E+06 | 0.706 | 5.8E-05 | 1311.8 | 2.4E-01 | 0.230 | 1.540 | 1.3E+00 | 0.858 | 8.0E-01 |
| 3 | 3S----- | B3run4-2 | 0.211 | 2.6E-03 | 1.65E+08 | 2.6E+06 | 1.73E+08 | 2.7E+06 | 0.706 | 6.7E-05 | 1309.1 | 1.2E-01 | 0.231 | 1.419 | 5.3E+00 | 0.803 | 3.2E+00 |
| 3 | 3S----- | B3run5-1 | 0.113 | 1.2E-03 | 1.53E+08 | 1.8E+06 | 1.66E+08 | 1.8E+06 | 0.706 | 5.0E-05 | 1328.9 | 2.0E-01 | 0.391 | 1.010 | 1.6E+00 | 0.530 | 9.0E-01 |
| 3 | 3S----- | B3run5-3 | 0.120 | 7.8E-04 | 1.76E+08 | 1.3E+06 | 1.90E+08 | 1.5E+06 | 0.707 | 3.3E-05 | 1324.0 | 0.0E+00 | 0.391 | 0.857 | 7.4E-01 | 0.477 | 3.4E-01 |
| 3 | 3S----- | B3run6-1 | 0.123 | 1.0E-03 | 1.51E+08 | 1.5E+06 | 1.65E+08 | 1.4E+06 | 0.706 | 4.0E-05 | 1390.5 | 2.4E-01 | 0.359 | 1.136 | 9.2E-01 | 0.581 | 4.5E-01 |
| 3 | 3S----- | B3run6-2 | 0.121 | 3.8E-03 | 1.51E+08 | 6.3E+06 | 1.66E+08 | 6.5E+06 | 0.706 | 1.4E-04 | 1410.5 | 3.5E-01 | 0.357 | 1.095 | 1.2E+00 | 0.637 | 4.2E-01 |
| 3 | 3S----- | B3run6-3 | 0.122 | 1.1E-03 | 1.50E+08 | 1.4E+06 | 1.65E+08 | 1.4E+06 | 0.706 | 4.1E-05 | 1409.5 | 2.6E-01 | 0.357 | 1.037 | 1.5E+00 | 0.569 | 8.8E-01 |
| 3 | 3S----- | B3run7-1 | 0.274 | 2.7E-03 | 1.82E+08 | 2.7E+06 | 1.97E+08 | 2.5E+06 | 0.708 | 4.1E-05 | 1342.1 | 2.6E-01 | 0.183 | 2.175 | 6.0E+00 | 1.351 | 3.2E+00 |
| 3 | 3S----- | B3run7-2 | 0.262 | 3.1E-03 | 1.47E+08 | 2.1E+06 | 1.58E+08 | 2.2E+06 | 0.705 | 6.6E-05 | 1366.2 | 1.9E-01 | 0.180 | 2.096 | 2.6E+00 | 1.173 | 1.5E+00 |
| 3 | 3S----- | B3run7-4 | 0.157 | 2.1E-03 | 5.08E+07 | 8.4E+05 | 5.26E+07 | 8.4E+05 | 0.705 | 8.5E-05 | 766.2 | 1.2E-01 | 0.180 | 1.517 | 1.2E+01 | 0.689 | 5.3E+00 |
| 3 | 3S----- | B3run8-1 | 0.155 | 1.1E-03 | 4.44E+07 | 4.0E+05 | 4.60E+07 | 4.2E+05 | 0.705 | 4.8E-05 | 656.0 | 2.8E-01 | 0.174 | 1.638 | 1.8E+00 | 0.659 | 5.9E-01 |
| 3 | 3S----- | B3run9-1 | 0.044 | 3.1E-04 | 4.19E+07 | 3.4E+05 | 4.66E+07 | 3.9E+05 | 0.706 | 4.1E-05 | 623.0 | 3.4E-02 | 0.556 | -0.174 | 1.2E-01 | 0.224 | 1.7E-01 |
| 3 | 3S----- | B3run9-2 | 0.404 | 3.1E-03 | 3.47E+08 | 3.2E+06 | 3.51E+08 | 3.0E+06 | 0.706 | 4.2E-05 | 2177.8 | 8.2E-02 | 0.178 | 1.711 | 3.2E-01 | 1.126 | 2.3E-01 |
| 3 | 3S----- | B3run9-3 | 0.403 | 3.0E-03 | 3.45E+08 | 3.1E+06 | 3.49E+08 | 3.1E+06 | 0.705 | 4.0E-05 | 2178.1 | 1.9E-01 | 0.178 | 1.760 | 3.3E-01 | 1.162 | 1.8E-01 |
| 3 | 3S----- | B3run9-4 | 0.173 | 1.8E-03 | 3.51E+08 | 3.6E+06 | 3.45E+08 | 3.4E+06 | 0.705 | 4.6E-05 | 2171.6 | 2.3E-01 | 0.382 | 1.047 | 1.3E+00 | 0.686 | 8.0E-01 |
| 3 | 3S----- | B3run12-1 | 0.175 | 1.1E-03 | 1.40E+08 | 1.0E+06 | 1.50E+08 | 1.1E+06 | 0.705 | 3.5E-05 | 1403.0 | 2.0E-02 | 0.256 | 1.160 | 1.6E-01 | 0.779 | 1.3E-01 |
| 3 | 3S----- | B3run13-1 | 0.062 | 6.8E-04 | 1.43E+08 | 1.8E+06 | 1.72E+08 | 1.8E+06 | 0.705 | 5.3E-05 | 1521.4 | 3.3E-01 | 0.726 | 0.123 | 3.0E-02 | 0.169 | |

| Build | Exp Code | Run | Bo* | +/- | Grsh,C2 | +/- | Grsh,C3 | +/- | Pr | +/- | Rot Speed (rpm) | +/- | Mass Flow (kg/s) | Ø | +/- | finØ | +/- |
|-------|----------|------------|-------|---------|----------|---------|----------|---------|-------|---------|-----------------|---------|------------------|-------|---------|-------|---------|
| 3 | 3S----- | B3run33-2 | 0.278 | 2.8E-03 | 6.91E+08 | 6.4E+06 | 6.79E+08 | 6.5E+06 | 0.707 | 3.5E-05 | 3028.8 | 3.1E-01 | 0.318 | 1.414 | 1.6E+00 | 0.997 | 9.1E-01 |
| 3 | 3S----- | B3run33-3 | 0.364 | 2.3E-03 | 2.18E+08 | 1.8E+06 | 2.25E+08 | 1.8E+06 | 0.707 | 3.3E-05 | 1480.7 | 1.6E-01 | 0.158 | 1.492 | 1.5E-01 | 0.859 | 7.0E-02 |
| 3 | 3S----- | B3run33-4 | 0.070 | 6.7E-04 | 1.67E+08 | 1.5E+06 | 1.82E+08 | 1.6E+06 | 0.706 | 4.0E-05 | 1477.4 | 2.1E-01 | 0.635 | 0.305 | 3.3E-02 | 0.177 | 1.7E-02 |
| 3 | 3S----- | B3run33-5 | 0.234 | 4.1E-03 | 2.17E+09 | 2.5E+07 | 1.98E+09 | 2.5E+07 | 0.705 | 4.3E-05 | 4915.7 | 9.3E-02 | 0.639 | 1.126 | 6.1E-01 | 0.837 | 4.3E-01 |
| 3 | 3S----- | B3run33-6 | 1.667 | 1.3E-02 | 2.44E+09 | 2.5E+07 | 2.41E+09 | 2.4E+07 | 0.706 | 3.6E-05 | 5060.8 | 1.8E-01 | 0.126 | 2.176 | 2.4E-01 | 1.619 | 1.2E-01 |
| 3 | 3S----- | B3run33-7 | 0.170 | 1.1E-03 | 2.08E+08 | 1.6E+06 | 2.12E+08 | 1.5E+06 | 0.706 | 3.3E-05 | 1575.8 | 2.6E-01 | 0.320 | 0.749 | 1.2E-01 | 0.435 | 5.8E-02 |
| 3 | 3S----- | B3run33-8 | 0.142 | 1.4E-03 | 6.88E+08 | 6.4E+06 | 6.75E+08 | 6.4E+06 | 0.705 | 3.8E-05 | 3177.2 | 1.3E-01 | 0.640 | 0.496 | 5.9E-02 | 0.314 | 2.9E-02 |
| 3 | 3S----- | B3run33-9 | 0.615 | 4.4E-03 | 6.88E+08 | 5.9E+06 | 6.87E+08 | 5.9E+06 | 0.705 | 3.3E-05 | 3317.8 | 8.3E-02 | 0.166 | 1.880 | 2.5E-01 | 1.354 | 1.6E-01 |
| 3 | 3S----- | B3run33-10 | 0.590 | 5.3E-03 | 2.47E+09 | 2.6E+07 | 2.30E+09 | 2.4E+07 | 0.705 | 3.3E-05 | 4748.1 | 2.3E-01 | 0.315 | 1.306 | 1.7E-01 | 0.925 | 1.0E-01 |
| 3 | 3S----- | B3run34-1 | 0.122 | 1.1E-03 | 2.00E+08 | 1.7E+06 | 2.03E+08 | 1.9E+06 | 0.706 | 4.0E-05 | 1506.3 | 2.1E-01 | 0.413 | 0.664 | 3.2E-01 | 0.388 | 2.5E-01 |
| 3 | 3S----- | B3run34-2 | 0.384 | 5.2E-03 | 2.17E+09 | 2.3E+07 | 2.05E+09 | 2.2E+07 | 0.705 | 3.5E-05 | 5366.8 | 2.1E-01 | 0.403 | 1.636 | 1.7E+00 | 1.221 | 1.6E+00 |
| 3 | 3S----- | B3run34-3 | 0.373 | 4.2E-03 | 6.70E+08 | 7.0E+06 | 6.55E+08 | 7.1E+06 | 0.706 | 3.9E-05 | 2715.3 | 2.2E-01 | 0.239 | 2.083 | 4.7E+01 | 1.380 | 3.4E+01 |
| 3 | 3S----- | B3run34-4 | 0.217 | 1.6E-03 | 2.03E+08 | 1.6E+06 | 2.01E+08 | 1.8E+06 | 0.706 | 3.4E-05 | 1381.6 | 2.6E-01 | 0.242 | 1.005 | 1.4E-01 | 0.590 | 6.9E-02 |
| 3 | 3S----- | B3run34-5 | 0.209 | 2.3E-03 | 6.62E+08 | 6.3E+06 | 6.38E+08 | 6.4E+06 | 0.705 | 3.8E-05 | 3008.9 | 2.0E-01 | 0.405 | 1.317 | 3.4E+00 | 0.846 | 2.5E+00 |
| 3 | 3S----- | B3run34-6 | 0.184 | 2.8E-03 | 2.02E+09 | 2.1E+07 | 1.82E+09 | 1.9E+07 | 0.704 | 3.2E-05 | 5495.9 | 1.0E-01 | 0.721 | 1.587 | 1.3E+00 | 1.153 | 1.1E+00 |
| 3 | 3S----- | B3run34-7 | 0.740 | 8.3E-03 | 2.11E+09 | 2.3E+07 | 2.01E+09 | 2.3E+07 | 0.705 | 3.6E-05 | 5270.8 | 2.0E-01 | 0.230 | 2.505 | 1.0E+02 | 1.992 | 1.0E+02 |
| 3 | 3S----- | B3run34-8 | 0.109 | 1.4E-03 | 6.39E+08 | 6.2E+06 | 6.15E+08 | 5.8E+06 | 0.704 | 3.8E-05 | 3071.6 | 1.1E-01 | 0.719 | 0.652 | 3.0E-01 | 0.403 | 1.9E-01 |
| 3 | 3S----- | B3run34-9 | 0.063 | 7.1E-04 | 1.78E+08 | 1.9E+06 | 1.93E+08 | 1.9E+06 | 0.704 | 4.4E-05 | 1546.8 | 2.1E-01 | 0.716 | 0.366 | 1.3E-01 | 0.187 | 8.7E-02 |
| 3 | 3S----- | B3run35-1 | 0.158 | 1.2E-03 | 2.04E+08 | 1.7E+06 | 2.19E+08 | 1.7E+06 | 0.705 | 3.6E-05 | 1559.0 | 3.9E-02 | 0.329 | 1.093 | 7.5E-01 | 0.690 | 6.4E-01 |
| 3 | 3S----- | B3run35-2 | 0.119 | 1.8E-03 | 6.21E+08 | 6.6E+06 | 5.64E+08 | 6.5E+06 | 0.705 | 4.1E-05 | 2969.4 | 1.8E-01 | 0.641 | 0.922 | 6.9E-01 | 0.647 | 3.7E-01 |
| 3 | 3S----- | B3run35-3 | 0.255 | 3.0E-03 | 6.17E+08 | 5.9E+06 | 5.69E+08 | 5.8E+06 | 0.705 | 3.6E-05 | 2952.9 | 3.3E-01 | 0.319 | 2.153 | 1.6E+00 | 1.316 | 8.4E-01 |
| 3 | 3S----- | B3run35-4 | 0.191 | 5.6E-03 | 1.73E+09 | 2.6E+07 | 1.31E+09 | 2.6E+07 | 0.704 | 5.0E-05 | 5078.7 | 2.4E+00 | 0.643 | 1.730 | 2.7E+00 | 1.156 | 1.7E+00 |
| 3 | 3S----- | B3run35-5 | 0.638 | 5.2E-03 | 6.19E+08 | 5.9E+06 | 5.76E+08 | 5.7E+06 | 0.704 | 3.7E-05 | 3183.8 | 2.9E-01 | 0.153 | 1.793 | 3.0E-01 | 1.258 | 2.0E-01 |
| 3 | 3S----- | B3run35-6 | 0.466 | 7.0E-03 | 1.81E+09 | 2.0E+07 | 1.42E+09 | 2.0E+07 | 0.704 | 3.8E-05 | 5173.5 | 1.4E+00 | 0.318 | 1.674 | 7.6E+00 | 1.126 | 3.7E+00 |
| 3 | 3S----- | B3run35-7 | 0.313 | 2.0E-03 | 1.96E+08 | 1.5E+06 | 2.04E+08 | 1.6E+06 | 0.704 | 3.6E-05 | 1842.0 | 3.9E-02 | 0.174 | 2.356 | 2.4E+00 | 1.232 | 1.3E+00 |
| 3 | 3S----- | B3run35-8 | 0.072 | 7.3E-04 | 1.80E+08 | 1.7E+06 | 1.94E+08 | 1.8E+06 | 0.703 | 4.5E-05 | 1861.1 | 6.6E-02 | 0.651 | 0.367 | 6.2E-02 | 0.224 | 3.0E-02 |
| 3 | 3S----- | B3run35-9 | 1.068 | 1.4E-02 | 1.71E+09 | 2.3E+07 | 1.33E+09 | 2.2E+07 | 0.703 | 4.1E-05 | 5542.1 | 2.4E-01 | 0.150 | 2.566 | 1.7E+00 | 1.928 | 1.8E+00 |

| Build | Exp Code | Run | Bo* | +/- | Grsh,C2 | +/- | Grsh,C3 | +/- | Pr | +/- | Rot Speed (rpm) | +/- | Mass Flow (kg/s) | Ø | +/- | finØ | +/- |
|-------|-------------|-----------|-------|---------|-----------|---------|-----------|---------|-------|---------|-----------------------|---------|------------------------|--------|---------|--------|---------|
| 3 | 3S----- | B3run36-1 | 0.131 | 1.0E-03 | 1.96E+08 | 1.9E+06 | 2.12E+08 | 1.9E+06 | 0.704 | 4.3E-05 | 1611.3 | 3.2E-01 | 0.412 | 0.711 | 1.0E+00 | 0.420 | 5.3E-01 |
| 3 | 3S----- | B3run37-1 | 0.065 | 2.9E-02 | 3.02E+08 | 2.1E+07 | 2.70E+08 | 2.2E+07 | 0.705 | 1.4E-04 | 2964.1 | 3.6E-01 | 0.203 | -1.622 | 1.6E+00 | -1.314 | 8.9E-01 |
| 3 | 3S----- | B3run38-1 | 0.034 | 2.4E-04 | 2.65E+07 | 2.2E+05 | 3.25E+07 | 2.5E+05 | 0.704 | 4.6E-05 | 655.0 | 0.0E+00 | 0.595 | 0.146 | 1.5E-02 | 0.069 | 7.1E-03 |
| 3 | 3S----- | B3run39-1 | 0.000 | 1.5E-02 | 2.85E+08 | 3.7E+06 | 2.49E+08 | 3.7E+06 | 0.704 | 4.0E-05 | 3143.4 | 3.0E-01 | 0.200 | -1.712 | 4.6E-01 | -1.193 | 2.8E-01 |
| 3 | 3S----- | B3run40-1 | 0.110 | 4.8E-04 | 3.18E+07 | 2.0E+05 | 3.37E+07 | 2.1E+05 | 0.704 | 3.9E-05 | 608.8 | 1.9E-01 | 0.227 | 0.399 | 5.5E-02 | 0.310 | 2.7E-02 |
| 3 | 3S----- | B3run44-1 | 0.040 | 1.1E-02 | 4.05E+08 | 3.1E+07 | 3.70E+08 | 3.1E+07 | 0.707 | 1.4E-04 | 2780.1 | 9.8E-02 | 0.596 | -0.759 | 4.1E+01 | -0.712 | 7.8E-01 |
| 3 | 3S----- | B3run44-2 | 0.068 | 1.1E-02 | 4.17E+08 | 2.6E+07 | 3.84E+08 | 2.6E+07 | 0.707 | 1.6E-04 | 2775.5 | 1.7E-01 | 0.596 | -1.657 | 2.1E+00 | -1.449 | 1.3E-01 |
| 3 | 3S----- | B3run44-3 | 0.078 | 1.9E-03 | 4.12E+08 | 4.8E+06 | 3.81E+08 | 4.5E+06 | 0.706 | 3.9E-05 | 2768.6 | 2.7E-01 | 0.595 | -5.395 | 1.3E+01 | -4.043 | 7.4E+00 |
| 3 | 3S----- | B3run45-1 | 0.000 | 3.9E-06 | -5.47E+00 | 4.6E-01 | -6.08E+00 | 5.0E-01 | 0.707 | 4.4E-05 | 1.0 | 0.0E+00 | 0.183 | 0.069 | 3.0E-02 | 0.001 | 3.2E-03 |

| Build | Exp Code | Run | Tcav2 (K) | +/- | Tcav3 (K) | +/- | Tinlet (K) | +/- | Texit (K) | +/- | Dtmeas (K) | +/- | Dtcalc (K) | +/- | Tsh,C2 (K) | +/- | Tsh,C3 (K) | +/- |
|-------|----------|-----------|-----------|---------|-----------|---------|------------|---------|-----------|---------|------------|---------|------------|---------|------------|---------|------------|---------|
| 3 | 3S---- | B3run2-1 | 323.49 | 1.5E+00 | 335.25 | 1.3E+00 | 316.87 | 3.2E-01 | 318.80 | 3.0E-01 | 1.93 | 4.3E-01 | 1.92 | 4.9E-02 | 368.17 | 3.2E-01 | 377.00 | 3.0E-01 |
| 3 | 3S---- | B3run2-2 | 324.31 | 1.1E+00 | 335.86 | 8.6E-01 | 317.32 | 2.4E-01 | 319.27 | 2.4E-01 | 1.95 | 3.2E-01 | 1.91 | 3.6E-02 | 368.92 | 2.3E-01 | 377.76 | 1.9E-01 |
| 3 | 3S---- | B3run3-1 | 326.92 | 1.2E+00 | 337.38 | 8.3E-01 | 319.73 | 2.9E-01 | 321.79 | 2.9E-01 | 2.06 | 4.0E-01 | 2.09 | 4.6E-02 | 371.77 | 2.4E-01 | 379.69 | 1.9E-01 |
| 3 | 3S---- | B3run4-1 | 325.12 | 1.3E+00 | 324.68 | 1.3E+00 | 314.06 | 3.6E-01 | 319.43 | 3.4E-01 | 5.36 | 4.8E-01 | 5.36 | 1.2E-01 | 369.53 | 2.8E-01 | 372.49 | 2.5E-01 |
| 3 | 3S---- | B3run4-2 | 325.39 | 2.9E+00 | 325.08 | 2.8E+00 | 313.83 | 4.2E-01 | 319.49 | 3.7E-01 | 5.67 | 5.6E-01 | 5.34 | 2.9E-01 | 369.99 | 5.1E-01 | 372.80 | 5.0E-01 |
| 3 | 3S---- | B3run5-1 | 321.39 | 1.4E+00 | 324.62 | 1.3E+00 | 314.14 | 3.2E-01 | 317.48 | 3.4E-01 | 3.34 | 5.0E-01 | 3.35 | 8.6E-02 | 367.95 | 2.7E-01 | 372.65 | 2.7E-01 |
| 3 | 3S---- | B3run5-3 | 316.66 | 1.0E+00 | 320.43 | 1.1E+00 | 308.14 | 2.2E-01 | 311.87 | 2.5E-01 | 3.73 | 3.6E-01 | 3.56 | 6.9E-02 | 365.43 | 1.9E-01 | 370.08 | 2.1E-01 |
| 3 | 3S---- | B3run6-1 | 319.87 | 9.8E-01 | 322.46 | 1.1E+00 | 311.65 | 2.6E-01 | 315.41 | 2.6E-01 | 3.75 | 3.4E-01 | 3.74 | 7.7E-02 | 367.34 | 1.9E-01 | 372.73 | 2.0E-01 |
| 3 | 3S---- | B3run6-2 | 318.94 | 7.8E+00 | 321.56 | 7.6E+00 | 311.73 | 9.1E-01 | 315.61 | 1.0E+00 | 3.88 | 1.4E+00 | 3.74 | 6.1E-01 | 366.01 | 1.6E+00 | 371.39 | 1.5E+00 |
| 3 | 3S---- | B3run6-3 | 320.56 | 1.2E+00 | 323.76 | 1.2E+00 | 312.84 | 2.6E-01 | 316.69 | 2.6E-01 | 3.85 | 3.5E-01 | 3.73 | 7.4E-02 | 367.74 | 2.8E-01 | 373.12 | 2.2E-01 |
| 3 | 3S---- | B3run7-1 | 310.31 | 3.4E+00 | 311.38 | 2.7E+00 | 302.03 | 2.8E-01 | 309.55 | 2.2E-01 | 7.52 | 3.8E-01 | 7.42 | 4.1E-01 | 358.82 | 6.5E-01 | 363.56 | 5.7E-01 |
| 3 | 3S---- | B3run7-2 | 327.62 | 1.6E+00 | 327.72 | 1.3E+00 | 317.74 | 4.1E-01 | 324.53 | 3.6E-01 | 6.79 | 5.1E-01 | 6.85 | 1.9E-01 | 372.40 | 3.4E-01 | 376.50 | 2.8E-01 |
| 3 | 3S---- | B3run7-4 | 333.15 | 2.5E+00 | 334.42 | 2.3E+00 | 320.84 | 5.2E-01 | 326.87 | 4.9E-01 | 6.03 | 7.6E-01 | 5.72 | 2.5E-01 | 383.08 | 3.5E-01 | 385.33 | 3.2E-01 |
| 3 | 3S---- | B3run8-1 | 331.45 | 1.5E+00 | 331.56 | 1.4E+00 | 316.35 | 3.0E-01 | 322.94 | 2.6E-01 | 6.59 | 4.0E-01 | 6.08 | 1.5E-01 | 384.32 | 2.2E-01 | 386.74 | 2.0E-01 |
| 3 | 3S---- | B3run9-1 | 326.00 | 1.5E+00 | 328.75 | 1.5E+00 | 315.21 | 2.6E-01 | 317.45 | 3.1E-01 | 2.24 | 4.1E-01 | 1.84 | 4.8E-02 | 373.59 | 1.8E-01 | 380.10 | 2.0E-01 |
| 3 | 3S---- | B3run9-2 | 329.37 | 7.3E-01 | 327.47 | 7.3E-01 | 316.22 | 2.6E-01 | 323.07 | 2.5E-01 | 6.85 | 3.5E-01 | 7.46 | 1.2E-01 | 370.11 | 1.7E-01 | 370.75 | 1.8E-01 |
| 3 | 3S---- | B3run9-3 | 329.35 | 7.6E-01 | 327.42 | 6.1E-01 | 316.48 | 2.5E-01 | 323.32 | 2.3E-01 | 6.84 | 3.5E-01 | 7.48 | 1.0E-01 | 370.26 | 1.7E-01 | 370.80 | 1.5E-01 |
| 3 | 3S---- | B3run9-4 | 329.81 | 8.2E-01 | 326.35 | 7.2E-01 | 317.57 | 2.8E-01 | 320.90 | 2.8E-01 | 3.32 | 4.0E-01 | 3.55 | 5.7E-02 | 370.29 | 1.8E-01 | 369.48 | 1.6E-01 |
| 3 | 3S---- | B3run12-1 | 326.48 | 9.8E-01 | 328.18 | 8.4E-01 | 317.07 | 2.2E-01 | 322.42 | 2.3E-01 | 5.35 | 3.1E-01 | 5.17 | 8.5E-02 | 375.45 | 1.8E-01 | 379.54 | 1.5E-01 |
| 3 | 3S---- | B3run13-1 | 324.84 | 1.1E+00 | 335.77 | 9.6E-01 | 319.73 | 3.2E-01 | 321.25 | 3.6E-01 | 1.52 | 4.5E-01 | 1.71 | 3.7E-02 | 369.90 | 2.5E-01 | 379.94 | 2.0E-01 |
| 3 | 3S---- | B3run13-2 | 324.95 | 9.6E-01 | 335.60 | 7.7E-01 | 319.29 | 4.5E-01 | 320.82 | 3.8E-01 | 1.54 | 5.1E-01 | 1.70 | 3.0E-02 | 369.96 | 2.0E-01 | 380.01 | 1.6E-01 |
| 3 | 3S---- | B3run13-3 | 321.14 | 5.4E-01 | 321.38 | 4.9E-01 | 311.31 | 2.4E-01 | 320.49 | 2.0E-01 | 9.18 | 2.9E-01 | 7.20 | 8.1E-02 | 361.74 | 1.6E-01 | 360.46 | 1.5E-01 |
| 3 | 3S---- | B3run15-1 | 319.85 | 1.0E+00 | 324.67 | 7.1E-01 | 314.98 | 2.7E-01 | 319.32 | 2.5E-01 | 4.34 | 3.7E-01 | 4.29 | 6.9E-02 | 368.78 | 2.1E-01 | 372.76 | 1.4E-01 |
| 3 | 3S---- | B3run15-2 | 318.24 | 5.9E-01 | 321.17 | 6.5E-01 | 310.88 | 2.3E-01 | 319.44 | 2.6E-01 | 8.57 | 3.4E-01 | 9.25 | 1.3E-01 | 359.18 | 1.7E-01 | 359.20 | 1.8E-01 |
| 3 | 3S---- | B3run16-1 | 322.98 | 5.6E-01 | 321.75 | 6.2E-01 | 315.54 | 2.5E-01 | 319.84 | 2.2E-01 | 4.31 | 3.1E-01 | 4.55 | 6.4E-02 | 361.74 | 1.3E-01 | 360.65 | 1.6E-01 |

| Build | Exp Code | Run | Tcav2 (K) | +/- | Tcav3 (K) | +/- | Tinlet (K) | +/- | Texit (K) | +/- | Dtmeas (K) | +/- | Dtcalc (K) | +/- | Tsh,C2 (K) | +/- | Tsh,C3 (K) | +/- |
|-------|----------|-----------|-----------|---------|-----------|---------|------------|---------|-----------|---------|------------|---------|------------|---------|------------|---------|------------|---------|
| 3 | 3S---- | B3run16-2 | 327.05 | 5.1E-01 | 327.33 | 5.5E-01 | 318.66 | 2.4E-01 | 326.17 | 2.5E-01 | 7.51 | 3.3E-01 | 5.97 | 7.6E-02 | 364.98 | 1.6E-01 | 363.75 | 1.7E-01 |
| 3 | 3S---- | B3run17-1 | 322.88 | 5.8E-01 | 321.59 | 5.4E-01 | 315.31 | 2.2E-01 | 317.69 | 2.7E-01 | 2.38 | 3.5E-01 | 2.39 | 3.1E-02 | 362.11 | 1.5E-01 | 361.67 | 1.5E-01 |
| 3 | 3S---- | B3run17-2 | 323.18 | 6.1E-01 | 321.94 | 6.9E-01 | 315.54 | 2.6E-01 | 317.91 | 2.6E-01 | 2.37 | 4.0E-01 | 2.39 | 3.5E-02 | 362.26 | 1.7E-01 | 361.87 | 1.7E-01 |
| 3 | 3S---- | B3run17-3 | 325.70 | 5.2E-01 | 322.46 | 5.2E-01 | 316.92 | 2.5E-01 | 320.35 | 2.2E-01 | 3.43 | 3.5E-01 | 2.92 | 3.6E-02 | 362.31 | 1.6E-01 | 359.14 | 1.7E-01 |
| 3 | 3S---- | B3run17-4 | 325.81 | 5.0E-01 | 322.50 | 5.5E-01 | 317.11 | 2.4E-01 | 320.53 | 2.4E-01 | 3.43 | 3.0E-01 | 2.94 | 3.8E-02 | 362.48 | 1.6E-01 | 359.28 | 1.7E-01 |
| 3 | 3S---- | B3run17-5 | 324.25 | 9.1E-01 | 324.27 | 7.2E-01 | 313.07 | 2.1E-01 | 322.04 | 2.1E-01 | 8.97 | 2.8E-01 | 8.27 | 1.1E-01 | 369.36 | 2.1E-01 | 374.09 | 1.5E-01 |
| 3 | 3S---- | B3run17-6 | 324.49 | 9.9E-01 | 324.49 | 8.8E-01 | 313.06 | 2.5E-01 | 322.05 | 2.1E-01 | 8.99 | 3.4E-01 | 8.41 | 1.4E-01 | 369.41 | 1.7E-01 | 374.17 | 1.8E-01 |
| 3 | 3S---- | B3run18-1 | 324.58 | 1.4E+00 | 328.74 | 1.4E+00 | 313.93 | 2.5E-01 | 320.68 | 2.3E-01 | 6.74 | 3.3E-01 | 5.84 | 1.3E-01 | 378.60 | 2.0E-01 | 382.03 | 2.0E-01 |
| 3 | 3S---- | B3run18-2 | 331.58 | 7.8E-01 | 330.17 | 7.0E-01 | 321.42 | 2.4E-01 | 328.22 | 2.4E-01 | 6.80 | 3.3E-01 | 7.02 | 1.3E-01 | 370.90 | 1.9E-01 | 372.70 | 1.7E-01 |
| 3 | 3S---- | B3run18-3 | 326.31 | 8.5E-01 | 328.25 | 9.5E-01 | 318.78 | 2.2E-01 | 322.39 | 2.4E-01 | 3.61 | 3.5E-01 | 3.52 | 5.7E-02 | 371.64 | 1.7E-01 | 375.72 | 1.9E-01 |
| 3 | 3S---- | B3run19-1 | 327.11 | 3.8E+00 | 330.83 | 4.0E+00 | 319.99 | 3.3E-01 | 323.37 | 3.2E-01 | 3.38 | 4.4E-01 | 3.33 | 2.2E-01 | 372.16 | 8.1E-01 | 376.00 | 7.9E-01 |
| 3 | 3S---- | B3run20-1 | 324.28 | 9.0E-01 | 328.48 | 8.3E-01 | 318.57 | 5.9E-01 | 322.33 | 5.8E-01 | 3.76 | 8.9E-01 | 3.62 | 8.3E-02 | 367.51 | 1.9E-01 | 370.73 | 1.7E-01 |
| 3 | 3S---- | B3run21-1 | 323.29 | 1.0E+00 | 326.62 | 8.3E-01 | 316.75 | 2.4E-01 | 320.03 | 2.4E-01 | 3.29 | 3.5E-01 | 3.09 | 6.7E-02 | 361.92 | 2.1E-01 | 363.63 | 1.7E-01 |
| 3 | 3S---- | B3run22-1 | 320.09 | 6.5E-01 | 322.38 | 7.5E-01 | 311.53 | 2.5E-01 | 318.97 | 2.8E-01 | 7.45 | 3.9E-01 | 6.79 | 1.1E-01 | 361.80 | 1.7E-01 | 365.30 | 1.6E-01 |
| 3 | 3S---- | B3run22-2 | 329.31 | 8.7E-01 | 327.62 | 7.8E-01 | 321.35 | 2.6E-01 | 325.00 | 2.2E-01 | 3.65 | 3.5E-01 | 4.01 | 8.8E-02 | 363.34 | 2.4E-01 | 361.42 | 2.3E-01 |
| 3 | 3S---- | B3run23-1 | 326.87 | 6.7E-01 | 323.72 | 7.1E-01 | 318.33 | 2.1E-01 | 321.49 | 2.2E-01 | 3.16 | 2.9E-01 | 3.51 | 5.0E-02 | 363.33 | 1.9E-01 | 360.06 | 1.8E-01 |
| 3 | 3S---- | B3run24-1 | 312.77 | 1.0E+00 | 315.35 | 8.5E-01 | 306.98 | 2.4E-01 | 311.10 | 2.8E-01 | 4.12 | 3.9E-01 | 3.60 | 6.8E-02 | 354.74 | 2.4E-01 | 358.35 | 2.1E-01 |
| 3 | 3S---- | B3run24-2 | 351.69 | 1.5E+00 | 353.24 | 1.3E+00 | 338.71 | 3.2E-01 | 342.13 | 2.9E-01 | 3.42 | 4.6E-01 | 2.75 | 7.6E-02 | 385.45 | 3.0E-01 | 388.43 | 2.5E-01 |
| 3 | 3S---- | B3run25-1 | 335.67 | 1.2E+00 | 342.81 | 9.2E-01 | 327.80 | 2.2E-01 | 329.73 | 2.3E-01 | 1.93 | 3.3E-01 | 1.81 | 3.7E-02 | 376.05 | 2.5E-01 | 382.24 | 1.9E-01 |
| 3 | 3S---- | B3run26-1 | 335.43 | 5.5E-01 | 332.95 | 5.9E-01 | 324.18 | 2.1E-01 | 330.50 | 2.3E-01 | 6.32 | 3.0E-01 | 6.75 | 1.1E-01 | 367.27 | 1.6E-01 | 364.86 | 1.5E-01 |
| 3 | 3S---- | B3run27-1 | 328.08 | 6.5E-01 | 323.32 | 6.4E-01 | 319.31 | 2.5E-01 | 321.20 | 2.4E-01 | 1.90 | 3.7E-01 | 1.97 | 3.0E-02 | 363.61 | 1.7E-01 | 359.84 | 1.8E-01 |
| 3 | 3S---- | B3run28-1 | 324.92 | 6.7E-01 | 319.62 | 6.7E-01 | 314.83 | 2.5E-01 | 318.29 | 2.3E-01 | 3.45 | 3.3E-01 | 3.28 | 5.5E-02 | 354.86 | 2.4E-01 | 349.15 | 2.3E-01 |
| 3 | 3S---- | B3run28-2 | 329.67 | 5.9E-01 | 325.17 | 4.5E-01 | 316.59 | 2.5E-01 | 322.92 | 2.4E-01 | 6.33 | 3.3E-01 | 6.15 | 9.8E-02 | 358.84 | 1.9E-01 | 353.69 | 1.3E-01 |
| 3 | 3S---- | B3run30-1 | 318.73 | 9.8E-01 | 317.80 | 9.9E-01 | 313.79 | 2.3E-01 | 316.00 | 2.5E-01 | 2.21 | 3.5E-01 | 2.13 | 4.3E-02 | 353.22 | 3.2E-01 | 347.02 | 2.8E-01 |
| 3 | 3S---- | B3run31-1 | 310.28 | 1.2E+00 | 334.08 | 1.0E+00 | 320.16 | 2.7E-01 | 321.89 | 2.7E-01 | 1.74 | 3.8E-01 | 1.78 | 2.6E-02 | 372.45 | 1.8E-01 | 382.58 | 1.5E-01 |
| 3 | 3S---- | B3run32-1 | 317.27 | 1.3E+00 | 328.33 | 1.0E+00 | 309.75 | 2.4E-01 | 313.60 | 2.4E-01 | 3.85 | 3.3E-01 | 3.42 | 6.3E-02 | 377.70 | 2.0E-01 | 383.50 | 1.5E-01 |
| 3 | 3S---- | B3run33-1 | 317.52 | 1.1E+00 | 323.38 | 1.3E+00 | 306.02 | 2.9E-01 | 309.49 | 2.6E-01 | 3.47 | 3.9E-01 | 3.46 | 7.0E-02 | 363.88 | 2.1E-01 | 364.53 | 2.3E-01 |

| Build | Exp Code | Run | Tcav2 (K) | +/- | Tcav3 (K) | +/- | Tinlet (K) | +/- | Texit (K) | +/- | Dtmeas (K) | +/- | Dtcalc (K) | +/- | Tsh,C2 (K) | +/- | Tsh,C3 (K) | +/- |
|-------|----------|------------|-----------|---------|-----------|---------|------------|---------|-----------|---------|------------|---------|------------|---------|------------|---------|------------|---------|
| 3 | 3S---- | B3run33-2 | 316.77 | 6.2E-01 | 316.24 | 5.9E-01 | 308.68 | 2.3E-01 | 313.18 | 2.4E-01 | 4.50 | 3.3E-01 | 4.58 | 6.1E-02 | 355.25 | 1.7E-01 | 354.47 | 1.7E-01 |
| 3 | 3S---- | B3run33-3 | 315.73 | 7.4E-01 | 321.08 | 7.1E-01 | 307.36 | 2.2E-01 | 314.91 | 2.2E-01 | 7.55 | 3.5E-01 | 7.51 | 1.3E-01 | 360.54 | 1.8E-01 | 362.30 | 1.5E-01 |
| 3 | 3S---- | B3run33-4 | 319.72 | 8.9E-01 | 332.34 | 7.9E-01 | 316.21 | 2.5E-01 | 317.87 | 2.1E-01 | 1.67 | 3.4E-01 | 1.95 | 3.4E-02 | 366.89 | 1.8E-01 | 371.15 | 1.5E-01 |
| 3 | 3S---- | B3run33-5 | 327.20 | 5.0E-01 | 322.54 | 5.4E-01 | 317.30 | 2.7E-01 | 320.45 | 3.0E-01 | 3.15 | 3.7E-01 | 2.57 | 3.4E-02 | 358.03 | 1.7E-01 | 354.98 | 1.8E-01 |
| 3 | 3S---- | B3run33-6 | 329.50 | 6.9E-01 | 328.83 | 5.5E-01 | 315.24 | 2.3E-01 | 329.05 | 2.6E-01 | 13.81 | 3.5E-01 | 13.20 | 1.8E-01 | 361.88 | 2.6E-01 | 361.34 | 1.8E-01 |
| 3 | 3S---- | B3run33-7 | 324.85 | 7.9E-01 | 328.37 | 7.5E-01 | 314.78 | 2.1E-01 | 318.41 | 2.5E-01 | 3.63 | 3.2E-01 | 3.71 | 6.7E-02 | 368.60 | 1.7E-01 | 369.77 | 1.4E-01 |
| 3 | 3S---- | B3run33-8 | 329.54 | 6.1E-01 | 329.37 | 6.3E-01 | 318.37 | 2.3E-01 | 320.65 | 2.4E-01 | 2.28 | 3.3E-01 | 2.12 | 3.1E-02 | 365.44 | 1.8E-01 | 364.60 | 1.7E-01 |
| 3 | 3S---- | B3run33-9 | 327.03 | 6.8E-01 | 327.59 | 5.7E-01 | 317.09 | 2.1E-01 | 326.21 | 2.2E-01 | 9.13 | 3.2E-01 | 8.67 | 1.3E-01 | 365.51 | 2.0E-01 | 365.44 | 1.6E-01 |
| 3 | 3S---- | B3run33-10 | 331.38 | 6.4E-01 | 327.76 | 5.3E-01 | 318.72 | 2.0E-01 | 324.84 | 2.1E-01 | 6.12 | 3.0E-01 | 4.94 | 6.7E-02 | 360.54 | 2.3E-01 | 357.94 | 1.7E-01 |
| 3 | 3S---- | B3run34-1 | 321.50 | 7.6E-01 | 326.03 | 7.5E-01 | 314.44 | 2.5E-01 | 317.19 | 2.3E-01 | 2.76 | 3.5E-01 | 2.87 | 4.7E-02 | 365.05 | 1.6E-01 | 365.90 | 1.5E-01 |
| 3 | 3S---- | B3run34-2 | 326.39 | 5.1E-01 | 323.02 | 5.1E-01 | 316.88 | 2.2E-01 | 322.73 | 2.2E-01 | 5.86 | 3.0E-01 | 4.42 | 6.0E-02 | 359.97 | 1.7E-01 | 357.92 | 1.7E-01 |
| 3 | 3S---- | B3run34-3 | 322.47 | 6.2E-01 | 320.99 | 6.7E-01 | 314.14 | 2.5E-01 | 319.43 | 2.3E-01 | 5.29 | 3.5E-01 | 5.68 | 8.5E-02 | 357.96 | 1.9E-01 | 357.01 | 1.8E-01 |
| 3 | 3S---- | B3run34-4 | 323.57 | 7.4E-01 | 326.36 | 7.0E-01 | 314.62 | 2.2E-01 | 319.16 | 2.4E-01 | 4.54 | 3.3E-01 | 4.41 | 8.5E-02 | 363.88 | 1.8E-01 | 363.36 | 1.7E-01 |
| 3 | 3S---- | B3run34-5 | 325.75 | 6.2E-01 | 324.54 | 6.0E-01 | 317.89 | 2.3E-01 | 321.24 | 2.5E-01 | 3.34 | 3.3E-01 | 3.42 | 5.1E-02 | 362.06 | 1.8E-01 | 360.50 | 1.7E-01 |
| 3 | 3S---- | B3run34-6 | 332.64 | 5.4E-01 | 327.48 | 4.9E-01 | 323.89 | 1.9E-01 | 327.09 | 2.2E-01 | 3.20 | 3.2E-01 | 2.43 | 3.2E-02 | 365.30 | 1.7E-01 | 362.01 | 1.5E-01 |
| 3 | 3S---- | B3run34-7 | 329.28 | 6.1E-01 | 327.01 | 5.1E-01 | 320.65 | 2.2E-01 | 329.29 | 2.1E-01 | 8.64 | 2.9E-01 | 7.55 | 1.0E-01 | 362.68 | 2.2E-01 | 360.99 | 1.7E-01 |
| 3 | 3S---- | B3run34-8 | 332.94 | 6.6E-01 | 331.47 | 5.6E-01 | 323.86 | 2.3E-01 | 325.77 | 2.5E-01 | 1.90 | 3.5E-01 | 1.86 | 3.1E-02 | 367.67 | 1.9E-01 | 366.10 | 1.6E-01 |
| 3 | 3S---- | B3run34-9 | 326.29 | 8.5E-01 | 337.93 | 7.7E-01 | 324.68 | 2.6E-01 | 326.02 | 2.3E-01 | 1.34 | 3.3E-01 | 1.68 | 3.0E-02 | 370.50 | 2.0E-01 | 374.46 | 1.7E-01 |
| 3 | 3S---- | B3run35-1 | 327.25 | 7.8E-01 | 326.77 | 8.3E-01 | 317.09 | 2.2E-01 | 321.19 | 2.4E-01 | 4.10 | 3.5E-01 | 3.88 | 5.9E-02 | 369.14 | 1.6E-01 | 372.92 | 1.8E-01 |
| 3 | 3S---- | B3run35-2 | 329.68 | 5.5E-01 | 326.44 | 6.2E-01 | 321.44 | 2.5E-01 | 323.31 | 2.5E-01 | 1.87 | 3.4E-01 | 1.98 | 3.3E-02 | 363.45 | 1.6E-01 | 359.77 | 1.6E-01 |
| 3 | 3S---- | B3run35-3 | 329.46 | 5.6E-01 | 326.52 | 5.4E-01 | 321.51 | 2.2E-01 | 325.06 | 2.3E-01 | 3.55 | 3.2E-01 | 4.00 | 5.7E-02 | 363.28 | 1.7E-01 | 360.21 | 1.4E-01 |
| 3 | 3S---- | B3run35-4 | 335.04 | 4.9E-01 | 329.06 | 5.5E-01 | 326.36 | 2.9E-01 | 328.45 | 2.9E-01 | 2.09 | 4.2E-01 | 1.95 | 3.4E-02 | 361.39 | 1.5E-01 | 354.40 | 1.5E-01 |
| 3 | 3S---- | B3run35-5 | 333.80 | 6.2E-01 | 333.38 | 6.0E-01 | 323.81 | 2.2E-01 | 330.65 | 2.2E-01 | 6.83 | 3.5E-01 | 7.60 | 1.5E-01 | 367.14 | 1.9E-01 | 364.29 | 1.5E-01 |
| 3 | 3S---- | B3run35-6 | 336.38 | 4.9E-01 | 331.51 | 5.4E-01 | 326.11 | 2.2E-01 | 330.21 | 2.5E-01 | 4.10 | 3.4E-01 | 3.85 | 7.0E-02 | 362.83 | 1.4E-01 | 356.24 | 1.5E-01 |
| 3 | 3S---- | B3run35-7 | 337.40 | 8.5E-01 | 334.92 | 7.4E-01 | 324.92 | 2.1E-01 | 332.10 | 2.6E-01 | 7.19 | 3.4E-01 | 7.12 | 1.0E-01 | 378.49 | 1.7E-01 | 380.72 | 1.7E-01 |
| 3 | 3S---- | B3run35-8 | 336.97 | 9.4E-01 | 342.21 | 7.8E-01 | 328.91 | 2.6E-01 | 330.55 | 2.6E-01 | 1.64 | 3.5E-01 | 1.86 | 3.4E-02 | 378.94 | 1.9E-01 | 382.74 | 1.5E-01 |
| 3 | 3S---- | B3run35-9 | 338.10 | 6.4E-01 | 334.90 | 6.6E-01 | 328.60 | 2.4E-01 | 335.08 | 2.4E-01 | 6.48 | 3.5E-01 | 7.99 | 1.7E-01 | 365.64 | 2.0E-01 | 358.98 | 1.9E-01 |

| Build | Exp Code | Run | Tcav2 (K) | +/- | Tcav3 (K) | +/- | Tinlet (K) | +/- | Texit (K) | +/- | Dtmeas (K) | +/- | Dtcalc (K) | +/- | Tsh,C2 (K) | +/- | Tsh,C3 (K) | +/- |
|-------|----------|-----------|-----------|---------|-----------|---------|------------|---------|-----------|---------|------------|---------|------------|---------|------------|---------|------------|---------|
| 3 | 3S---- | B3run36-1 | 333.08 | 8.3E-01 | 335.08 | 8.3E-01 | 322.86 | 2.6E-01 | 325.83 | 2.7E-01 | 2.96 | 3.6E-01 | 3.05 | 5.7E-02 | 375.40 | 1.9E-01 | 379.71 | 1.8E-01 |
| 3 | 3S---- | B3run37-1 | 312.94 | 1.3E+01 | 311.58 | 3.2E+01 | 319.16 | 8.4E-01 | 324.02 | 9.3E-01 | 4.86 | 1.4E+00 | 6.48 | 7.4E+00 | 351.77 | 1.9E+00 | 348.46 | 1.8E+00 |
| 3 | 3S---- | B3run38-1 | 319.08 | 1.3E+00 | 361.17 | 1.7E+00 | 327.51 | 2.7E-01 | 329.38 | 2.6E-01 | 1.87 | 3.6E-01 | 1.86 | 3.6E-02 | 386.76 | 1.5E-01 | 400.04 | 1.7E-01 |
| 3 | 3S---- | B3run39-1 | 318.42 | 5.7E-01 | 317.72 | 5.5E-01 | 325.74 | 2.4E-01 | 330.37 | 2.2E-01 | 4.63 | 3.2E-01 | 6.16 | 8.9E-01 | 356.11 | 1.5E-01 | 352.56 | 1.4E-01 |
| 3 | 3S---- | B3run40-1 | 337.34 | 1.4E+00 | 345.19 | 1.1E+00 | 322.63 | 2.3E-01 | 327.62 | 2.5E-01 | 4.99 | 3.6E-01 | 4.45 | 8.5E-02 | 396.52 | 1.7E-01 | 400.95 | 1.5E-01 |
| 3 | 3S---- | B3run44-1 | 301.98 | 8.4E+00 | 301.51 | 8.6E+00 | 306.80 | 9.3E-01 | 308.84 | 1.0E+00 | 2.03 | 1.4E+00 | 2.45 | 1.0E+00 | 343.45 | 2.2E+00 | 340.42 | 2.1E+00 |
| 3 | 3S---- | B3run44-2 | 306.96 | 6.2E+00 | 306.62 | 8.7E+00 | 309.18 | 1.0E+00 | 311.14 | 1.0E+00 | 1.97 | 1.5E+00 | 2.34 | 1.9E+01 | 347.91 | 1.6E+00 | 344.97 | 2.0E+00 |
| 3 | 3S---- | B3run44-3 | 311.32 | 7.2E-01 | 311.06 | 7.3E-01 | 311.96 | 2.5E-01 | 313.86 | 2.5E-01 | 1.90 | 3.3E-01 | 2.31 | 3.9E-02 | 352.03 | 2.1E-01 | 349.16 | 1.8E-01 |
| 3 | 3S---- | B3run45-1 | 358.76 | 3.3E+01 | 299.35 | 3.2E+01 | 306.17 | 3.0E-01 | 305.75 | 2.6E-01 | -0.43 | 4.0E-01 | -0.11 | 1.5E-01 | 302.31 | 2.0E-01 | 301.88 | 1.9E-01 |

| Build | Exp Code | Run | Pcav (abs Pa) | Ts,C2,d,av (K) | +/- | Ts,C3,u,av (K) | +/- | TS,C3,d,av (K) | +/- | Ts,C4,u,av (K) | +/- | finTs,D2,av (K) | +/- | finTs,D3,av (K) | +/- |
|-------|----------|-----------|---------------|----------------|---------|----------------|---------|----------------|---------|----------------|---------|-----------------|---------|-----------------|---------|
| 3 | 3S---- | B3run2-1 | 2.424E+05 | 332.64 | 1.7E-01 | 331.84 | 2.2E-01 | 336.39 | 2.2E-01 | 337.41 | 2.0E-01 | 332.24 | 1.5E-01 | 336.90 | 1.5E-01 |
| 3 | 3S---- | B3run2-2 | 2.427E+05 | 333.22 | 1.3E-01 | 332.46 | 1.2E-01 | 337.05 | 1.3E-01 | 337.95 | 1.3E-01 | 332.84 | 9.1E-02 | 337.50 | 9.1E-02 |
| 3 | 3S---- | B3run3-1 | 2.387E+05 | 335.71 | 1.8E-01 | 334.49 | 1.7E-01 | 338.56 | 1.7E-01 | 339.26 | 1.7E-01 | 335.10 | 1.3E-01 | 338.91 | 1.2E-01 |
| 3 | 3S---- | B3run4-1 | 2.605E+05 | 333.74 | 2.1E-01 | 332.18 | 1.9E-01 | 334.04 | 1.6E-01 | 334.36 | 1.9E-01 | 332.96 | 1.4E-01 | 334.20 | 1.3E-01 |
| 3 | 3S---- | B3run4-2 | 2.606E+05 | 333.86 | 3.7E-01 | 332.38 | 4.4E-01 | 334.19 | 3.9E-01 | 334.50 | 3.6E-01 | 333.12 | 2.9E-01 | 334.34 | 2.6E-01 |
| 3 | 3S---- | B3run5-1 | 2.535E+05 | 330.49 | 1.7E-01 | 328.59 | 2.0E-01 | 332.55 | 2.1E-01 | 332.85 | 1.6E-01 | 329.54 | 1.4E-01 | 332.70 | 1.3E-01 |
| 3 | 3S---- | B3run5-3 | 2.529E+05 | 325.49 | 1.4E-01 | 323.70 | 1.5E-01 | 327.70 | 1.3E-01 | 327.97 | 1.3E-01 | 324.59 | 1.1E-01 | 327.84 | 8.8E-02 |
| 3 | 3S---- | B3run6-1 | 2.324E+05 | 329.08 | 1.3E-01 | 326.98 | 1.2E-01 | 331.07 | 1.5E-01 | 331.40 | 1.3E-01 | 328.03 | 9.3E-02 | 331.23 | 1.1E-01 |
| 3 | 3S---- | B3run6-2 | 2.322E+05 | 327.73 | 1.0E+00 | 326.18 | 1.2E+00 | 329.84 | 1.2E+00 | 330.07 | 1.2E+00 | 326.95 | 7.7E-01 | 329.96 | 8.5E-01 |
| 3 | 3S---- | B3run6-3 | 2.322E+05 | 329.56 | 2.1E-01 | 327.74 | 1.7E-01 | 331.49 | 2.1E-01 | 331.71 | 2.1E-01 | 328.65 | 1.5E-01 | 331.60 | 1.5E-01 |
| 3 | 3S---- | B3run7-1 | 2.440E+05 | 321.94 | 4.4E-01 | 320.88 | 2.3E-01 | 323.45 | 4.2E-01 | 323.97 | 4.9E-01 | 321.41 | 2.3E-01 | 323.71 | 3.2E-01 |
| 3 | 3S---- | B3run7-2 | 2.464E+05 | 337.47 | 2.2E-01 | 335.88 | 2.2E-01 | 338.57 | 2.0E-01 | 338.84 | 2.2E-01 | 336.67 | 1.7E-01 | 338.71 | 1.4E-01 |
| 3 | 3S---- | B3run7-4 | 2.459E+05 | 343.25 | 3.3E-01 | 341.53 | 2.2E-01 | 346.25 | 2.4E-01 | 346.81 | 3.1E-01 | 342.39 | 2.1E-01 | 346.53 | 2.1E-01 |
| 3 | 3S---- | B3run8-1 | 2.489E+05 | 342.55 | 1.1E-01 | 340.33 | 1.4E-01 | 345.67 | 1.4E-01 | 346.40 | 1.7E-01 | 341.44 | 9.0E-02 | 346.03 | 1.0E-01 |
| 3 | 3S---- | B3run9-1 | 2.725E+05 | 330.94 | 1.5E-01 | 333.59 | 1.5E-01 | 340.41 | 1.5E-01 | 339.30 | 1.5E-01 | 332.26 | 1.1E-01 | 339.86 | 9.8E-02 |
| 3 | 3S---- | B3run9-2 | 2.390E+05 | 335.48 | 1.2E-01 | 333.85 | 1.0E-01 | 336.26 | 1.1E-01 | 335.92 | 1.1E-01 | 334.67 | 8.2E-02 | 336.09 | 8.2E-02 |
| 3 | 3S---- | B3run9-3 | 2.392E+05 | 335.67 | 1.1E-01 | 334.05 | 9.2E-02 | 336.39 | 1.3E-01 | 336.06 | 1.1E-01 | 334.86 | 7.4E-02 | 336.22 | 8.3E-02 |
| 3 | 3S---- | B3run9-4 | 2.461E+05 | 334.14 | 1.4E-01 | 332.50 | 1.3E-01 | 333.12 | 1.1E-01 | 332.81 | 1.5E-01 | 333.32 | 9.9E-02 | 332.96 | 9.3E-02 |
| 3 | 3S---- | B3run12-1 | 2.256E+05 | 336.53 | 1.1E-01 | 335.47 | 1.3E-01 | 338.44 | 1.2E-01 | 338.53 | 1.2E-01 | 336.00 | 9.0E-02 | 338.49 | 8.3E-02 |
| 3 | 3S---- | B3run13-1 | 2.316E+05 | 333.58 | 1.6E-01 | 333.90 | 1.6E-01 | 339.75 | 1.4E-01 | 339.26 | 1.9E-01 | 333.74 | 1.2E-01 | 339.51 | 1.2E-01 |
| 3 | 3S---- | B3run13-2 | 2.318E+05 | 333.29 | 1.2E-01 | 333.70 | 1.2E-01 | 339.66 | 1.2E-01 | 339.07 | 1.3E-01 | 333.50 | 8.4E-02 | 339.36 | 8.3E-02 |
| 3 | 3S---- | B3run13-3 | 2.269E+05 | 324.95 | 1.2E-01 | 322.77 | 1.1E-01 | 326.77 | 1.0E-01 | 325.49 | 1.2E-01 | 323.86 | 7.3E-02 | 326.13 | 7.1E-02 |
| 3 | 3S---- | B3run15-1 | 2.569E+05 | 331.73 | 1.2E-01 | 330.49 | 1.1E-01 | 333.71 | 1.3E-01 | 333.92 | 1.3E-01 | 331.11 | 8.1E-02 | 333.81 | 9.6E-02 |
| 3 | 3S---- | B3run15-2 | 2.410E+05 | 327.24 | 1.1E-01 | 325.95 | 1.0E-01 | 329.05 | 1.1E-01 | 328.80 | 1.2E-01 | 326.59 | 7.7E-02 | 328.92 | 8.7E-02 |
| 3 | 3S---- | B3run16-1 | 2.498E+05 | 328.54 | 1.1E-01 | 327.01 | 1.3E-01 | 327.67 | 1.2E-01 | 327.43 | 1.2E-01 | 327.77 | 8.6E-02 | 327.55 | 8.6E-02 |

| Build | Exp Code | Run | Pcav (abs Pa) | Ts,C2,d,av (K) | +/- | Ts,C3,u,av (K) | +/- | TS,C3,d,av (K) | +/- | Ts,C4,u,av (K) | +/- | finTs,D2,av (K) | +/- | finTs,D3,av (K) | +/- |
|-------|----------|-----------|---------------|----------------|---------|----------------|---------|----------------|---------|----------------|---------|-----------------|---------|-----------------|---------|
| 3 | 3S---- | B3run16-2 | 2.504E+05 | 330.70 | 1.2E-01 | 328.48 | 1.4E-01 | 331.93 | 1.1E-01 | 330.75 | 1.4E-01 | 329.59 | 1.0E-01 | 331.34 | 9.7E-02 |
| 3 | 3S---- | B3run17-1 | 2.576E+05 | 326.92 | 1.3E-01 | 325.20 | 1.3E-01 | 326.77 | 1.1E-01 | 326.29 | 1.4E-01 | 326.06 | 9.9E-02 | 326.53 | 8.7E-02 |
| 3 | 3S---- | B3run17-2 | 2.578E+05 | 327.06 | 1.2E-01 | 325.34 | 1.4E-01 | 326.93 | 1.3E-01 | 326.50 | 1.3E-01 | 326.20 | 9.0E-02 | 326.71 | 9.7E-02 |
| 3 | 3S---- | B3run17-3 | 2.574E+05 | 327.01 | 1.3E-01 | 324.52 | 1.5E-01 | 325.99 | 1.2E-01 | 324.50 | 1.5E-01 | 325.77 | 1.0E-01 | 325.24 | 9.7E-02 |
| 3 | 3S---- | B3run17-4 | 2.573E+05 | 327.14 | 1.3E-01 | 324.65 | 1.3E-01 | 326.14 | 1.1E-01 | 324.65 | 1.2E-01 | 325.90 | 8.6E-02 | 325.39 | 9.0E-02 |
| 3 | 3S---- | B3run17-5 | 2.335E+05 | 333.88 | 1.1E-01 | 332.57 | 1.2E-01 | 336.19 | 1.3E-01 | 336.56 | 1.1E-01 | 333.22 | 8.7E-02 | 336.37 | 7.6E-02 |
| 3 | 3S---- | B3run17-6 | 2.341E+05 | 333.95 | 1.2E-01 | 332.67 | 1.3E-01 | 336.23 | 1.2E-01 | 336.56 | 1.3E-01 | 333.31 | 9.2E-02 | 336.39 | 8.4E-02 |
| 3 | 3S---- | B3run18-1 | 2.540E+05 | 337.91 | 1.4E-01 | 335.88 | 1.6E-01 | 341.41 | 1.6E-01 | 342.31 | 1.4E-01 | 336.90 | 1.1E-01 | 341.86 | 1.1E-01 |
| 3 | 3S---- | B3run18-2 | 2.548E+05 | 337.78 | 1.3E-01 | 336.44 | 1.4E-01 | 339.06 | 1.3E-01 | 339.03 | 1.5E-01 | 337.11 | 9.5E-02 | 339.05 | 1.0E-01 |
| 3 | 3S---- | B3run18-3 | 2.396E+05 | 334.11 | 1.2E-01 | 332.28 | 1.1E-01 | 335.84 | 1.2E-01 | 336.27 | 1.2E-01 | 333.20 | 8.3E-02 | 336.05 | 9.0E-02 |
| 3 | 3S---- | B3run19-1 | 2.618E+05 | 335.90 | 5.8E-01 | 333.81 | 3.4E-01 | 337.52 | 5.3E-01 | 338.09 | 5.0E-01 | 334.86 | 3.4E-01 | 337.80 | 3.2E-01 |
| 3 | 3S---- | B3run20-1 | 2.836E+05 | 334.06 | 2.0E-01 | 333.03 | 2.1E-01 | 335.67 | 2.0E-01 | 335.68 | 1.9E-01 | 333.55 | 1.5E-01 | 335.67 | 1.3E-01 |
| 3 | 3S---- | B3run21-1 | 2.884E+05 | 331.05 | 1.3E-01 | 329.91 | 1.3E-01 | 331.96 | 1.4E-01 | 331.91 | 1.4E-01 | 330.48 | 9.7E-02 | 331.94 | 1.0E-01 |
| 3 | 3S---- | B3run22-1 | 2.872E+05 | 329.65 | 1.0E-01 | 328.55 | 1.1E-01 | 331.54 | 1.1E-01 | 331.84 | 1.2E-01 | 329.10 | 7.8E-02 | 331.69 | 8.8E-02 |
| 3 | 3S---- | B3run22-2 | 2.857E+05 | 333.04 | 1.9E-01 | 331.29 | 1.9E-01 | 332.04 | 1.9E-01 | 331.41 | 1.9E-01 | 332.16 | 1.3E-01 | 331.73 | 1.2E-01 |
| 3 | 3S---- | B3run23-1 | 2.526E+05 | 330.79 | 1.1E-01 | 329.26 | 1.2E-01 | 329.22 | 1.2E-01 | 328.68 | 1.1E-01 | 330.03 | 8.1E-02 | 328.95 | 7.8E-02 |
| 3 | 3S---- | B3run24-1 | 2.940E+05 | 321.77 | 1.3E-01 | 320.91 | 1.3E-01 | 323.13 | 1.4E-01 | 323.17 | 1.3E-01 | 321.34 | 9.1E-02 | 323.15 | 9.2E-02 |
| 3 | 3S---- | B3run24-2 | 3.002E+05 | 354.15 | 1.8E-01 | 352.29 | 1.6E-01 | 356.28 | 1.8E-01 | 356.22 | 1.7E-01 | 353.22 | 1.2E-01 | 356.25 | 1.3E-01 |
| 3 | 3S---- | B3run25-1 | 2.714E+05 | 341.23 | 1.3E-01 | 340.18 | 1.2E-01 | 344.21 | 1.2E-01 | 344.48 | 1.2E-01 | 340.70 | 9.6E-02 | 344.35 | 8.6E-02 |
| 3 | 3S---- | B3run26-1 | 3.082E+05 | 338.82 | 1.3E-01 | 336.49 | 1.3E-01 | 339.52 | 1.2E-01 | 338.29 | 1.1E-01 | 337.65 | 8.6E-02 | 338.91 | 8.5E-02 |
| 3 | 3S---- | B3run27-1 | 2.425E+05 | 329.90 | 1.1E-01 | 327.47 | 1.2E-01 | 328.82 | 1.2E-01 | 327.71 | 9.9E-02 | 328.69 | 8.8E-02 | 328.26 | 8.3E-02 |
| 3 | 3S---- | B3run28-1 | 3.094E+05 | 325.12 | 1.4E-01 | 321.97 | 1.2E-01 | 324.19 | 1.4E-01 | 322.26 | 1.6E-01 | 323.54 | 8.9E-02 | 323.22 | 1.1E-01 |
| 3 | 3S---- | B3run28-2 | 3.113E+05 | 330.20 | 1.2E-01 | 327.11 | 1.1E-01 | 330.64 | 1.1E-01 | 328.68 | 1.2E-01 | 328.65 | 8.5E-02 | 329.66 | 7.7E-02 |
| 3 | 3S---- | B3run30-1 | 2.540E+05 | 321.84 | 1.5E-01 | 320.42 | 1.3E-01 | 320.38 | 1.3E-01 | 319.32 | 1.4E-01 | 321.13 | 1.0E-01 | 319.85 | 9.5E-02 |
| 3 | 3S---- | B3run31-1 | 2.603E+05 | 333.26 | 1.4E-01 | 332.79 | 1.5E-01 | 339.21 | 1.4E-01 | 338.94 | 1.4E-01 | 333.02 | 1.1E-01 | 339.08 | 1.0E-01 |
| 3 | 3S---- | B3run32-1 | 2.453E+05 | 333.50 | 1.3E-01 | 333.65 | 1.3E-01 | 340.66 | 1.4E-01 | 341.55 | 1.2E-01 | 333.57 | 9.3E-02 | 341.11 | 9.2E-02 |
| 3 | 3S---- | B3run33-1 | 2.512E+05 | 325.59 | 1.3E-01 | 324.59 | 1.4E-01 | 327.53 | 1.3E-01 | 327.84 | 1.5E-01 | 325.09 | 9.8E-02 | 327.69 | 1.0E-01 |

| Build | Exp Code | Run | Pcav (abs Pa) | Ts,C2,d,av (K) | +/- | Ts,C3,u,av (K) | +/- | TS,C3,d,av (K) | +/- | Ts,C4,u,av (K) | +/- | finTs,D2,av (K) | +/- | finTs,D3,av (K) | +/- |
|-------|----------|------------|---------------|----------------|---------|----------------|---------|----------------|---------|----------------|---------|-----------------|---------|-----------------|---------|
| 3 | 3S---- | B3run33-2 | 2.524E+05 | 322.27 | 1.3E-01 | 320.34 | 1.1E-01 | 321.45 | 1.2E-01 | 320.63 | 1.3E-01 | 321.30 | 8.8E-02 | 321.04 | 8.7E-02 |
| 3 | 3S---- | B3run33-3 | 2.606E+05 | 327.85 | 1.4E-01 | 326.31 | 1.2E-01 | 328.56 | 1.2E-01 | 328.56 | 1.4E-01 | 327.08 | 9.5E-02 | 328.56 | 9.4E-02 |
| 3 | 3S---- | B3run33-4 | 2.501E+05 | 329.98 | 1.1E-01 | 328.34 | 1.2E-01 | 332.68 | 1.2E-01 | 332.48 | 1.2E-01 | 329.16 | 8.0E-02 | 332.58 | 9.1E-02 |
| 3 | 3S---- | B3run33-5 | 3.385E+05 | 326.94 | 1.3E-01 | 323.91 | 1.4E-01 | 326.49 | 1.3E-01 | 324.54 | 1.1E-01 | 325.43 | 1.1E-01 | 325.51 | 8.9E-02 |
| 3 | 3S---- | B3run33-6 | 3.203E+05 | 331.80 | 1.3E-01 | 329.16 | 1.8E-01 | 334.14 | 1.4E-01 | 332.57 | 1.4E-01 | 330.48 | 1.1E-01 | 333.36 | 1.1E-01 |
| 3 | 3S---- | B3run33-7 | 2.509E+05 | 333.62 | 1.2E-01 | 332.32 | 1.3E-01 | 334.04 | 1.6E-01 | 334.27 | 1.3E-01 | 332.97 | 9.0E-02 | 334.16 | 1.2E-01 |
| 3 | 3S---- | B3run33-8 | 2.569E+05 | 332.29 | 1.1E-01 | 330.16 | 1.2E-01 | 331.15 | 1.3E-01 | 330.45 | 1.3E-01 | 331.23 | 8.4E-02 | 330.80 | 9.5E-02 |
| 3 | 3S---- | B3run33-9 | 2.412E+05 | 333.58 | 1.2E-01 | 331.72 | 1.2E-01 | 334.88 | 1.2E-01 | 334.03 | 1.2E-01 | 332.65 | 8.0E-02 | 334.45 | 8.4E-02 |
| 3 | 3S---- | B3run33-10 | 3.692E+05 | 331.38 | 1.4E-01 | 328.35 | 1.4E-01 | 331.57 | 1.3E-01 | 329.76 | 1.3E-01 | 329.86 | 1.0E-01 | 330.66 | 8.6E-02 |
| 3 | 3S---- | B3run34-1 | 2.648E+05 | 330.02 | 1.2E-01 | 328.50 | 1.5E-01 | 330.86 | 1.3E-01 | 330.82 | 1.5E-01 | 329.26 | 1.0E-01 | 330.84 | 9.6E-02 |
| 3 | 3S---- | B3run34-2 | 3.061E+05 | 326.72 | 1.2E-01 | 323.84 | 1.3E-01 | 327.22 | 1.3E-01 | 325.50 | 1.6E-01 | 325.28 | 8.4E-02 | 326.36 | 1.0E-01 |
| 3 | 3S---- | B3run34-3 | 2.954E+05 | 327.23 | 1.2E-01 | 325.26 | 1.2E-01 | 326.71 | 1.1E-01 | 326.02 | 1.1E-01 | 326.24 | 8.8E-02 | 326.37 | 8.1E-02 |
| 3 | 3S---- | B3run34-4 | 2.950E+05 | 331.23 | 1.0E-01 | 329.93 | 1.3E-01 | 331.38 | 1.4E-01 | 331.39 | 1.3E-01 | 330.58 | 9.2E-02 | 331.38 | 9.6E-02 |
| 3 | 3S---- | B3run34-5 | 2.731E+05 | 329.82 | 1.2E-01 | 327.71 | 1.2E-01 | 329.00 | 1.0E-01 | 328.32 | 1.2E-01 | 328.76 | 8.5E-02 | 328.66 | 8.2E-02 |
| 3 | 3S---- | B3run34-6 | 3.118E+05 | 331.69 | 1.1E-01 | 328.77 | 1.1E-01 | 331.33 | 1.1E-01 | 329.68 | 1.2E-01 | 330.23 | 8.0E-02 | 330.51 | 8.8E-02 |
| 3 | 3S---- | B3run34-7 | 3.184E+05 | 331.23 | 1.3E-01 | 328.91 | 1.4E-01 | 333.50 | 1.1E-01 | 332.01 | 1.1E-01 | 330.07 | 9.5E-02 | 332.76 | 7.4E-02 |
| 3 | 3S---- | B3run34-8 | 2.757E+05 | 334.36 | 1.2E-01 | 332.11 | 1.6E-01 | 333.42 | 1.3E-01 | 332.69 | 1.5E-01 | 333.23 | 1.1E-01 | 333.06 | 9.8E-02 |
| 3 | 3S---- | B3run34-9 | 2.750E+05 | 336.02 | 1.6E-01 | 334.21 | 1.8E-01 | 338.27 | 1.8E-01 | 338.36 | 1.5E-01 | 335.12 | 1.2E-01 | 338.32 | 1.2E-01 |
| 3 | 3S---- | B3run35-1 | 2.599E+05 | 333.87 | 1.2E-01 | 332.58 | 1.3E-01 | 334.24 | 1.1E-01 | 334.37 | 1.6E-01 | 333.22 | 8.5E-02 | 334.30 | 9.3E-02 |
| 3 | 3S---- | B3run35-2 | 2.818E+05 | 331.83 | 9.9E-02 | 329.97 | 1.2E-01 | 329.97 | 1.2E-01 | 329.09 | 1.3E-01 | 330.90 | 7.7E-02 | 329.53 | 9.7E-02 |
| 3 | 3S---- | B3run35-3 | 2.832E+05 | 332.73 | 1.2E-01 | 330.56 | 1.1E-01 | 331.69 | 1.1E-01 | 331.04 | 1.1E-01 | 331.65 | 8.7E-02 | 331.37 | 7.8E-02 |
| 3 | 3S---- | B3run35-4 | 3.433E+05 | 333.77 | 1.0E-01 | 330.76 | 1.2E-01 | 332.30 | 1.3E-01 | 330.42 | 1.2E-01 | 332.27 | 8.3E-02 | 331.36 | 8.7E-02 |
| 3 | 3S---- | B3run35-5 | 2.640E+05 | 339.03 | 1.1E-01 | 337.03 | 1.1E-01 | 339.20 | 1.5E-01 | 338.07 | 1.3E-01 | 338.03 | 8.3E-02 | 338.63 | 9.5E-02 |
| 3 | 3S---- | B3run35-6 | 3.365E+05 | 335.72 | 1.2E-01 | 332.85 | 1.3E-01 | 334.77 | 1.1E-01 | 332.98 | 1.2E-01 | 334.29 | 7.9E-02 | 333.87 | 7.9E-02 |
| 3 | 3S---- | B3run35-7 | 2.251E+05 | 343.68 | 1.2E-01 | 341.76 | 1.3E-01 | 344.56 | 1.3E-01 | 344.86 | 1.3E-01 | 342.72 | 9.2E-02 | 344.71 | 9.6E-02 |
| 3 | 3S---- | B3run35-8 | 2.278E+05 | 342.54 | 1.1E-01 | 340.86 | 1.4E-01 | 343.55 | 1.2E-01 | 343.20 | 1.3E-01 | 341.70 | 9.1E-02 | 343.37 | 9.0E-02 |
| 3 | 3S---- | B3run35-9 | 3.154E+05 | 339.25 | 1.4E-01 | 336.96 | 1.5E-01 | 338.81 | 1.3E-01 | 337.19 | 1.0E-01 | 338.10 | 1.0E-01 | 338.00 | 8.6E-02 |

| Build | Exp Code | Run | Pcav (abs Pa) | Ts,C2,d,av (K) | +/- | Ts,C3,u,av (K) | +/- | TS,C3,d,av (K) | +/- | Ts,C4,u,av (K) | +/- | finTs,D2,av (K) | +/- | finTs,D3,av (K) | +/- |
|-------|----------|-----------|---------------|----------------|---------|----------------|---------|----------------|---------|----------------|---------|-----------------|---------|-----------------|---------|
| 3 | 3S---- | B3run36-1 | 2.554E+05 | 340.73 | 1.3E-01 | 339.70 | 1.3E-01 | 342.19 | 1.3E-01 | 342.79 | 1.4E-01 | 340.22 | 9.1E-02 | 342.49 | 9.7E-02 |
| 3 | 3S---- | B3run37-1 | 2.216E+05 | 320.28 | 1.3E+00 | 319.22 | 9.5E-01 | 319.69 | 1.4E+00 | 319.12 | 1.2E+00 | 319.75 | 8.2E-01 | 319.40 | 9.1E-01 |
| 3 | 3S---- | B3run38-1 | 2.232E+05 | 346.72 | 1.4E-01 | 345.94 | 1.5E-01 | 352.02 | 1.3E-01 | 352.92 | 1.3E-01 | 346.33 | 1.0E-01 | 352.47 | 9.6E-02 |
| 3 | 3S---- | B3run39-1 | 2.220E+05 | 326.22 | 1.1E-01 | 324.74 | 1.1E-01 | 325.57 | 1.0E-01 | 325.01 | 1.1E-01 | 325.48 | 7.8E-02 | 325.29 | 8.3E-02 |
| 3 | 3S---- | B3run40-1 | 2.274E+05 | 353.32 | 1.3E-01 | 354.01 | 1.5E-01 | 359.91 | 1.4E-01 | 360.99 | 1.4E-01 | 353.67 | 9.5E-02 | 360.45 | 9.7E-02 |
| 3 | 3S---- | B3run44-1 | 2.340E+05 | 308.79 | 1.5E+00 | 308.25 | 1.1E+00 | 307.79 | 1.6E+00 | 307.34 | 1.4E+00 | 308.52 | 9.4E-01 | 307.57 | 1.0E+00 |
| 3 | 3S---- | B3run44-2 | 2.351E+05 | 313.69 | 1.5E+00 | 313.02 | 1.1E+00 | 312.41 | 1.4E+00 | 312.04 | 1.3E+00 | 313.36 | 9.2E-01 | 312.23 | 9.5E-01 |
| 3 | 3S---- | B3run44-3 | 2.347E+05 | 317.90 | 1.4E-01 | 316.79 | 1.5E-01 | 316.65 | 1.6E-01 | 316.33 | 1.1E-01 | 317.35 | 1.1E-01 | 316.49 | 9.4E-02 |
| 3 | 3S---- | B3run45-1 | 2.229E+05 | 303.21 | 1.4E-01 | 303.18 | 1.3E-01 | 302.98 | 1.2E-01 | 302.72 | 1.3E-01 | 303.20 | 9.3E-02 | 302.85 | 8.7E-02 |

| Build | Exp Code | Run | Nu,C2,d,av +/- | | Nu,C3,u,av +/- | | Nu,C3,d,av +/- | | Nu,C4,u,av +/- | | Nush,C2 | +/- | Nush,C3 | +/- | finNu,D2,av +/- | | finNu,D3,av +/- | |
|-------|----------|-----------|----------------|---------|----------------|---------|----------------|---------|----------------|--------|---------|-----|---------|---------|-----------------|---------|-----------------|---------|
| 3 | 3S----- | B3run2-1 | -223.3 | 1.2E+02 | 590.5 | 1.4E+02 | 520.2 | 1.2E+02 | -2.5E+02 | 105.5 | 5.5E+01 | 2 | 42.5 | 1.7E+00 | 148.0 | 1.3E+01 | 116.8 | 1.2E+01 |
| 3 | 3S----- | B3run2-2 | -204.2 | 8.2E+01 | 563.7 | 9.1E+01 | 471.1 | 7.1E+01 | -2.1E+02 | 64.2 | 5.4E+01 | 2 | 42.4 | 1.1E+00 | 145.4 | 1.1E+01 | 115.7 | 7.7E+00 |
| 3 | 3S----- | B3run3-1 | -409.4 | 1.1E+02 | 807.3 | 1.3E+02 | 429.0 | 9.3E+01 | -1.3E+02 | 86.1 | 5.4E+01 | 2 | 43.6 | 1.1E+00 | 149.8 | 1.4E+01 | 135.2 | 9.8E+00 |
| 3 | 3S----- | B3run4-1 | -469.2 | 1.1E+02 | 776.7 | 1.4E+02 | 266.5 | 9.4E+01 | 8.2E+00 | 88.2 | 5.1E+01 | 2 | 53.0 | 1.8E+00 | 112.9 | 1.1E+01 | 127.0 | 1.2E+01 |
| 3 | 3S----- | B3run4-2 | -437.5 | 2.1E+02 | 732.4 | 2.5E+02 | 254.2 | 2.1E+02 | 1.4E+01 | 199.0 | 5.0E+01 | 4 | 52.4 | 3.9E+00 | 110.0 | 2.6E+01 | 123.9 | 2.0E+01 |
| 3 | 3S----- | B3run5-1 | -703.5 | 1.1E+02 | 1191.0 | 1.4E+02 | 300.1 | 1.2E+02 | 3.4E+01 | 104.7 | 5.5E+01 | 2 | 52.7 | 1.8E+00 | 161.7 | 1.3E+01 | 157.6 | 1.4E+01 |
| 3 | 3S----- | B3run5-3 | -613.1 | 7.8E+01 | 1076.9 | 1.0E+02 | 276.8 | 7.6E+01 | 5.6E+01 | 73.9 | 5.6E+01 | 1 | 53.4 | 1.5E+00 | 163.7 | 1.3E+01 | 157.1 | 1.0E+01 |
| 3 | 3S----- | B3run6-1 | -742.6 | 6.9E+01 | 1233.4 | 8.9E+01 | 301.3 | 7.6E+01 | 2.8E+01 | 71.7 | 5.4E+01 | 1 | 52.8 | 1.4E+00 | 159.1 | 1.0E+01 | 154.8 | 8.8E+00 |
| 3 | 3S----- | B3run6-2 | -557.0 | 1.2E+04 | 1017.0 | 3.4E+03 | 282.0 | 1.3E+03 | 6.7E+01 | 2185.0 | 5.5E+01 | 10 | 53.9 | 1.1E+01 | 166.5 | 2.3E+02 | 164.6 | 6.9E+01 |
| 3 | 3S----- | B3run6-3 | -648.5 | 1.1E+02 | 1118.3 | 1.4E+02 | 271.4 | 1.2E+02 | 7.0E+01 | 112.6 | 5.4E+01 | 2 | 52.3 | 1.6E+00 | 161.1 | 1.6E+01 | 161.1 | 1.2E+01 |
| 3 | 3S----- | B3run7-1 | -281.8 | 2.3E+02 | 577.9 | 2.2E+02 | 326.4 | 2.4E+02 | -5.5E+01 | 239.6 | 5.7E+01 | 5 | 58.3 | 3.7E+00 | 120.7 | 2.9E+01 | 125.3 | 2.5E+01 |
| 3 | 3S----- | B3run7-2 | -477.1 | 1.0E+02 | 788.2 | 1.3E+02 | 231.6 | 1.1E+02 | 2.6E+01 | 103.1 | 5.0E+01 | 2 | 52.6 | 1.9E+00 | 112.2 | 1.3E+01 | 120.0 | 1.2E+01 |
| 3 | 3S----- | B3run7-4 | -449.2 | 1.2E+02 | 754.1 | 1.4E+02 | 264.8 | 1.1E+02 | -6.2E+01 | 103.5 | 3.4E+01 | 2 | 34.1 | 1.9E+00 | 110.0 | 1.6E+01 | 94.4 | 1.4E+01 |
| 3 | 3S----- | B3run8-1 | -519.2 | 4.9E+01 | 818.5 | 6.0E+01 | 281.5 | 5.9E+01 | -9.5E+01 | 57.3 | 3.2E+01 | 1 | 32.3 | 1.1E+00 | 103.0 | 8.0E+00 | 86.7 | 7.5E+00 |
| 3 | 3S----- | B3run9-1 | 1400.8 | 1.1E+02 | -857.6 | 7.8E+01 | -208.0 | 6.9E+01 | 4.5E+02 | 74.1 | 3.3E+01 | 1 | 32.8 | 1.2E+00 | 157.7 | 1.2E+01 | 108.1 | 8.5E+00 |
| 3 | 3S----- | B3run9-2 | -510.2 | 5.8E+01 | 815.2 | 7.2E+01 | -0.6 | 5.6E+01 | 2.3E+02 | 58.4 | 6.0E+01 | 1 | 64.6 | 1.3E+00 | 108.2 | 7.4E+00 | 105.6 | 7.8E+00 |
| 3 | 3S----- | B3run9-3 | -506.3 | 5.3E+01 | 811.1 | 6.5E+01 | 1.0 | 6.7E+01 | 2.3E+02 | 67.8 | 6.1E+01 | 1 | 65.0 | 1.2E+00 | 108.5 | 7.4E+00 | 105.7 | 7.2E+00 |
| 3 | 3S----- | B3run9-4 | -590.4 | 7.6E+01 | 967.6 | 9.8E+01 | 29.1 | 8.8E+01 | 2.9E+02 | 93.4 | 6.2E+01 | 2 | 68.3 | 1.5E+00 | 133.1 | 1.2E+01 | 144.1 | 1.0E+01 |
| 3 | 3S----- | B3run12-1 | -267.1 | 6.4E+01 | 590.6 | 7.5E+01 | 177.8 | 6.0E+01 | 9.2E+01 | 59.5 | 5.1E+01 | 1 | 51.0 | 1.0E+00 | 131.4 | 8.2E+00 | 126.1 | 7.8E+00 |
| 3 | 3S----- | B3run13-1 | 343.0 | 1.3E+02 | 68.4 | 1.1E+02 | -43.8 | 8.2E+01 | 3.0E+02 | 84.8 | 5.6E+01 | 2 | 45.9 | 1.2E+00 | 174.0 | 1.5E+01 | 121.3 | 9.9E+00 |
| 3 | 3S----- | B3run13-2 | 388.7 | 1.0E+02 | 20.6 | 8.9E+01 | -79.9 | 6.7E+01 | 3.3E+02 | 69.6 | 5.6E+01 | 2 | 45.8 | 1.0E+00 | 172.0 | 1.2E+01 | 118.6 | 7.8E+00 |
| 3 | 3S----- | B3run13-3 | -1051.5 | 9.6E+01 | 1599.5 | 1.4E+02 | -487.8 | 8.2E+01 | 7.8E+02 | 94.2 | 1.1E+02 | 2 | 109.3 | 1.7E+00 | 136.6 | 1.3E+01 | 112.9 | 9.4E+00 |
| 3 | 3S----- | B3run15-1 | -393.4 | 7.4E+01 | 777.2 | 8.7E+01 | 244.3 | 6.6E+01 | 6.1E+01 | 64.3 | 5.9E+01 | 2 | 53.9 | 9.9E-01 | 149.0 | 9.1E+00 | 143.4 | 8.6E+00 |
| 3 | 3S----- | B3run15-2 | -470.1 | 6.7E+01 | 781.0 | 8.0E+01 | 4.7 | 6.7E+01 | 2.0E+02 | 67.7 | 8.2E+01 | 1 | 73.9 | 1.5E+00 | 113.9 | 8.6E+00 | 95.9 | 8.0E+00 |
| 3 | 3S----- | B3run16-1 | -711.9 | 9.2E+01 | 1166.8 | 1.2E+02 | 51.2 | 9.8E+01 | 3.1E+02 | 103.1 | 7.8E+01 | 1 | 79.8 | 1.6E+00 | 152.1 | 1.2E+01 | 164.6 | 1.2E+01 |

| Build | Exp Code | Run | Nu,C2,d,av +/- | | Nu,C3,u,av +/- | | Nu,C3,d,av +/- | | Nu,C4,u,av +/- | | Nush,C2 +/- | | Nush,C3 +/- | | finNu,D2,av +/- | | finNu,D3,av +/- | |
|-------|----------|-----------|----------------|---------|----------------|---------|----------------|---------|----------------|-------|-------------|---|-------------|---------|-----------------|---------|-----------------|---------|
| | 3 S----- | B3run16-2 | -1196.3 | 9.7E+01 | 1838.4 | 1.6E+02 | -508.7 | 9.2E+01 | 8.3E+02 | 115.1 | 1.1E+02 | 2 | 108.6 | 2.0E+00 | 143.6 | 1.4E+01 | 125.1 | 1.3E+01 |
| | 3 S----- | B3run17-1 | -907.4 | 9.9E+01 | 1524.9 | 1.4E+02 | -88.4 | 1.2E+02 | 5.2E+02 | 125.6 | 7.9E+01 | 1 | 81.9 | 1.4E+00 | 188.6 | 1.3E+01 | 194.9 | 1.2E+01 |
| | 3 S----- | B3run17-2 | -912.1 | 1.2E+02 | 1534.9 | 1.8E+02 | -55.8 | 1.2E+02 | 4.9E+02 | 133.6 | 7.9E+01 | 2 | 81.6 | 1.8E+00 | 189.8 | 1.4E+01 | 194.8 | 1.3E+01 |
| | 3 S----- | B3run17-3 | -1634.5 | 1.3E+02 | 2651.4 | 2.4E+02 | -960.3 | 1.5E+02 | 1.6E+03 | 227.3 | 1.1E+02 | 2 | 118.0 | 2.0E+00 | 182.1 | 1.8E+01 | 211.4 | 2.1E+01 |
| | 3 S----- | B3run17-4 | -1639.7 | 1.4E+02 | 2666.8 | 2.8E+02 | -964.6 | 1.1E+02 | 1.7E+03 | 180.6 | 1.1E+02 | 2 | 118.4 | 2.3E+00 | 184.1 | 1.8E+01 | 212.7 | 2.1E+01 |
| | 3 S----- | B3run17-5 | -353.0 | 5.3E+01 | 639.7 | 6.1E+01 | 239.1 | 6.0E+01 | -1.5E+01 | 56.5 | 5.1E+01 | 1 | 54.1 | 9.9E-01 | 110.5 | 6.8E+00 | 104.9 | 6.7E+00 |
| | 3 S----- | B3run17-6 | -342.0 | 6.0E+01 | 625.3 | 6.7E+01 | 223.6 | 6.0E+01 | 7.4E-01 | 58.3 | 5.1E+01 | 1 | 53.9 | 1.2E+00 | 109.7 | 7.1E+00 | 105.1 | 7.1E+00 |
| | 3 S----- | B3run18-1 | -509.0 | 6.0E+01 | 841.5 | 7.4E+01 | 354.1 | 5.4E+01 | -1.3E+02 | 50.4 | 3.5E+01 | 1 | 32.9 | 1.1E+00 | 118.1 | 7.7E+00 | 102.3 | 6.8E+00 |
| | 3 S----- | B3run18-2 | -467.1 | 8.0E+01 | 804.1 | 1.0E+02 | 120.4 | 8.1E+01 | 1.3E+02 | 82.7 | 6.2E+01 | 2 | 66.4 | 1.4E+00 | 123.4 | 1.1E+01 | 116.3 | 8.2E+00 |
| | 3 S----- | B3run18-3 | -698.3 | 7.0E+01 | 1215.5 | 9.1E+01 | 364.5 | 7.6E+01 | -1.8E+01 | 71.7 | 5.5E+01 | 1 | 54.3 | 1.4E+00 | 173.6 | 1.0E+01 | 161.4 | 1.0E+01 |
| | 3 S----- | B3run19-1 | -797.6 | 3.5E+02 | 1322.2 | 3.8E+02 | 410.7 | 3.6E+02 | -7.6E+01 | 394.9 | 5.5E+01 | 6 | 51.4 | 5.5E+00 | 164.2 | 4.9E+01 | 154.9 | 4.0E+01 |
| | 3 S----- | B3run20-1 | -343.7 | 1.4E+02 | 694.5 | 1.7E+02 | 157.5 | 1.2E+02 | 1.3E+02 | 119.0 | 6.0E+01 | 2 | 55.0 | 1.5E+00 | 138.5 | 1.6E+01 | 134.7 | 1.4E+01 |
| | 3 S----- | B3run21-1 | -445.4 | 8.7E+01 | 803.1 | 1.1E+02 | 129.9 | 9.6E+01 | 1.6E+02 | 95.9 | 5.8E+01 | 2 | 52.4 | 1.5E+00 | 135.6 | 1.2E+01 | 135.2 | 1.1E+01 |
| | 3 S----- | B3run22-1 | -336.2 | 5.9E+01 | 623.7 | 6.9E+01 | 237.6 | 6.3E+01 | -3.7E+00 | 60.0 | 5.9E+01 | 1 | 57.6 | 1.3E+00 | 112.7 | 8.9E+00 | 109.7 | 7.5E+00 |
| | 3 S----- | B3run22-2 | -937.2 | 1.6E+02 | 1462.7 | 2.4E+02 | -236.5 | 2.0E+02 | 6.1E+02 | 230.9 | 8.1E+01 | 3 | 84.4 | 2.4E+00 | 148.2 | 2.4E+01 | 160.2 | 2.3E+01 |
| | 3 S----- | B3run23-1 | -741.3 | 9.0E+01 | 1198.8 | 1.2E+02 | -157.9 | 1.2E+02 | 5.5E+02 | 128.9 | 7.5E+01 | 2 | 80.0 | 2.0E+00 | 151.0 | 1.2E+01 | 170.2 | 1.5E+01 |
| | 3 S----- | B3run24-1 | -289.1 | 9.7E+01 | 645.2 | 1.2E+02 | 187.3 | 9.3E+01 | 1.2E+02 | 91.7 | 6.6E+01 | 2 | 62.9 | 1.6E+00 | 146.3 | 1.3E+01 | 147.0 | 1.3E+01 |
| | 3 S----- | B3run24-2 | -708.3 | 1.1E+02 | 1129.4 | 1.4E+02 | 113.0 | 9.6E+01 | 1.5E+02 | 95.1 | 4.0E+01 | 2 | 40.0 | 1.9E+00 | 128.1 | 1.4E+01 | 123.2 | 1.3E+01 |
| | 3 S----- | B3run25-1 | -381.6 | 8.5E+01 | 831.4 | 1.0E+02 | 278.4 | 7.7E+01 | 2.5E+01 | 72.2 | 5.5E+01 | 2 | 47.2 | 1.4E+00 | 172.1 | 1.2E+01 | 141.8 | 1.1E+01 |
| | 3 S----- | B3run26-1 | -1034.9 | 8.9E+01 | 1500.3 | 1.3E+02 | -467.8 | 6.7E+01 | 7.2E+02 | 80.0 | 7.3E+01 | 1 | 77.9 | 1.8E+00 | 105.9 | 1.0E+01 | 97.5 | 1.0E+01 |
| | 3 S----- | B3run27-1 | -1483.0 | 9.6E+01 | 2403.4 | 1.9E+02 | -609.2 | 1.1E+02 | 1.2E+03 | 139.8 | 7.3E+01 | 2 | 82.1 | 1.8E+00 | 186.7 | 1.8E+01 | 211.9 | 1.9E+01 |
| | 3 S----- | B3run28-1 | -2103.6 | 1.3E+02 | 3453.3 | 2.8E+02 | -1331.1 | 1.5E+02 | 2.1E+03 | 257.2 | 1.0E+02 | 3 | 113.2 | 3.0E+00 | 152.0 | 1.9E+01 | 168.5 | 1.9E+01 |
| | 3 S----- | B3run28-2 | -1557.5 | 8.0E+01 | 2286.4 | 1.3E+02 | -919.4 | 8.9E+01 | 1.3E+03 | 126.5 | 9.0E+01 | 2 | 98.8 | 1.9E+00 | 101.0 | 1.4E+01 | 90.9 | 1.1E+01 |
| | 3 S----- | B3run30-1 | -1114.4 | 1.6E+02 | 1808.6 | 2.8E+02 | -943.6 | 2.3E+02 | 1.6E+03 | 333.3 | 1.2E+02 | 4 | 110.2 | 4.7E+00 | 189.9 | 2.3E+01 | 205.2 | 2.6E+01 |
| | 3 S----- | B3run31-1 | -34.8 | 1.0E+02 | 556.7 | 1.1E+02 | 85.0 | 7.5E+01 | 2.8E+02 | 75.6 | 5.4E+01 | 1 | 32.3 | 8.7E-01 | 218.7 | 1.1E+01 | 175.0 | 8.0E+00 |
| | 3 S----- | B3run32-1 | 166.7 | 5.6E+01 | 108.7 | 5.5E+01 | 299.5 | 4.5E+01 | -1.4E+02 | 42.3 | 3.9E+01 | 1 | 31.8 | 7.5E-01 | 117.8 | 7.4E+00 | 75.6 | 5.8E+00 |
| | 3 S----- | B3run33-1 | -265.7 | 7.3E+01 | 557.4 | 8.4E+01 | 224.9 | 6.6E+01 | -6.0E+00 | 62.6 | 4.9E+01 | 1 | 41.2 | 1.6E+00 | 120.0 | 8.6E+00 | 102.1 | 8.1E+00 |

| Build | Exp Code | Run | Nu,C2,d,av +/- | | Nu,C3,u,av +/- | | Nu,C3,d,av +/- | | Nu,C4,u,av +/- | | Nush,C2 +/- | | Nush,C3 +/- | | finNu,D2,av +/- | | finNu,D3,av +/- | |
|-------|----------|------------|----------------|---------|----------------|---------|----------------|---------|----------------|--------|-------------|---|-------------|---------|-----------------|---------|-----------------|---------|
| | 3 S----- | B3run33-2 | -913.9 | 8.8E+01 | 1419.7 | 1.3E+02 | -290.9 | 1.0E+02 | 6.7E+02 | 122.8 | 8.5E+01 | 2 | 86.1 | 1.7E+00 | 147.6 | 1.2E+01 | 162.0 | 1.3E+01 |
| | 3 S----- | B3run33-3 | -463.9 | 6.4E+01 | 736.4 | 7.5E+01 | 114.3 | 6.4E+01 | 9.7E+01 | 65.0 | 6.0E+01 | 1 | 51.7 | 1.1E+00 | 100.0 | 7.5E+00 | 99.0 | 7.4E+00 |
| | 3 S----- | B3run33-4 | -683.9 | 8.6E+01 | 1256.3 | 1.2E+02 | 79.3 | 7.2E+01 | 2.4E+02 | 72.8 | 6.2E+01 | 2 | 44.4 | 1.1E+00 | 197.5 | 1.4E+01 | 152.9 | 9.5E+00 |
| | 3 S----- | B3run33-5 | -2124.1 | 1.2E+02 | 3596.0 | 3.4E+02 | -1344.7 | 1.4E+02 | 2.2E+03 | 269.6 | 1.1E+02 | 2 | 128.2 | 2.7E+00 | 177.5 | 2.2E+01 | 188.3 | 2.3E+01 |
| | 3 S----- | B3run33-6 | -1076.4 | 9.6E+01 | 1516.4 | 1.5E+02 | -523.8 | 7.5E+01 | 7.4E+02 | 89.1 | 1.0E+02 | 2 | 104.9 | 2.1E+00 | 89.7 | 1.1E+01 | 80.8 | 8.8E+00 |
| | 3 S----- | B3run33-7 | -391.4 | 6.7E+01 | 684.7 | 7.9E+01 | 217.6 | 7.4E+01 | 1.9E+01 | 69.7 | 5.6E+01 | 1 | 50.9 | 1.2E+00 | 113.3 | 7.6E+00 | 109.1 | 8.3E+00 |
| | 3 S----- | B3run33-8 | -981.0 | 8.3E+01 | 1489.1 | 1.2E+02 | -215.3 | 1.1E+02 | 5.7E+02 | 117.0 | 7.7E+01 | 2 | 76.1 | 1.6E+00 | 134.9 | 1.0E+01 | 153.5 | 1.1E+01 |
| | 3 S----- | B3run33-9 | -708.3 | 7.2E+01 | 1074.5 | 9.2E+01 | -233.1 | 7.1E+01 | 4.7E+02 | 76.6 | 8.1E+01 | 2 | 79.3 | 1.5E+00 | 113.2 | 9.8E+00 | 103.1 | 8.5E+00 |
| | 3 S----- | B3run33-10 | -1623.6 | 9.6E+01 | 2449.2 | 1.7E+02 | -889.5 | 1.0E+02 | 1.3E+03 | 140.1 | 1.0E+02 | 3 | 115.9 | 2.4E+00 | 116.1 | 1.2E+01 | 117.4 | 1.4E+01 |
| | 3 S----- | B3run34-1 | -572.7 | 8.2E+01 | 978.2 | 1.0E+02 | 139.6 | 9.6E+01 | 1.5E+02 | 96.2 | 6.0E+01 | 1 | 52.1 | 1.3E+00 | 145.1 | 9.8E+00 | 137.4 | 1.0E+01 |
| | 3 S----- | B3run34-2 | -1960.0 | 1.3E+02 | 3278.0 | 3.0E+02 | -1031.8 | 1.4E+02 | 1.6E+03 | 226.1 | 1.2E+02 | 2 | 127.5 | 2.3E+00 | 182.2 | 2.1E+01 | 172.1 | 2.2E+01 |
| | 3 S----- | B3run34-3 | -972.6 | 9.2E+01 | 1486.8 | 1.3E+02 | -227.5 | 8.8E+01 | 5.7E+02 | 93.4 | 8.2E+01 | 2 | 85.8 | 2.0E+00 | 139.8 | 1.4E+01 | 150.3 | 1.1E+01 |
| | 3 S----- | B3run34-4 | -462.4 | 6.4E+01 | 770.6 | 8.0E+01 | 141.5 | 8.6E+01 | 1.1E+02 | 85.8 | 5.6E+01 | 1 | 50.6 | 1.2E+00 | 115.1 | 9.2E+00 | 115.4 | 9.7E+00 |
| | 3 S----- | B3run34-5 | -1130.9 | 1.1E+02 | 1771.3 | 1.6E+02 | -241.5 | 1.0E+02 | 6.5E+02 | 114.8 | 8.4E+01 | 2 | 85.5 | 1.8E+00 | 160.3 | 1.3E+01 | 176.4 | 1.4E+01 |
| | 3 S----- | B3run34-6 | -2464.7 | 1.3E+02 | 4641.5 | 3.8E+02 | -1340.3 | 1.4E+02 | 2.3E+03 | 256.1 | 1.1E+02 | 2 | 133.4 | 2.4E+00 | 236.4 | 2.3E+01 | 232.7 | 2.4E+01 |
| | 3 S----- | B3run34-7 | -1436.6 | 1.1E+02 | 2242.6 | 1.9E+02 | -719.8 | 9.4E+01 | 1.1E+03 | 128.4 | 1.2E+02 | 3 | 125.1 | 2.4E+00 | 148.8 | 1.6E+01 | 113.1 | 1.2E+01 |
| | 3 S----- | B3run34-8 | -1359.3 | 1.3E+02 | 2208.5 | 2.5E+02 | -299.8 | 1.4E+02 | 8.0E+02 | 174.7 | 7.9E+01 | 2 | 81.3 | 1.6E+00 | 186.3 | 1.7E+01 | 209.5 | 1.7E+01 |
| | 3 S----- | B3run34-9 | -934.1 | 1.4E+02 | 1660.2 | 2.3E+02 | 248.4 | 1.3E+02 | 1.2E+02 | 126.0 | 6.7E+01 | 2 | 47.7 | 1.3E+00 | 218.0 | 1.9E+01 | 172.0 | 1.3E+01 |
| | 3 S----- | B3run35-1 | -427.1 | 7.9E+01 | 778.8 | 9.4E+01 | 227.6 | 8.6E+01 | 9.3E+01 | 87.5 | 5.5E+01 | 1 | 58.3 | 1.3E+00 | 134.1 | 9.4E+00 | 148.7 | 8.8E+00 |
| | 3 S----- | B3run35-2 | -1119.7 | 1.1E+02 | 1782.5 | 1.7E+02 | -494.5 | 1.4E+02 | 1.1E+03 | 170.3 | 7.9E+01 | 2 | 85.4 | 2.0E+00 | 171.8 | 1.5E+01 | 215.6 | 2.2E+01 |
| | 3 S----- | B3run35-3 | -1239.8 | 9.2E+01 | 1937.3 | 1.6E+02 | -255.4 | 1.2E+02 | 6.7E+02 | 135.2 | 8.0E+01 | 2 | 85.8 | 1.8E+00 | 159.9 | 1.6E+01 | 177.6 | 1.6E+01 |
| | 3 S----- | B3run35-4 | -2718.6 | 2.0E+02 | 5156.8 | 1.6E+03 | -2016.5 | 3.1E+02 | 3.5E+03 | 8123.9 | 1.0E+02 | 2 | 119.1 | 3.5E+00 | 190.8 | 2.7E+01 | 220.0 | 6.2E+01 |
| | 3 S----- | B3run35-5 | -841.0 | 7.0E+01 | 1210.5 | 9.7E+01 | -420.5 | 9.2E+01 | 6.6E+02 | 107.4 | 7.5E+01 | 2 | 72.3 | 1.8E+00 | 98.8 | 1.0E+01 | 93.9 | 1.1E+01 |
| | 3 S----- | B3run35-6 | -1997.8 | 1.4E+02 | 3214.3 | 3.4E+02 | -1332.8 | 1.3E+02 | 2.0E+03 | 211.5 | 9.8E+01 | 2 | 107.5 | 3.0E+00 | 132.6 | 1.7E+01 | 130.0 | 1.8E+01 |
| | 3 S----- | B3run35-7 | -617.5 | 6.0E+01 | 967.6 | 7.8E+01 | 246.2 | 6.4E+01 | 6.8E+00 | 63.0 | 5.1E+01 | 1 | 56.3 | 1.1E+00 | 114.5 | 8.1E+00 | 117.6 | 8.2E+00 |
| | 3 S----- | B3run35-8 | -703.6 | 8.4E+01 | 1238.7 | 1.1E+02 | 20.6 | 8.6E+01 | 3.3E+02 | 87.3 | 5.6E+01 | 2 | 49.4 | 1.2E+00 | 177.7 | 1.3E+01 | 161.4 | 9.9E+00 |
| | 3 S----- | B3run35-9 | -1416.1 | 1.3E+02 | 2085.6 | 2.2E+02 | -1019.8 | 1.1E+02 | 1.4E+03 | 161.6 | 1.0E+02 | 3 | 100.3 | 3.5E+00 | 108.9 | 1.7E+01 | 91.9 | 1.7E+01 |
| | 3 S----- | B3run36-1 | -291.6 | 7.4E+01 | 604.3 | 8.4E+01 | 365.3 | 7.0E+01 | -9.3E+01 | 66.0 | 5.4E+01 | 1 | 53.8 | 1.3E+00 | 125.1 | 9.2E+00 | 123.8 | 8.8E+00 |

| Build | Exp Code | Run | Nu,C2,d,av +/- | | Nu,C3,u,av +/- | | Nu,C3,d,av +/- | | Nu,C4,u,av +/- | | Nush,C2 | +/- | Nush,C3 | +/- | finNu,D2,av +/- | | finNu,D3,av +/- | |
|-------|-------------|-----------|----------------|---------|----------------|---------|----------------|---------|----------------|----------|---------|-----|---------|---------|-----------------|---------|-----------------|---------|
| 3 | 3S----- | B3run37-1 | -5314.6 | 2.5E+04 | 154778.8 | 1.9E+04 | -3997.5 | 5.8E+04 | -1.5E+05 | 142311.6 | 1.1E+02 | 365 | 109.4 | 1.1E+03 | 2777.2 | 2.3E+03 | 6843.4 | 2.2E+03 |
| 3 | 3S----- | B3run38-1 | -138.1 | 7.3E+01 | 505.0 | 7.7E+01 | 408.0 | 5.2E+01 | -1.2E+02 | 48.2 | 4.4E+01 | 1 | 17.3 | 9.9E-01 | 147.7 | 8.8E+00 | 132.7 | 6.0E+00 |
| 3 | 3S----- | B3run39-1 | -18455.3 | 6.5E+04 | -12412.0 | 7.1E+04 | 12523.5 | 6.4E+03 | -7.5E+03 | 78773.6 | 1.1E+02 | 2 | 111.0 | 2.5E+00 | -6119.2 | 2.0E+04 | -3408.3 | 1.7E+04 |
| 3 | 3S----- | B3run40-1 | 244.4 | 4.8E+01 | -48.0 | 4.7E+01 | 273.0 | 3.9E+01 | -1.4E+02 | 36.7 | 2.9E+01 | 1 | 25.6 | 6.4E-01 | 81.5 | 4.8E+00 | 58.9 | 4.4E+00 |
| 3 | 3S----- | B3run44-1 | -1066.3 | 4.6E+03 | 4337.9 | 2.5E+04 | -1027.4 | 8.6E+03 | 9.9E+03 | 19490.9 | 1.1E+02 | 49 | 108.8 | 5.1E+01 | 1117.8 | 1.9E+03 | 2610.7 | 6.4E+03 |
| 3 | 3S----- | B3run44-2 | -677.2 | 6.4E+03 | 1840.0 | 1.1E+04 | -116.2 | 1.1E+04 | 1.7E+03 | ##### | 1.0E+02 | 23 | 99.4 | 4.3E+01 | 444.9 | 2.7E+04 | 657.1 | 8.7E+02 |
| 3 | 3S----- | B3run44-3 | -1045.9 | 2.1E+02 | 2154.5 | 3.6E+02 | -11.8 | 4.1E+02 | 9.9E+02 | 492.7 | 9.5E+01 | 2 | 93.3 | 2.3E+00 | 358.9 | 3.4E+01 | 431.3 | 5.1E+01 |
| 3 | 3S----- | B3run45-1 | 91.4 | 1.2E+03 | -54.5 | 1.9E+03 | 623.1 | 9.3E+02 | -5.7E+02 | 767.0 | 2.8E+01 | 37 | -1.7 | 4.8E+01 | 15.9 | 9.9E+01 | 1.4 | 1.0E+03 |

| Build | Exp Code | Run | Tsh,C2,out (K) | +/- | Tsh,C3,out(K) | +/- | k_air(Tin) | +/- | V (m/s) | +/- |
|-------|-------------|-----------|-------------------|-----|---------------|------|------------|----------|---------|---------|
| | 3 3S----- | B3run2-1 | 379 | 0.3 | 3.9E+02 | 0.30 | 2.7E-02 | 2.38E-05 | 32.97 | 3.3E-02 |
| | 3 3S----- | B3run2-2 | 380 | 0.2 | 3.9E+02 | 0.19 | 2.8E-02 | 1.77E-05 | 32.97 | 2.5E-02 |
| | 3 3S----- | B3run3-1 | 383 | 0.2 | 3.9E+02 | 0.19 | 2.8E-02 | 2.14E-05 | 32.00 | 2.9E-02 |
| | 3 3S----- | B3run4-1 | 381 | 0.2 | 3.8E+02 | 0.25 | 2.7E-02 | 2.69E-05 | 11.46 | 1.3E-02 |
| | 3 3S----- | B3run4-2 | 381 | 0.5 | 3.9E+02 | 0.55 | 2.7E-02 | 3.14E-05 | 11.50 | 1.6E-02 |
| | 3 3S----- | B3run5-1 | 380 | 0.3 | 3.8E+02 | 0.26 | 2.7E-02 | 2.36E-05 | 20.03 | 2.0E-02 |
| | 3 3S----- | B3run5-3 | 378 | 0.2 | 3.8E+02 | 0.21 | 2.7E-02 | 1.60E-05 | 19.69 | 1.4E-02 |
| | 3 3S----- | B3run6-1 | 379 | 0.2 | 3.9E+02 | 0.21 | 2.7E-02 | 1.93E-05 | 19.90 | 1.7E-02 |
| | 3 3S----- | B3run6-2 | 378 | 1.6 | 3.8E+02 | 1.60 | 2.7E-02 | 6.75E-05 | 19.81 | 5.8E-02 |
| | 3 3S----- | B3run6-3 | 379 | 0.3 | 3.9E+02 | 0.24 | 2.7E-02 | 1.93E-05 | 19.88 | 1.7E-02 |
| | 3 3S----- | B3run7-1 | 371 | 0.7 | 3.8E+02 | 0.66 | 2.6E-02 | 2.11E-05 | 9.36 | 8.8E-03 |
| | 3 3S----- | B3run7-2 | 383 | 0.3 | 3.9E+02 | 0.30 | 2.8E-02 | 3.00E-05 | 9.59 | 1.2E-02 |
| | 3 3S----- | B3run7-4 | 392 | 0.3 | 3.9E+02 | 0.35 | 2.8E-02 | 3.82E-05 | 9.71 | 1.6E-02 |
| | 3 3S----- | B3run8-1 | 393 | 0.2 | 4.0E+02 | 0.21 | 2.7E-02 | 2.20E-05 | 9.14 | 8.6E-03 |
| | 3 3S----- | B3run9-1 | 381 | 0.2 | 3.9E+02 | 0.24 | 2.7E-02 | 1.89E-05 | 26.58 | 2.2E-02 |
| | 3 3S----- | B3run9-2 | 383 | 0.2 | 3.8E+02 | 0.16 | 2.7E-02 | 1.92E-05 | 9.74 | 8.0E-03 |
| | 3 3S----- | B3run9-3 | 383 | 0.2 | 3.8E+02 | 0.16 | 2.7E-02 | 1.84E-05 | 9.74 | 7.6E-03 |
| | 3 3S----- | B3run9-4 | 383 | 0.2 | 3.8E+02 | 0.16 | 2.8E-02 | 2.10E-05 | 20.38 | 1.8E-02 |
| | 3 3S----- | B3run12-1 | 387 | 0.2 | 3.9E+02 | 0.16 | 2.8E-02 | 1.63E-05 | 14.87 | 1.0E-02 |
| | 3 3S----- | B3run13-1 | 381 | 0.2 | 3.9E+02 | 0.18 | 2.8E-02 | 2.41E-05 | 41.44 | 4.2E-02 |
| | 3 3S----- | B3run13-2 | 381 | 0.2 | 3.9E+02 | 0.16 | 2.8E-02 | 3.32E-05 | 41.40 | 5.8E-02 |
| | 3 3S----- | B3run13-3 | 381 | 0.2 | 3.8E+02 | 0.17 | 2.7E-02 | 1.75E-05 | 14.92 | 1.1E-02 |
| | 3 3S----- | B3run15-1 | 381 | 0.2 | 3.9E+02 | 0.16 | 2.7E-02 | 1.97E-05 | 15.71 | 1.3E-02 |
| | 3 3S----- | B3run15-2 | 374 | 0.2 | 3.7E+02 | 0.16 | 2.7E-02 | 1.73E-05 | 7.94 | 6.0E-03 |
| | 3 3S----- | B3run16-1 | 376 | 0.2 | 3.7E+02 | 0.16 | 2.7E-02 | 1.83E-05 | 15.71 | 1.2E-02 |

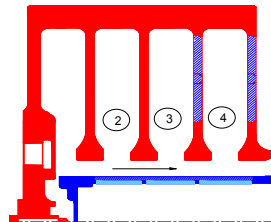
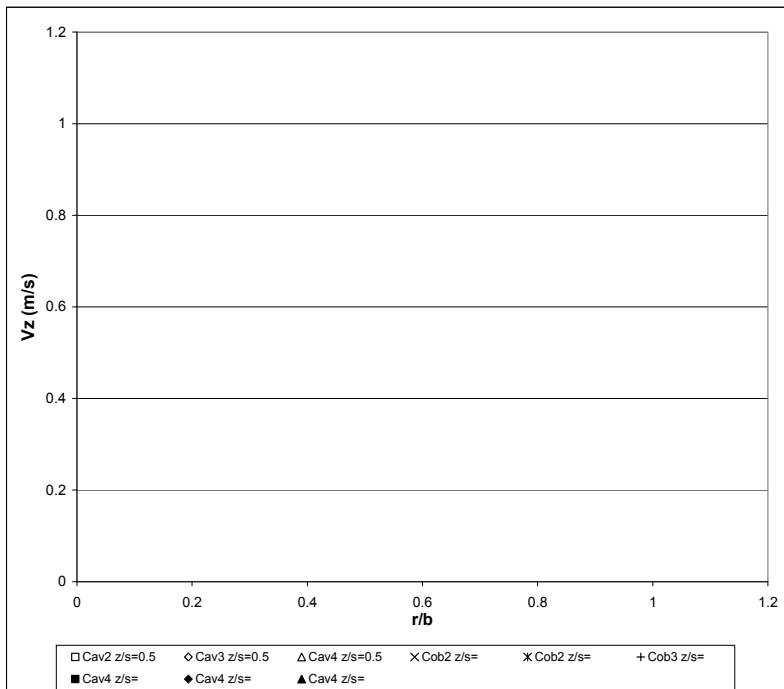
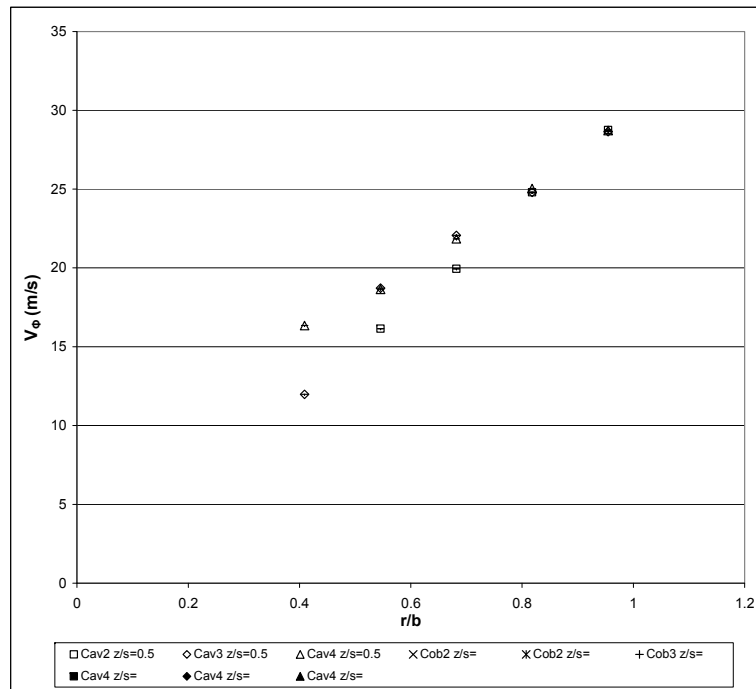
| Build | Exp Code | Run | Tsh,C2,out (K) | +/- | Tsh,C3,out(K) | +/- | k_air(Tin) | +/- | V (m/s) | +/- |
|-------|-------------|-----------|-------------------|-----|---------------|------|------------|----------|---------|---------|
| | 3 3S----- | B3run16-2 | 383 | 0.2 | 3.8E+02 | 0.18 | 2.8E-02 | 1.81E-05 | 15.57 | 1.2E-02 |
| | 3 3S----- | B3run17-1 | 376 | 0.2 | 3.8E+02 | 0.16 | 2.7E-02 | 1.64E-05 | 30.56 | 2.2E-02 |
| | 3 3S----- | B3run17-2 | 377 | 0.2 | 3.8E+02 | 0.16 | 2.7E-02 | 1.90E-05 | 30.40 | 2.5E-02 |
| | 3 3S----- | B3run17-3 | 380 | 0.2 | 3.8E+02 | 0.17 | 2.7E-02 | 1.83E-05 | 30.69 | 2.4E-02 |
| | 3 3S----- | B3run17-4 | 380 | 0.2 | 3.8E+02 | 0.16 | 2.8E-02 | 1.81E-05 | 30.62 | 2.4E-02 |
| | 3 3S----- | B3run17-5 | 381 | 0.2 | 3.9E+02 | 0.16 | 2.7E-02 | 1.58E-05 | 8.53 | 5.8E-03 |
| | 3 3S----- | B3run17-6 | 381 | 0.2 | 3.9E+02 | 0.18 | 2.7E-02 | 1.87E-05 | 8.35 | 6.7E-03 |
| | 3 3S----- | B3run18-1 | 388 | 0.2 | 3.9E+02 | 0.19 | 2.7E-02 | 1.83E-05 | 9.61 | 7.6E-03 |
| | 3 3S----- | B3run18-2 | 383 | 0.2 | 3.9E+02 | 0.19 | 2.8E-02 | 1.81E-05 | 9.60 | 7.3E-03 |
| | 3 3S----- | B3run18-3 | 383 | 0.2 | 3.9E+02 | 0.17 | 2.8E-02 | 1.65E-05 | 20.52 | 1.4E-02 |
| | 3 3S----- | B3run19-1 | 384 | 0.7 | 3.9E+02 | 0.77 | 2.8E-02 | 2.44E-05 | 19.30 | 2.0E-02 |
| | 3 3S----- | B3run20-1 | 379 | 0.2 | 3.8E+02 | 0.18 | 2.8E-02 | 4.39E-05 | 15.60 | 2.9E-02 |
| | 3 3S----- | B3run21-1 | 372 | 0.2 | 3.7E+02 | 0.17 | 2.7E-02 | 1.79E-05 | 15.57 | 1.2E-02 |
| | 3 3S----- | B3run22-1 | 374 | 0.2 | 3.8E+02 | 0.16 | 2.7E-02 | 1.83E-05 | 8.02 | 6.4E-03 |
| | 3 3S----- | B3run22-2 | 377 | 0.2 | 3.7E+02 | 0.25 | 2.8E-02 | 1.91E-05 | 14.92 | 1.2E-02 |
| | 3 3S----- | B3run23-1 | 376 | 0.2 | 3.7E+02 | 0.17 | 2.8E-02 | 1.59E-05 | 19.12 | 1.3E-02 |
| | 3 3S----- | B3run24-1 | 367 | 0.2 | 3.7E+02 | 0.23 | 2.7E-02 | 1.81E-05 | 15.15 | 1.2E-02 |
| | 3 3S----- | B3run24-2 | 393 | 0.2 | 4.0E+02 | 0.22 | 2.9E-02 | 2.40E-05 | 15.90 | 1.5E-02 |
| | 3 3S----- | B3run25-1 | 387 | 0.2 | 3.9E+02 | 0.21 | 2.8E-02 | 1.64E-05 | 32.25 | 2.2E-02 |
| | 3 3S----- | B3run26-1 | 380 | 0.2 | 3.8E+02 | 0.16 | 2.8E-02 | 1.58E-05 | 7.70 | 5.0E-03 |
| | 3 3S----- | B3run27-1 | 376 | 0.2 | 3.7E+02 | 0.17 | 2.8E-02 | 1.84E-05 | 35.27 | 2.7E-02 |
| | 3 3S----- | B3run28-1 | 369 | 0.2 | 3.6E+02 | 0.19 | 2.7E-02 | 1.82E-05 | 17.88 | 1.4E-02 |
| | 3 3S----- | B3run28-2 | 373 | 0.2 | 3.7E+02 | 0.16 | 2.7E-02 | 1.87E-05 | 8.96 | 7.2E-03 |
| | 3 3S----- | B3run30-1 | 369 | 0.3 | 3.6E+02 | 0.28 | 2.7E-02 | 1.71E-05 | 34.37 | 2.5E-02 |
| | 3 3S----- | B3run31-1 | 384 | 0.2 | 3.9E+02 | 0.16 | 2.8E-02 | 2.00E-05 | 34.98 | 2.9E-02 |
| | 3 3S----- | B3run32-1 | 388 | 0.2 | 3.9E+02 | 0.17 | 2.7E-02 | 1.81E-05 | 17.38 | 1.4E-02 |
| | 3 3S----- | B3run33-1 | 375 | 0.2 | 3.7E+02 | 0.22 | 2.7E-02 | 2.15E-05 | 16.01 | 1.5E-02 |

| Build | Exp Code | Run | Tsh,C2,out (K) | +/- | Tsh,C3,out(K) | +/- | k_air(Tin) | +/- | V (m/s) | +/- |
|-------|-------------|------------|-------------------|-----|---------------|------|------------|----------|---------|---------|
| | 3 3S----- | B3run33-2 | 370 | 0.2 | 3.7E+02 | 0.17 | 2.7E-02 | 1.69E-05 | 16.08 | 1.2E-02 |
| | 3 3S----- | B3run33-3 | 373 | 0.2 | 3.7E+02 | 0.16 | 2.7E-02 | 1.61E-05 | 7.70 | 5.4E-03 |
| | 3 3S----- | B3run33-4 | 379 | 0.2 | 3.8E+02 | 0.16 | 2.7E-02 | 1.84E-05 | 33.19 | 2.6E-02 |
| | 3 3S----- | B3run33-5 | 374 | 0.2 | 3.7E+02 | 0.17 | 2.8E-02 | 1.98E-05 | 24.76 | 2.1E-02 |
| | 3 3S----- | B3run33-6 | 379 | 0.2 | 3.8E+02 | 0.18 | 2.7E-02 | 1.70E-05 | 5.13 | 3.7E-03 |
| | 3 3S----- | B3run33-7 | 381 | 0.2 | 3.8E+02 | 0.16 | 2.7E-02 | 1.55E-05 | 16.60 | 1.1E-02 |
| | 3 3S----- | B3run33-8 | 379 | 0.2 | 3.8E+02 | 0.16 | 2.8E-02 | 1.72E-05 | 32.79 | 2.4E-02 |
| | 3 3S----- | B3run33-9 | 381 | 0.2 | 3.8E+02 | 0.16 | 2.8E-02 | 1.54E-05 | 9.02 | 5.9E-03 |
| | 3 3S----- | B3run33-10 | 376 | 0.2 | 3.7E+02 | 0.19 | 2.8E-02 | 1.52E-05 | 11.24 | 7.2E-03 |
| | 3 3S----- | B3run34-1 | 377 | 0.2 | 3.8E+02 | 0.16 | 2.7E-02 | 1.86E-05 | 20.27 | 1.6E-02 |
| | 3 3S----- | B3run34-2 | 377 | 0.2 | 3.8E+02 | 0.17 | 2.7E-02 | 1.61E-05 | 17.25 | 1.2E-02 |
| | 3 3S----- | B3run34-3 | 372 | 0.2 | 3.7E+02 | 0.16 | 2.7E-02 | 1.84E-05 | 10.51 | 8.3E-03 |
| | 3 3S----- | B3run34-4 | 375 | 0.2 | 3.7E+02 | 0.16 | 2.7E-02 | 1.60E-05 | 10.67 | 7.3E-03 |
| | 3 3S----- | B3run34-5 | 376 | 0.2 | 3.7E+02 | 0.17 | 2.8E-02 | 1.72E-05 | 19.49 | 1.4E-02 |
| | 3 3S----- | B3run34-6 | 382 | 0.2 | 3.8E+02 | 0.16 | 2.8E-02 | 1.42E-05 | 30.96 | 1.8E-02 |
| | 3 3S----- | B3run34-7 | 380 | 0.2 | 3.8E+02 | 0.16 | 2.8E-02 | 1.64E-05 | 9.57 | 6.6E-03 |
| | 3 3S----- | B3run34-8 | 381 | 0.2 | 3.8E+02 | 0.16 | 2.8E-02 | 1.69E-05 | 34.92 | 2.5E-02 |
| | 3 3S----- | B3run34-9 | 383 | 0.2 | 3.8E+02 | 0.17 | 2.8E-02 | 1.93E-05 | 34.95 | 2.8E-02 |
| | 3 3S----- | B3run35-1 | 381 | 0.2 | 3.9E+02 | 0.17 | 2.8E-02 | 1.64E-05 | 16.59 | 1.2E-02 |
| | 3 3S----- | B3run35-2 | 376 | 0.2 | 3.7E+02 | 0.16 | 2.8E-02 | 1.84E-05 | 30.22 | 2.3E-02 |
| | 3 3S----- | B3run35-3 | 376 | 0.2 | 3.7E+02 | 0.16 | 2.8E-02 | 1.63E-05 | 14.97 | 1.0E-02 |
| | 3 3S----- | B3run35-4 | 374 | 0.2 | 3.7E+02 | 0.17 | 2.8E-02 | 2.18E-05 | 25.27 | 2.3E-02 |
| | 3 3S----- | B3run35-5 | 380 | 0.2 | 3.8E+02 | 0.16 | 2.8E-02 | 1.63E-05 | 7.76 | 5.3E-03 |
| | 3 3S----- | B3run35-6 | 375 | 0.2 | 3.7E+02 | 0.16 | 2.8E-02 | 1.67E-05 | 12.74 | 8.8E-03 |
| | 3 3S----- | B3run35-7 | 390 | 0.2 | 3.9E+02 | 0.16 | 2.8E-02 | 1.58E-05 | 10.38 | 6.8E-03 |
| | 3 3S----- | B3run35-8 | 390 | 0.2 | 3.9E+02 | 0.16 | 2.8E-02 | 1.94E-05 | 38.86 | 3.1E-02 |
| | 3 3S----- | B3run35-9 | 378 | 0.2 | 3.7E+02 | 0.19 | 2.8E-02 | 1.79E-05 | 6.46 | 4.7E-03 |
| | 3 3S----- | B3run36-1 | 387 | 0.2 | 3.9E+02 | 0.17 | 2.8E-02 | 1.93E-05 | 21.53 | 1.7E-02 |

| Build | Exp Code | Run | Tsh,C2,out (K) | +/- | Tsh,C3,out(K) | +/- | k _{air} (T _{in}) | +/- | V (m/s) | +/- |
|-------|-------------|-----------|-------------------|-----|---------------|------|-------------------------------------|----------|---------|---------|
| 3 | 3S----- | B3run37-1 | 365 | 1.9 | 3.6E+02 | 1.88 | 2.8E-02 | 6.21E-05 | 12.09 | 3.2E-02 |
| 3 | 3S----- | B3run38-1 | 398 | 0.2 | 4.1E+02 | 0.19 | 2.8E-02 | 1.99E-05 | 36.10 | 3.0E-02 |
| 3 | 3S----- | B3run39-1 | 369 | 0.2 | 3.6E+02 | 0.16 | 2.8E-02 | 1.78E-05 | 12.13 | 8.9E-03 |
| 3 | 3S----- | B3run40-1 | 405 | 0.2 | 4.1E+02 | 0.16 | 2.8E-02 | 1.73E-05 | 13.31 | 9.6E-03 |
| 3 | 3S----- | B3run44-1 | 358 | 2.0 | 3.5E+02 | 1.99 | 2.7E-02 | 6.86E-05 | 32.31 | 9.7E-02 |
| 3 | 3S----- | B3run44-2 | 363 | 2.0 | 3.6E+02 | 1.92 | 2.7E-02 | 7.78E-05 | 32.40 | 1.1E-01 |
| 3 | 3S----- | B3run44-3 | 367 | 0.2 | 3.6E+02 | 0.18 | 2.7E-02 | 1.86E-05 | 32.70 | 2.6E-02 |
| 3 | 3S----- | B3run45-1 | 302 | 0.2 | 3.0E+02 | 0.18 | 2.7E-02 | 2.20E-05 | 10.39 | 1.0E-02 |

Appendix 6.4: Selected LDA test reports

| | | | | | | |
|------------------------------------|--|--------------------------------------|-----------------------|-----------------------|----------------------|----------|
| LDA measurement | | LDA code: 100 | rpm: | Perfd by: AA | Printed: 26/09/2008 | Page 1/3 |
| Build: 3 | | ReΦ: 1.00E+06 | Date: | LDA1: V Φ | Comments: | |
| LDA test # : B3run4 | | Rez: 6.27E+04 | raw LDA file: | LDA2: Vz | Traverse ref: | |
| Heat Transfer run: B3run4-1 | | Ro: 1.19 | LDA list file: | W (m/s): 11.46 | | |



LDA measurement

Build: 3
LDA test # : B3run4
Heat Transfer run: B3run4-1

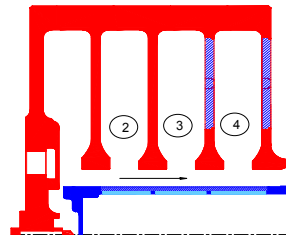
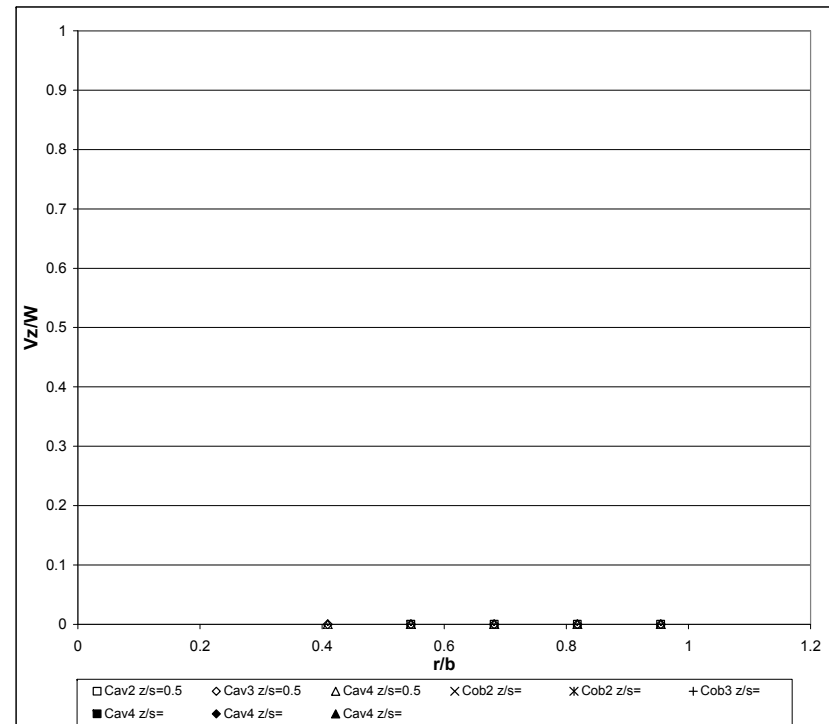
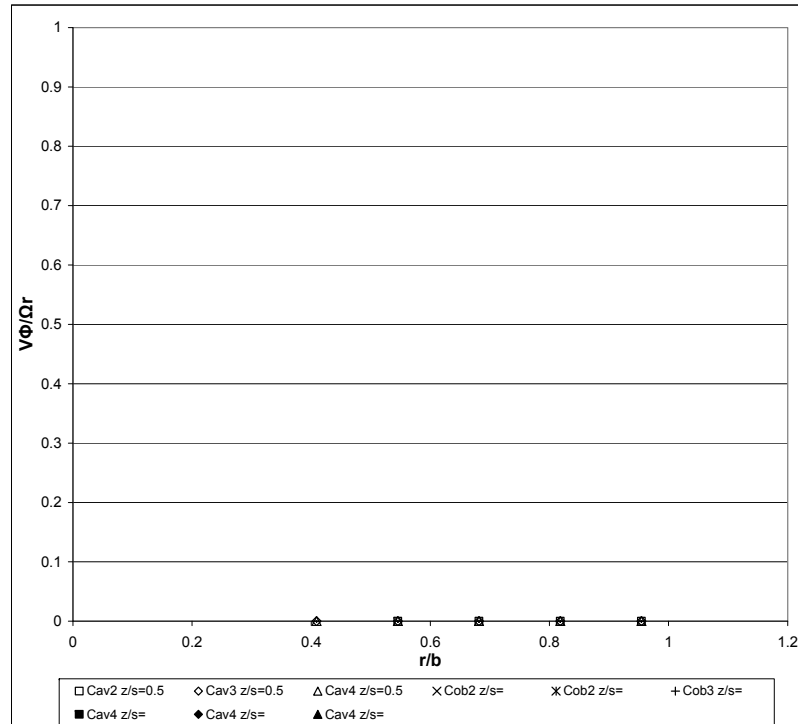
LDA code: 101
Re_φ: 1.00E+06
Re_z: 6.27E+04
Ro: 1.19

rpm: 0.0
Date: 00/01/1900
raw LDA file: 0
LDA list file: 0

Perfd by: AA
LDA1: Vφ
LDA2: Vz
W (m/s): 11.46

Printed: 26/09/2008
Comments: 0
Traverse corresp: 0

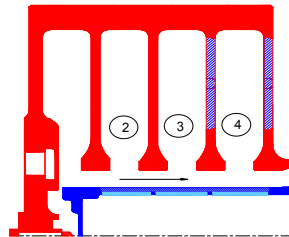
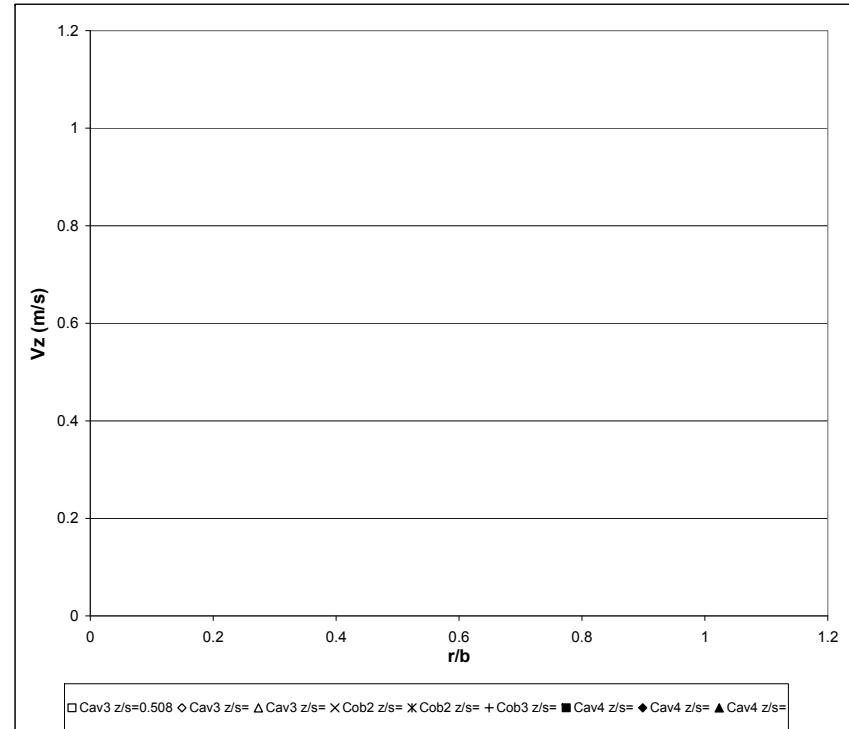
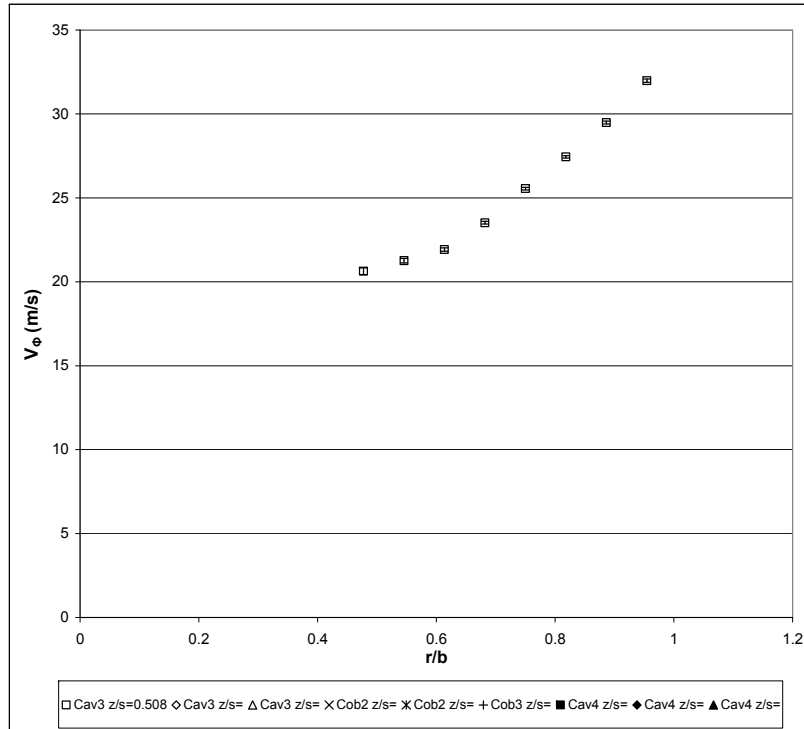
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LDA measurement
Build: 3 **LDA code:** 100 **rpm:** 1311.0 **Perfd by:** AA **Printed:** 26/09/2008 **Page 3/3**
ReΦ: 1.00E+06 **Date:** 00/01/1900 **LDA1:** VΦ **Comments:** 0
LDA test # : B3run4 **Rez:** 6.27E+04 **raw LDA file:** 0 **LDA2:** Vz **Traversal corresp:** 0
Heat Transfer run: B3run4-1 **Ro:** 1.19 **LDA list file:** 0 **W (m/s):** 11.46

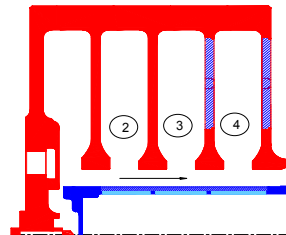
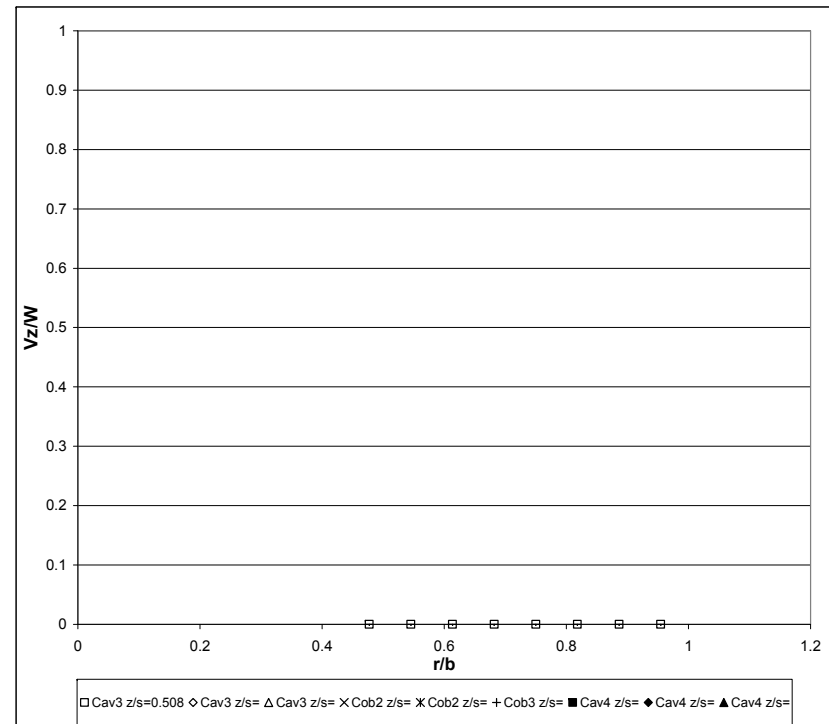
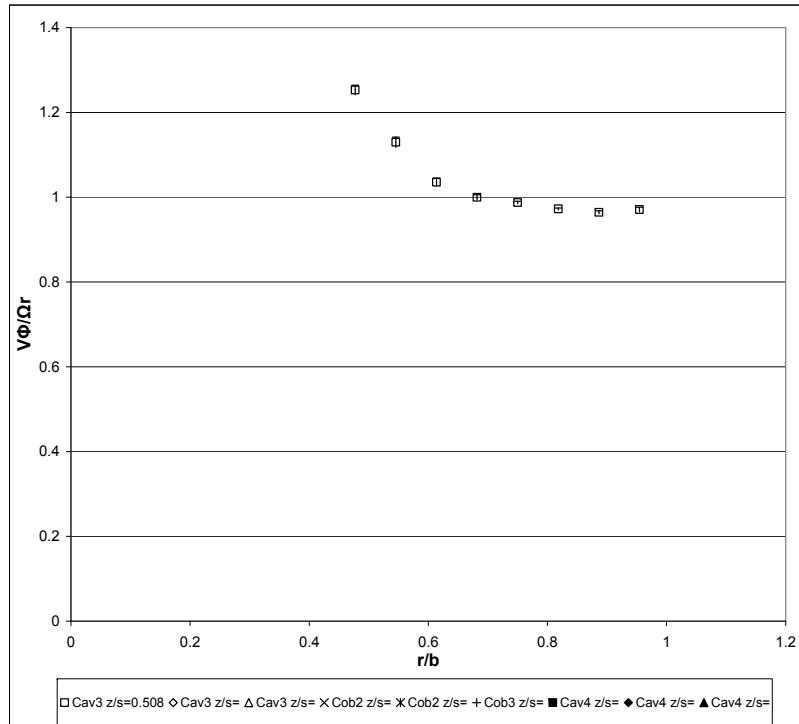
| X [mm] | correcte | Y [mm] | corrected Y / | Count{1} | Count{2} | Data Rate{1} | Data Rate{2} | Validation | Validation | LDA1 | LDA2 | LDA1 | LDA2 | LDA1 | LDA2 | LDA1 | LDA2 | Cavity | r/b | z/s | Ωr (m/s) | V _{ΦU95%} / | V _{Φ(RMS)} / | Vz/W | Vz _(RMS) / | Vz _{U95%} /W | |
|--------|----------|--------|---------------|----------|----------|--------------|--------------|------------|------------|----------|---------|------|------|------|------|----------|---------|--------|-------|-------|----------|----------------------|-----------------------|------|-----------------------|-----------------------|----------|
| | d X | | Radius (mm) | | | #/s | #/s | | | n{1} [%] | {2} [%] | Mean | Mean | RMS | RMS | MeanConf | MeanCon | | | | | RMSCConf | RMSCConf | | r | | Ωr |
| 115 | | 140 | 210 | 100000 | | 7211.1 | | 70.00 | | 28.75 | | 0.97 | | 0.01 | | 0.00 | | 2 | 0.955 | 0.500 | 28.83 | 1.00 | 2.09E-04 | 0.03 | 0.00 | 0.00 | 0.00E+00 |
| 115 | | 110 | 180 | 5000 | | 332.2 | | 70.00 | | 24.80 | | 1.05 | | 0.03 | | 0.02 | | 2 | 0.818 | 0.500 | 24.71 | 1.00 | 1.18E-03 | 0.04 | 0.00 | 0.00 | 0.00E+00 |
| 115 | | 80 | 150 | 781 | | 39.1 | | 70.00 | | 19.94 | | 1.34 | | 0.09 | | 0.07 | | 2 | 0.682 | 0.500 | 20.59 | 0.97 | 4.58E-03 | 0.07 | 0.00 | 0.00 | 0.00E+00 |
| 115 | | 50 | 120 | 548 | | 27.5 | | 70.00 | | 16.15 | | 2.17 | | 0.18 | | 0.13 | | 2 | 0.545 | 0.500 | 16.47 | 0.98 | 1.10E-02 | 0.13 | 0.00 | 0.00 | 0.00E+00 |
| 64 | | 140 | 210 | 544 | | 27.3 | | 70.00 | | 28.64 | | 2.16 | | 0.18 | | 0.13 | | 3 | 0.955 | 0.500 | 28.83 | 0.99 | 6.31E-03 | 0.07 | 0.00 | 0.00 | 0.00E+00 |
| 64 | | 110 | 180 | 396 | | 19.9 | | 70.00 | | 24.80 | | 1.95 | | 0.19 | | 0.14 | | 3 | 0.818 | 0.500 | 24.71 | 1.00 | 7.77E-03 | 0.08 | 0.00 | 0.00 | 0.00E+00 |
| 64 | | 80 | 150 | 327 | | 16.4 | | | | 22.06 | | 1.48 | | 0.16 | | 0.11 | | 3 | 0.682 | 0.500 | 20.59 | 1.07 | 7.80E-03 | 0.07 | 0.00 | 0.00 | 0.00E+00 |
| 64 | | 50 | 120 | 722 | | 36.3 | | | | 18.72 | | 1.57 | | 0.11 | | 0.08 | | 3 | 0.545 | 0.500 | 16.47 | 1.14 | 6.94E-03 | 0.10 | 0.00 | 0.00 | 0.00E+00 |
| 64 | | 20 | 90 | 68 | | 3.5 | | | | 11.98 | | 9.07 | | 2.17 | | 1.54 | | 3 | 0.409 | 0.500 | 12.36 | 0.97 | 1.76E-01 | 0.73 | 0.00 | 0.00 | 0.00E+00 |
| 13 | | 140 | 210 | 5000 | | 344.2 | | | | 28.72 | | 2.23 | | 0.06 | | 0.04 | | 4 | 0.955 | 0.500 | 28.83 | 1.00 | 2.15E-03 | 0.08 | 0.00 | 0.00 | 0.00E+00 |
| 13 | | 110 | 180 | 5000 | | 1990.0 | | | | 25.03 | | 2.19 | | 0.06 | | 0.04 | | 4 | 0.818 | 0.500 | 24.71 | 1.01 | 2.46E-03 | 0.09 | 0.00 | 0.00 | 0.00E+00 |
| 13 | | 80 | 150 | 5000 | | 2313.3 | | | | 21.85 | | 1.49 | | 0.04 | | 0.03 | | 4 | 0.682 | 0.500 | 20.59 | 1.06 | 2.01E-03 | 0.07 | 0.00 | 0.00 | 0.00E+00 |
| 13 | | 50 | 120 | 5000 | | 2781.0 | | | | 18.65 | | 1.77 | | 0.05 | | 0.03 | | 4 | 0.545 | 0.500 | 16.47 | 1.13 | 2.99E-03 | 0.11 | 0.00 | 0.00 | 0.00E+00 |
| 13 | | 20 | 90 | 5000 | | 521.0 | | | | 16.35 | | 3.14 | | 0.09 | | 0.06 | | 4 | 0.409 | 0.500 | 12.36 | 1.32 | 7.04E-03 | 0.25 | 0.00 | 0.00 | 0.00E+00 |

| | | | | | | |
|------------------------------|----------------------|---|----------------|------------------|---------------------|----------|
| LDA measurement | | LDA code: 100 | rpm: 1497.8 | Perfd by: NM | Printed: 26/09/2008 | Page 1/3 |
| Build: 3 | Re Φ : 1.03E+06 | Date: 21/06/2003 | LDA1: V Φ | Comments: run OK | | |
| LDA test #: B3run18-3 | Rez: 1.01E+05 | raw LDA file: \MCRLDA\Build3\B3run18-3.lda | LDA2: Vz | Traverse ref: | | |
| Heat Transfer run: B3run18-3 | Ro: 1.87 | LDA list file: \MCRLDA\Build3\B3run18-3.xls | W (m/s): 20.52 | | | |



LDA measurement**Build:** 3**LDA test # :** B3run18-3**Heat Transfer run:** B3run18-3**LDA code:** 101**Re_φ:** 1.03E+06**Re_z:** 1.01E+05**Ro:** 1.87**rpm:** 1497.8**Date:** 21/06/2003**raw LDA file:** \MCRLDA\Build3\B3run18-3.lda**LDA list file:** \MCRLDA\Build3\B3run18-3.xls**Perfd by:** NM**LDA1:** Vφ**LDA2:** Vz**W (m/s):** 20.52**Printed:** 26/09/2008**Comments:** run OK**Traverse corresp:** 0

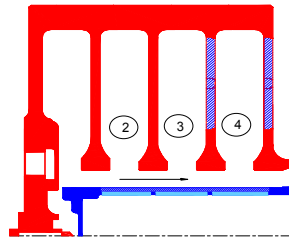
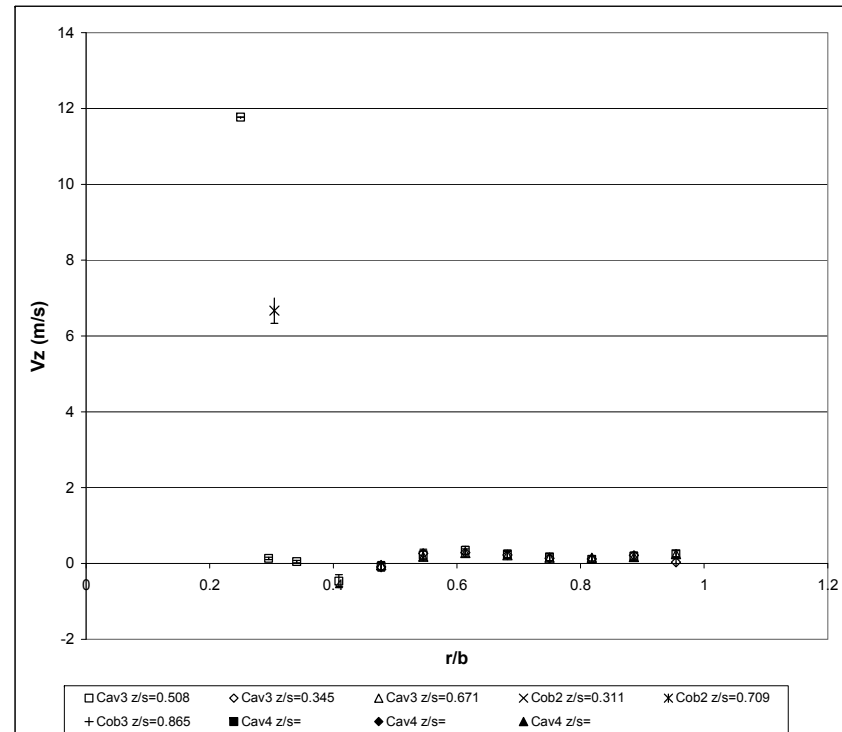
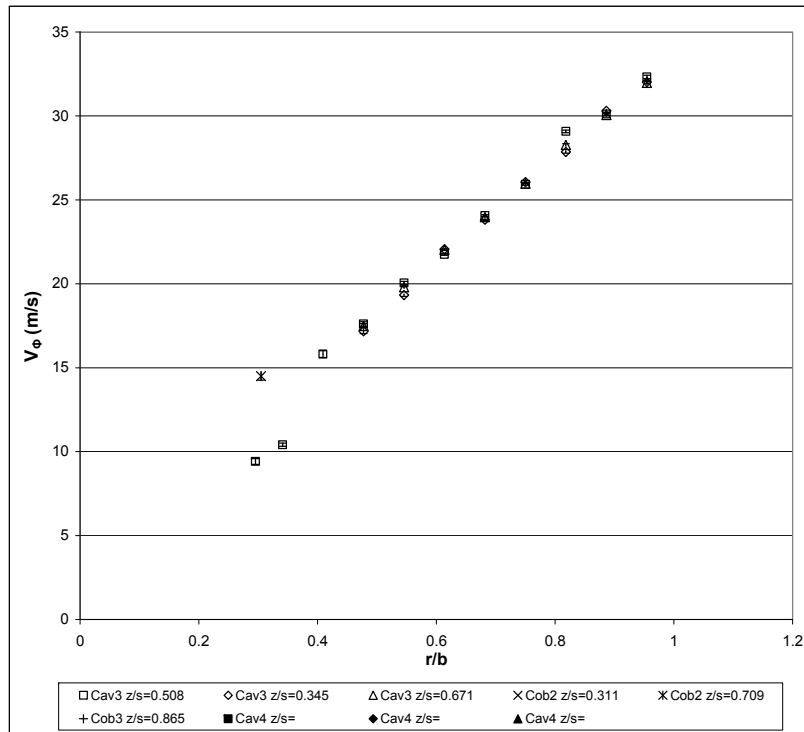
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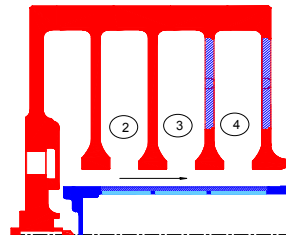
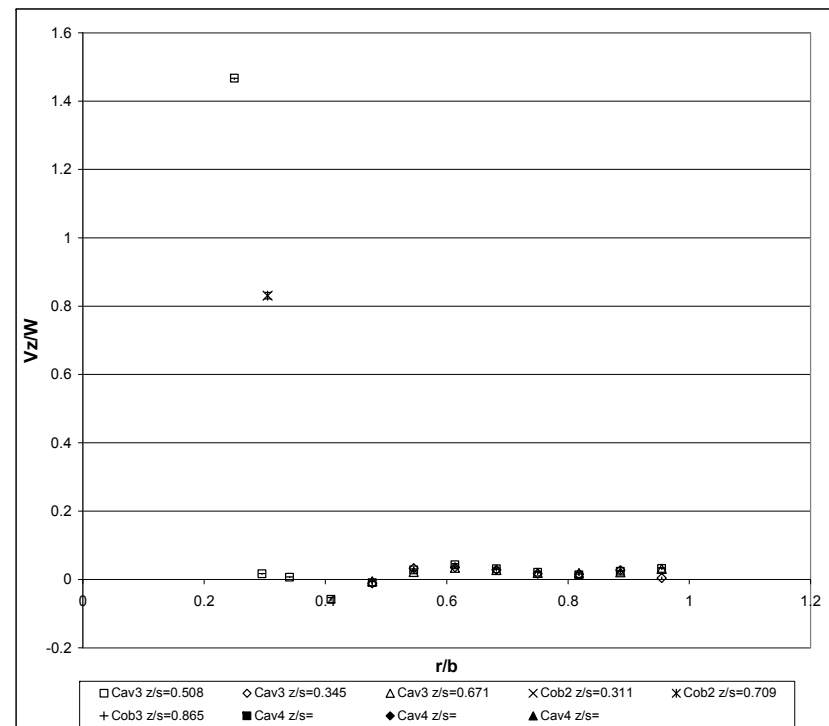
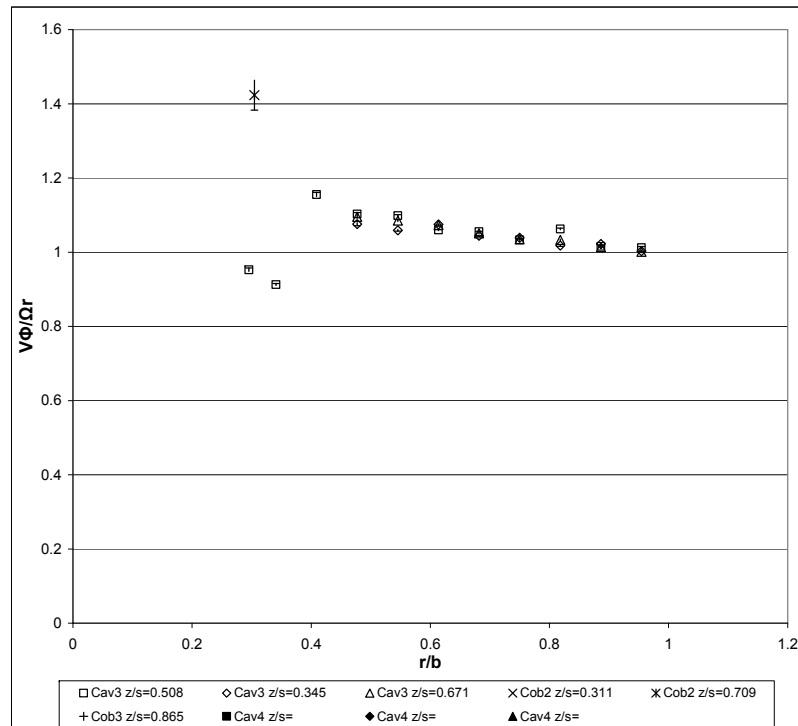
LDA measurement
Build: 3 **LDA code:** 100 **rpm:** 1497.8 **Perfd by:** NM
ReΦ: 1.03E+06 **Date:** 21/06/2003 **LDA1:** VΦ **Printed:** 26/09/2008
LDA test # : B3run18-3 **Rez:** 1.01E+05 **raw LDA file:** \MCRLDA\Build3\B3run18-3.lda **LDA2:** Vz **Comments:** run OK
Heat Transfer run: B3run18-3 **Ro:** 1.87 **LDA list file:** \MCRLDA\Build3\B3run18-3.xls **W (m/s):** 20.52 **Traversal corresp:** 0

| X [mm] | corrected X | Y [mm] | corrected Y / Radius (mm) | Count{1} | Count{2} | Data Rate{1} [#s] | Data Rate{2} [#s] | Validation n{1} [%] | Validation {2} [%] | LDA1 Mean [m/s] | LDA2 Mean [m/s] | LDA1 RMS [m/s] | LDA2 RMS [m/s] | LDA1 MeanConf [m/s] | LDA2 MeanConf [m/s] | LDA1 RMSConf [m/s] | LDA2 RMSConf [m/s] | Cavity | r/b | z/s | Ωr (m/s) | V _θ /Ωr | V _θ 95%/Ωr | V _θ (RMS)/Ωr | Vz/W | Vz(RMS)/W | Vz _{95%} /W |
|--------|-------------|--------|---------------------------|----------|----------|-------------------|-------------------|---------------------|--------------------|-----------------|-----------------|----------------|----------------|---------------------|---------------------|--------------------|--------------------|--------|-------|-------|----------|--------------------|-----------------------|-------------------------|------|-----------|----------------------|
| 64 | | 140 | 210 | 1311 | | 65.6 | | | | 31.98 | | 3.39 | | 0.18 | | 0.13 | | 3 | 0.955 | 0.508 | 32.94 | 0.97 | 5.57E-03 | 0.10 | 0.00 | 0.00 | 0.00E+00 |
| 64 | | 125 | 195 | 1998 | | 100.2 | | | | 29.49 | | 2.52 | | 0.11 | | 0.08 | | 3 | 0.886 | 0.508 | 30.59 | 0.96 | 3.62E-03 | 0.08 | 0.00 | 0.00 | 0.00E+00 |
| 64 | | 110 | 180 | 4534 | | 226.8 | | | | 27.45 | | 2.16 | | 0.06 | | 0.04 | | 3 | 0.818 | 0.508 | 28.23 | 0.97 | 2.23E-03 | 0.08 | 0.00 | 0.00 | 0.00E+00 |
| 64 | | 95 | 165 | 2427 | | 121.4 | | | | 25.55 | | 1.92 | | 0.08 | | 0.05 | | 3 | 0.750 | 0.508 | 25.88 | 0.99 | 2.95E-03 | 0.07 | 0.00 | 0.00 | 0.00E+00 |
| 64 | | 80 | 150 | 806 | | 40.4 | | | | 23.52 | | 1.95 | | 0.14 | | 0.10 | | 3 | 0.682 | 0.508 | 23.53 | 1.00 | 5.74E-03 | 0.08 | 0.00 | 0.00 | 0.00E+00 |
| 64 | | 65 | 135 | 431 | | 21.6 | | | | 21.92 | | 2.11 | | 0.20 | | 0.14 | | 3 | 0.614 | 0.508 | 21.18 | 1.04 | 9.40E-03 | 0.10 | 0.00 | 0.00 | 0.00E+00 |
| 64 | | 50 | 120 | 453 | | 22.8 | | | | 21.26 | | 2.51 | | 0.23 | | 0.16 | | 3 | 0.545 | 0.508 | 18.82 | 1.13 | 1.23E-02 | 0.13 | 0.00 | 0.00 | 0.00E+00 |
| 64 | | 35 | 105 | 1136 | | 56.8 | | | | 20.63 | | 3.22 | | 0.19 | | 0.13 | | 3 | 0.477 | 0.508 | 16.47 | 1.25 | 1.14E-02 | 0.20 | 0.00 | 0.00 | 0.00E+00 |

| | | | | | | |
|------------------------------|---------------|---|---------------|------------------|---------------------|----------|
| LDA measurement | | LDA code: 101 | rpm: 1452.2 | Perfd by: NM | Printed: 26/09/2008 | Page 1/3 |
| Build: 3 | ReΦ: 1.24E+06 | Date: 07/07/2003 | LDA1: VΦ | Comments: run OK | | |
| LDA test #: B3run22-1 | Rez: 4.91E+04 | raw LDA file: \MCRLDA\Build3\B3run22-1.lda | LDA2: Vz | Traversal ref: | | |
| Heat Transfer run: B3run22-1 | Ro: 0.75 | LDA list file: \MCRLDA\Build3\B3run22-1.xls | W (m/s): 8.02 | | | |



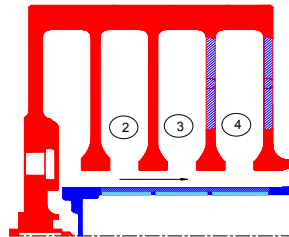
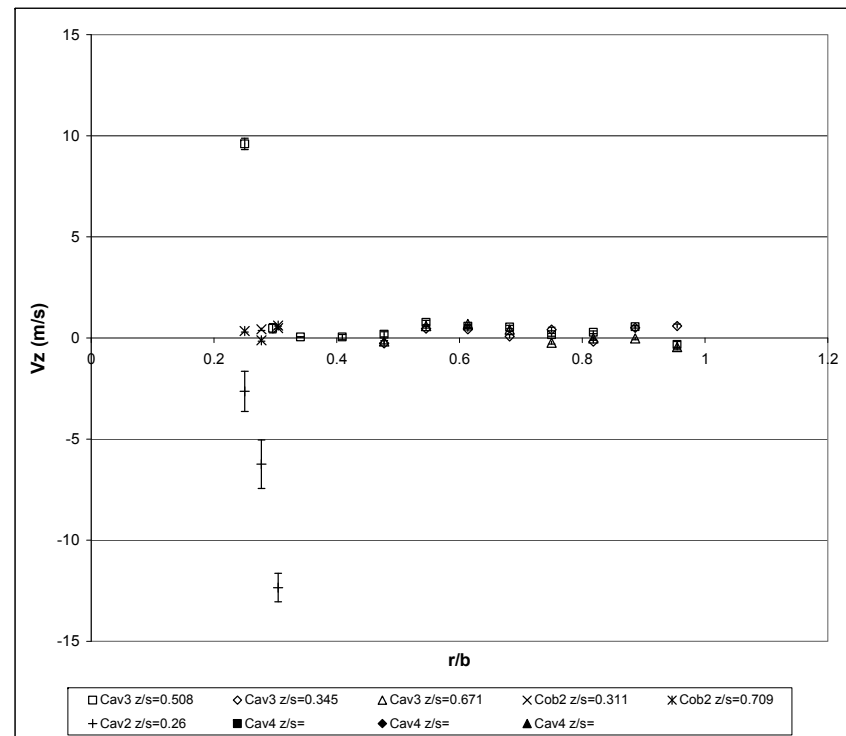
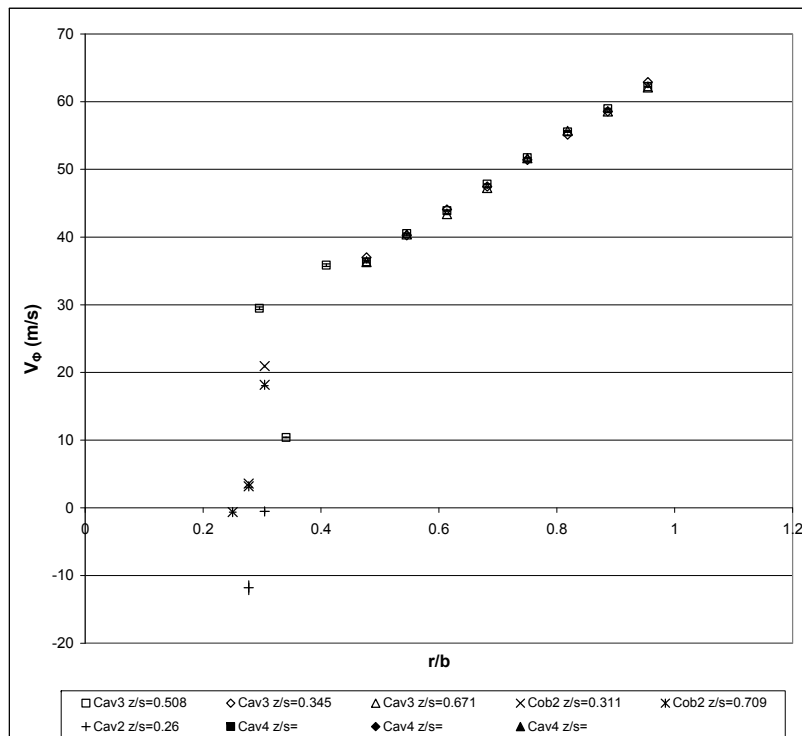
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|------------------------------|----------------------------|---|----------------|---------------------|---------------------|----------|
| LDA measurement | | LDA code: 101 | rpm: 1452.2 | Perfd by: NM | Printed: 26/09/2008 | Page 2/3 |
| Build: 3 | Re _φ : 1.24E+06 | Date: 07/07/2003 | LDA1: Vφ | Comments: run OK | | |
| LDA test #: B3run22-1 | Re _z : 4.91E+04 | raw LDA file: \MCRLDA\Build3\B3run22-1.lda | LDA2: Vz | Traverse corresp: 0 | | |
| Heat Transfer run: B3run22-1 | Ro: 0.75 | LDA list file: \MCRLDA\Build3\B3run22-1.xls | W (m/s): 8.025 | | | |



LDA measurement
Build: 3 **LDA code:** 101 **rpm:** 1452.2 **Perfd by:** NM
ReΦ: 1.24E+06 **Date:** 07/07/2003 **LDA1:** VΦ **Printed:** 26/09/2008 **Page 3/3**
LDA test # : B3run22-1 **Rez:** 4.91E+04 **raw LDA file:** \MCRLDA\Build3\B3run22-1.la **LDA2:** Vz **Comments:** run OK
Heat Transfer run: B3run22-1 **Ro:** 0.75 **LDA list file:** \MCRLDA\Build3\B3run22-1.xls **W (m/s):** 8.025 **Traverse corresp:** 0

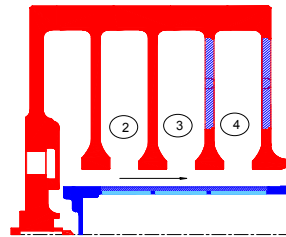
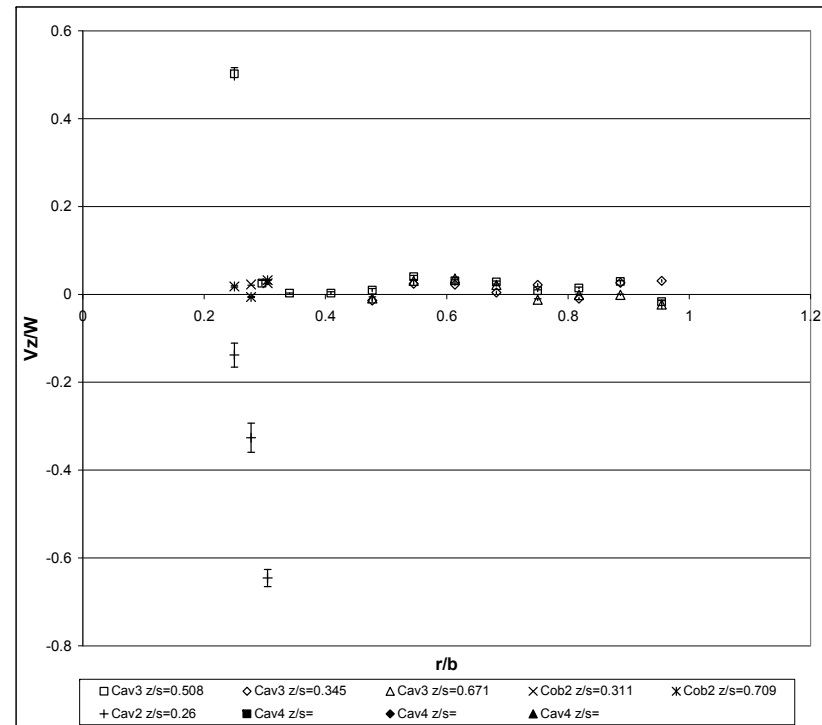
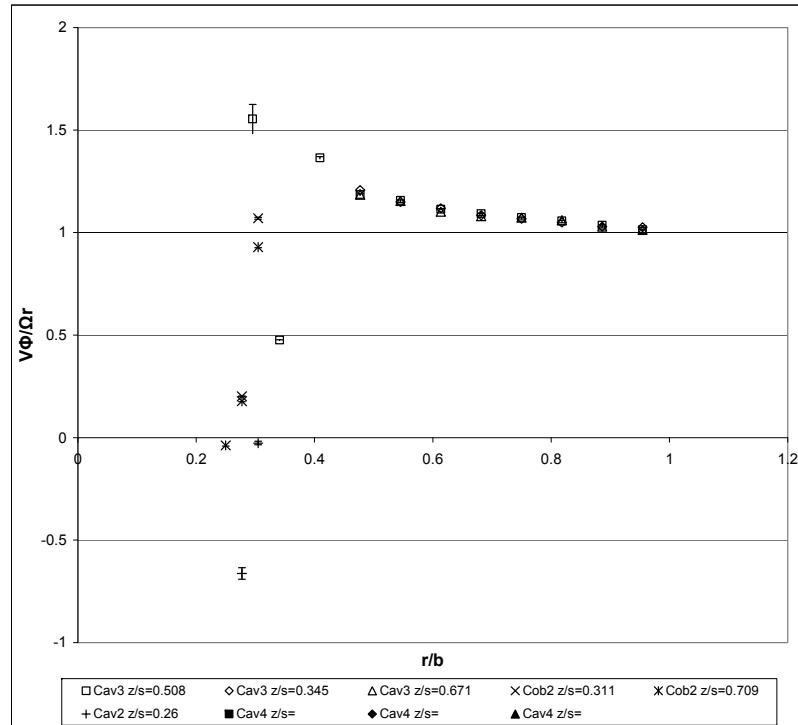
| X [mm] | correcte d X | Y [mm] | corrected Y / Radius (mm) | Count{1} | Count{2} | Data Rate{1} [#/s] | Data Rate{2} [#/s] | Validation n{1} [%] | Validation {2} [%] | LDA1 Mean [m/s] | LDA2 Mean [m/s] | LDA1 RMS [m/s] | LDA2 RMS [m/s] | LDA1 MeanConf [m/s] | LDA2 MeanCon f [m/s] | LDA1 RMSConf [m/s] | LDA2 RMSConf [m/s] | Cavity | r/b | z/s | Ωr (m/s) | V _φ /Ωr | V _φ U95%/Ωr | V _φ (RMS)/ Ωr | Vz(RMS)/ W | Vz(W) | Vz _{U95%/W} |
|--------|-----------------|--------|------------------------------|----------|----------|-----------------------|-----------------------|------------------------|-----------------------|-----------------------|-----------------------|----------------------|----------------------|---------------------------|----------------------------|--------------------------|--------------------------|--------|-------|-------|----------|--------------------|------------------------|-----------------------------|---------------|-------|----------------------|
| 64 | | 140 | 210 | 5000 | 5000 | 23477.7 | 7540.6 | | | 32.33 | 0.26 | 3.25 | 0.67 | 0.09 | 0.02 | 0.06 | 0.01 | 3 | 0.955 | 0.508 | 31.94 | 1.01 | 2.82E-03 | 0.10 | 0.03 | 0.08 | 2.31E-03 |
| 64 | | 125 | 195 | 5000 | 5000 | 21020.9 | 7589.7 | | | 30.12 | 0.19 | 2.81 | 0.72 | 0.08 | 0.02 | 0.06 | 0.01 | 3 | 0.886 | 0.508 | 29.65 | 1.02 | 2.63E-03 | 0.09 | 0.02 | 0.09 | 2.48E-03 |
| 64 | | 110 | 180 | 5000 | 5000 | 23667.2 | 8088.8 | | | 29.08 | 0.11 | 2.12 | 0.76 | 0.06 | 0.02 | 0.04 | 0.01 | 3 | 0.818 | 0.508 | 27.37 | 1.06 | 2.15E-03 | 0.08 | 0.01 | 0.10 | 2.64E-03 |
| 64 | | 95 | 165 | 5000 | 5000 | 16583.0 | 5873.9 | | | 25.93 | 0.17 | 1.45 | 0.79 | 0.04 | 0.02 | 0.03 | 0.02 | 3 | 0.750 | 0.508 | 25.09 | 1.03 | 1.60E-03 | 0.06 | 0.02 | 0.10 | 2.73E-03 |
| 64 | | 80 | 150 | 5000 | 5000 | 2113.1 | 771.1 | | | 24.07 | 0.25 | 1.15 | 0.82 | 0.03 | 0.02 | 0.02 | 0.02 | 3 | 0.682 | 0.508 | 22.81 | 1.06 | 1.40E-03 | 0.05 | 0.03 | 0.10 | 2.83E-03 |
| 64 | | 65 | 135 | 5000 | 5000 | 1723.7 | 700.0 | | | 21.75 | 0.35 | 1.10 | 0.85 | 0.03 | 0.02 | 0.02 | 0.02 | 3 | 0.614 | 0.508 | 20.53 | 1.06 | 1.49E-03 | 0.05 | 0.04 | 0.11 | 2.94E-03 |
| 64 | | 50 | 120 | 5000 | 5000 | 1873.1 | 687.4 | | | 20.05 | 0.22 | 1.69 | 0.91 | 0.05 | 0.03 | 0.03 | 0.02 | 3 | 0.545 | 0.508 | 18.25 | 1.10 | 2.57E-03 | 0.09 | 0.03 | 0.11 | 3.15E-03 |
| 64 | | 35 | 105 | 5000 | 5000 | 1800.7 | 745.3 | | | 17.61 | -0.08 | 2.12 | 1.03 | 0.06 | 0.03 | 0.04 | 0.02 | 3 | 0.477 | 0.508 | 15.97 | 1.10 | 3.68E-03 | 0.13 | -0.01 | 0.13 | 3.55E-03 |
| 64 | | 20 | 90 | 5000 | 5000 | 1879.8 | 656.5 | | | 15.81 | -0.46 | 2.76 | 1.36 | 0.08 | 0.04 | 0.05 | 0.03 | 3 | 0.409 | 0.508 | 13.69 | 1.16 | 5.59E-03 | 0.20 | -0.06 | 0.17 | 4.68E-03 |
| 71 | | 140 | 210 | 5000 | 5000 | 2408.5 | 915.8 | | | 32.02 | 0.03 | 2.87 | 0.77 | 0.08 | 0.02 | 0.06 | 0.02 | 3 | 0.955 | 0.345 | 31.94 | 1.00 | 2.49E-03 | 0.09 | 0.00 | 0.10 | 2.65E-03 |
| 71 | | 125 | 195 | 5000 | 5000 | 2249.9 | 692.1 | | | 30.31 | 0.22 | 2.79 | 0.64 | 0.08 | 0.02 | 0.05 | 0.01 | 3 | 0.886 | 0.345 | 29.65 | 1.02 | 2.61E-03 | 0.09 | 0.03 | 0.08 | 2.22E-03 |
| 71 | | 110 | 180 | 5000 | 5000 | 2396.4 | 670.6 | | | 27.86 | 0.12 | 2.08 | 0.66 | 0.06 | 0.02 | 0.04 | 0.01 | 3 | 0.818 | 0.345 | 27.37 | 1.02 | 2.11E-03 | 0.08 | 0.02 | 0.08 | 2.27E-03 |
| 71 | | 95 | 165 | 5000 | 5000 | 2245.5 | 692.0 | | | 26.06 | 0.14 | 1.60 | 0.68 | 0.04 | 0.02 | 0.03 | 0.01 | 3 | 0.750 | 0.345 | 25.09 | 1.04 | 1.77E-03 | 0.06 | 0.02 | 0.08 | 2.36E-03 |
| 71 | | 80 | 150 | 5000 | 5000 | 2032.7 | 725.3 | | | 23.82 | 0.23 | 1.03 | 0.75 | 0.03 | 0.02 | 0.02 | 0.01 | 3 | 0.682 | 0.345 | 22.81 | 1.04 | 1.25E-03 | 0.05 | 0.03 | 0.09 | 2.59E-03 |
| 71 | | 65 | 135 | 5000 | 5000 | 1818.0 | 703.1 | | | 22.07 | 0.29 | 1.23 | 0.81 | 0.03 | 0.02 | 0.02 | 0.02 | 3 | 0.614 | 0.345 | 20.53 | 1.07 | 1.66E-03 | 0.06 | 0.04 | 0.10 | 2.79E-03 |
| 57 | | 140 | 210 | 5000 | 5000 | 19186.2 | 569.0 | | | 31.97 | 0.25 | 3.03 | 0.58 | 0.08 | 0.02 | 0.06 | 0.01 | 3 | 0.955 | 0.671 | 31.94 | 1.00 | 2.63E-03 | 0.09 | 0.03 | 0.07 | 1.99E-03 |
| 57 | | 125 | 195 | 5000 | 5000 | 15926.5 | 595.0 | | | 30.06 | 0.17 | 2.64 | 0.67 | 0.07 | 0.02 | 0.05 | 0.01 | 3 | 0.886 | 0.671 | 29.65 | 1.01 | 2.46E-03 | 0.09 | 0.02 | 0.08 | 2.31E-03 |
| 57 | | 110 | 180 | 5000 | 5000 | 17180.4 | 665.9 | | | 28.27 | 0.15 | 2.05 | 0.69 | 0.06 | 0.02 | 0.04 | 0.01 | 3 | 0.818 | 0.671 | 27.37 | 1.03 | 2.07E-03 | 0.07 | 0.02 | 0.09 | 2.37E-03 |
| 57 | | 95 | 165 | 5000 | 5000 | 16946.5 | 624.1 | | | 25.97 | 0.15 | 1.48 | 0.71 | 0.04 | 0.02 | 0.03 | 0.01 | 3 | 0.750 | 0.671 | 25.09 | 1.03 | 1.64E-03 | 0.06 | 0.02 | 0.09 | 2.45E-03 |
| 57 | | 80 | 150 | 5000 | 5000 | 16872.2 | 667.4 | | | 23.97 | 0.22 | 1.04 | 0.74 | 0.03 | 0.02 | 0.02 | 0.01 | 3 | 0.682 | 0.671 | 22.81 | 1.05 | 1.27E-03 | 0.05 | 0.03 | 0.09 | 2.56E-03 |
| 57 | | 65 | 135 | 5000 | 5000 | 15135.2 | 663.4 | | | 22.03 | 0.28 | 1.12 | 0.78 | 0.03 | 0.02 | 0.02 | 0.02 | 3 | 0.614 | 0.671 | 20.53 | 1.07 | 1.52E-03 | 0.05 | 0.03 | 0.10 | 2.70E-03 |
| 94 | | -3 | 67 | #N/A | #N/A | #N/A | #N/A | | | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | cob2 | 0.305 | 0.311 | 10.19 | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A |
| 94 | | -9 | 61 | #N/A | #N/A | #N/A | #N/A | | | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | cob2 | 0.277 | 0.311 | 9.28 | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A |
| 94 | | -13 | 57 | #N/A | #N/A | #N/A | #N/A | | | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | cob2 | 0.259 | 0.311 | 8.67 | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A |
| 84 | | -3 | 67 | 141 | 5000 | 7.6 | 429.5 | | | 14.50 | 6.67 | 2.47 | 2.68 | 0.41 | 0.07 | 0.29 | 0.05 | cob2 | 0.305 | 0.709 | 10.19 | 1.42 | 4.02E-02 | 0.24 | 0.83 | 0.33 | 9.24E-03 |
| 84 | | -9 | 61 | #N/A | #N/A | #N/A | #N/A | | | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | cob2 | 0.277 | 0.709 | 9.28 | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A |
| 84 | | -15 | 55 | #N/A | #N/A | #N/A | #N/A | | | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | cob2 | 0.250 | 0.709 | 8.36 | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A |
| 131 | | -3 | 67 | #N/A | #N/A | #N/A | #N/A | | | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | cob1 | 0.305 | 0.865 | 10.19 | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A |
| 131 | | -9 | 61 | #N/A | #N/A | #N/A | #N/A | | | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | cob1 | 0.277 | 0.865 | 9.28 | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A |
| 131 | | -15 | 55 | #N/A | #N/A | #N/A | #N/A | | | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | cob1 | 0.250 | 0.865 | 8.36 | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A |
| 64 | | 5 | 75 | 5000 | 5000 | 1429.8 | 226.4 | | | 10.41 | 0.05 | 1.08 | 0.27 | 0.03 | 0.01 | 0.02 | 0.01 | 3 | 0.341 | 0.508 | 11.41 | 0.91 | 2.63E-03 | 0.09 | 0.01 | 0.03 | 9.26E-04 |
| 64 | | -5 | 65 | 5000 | 5000 | 656.9 | 535.2 | | | 9.42 | 0.13 | 1.74 | 0.24 | 0.05 | 0.01 | 0.03 | 0.00 | 3 | 0.295 | 0.508 | 9.88 | 0.95 | 4.89E-03 | 0.18 | 0.02 | 0.03 | 8.43E-04 |
| 64 | | -15 | 55 | 0 | 1 | #N/A | #N/A | | | #N/A | 11.77 | #N/A | 0.00 | #N/A | 0.00 | #N/A | 0.00 | 3 | 0.250 | 0.508 | 8.36 | #N/A | #N/A | #N/A | 1.47 | 0.00 | 0.00E+00 |
| 71 | | 50 | 120 | 5000 | 5000 | 15467.2 | 629.4 | | | 19.33 | 0.26 | 1.47 | 0.90 | 0.04 | 0.02 | 0.03 | 0.02 | 3 | 0.545 | 0.345 | 18.25 | 1.06 | 2.23E-03 | 0.08 | 0.03 | 0.11 | 3.10E-03 |
| 71 | | 35 | 105 | 5000 | 5000 | 12501.5 | 584.2 | | | 17.17 | -0.09 | 2.27 | 1.03 | 0.06 | 0.03 | 0.04 | 0.02 | 3 | 0.477 | 0.345 | 15.97 | 1.08 | 3.94E-03 | 0.14 | -0.01 | 0.13 | 3.56E-03 |
| 57 | | 50 | 120 | 5000 | 5000 | 1404.8 | 645.5 | | | 19.80 | 0.18 | 1.52 | 0.87 | 0.04 | 0.02 | 0.03 | 0.02 | 3 | 0.545 | 0.671 | 18.25 | 1.09 | 2.31E-03 | 0.08 | 0.02 | 0.11 | 2.99E-03 |
| 57 | | 35 | 105 | 5000 | 5000 | 1616.3 | 619.5 | | | 17.49 | -0.04 | 2.10 | 1.00 | 0.06 | 0.03 | 0.04 | 0.02 | 3 | 0.477 | 0.671 | 15.97 | 1.10 | 3.64E-03 | 0.13 | -0.01 | 0.12 | 3.44E-03 |

| | | | | | | |
|------------------------------|---------------|---|----------------|--------------|---------------------|----------|
| LDA measurement | | LDA code: 101 | rpm: 2787.3 | Perfd by: NM | Printed: 26/09/2008 | Page 1/3 |
| Build: 3 | ReΦ: 2.02E+06 | Date: 09/07/2003 | LDA1: VΦ | Comments: | | |
| LDA test #: B3run23-1 | Rez: 9.91E+04 | raw LDA file: \MCRLDA\Build3\B3run23-1.lda | LDA2: Vz | Comments: | | |
| Heat Transfer run: B3run23-1 | Ro: 0.93 | LDA list file: \MCRLDA\Build3\B3run23-1.xls | W (m/s): 19.12 | Comments: | | |



LDA measurement**Build:** 3**LDA test # :** B3run23-1**Heat Transfer run:** B3run23-1**LDA code:** 101**Re_φ:** 2.02E+06**Re_z:** 9.91E+04**Ro:** 0.93**rpm:** 2787.3**Date:** 09/07/2003**raw LDA file:** \MCRLDA\Build3\B3run23-1.lda**LDA list file:** \MCRLDA\Build3\B3run23-1.xls**Perfd by:** NM**LDA1:** Vφ**LDA2:** Vz**W (m/s):** 19.12**Printed:** 26/09/2008**Comments:** 0**Traverse corresp:** 0

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LDA measurement
Build: 3 **LDA code:** 101 **rpm:** 2787.3 **Perfd by:** NM **Printed:** 26/09/2008 **Page 3/3**
ReΦ: 2.02E+06 **Date:** 09/07/2003 **LDA1:** VΦ **Comments:** 0
LDA test # : B3run23-1 **Rez:** 9.91E+04 **raw LDA file:** \MCRLDA\Build3\B3run23-1.lda **LDA2:** Vz **Traverse corresp:** 0
Heat Transfer run: B3run23-1 **Ro:** 0.93 **LDA list file:** \MCRLDA\Build3\B3run23-1.xls **W (m/s):** 19.12

| X [mm] | correcte | Y [mm] | corrected Y / | | Data Rate(1) | Data Rate(2) | Validation | Validation | LDA1 | LDA2 | LDA1 | LDA2 | LDA1 | LDA2 | LDA1 | LDA2 | Cavity | r/lb | z/s | Ωr (m/s) | V _φ /Ωr | V _φ U _{95%} /Ωr | V _φ (RMS)/ | Vz/W | Vz(RMS)/ | VzU _{95%} /W | |
|--------|----------|--------|---------------|----------|--------------|--------------|------------|------------|--------|--------|-------|-------|------|------|------|------|--------|-------|-------|----------|--------------------|-------------------------------------|-----------------------|-------|----------|-----------------------|----------|
| | d X | | Radius (mm) | Count{1} | | | | | | | | | | | | | | | | | | | Count{2} | | #/s] | | #/s] |
| 64 | 140 | 210 | 5000 | 5000 | 2344.2 | 393.4 | | | 62.20 | -0.32 | 3.97 | 2.71 | 0.11 | 0.07 | 0.08 | 0.05 | 3 | 0.955 | 0.508 | 61.30 | 1.01 | 1.79E-03 | 0.06 | -0.02 | 0.14 | 3.92E-03 | |
| 64 | 125 | 195 | 5000 | 5000 | 2645.8 | 284.3 | | | 58.97 | 0.56 | 4.03 | 1.27 | 0.11 | 0.04 | 0.08 | 0.02 | 3 | 0.886 | 0.508 | 56.92 | 1.04 | 1.96E-03 | 0.07 | 0.03 | 0.07 | 1.84E-03 | |
| 64 | 110 | 180 | 5000 | 5000 | 2362.4 | 544.1 | | | 55.55 | 0.28 | 3.42 | 1.34 | 0.09 | 0.04 | 0.07 | 0.03 | 3 | 0.818 | 0.508 | 52.54 | 1.06 | 1.80E-03 | 0.07 | 0.01 | 0.07 | 1.94E-03 | |
| 64 | 95 | 165 | 5000 | 5000 | 2312.0 | 1030.2 | | | 51.73 | 0.16 | 2.82 | 1.46 | 0.08 | 0.04 | 0.06 | 0.03 | 3 | 0.750 | 0.508 | 48.16 | 1.07 | 1.62E-03 | 0.06 | 0.01 | 0.08 | 2.11E-03 | |
| 64 | 80 | 150 | 5000 | 5000 | 442.7 | 292.7 | | | 47.83 | 0.54 | 2.24 | 1.52 | 0.06 | 0.04 | 0.04 | 0.03 | 3 | 0.682 | 0.508 | 43.78 | 1.09 | 1.42E-03 | 0.05 | 0.03 | 0.08 | 2.20E-03 | |
| 64 | 65 | 135 | 5000 | 5000 | 539.3 | 429.1 | | | 43.90 | 0.59 | 2.32 | 1.78 | 0.06 | 0.05 | 0.05 | 0.03 | 3 | 0.614 | 0.508 | 39.40 | 1.11 | 1.63E-03 | 0.06 | 0.03 | 0.09 | 2.59E-03 | |
| 64 | 50 | 120 | 5000 | 5000 | 401.6 | 297.9 | | | 40.53 | 0.77 | 2.85 | 2.03 | 0.08 | 0.06 | 0.06 | 0.04 | 3 | 0.545 | 0.508 | 35.03 | 1.16 | 2.25E-03 | 0.08 | 0.04 | 0.11 | 2.94E-03 | |
| 64 | 35 | 105 | 5000 | 5000 | 439.2 | 429.5 | | | 36.34 | 0.18 | 3.99 | 2.50 | 0.11 | 0.07 | 0.08 | 0.05 | 3 | 0.477 | 0.508 | 30.65 | 1.19 | 3.61E-03 | 0.13 | 0.01 | 0.13 | 3.63E-03 | |
| 71 | 140 | 210 | 5000 | 5000 | 9160.3 | 1332.3 | | | 62.88 | 0.59 | 1.88 | 1.93 | 0.05 | 0.05 | 0.04 | 0.04 | 3 | 0.955 | 0.345 | 61.30 | 1.03 | 8.49E-04 | 0.03 | 0.03 | 0.10 | 2.80E-03 | |
| 71 | 125 | 195 | 5000 | 5000 | 447.9 | 160.1 | | | 58.57 | 0.51 | 3.74 | 1.71 | 0.10 | 0.05 | 0.07 | 0.03 | 3 | 0.886 | 0.345 | 56.92 | 1.03 | 1.82E-03 | 0.07 | 0.03 | 0.09 | 2.48E-03 | |
| 71 | 110 | 180 | 5000 | 2467 | 339.2 | 61.7 | | | 55.14 | -0.18 | 3.41 | 1.39 | 0.09 | 0.05 | 0.07 | 0.04 | 3 | 0.818 | 0.345 | 52.54 | 1.05 | 1.80E-03 | 0.06 | -0.01 | 0.07 | 2.86E-03 | |
| 71 | 95 | 165 | 5000 | 5000 | 352.7 | 164.7 | | | 51.41 | 0.41 | 2.84 | 2.27 | 0.08 | 0.06 | 0.06 | 0.04 | 3 | 0.750 | 0.345 | 48.16 | 1.07 | 1.63E-03 | 0.06 | 0.02 | 0.12 | 3.29E-03 | |
| 71 | 80 | 150 | 5000 | 5000 | 516.1 | 184.5 | | | 47.48 | 0.08 | 2.47 | 1.63 | 0.07 | 0.05 | 0.05 | 0.03 | 3 | 0.682 | 0.345 | 43.78 | 1.08 | 1.57E-03 | 0.06 | 0.00 | 0.09 | 2.37E-03 | |
| 71 | 65 | 135 | 5000 | 5000 | 589.6 | 354.8 | | | 44.08 | 0.43 | 2.35 | 1.85 | 0.07 | 0.05 | 0.05 | 0.04 | 3 | 0.614 | 0.345 | 39.40 | 1.12 | 1.66E-03 | 0.06 | 0.02 | 0.10 | 2.68E-03 | |
| 57 | 140 | 210 | 5000 | 876 | 867.9 | 14.6 | | | 62.11 | -0.44 | 4.18 | 2.67 | 0.12 | 0.18 | 0.08 | 0.13 | 3 | 0.955 | 0.671 | 61.30 | 1.01 | 1.89E-03 | 0.07 | -0.02 | 0.14 | 9.26E-03 | |
| 57 | 125 | 195 | 5000 | 5000 | 1174.5 | 137.2 | | | 58.56 | -0.02 | 3.97 | 1.20 | 0.11 | 0.03 | 0.08 | 0.02 | 3 | 0.886 | 0.671 | 56.92 | 1.03 | 1.93E-03 | 0.07 | 0.00 | 0.06 | 1.73E-03 | |
| 57 | 110 | 180 | 5000 | 5000 | 1250.3 | 264.4 | | | 55.68 | -0.03 | 3.45 | 1.30 | 0.10 | 0.04 | 0.07 | 0.03 | 3 | 0.818 | 0.671 | 52.54 | 1.06 | 1.82E-03 | 0.07 | 0.00 | 0.07 | 1.88E-03 | |
| 57 | 95 | 165 | 5000 | 5000 | 1019.0 | 245.3 | | | 51.66 | -0.23 | 2.91 | 1.46 | 0.08 | 0.04 | 0.06 | 0.03 | 3 | 0.750 | 0.671 | 48.16 | 1.07 | 1.68E-03 | 0.06 | -0.01 | 0.08 | 2.12E-03 | |
| 57 | 80 | 150 | 5000 | 5000 | 1061.4 | 115.1 | | | 47.26 | 0.41 | 2.51 | 1.52 | 0.07 | 0.04 | 0.05 | 0.03 | 3 | 0.682 | 0.671 | 43.78 | 1.08 | 1.59E-03 | 0.06 | 0.02 | 0.08 | 2.20E-03 | |
| 57 | 65 | 135 | 5000 | 5000 | 929.8 | 356.9 | | | 43.42 | 0.69 | 2.42 | 1.59 | 0.07 | 0.04 | 0.05 | 0.03 | 3 | 0.614 | 0.671 | 39.40 | 1.10 | 1.70E-03 | 0.06 | 0.04 | 0.08 | 2.30E-03 | |
| 94 | -3 | 67 | 2930 | 5000 | 24.5 | 31402.5 | | | 20.92 | 0.49 | 2.63 | 0.66 | 0.10 | 0.02 | 0.07 | 0.01 | cob2 | 0.305 | 0.311 | 19.56 | 1.07 | 4.87E-03 | 0.13 | 0.03 | 0.03 | 9.51E-04 | |
| 94 | -9 | 61 | 4834 | 5000 | 40.3 | 647.9 | | | 3.59 | 0.43 | 1.54 | 1.12 | 0.04 | 0.03 | 0.03 | 0.02 | cob2 | 0.277 | 0.311 | 17.80 | 0.20 | 2.44E-03 | 0.09 | 0.02 | 0.06 | 1.63E-03 | |
| 94 | -13 | 57 | #N/A | #N/A | #N/A | #N/A | | | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | cob2 | 0.259 | 0.311 | 16.64 | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | |
| 84 | -3 | 67 | 5000 | 5000 | 1428.1 | 779.7 | | | 18.17 | 0.62 | 2.31 | 0.65 | 0.06 | 0.02 | 0.05 | 0.01 | cob2 | 0.305 | 0.709 | 19.56 | 0.93 | 3.28E-03 | 0.12 | 0.03 | 0.03 | 9.46E-04 | |
| 84 | -9 | 61 | 5000 | 5000 | 258.6 | 320.0 | | | 3.17 | -0.12 | 1.87 | 0.74 | 0.05 | 0.02 | 0.04 | 0.01 | cob2 | 0.277 | 0.709 | 17.80 | 0.18 | 2.91E-03 | 0.10 | -0.01 | 0.04 | 1.07E-03 | |
| 84 | -15 | 55 | 133 | 5000 | 6.9 | 4201.4 | | | -0.61 | 0.35 | 0.26 | 1.01 | 0.05 | 0.03 | 0.03 | 0.02 | cob2 | 0.250 | 0.709 | 16.05 | -0.04 | 2.81E-03 | 0.02 | 0.02 | 0.05 | 1.46E-03 | |
| 125 | -3 | 67 | 5000 | 5000 | 115.9 | 48.6 | | | -0.52 | -12.35 | 6.19 | 13.45 | 0.17 | 0.37 | 0.12 | 0.26 | 2 | 0.305 | 0.260 | 19.56 | -0.03 | 8.77E-03 | 0.32 | -0.65 | 0.70 | 1.95E-02 | |
| 125 | -9 | 61 | 5000 | 5000 | 65.9 | 1106.6 | | | -11.80 | -6.24 | 18.11 | 22.87 | 0.50 | 0.63 | 0.35 | 0.45 | 2 | 0.277 | 0.260 | 17.80 | -0.66 | 2.82E-02 | 1.02 | -0.33 | 1.20 | 3.32E-02 | |
| 125 | -15 | 55 | 0 | 5000 | #N/A | 3466.7 | | | #N/A | -2.64 | #N/A | 18.92 | #N/A | 0.52 | #N/A | 0.37 | 2 | 0.250 | 0.260 | 16.05 | #N/A | #N/A | #N/A | #N/A | -0.14 | 0.99 | 2.74E-02 |
| 64 | 20 | 90 | 5000 | 5000 | 42.1 | 220.3 | | | 35.85 | 0.05 | 4.82 | 2.24 | 0.13 | 0.06 | 0.09 | 0.04 | 3 | 0.409 | 0.508 | 26.27 | 1.36 | 5.09E-03 | 0.18 | 0.00 | 0.12 | 3.24E-03 | |
| 64 | 5 | 75 | 5000 | 5000 | 1429.8 | 226.4 | | | 10.41 | 0.05 | 1.08 | 0.27 | 0.03 | 0.01 | 0.02 | 0.01 | 3 | 0.341 | 0.508 | 21.89 | 0.48 | 1.37E-03 | 0.05 | 0.00 | 0.01 | 3.89E-04 | |
| 64 | -5 | 65 | 26 | 5000 | 0.2 | 318.5 | | | 29.48 | 0.48 | 3.43 | 4.40 | 1.34 | 0.12 | 0.95 | 0.09 | 3 | 0.295 | 0.508 | 18.97 | 1.55 | 7.09E-02 | 0.18 | 0.03 | 0.23 | 6.38E-03 | |
| 64 | -15 | 55 | 0 | 1465 | #N/A | 12.2 | | | #N/A | 9.60 | #N/A | 5.30 | #N/A | 0.27 | #N/A | 0.19 | 3 | 0.250 | 0.508 | 16.05 | #N/A | #N/A | #N/A | #N/A | 0.50 | 0.28 | 1.42E-02 |
| 71 | 50 | 120 | 5000 | 4511 | 189.6 | 75.2 | | | 40.27 | 0.46 | 2.79 | 2.09 | 0.08 | 0.06 | 0.05 | 0.04 | 3 | 0.545 | 0.345 | 35.03 | 1.15 | 2.21E-03 | 0.08 | 0.02 | 0.11 | 3.19E-03 | |
| 71 | 35 | 105 | 5000 | 4528 | 138.8 | 75.5 | | | 36.99 | -0.26 | 3.81 | 2.71 | 0.11 | 0.08 | 0.07 | 0.06 | 3 | 0.477 | 0.345 | 30.65 | 1.21 | 3.44E-03 | 0.12 | -0.01 | 0.14 | 4.12E-03 | |
| 57 | 50 | 120 | 5000 | 5000 | 390.5 | 179.8 | | | 40.42 | 0.59 | 2.99 | 1.91 | 0.08 | 0.05 | 0.06 | 0.04 | 3 | 0.545 | 0.671 | 35.03 | 1.15 | 2.37E-03 | 0.09 | 0.03 | 0.10 | 2.77E-03 | |
| 57 | 35 | 105 | 5000 | 5000 | 365.5 | 124.4 | | | 36.32 | -0.17 | 3.90 | 2.38 | 0.11 | 0.07 | 0.08 | 0.05 | 3 | 0.477 | 0.671 | 30.65 | 1.19 | 3.53E-03 | 0.13 | -0.01 | 0.12 | 3.45E-03 | |

LDA measurement

Build: 3
LDA test # : B3run25-1
Heat Transfer run: B3run25-1

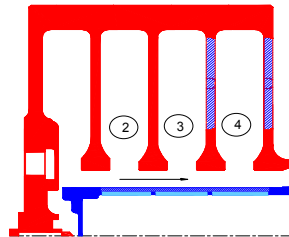
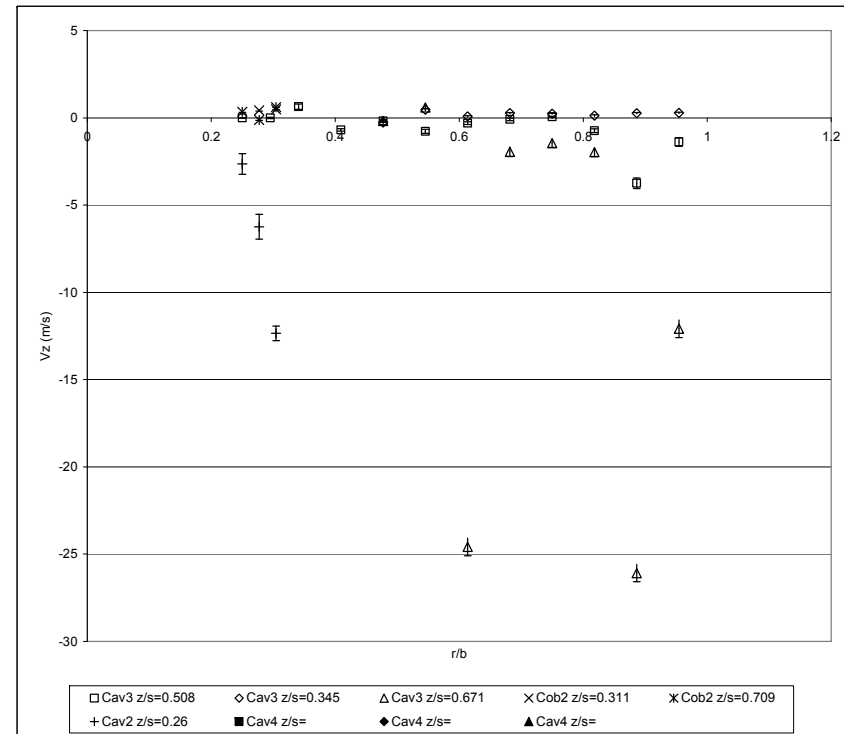
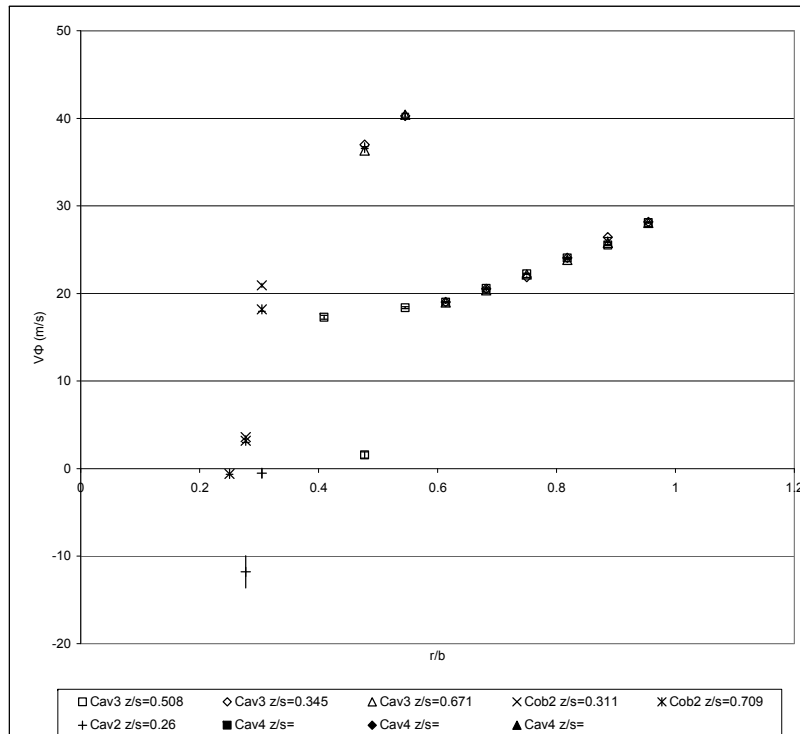
LDA code: 101
Re Φ : 1.15E+06
Rez: 1.71E+05
Ro: 2.84

rpm: 1546.8
Date: 27/08/2003
raw LDA file: \MCRLDA\Build3\B3run25-1.lda
LDA list file: \MCRLDA\Build3\B3run25-1.xls

Perfd by: NM
LDA1: V Φ
LDA2: Vz
W (m/s): 32.25

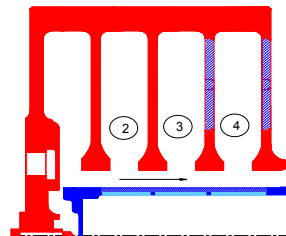
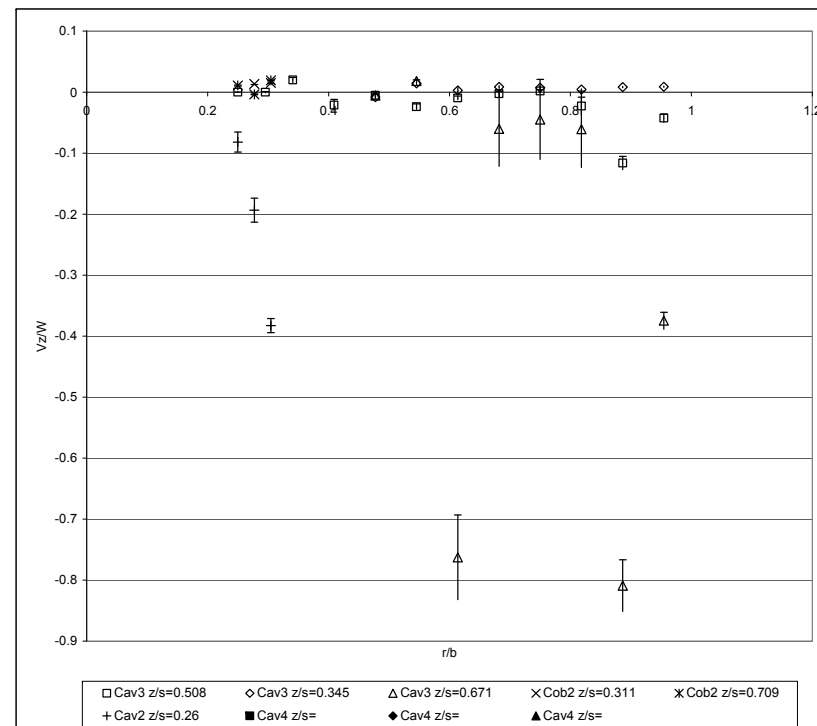
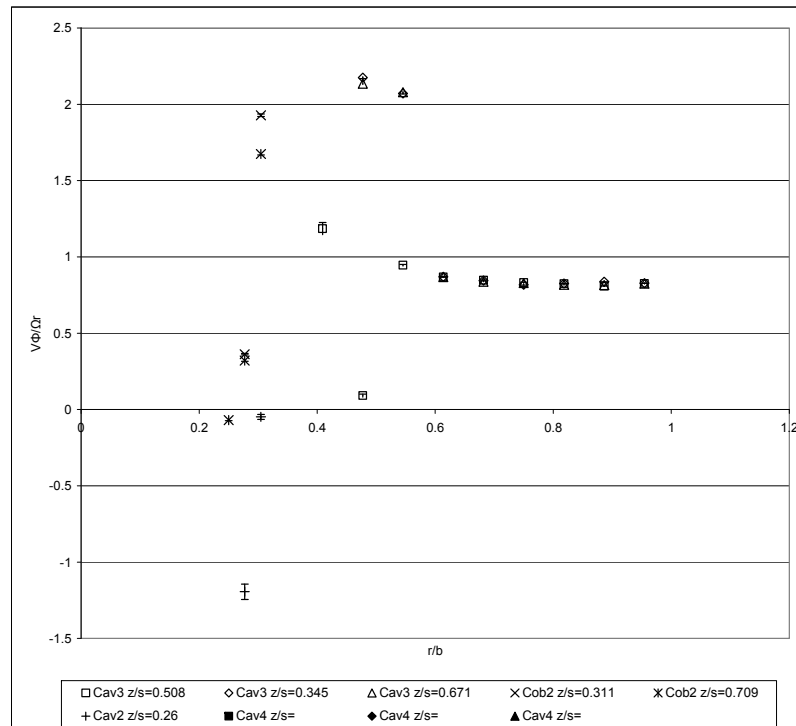
Printed: 26/09/2008
Comments: run OK
Traverse ref:

Page 1/3



LDA measurement**Build:** 3**LDA test # :** B3run25-1**Heat Transfer run:** B3run25-1**LDA code:** 101**Re_φ:** 1.15E+06**Re_z:** 1.71E+05**Ro:** 2.84**rpm:** 1546.8**Date:** 27/08/2003**raw LDA file:** \MCRLDA\Build3\B3run25-1.lda**LDA list file:** \MCRLDA\Build3\B3run25-1.xls**Perfd by:** NM**LDA1:** Vφ**LDA2:** Vz**W (m/s):** 32.25**Printed:** 26/09/2008**Comments:** run OK**Traverse corresp:** 0

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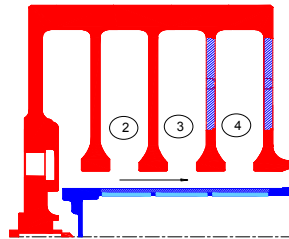
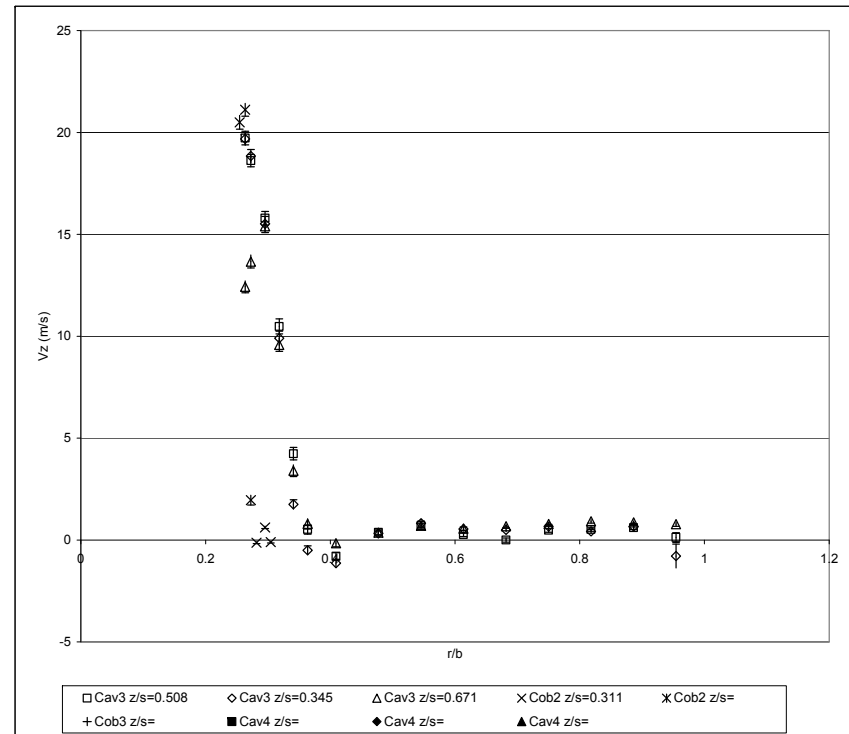
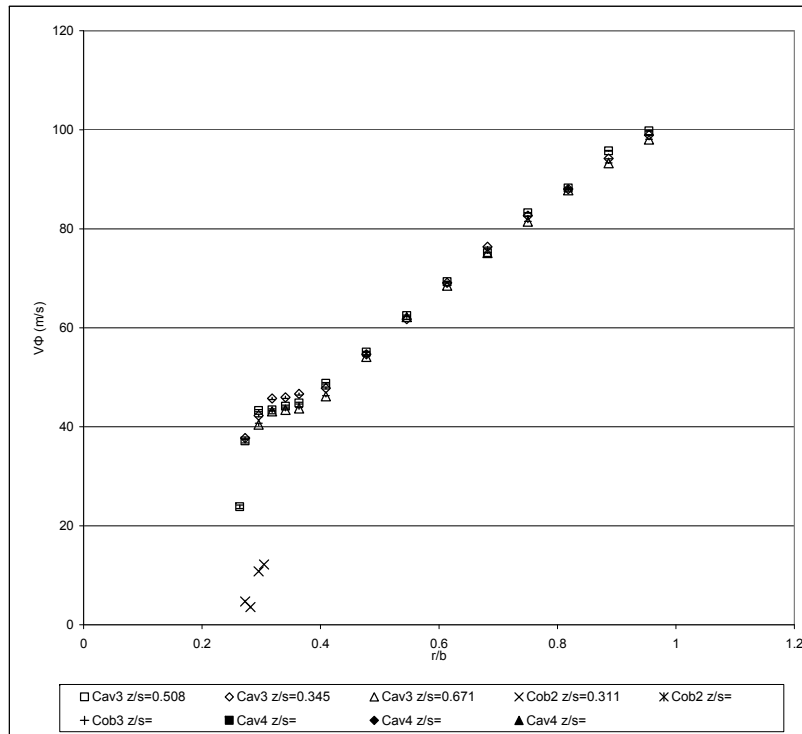
LDA measurement
Build: 3 **LDA code:** 101 **rpm:** 1546.8 **Perf by:** NM
LDA test # : B3run25-1 **ReΦ:** 1.15E+06 **Date:** 27/08/2003 **LDA1:** VΦ
Heat Transfer run: B3run25-1 **Rez:** 1.71E+05 **raw LDA file:** \MCRLDA\Build3\B3run25-1.lda **LDA2:** Vz
Ro: 2.84 **LDA list file:** \MCRLDA\Build3\B3run25-1.xls **W (m/s):** 32.25

Printed: 26/09/2008
 Comments: run OK
 Traverse corresp: 0

Page 3/3

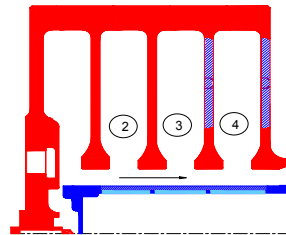
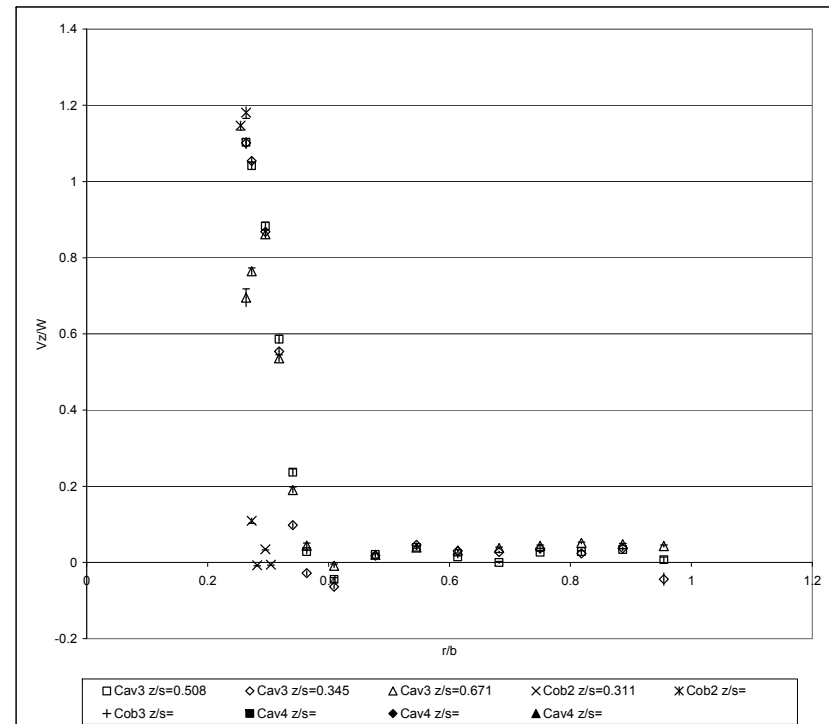
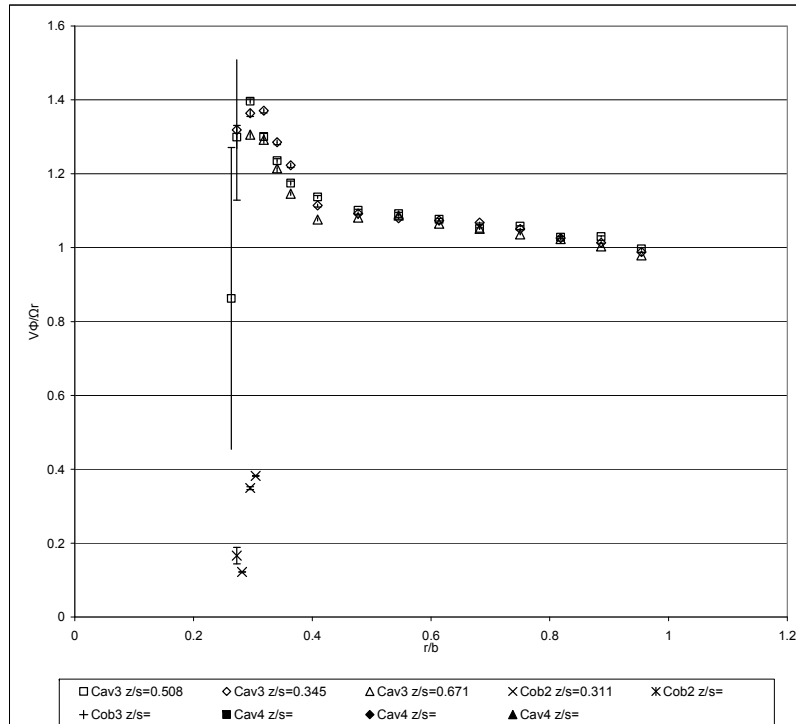
| X [mm] | correcte | Y [mm] | corrected Y / | Count{1} | Count{2} | Data Rate{1} | Data Rate{2} | Validation | Validation | LDA1 | LDA2 | LDA1 | LDA2 | LDA1 | LDA2 | LDA1 | LDA2 | Cavity | r/b | z/s | Ωr (m/s) | V _θ /Ωr | V _θ 95%/Ωr | V _θ (RMS)/ | Vz/W | Vz (RMS)/ | Vz95%/W |
|--------|----------|--------|---------------|----------|----------|--------------|--------------|------------|------------|----------|---------|------|------|------|------|----------|---------|--------|-------|----------|----------|--------------------|-----------------------|-----------------------|------|-----------|---------|
| | d X | | Radius (mm) | | | [#/s] | [#/s] | | | n{1} [%] | {2} [%] | Mean | Mean | RMS | RMS | MeanConf | MeanCon | | | | | | | RMSCConf | | RMSCConf | |
| 64 | 140 | 210 | 5000 | 5000 | 24481.9 | 1829.5 | 28.09 | -1.38 | 1.93 | 7.99 | 0.05 | 0.22 | 0.04 | 0.16 | 3 | 0.955 | 0.508 | 34.02 | 0.83 | 1.57E-03 | 0.06 | -0.04 | 0.25 | 6.87E-03 | | | |
| 64 | 125 | 195 | 5000 | 2957 | 1577.1 | 49.3 | 25.53 | -3.75 | 2.01 | 9.76 | 0.06 | 0.35 | 0.04 | 0.25 | 3 | 0.886 | 0.508 | 31.59 | 0.81 | 1.76E-03 | 0.06 | -0.12 | 0.30 | 1.09E-02 | | | |
| 64 | 110 | 180 | 5000 | 225 | 1310.1 | 3.8 | 24.05 | -0.74 | 1.90 | 3.56 | 0.05 | 0.47 | 0.04 | 0.33 | 3 | 0.818 | 0.508 | 29.16 | 0.82 | 1.81E-03 | 0.07 | -0.02 | 0.11 | 1.45E-02 | | | |
| 64 | 95 | 165 | 5000 | 539 | 1159.7 | 9.0 | 22.25 | 0.07 | 1.58 | 0.75 | 0.04 | 0.06 | 0.03 | 0.04 | 3 | 0.750 | 0.508 | 26.73 | 0.83 | 1.64E-03 | 0.06 | 0.00 | 0.02 | 1.97E-03 | | | |
| 64 | 80 | 150 | 2349 | 485 | 117.6 | 8.1 | 20.58 | -0.09 | 1.42 | 2.27 | 0.06 | 0.20 | 0.04 | 0.14 | 3 | 0.682 | 0.508 | 24.30 | 0.85 | 2.36E-03 | 0.06 | 0.00 | 0.07 | 6.27E-03 | | | |
| 64 | 65 | 135 | 4255 | 2776 | 212.8 | 46.3 | 19.01 | -0.30 | 1.48 | 2.48 | 0.04 | 0.09 | 0.03 | 0.07 | 3 | 0.614 | 0.508 | 21.87 | 0.87 | 2.03E-03 | 0.07 | -0.01 | 0.08 | 2.86E-03 | | | |
| 64 | 50 | 120 | 1645 | 2412 | 82.3 | 40.3 | 18.38 | -0.77 | 2.05 | 3.79 | 0.10 | 0.15 | 0.07 | 0.11 | 3 | 0.545 | 0.508 | 19.44 | 0.95 | 5.11E-03 | 0.11 | -0.02 | 0.12 | 4.69E-03 | | | |
| 64 | 35 | 105 | 5000 | 1300 | 5020.5 | 21.7 | 1.56 | -0.17 | 5.99 | 2.22 | 0.17 | 0.12 | 0.12 | 0.09 | 3 | 0.477 | 0.508 | 17.01 | 0.09 | 9.77E-03 | 0.35 | -0.01 | 0.07 | 3.75E-03 | | | |
| 71 | 140 | 210 | 5000 | 5000 | 33613.1 | 19024.5 | 28.16 | 0.29 | 1.34 | 1.11 | 0.04 | 0.03 | 0.03 | 0.02 | 3 | 0.955 | 0.345 | 34.02 | 0.83 | 1.09E-03 | 0.04 | 0.01 | 0.03 | 9.52E-04 | | | |
| 71 | 125 | 195 | 5000 | 5000 | 5628.0 | 7110.6 | 26.43 | 0.27 | 0.87 | 1.28 | 0.02 | 0.04 | 0.02 | 0.03 | 3 | 0.886 | 0.345 | 31.59 | 0.84 | 7.67E-04 | 0.03 | 0.01 | 0.04 | 1.10E-03 | | | |
| 71 | 110 | 180 | 3964 | 5000 | 66.1 | 173.8 | 24.08 | 0.13 | 1.55 | 1.48 | 0.05 | 0.04 | 0.03 | 0.03 | 3 | 0.818 | 0.345 | 29.16 | 0.83 | 1.66E-03 | 0.05 | 0.00 | 0.05 | 1.28E-03 | | | |
| 71 | 95 | 165 | 3637 | 179 | 60.7 | 3.0 | 21.86 | 0.24 | 1.50 | 0.73 | 0.05 | 0.11 | 0.03 | 0.08 | 3 | 0.750 | 0.345 | 26.73 | 0.82 | 1.83E-03 | 0.06 | 0.01 | 0.02 | 3.33E-03 | | | |
| 71 | 80 | 150 | 3911 | 589 | 65.2 | 9.9 | 20.54 | 0.27 | 1.46 | 0.87 | 0.05 | 0.07 | 0.03 | 0.05 | 3 | 0.682 | 0.345 | 24.30 | 0.85 | 1.88E-03 | 0.06 | 0.01 | 0.03 | 2.17E-03 | | | |
| 71 | 65 | 135 | 1548 | 171 | 25.8 | 2.9 | 19.03 | 0.09 | 1.45 | 1.08 | 0.07 | 0.16 | 0.05 | 0.11 | 3 | 0.614 | 0.345 | 21.87 | 0.87 | 3.31E-03 | 0.07 | 0.00 | 0.03 | 5.03E-03 | | | |
| 57 | 140 | 210 | 5000 | 5000 | 2062.0 | 1602.4 | 28.06 | -12.09 | 1.91 | 16.14 | 0.05 | 0.45 | 0.04 | 0.32 | 3 | 0.955 | 0.671 | 34.02 | 0.82 | 1.55E-03 | 0.06 | -0.37 | 0.50 | 1.39E-02 | | | |
| 57 | 125 | 195 | 5000 | 526 | 1124.1 | 8.8 | 25.78 | -26.09 | 1.98 | 15.98 | 0.05 | 1.37 | 0.04 | 0.97 | 3 | 0.886 | 0.671 | 31.59 | 0.82 | 1.73E-03 | 0.06 | -0.81 | 0.50 | 4.24E-02 | | | |
| 57 | 110 | 180 | 5000 | 63 | 979.0 | 1.1 | 23.84 | -1.96 | 1.75 | 8.10 | 0.05 | 2.02 | 0.03 | 1.43 | 3 | 0.818 | 0.671 | 29.16 | 0.82 | 1.66E-03 | 0.06 | -0.06 | 0.25 | 6.25E-02 | | | |
| 57 | 95 | 165 | 5000 | 52 | 705.8 | 0.9 | 22.17 | -1.45 | 1.56 | 7.72 | 0.04 | 2.12 | 0.03 | 1.50 | 3 | 0.750 | 0.671 | 26.73 | 0.83 | 1.62E-03 | 0.06 | -0.04 | 0.24 | 6.57E-02 | | | |
| 57 | 80 | 150 | 5000 | 75 | 649.0 | 1.3 | 20.37 | -1.94 | 1.57 | 8.69 | 0.04 | 1.98 | 0.03 | 1.40 | 3 | 0.682 | 0.671 | 24.30 | 0.84 | 1.79E-03 | 0.06 | -0.06 | 0.27 | 6.14E-02 | | | |
| 57 | 65 | 135 | 5000 | 201 | 166.5 | 3.4 | 18.97 | -24.59 | 2.43 | 16.16 | 0.07 | 2.24 | 0.05 | 1.58 | 3 | 0.614 | 0.671 | 21.87 | 0.87 | 3.08E-03 | 0.11 | -0.76 | 0.50 | 6.94E-02 | | | |
| 94 | -3 | 67 | 2930 | 5000 | 24.5 | 31402.5 | 20.92 | 0.49 | 2.63 | 0.66 | 0.10 | 0.02 | 0.07 | 0.01 | cob2 | 0.305 | 0.311 | 10.85 | 1.93 | 8.78E-03 | 0.24 | 0.02 | 0.02 | 5.64E-04 | | | |
| 94 | -9 | 61 | 4834 | 5000 | 40.3 | 647.9 | 3.59 | 0.43 | 1.54 | 1.12 | 0.04 | 0.03 | 0.03 | 0.02 | cob2 | 0.277 | 0.311 | 9.88 | 0.36 | 4.39E-03 | 0.16 | 0.01 | 0.03 | 9.65E-04 | | | |
| 94 | -13 | 57 | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | cob2 | 0.259 | 0.311 | 9.23 | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | #N/A | | |
| 84 | -3 | 67 | 5000 | 5000 | 1428.1 | 779.7 | 18.17 | 0.62 | 2.31 | 0.65 | 0.06 | 0.02 | 0.05 | 0.01 | cob2 | 0.305 | 0.709 | 10.85 | 1.67 | 5.91E-03 | 0.21 | 0.02 | 0.02 | 5.61E-04 | | | |
| 84 | -9 | 61 | 5000 | 5000 | 258.6 | 320.0 | 3.17 | -0.12 | 1.87 | 0.74 | 0.05 | 0.02 | 0.04 | 0.01 | cob2 | 0.277 | 0.709 | 9.88 | 0.32 | 5.24E-03 | 0.19 | 0.00 | 0.02 | 6.34E-04 | | | |
| 84 | -15 | 55 | 133 | 5000 | 6.9 | 4201.4 | -0.61 | 0.35 | 0.26 | 1.01 | 0.05 | 0.03 | 0.03 | 0.02 | cob2 | 0.250 | 0.709 | 8.91 | -0.07 | 5.07E-03 | 0.03 | 0.01 | 0.03 | 8.65E-04 | | | |
| 125 | -3 | 67 | 5000 | 5000 | 115.9 | 48.6 | -0.52 | -12.35 | 6.19 | 13.45 | 0.17 | 0.37 | 0.12 | 0.26 | 2 | 0.305 | 0.260 | 10.85 | -0.05 | 1.58E-02 | 0.57 | -0.38 | 0.42 | 1.16E-02 | | | |
| 125 | -9 | 61 | 5000 | 5000 | 65.9 | 1106.6 | -11.80 | -6.24 | 18.11 | 22.87 | 0.50 | 0.63 | 0.35 | 0.45 | 2 | 0.277 | 0.260 | 9.88 | -1.19 | 5.08E-02 | 1.83 | -0.19 | 0.71 | 1.97E-02 | | | |
| 125 | -15 | 55 | 0 | 5000 | #N/A | 3466.7 | #N/A | -2.64 | #N/A | 18.92 | #N/A | 0.52 | #N/A | 0.37 | 2 | 0.250 | 0.260 | 8.91 | #N/A | #N/A | #N/A | -0.08 | 0.59 | 1.63E-02 | | | |
| 64 | 20 | 90 | 113 | 212 | 1.9 | 3.6 | 17.28 | -0.69 | 3.11 | 2.25 | 0.58 | 0.30 | 0.41 | 0.22 | 3 | 0.409 | 0.508 | 14.58 | 1.19 | 3.95E-02 | 0.21 | -0.02 | 0.07 | 9.43E-03 | | | |
| 64 | 5 | 75 | 0 | 5000 | #N/A | 745.7 | #N/A | 0.64 | #N/A | 4.80 | #N/A | 0.13 | #N/A | 0.09 | 3 | 0.341 | 0.508 | 12.15 | #N/A | #N/A | #N/A | 0.02 | 0.15 | 4.13E-03 | | | |
| 64 | -5 | 65 | 0 | 5000 | #N/A | 41222.0 | #N/A | 0.00 | #N/A | 1.22 | #N/A | 0.03 | #N/A | 0.02 | 3 | 0.295 | 0.508 | 10.53 | #N/A | #N/A | #N/A | 0.00 | 0.04 | 1.05E-03 | | | |
| 64 | -15 | 55 | 0 | 5000 | #N/A | 4183.2 | #N/A | -0.01 | #N/A | 0.27 | #N/A | 0.01 | #N/A | 0.01 | 3 | 0.250 | 0.508 | 8.91 | #N/A | #N/A | #N/A | 0.00 | 0.01 | 2.31E-04 | | | |
| 71 | 50 | 120 | 5000 | 4511 | 189.6 | 75.2 | 40.27 | 0.46 | 2.79 | 2.09 | 0.08 | 0.06 | 0.05 | 0.04 | 3 | 0.545 | 0.345 | 19.44 | 2.07 | 3.98E-03 | 0.14 | 0.01 | 0.06 | 1.89E-03 | | | |
| 71 | 35 | 105 | 5000 | 4528 | 138.8 | 75.5 | 36.99 | -0.26 | 3.81 | 2.71 | 0.11 | 0.08 | 0.07 | 0.06 | 3 | 0.477 | 0.345 | 17.01 | 2.17 | 6.20E-03 | 0.22 | -0.01 | 0.08 | 2.44E-03 | | | |
| 57 | 50 | 120 | 5000 | 5000 | 390.5 | 179.8 | 40.42 | 0.59 | 2.99 | 1.91 | 0.08 | 0.05 | 0.06 | 0.04 | 3 | 0.545 | 0.671 | 19.44 | 2.08 | 4.27E-03 | 0.15 | 0.02 | 0.06 | 1.64E-03 | | | |
| 57 | 35 | 105 | 5000 | 5000 | 365.5 | 124.4 | 36.32 | -0.17 | 3.90 | 2.38 | 0.11 | 0.07 | 0.08 | 0.05 | 3 | 0.477 | 0.671 | 17.01 | 2.14 | 6.36E-03 | 0.23 | -0.01 | 0.07 | 2.05E-03 | | | |

| | | | | | | |
|------------------------------|----------------------|---|----------------|------------------|---------------------|----------|
| LDA measurement | | LDA code: 101 | rpm: 4552.2 | Perfd by: NM | Printed: 26/09/2008 | Page 1/3 |
| Build: 3 | Re Φ : 4.12E+06 | Date: 12/11/2003 | LDA1: V Φ | Comments: run OK | | |
| LDA test #: B3run28-1 | Rez: 1.16E+05 | raw LDA file: \MCRLDA\Build3\B3run28-1.lda | LDA2: Vz | Traversal ref: | | |
| Heat Transfer run: B3run28-1 | Ro: 0.54 | LDA list file: \MCRLDA\Build3\B3run28-1.xls | W (m/s): 17.88 | | | |



LDA measurement

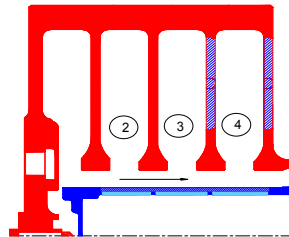
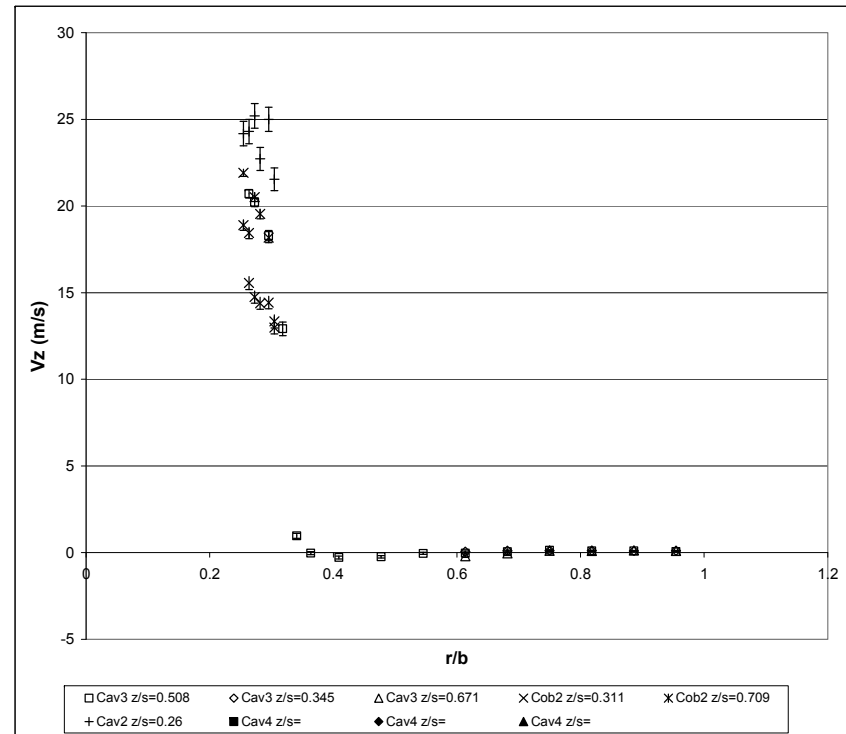
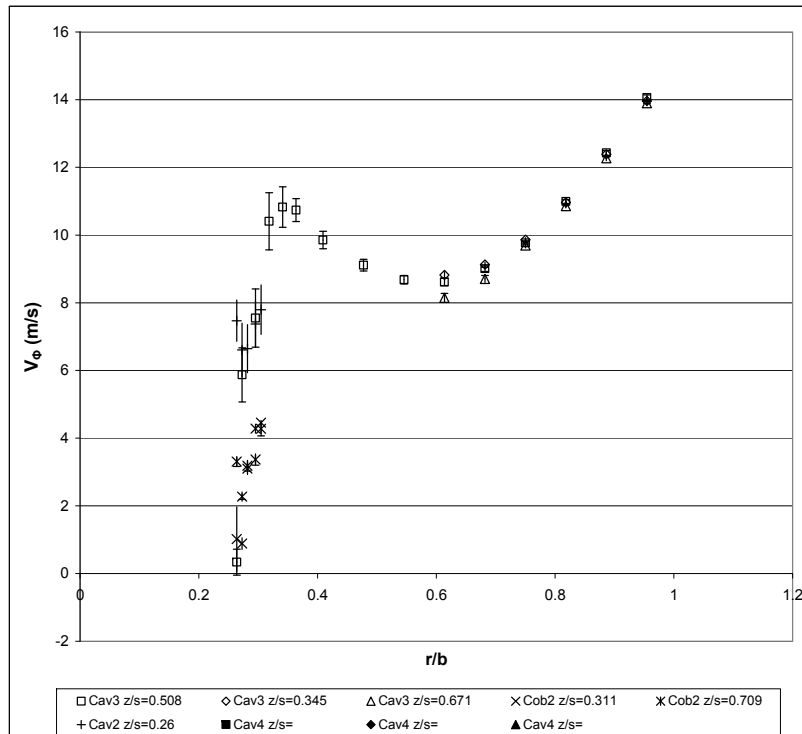
Build: 3 **LDA code:** 101 **rpm:** 4552.2 **Perfd by:** NM **Printed:** 26/09/2008 **Page 2/3**
LDA test # : B3run28-1 **Re_φ:** 4.12E+06 **Date:** 12/11/2003 **LDA1:** Vφ **Comments:** run OK
Heat Transfer run: B3run28-1 **Re_z:** 1.16E+05 **raw LDA file:** \MCRLDA\Build3\B3run28-1.lda **LDA2:** Vz **Traverse corresp:** 0
Ro: 0.54 **LDA list file:** \MCRLDA\Build3\B3run28-1.xls **W (m/s):** 17.88



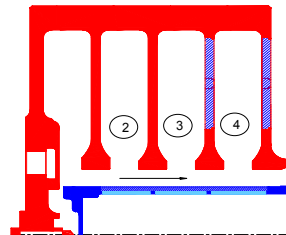
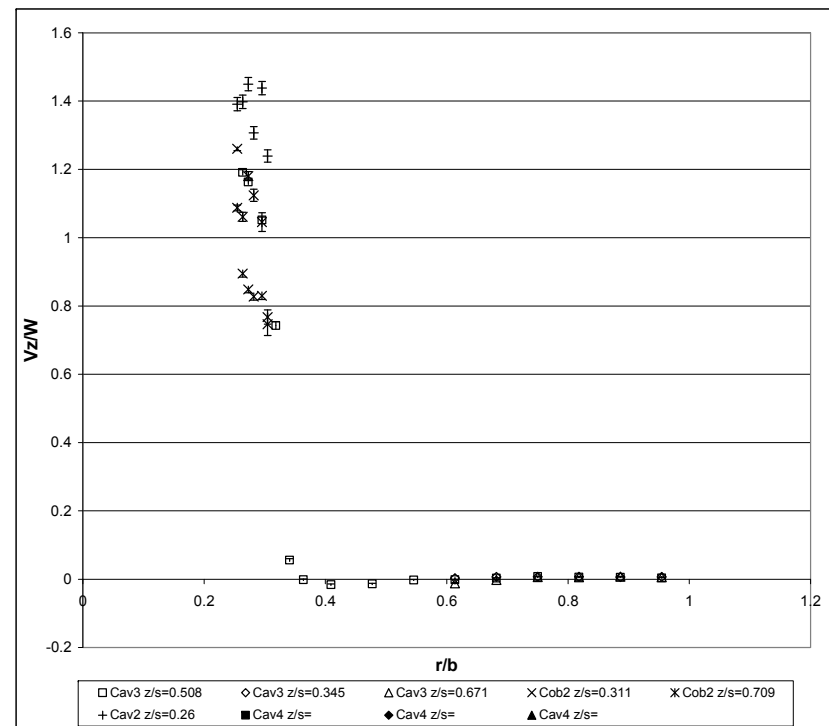
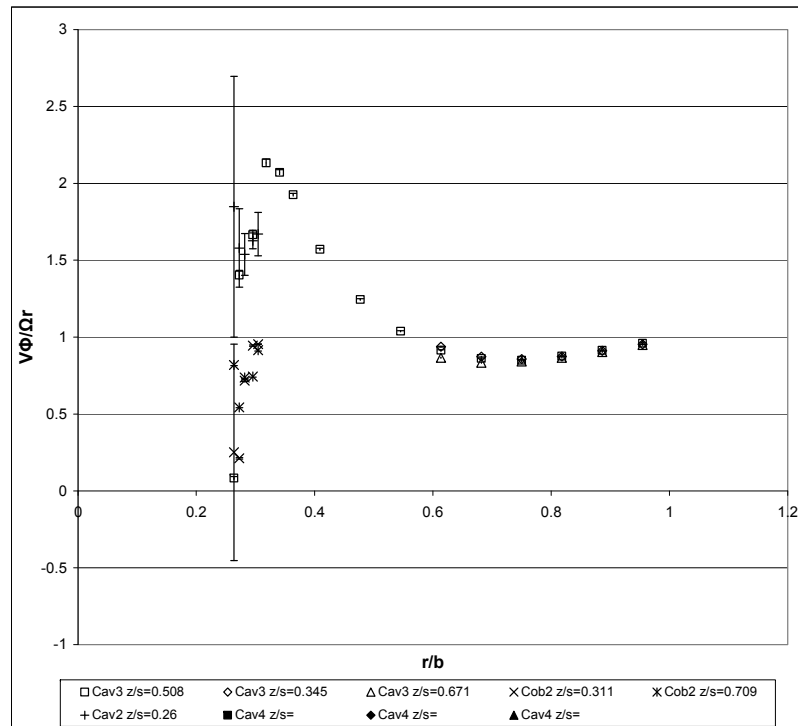
| | | | | | | |
|---------------------------|-----------|---------------|---|----------------|---------------------|----------|
| LDA measurement | | LDA code: 101 | rpm: 4552.2 | Perfd by: NM | Printed: 26/09/2008 | Page 3/3 |
| Build: | 3 | ReΦ: 4.12E+06 | Date: 12/11/2003 | LDA1: VΦ | Comments: run OK | |
| LDA test # : | B3run28-1 | Rez: 1.16E+05 | raw LDA file: \MCR\LDABuild3\B3run28-1.lda | LDA2: Vz | Traverse corresp: 0 | |
| Heat Transfer run: | B3run28-1 | Ro: 0.54 | LDA list file: \MCR\LDABuild3\B3run28-1.xls | W (m/s): 17.88 | | |

| X [mm] | correcte d X | Y [mm] | corrected Y / Radius (mm) | Count{1} | Count{2} | Data Rate{1} [#/s] | Data Rate{2} [#/s] | Validation n{1} [%] | Validation {2} [%] | LDA1 Mean [m/s] | LDA2 Mean [m/s] | LDA1 RMS [m/s] | LDA2 RMS [m/s] | LDA1 MeanConf [m/s] | LDA2 MeanCon f [m/s] | LDA1 RMSConf [m/s] | LDA2 RMSConf [m/s] | Cavity | r/b | z/s | Ωr (m/s) | V _g /Ωr | V _g 95%/Ωr | V _g (RMS)/ Ωr | Vz/W | Vz(RMS)/ W | Vz95%/W |
|--------|-----------------|--------|------------------------------|----------|----------|-----------------------|-----------------------|------------------------|-----------------------|-----------------------|-----------------------|----------------------|----------------------|---------------------------|----------------------------|--------------------------|--------------------------|--------|-------|-------|----------|--------------------|-----------------------|-----------------------------|-------|---------------|----------|
| 64 | | 140 | 210 | 5000 | 5000 | 10221.0 | 3203.8 | | | 99.79 | 0.13 | 9.62 | 4.21 | 0.27 | 0.12 | 0.19 | 0.08 | 3 | 0.955 | 0.508 | 100.11 | 1.00 | 2.66E-03 | 0.10 | 0.01 | 0.24 | 6.53E-03 |
| 64 | | 125 | 195 | 5000 | 5000 | 12325.4 | 4253.1 | | | 95.76 | 0.61 | 6.38 | 1.65 | 0.18 | 0.05 | 0.13 | 0.03 | 3 | 0.886 | 0.508 | 92.96 | 1.03 | 1.90E-03 | 0.07 | 0.03 | 0.09 | 2.57E-03 |
| 64 | | 110 | 180 | 5000 | 5000 | 13403.7 | 5287.8 | | | 88.24 | 0.51 | 4.52 | 1.51 | 0.13 | 0.04 | 0.09 | 0.03 | 3 | 0.818 | 0.508 | 85.81 | 1.03 | 1.46E-03 | 0.05 | 0.03 | 0.08 | 2.34E-03 |
| 64 | | 95 | 165 | 5000 | 5000 | 13316.7 | 5092.9 | | | 83.22 | 0.48 | 4.29 | 1.65 | 0.12 | 0.05 | 0.08 | 0.03 | 3 | 0.750 | 0.508 | 78.66 | 1.06 | 1.51E-03 | 0.05 | 0.03 | 0.09 | 2.56E-03 |
| 64 | | 80 | 150 | 5000 | 4941 | 28626.4 | 82.4 | | | 75.20 | 0.00 | 5.08 | 1.78 | 0.14 | 0.05 | 0.10 | 0.04 | 3 | 0.682 | 0.508 | 71.51 | 1.05 | 1.97E-03 | 0.07 | 0.00 | 0.10 | 2.77E-03 |
| 64 | | 65 | 135 | 5000 | 5000 | 28517.4 | 234.1 | | | 69.30 | 0.26 | 5.45 | 1.76 | 0.15 | 0.05 | 0.11 | 0.03 | 3 | 0.614 | 0.508 | 64.36 | 1.08 | 2.35E-03 | 0.08 | 0.01 | 0.10 | 2.72E-03 |
| 64 | | 50 | 120 | 5000 | 5000 | 12005.9 | 596.4 | | | 62.47 | 0.71 | 6.98 | 2.02 | 0.19 | 0.06 | 0.14 | 0.04 | 3 | 0.545 | 0.508 | 57.20 | 1.09 | 3.38E-03 | 0.12 | 0.04 | 0.11 | 3.13E-03 |
| 64 | | 35 | 105 | 5000 | 5000 | 396.0 | 1118.5 | | | 55.12 | 0.38 | 5.09 | 2.21 | 0.14 | 0.06 | 0.10 | 0.04 | 3 | 0.477 | 0.508 | 50.05 | 1.10 | 2.82E-03 | 0.10 | 0.02 | 0.12 | 3.43E-03 |
| 71 | | 140 | 210 | 5000 | 5000 | 3371.3 | 7047.3 | | | 98.93 | -0.79 | 5.22 | 10.37 | 0.14 | 0.29 | 0.10 | 0.20 | 3 | 0.955 | 0.345 | 100.11 | 0.99 | 1.44E-03 | 0.05 | -0.04 | 0.58 | 1.61E-02 |
| 71 | | 125 | 195 | 5000 | 5000 | 3076.8 | 6422.7 | | | 94.19 | 0.66 | 4.98 | 1.66 | 0.14 | 0.05 | 0.10 | 0.03 | 3 | 0.886 | 0.345 | 92.96 | 1.01 | 1.48E-03 | 0.05 | 0.04 | 0.09 | 2.57E-03 |
| 71 | | 110 | 180 | 5000 | 5000 | 2985.2 | 5562.2 | | | 88.02 | 0.42 | 5.04 | 1.69 | 0.14 | 0.05 | 0.10 | 0.03 | 3 | 0.818 | 0.345 | 85.81 | 1.03 | 1.63E-03 | 0.06 | 0.02 | 0.09 | 2.62E-03 |
| 71 | | 95 | 165 | 5000 | 5000 | 3270.3 | 7008.6 | | | 82.64 | 0.66 | 4.27 | 1.79 | 0.12 | 0.05 | 0.08 | 0.04 | 3 | 0.750 | 0.345 | 78.66 | 1.05 | 1.50E-03 | 0.05 | 0.04 | 0.10 | 2.77E-03 |
| 71 | | 80 | 150 | 5000 | 5000 | 3670.7 | 8791.2 | | | 76.35 | 0.49 | 3.34 | 1.56 | 0.09 | 0.04 | 0.07 | 0.03 | 3 | 0.682 | 0.345 | 71.51 | 1.07 | 1.30E-03 | 0.05 | 0.03 | 0.09 | 2.42E-03 |
| 71 | | 65 | 135 | 5000 | 5000 | 3320.9 | 8138.1 | | | 69.08 | 0.54 | 2.91 | 1.74 | 0.08 | 0.05 | 0.06 | 0.03 | 3 | 0.614 | 0.345 | 64.36 | 1.07 | 1.25E-03 | 0.05 | 0.03 | 0.10 | 2.70E-03 |
| 57 | | 140 | 210 | 5000 | 5000 | 1064.4 | 1723.1 | | | 98.02 | 0.77 | 5.25 | 1.92 | 0.15 | 0.05 | 0.10 | 0.04 | 3 | 0.955 | 0.671 | 100.11 | 0.98 | 1.45E-03 | 0.05 | 0.04 | 0.11 | 2.97E-03 |
| 57 | | 125 | 195 | 5000 | 5000 | 1465.3 | 2380.8 | | | 93.26 | 0.86 | 5.01 | 1.40 | 0.14 | 0.04 | 0.10 | 0.03 | 3 | 0.886 | 0.671 | 92.96 | 1.00 | 1.50E-03 | 0.05 | 0.05 | 0.08 | 2.18E-03 |
| 57 | | 110 | 180 | 5000 | 5000 | 1861.4 | 2430.6 | | | 87.78 | 0.91 | 4.74 | 1.50 | 0.13 | 0.04 | 0.09 | 0.03 | 3 | 0.818 | 0.671 | 85.81 | 1.02 | 1.53E-03 | 0.06 | 0.05 | 0.08 | 2.33E-03 |
| 57 | | 95 | 165 | 5000 | 5000 | 2097.1 | 2880.4 | | | 81.47 | 0.78 | 3.68 | 1.49 | 0.10 | 0.04 | 0.07 | 0.03 | 3 | 0.750 | 0.671 | 78.66 | 1.04 | 1.30E-03 | 0.05 | 0.04 | 0.08 | 2.31E-03 |
| 57 | | 80 | 150 | 5000 | 5000 | 1958.9 | 3122.6 | | | 75.14 | 0.68 | 3.20 | 1.52 | 0.09 | 0.04 | 0.06 | 0.03 | 3 | 0.682 | 0.671 | 71.51 | 1.05 | 1.24E-03 | 0.04 | 0.04 | 0.08 | 2.36E-03 |
| 57 | | 65 | 135 | 5000 | 5000 | 2950.1 | 3139.6 | | | 68.53 | 0.56 | 2.88 | 1.47 | 0.08 | 0.04 | 0.06 | 0.03 | 3 | 0.614 | 0.671 | 64.36 | 1.06 | 1.24E-03 | 0.04 | 0.03 | 0.08 | 2.28E-03 |
| 94 | | -3 | 67 | 5000 | 5000 | 54.3 | 1694.6 | | | 12.20 | -0.10 | 1.46 | 0.80 | 0.04 | 0.02 | 0.03 | 0.02 | cob2 | 0.305 | 0.311 | 31.94 | 0.38 | 1.27E-03 | 0.05 | -0.01 | 0.04 | 1.24E-03 |
| 94 | | -5 | 65 | 5000 | 5000 | 43.8 | 521.3 | | | 10.81 | 0.62 | 4.42 | 1.00 | 0.12 | 0.03 | 0.09 | 0.02 | cob2 | 0.295 | 0.311 | 30.99 | 0.35 | 3.95E-03 | 0.14 | 0.03 | 0.06 | 1.55E-03 |
| 94 | | -8 | 62 | 5000 | 5000 | 328.6 | 288.2 | | | 3.60 | -0.13 | 0.48 | 0.79 | 0.01 | 0.02 | 0.01 | 0.02 | cob2 | 0.282 | 0.311 | 29.56 | 0.12 | 4.50E-04 | 0.02 | -0.01 | 0.04 | 1.22E-03 |
| 94 | | -10 | 60 | 352 | 5000 | 2.0 | 288.1 | | | 4.75 | 1.96 | 6.08 | 4.28 | 0.64 | 0.12 | 0.45 | 0.08 | cob2 | 0.273 | 0.311 | 28.60 | 0.17 | 2.22E-02 | 0.21 | 0.11 | 0.24 | 6.63E-03 |
| 94 | | -12 | 58 | 0 | 1613 | #N/A | 13.5 | | | #N/A | 21.11 | #N/A | 5.57 | #N/A | 0.27 | #N/A | 0.19 | cob2 | 0.264 | 0.311 | 27.65 | #N/A | #N/A | #N/A | 1.18 | 0.31 | 1.52E-02 |
| 94 | | -14 | 56 | 0 | 3321 | #N/A | 27.7 | | | #N/A | 20.50 | #N/A | 6.01 | #N/A | 0.20 | #N/A | 0.14 | cob2 | 0.255 | 0.311 | 26.70 | #N/A | #N/A | #N/A | 1.15 | 0.34 | 1.14E-02 |
| 64 | | 20 | 90 | 5000 | 5000 | 290.5 | 215.8 | | | 48.79 | -0.80 | 6.10 | 3.02 | 0.17 | 0.08 | 0.12 | 0.06 | 3 | 0.409 | 0.508 | 42.90 | 1.14 | 3.94E-03 | 0.14 | -0.04 | 0.17 | 4.68E-03 |
| 64 | | 10 | 80 | 4277 | 5000 | 42.8 | 222.7 | | | 44.80 | 0.51 | 6.75 | 4.07 | 0.20 | 0.11 | 0.14 | 0.08 | 3 | 0.364 | 0.508 | 38.14 | 1.17 | 5.30E-03 | 0.18 | 0.03 | 0.23 | 6.32E-03 |
| 64 | | 5 | 75 | 5000 | 5000 | 70.1 | 263.8 | | | 44.17 | 4.24 | 7.06 | 5.41 | 0.20 | 0.15 | 0.14 | 0.11 | 3 | 0.341 | 0.508 | 35.75 | 1.24 | 5.47E-03 | 0.20 | 0.24 | 0.30 | 8.39E-03 |
| 64 | | 0 | 70 | 2508 | 5000 | 25.1 | 115.2 | | | 43.40 | 10.48 | 7.39 | 6.65 | 0.29 | 0.18 | 0.20 | 0.13 | 3 | 0.318 | 0.508 | 33.37 | 1.30 | 8.67E-03 | 0.22 | 0.59 | 0.37 | 1.03E-02 |
| 64 | | -5 | 65 | 5000 | 5000 | 62.3 | 104.1 | | | 43.27 | 15.79 | 6.56 | 6.06 | 0.18 | 0.17 | 0.13 | 0.12 | 3 | 0.295 | 0.508 | 30.99 | 1.40 | 5.87E-03 | 0.21 | 0.88 | 0.34 | 9.39E-03 |
| 64 | | -10 | 60 | 681 | 5000 | 4.5 | 187.3 | | | 37.17 | 18.62 | 11.96 | 5.38 | 0.90 | 0.15 | 0.64 | 0.11 | 3 | 0.273 | 0.508 | 28.60 | 1.30 | 3.14E-02 | 0.42 | 1.04 | 0.30 | 8.33E-03 |
| 64 | | -12 | 58 | 4 | 5000 | 0.1 | 142.1 | | | 23.85 | 19.72 | 9.98 | 5.96 | 11.29 | 0.17 | 7.98 | 0.12 | 3 | 0.264 | 0.508 | 27.65 | 0.86 | 4.08E-01 | 0.36 | 1.10 | 0.33 | 9.24E-03 |
| 71 | | 50 | 120 | 5000 | 5000 | 1474.2 | 652.3 | | | 61.75 | 0.82 | 3.81 | 1.94 | 0.11 | 0.05 | 0.07 | 0.04 | 3 | 0.545 | 0.345 | 57.20 | 1.08 | 1.85E-03 | 0.07 | 0.05 | 0.11 | 3.01E-03 |
| 71 | | 35 | 105 | 5000 | 5000 | 1860.7 | 930.4 | | | 54.67 | 0.32 | 5.05 | 2.30 | 0.14 | 0.06 | 0.10 | 0.05 | 3 | 0.477 | 0.345 | 50.05 | 1.09 | 2.80E-03 | 0.10 | 0.02 | 0.13 | 3.57E-03 |
| 57 | | 50 | 120 | 5000 | 5000 | 1343.4 | 664.6 | | | 62.18 | 0.70 | 3.76 | 1.91 | 0.10 | 0.05 | 0.07 | 0.04 | 3 | 0.545 | 0.671 | 57.20 | 1.09 | 1.82E-03 | 0.07 | 0.04 | 0.11 | 2.96E-03 |
| 57 | | 35 | 105 | 5000 | 5000 | 133.8 | 1185.8 | | | 54.13 | 0.36 | 5.19 | 2.14 | 0.14 | 0.06 | 0.10 | 0.04 | 3 | 0.477 | 0.671 | 50.05 | 1.08 | 2.88E-03 | 0.10 | 0.02 | 0.12 | 3.32E-03 |
| 71 | | 20 | 90 | 5000 | 5000 | 86.6 | 102.0 | | | 47.79 | -1.14 | 6.40 | 3.54 | 0.18 | 0.10 | 0.13 | 0.07 | 3 | 0.409 | 0.345 | 42.90 | 1.11 | 4.14E-03 | 0.15 | -0.06 | 0.20 | 5.50E-03 |
| 71 | | 10 | 80 | 4081 | 5000 | 68.0 | 93.3 | | | 46.64 | -0.51 | 7.59 | 3.94 | 0.23 | 0.11 | 0.16 | 0.08 | 3 | 0.364 | 0.345 | 38.14 | 1.22 | 6.11E-03 | 0.20 | -0.03 | 0.22 | 6.11E-03 |
| 71 | | 5 | 75 | 5000 | 5000 | 88.4 | 119.6 | | | 45.95 | 1.75 | 8.14 | 4.10 | 0.23 | 0.11 | 0.16 | 0.08 | 3 | 0.341 | 0.345 | 35.75 | 1.29 | 6.31E-03 | 0.23 | 0.10 | 0.23 | 6.36E-03 |
| 71 | | 0 | 70 | 5000 | 4902 | 106.3 | 81.7 | | | 45.73 | 9.90 | 6.98 | 6.32 | 0.19 | 0.18 | 0.14 | 0.13 | 3 | 0.318 | 0.345 | 33.37 | 1.37 | 5.80E-03 | 0.21 | 0.55 | 0.35 | 9.90E-03 |
| 71 | | -5 | 65 | 3050 | 5000 | 17.0 | 45.0 | | | 42.26 | 15.52 | 7.79 | 6.46 | 0.28 | 0.18 | 0.20 | 0.13 | 3 | 0.295 | 0.345 | 30.99 | 1.36 | 8.92E-03 | 0.25 | 0.87 | 0.36 | 1.00E-02 |
| 71 | | -10 | 60 | 7 | 4447 | 0.1 | 24.7 | | | 37.71 | 18.84 | 6.80 | 5.70 | 5.44 | 0.17 | 3.85 | 0.12 | 3 | 0.273 | 0.345 | 28.60 | 1.32 | 1.90E-01 | 0.24 | 1.05 | 0.32 | 9.38E-03 |
| 71 | | -12 | 58 | 0 | 1797 | #N/A | 10.0 | | | #N/A | 19.68 | #N/A | 5.55 | #N/A | 0.26 | #N/A | 0.18 | 3 | 0.264 | 0.345 | 27.65 | #N/A | #N/A | #N/A | 1.10 | 0.31 | 1.43E-02 |
| 57 | | 20 | 90 | 5000 | 5000 | 99.3 | 73.2 | | | 46.17 | -0.16 | 6.31 | 3.11 | 0.17 | 0.09 | 0.12 | 0.06 | 3 | 0.409 | 0.671 | 42.90 | 1.08 | 4.07E-03 | 0.15 | -0.01 | 0.17 | 4.82E-03 |
| 57 | | 10 | 80 | 5000 | 5000 | 115.3 | 80.6 | | | 43.71 | 0.79 | 6.55 | 4.48 | 0.18 | 0.12 | 0.13 | 0.09 | 3 | 0.364 | 0.671 | 38.14 | 1.15 | 4.76E-03 | 0.17 | 0.04 | 0.25 | 6.94E-03 |
| 57 | | 5 | 75 | 1429 | 5000 | 7.9 | 71.2 | | | 43.42 | 3.40 | 7.18 | 5.13 | 0.37 | 0.14 | 0.26 | 0.10 | 3 | 0.341 | 0.671 | 35.75 | 1.21 | 1.04E-02 | 0.20 | 0.19 | 0.29 | 7.96E-03 |

| | | | | | | |
|------------------------------|----------------------|---|----------------|------------------|---------------------|----------|
| LDA measurement | | LDA code: 101 | rpm: 665.6 | Perfd by: NM | Printed: 26/09/2008 | Page 1/3 |
| Build: 3 | Re Φ : 4.92E+05 | Date: 16/01/2004 | LDA1: V Φ | Comments: run OK | | |
| LDA test #: B3run32-1 | Rez: 9.18E+04 | raw LDA file: \MCRLDA\Build3\B3run32-1.lda | LDA2: Vz | Traverse ref: | | |
| Heat Transfer run: B3run32-1 | Ro: 3.56 | LDA list file: \MCRLDA\Build3\B3run32-1.xls | W (m/s): 17.38 | | | |



| | | | | | | |
|------------------------------|----------------------|---|----------------|---------------------|---------------------|----------|
| LDA measurement | | LDA code: 101 | rpm: 665.6 | Perfd by: NM | Printed: 26/09/2008 | Page 2/3 |
| Build: 3 | Re_ϕ : 4.92E+05 | Date: 16/01/2004 | LDA1: $V\phi$ | Comments: run OK | | |
| LDA test #: B3run32-1 | Re_z : 9.18E+04 | raw LDA file: \MCRLDA\Build3\B3run32-1.lda | LDA2: Vz | Traverse corresp: 0 | | |
| Heat Transfer run: B3run32-1 | Ro: 3.56 | LDA list file: \MCRLDA\Build3\B3run32-1.xls | W (m/s): 17.38 | | | |



| X [mm] | correcte | | corrected Y / | | Count{1} | Count{2} | Data Rate{1} | | Data Rate{2} | | Validation | | Mean [m/s] | LDA1 | LDA2 | LDA1 | LDA2 | LDA1 | LDA2 | LDA1 | LDA2 | V ₀ RMS/Ωr | V _z RMS/ W | |
|--------|----------|--------|---------------|----------|----------|----------|--------------|-------|--------------|----------|--------------------|-----------|------------|------|------|-------|-------|-------|------|----------|------|-----------------------|-----------------------|-----------|
| | d X | Y [mm] | Radius (mm) | Count{1} | | | Count{2} | [#/s] | [#/s] | n{1} [%] | Validation {2} [%] | RMS [m/s] | | | | | | | | | | | | RMS [m/s] |
| 64 | 140 | 210 | 5000 | 5000 | 87.5 | 83.9 | 14.06 | 0.07 | 1.13 | 0.55 | 0.03 | 0.02 | 0.02 | 0.01 | 3 | 0.955 | 0.508 | 14.64 | 0.96 | 2.13E-03 | 0.08 | 0.00 | 0.03 | 8.72E-04 |
| 64 | 125 | 195 | 5000 | 5000 | 131.2 | 116.3 | 12.43 | 0.10 | 0.90 | 0.34 | 0.02 | 0.01 | 0.02 | 0.01 | 3 | 0.886 | 0.508 | 13.59 | 0.91 | 1.83E-03 | 0.07 | 0.01 | 0.02 | 5.48E-04 |
| 64 | 110 | 180 | 5000 | 5000 | 860.3 | 621.7 | 10.99 | 0.10 | 0.69 | 0.37 | 0.02 | 0.01 | 0.01 | 0.01 | 3 | 0.818 | 0.508 | 12.55 | 0.88 | 1.53E-03 | 0.06 | 0.01 | 0.02 | 5.89E-04 |
| 64 | 95 | 165 | 5000 | 5000 | 82.1 | 75.2 | 9.77 | 0.14 | 0.70 | 0.48 | 0.02 | 0.01 | 0.01 | 0.01 | 3 | 0.750 | 0.508 | 11.50 | 0.85 | 1.68E-03 | 0.06 | 0.01 | 0.03 | 7.58E-04 |
| 64 | 80 | 150 | 5000 | 5000 | 274.9 | 325.7 | 9.01 | 0.05 | 0.87 | 0.71 | 0.02 | 0.02 | 0.02 | 0.01 | 3 | 0.682 | 0.508 | 10.46 | 0.86 | 2.31E-03 | 0.08 | 0.00 | 0.04 | 1.13E-03 |
| 64 | 65 | 135 | 5000 | 5000 | 105.0 | 120.2 | 8.61 | -0.03 | 0.99 | 0.90 | 0.03 | 0.02 | 0.02 | 0.02 | 3 | 0.614 | 0.508 | 9.41 | 0.92 | 2.92E-03 | 0.11 | 0.00 | 0.05 | 1.44E-03 |
| 64 | 50 | 120 | 5000 | 5000 | 208.5 | 243.7 | 8.68 | -0.04 | 1.05 | 1.04 | 0.03 | 0.03 | 0.02 | 0.02 | 3 | 0.545 | 0.508 | 8.36 | 1.04 | 3.49E-03 | 0.13 | 0.00 | 0.06 | 1.65E-03 |
| 64 | 35 | 105 | 5000 | 5000 | 186.2 | 2793.3 | 9.11 | -0.24 | 1.26 | 1.13 | 0.03 | 0.02 | 0.02 | 0.02 | 3 | 0.508 | 0.508 | 7.32 | 1.44 | 4.76E-03 | 0.17 | 0.01 | 0.07 | 1.91E-03 |
| 64 | 20 | 90 | 5000 | 5000 | 20.8 | 20.8 | 13.96 | 0.12 | 0.36 | 0.03 | 0.01 | 0.02 | 0.01 | 0.01 | 3 | 0.395 | 0.345 | 1.95 | 0.95 | 1.43E-03 | 0.08 | 0.01 | 0.02 | 4.87E-04 |
| 71 | 125 | 195 | 5000 | 5000 | 81.1 | 68.0 | 12.39 | 0.10 | 0.94 | 0.33 | 0.03 | 0.01 | 0.02 | 0.01 | 3 | 0.886 | 0.345 | 13.59 | 0.91 | 1.92E-03 | 0.07 | 0.01 | 0.02 | 5.35E-04 |
| 71 | 110 | 180 | 5000 | 5000 | 107.7 | 88.8 | 10.97 | 0.12 | 0.75 | 0.36 | 0.02 | 0.01 | 0.01 | 0.01 | 3 | 0.818 | 0.345 | 12.55 | 0.87 | 1.65E-03 | 0.06 | 0.01 | 0.02 | 5.67E-04 |
| 71 | 95 | 165 | 5000 | 5000 | 101.0 | 84.5 | 9.86 | 0.12 | 0.70 | 0.42 | 0.02 | 0.01 | 0.01 | 0.01 | 3 | 0.750 | 0.345 | 11.50 | 0.86 | 1.70E-03 | 0.08 | 0.01 | 0.02 | 6.73E-04 |
| 71 | 80 | 150 | 5000 | 5000 | 74.2 | 62.0 | 9.13 | 0.11 | 0.81 | 0.58 | 0.02 | 0.02 | 0.02 | 0.01 | 3 | 0.682 | 0.345 | 10.46 | 0.87 | 2.15E-03 | 0.08 | 0.01 | 0.03 | 9.31E-04 |
| 71 | 65 | 135 | 5000 | 5000 | 93.5 | 85.4 | 8.82 | 0.05 | 0.88 | 0.77 | 0.02 | 0.02 | 0.02 | 0.02 | 3 | 0.614 | 0.345 | 9.41 | 0.94 | 2.59E-03 | 0.09 | 0.00 | 0.04 | 1.23E-03 |
| 57 | 140 | 210 | 5000 | 5000 | 111.3 | 103.4 | 13.90 | 0.10 | 1.18 | 0.32 | 0.03 | 0.01 | 0.02 | 0.01 | 3 | 0.955 | 0.671 | 14.64 | 0.95 | 2.23E-03 | 0.08 | 0.01 | 0.02 | 5.14E-04 |
| 57 | 125 | 195 | 5000 | 5000 | 105.7 | 97.5 | 12.27 | 0.13 | 0.96 | 0.33 | 0.03 | 0.01 | 0.02 | 0.01 | 3 | | | | | | | | | |

LDA measurement

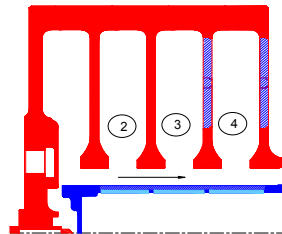
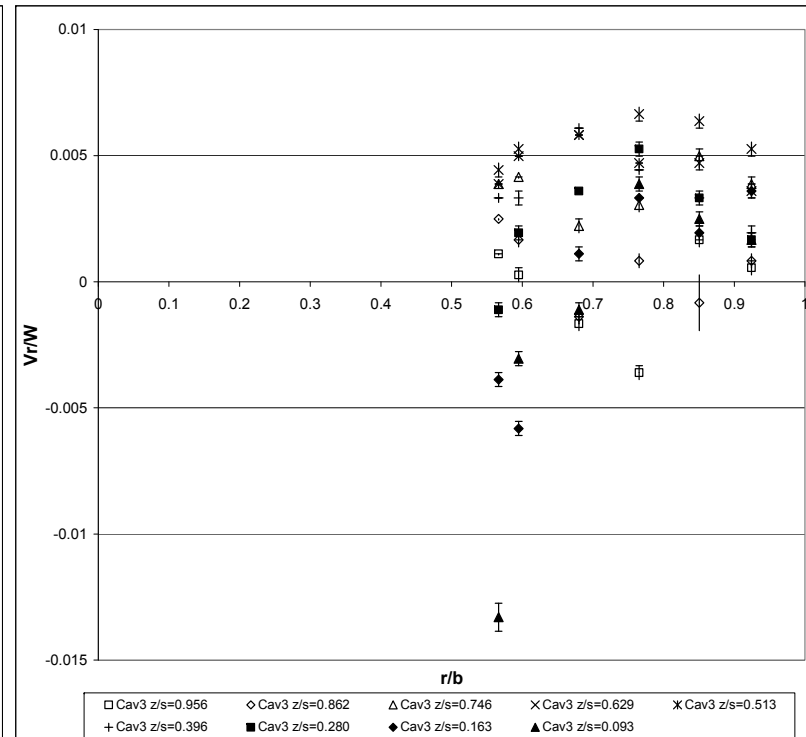
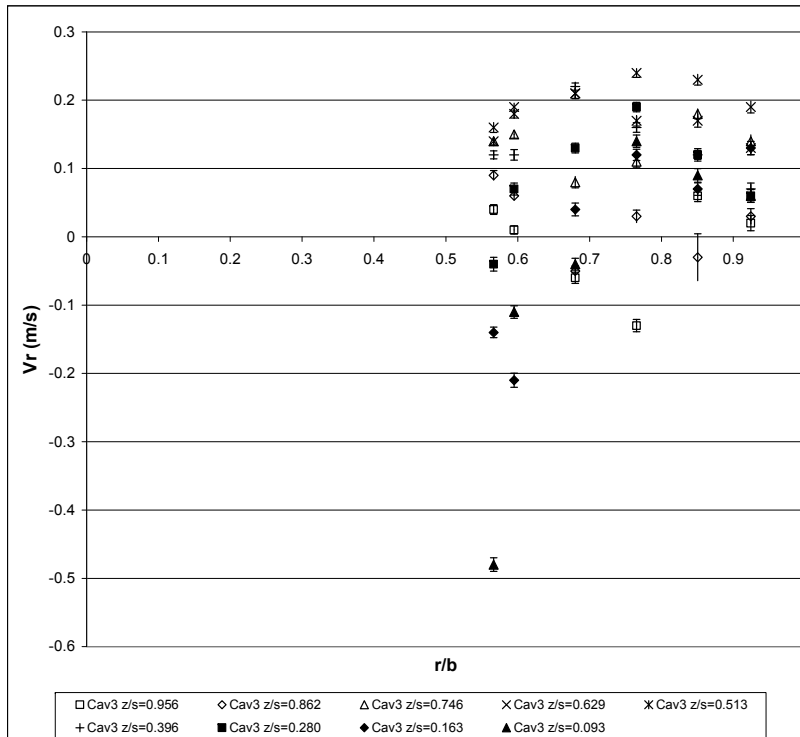
Build: 3
 LDA test #: B3run38-1
 Heat Transfer run: B3run38-1

LDA code: 110
 Re Φ : 3.99E+05
 Rez: 1.57E+05
 Ro: 7.51

rpm: 655.0
 Date: 27/09/2004
 raw LDA file: \MCRLDA\Build3\B3run38-1.lda
 LDA list file: \MCRLDA\Build3\B3run38-1.xls

Perfd by: NM
 LDA1: Vr
 LDA2: V Φ
 W (m/s): 36.10

Printed: 27/09/2008
 Comments: Probe angular error corrected -1.389deg
 Traverse ref:

CAVITY 3

LDA measurement

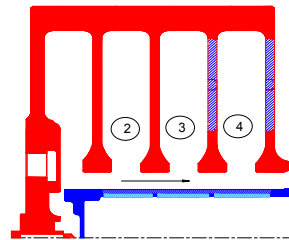
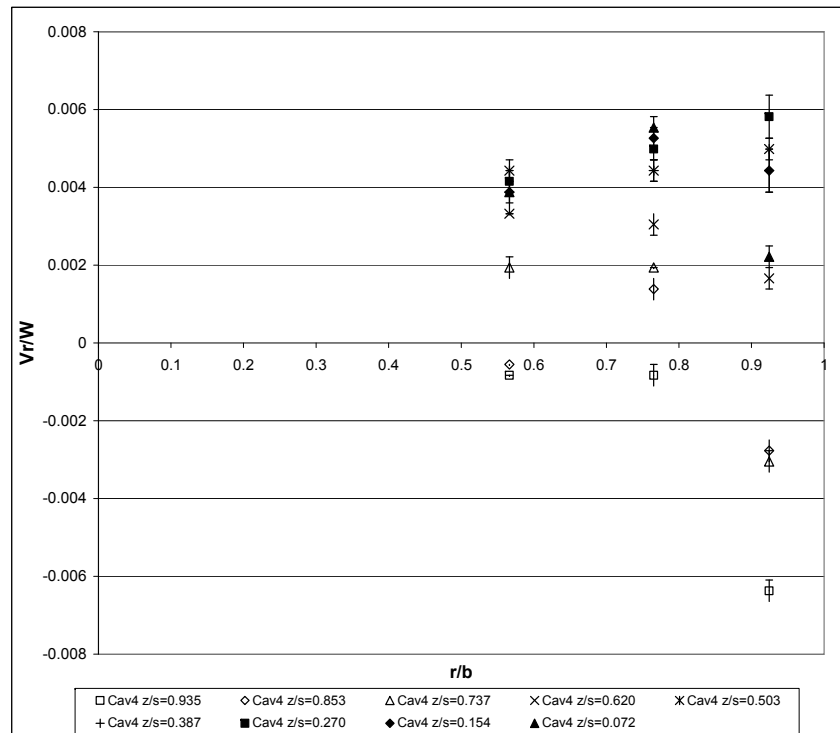
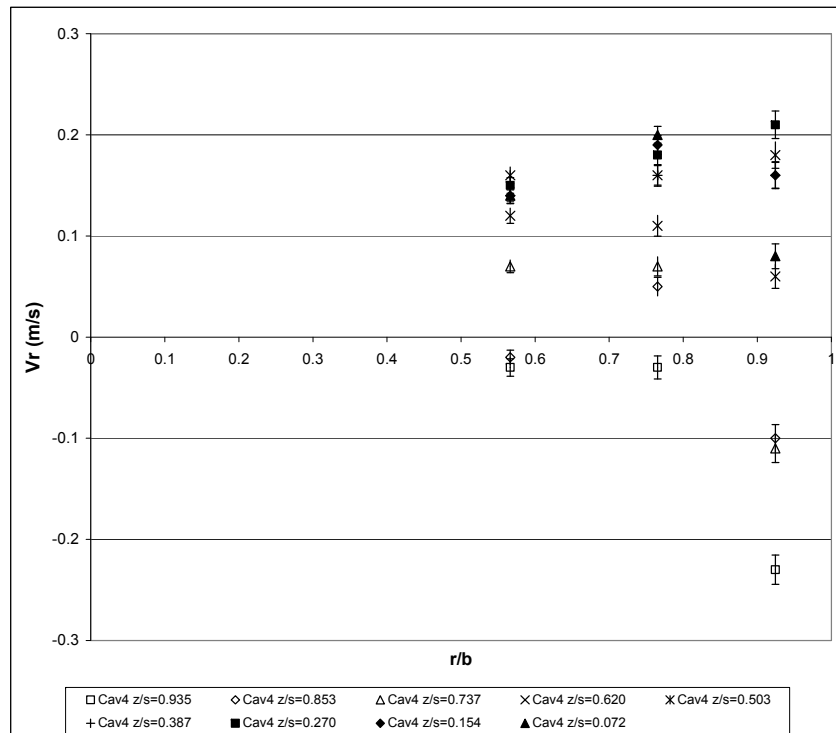
Build: 3
LDA test # : B3run38-1
Heat Transfer run: B3run38-1

LDA code: 110
Re Φ : 3.99E+05
Rez: 1.57E+05
Ro: 7.51

rpm: 655.0
Date: 27/09/2004
raw LDA file: \MCRLDA\Build3\B3run38-1.lda
LDA list file: \MCRLDA\Build3\B3run38-1.xls

Perfd by: NM
LDA1: Vr
LDA2: V Φ
W (m/s): 36.10

Printed: 27/09/2008 Page 1/2
Comments: Probe angular error corrected -1.389deg
Traverse ref:

CAVITY 4

| X [mm] | corrected d X | Y [mm] | corrected Y / Radius [mm] | | Count(1) | Count(2) | Data Rate(1) | | Data Rate(2) | | Validatio [Hz] | Validation [Hz] | LDA2 | | LDA1 | | LDA1 | | LDA2 | | LDA1 | | Cavity | r/b | z/s | Qr (m/s) | V _{Qr} /Qr | V _{BUSSV} /Qr | V _{Qr} | V _z /W | V _z /W | V _z /W | V _z /W |
|--------|------------------|--------|------------------------------|-------|----------|----------|---------------|--------------|---------------|--------------|-------------------|--------------------|---------------|-------------------|-------------------|-------------------|------------------|------------------|------|--|------|---|--------|-------|-------|----------|---------------------|------------------------|-----------------|-------------------|-------------------|-------------------|-------------------|
| | | | [Hz] | [Hz] | | | Mean [m/s] | RMS [m/s] | Mean [m/s] | RMS [m/s] | | | Mean [m/s] | MeanConf [m/s] | MeanConf [m/s] | MeanConf [m/s] | RMSConf [m/s] | RMSConf [m/s] | | | | | | | | | | | | | | | |
| 151 | 173.5 | 163 | 203.375 | 7142 | | | 246.6 | 45.61 | 0.02 | | | | 0.02 | | 0.40 | | 0.01 | | 0.01 | | 0.01 | 3 | 0.924 | 0.956 | 13.95 | 0.01 | 0.01 | 2.77E-04 | 0.01 | 2.77E-04 | | | |
| 155 | 177.5 | 163 | 203.375 | 4472 | | | 56.1 | 11.5 | 0.03 | | | | 0.41 | | 0.49 | | 0.01 | | 0.01 | | 0.01 | 3 | 0.924 | 0.956 | 13.95 | 0.01 | 0.01 | 2.77E-04 | 0.01 | 2.77E-04 | | | |
| 160 | 182.5 | 163 | 203.375 | 10054 | | | 141.3 | 73.83 | 0.14 | | | | 0.14 | | 0.31 | | 0.01 | | 0.00 | | 0.00 | 3 | 0.924 | 0.924 | 0.746 | 13.95 | 0.01 | 0.01 | 2.77E-04 | 0.01 | 2.77E-04 | | |
| 165 | 187.5 | 163 | 203.375 | 5266 | | | 596.7 | 67.60 | 0.19 | | | | 0.19 | | 0.31 | | 0.01 | | 0.01 | | 0.01 | 3 | 0.924 | 0.629 | 13.95 | 0.01 | 0.01 | 2.77E-04 | 0.01 | 2.77E-04 | | | |
| 170 | 192.5 | 163 | 203.375 | 11583 | | | 211.6 | 55.95 | 0.13 | | | | 0.13 | | 0.35 | | 0.01 | | 0.00 | | 0.00 | 3 | 0.924 | 0.513 | 13.95 | 0.01 | 0.01 | 2.77E-04 | 0.01 | 2.77E-04 | | | |
| 175 | 197.5 | 163 | 203.375 | 7101 | | | 540.7 | 56.17 | 0.17 | | | | 0.17 | | 0.32 | | 0.01 | | 0.01 | | 0.01 | 3 | 0.924 | 0.513 | 13.95 | 0.01 | 0.01 | 2.77E-04 | 0.01 | 2.77E-04 | | | |
| 180 | 202.5 | 163 | 203.375 | 7100 | | | 246.8 | 60.06 | 0.06 | | | | 0.06 | | 0.33 | | 0.01 | | 0.01 | | 0.01 | 3 | 0.924 | 0.284 | 13.95 | 0.00 | 0.01 | 2.77E-04 | 0.00 | 2.77E-04 | | | |
| 185 | 207.5 | 163 | 203.375 | 9604 | | | 53.4 | 60.77 | 0.13 | | | | 0.13 | | 0.35 | | 0.01 | | 0.00 | | 0.00 | 3 | 0.924 | 0.163 | 13.95 | 0.00 | 0.01 | 2.77E-04 | 0.00 | 2.77E-04 | | | |
| 188 | 210.5 | 163 | 203.375 | 2801 | | | 15.6 | 64.77 | 0.06 | | | | 0.06 | | 0.35 | | 0.01 | | 0.01 | | 0.01 | 3 | 0.924 | 0.093 | 13.95 | 0.00 | 0.01 | 2.77E-04 | 0.00 | 2.77E-04 | | | |
| 151 | 173.5 | 150 | 187.125 | 3550 | | | 489.3 | 62.28 | 0.17 | | | | 0.17 | | 0.28 | | 0.01 | | 0.01 | | 0.01 | 3 | 0.851 | 0.754 | 12.84 | 0.00 | 0.01 | 2.77E-04 | 0.00 | 2.77E-04 | | | |
| 155 | 177.5 | 150 | 187.125 | 3205 | | | 755.2 | 14.72 | -0.03 | | | | 0.14 | | 0.04 | | 0.03 | | 0.03 | | 0.03 | 3 | 0.851 | 0.861 | 12.84 | 0.00 | 0.03 | 1.11E-03 | 0.00 | 1.11E-03 | | | |
| 160 | 182.5 | 150 | 187.125 | 7053 | | | 555.1 | 61.83 | 0.18 | | | | 0.26 | | 0.01 | | 0.00 | | 0.03 | | 0.03 | 3 | 0.851 | 0.746 | 12.84 | 0.00 | 0.01 | 2.77E-04 | 0.00 | 2.77E-04 | | | |
| 165 | 187.5 | 150 | 187.125 | 10469 | | | 2020.6 | 60.32 | 0.23 | | | | 0.23 | | 0.28 | | 0.01 | | 0.00 | | 0.00 | 3 | 0.851 | 0.629 | 12.84 | 0.01 | 0.01 | 2.77E-04 | 0.01 | 2.77E-04 | | | |
| 170 | 192.5 | 150 | 187.125 | 7855 | | | 1669.1 | 60.73 | 0.17 | | | | 0.17 | | 0.24 | | 0.01 | | 0.01 | | 0.01 | 3 | 0.851 | 0.754 | 12.84 | 0.01 | 0.01 | 2.77E-04 | 0.01 | 2.77E-04 | | | |
| 175 | 197.5 | 150 | 187.125 | 6452 | | | 191.0 | 70.51 | 0.12 | | | | | | | | | | | | | | | | | | | | | | | | |

LDA measurement

Build: 3
LDA test # : B3run39-1
Heat Transfer run: B3run39-1

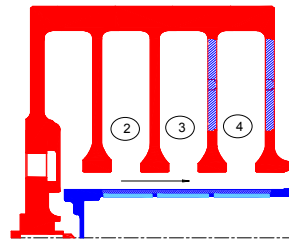
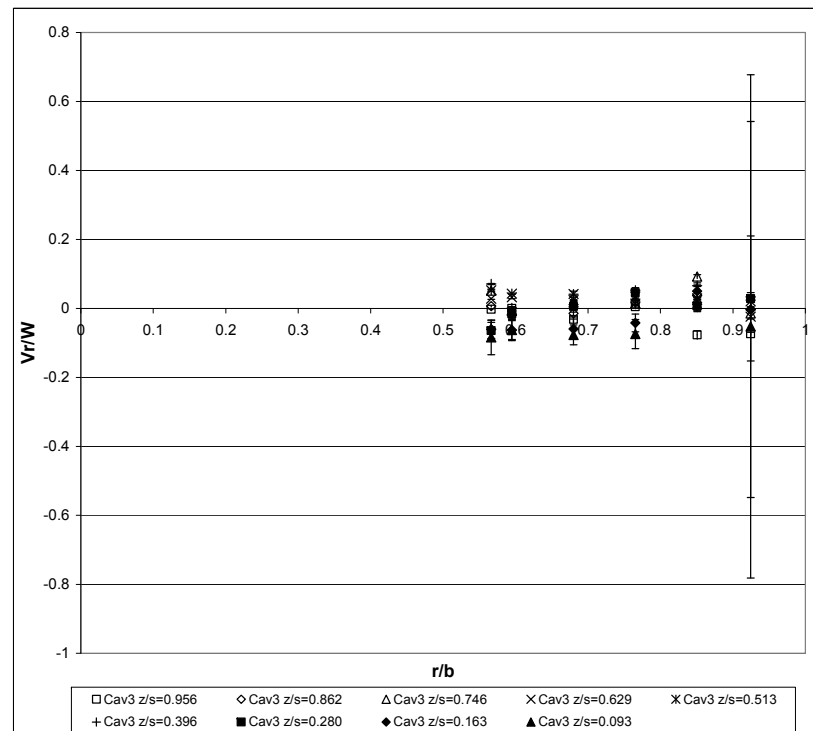
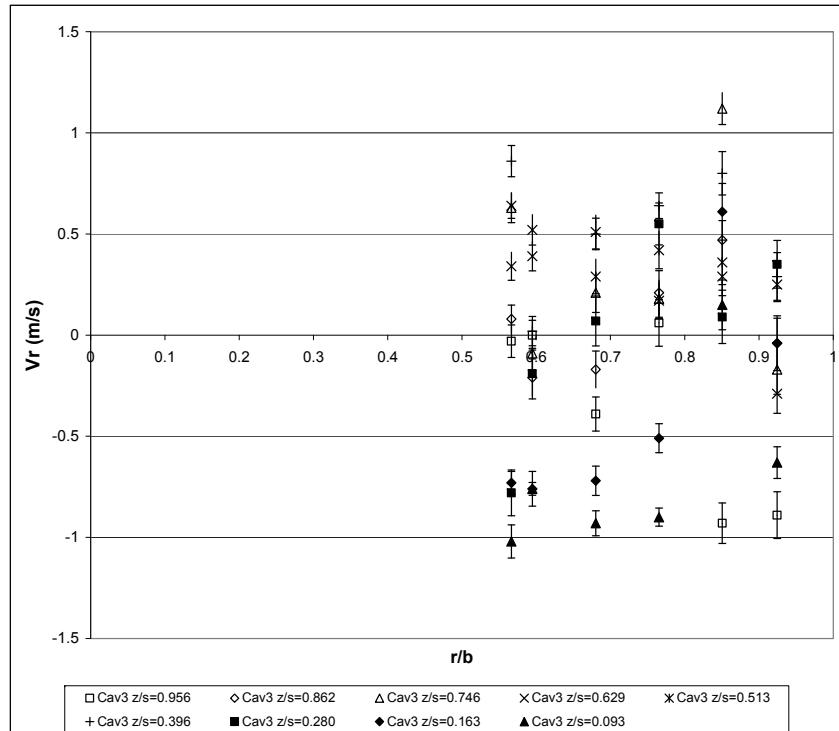
LDA code: 110
Re Φ : 1.93E+06
Rez: 5.31E+04
Ro: 0.53

rpm: 3143.4
Date: 28/09/2004
raw LDA file: \MCR\DA\Build3\B3run39-1.lda
LDA list file: \MCR\DA\Build3\B3run39-1.xls

Perfd by: NM
LDA1: Vr
LDA2: V Φ
W (m/s): 12.13

Printed: 27/09/2008
Comments: Probe angular error corrected -1.389deg
 Traverse ref:

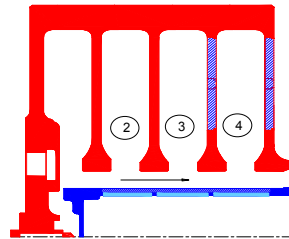
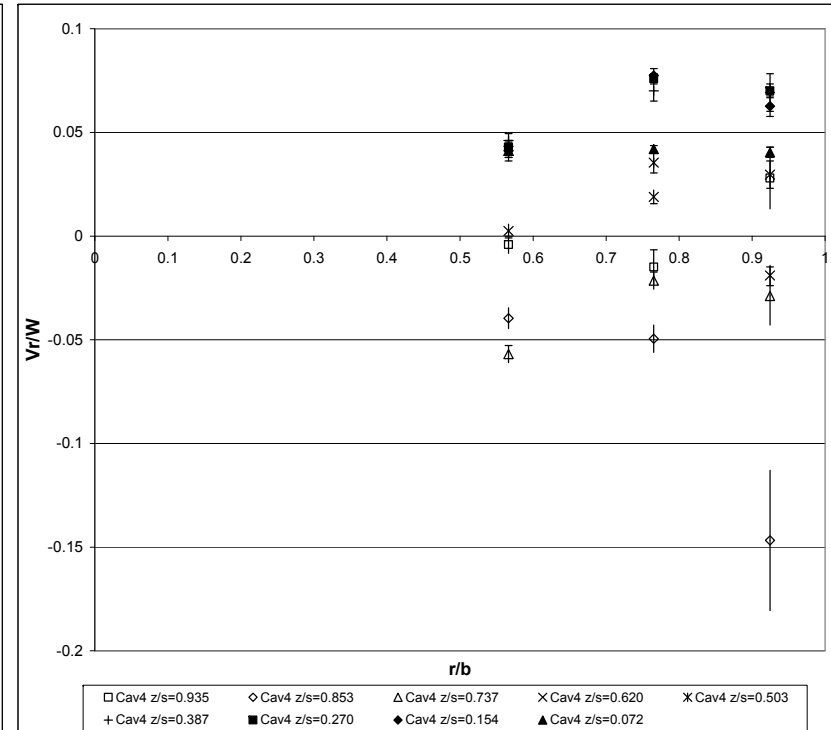
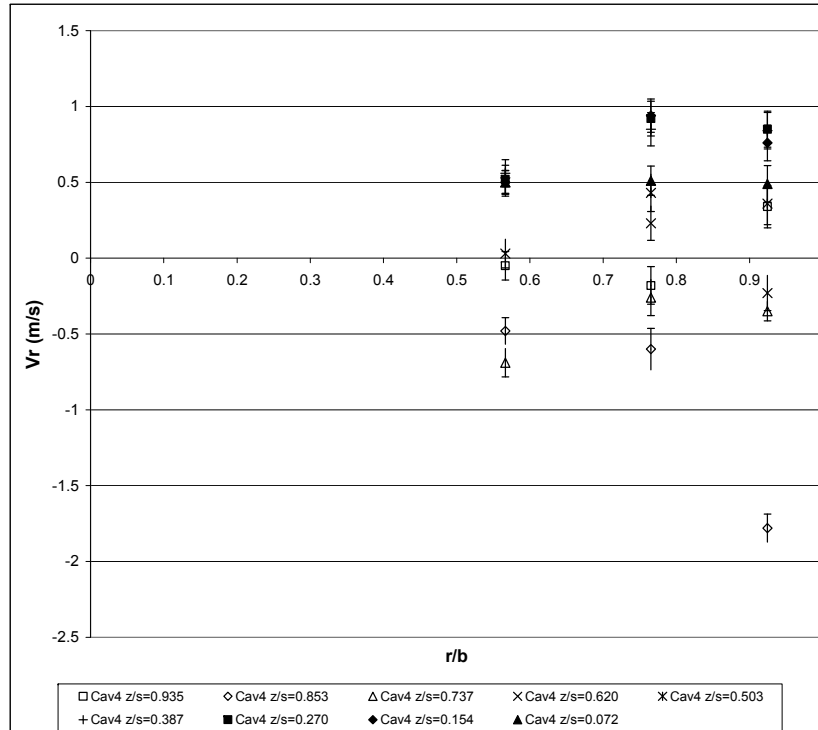
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CAVITY 3

LDA measurement**Build:** 3**LDA test # :** B3run39-1**Heat Transfer run:** B3run39-1**LDA code:** 110**Re Φ :** 1.93E+06**Rez:** 5.31E+04**Ro:** 0.53**rpm:** 3143.4**Date:** 28/09/2004**raw LDA file:** \MCR\LD\Build3\B3run39-1.lda**LDA list file:** \MCR\LD\Build3\B3run39-1.xls**Perfd by:** NM**LDA1:** Vr**LDA2:** V Φ **W (m/s):** 12.13

Printed: 27/09/2008

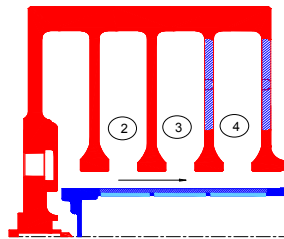
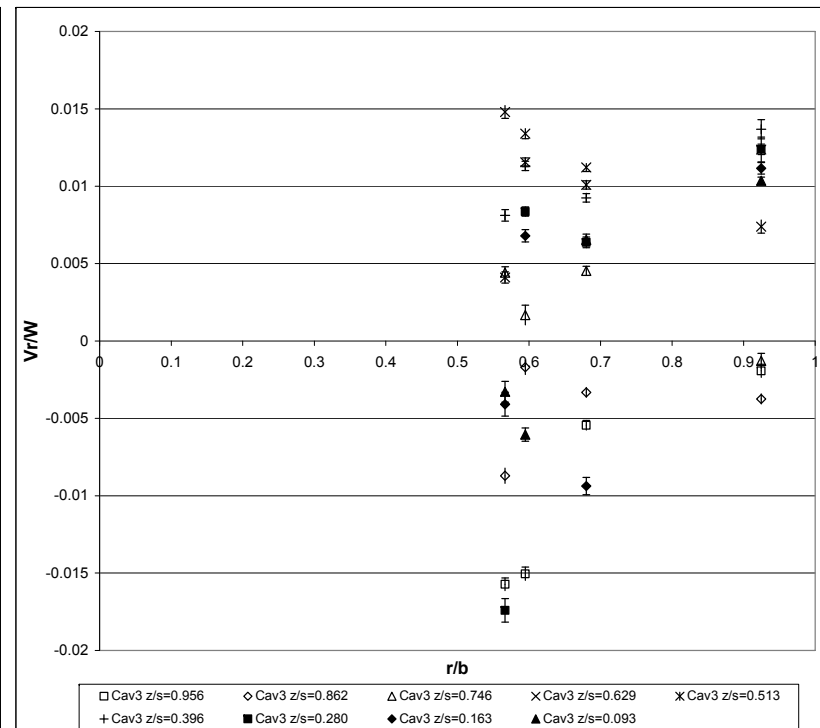
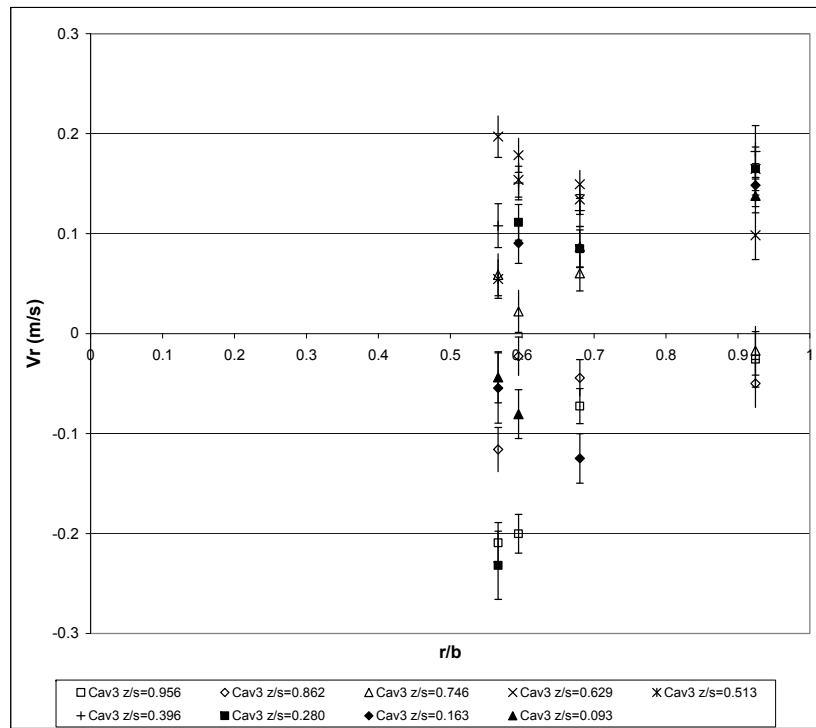
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Comments: Probe angular error corrected -1.389deg**Traverse ref:****CAVITY 4**

| X [mm] | corrected Δ X | Y [mm] | corrected Y/ Radius [mm] | Count(1) | Count(2) | Data Rate(1) [dBs] | Data Rate(2) [dBs] | Validation n1[%] | Validation n2 [%] | LDA1 Mean [m/s] | LDA2 Mean [m/s] | RMS LDA1 | RMS LDA2 | LDA1 MeanConf [m/s] | LDA2 RMSConf [m/s] | Cavity | r_bz | z/s | Qr | V ₀ /Qr | V ₀ /Qr V _{D95%} /Qr | V ₀ /Qr Qr | V ₀ /Qr V _{Z95%} /Qr | V ₀ /Qr V _{Z95%} /Qr |
|--------|------------------|--------|-----------------------------|----------|----------|-----------------------|-----------------------|---------------------|----------------------|-----------------------|-----------------------|-------------|-------------|---------------------------|--------------------------|--------|-------|-------|-------|--------------------|---|--------------------------|---|---|
| 151 | 173.5 | 163 | 203.375 | 2610 | | 48.2 | 16.57 | -0.89 | | 1.40 | | 0.05 | | 0.04 | 3 | 0.924 | 0.956 | 66.95 | -0.07 | 0.12 | 4.12E-03 | | | |
| 155 | 177.5 | 163 | 203.375 | 3991 | | 10.0 | 19.42 | -0.04 | | 1.50 | | 0.05 | | 0.03 | 3 | 0.924 | 0.962 | 66.95 | -0.00 | 0.12 | 0.00E+00 | | | |
| 160 | 182.5 | 163 | 203.375 | 1273 | | 55.52 | 19.07 | 0.52 | | 1.44 | | 0.17 | | 0.04 | 3 | 0.924 | 0.966 | 66.95 | -0.01 | 0.12 | 0.00E+00 | | | |
| 165 | 187.5 | 163 | 203.375 | 1466 | | 40.0 | 27.42 | -0.29 | | 1.17 | | 0.06 | | 0.04 | 3 | 0.924 | 0.929 | 66.95 | -0.02 | 0.10 | 0.00E+00 | | | |
| 170 | 192.5 | 163 | 203.375 | 3898 | | 50.3 | 36.56 | 0.25 | | 1.02 | | 0.03 | | 0.02 | 3 | 0.924 | 0.951 | 66.95 | -0.02 | 0.08 | 0.00E+00 | | | |
| 175 | 197.5 | 163 | 203.375 | 197 | | 1.1 | 23.51 | 0.29 | | 1.43 | | 0.20 | | 0.14 | 3 | 0.924 | 0.936 | 66.95 | -0.02 | 0.12 | 0.00E+00 | | | |
| 180 | 202.5 | 163 | 203.375 | 101 | | 1.43 | 35 | 0.26 | | 1.35 | | 0.28 | | 0.26 | 3 | 0.924 | 0.930 | 66.95 | -0.03 | 0.12 | 0.00E+00 | | | |
| 185 | 207.5 | 163 | 203.375 | 41 | | 0.2 | 15.46 | -0.04 | | 1.64 | | 0.51 | | 0.36 | 3 | 0.924 | 0.163 | 66.95 | -0.00 | 0.14 | 0.00E+00 | | | |
| 188 | 210.5 | 163 | 203.375 | 31 | | 0.2 | 64.21 | -0.63 | | 0.95 | | 0.34 | | 0.24 | 3 | 0.924 | 0.093 | 66.95 | -0.05 | 0.08 | 0.00E+00 | | | |
| 151 | 173.5 | 150 | 187.125 | 1249 | | 122.5 | 28.69 | -0.93 | | 1.22 | | 0.07 | | 0.05 | 3 | 0.851 | 0.956 | 61.60 | -0.08 | 0.10 | 0.00E+00 | | | |
| 155 | 177.5 | 150 | 187.125 | 2135 | | 94.35 | 86.03 | 0.47 | | 0.81 | | 0.07 | | 0.05 | 3 | 0.851 | 0.962 | 61.60 | -0.04 | 0.10 | 0.00E+00 | | | |
| 160 | 182.5 | 150 | 187.125 | 1240 | | 24.18 | 68.76 | 0.12 | | 0.96 | | 0.04 | | 0.03 | 3 | 0.851 | 0.746 | 61.60 | -0.09 | 0.08 | 0.00E+00 | | | |
| 165 | 187.5 | 150 | 187.125 | 937 | | 397.7 | 77.17 | 0.29 | | 1.15 | | 0.07 | | 0.05 | 3 | 0.851 | 0.629 | 61.60 | -0.02 | 0.09 | 0.00E+00 | | | |
| 170 | 192.5 | 150 | 187.125 | 6429 | | 190.3 | 79.33 | 0.36 | | 1.33 | | 0.03 | | 0.02 | 3 | 0.851 | 0.513 | 61.60 | -0.03 | 0.11 | 0.00E+00 | | | |
| 175 | 197.5 | 150 | 187.125 | 9758 | | 195.8 | 46.96 | 0.80 | | 1.30 | | 0.02 | | 0.01 | 3 | 0.851 | 0.390 | 61.60 | -0.02 | 0.11 | 0.00E+00 | | | |
| 180 | 202.5 | 150 | 187.125 | 9426 | | 156.6 | 51.13 | 0.09 | | 1.60 | | 0.03 | | 0.02 | 3 | 0.851 | 0.280 | 61.60 | -0.01 | 0.13 | 0.00E+00 | | | |
| 185 | 207.5 | 150 | 187.125 | 4093 | | 294.0 | 78.98 | 0.61 | | 1.70 | | 0.05 | | 0.04 | 3 | 0.851 | 0.163 | 61.60 | -0.05 | 0.14 | 0.00E+00 | | | |
| 188 | 210.5 | 150 | 187.125 | 4854 | | 75.9 | 53.70 | 0.15 | | 1.49 | | 0.04 | | 0.03 | 3 | 0.851 | 0.093 | 61.60 | -0.01 | 0.12 | 0.00E+00 | | | |
| 151 | 173.5 | 135 | 168.375 | 4038 | | 120.4 | 70.40 | 0.06 | | 1.03 | | 0.04 | | 0.03 | 3 | 0.785 | 0.966 | 55.43 | -0.07 | 0.11 | 0.00E+00 | | | |
| 155 | 177.5 | 135 | 168.375 | 6017 | | 802.4 | 80.23 | 0.21 | | 1.32 | | 0.02 | | 0.01 | 3 | 0.785 | 0.866 | 55.43 | -0.11 | 0.02 | 0.00E+00 | | | |
| 160 | 182.5 | 135 | 168.375 | 2623 | | 325.2 | 53.08 | 0.18 | | 1.11 | | 0.04 | | 0.03 | 3 | 0.785 | 0.746 | 55.43 | -0.01 | 0.09 | 0.00E+00 | | | |
| 165 | 187.5 | 135 | 168.375 | 1755 | | 713.0 | 63.67 | 0.17 | | 1.08 | | 0.05 | | 0.04 | 3 | 0.785 | 0.629 | 55.43 | -0.01 | 0.09 | 0.00E+00 | | | |
| 170 | 192.5 | 135 | 168.375 | 1329 | | 254.9 | 45.86 | 0.14 | | 1.42 | | 0.19 | | 0.14 | 3 | 0.785 | 0.513 | 55.43 | -0.04 | 0.09 | 0.00E+00 | | | |
| 175 | 197.5 | 135 | 168.375 | 4264 | | 193.2 | 72.33 | 0.64 | | 0.77 | | 0.02 | | 0.02 | 3 | 0.785 | 0.396 | 55.43 | -0.05 | 0.06 | 0.00E+00 | | | |
| 180 | 202.5 | 135 | 168.375 | 6962 | | 104.6 | 65.65 | 0.55 | | 1.26 | | 0.03 | | 0.02 | 3 | 0.785 | 0.280 | 55.43 | -0.05 | 0.10 | 0.00E+00 | | | |
| 185 | 207.5 | 135 | 168.375 | 3423 | | 103.6 | 72.07 | -0.51 | | 0.87 | | 0.03 | | 0.02 | 3 | 0.785 | 0.163 | 55.43 | -0.04 | 0.07 | 0.00E+00 | | | |
| 188 | 210.5 | 135 | 168.375 | 1900 | | 21.1 | 87.98 | -0.90 | | 0.92 | | 0.03 | | 0.02 | 3 | 0.785 | 0.093 | 55.43 | -0.02 | 0.09 | 0.00E+00 | | | |
| 151 | 173.5 | 120 | 149.625 | 2772 | | 103.6 | 82.85 | -0.39 | | 1.02 | | 0.04 | | 0.03 | 3 | 0.680 | 0.956 | 49.25 | -0.03 | 0.08 | 0.00E+00 | | | |
| 155 | 177.5 | 120 | 149.625 | 7963 | | 88.7 | 49.99 | -0.17 | | 1.10 | | 0.02 | | 0.02 | 3 | 0.680 | 0.862 | 49.25 | -0.01 | 0.09 | 0.00E+00 | | | |
| 160 | 182.5 | 120 | 149.625 | 12403 | | 137.6 | 43.01 | 0.21 | | 1.18 | | 0.02 | | 0.01 | 3 | 0.680 | 0.746 | 49.25 | -0.02 | 0.10 | 0.00E+00 | | | |
| 165 | 187.5 | 120 | 149.625 | 1068 | | 246.7 | 58.73 | 0.51 | | 0.91 | | 0.03 | | 0.02 | 3 | 0.680 | 0.629 | 49.25 | -0.04 | 0.08 | 0.00E+00 | | | |
| 170 | 192.5 | 120 | 149.625 | 12185 | | 174.4 | 60.27 | 0.29 | | 1.03 | | 0.02 | | 0.01 | 3 | 0.680 | 0.513 | 49.25 | -0.02 | 0.08 | 0.00E+00 | | | |
| 175 | 197.5 | 120 | 149.625 | 2040 | | 346.4 | 52.32 | 0.50 | | 0.95 | | 0.04 | | 0.03 | 3 | 0.680 | 0.396 | 49.25 | -0.04 | 0.08 | 0.00E+00 | | | |
| 180 | 202.5 | 120 | 149.625 | 6747 | | 114.1 | 50.48 | 0.07 | | 1.50 | | 0.04 | | 0.03 | 3 | 0.680 | 0.280 | 49.25 | -0.01 | 0.12 | 0.00E+00 | | | |
| 185 | 207.5 | 120 | 149.625 | 4848 | | 28.6 | 70.43 | -0.72 | | 1.03 | | 0.02 | | 0.01 | 3 | 0.680 | 0.163 | 49.25 | -0.07 | 0.07 | 0.00E+00 | | | |
| 188 | 210.5 | 120 | 149.625 | 1006 | | 21.3 | 70.01 | -0.93 | | 0.75 | | 0.05 | | 0.03 | 3 | 0.680 | 0.093 | 49.25 | -0.08 | 0.06 | 0.00E+00 | | | |
| 151 | 173.5 | 105 | 130.875 | 2099 | | 160.0 | 60.66 | 0.00 | | 0.89 | | 0.03 | | 0.02 | 3 | 0.595 | 0.956 | 43.08 | -0.00 | 0.07 | 0.00E+00 | | | |
| 155 | 177.5 | 105 | 130.875 | 4744 | | 520.8 | 49.47 | -0.21 | | 1.13 | | 0.03 | | 0.02 | 3 | 0.595 | 0.862 | 43.08 | -0.02 | 0.09 | 0.00E+00 | | | |
| 160 | 182.5 | 105 | 130.875 | 3466 | | 606.2 | 54.89 | 0.09 | | 1.04 | | 0.02 | | 0.01 | 3 | 0.595 | 0.746 | 43.08 | -0.01 | 0.09 | 0.00E+00 | | | |
| 165 | 187.5 | 105 | 130.875 | 1800 | | 347.8 | 56.39 | 0.08 | | 0.93 | | 0.03 | | 0.02 | 3 | 0.595 | 0.629 | 43.08 | -0.03 | 0.07 | 0.00E+00 | | | |
| 170 | 192.5 | 105 | 130.875 | 8885 | | 90.1 | 82.33 | 0.52 | | 0.91 | | 0.02 | | 0.01 | 3 | 0.595 | 0.513 | 43.08 | -0.04 | 0.08 | 0.00E+00 | | | |
| 175 | 197.5 | 105 | 130.875 | 10757 | | 233.2 | 86.07 | 0.02 | | 0.88 | | 0.02 | | 0.01 | 3 | 0.595 | 0.396 | 43.08 | -0.00 | 0.07 | 0.00E+00 | | | |
| 180 | 202.5 | 105 | 130.875 | 3906 | | 192.8 | 52.91 | 0.03 | | 0.92 | | 0.02 | | 0.01 | 3 | 0.595 | 0.280 | 43.08 | -0.02 | 0.13 | 0.00E+00 | | | |
| 185 | 207.5 | 105 | 130.875 | 3461 | | 211.7 | 76.47 | -0.76 | | 1.04 | | 0.03 | | 0.02 | 3 | 0.595 | 0.163 | 43.08 | -0.06 | 0.09 | 0.00E+00 | | | |
| 188 | 210.5 | 105 | 130.875 | 1818 | | 29.0 | 89.99 | -0.76 | | 0.38 | | 0.02 | | 0.01 | 3 | 0.595 | 0.093 | 43.08 | -0.06 | 0.03 | 0.00E+00 | | | |
| 151 | 173.5 | 100 | 124.625 | 9888 | | 546.2 | 42.02 | 0.03 | | 0.97 | | 0.02 | | 0.01 | 3 | 0.566 | 0.956 | 41.02 | -0.00 | 0.08 | 0.00E+00 | | | |
| 155 | 177.5 | 100 | 124.625 | 10287 | | 340.8 | 54.41 | 0.01 | | 0.98 | | 0.03 | | 0.02 | 3 | 0.566 | 0.862 | 41.02 | -0.01 | 0.07 | 0.00E+00 | | | |
| 160 | 182.5 | 100 | 124.625 | 9715 | | 1082.7 | 56.81 | 0.63 | | 0.90 | | 0.02 | | 0.01 | 3 | 0.566 | 0.746 | 41.02 | -0.05 | 0.07 | 0.00E+00 | | | |
| 165 | 187.5 | 100 | 124.625 | 6751 | | 778.4 | 58.46 | 0.34 | | 0.84 | | 0.02 | | 0.01 | 3 | 0.566 | 0.629 | 41.02 | -0.03 | 0.07 | 0.00E+00 | | | |
| 170 | 192.5 | 100 | 124.625 | 2909 | | 578.1 | 66.97 | 0.74 | | 0.76 | | 0.03 | | 0.02 | 3 | 0.566 | 0.513 | 41.02 | -0.05 | 0.06 | 0.00E+00 | | | |
| 175 | 197.5 | 100 | 124.625 | 13456 | | 144.3 | 52.65 | 0.86 | | 0.94 | | 0.02 | | 0.01 | 3 | 0.566 | 0.396 | 41.02 | -0.01 | 0.09 | 0.00E+00 | | | |
| 180 | 202.5 | 100 | 124.625 | 7271 | | 52.22 | 66.78 | -1.37 | | 0.62 | | 0.03 | | 0.02 | 3 | 0.566 | 0.280 | 41.02 | -0.11 | -0.06 | 0.00E+00 | | | |
| 185 | 207.5 | 100 | 124.625 | 1820 | | 61.4 | 74.90 | -0.73 | | 0.68 | | 0.03 | | 0.02 | 3 | 0.566 | 0.163 | 41.02 | -0.06 | -0.06 | 0.00E+00 | | | |
| 188 | 210.5 | 100 | 124.625 | 813 | | 6.8 | 80.52 | -1.02 | | 0.99 | | 0.07 | | 0.05 | 3 | 0.566 | 0.093 | 41.02 | -0.08 | -0.08 | 0.00E+00 | | | |
| 101 | 123.5 | 163 | 203.375 | 1915 | | 16.0 | 68.31 | 0.34 | | 1.68 | | 0.03 | | 0.02 | 3 | 0.924 | 0.935 | 66.95 | -0.03 | 0.14 | 0.00E+00 | | | |
| 104.5 | 127 | 163 | 203.375 | 196 | | 1.2 | 74.52 | -1.12 | | 1.13 | | 0.18 | | 0.14 | 3 | 0.924 | 0.862 | 66.95 | -0.05 | 0.15 | 0.00E+00 | | | |
| 109.5 | 132 | 163 | 203.375 | 142 | | 1.2 | 68.31 | -0.35 | | 0.78 | | 0.13 | | 0.09 | 4 | 0.924 | 0.737 | 66.95 | -0.03 | 0.06 | 0.00E+00 | | | |
| 114.5 | 137 | 163 | 203.375 | 909 | | 75.3 | 75.14 | -0.23 | | 1.40 | | 0.03 | | 0.02 | 4 | 0.924 | 0.620 | 66.95 | -0.02 | 0.12 | 0.00E+00 | | | |
| 119.5 | 142 | 163 | 203.375 | 9253 | | 168.0 | 72.93 | 0.36 | | 1.69 | | 0.03 | | 0.02 | 4 | 0.924 | 0.503 | 66.95 | -0.08 | 0.13 | 0.00E+00 | | | |
| 124.5 | 147 | 163 | 203.375 | 8181 | | 1473.8 | 66.40 | 0.84 | | 1.38 | | 0.02 | | 0.01 | 4 | 0.924 | 0.387 | 66.95 | -0.01 | 0.09 | 0.00E+00 | | | |
| 129.5 | 152 | 163 | 203.375 | 7777 | | 1481.1 | 58.86 | 0.85 | | 1.44 | | 0.03 | | 0.02 | 4 | 0.924 | 0.270 | 66.95 | -0.07 | 0.12 | 0.00E+00 | | | |
| | | | | | | | | | | | | | | | | | | | | | | | | |

LDA measurement**Build:** 3**LDA test # :** B3run40-1**Heat Transfer run:** B3run40-1**LDA code:** 110**Re Φ :** 3.88E+05**Rez:** 6.07E+04**Ro:** 2.98**rpm:** 608.8**Date:** 05/10/2004**raw LDA file:** \MCRLDA\Build3\B3run40-1.lda**LDA list file:** \MCRLDA\Build3\B3run40-1.xls**Perfd by:** NM**LDA1:** Vr**LDA2:** V ϕ **W (m/s):** 13.31**Printed:** 27/09/2008**Comments:****Traverse ref:**

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CAVITY 3

LDA measurement

Build: 3
 LDA test #: B3run40-1
 Heat Transfer run: B3run40-1

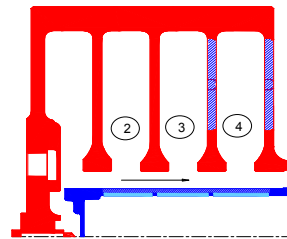
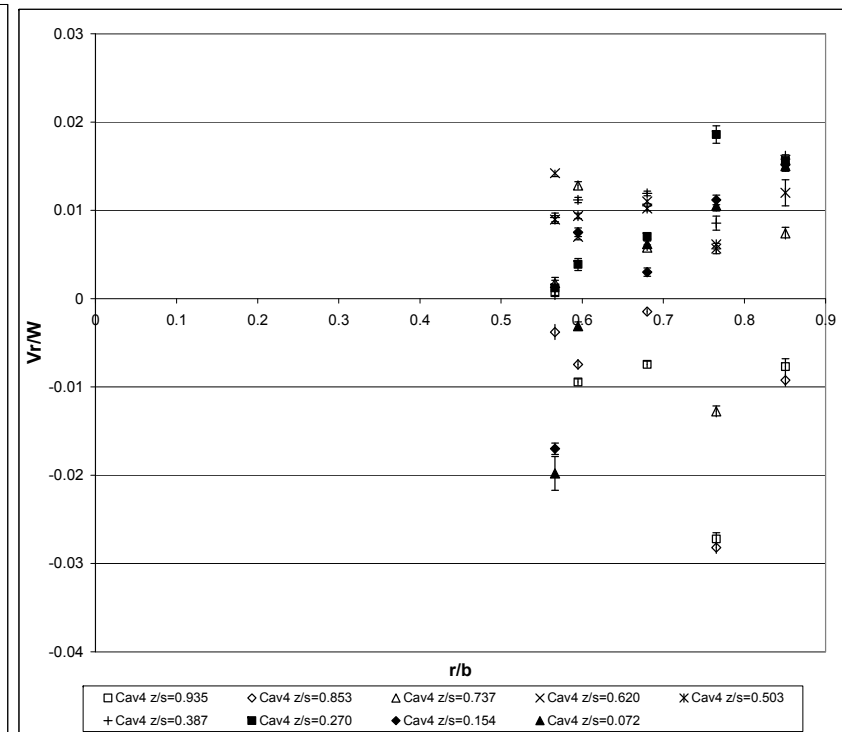
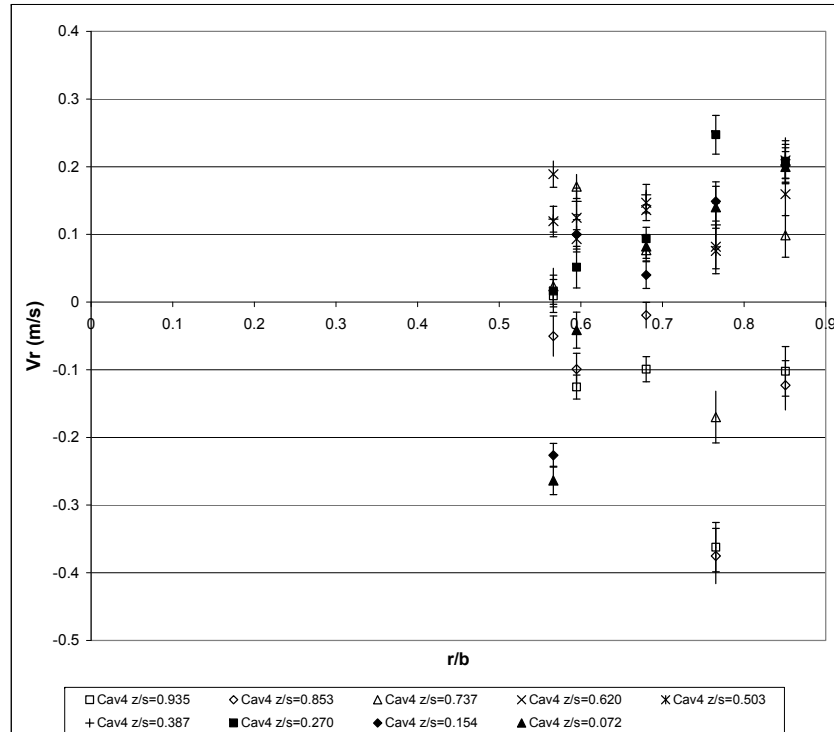
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 Rez: 6.07E+04
 Ro: 2.98

rpm: 608.8
 Date: 05/10/2004
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 LDA list file: \MCR\LD\Build3\B3run40-1.xls

Perfd by: NM
 LDA1: Vr
 LDA2: V Φ
 W (m/s): 13.31

Printed: 27/09/2008
 Comments:
 Traverse ref:

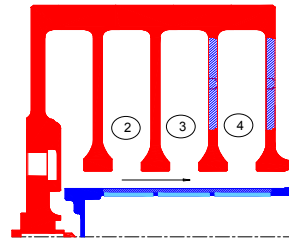
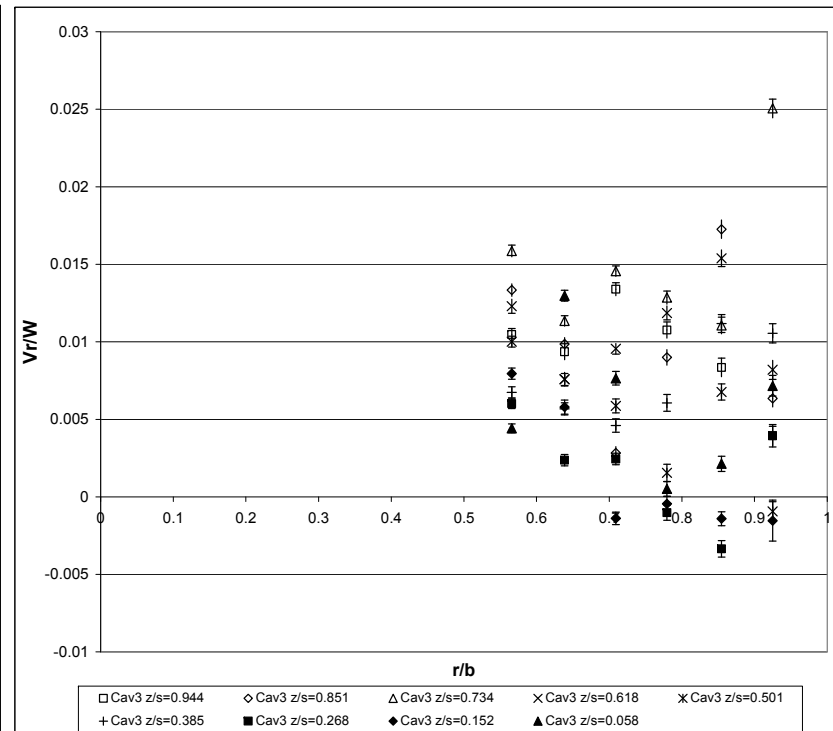
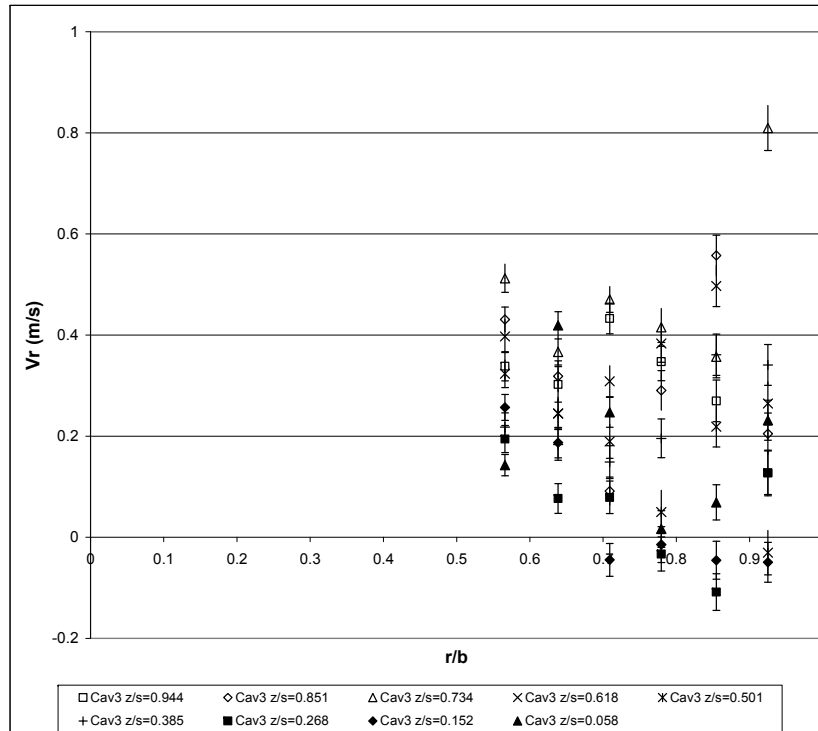
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CAVITY 4

| LDA measurement | | | | | | | | | | LDA code: 110 | rpm: 608.8 | Perfd by: NM | Printed: 27/09/2008 | Page 2/2 | | | | | | | | | | | | | |
|-------------------------------------|-------------|--------|-------------|-------------|----------|----------|-------------------|-------------------|---------------------|----------------------|--|-----------------------|---------------------|----------------|------------------|---------------------|---------------------|--------|-------|-------|----------|--------------------|--------------------------------------|-----------------------|-------|------|----------------------|
| Build: 3 | | | | | | | | | | ReΦ: 3.88E+05 | Date: 05/10/2004 | LDA1: Vφ | Comments: 0 | | | | | | | | | | | | | | |
| LDA test # : B3run40-1 | | | | | | | | | | Rez: 6.07E+04 | raw LDA file: MCRLDA\Build3\B3run40-1.lda | LDA2: Vφ | Traverse corresp: 0 | | | | | | | | | | | | | | |
| Heat Transfer run: B3run40-1 | | | | | | | | | | Ro: 2.98 | LDA list file: MCRLDA\Build3\B3run40-1.xls | W (m/s): 13.31 | | | | | | | | | | | | | | | |
| X [mm] | corrected X | Y [mm] | corrected Y | Radius [mm] | Count(1) | Count(2) | Data Rate(1) [Hz] | Data Rate(2) [Hz] | Validation n(1) [%] | Validation (2) [%] | LDA1 Mean [m/s] | LDA2 Mean [m/s] | LDA1 RMS [m/s] | LDA2 RMS [m/s] | MeanConf f [m/s] | LDA1 MeanConf [m/s] | LDA2 MeanConf [m/s] | Cavity | r/b | z/s | Ωr (m/s) | V _φ /Qr | V _φ /Q _{USS} /Qr | V _φ RMS/Ωr | Vz/W | W | Vz _{RMS} /W |
| 101 | 123.5 | 100 | 124.625 | 3783 | | | 15.8 | 41.78 | 0.01 | | 0.32 | 0.01 | 0.01 | | 0.01 | | 0.01 | 4 | 0.566 | 0.935 | 7.95 | | | | 0.00 | 0.02 | 7.74E-04 |
| 104.5 | 127 | 100 | 124.625 | 4566 | | | 19.0 | 43.68 | -0.05 | | 0.40 | 0.01 | 0.01 | | 0.01 | | 0.01 | 4 | 0.566 | 0.853 | 7.95 | | | | 0.00 | 0.03 | 8.62E-04 |
| 109.5 | 132 | 100 | 124.625 | 6179 | | | 25.8 | 55.35 | 0.02 | | 0.35 | 0.01 | 0.01 | | 0.01 | | 0.01 | 4 | 0.566 | 0.737 | 7.95 | | | | 0.00 | 0.03 | 6.61E-04 |
| 114.5 | 137 | 100 | 124.625 | 11017 | | | 63.4 | 57.83 | 0.19 | | 0.26 | 0.00 | 0.00 | | 0.00 | | 0.00 | 4 | 0.566 | 0.620 | 7.95 | | | | 0.01 | 0.02 | 3.61E-04 |
| 119.5 | 142 | 100 | 124.625 | 10066 | | | 51.2 | 60.21 | 0.12 | | 0.30 | 0.01 | 0.00 | | 0.00 | | 0.00 | 4 | 0.566 | 0.503 | 7.95 | | | | 0.01 | 0.02 | 4.44E-04 |
| 124.5 | 147 | 100 | 124.625 | 5900 | | | 24.6 | 59.92 | 0.12 | | 0.25 | 0.01 | 0.00 | | 0.00 | | 0.00 | 4 | 0.566 | 0.387 | 7.95 | | | | 0.01 | 0.02 | 4.87E-04 |
| 129.5 | 152 | 100 | 124.625 | 2864 | | | 11.9 | 58.76 | 0.02 | | 0.31 | 0.01 | 0.01 | | 0.01 | | 0.01 | 4 | 0.566 | 0.270 | 7.95 | | | | 0.00 | 0.02 | 8.59E-04 |
| 134.5 | 157 | 100 | 124.625 | 2763 | | | 11.5 | 75.45 | -0.23 | | 0.23 | 0.01 | 0.01 | | 0.01 | | 0.01 | 4 | 0.566 | 0.154 | 7.95 | | | | -0.02 | 0.02 | 6.50E-04 |
| 138 | 160.5 | 100 | 124.625 | 459 | | | 1.9 | 60.53 | -0.26 | | 0.28 | 0.03 | 0.02 | | 0.02 | | 0.02 | 4 | 0.566 | 0.072 | 7.95 | | | | -0.02 | 0.02 | 1.91E-03 |
| 151 | 173.5 | 100 | 124.625 | 9702 | | | 40.4 | 59.23 | -0.21 | | 0.27 | 0.01 | 0.00 | | 0.00 | | 0.00 | 3 | 0.566 | 0.956 | 7.95 | | | | -0.02 | 0.02 | 4.06E-04 |
| 155 | 177.5 | 100 | 124.625 | 7645 | | | 31.9 | 44.79 | -0.12 | | 0.29 | 0.01 | 0.00 | | 0.00 | | 0.00 | 3 | 0.566 | 0.862 | 7.95 | | | | -0.01 | 0.02 | 4.96E-04 |
| 160 | 182.5 | 100 | 124.625 | 11330 | | | 77.0 | 56.00 | 0.06 | | 0.28 | 0.01 | 0.00 | | 0.00 | | 0.00 | 3 | 0.566 | 0.746 | 7.95 | | | | 0.00 | 0.02 | 3.86E-04 |
| 165 | 187.5 | 100 | 124.625 | 9931 | | | 86.2 | 64.77 | 0.20 | | 0.28 | 0.01 | 0.00 | | 0.00 | | 0.00 | 3 | 0.566 | 0.629 | 7.95 | | | | 0.01 | 0.02 | 4.09E-04 |
| 170 | 192.5 | 105 | 130.875 | 13579 | | | 87.5 | 65.05 | 0.05 | | 0.26 | 0.00 | 0.00 | | 0.00 | | 0.00 | 3 | 0.595 | 0.513 | 7.95 | | | | 0.00 | 0.02 | 3.62E-04 |
| 175 | 197.5 | 100 | 124.625 | 13277 | | | 82.2 | 57.12 | 0.11 | | 0.29 | 0.00 | 0.00 | | 0.00 | | 0.00 | 3 | 0.566 | 0.396 | 7.95 | | | | 0.01 | 0.02 | 3.73E-04 |
| 180 | 202.5 | 100 | 124.625 | 7703 | | | 46.5 | 36.94 | -0.23 | | 0.45 | 0.01 | 0.01 | | 0.01 | | 0.01 | 3 | 0.566 | 0.280 | 7.95 | | | | -0.02 | 0.03 | 7.61E-04 |
| 185 | 207.5 | 100 | 124.625 | 8009 | | | 97.8 | 42.99 | -0.05 | | 0.47 | 0.01 | 0.01 | | 0.01 | | 0.01 | 3 | 0.566 | 0.163 | 7.95 | | | | 0.00 | 0.04 | 7.71E-04 |
| 188 | 210.5 | 105 | 130.875 | 10499 | | | 53.8 | 55.49 | -0.04 | | 0.24 | 0.00 | 0.00 | | 0.00 | | 0.00 | 3 | 0.595 | 0.093 | 7.95 | | | | 0.01 | 0.03 | 7.74E-04 |
| 101 | 123.5 | 105 | 130.875 | 10499 | | | 53.8 | 64.79 | -0.13 | | 0.24 | 0.00 | 0.00 | | 0.00 | | 0.00 | 3 | 0.595 | 0.935 | 8.34 | | | | -0.01 | 0.02 | 3.40E-04 |
| 104.5 | 127 | 105 | 130.875 | 8862 | | | 28.0 | 66.00 | -0.10 | | 0.31 | 0.01 | 0.01 | | 0.01 | | 0.01 | 4 | 0.595 | 0.853 | 8.34 | | | | -0.01 | 0.02 | 5.58E-04 |
| 109.5 | 132 | 105 | 130.875 | 8862 | | | 28.6 | 73.29 | 0.17 | | 0.24 | 0.01 | 0.00 | | 0.00 | | 0.00 | 4 | 0.595 | 0.737 | 8.34 | | | | 0.01 | 0.02 | 4.20E-04 |
| 114.5 | 137 | 105 | 130.875 | 12111 | | | 85.1 | 69.61 | 0.09 | | 0.26 | 0.00 | 0.00 | | 0.00 | | 0.00 | 4 | 0.595 | 0.620 | 8.34 | | | | 0.01 | 0.02 | 3.44E-04 |
| 119.5 | 142 | 105 | 130.875 | 12338 | | | 85.5 | 64.48 | 0.12 | | 0.23 | 0.00 | 0.00 | | 0.00 | | 0.00 | 4 | 0.595 | 0.503 | 8.34 | | | | 0.01 | 0.02 | 3.10E-04 |
| 124.5 | 147 | 105 | 130.875 | 16426 | | | 110.9 | 75.39 | 0.15 | | 0.26 | 0.00 | 0.00 | | 0.00 | | 0.00 | 4 | 0.595 | 0.387 | 8.34 | | | | 0.01 | 0.02 | 3.03E-04 |
| 129.5 | 152 | 105 | 130.875 | 7822 | | | 43.8 | 44.88 | 0.05 | | 0.41 | 0.01 | 0.01 | | 0.01 | | 0.01 | 4 | 0.595 | 0.270 | 8.34 | | | | 0.00 | 0.03 | 6.81E-04 |
| 134.5 | 157 | 105 | 130.875 | 7610 | | | 33.3 | 58.10 | 0.10 | | 0.29 | 0.01 | 0.01 | | 0.01 | | 0.01 | 4 | 0.595 | 0.154 | 8.34 | | | | 0.01 | 0.02 | 4.89E-04 |
| 138 | 160.5 | 105 | 130.875 | 12200 | | | 113.4 | 40.35 | -0.04 | | 0.35 | 0.01 | 0.00 | | 0.00 | | 0.00 | 4 | 0.595 | 0.072 | 8.34 | | | | 0.00 | 0.03 | 4.72E-04 |
| 151 | 173.5 | 105 | 130.875 | 7866 | | | 50.2 | 59.68 | -0.20 | | 0.26 | 0.01 | 0.00 | | 0.00 | | 0.00 | 3 | 0.595 | 0.956 | 8.34 | | | | -0.02 | 0.02 | 4.28E-04 |
| 155 | 177.5 | 105 | 130.875 | 6521 | | | 38.1 | 63.19 | -0.02 | | 0.26 | 0.01 | 0.00 | | 0.00 | | 0.00 | 3 | 0.595 | 0.862 | 8.34 | | | | 0.00 | 0.02 | 4.67E-04 |
| 160 | 182.5 | 105 | 130.875 | 4202 | | | 88.5 | 70.87 | 0.02 | | 0.28 | 0.01 | 0.00 | | 0.00 | | 0.00 | 3 | 0.595 | 0.746 | 8.34 | | | | 0.01 | 0.02 | 3.37E-04 |
| 165 | 187.5 | 105 | 130.875 | 10343 | | | 146.2 | 62.83 | 0.18 | | 0.23 | 0.00 | 0.00 | | 0.00 | | 0.00 | 3 | 0.595 | 0.629 | 8.34 | | | | 0.01 | 0.02 | 3.26E-04 |
| 170 | 192.5 | 105 | 130.875 | 13821 | | | 146.9 | 66.32 | 0.15 | | 0.23 | 0.00 | 0.00 | | 0.00 | | 0.00 | 3 | 0.595 | 0.513 | 8.34 | | | | 0.01 | 0.02 | 2.87E-04 |
| 175 | 197.5 | 105 | 130.875 | 12120 | | | 80.2 | 70.95 | 0.15 | | 0.22 | 0.00 | 0.00 | | 0.00 | | 0.00 | 3 | 0.595 | 0.396 | 8.34 | | | | 0.01 | 0.02 | 2.99E-04 |
| 180 | 202.5 | 105 | 130.875 | 13579 | | | 66.8 | 62.83 | 0.11 | | 0.24 | 0.00 | 0.00 | | 0.00 | | 0.00 | 3 | 0.595 | 0.280 | 8.34 | | | | 0.01 | 0.02 | 3.02E-04 |
| 185 | 207.5 | 105 | 130.875 | 10061 | | | 72.0 | 59.19 | 0.09 | | 0.27 | 0.01 | 0.00 | | 0.00 | | 0.00 | 3 | 0.595 | 0.163 | 8.34 | | | | 0.01 | 0.02 | 3.95E-04 |
| 188 | 210.5 | 105 | 130.875 | 12500 | | | 208.1 | 36.98 | -0.08 | | 0.32 | 0.01 | 0.00 | | 0.00 | | 0.00 | 3 | 0.595 | 0.093 | 8.34 | | | | -0.01 | 0.02 | 4.27E-04 |
| 101 | 123.5 | 120 | 149.625 | 10257 | | | 147.0 | 51.08 | -0.10 | | 0.25 | 0.00 | 0.00 | | 0.00 | | 0.00 | 4 | 0.680 | 0.935 | 9.54 | | | | -0.01 | 0.02 | 3.60E-04 |
| 104.5 | 127 | 120 | 149.625 | 12461 | | | 173.0 | 54.17 | -0.02 | | 0.25 | 0.00 | 0.00 | | 0.00 | | 0.00 | 4 | 0.680 | 0.823 | 9.54 | | | | 0.00 | 0.02 | 3.34E-04 |
| 109.5 | 132 | 120 | 149.625 | 10514 | | | 98.5 | 65.22 | 0.08 | | 0.21 | 0.00 | 0.00 | | 0.00 | | 0.00 | 4 | 0.680 | 0.737 | 9.54 | | | | 0.01 | 0.02 | 3.06E-04 |
| 114.5 | 137 | 120 | 149.625 | 11957 | | | 106.8 | 71.24 | 0.15 | | 0.20 | 0.00 | 0.00 | | 0.00 | | 0.00 | 4 | 0.680 | 0.620 | 9.54 | | | | 0.01 | 0.01 | 2.69E-04 |
| 119.5 | 142 | 120 | 149.625 | 14503 | | | 170.7 | 72.51 | 0.14 | | 0.21 | 0.00 | 0.00 | | 0.00 | | 0.00 | 4 | 0.680 | 0.503 | 9.54 | | | | 0.01 | 0.02 | 2.52E-04 |
| 124.5 | 147 | 120 | 149.625 | 14160 | | | 133.0 | 71.39 | 0.16 | | 0.20 | 0.00 | 0.00 | | 0.00 | | 0.00 | 4 | 0.680 | 0.387 | 9.54 | | | | 0.01 | 0.02 | 2.50E-04 |
| 129.5 | 152 | 120 | 149.625 | 12940 | | | 166.8 | 60.63 | 0.09 | | 0.23 | 0.00 | 0.00 | | 0.00 | | 0.00 | 4 | 0.680 | 0.270 | 9.54 | | | | 0.01 | 0.02 | 2.92E-04 |
| 134.5 | 157 | 120 | 149.625 | 6814 | | | 229.7 | 29.98 | 0.04 | | 0.26 | 0.01 | 0.00 | | 0.00 | | 0.00 | 4 | 0.680 | 0.154 | 9.54 | | | | 0.00 | 0.02 | 4.70E-04 |
| 138 | 160.5 | 120 | 149.625 | 10856 | | | 477.1 | 38.99 | 0.08 | | 0.23 | 0.00 | 0.00 | | 0.00 | | 0.00 | 4 | 0.680 | 0.072 | 9.54 | | | | 0.01 | 0.02 | 3.30E-04 |
| 151 | 173.5 | 120 | 149.625 | 12037 | | | 262.7 | 43.90 | -0.07 | | 0.23 | 0.00 | 0.00 | | 0.00 | | 0.00 | 3 | 0.680 | 0.956 | 9.54 | | | | -0.01 | 0.02 | 3.15E-04 |
| 155 | 177.5 | 120 | 149.625 | 12286 | | | 320.2 | 47.00 | -0.04 | | 0.24 | 0.00 | 0.00 | | 0.00 | | 0.00 | 3 | 0.680 | 0.862 | 9.54 | | | | 0.00 | 0.02 | 3.23E-04 |
| 160 | 182.5 | 120 | 149.625 | 13281 | | | 181.5 | 56.82 | 0.06 | | 0.24 | 0.00 | 0.00 | | 0.00 | | 0.00 | 3 | 0.680 | 0.746 | 9.54 | | | | 0.00 | 0.02 | 3.00E-04 |
| 165 | 187.5 | 120 | 149.625 | 11538 | | | 140.6 | 74.98 | 0.15 | | 0.19 | 0.00 | 0.00 | | 0.00 | | 0.00 | 3 | 0.680 | 0.629 | 9.54 | | | | 0.01 | 0.01 | 2.55E-04 |
| 170 | 192.5 | 120 | 149.625 | 11901 | | | 110.9 | 74.53 | 0.13 | | 0.20 | 0.00 | 0.00 | | 0.00 | | 0.00 | 3 | 0.680 | 0.513 | 9.54 | | | | 0.01 | 0.02 | 2.70E-04 |
| 175 | 197.5 | 120 | 149.625 | 13243 | | | 148.2 | 72.46 | 0.12 | | 0.21 | 0.00 | 0.00 | | 0.00 | | | | | | | | | | | | |

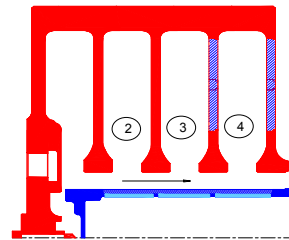
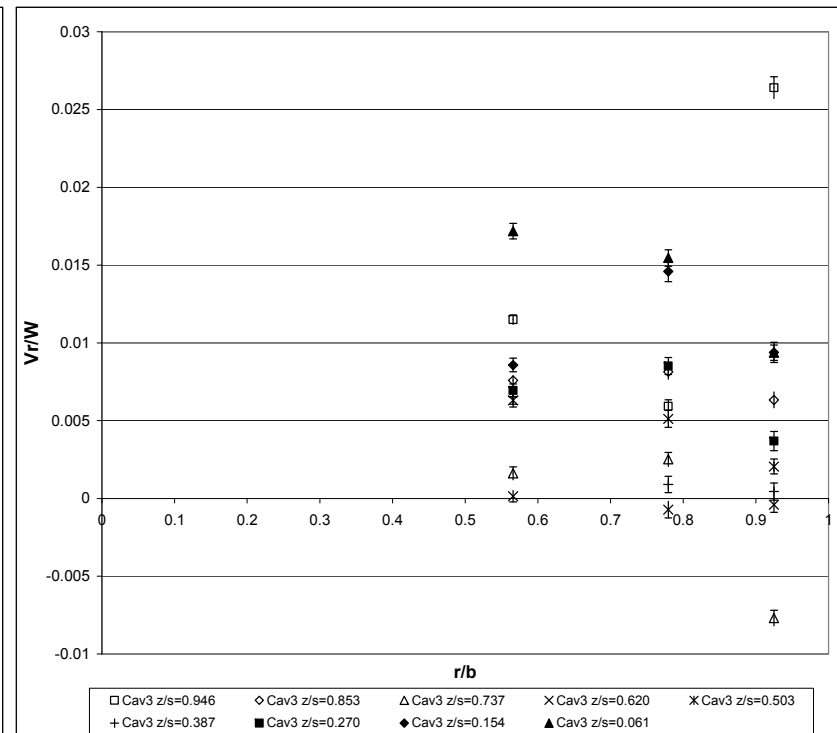
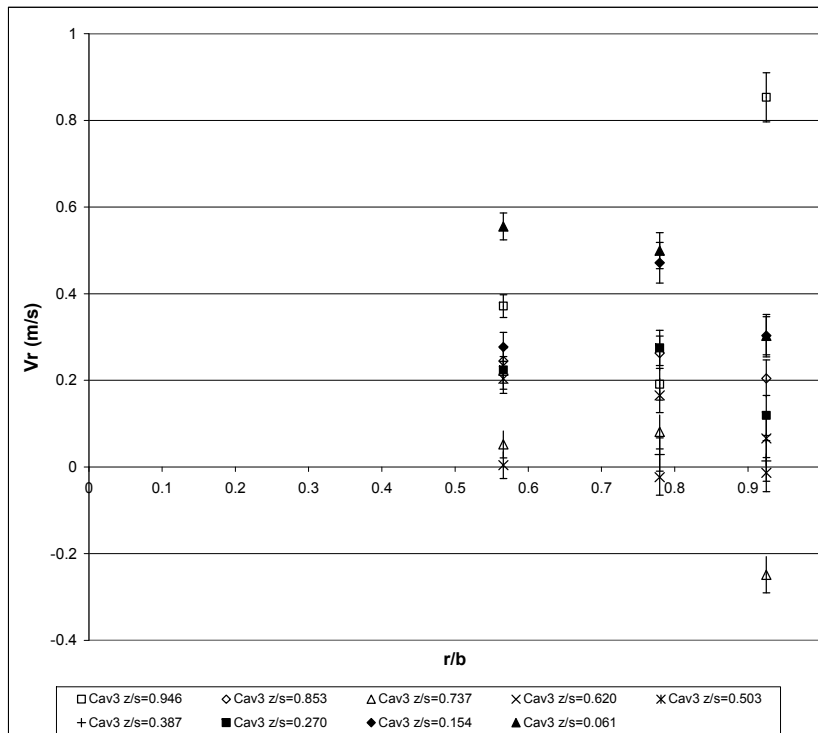
LDA measurement**Build:** 3**LDA test # :** B3run44-1**Heat Transfer run:** B3run44-2**LDA code:** 110**Re Φ :** 1.97E+06**Rez:** 1.64E+05**Ro:** 1.59**rpm:** 2775.5**Date:** 20/12/2004**raw LDA file:** \MCR\LD\Build3\B3run44-1.lda**LDA list file:** \MCR\LD\Build3\B3run44-1.xls**Perfd by:** NM**LDA1:** Vr**LDA2:** V Φ **W (m/s):** 32.31**Printed:** 27/09/2008**Comments:** run OK**Traverse ref:**

Page 1/2

CAVITY 3

LDA measurement**Build:** 3**LDA test # :** B3run44-1**Heat Transfer run:** B3run44-2**LDA code:** 110**Re Φ :** 1.97E+06**Rez:** 1.64E+05**Ro:** 1.59**rpm:** 2775.5**Date:** 20/12/2004**raw LDA file:** \MCR\LD\Build3\B3run44-1.lda**LDA list file:** \MCR\LD\Build3\B3run44-1.xls**Perfd by:** NM**LDA1:** Vr**LDA2:** V Φ **W (m/s):** 32.31**Printed:** 27/09/2008**Comments:** run OK**Traverse ref:**

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CAVITY 4

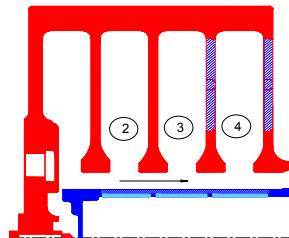
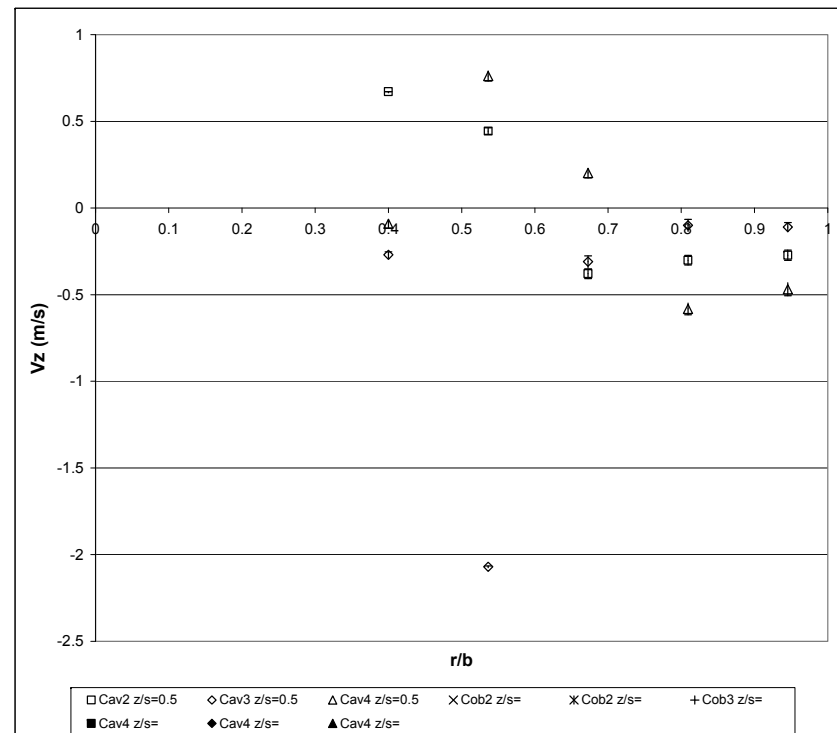
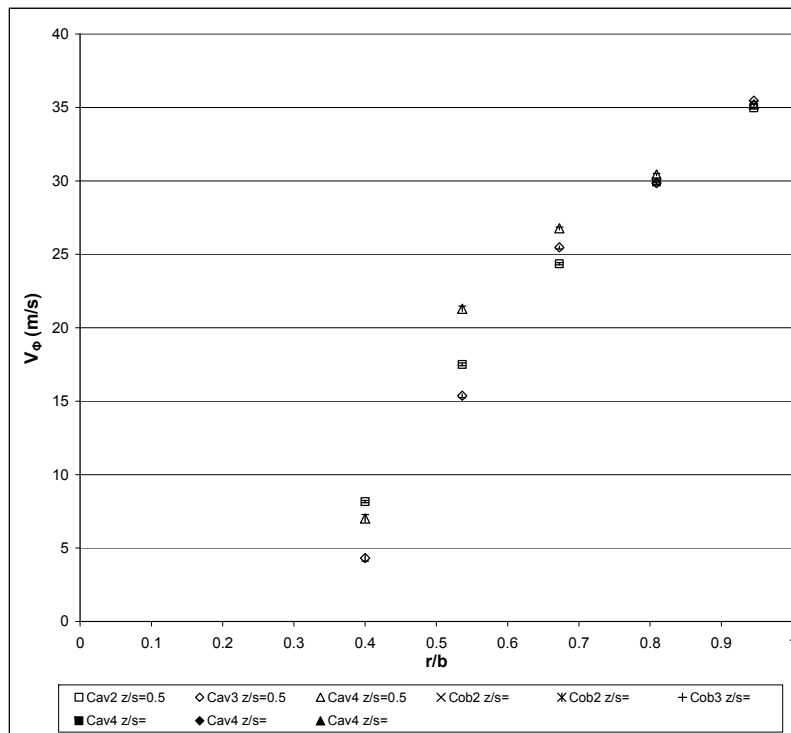
[illegible]

LDA measurement**Build:** 2**LDA test # :** test6**Heat Transfer run:** B2t06**LDA code:** 101**Re Φ :** 1.09E+06**Rez:** 1.14E+05**Ro:** 3.56**rpm:** 1655.0**Date:** 07/08/2001**raw LDA file:** \MCR\LDABuild2&1\Projects\mcr3.lda , mcr2.lda**LDA list file:** MCR\b2Data\LDA-results\MultiCavityRig\Test6-250mm-3cav\cav2_list.txt**Perfd by:****LDA1:** $V\Phi$ **LDA2:** Vz **W (m/s):** 55.3

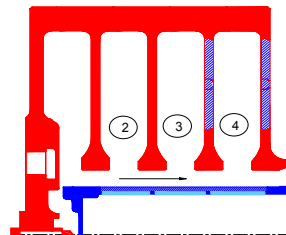
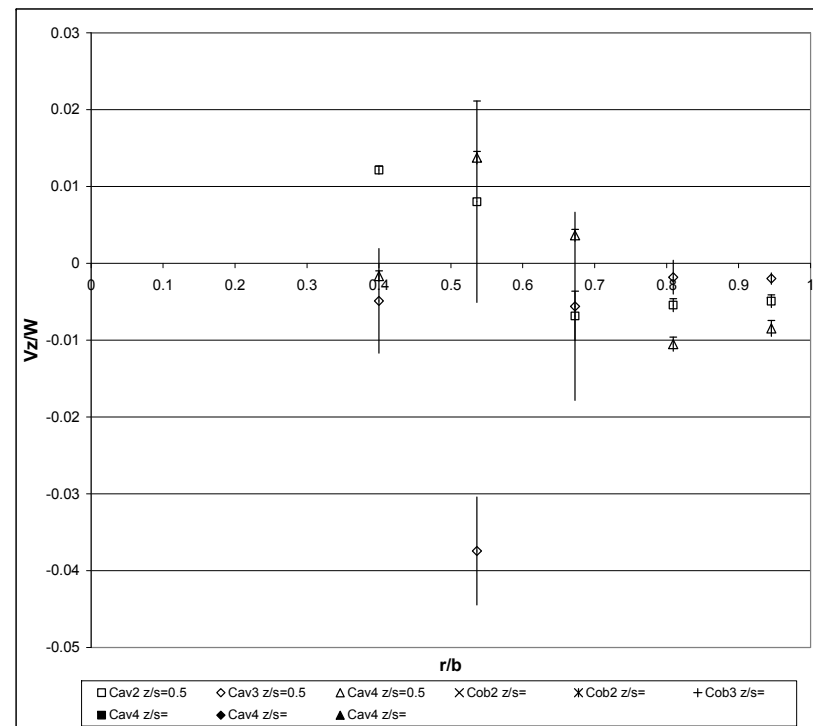
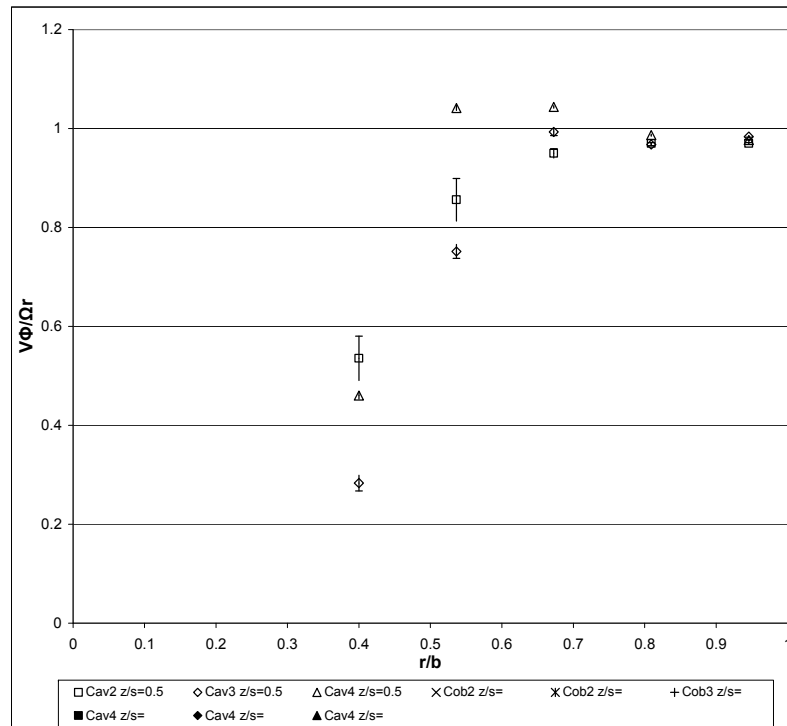
Printed: 27/09/2008

Comments:**Traverse ref:**

Page 1/3



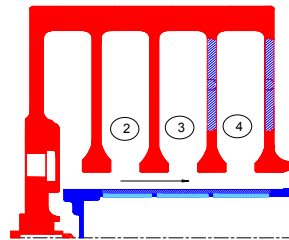
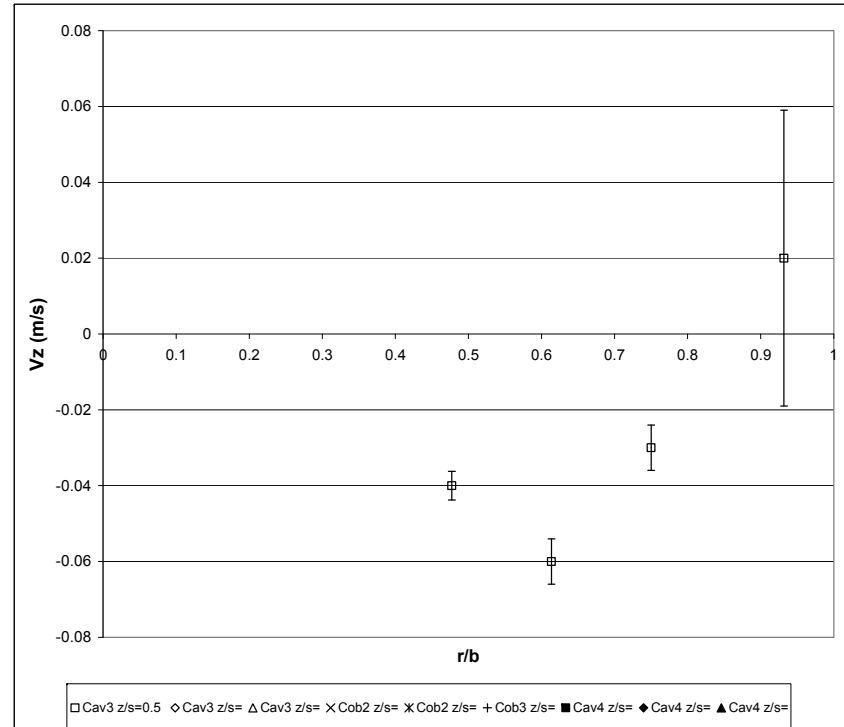
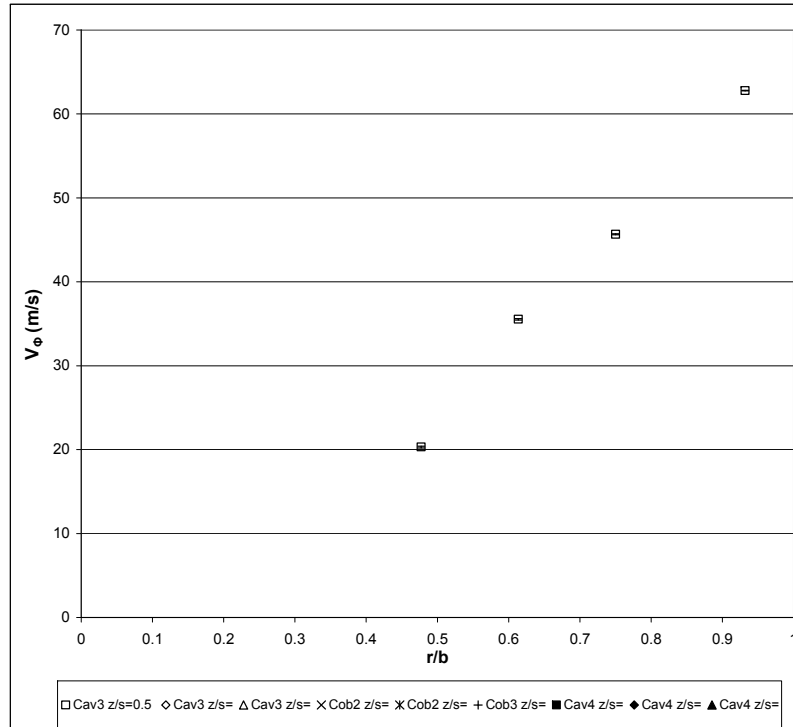
| | | | | | | |
|---------------------------------|--|---------------------------------|--|-----------------|----------------------------|----------|
| LDA measurement | | LDA code: 101 | rpm: 1655.0 | Perfd by: 0 | Printed: 27/09/2008 | Page 2/3 |
| Build: 2 | | Re_φ: 1.09E+06 | Date: 07/08/2001 | LDA1: Vφ | Comments: 0 | |
| LDA test # : test6 | | Re_z: 1.14E+05 | raw LDA file: \MCRLDA\Build2&1\Projects\mcr3.lda , m | LDA2: Vz | Traverse corresp: 0 | |
| Heat Transfer run: B2t06 | | Ro: 3.56 | LDA list file: MCR\b2Data\LDA-results\MultiCavityRig\TW (m/s): | 55.3 | | |



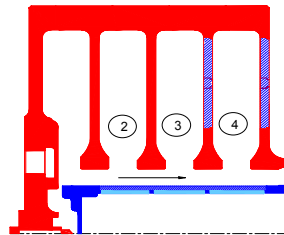
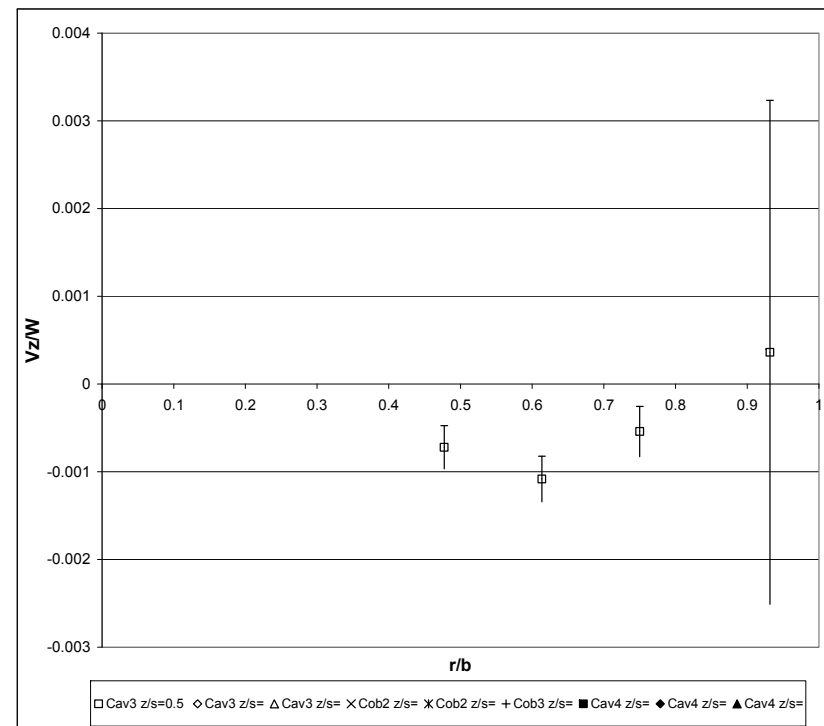
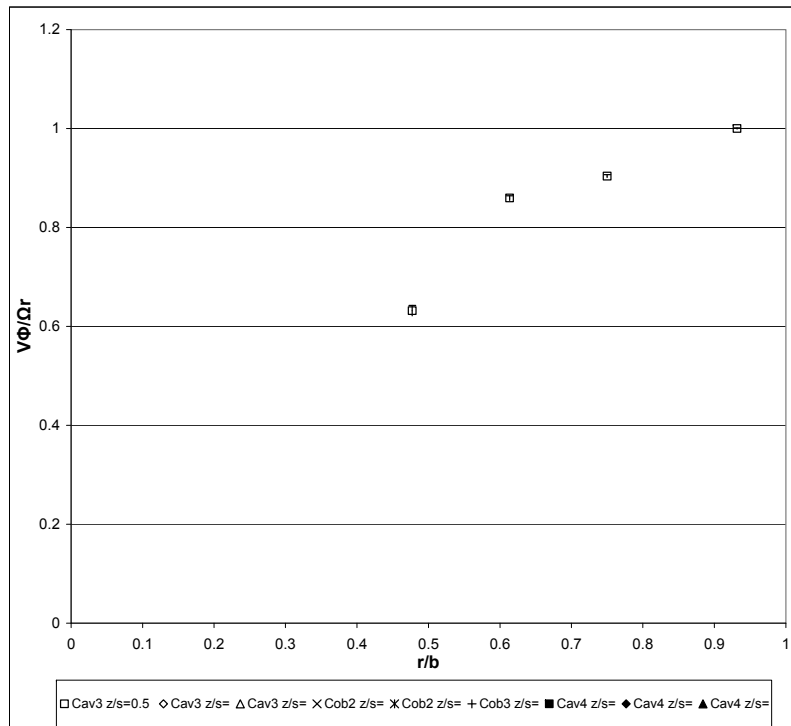
LDA measurement
Build: 2 **LDA code:** 101 **rpm:** 1655.0 **Perfd by:** 0 **Printed:** 27/09/2008 **Page 3/3**
LDA test # : test6 **ReΦ:** 1.09E+06 **Date:** 07/08/2001 **LDA1:** Vφ **Comments:** 0
Heat Transfer run: B2t06 **Rez:** 1.14E+05 **raw LDA file:** \MCR\LDABuild2&1\Projects\mcr3.lda , m **LDA2:** Vz **Traversal corresp:** 0
Ro: 3.56 **LDA list file:** MCR\b2Data\LDA-results\MultiCavityRig\TW (m/s): 55.3

| X [mm] | corrected X | Y [mm] | corrected Y / Radius (mm) | Count{1} | Count{2} | Data Rate{1} [#s] | Data Rate{2} [#s] | Validation n{1} [%] | Validation {2} [%] | LDA1 Mean [m/s] | LDA2 Mean [m/s] | LDA1 RMS [m/s] | LDA2 RMS [m/s] | LDA1 MeanConf [m/s] | LDA2 MeanConf [m/s] | LDA1 RMSConf [m/s] | LDA2 RMSConf [m/s] | Cavity | r/b | z/s | Ωr (m/s) | V _φ /Ωr | V _φ U95%/Ωr | V _φ (RMS)/Ωr | Vz/W | Vz(RMS)/W | Vz _{U95%} /W |
|--------|-------------|--------|---------------------------|----------|----------|-------------------|-------------------|---------------------|--------------------|-----------------|-----------------|----------------|----------------|---------------------|---------------------|--------------------|--------------------|--------|-------|-------|----------|--------------------|------------------------|-------------------------|-------|-----------|-----------------------|
| 75 | | 53 | 88 | 12 | 2 | 1.6 | 2.4 | | | 8.17 | 0.67 | 1.16 | 0.01 | 0.68 | 0.03 | 0.48 | 0.02 | 2 | 0.400 | 0.500 | 15.25 | 0.54 | 4.48E-02 | 0.08 | 0.01 | 0.00 | 5.26E-04 |
| 75 | | 83 | 118 | 19 | 9 | 2.2 | 1.2 | | | 17.51 | 0.44 | 1.91 | 1.05 | 0.88 | 0.72 | 0.62 | 0.51 | 2 | 0.536 | 0.500 | 20.45 | 0.86 | 4.31E-02 | 0.09 | 0.01 | 0.02 | 1.31E-02 |
| 75 | | 113 | 148 | 246 | 291 | 24.8 | 29.5 | | | 24.36 | -0.38 | 1.92 | 1.55 | 0.24 | 0.18 | 0.17 | 0.13 | 2 | 0.673 | 0.500 | 25.65 | 0.95 | 9.37E-03 | 0.07 | -0.01 | 0.03 | 3.22E-03 |
| 75 | | 143 | 178 | 5000 | 4197 | 797.0 | 421.7 | | | 29.93 | -0.30 | 1.67 | 1.53 | 0.05 | 0.05 | 0.03 | 0.03 | 2 | 0.809 | 0.500 | 30.85 | 0.97 | 1.50E-03 | 0.05 | -0.01 | 0.03 | 8.35E-04 |
| 75 | | 173 | 208 | 5000 | 5000 | 1747.6 | 3778.7 | | | 34.97 | -0.27 | 1.57 | 1.63 | 0.04 | 0.05 | 0.03 | 0.03 | 2 | 0.945 | 0.500 | 36.05 | 0.97 | 1.21E-03 | 0.04 | 0.00 | 0.03 | 8.16E-04 |
| 130 | | 53 | 88 | 670 | 23 | 67.4 | 2.6 | | | 4.32 | -0.27 | 3.18 | 0.92 | 0.24 | 0.38 | | | 3 | 0.400 | 0.500 | 15.25 | 0.28 | 1.58E-02 | 0.21 | 0.00 | 0.02 | 6.80E-03 |
| 130 | | 83 | 118 | 320 | 2 | 32.2 | 0.2 | | | 15.37 | -2.07 | 2.60 | 0.28 | 0.28 | 0.39 | | | 3 | 0.536 | 0.500 | 20.45 | 0.75 | 1.39E-02 | 0.13 | -0.04 | 0.01 | 7.02E-03 |
| 130 | | 113 | 148 | 688 | 29 | 68.9 | 2.9 | | | 25.47 | -0.31 | 2.52 | 1.86 | 0.19 | 0.68 | | | 3 | 0.673 | 0.500 | 25.65 | 0.99 | 7.34E-03 | 0.10 | -0.01 | 0.03 | 1.22E-02 |
| 130 | | 143 | 178 | 5000 | 945 | 4999.5 | 94.5 | | | 29.85 | -0.10 | 2.38 | 1.91 | 0.07 | 0.12 | | | 3 | 0.809 | 0.500 | 30.85 | 0.97 | 2.14E-03 | 0.08 | 0.00 | 0.03 | 2.20E-03 |
| 130 | | 173 | 208 | 5000 | 5000 | 24737.0 | 10899.2 | | | 35.46 | -0.11 | 1.37 | 1.47 | 0.04 | 0.04 | | | 3 | 0.945 | 0.500 | 36.05 | 0.98 | 1.05E-03 | 0.04 | 0.00 | 0.03 | 7.37E-04 |
| 175 | | 53 | 88 | 5000 | 5000 | 1074.2 | 853.9 | | | 7.02 | -0.09 | 4.09 | 1.32 | 0.11 | 0.04 | 0.08 | 0.03 | 4 | 0.400 | 0.500 | 15.25 | 0.46 | 7.43E-03 | 0.27 | 0.00 | 0.02 | 6.63E-04 |
| 175 | | 83 | 118 | 5000 | 5000 | 518.5 | 632.7 | | | 21.30 | 0.76 | 3.51 | 1.61 | 0.10 | 0.04 | 0.07 | 0.03 | 4 | 0.536 | 0.500 | 20.45 | 1.04 | 4.76E-03 | 0.17 | 0.01 | 0.03 | 8.06E-04 |
| 175 | | 113 | 148 | 5000 | 5000 | 887.9 | 2633.0 | | | 26.77 | 0.20 | 2.33 | 1.52 | 0.06 | 0.04 | 0.05 | 0.03 | 4 | 0.673 | 0.500 | 25.65 | 1.04 | 2.52E-03 | 0.09 | 0.00 | 0.03 | 7.64E-04 |
| 175 | | 143 | 178 | 5000 | 5000 | 8111.5 | 2777.0 | | | 30.44 | -0.58 | 2.23 | 1.84 | 0.06 | 0.05 | 0.04 | 0.04 | 4 | 0.809 | 0.500 | 30.85 | 0.99 | 2.01E-03 | 0.07 | -0.01 | 0.03 | 9.25E-04 |
| 175 | | 173 | 208 | 5000 | 5000 | 30104.5 | 28704.6 | | | 35.21 | -0.47 | 2.40 | 2.07 | 0.07 | 0.06 | 0.05 | 0.04 | 4 | 0.945 | 0.500 | 36.05 | 0.98 | 1.84E-03 | 0.07 | -0.01 | 0.04 | 1.04E-03 |

| | | | | | | |
|----------------------------|---------------|---|---------------|---------------|---------------------|----------|
| LDA measurement | | LDA code: 101 | rpm: 2925.0 | Perfd by: | Printed: 27/09/2008 | Page 1/3 |
| Build: 2 | ReΦ: 2.02E+06 | Date: 13/08/2001 | LDA1: VΦ | Comments: | | |
| LDA test #: Test7b | Rez: 1.23E+05 | raw LDA file: | LDA2: Vz | Traverse ref: | | |
| Heat Transfer run: B2t07-b | Ro: 2.07 | LDA list file: \\MCR\lb2Data\LDA-results\MultiCavityRig\Test7-250mm\test7b\avg2.txt | W (m/s): 55.4 | | | |



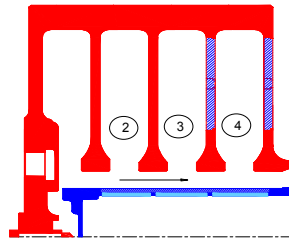
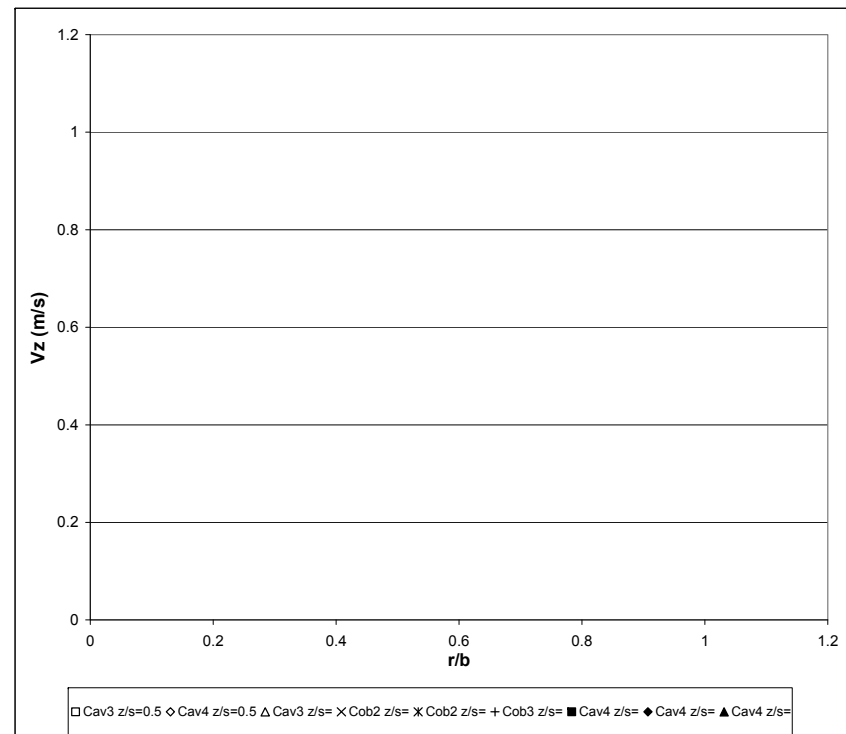
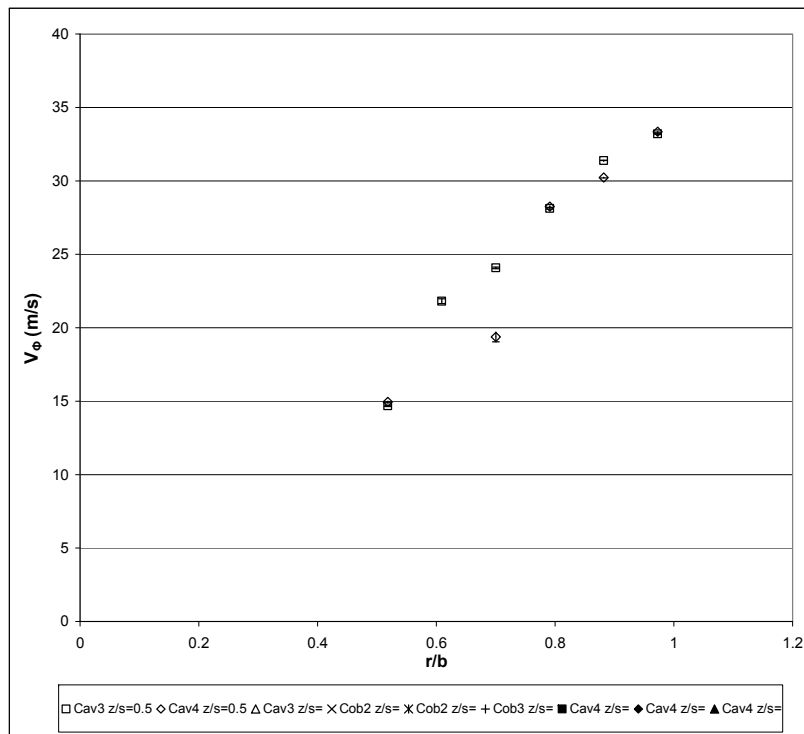
| | | | | | | |
|----------------------------|----------------------------|--|---------------|---------------------|---------------------|----------|
| LDA measurement | | LDA code: 101 | rpm: 2925.0 | Perfd by: 0 | Printed: 27/09/2008 | Page 2/3 |
| Build: 2 | Re _φ : 2.02E+06 | Date: 13/08/2001 | LDA1: Vφ | Comments: 0 | | |
| LDA test #: Test7b | Re _z : 1.23E+05 | raw LDA file: 0 | LDA2: Vz | Traverse corresp: 0 | | |
| Heat Transfer run: B2t07-b | Ro: 2.07 | LDA list file: \MCR\b2Data\LDA-results\MultiCavityRig\ | W (m/s): 55.4 | | | |



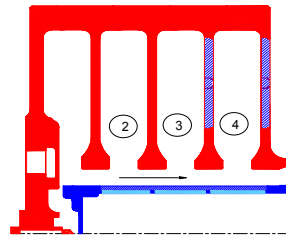
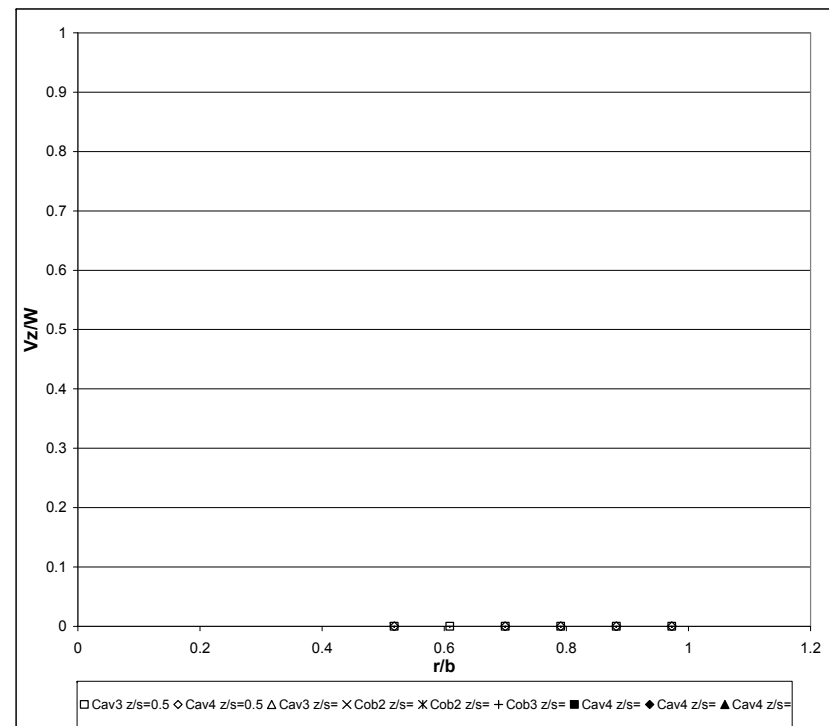
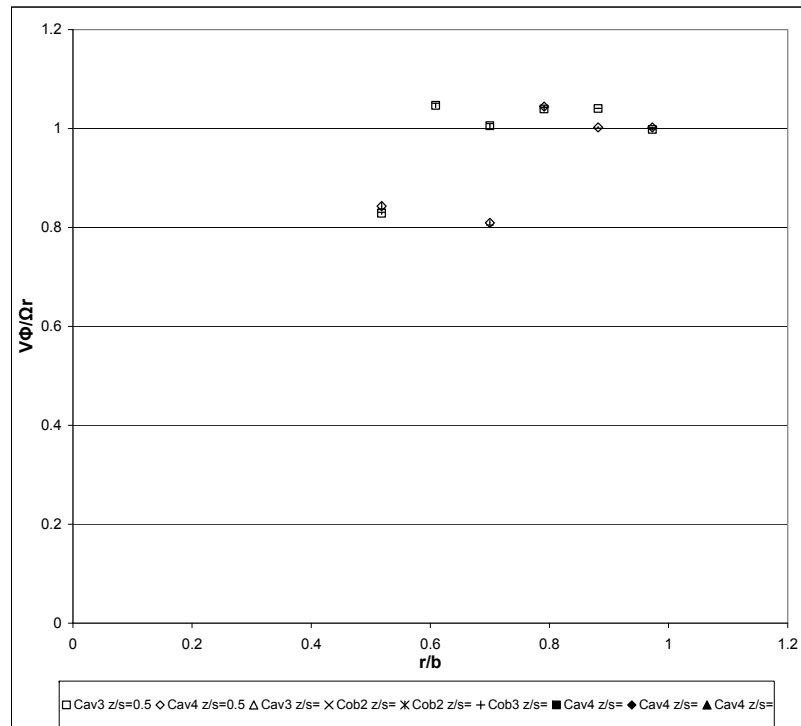
LDA measurement
Build: 2 **LDA code:** 101 **rpm:** 2925.0 **Perfd by:** 0 Printed: 27/09/2008 Page 3/3
LDA test # : Test7b **ReΦ:** 2.02E+06 **Date:** 13/08/2001 **LDA1:** VΦ **Comments:** 0
Heat Transfer run: B2t07-b **Rez:** 1.23E+05 **raw LDA file:** 0 **LDA2:** Vz **Traverse corresp:** 0
Ro: 2.07 **LDA list file:** \MCR\b2Data\LDA-results\MultiCavityRig\ **W (m/s):** 55.4

| X [mm] | correcte | Y [mm] | corrected Y / | Count{1} | Count{2} | Data Rate{1} | Data Rate{2} | Validatio | Validation | LDA1 | LDA2 | LDA1 | LDA2 | LDA1 | LDA2 | LDA1 | LDA2 | Cavity | r/b | z/s | Ωr (m/s) | V _φ /Ωr | V _{φ(RMS)} /Ωr | Vz/W | Vz _(RMS) / | Vz _{U95%} /W |
|--------|----------|--------|---------------|----------|----------|---------------|---------------|-----------|------------|------------|------------|-----------|-----------|----------------|----------------|----------------|----------------|----------|------|------|----------|--------------------|-------------------------|------|-----------------------|-----------------------|
| | d X | | Radius (mm) | | | Rate{1} [#/s] | Rate{2} [#/s] | n{1} [%] | {2} [%] | Mean [m/s] | Mean [m/s] | RMS [m/s] | RMS [m/s] | MeanConf [m/s] | MeanConf [m/s] | RMSCConf [m/s] | RMSCConf [m/s] | | | | | W | | | | |
| 125 | 70 | 105 | 373 | 911 | 6.4 | 15.3 | 20.32 | -0.04 | 3.34 | 0.21 | 0.34 | 0.01 | 3 | 0.477 | 0.500 | 32.16 | 0.63 | 1.05E-02 | 0.10 | 0.00 | 0.00 | 2.46E-04 | | | | |
| 125 | 100 | 135 | 1225 | 2000 | 20.4 | 91.4 | 35.54 | -0.06 | 3.52 | 0.33 | 0.20 | 0.01 | 3 | 0.614 | 0.500 | 41.35 | 0.86 | 4.77E-03 | 0.09 | 0.00 | 0.01 | 2.61E-04 | | | | |
| 125 | 130 | 165 | 1578 | 1671 | 26.3 | 27.9 | 45.67 | -0.03 | 2.88 | 0.33 | 0.14 | 0.02 | 3 | 0.750 | 0.500 | 50.54 | 0.90 | 2.81E-03 | 0.06 | 0.00 | 0.01 | 2.86E-04 | | | | |
| 125 | 170 | 205 | 2000 | 708 | 52.7 | 11.8 | 62.79 | 0.02 | 1.52 | 2.16 | 0.07 | 0.16 | 3 | 0.932 | 0.500 | 62.79 | 1.00 | 1.06E-03 | 0.02 | 0.00 | 0.04 | 2.87E-03 | | | | |

| | | | | | |
|----------------------------|----------------------|--|----------------|---------------------|----------|
| LDA measurement | LDA code: 100 | rpm: 1485.0 | Perfd by: | Printed: 27/09/2008 | Page 1/3 |
| Build: 2 | Re Φ : 1.05E+06 | Date: 17/10/2001 | LDA1: V Φ | Comments: | |
| LDA test #: Test13 | Rez: 1.25E+05 | raw LDA file: | LDA2: | Traverse ref: | |
| Heat Transfer run: B2t13-b | Ro: 4.10 | LDA list file: MCR\b2Data\LDA-results\MultiCavityRig\Test13\avgr | W (m/s): 56.1 | | |



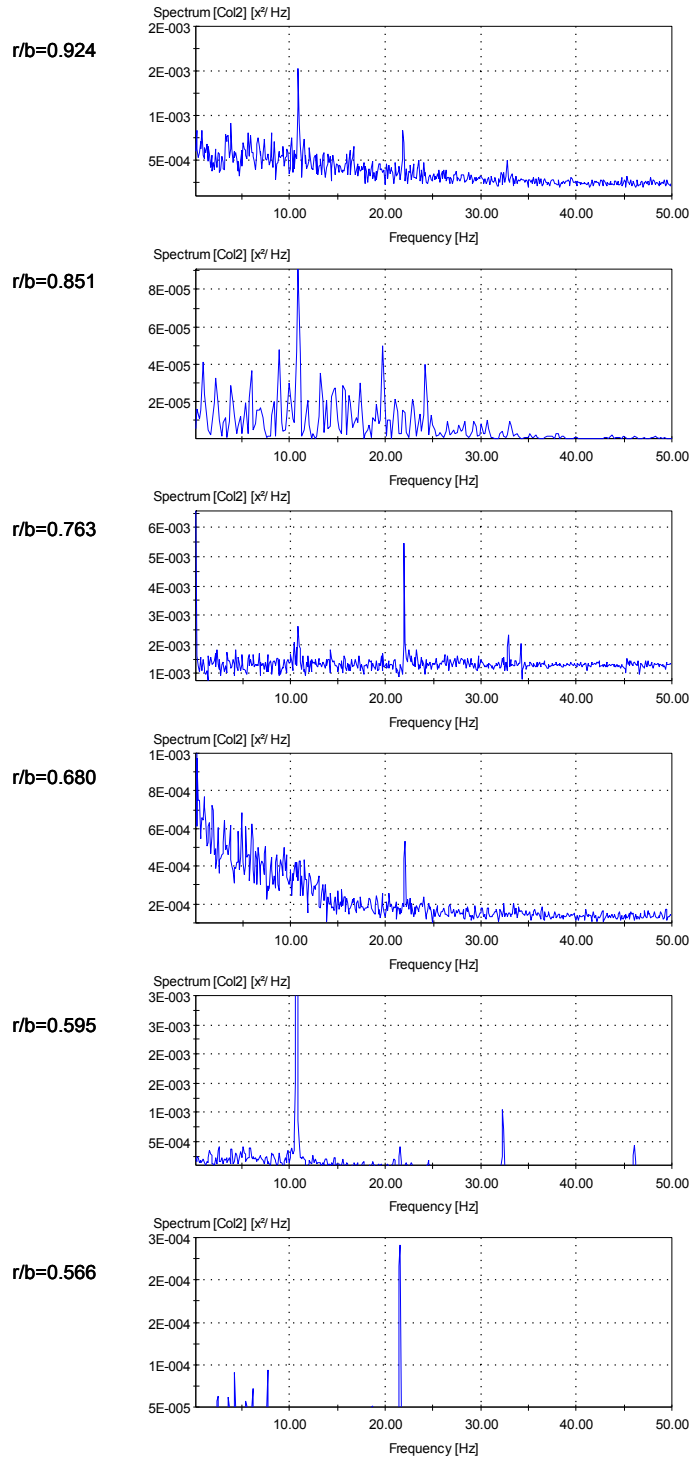
| | | | | | | |
|-----------------------------------|--|---------------------------------|---|-----------------|----------------------------|----------|
| LDA measurement | | LDA code: 101 | rpm: 1485.0 | Perfd by: 0 | Printed: 27/09/2008 | Page 2/3 |
| Build: 2 | | Re_φ: 1.05E+06 | Date: 17/10/2001 | LDA1: Vφ | Comments: 0 | |
| LDA test # : Test13 | | Re_z: 1.25E+05 | raw LDA file: 0 | LDA2: 0 | Traverse corresp: 0 | |
| Heat Transfer run: B2t13-b | | Ro: 4.10 | LDA list file: MCR\b2Data\LDA-results\MultiCavityRig\TW (m/s): 56.1 | | | |



LDA measurement
Build: 2 **LDA code:** 100 **rpm:** 1485.0 **Perfd by:** 0 **Printed:** 27/09/2008 **Page 3/3**
ReΦ: 1.05E+06 **Date:** 17/10/2001 **LDA1:** VΦ **Comments:** 0
LDA test # : Test13 **Rez:** 1.25E+05 **raw LDA file:** 0 **LDA2:** 0 **Traverse corresp:** 0
Heat Transfer run: B2t13-b **Ro:** 4.10 **LDA list file:** MCR\b2Data\LDA-results\MultiCavityRig\1W (m/s): 56.1

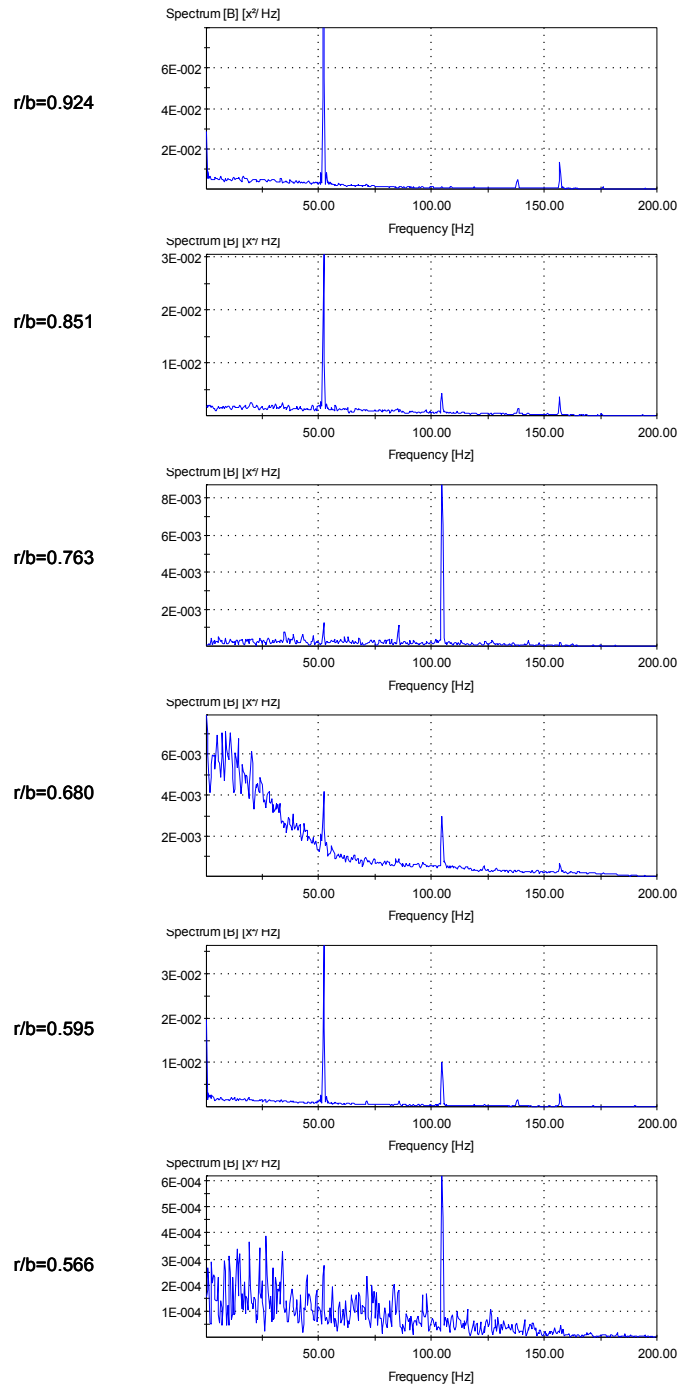
| X [mm] | corrected X | Y [mm] | corrected Y / Radius (mm) | Count{1} | Count{2} | Data Rate{1} [#/s] | Data Rate{2} [#/s] | Validation {1} [%] | Validation {2} [%] | LDA1 Mean [m/s] | LDA2 Mean [m/s] | LDA1 RMS [m/s] | LDA2 RMS [m/s] | LDA1 MeanConf [m/s] | LDA2 MeanConf [m/s] | LDA1 RMSConf [m/s] | LDA2 RMSConf [m/s] | Cavity | r/b | z/s | Ωr (m/s) | V _φ /Ωr | V _φ 95%/Ωr | V _φ (RMS)/Ωr | Vz/W | W | Vz _{95%} /W |
|--------|-------------|--------|---------------------------|----------|----------|--------------------|--------------------|--------------------|--------------------|-----------------|-----------------|----------------|----------------|---------------------|---------------------|--------------------|--------------------|--------|-------|-------|----------|--------------------|-----------------------|-------------------------|------|------|----------------------|
| 175 | | 180 | 214 | 5000 | | 7861.8 | | | | 33.35 | | 0.89 | | 0.02 | | | | 4 | 0.973 | 0.500 | 33.28 | 1.00 | 7.41E-04 | 0.03 | 0.00 | 0.00 | 0.00E+00 |
| 175 | | 160 | 194 | 745 | | 24.8 | | | | 30.23 | | 0.64 | | 0.05 | | | | 4 | 0.882 | 0.500 | 30.17 | 1.00 | 1.52E-03 | 0.02 | 0.00 | 0.00 | 0.00E+00 |
| 175 | | 140 | 174 | 5000 | | 206.1 | | | | 28.25 | | 1.43 | | 0.04 | | | | 4 | 0.791 | 0.500 | 27.06 | 1.04 | 1.46E-03 | 0.05 | 0.00 | 0.00 | 0.00E+00 |
| 175 | | 120 | 154 | 20000 | | 3204.5 | | | | 19.38 | | 7.86 | | 0.11 | | | | 4 | 0.700 | 0.500 | 23.95 | 0.81 | 4.55E-03 | 0.33 | 0.00 | 0.00 | 0.00E+00 |
| 175 | | 100 | 134 | 216 | | 7.3 | | | | 21.10 | | 3.61 | | 0.48 | | | | 4 | 0.609 | 0.520 | 20.84 | 1.01 | 2.31E-02 | 0.17 | 0.00 | 0.00 | 0.00E+00 |
| 175 | | 80 | 114 | 360 | | 12.0 | | | | 14.95 | | 0.98 | | 0.10 | | | | 4 | 0.518 | 0.500 | 17.73 | 0.84 | 5.71E-03 | 0.06 | 0.00 | 0.00 | 0.00E+00 |
| 125 | | 180 | 214 | 5000 | | 21080.9 | | | | 33.21 | | 1.08 | | 0.03 | | | | 3 | 0.973 | 0.500 | 33.28 | 1.00 | 9.00E-04 | 0.03 | 0.00 | 0.00 | 0.00E+00 |
| 125 | | 160 | 194 | 863 | | 14.4 | | | | 31.39 | | 0.15 | | 0.01 | | | | 3 | 0.882 | 0.500 | 30.17 | 1.04 | 3.32E-04 | 0.00 | 0.00 | 0.00 | 0.00E+00 |
| 125 | | 140 | 174 | 824 | | 13.8 | | | | 28.13 | | 1.37 | | 0.09 | | | | 3 | 0.791 | 0.500 | 27.06 | 1.04 | 3.46E-03 | 0.05 | 0.00 | 0.00 | 0.00E+00 |
| 125 | | 120 | 154 | 567 | | 19.0 | | | | 24.08 | | 1.32 | | 0.11 | | | | 3 | 0.700 | 0.500 | 23.95 | 1.01 | 4.54E-03 | 0.06 | 0.00 | 0.00 | 0.00E+00 |
| 125 | | 100 | 134 | 5000 | | 90.9 | | | | 21.81 | | 3.36 | | 0.09 | | | | 3 | 0.609 | 0.500 | 20.84 | 1.05 | 4.47E-03 | 0.16 | 0.00 | 0.00 | 0.00E+00 |
| 125 | | 80 | 114 | 1690 | | 28.2 | | | | 14.69 | | 0.77 | | 0.04 | | | | 3 | 0.518 | 0.500 | 17.73 | 0.83 | 2.07E-03 | 0.04 | 0.00 | 0.00 | 0.00E+00 |

Appendix 6.5: Spectral analysis plots



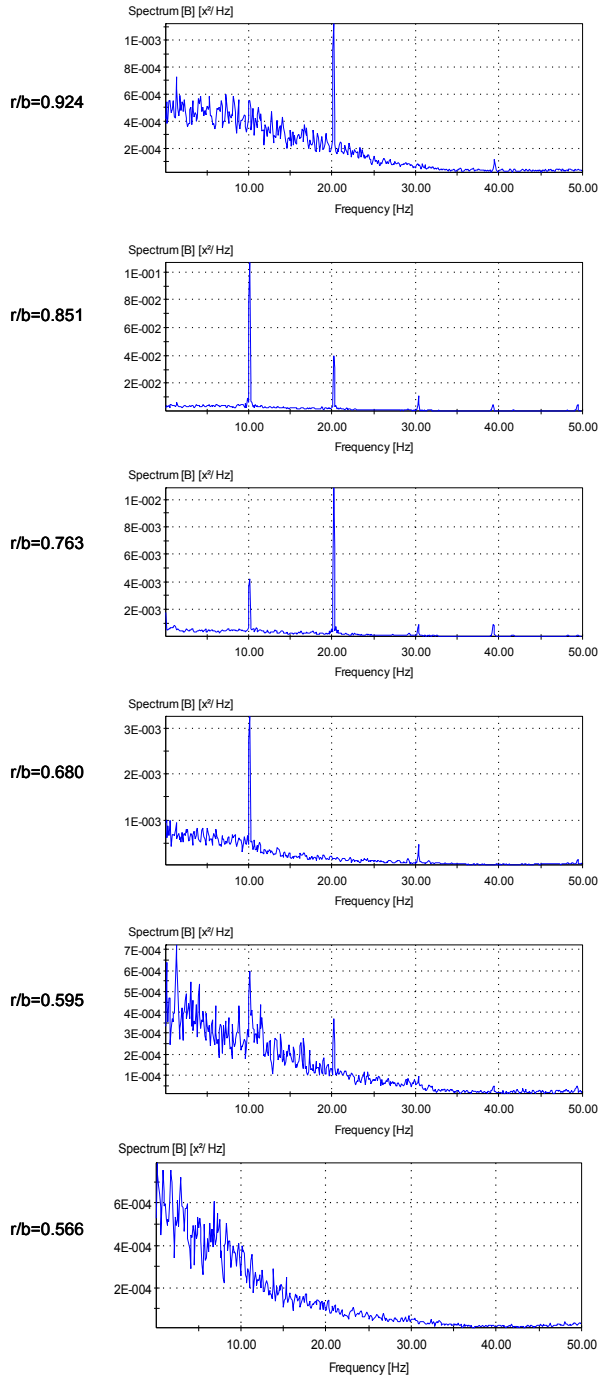
Project: SPECT_R38_pos50 - Pos: 0.00;0.00;0.00 - Date/Time: #N/A

Build 3, Run 38-1, Cavity 3, $r/b=0.508$, $Re_\phi=0.4 \times 10^6$, $Re_z=1.57 \times 10^5$, $Ro=7.51$



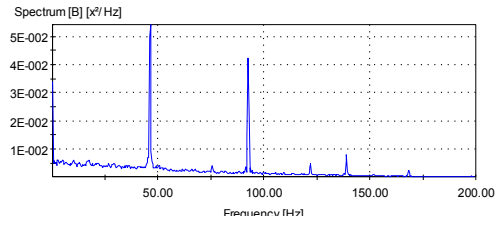
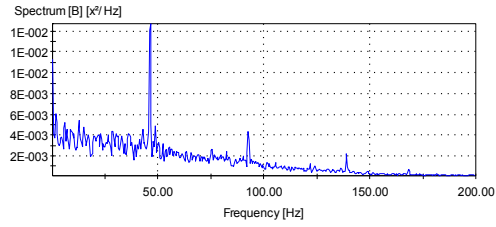
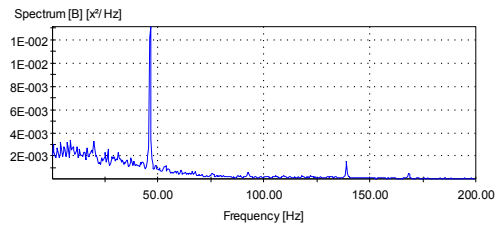
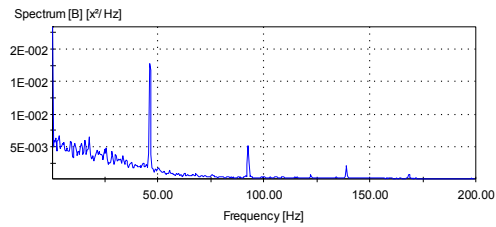
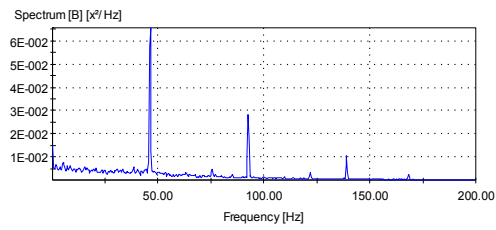
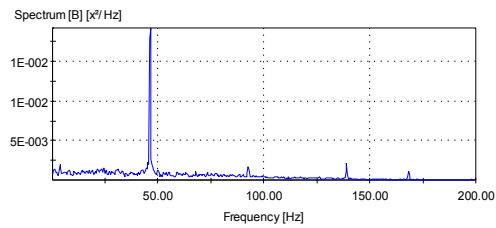
Project: SPECT_R39_pos50 - Pos: 0.00;0.00;0.00 - Date/Time: #N/A

Build 3, Run 39-1, Cavity 3, $r/b=0.508$, $Re_\phi=1.93 \times 10^6$, $Re_z=0.53 \times 10^5$, $Ro=0.53$



Project: SPECT_R40_pos50 - Pos: 0.00;0.00;0.00 - Date/Time: #N/A

Build 3, Run 40-1, Cavity 3, $r/b=0.508$, $Re_\phi=0.39 \times 10^6$, $Re_z=0.61 \times 10^5$, $Ro=2.98$

$r/b=0.924$  $r/b=0.851$  $r/b=0.763$  $r/b=0.680$  $r/b=0.595$  $r/b=0.566$ 

Project: SPECT_R44_pos50 - Pos: 0.00;0.00;0.00 - Date/Time: #N/A

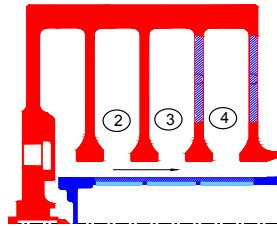
Build 3, Run 44-1, Cavity 3, $r/b=0.508$, $Re_\phi=1.99 \times 10^6$, $Re_z=1.65 \times 10^5$, $Ro=1.58$

Appendix 7.1: Selected heat transfer test reports

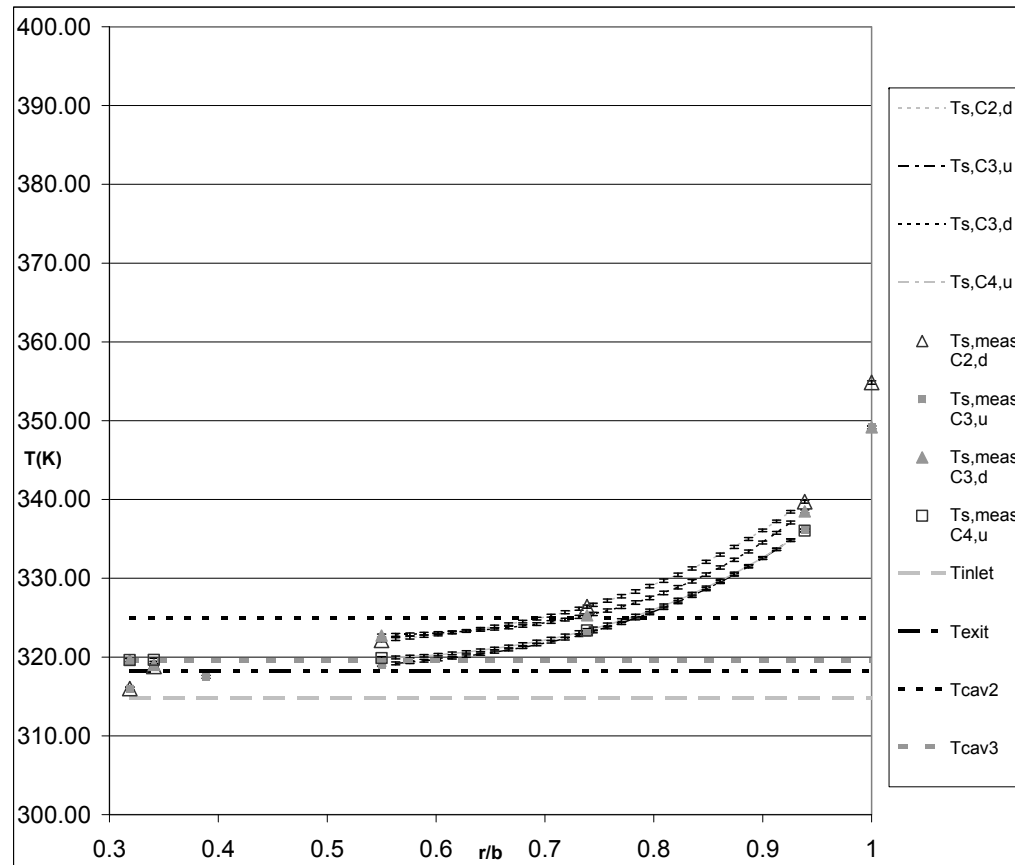
MULTIPLE CAVITY RIG

Build: 3 ExpDate: 11/12/2003
Exp Code: 3S----- Run: B3run28-1

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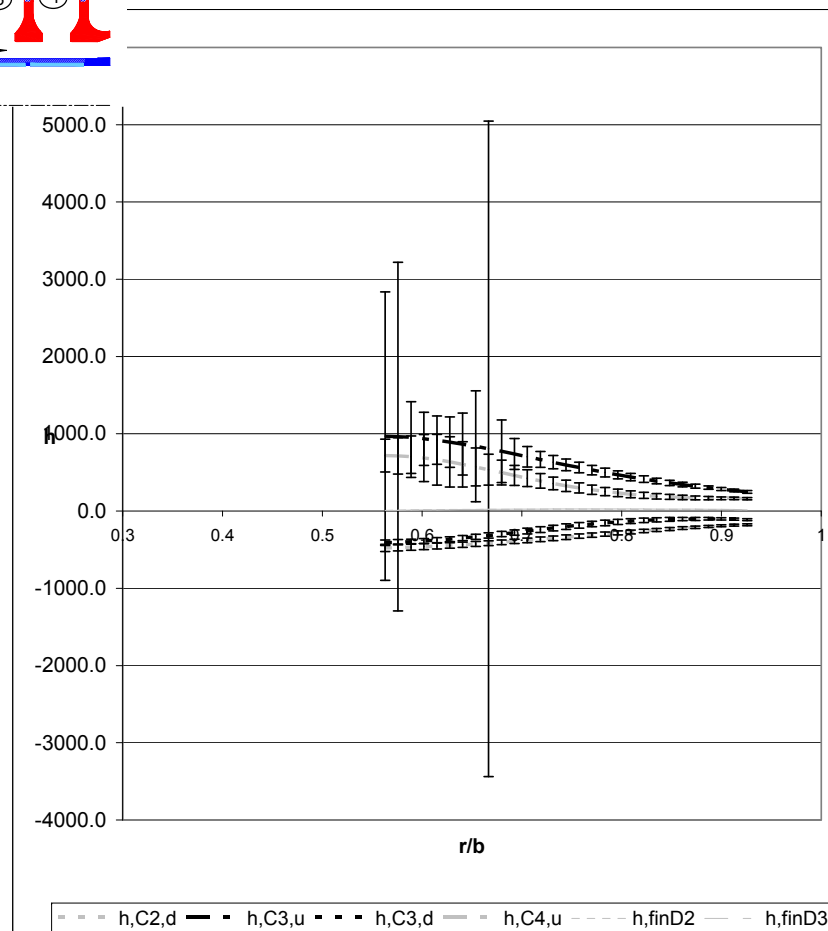
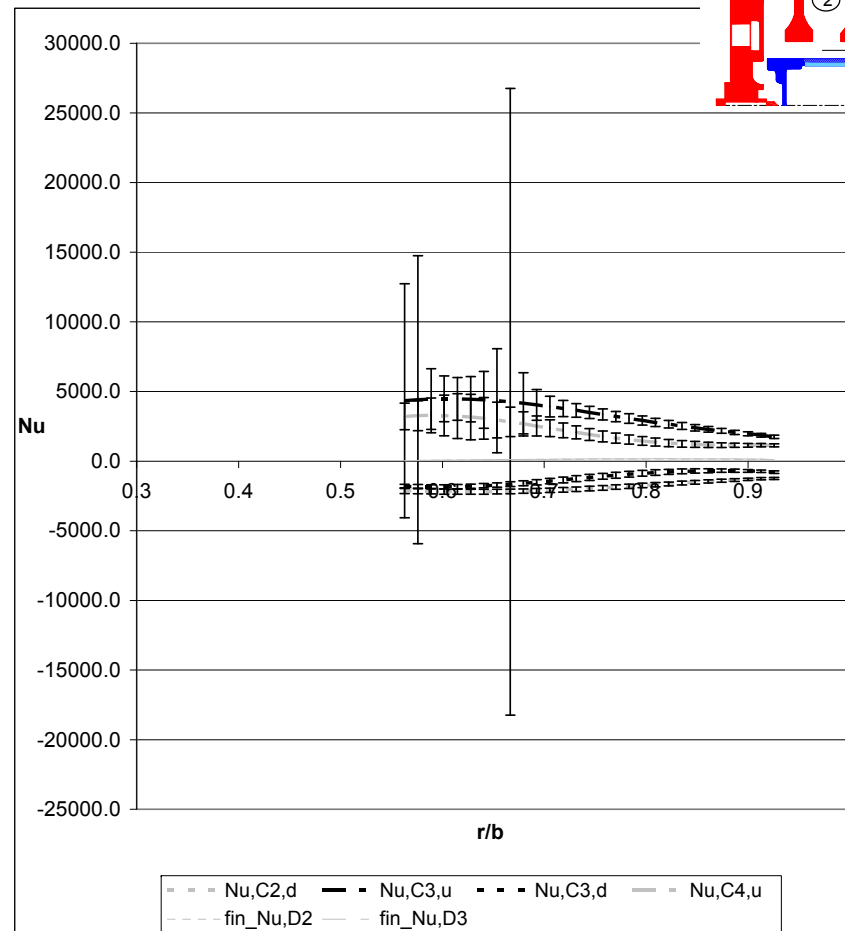
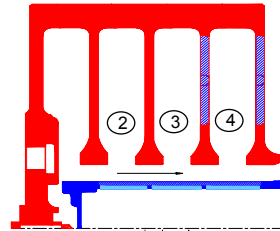
| | | | |
|------------------|--------------------|------------------|--------------------|
| Rez: | 1.16E+05 ± 6.9E+01 | ReΦ: | 4.12E+06 ± 5.7E+03 |
| Ro: | 0.54 ± 4.4E-04 | RotSpeed (rpm): | 4552.2 ± 1.0E-01 |
| Mflow (kg/s): | 0.425 | Pcav (abs Pa): | 3.094E+05 |
| Bo*: | 0.308 ± 4.7E-03 | Pr: | 0.706 ± 3.9E-05 |
| Tinlet (K): | 314.83 ± 2.4E-01 | Texit(K): | 318.29 ± 2.6E-01 |
| Tcav2 (K): | 324.92 ± 6.7E-01 | Tcav3 (K): | 319.62 ± 6.7E-01 |
| ΔTmeas (K): | 3.45 ± 3.3E-01 | ΔTcalc (K): | 3.28 ± 5.5E-02 |
| βΔTav: | 0.027 ± 7.7E-04 | finβΔT av: | 0.027 ± 7.7E-04 |
| Ø: | 1.646 ± 4.8E+00 | finØ: | 1.119 ± 5.1E+00 |
| Ts,C2, d,av(K): | 325.12 ± 1.4E-01 | Ts,C3, u,av(K): | 321.97 ± 1.2E-01 |
| Ts,C3, d,av(K): | 324.19 ± 1.4E-01 | Ts,C4, u,av(K): | 322.26 ± 1.6E-01 |
| finTs, D2,av(K): | 323.54 ± 8.9E-02 | finTs, D3,av(K): | 323.22 ± 1.1E-01 |
| Nu,C2, d,av: | -2103.6 ± 1.3E+02 | Nu,C3, u,av: | 3453.3 ± 2.8E+02 |
| Nu,C3, d,av: | -1331.1 ± 1.5E+02 | Nu,C4, u,av: | 2076.0 ± 2.6E+02 |
| finNu, D2,av: | 152.0 ± 1.9E+01 | finNu, D3,av: | 168.5 ± 1.9E+01 |
| Tsh,C2 (K): | 354.86 ± 2.4E-01 | Tsh,C3 (K): | 349.15 ± 2.3E-01 |
| Nush,C2: | 99.7 ± 2.5E+00 | Nush,C3: | 113.2 ± 3.0E+00 |
| Grsh,C2: | 1.63E+09 ± 1.9E+07 | Grsh,C3: | 1.37E+09 ± 1.7E+07 |
| g2: | 1.34 ± 2.9E-02 | g3: | 1.16 ± 2.5E-02 |



MULTIPLE CAVITY RIG

Build: 3 ExpDate: 11/12/2003
 Exp Code: 3S----- Run: B3run28-1

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MULTIPLE CAVITY RIG

```
Build:      3      ExpDate:    11/12/2003
Exp Code:   3S----- Run:      B3run28-1
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Printed: 27/09/2008

| Radius(m) | non-dim radius | Tsh/cob D2 | ± | Tsh/cob D3 | ± | TC,C2, dn (K) | ± | TC,C3, up (K) | ± | TC,C3, dn (K) | ± | TC,C4, up (K) | ± | Ts,C2, dn (K) | ± | Ts,C3, up (K) | ± | Ts,C3, dn (K) | ± | Ts,C4, up (K) | ± | finTs, D2 (K) | ± | finTs, D3 (K) | ± |
|-----------|-------------------|------------|------|------------|------|------------------|------|------------------|------|------------------|------|------------------|------|------------------|------|------------------|------|------------------|------|------------------|------|------------------|------|---------------|------|
| 0.0701 | 0.31864 | 316.01 | 0.18 | 319.63 | 0.20 | | | | | | | | | | | | | | | | | | | | |
| 0.075 | 0.34091 | | | | | 318.82 | 0.20 | | | 319.00 | 0.20 | 319.65 | 0.20 | | | | | | | | | | | | |
| 0.0855 | 0.38864 | | | | | | | 317.50 | 0.18 | | | | | | | | | | | | | | | | |
| 0.121 | 0.55 | | | | | 322.12 | 0.19 | 319.07 | 0.19 | 322.73 | 0.20 | 319.88 | 0.18 | | | | | | | | | | | | |
| 0.12385 | 0.56295 | | | | | | | | | | | | | 322.29 | 0.18 | 319.21 | 0.19 | 322.81 | 0.19 | 319.96 | 0.18 | 320.75 | 0.12 | 321.39 | 0.14 |
| 0.1267 | 0.57591 | | | | | | | | | | | | | 322.47 | 0.17 | 319.36 | 0.18 | 322.89 | 0.18 | 320.06 | 0.17 | 320.92 | 0.12 | 321.47 | 0.13 |
| 0.12955 | 0.58886 | | | | | | | | | | | | | 322.67 | 0.17 | 319.53 | 0.17 | 322.98 | 0.17 | 320.17 | 0.16 | 321.10 | 0.11 | 321.57 | 0.12 |
| 0.1324 | 0.60182 | | | | | | | | | | | | | 322.88 | 0.16 | 319.71 | 0.16 | 323.07 | 0.16 | 320.31 | 0.16 | 321.29 | 0.11 | 321.69 | 0.12 |
| 0.13525 | 0.61477 | | | | | | | | | | | | | 323.10 | 0.16 | 319.91 | 0.15 | 323.17 | 0.15 | 320.47 | 0.16 | 321.50 | 0.10 | 321.82 | 0.11 |
| 0.1381 | 0.62773 | | | | | | | | | | | | | 323.34 | 0.16 | 320.12 | 0.15 | 323.29 | 0.15 | 320.65 | 0.16 | 321.73 | 0.10 | 321.97 | 0.11 |
| 0.14095 | 0.64068 | | | | | | | | | | | | | 323.60 | 0.16 | 320.36 | 0.14 | 323.42 | 0.15 | 320.86 | 0.16 | 321.98 | 0.10 | 322.14 | 0.11 |
| 0.1438 | 0.65364 | | | | | | | | | | | | | 323.88 | 0.16 | 320.62 | 0.14 | 323.57 | 0.15 | 321.09 | 0.17 | 322.25 | 0.10 | 322.33 | 0.11 |
| 0.14665 | 0.66659 | | | | | | | | | | | | | 324.19 | 0.16 | 320.91 | 0.14 | 323.75 | 0.15 | 321.35 | 0.17 | 322.55 | 0.10 | 322.55 | 0.12 |
| 0.1495 | 0.67955 | | | | | | | | | | | | | 324.52 | 0.17 | 321.23 | 0.14 | 323.95 | 0.16 | 321.64 | 0.18 | 322.87 | 0.10 | 322.80 | 0.12 |
| 0.15235 | 0.6925 | | | | | | | | | | | | | 324.87 | 0.18 | 321.57 | 0.15 | 324.19 | 0.16 | 321.96 | 0.19 | 323.22 | 0.11 | 323.08 | 0.13 |
| 0.1552 | 0.70545 | | | | | | | | | | | | | 325.26 | 0.18 | 321.94 | 0.15 | 324.45 | 0.17 | 322.32 | 0.20 | 323.60 | 0.11 | 323.39 | 0.13 |
| 0.15805 | 0.71841 | | | | | | | | | | | | | 325.68 | 0.19 | 322.35 | 0.16 | 324.76 | 0.18 | 322.71 | 0.21 | 324.02 | 0.12 | 323.73 | 0.14 |
| 0.1609 | 0.73136 | | | | | | | | | | | | | 326.14 | 0.20 | 322.79 | 0.17 | 325.10 | 0.19 | 323.13 | 0.22 | 324.46 | 0.12 | 324.11 | 0.15 |
| 0.1625 | 0.73864 | | | | | 326.41 | 0.20 | 323.05 | 0.18 | 325.31 | 0.20 | 323.39 | 0.22 | | | | | | | | | | | | |
| 0.16375 | 0.74432 | | | | | | | | | | | | | 326.63 | 0.20 | 323.27 | 0.18 | 325.48 | 0.20 | 323.59 | 0.23 | 324.95 | 0.13 | 324.54 | 0.16 |
| 0.1666 | 0.75727 | | | | | | | | | | | | | 327.16 | 0.21 | 323.78 | 0.19 | 325.91 | 0.21 | 324.09 | 0.23 | 325.47 | 0.14 | 325.00 | 0.16 |
| 0.16945 | 0.77023 | | | | | | | | | | | | | 327.72 | 0.21 | 324.34 | 0.20 | 326.39 | 0.21 | 324.63 | 0.24 | 326.03 | 0.14 | 325.51 | 0.17 |
| 0.1723 | 0.78318 | | | | | | | | | | | | | 328.34 | 0.22 | 324.93 | 0.20 | 326.92 | 0.22 | 325.21 | 0.24 | 326.63 | 0.15 | 326.07 | 0.17 |
| 0.17515 | 0.79614 | | | | | | | | | | | | | 328.99 | 0.22 | 325.58 | 0.21 | 327.51 | 0.22 | 325.84 | 0.24 | 327.28 | 0.15 | 326.67 | 0.17 |
| 0.178 | 0.80909 | | | | | | | | | | | | | 329.69 | 0.22 | 326.26 | 0.21 | 328.15 | 0.22 | 326.51 | 0.24 | 327.98 | 0.15 | 327.33 | 0.17 |
| 0.18085 | 0.82205 | | | | | | | | | | | | | 330.44 | 0.21 | 327.00 | 0.21 | 328.85 | 0.22 | 327.23 | 0.23 | 328.72 | 0.15 | 328.04 | 0.17 |
| 0.1837 | 0.835 | | | | | | | | | | | | | 331.24 | 0.21 | 327.79 | 0.20 | 329.62 | 0.22 | 327.99 | 0.23 | 329.52 | 0.15 | 328.81 | 0.16 |
| 0.18655 | 0.84795 | | | | | | | | | | | | | 332.10 | 0.20 | 328.62 | 0.20 | 330.46 | 0.21 | 328.81 | 0.22 | 330.36 | 0.14 | 329.63 | 0.16 |
| 0.1894 | 0.86091 | | | | | | | | | | | | | 333.01 | 0.19 | 329.52 | 0.19 | 331.37 | 0.20 | 329.67 | 0.20 | 331.26 | 0.14 | 330.52 | 0.15 |
| 0.19225 | 0.87386 | | | | | | | | | | | | | 333.97 | 0.18 | 330.46 | 0.18 | 332.35 | 0.20 | 330.59 | 0.19 | 332.22 | 0.13 | 331.47 | 0.14 |
| 0.1951 | 0.88682 | | | | | | | | | | | | | 335.00 | 0.17 | 331.47 | 0.17 | 333.41 | 0.19 | 331.57 | 0.17 | 333.23 | 0.12 | 332.49 | 0.13 |
| 0.19795 | 0.89977 | | | | | | | | | | | | | 336.08 | 0.17 | 332.53 | 0.15 | 334.55 | 0.18 | 332.60 | 0.16 | 334.31 | 0.11 | 333.57 | 0.12 |
| 0.2008 | 0.91273 | | | | | | | | | | | | | 337.23 | 0.16 | 333.66 | 0.15 | 335.77 | 0.18 | 333.68 | 0.15 | 335.44 | 0.11 | 334.73 | 0.11 |
| 0.20365 | 0.92568 | | | | | | | | | | | | | 338.45 | 0.17 | 334.85 | 0.15 | 337.08 | 0.18 | 334.83 | 0.15 | 336.65 | 0.11 | 335.95 | 0.11 |
| 0.2065 | 0.93864 | | | | | 339.73 | 0.18 | 336.10 | 0.17 | 338.48 | 0.19 | 336.03 | 0.17 | | | | | | | | | | | | |
| 0.22 | 1 | 354.86 | 0.22 | 349.15 | 0.18 | | | | | | | | | | | | | | | | | | | | |
| 0.24565 | | 369.07 | 0.19 | 362.66 | 0.19 | | | | | | | | | | | | | | | | | | | | |

MULTIPLE CAVITY RIG

```
Build:      0      ExpDate:    11/12/2003
Exp Code:   0      Run:        B3run28-1
```

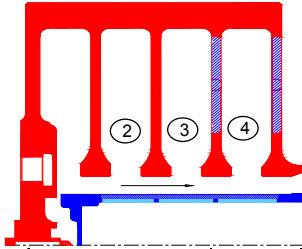
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| Radius(m) | non-dim radius | Nu_r, C2, d = | Nu_r, C3,u = | Nu_r, C3,d = | Nu_r, C4,u = | finNu_r, D2 = | finNu_r, D3 = | h_r, C2,d = | h_r, C3,u = | h_r, C3,d = | h_r, C4,u = | fin-h, D2 = | fin-h, D3 = | | | | | | | | | | | | |
|-----------|-------------------|---------------|--------------|--------------|--------------|---------------|---------------|-------------|-------------|-------------|-------------|-------------|-------------|--------|------|-------|--------|--------|------|-------|-------|------|-------|------|-------|
| 0.0701 | 0.31864 | | | | | | | | | | | | | | | | | | | | | | | | |
| 0.075 | 0.34091 | | | | | | | | | | | | | | | | | | | | | | | | |
| 0.0855 | 0.38864 | | | | | | | | | | | | | | | | | | | | | | | | |
| 0.121 | 0.55 | | | | | | | | | | | | | | | | | | | | | | | | |
| 0.12385 | 0.56295 | -2117.9 | 204.5 | 4340.5 | 8400.7 | -1815.1 | 155.2 | 3206.4 | 952.2 | 38.7 | 770.3 | 27.9 | 646.8 | -477.0 | 46.1 | 969.5 | 1867.5 | -409.3 | 35.1 | 717.6 | 212.5 | 8.7 | 172.6 | 6.3 | 145.3 |
| 0.1267 | 0.57591 | -2141.0 | 200.9 | 4410.1 | 10335.3 | -1833.4 | 153.7 | 3264.4 | 1074.6 | 55.2 | 192.3 | 42.3 | 163.2 | -471.6 | 44.3 | 963.3 | 2256.9 | -404.2 | 33.9 | 714.4 | 234.6 | 12.1 | 42.1 | 9.3 | 35.8 |
| 0.12955 | 0.58886 | -2163.2 | 196.3 | 4452.7 | 2174.2 | -1842.4 | 152.6 | 3285.5 | 1254.8 | 66.2 | 67.8 | 53.6 | 54.6 | -466.2 | 42.4 | 951.6 | 463.5 | -397.4 | 33.0 | 703.4 | 267.9 | 14.2 | 14.5 | 11.5 | 11.7 |
| 0.1324 | 0.60182 | -2181.1 | 190.9 | 4472.5 | 1647.8 | -1840.6 | 152.3 | 3274.1 | 1456.9 | 75.2 | 30.1 | 64.0 | 20.1 | -460.2 | 40.3 | 935.7 | 343.7 | -388.5 | 32.2 | 686.1 | 304.5 | 15.8 | 6.3 | 13.5 | 4.2 |
| 0.13525 | 0.61477 | -2193.4 | 184.8 | 4470.4 | 1532.6 | -1827.1 | 153.2 | 3232.2 | 1606.0 | 83.6 | 18.2 | 74.2 | 9.3 | -453.3 | 38.2 | 916.1 | 313.0 | -377.7 | 31.7 | 663.3 | 328.6 | 17.2 | 3.7 | 15.3 | 1.9 |
| 0.1381 | 0.62773 | -2199.5 | 178.5 | 4446.7 | 1626.4 | -1801.7 | 155.6 | 3162.5 | 1623.2 | 91.5 | 14.3 | 84.4 | 6.6 | -445.5 | 36.2 | 892.9 | 325.4 | -364.9 | 31.6 | 635.9 | 325.4 | 18.5 | 2.9 | 17.0 | 1.3 |
| 0.14095 | 0.64068 | -2199.0 | 172.5 | 4401.9 | 2040.0 | -1764.6 | 159.6 | 3067.9 | 1493.3 | 99.1 | 12.6 | 94.4 | 6.3 | -436.7 | 34.3 | 866.6 | 400.1 | -350.2 | 31.7 | 604.8 | 293.4 | 19.6 | 2.5 | 18.7 | 1.2 |
| 0.1438 | 0.65364 | -2191.8 | 167.2 | 4337.1 | 3730.7 | -1716.3 | 165.3 | 2952.2 | 1277.9 | 106.2 | 11.7 | 104.3 | 6.4 | -426.9 | 32.6 | 837.5 | 717.5 | -334.0 | 32.2 | 570.8 | 246.2 | 20.6 | 2.3 | 20.2 | 1.2 |
| 0.14665 | 0.66659 | -2177.9 | 163.3 | 4253.8 | 22498.0 | -1657.7 | 172.4 | 2819.6 | 1053.1 | 112.9 | 11.1 | 113.8 | 6.7 | -416.3 | 31.3 | 806.1 | 4243.3 | -316.5 | 33.0 | 534.9 | 199.0 | 21.5 | 2.1 | 21.7 | 1.3 |
| 0.1495 | 0.67955 | -2157.4 | 161.0 | 4154.1 | 2194.6 | -1590.0 | 180.6 | 2674.6 | 861.7 | 119.0 | 10.7 | 122.9 | 7.1 | -404.9 | 30.3 | 772.8 | 406.4 | -297.9 | 33.9 | 498.1 | 159.9 | 22.2 | 2.0 | 23.0 | 1.3 |
| 0.15235 | 0.6925 | -2130.6 | 160.3 | 4040.3 | 1100.9 | -1514.6 | 189.2 | 2521.9 | 714.6 | 124.5 | 10.5 | 131.4 | 7.6 | -392.7 | 29.6 | 738.3 | 200.2 | -278.7 | 34.9 | 461.3 | 130.2 | 22.9 | 1.9 | 24.1 | 1.4 |
| 0.1552 | 0.70545 | -2097.8 | 161.1 | 3915.0 | 743.7 | -1433.4 | 197.5 | 2365.8 | 606.7 | 129.4 | 10.3 | 139.1 | 8.0 | -380.0 | 29.2 | 702.9 | 132.9 | -259.1 | 35.8 | 425.2 | 108.6 | 23.3 | 1.9 | 25.1 | 1.4 |
| 0.15805 | 0.71841 | -2059.7 | 162.7 | 3780.7 | 581.8 | -1348.5 | 205.0 | 2210.4 | 528.3 | 133.7 | 10.2 | 146.0 | 8.5 | -366.8 | 29.0 | 667.3 | 102.3 | -239.5 | 36.5 | 390.5 | 93.0 | 23.7 | 1.8 | 25.9 | 1.5 |
| 0.1609 | 0.73136 | -2016.7 | 164.5 | 3640.1 | 495.0 | -1262.0 | 210.9 | 2059.3 | 470.5 | 137.3 | 10.2 | 152.1 | 8.9 | -353.2 | 28.9 | 631.8 | 85.6 | -220.4 | 36.9 | 357.8 | 81.4 | 23.9 | 1.8 | 26.5 | 1.5 |
| 0.1625 | 0.73864 | | | | | | | | | | | | | | | | | | | | | | | | |
| 0.16375 | 0.74432 | -1969.5 | 166.0 | 3495.5 | 441.6 | -1175.9 | 214.7 | 1915.4 | 426.0 | 140.3 | 10.1 | 157.3 | 9.3 | -339.3 | 28.6 | 596.9 | 75.1 | -202.0 | 36.9 | 327.4 | 72.6 | 24.1 | 1.7 | 26.9 | 1.6 |
| 0.1666 | 0.75727 | -1918.8 | 166.4 | 3349.1 | 403.9 | -1092.5 | 216.0 | 1781.0 | 390.0 | 142.6 | 10.1 | 161.5 | 9.6 | -325.4 | 28.3 | 562.9 | 67.6 | -184.7 | 36.5 | 299.6 | 65.4 | 24.1 | 1.7 | 27.2 | 1.6 |
| 0.16945 | 0.77023 | -1865.2 | 165.5 | 3202.8 | 373.6 | -1013.7 | 214.6 | 1658.1 | 359.1 | 144.3 | 10.1 | 164.8 | 9.9 | -311.4 | 27.7 | 530.0 | 61.6 | -168.7 | 35.7 | 274.6 | 59.3 | 24.0 | 1.7 | 27.4 | 1.6 |
| 0.1723 | 0.78318 | -1809.4 | 162.9 | 3058.1 | 346.8 | -941.1 | 210.4 | 1547.7 | 331.3 | 145.5 | 10.0 | 167.2 | 10.0 | -297.6 | 26.8 | 498.5 | 56.3 | -154.2 | 34.5 | 252.5 | 53.9 | 23.8 | 1.6 | 27.3 | 1.6 |
| 0.17515 | 0.79614 | -1752.2 | 158.6 | 2916.4 | 321.3 | -876.2 | 203.5 | 1450.8 | 305.3 | 146.1 | 9.9 | 168.8 | 10.1 | -284.0 | 25.8 | 468.5 | 51.4 | -141.5 | 32.9 | 233.2 | 48.9 | 23.6 | 1.6 | 27.2 | 1.6 |
| 0.178 | 0.80909 | -1694.1 | 152.5 | 2778.8 | 296.2 | -820.0 | 194.2 | 1367.5 | 280.3 | 146.3 | 9.8 | 169.6 | 10.1 | -270.7 | 24.4 | 440.0 | 46.7 | -130.5 | 30.9 | 216.7 | 44.3 | 23.3 | 1.6 | 26.9 | 1.6 |
| 0.18085 | 0.82205 | -1635.7 | 144.8 | 2646.0 | 271.3 | -773.2 | 182.8 | 1298.0 | 256.1 | 146.0 | 9.7 | 169.8 | 10.1 | -257.7 | 22.9 | 413.2 | 42.2 | -121.3 | 28.7 | 202.8 | 39.9 | 22.9 | 1.5 | 26.6 | 1.6 |
| 0.1837 | 0.835 | -1577.4 | 135.7 | 2518.6 | 246.4 | -736.2 | 169.7 | 1242.0 | 232.3 | 145.3 | 9.5 | 169.3 | 10.0 | -245.2 | 21.2 | 388.0 | 37.8 | -114.0 | 26.3 | 191.4 | 35.7 | 22.5 | 1.5 | 26.1 | 1.5 |
| 0.18655 | 0.84795 | -1519.9 | 125.6 | 2396.9 | 221.7 | -709.2 | 155.4 | 1199.0 | 209.2 | 144.3 | 9.3 | 168.2 | 9.8 | -233.2 | 19.3 | 364.4 | 33.6 | -108.3 | 23.8 | 182.4 | 31.7 | 22.0 | 1.4 | 25.6 | 1.5 |
| 0.1894 | 0.86091 | -1463.5 | 114.7 | 2281.1 | 197.7 | -692.2 | 140.5 | 1168.3 | 186.8 | 142.8 | 9.2 | 166.5 | 9.5 | -221.7 | 17.4 | 342.4 | 29.5 | -104.4 | 21.2 | 175.4 | 28.0 | 21.5 | 1.4 | 25.1 | 1.4 |
| 0.19225 | 0.87386 | -1408.7 | 103.8 | 2171.1 | 174.9 | -685.0 | 125.7 | 1149.1 | 165.6 | 140.9 | 9.3 | 164.1 | 9.4 | -210.7 | 15.6 | 321.8 | 25.8 | -102.0 | 18.7 | 170.4 | 24.5 | 21.0 | 1.4 | 24.4 | 1.4 |
| 0.1951 | 0.88682 | -1356.2 | 93.7 | 2066.2 | 154.2 | -687.7 | 111.9 | 1140.0 | 146.3 | 138.0 | 11.7 | 160.5 | 11.6 | -200.4 | 13.9 | 302.6 | 22.5 | -101.2 | 16.5 | 167.0 | 21.4 | 20.3 | 1.7 | 23.6 | 1.7 |
| 0.19795 | 0.89977 | -1307.5 | 85.4 | 1965.0 | 137.3 | -701.2 | 100.2 | 1138.6 | 130.1 | 132.8 | 23.8 | 154.2 | 24.1 | -191.0 | 12.5 | 284.4 | 19.8 | -102.0 | 14.6 | 164.8 | 18.8 | 19.3 | 3.5 | 22.4 | 3.5 |
| 0.2008 | 0.91273 | -1266.0 | 80.4 | 1863.0 | 126.0 | -729.0 | 92.1 | 1139.6 | 118.6 | 121.6 | 65.3 | 140.9 | 67.6 | -182.8 | 11.7 | 266.6 | 17.9 | -104.9 | 13.3 | 163.1 | 16.9 | 17.5 | 9.4 | 20.2 | 9.7 |
| 0.20365 | 0.92568 | -1242.2 | 79.8 | 1747.9 | 122.4 | -783.0 | 88.8 | 1129.3 | 114.0 | 93.4 | 245.9 | 108.0 | 255.9 | -177.4 | 11.4 | 247.4 | 17.2 | -111.4 | 12.7 | 159.8 | 16.1 | 13.3 | 35.0 | 15.3 | 36.3 |
| 0.2065 | 0.93864 | | | | | | | | | | | | | | | | | | | | | | | | |
| 0.22 | 1 | | | | | | | | | | | | | | | | | | | | | | | | |
| 0.24565 | | | | | | | | | | | | | | | | | | | | | | | | | |

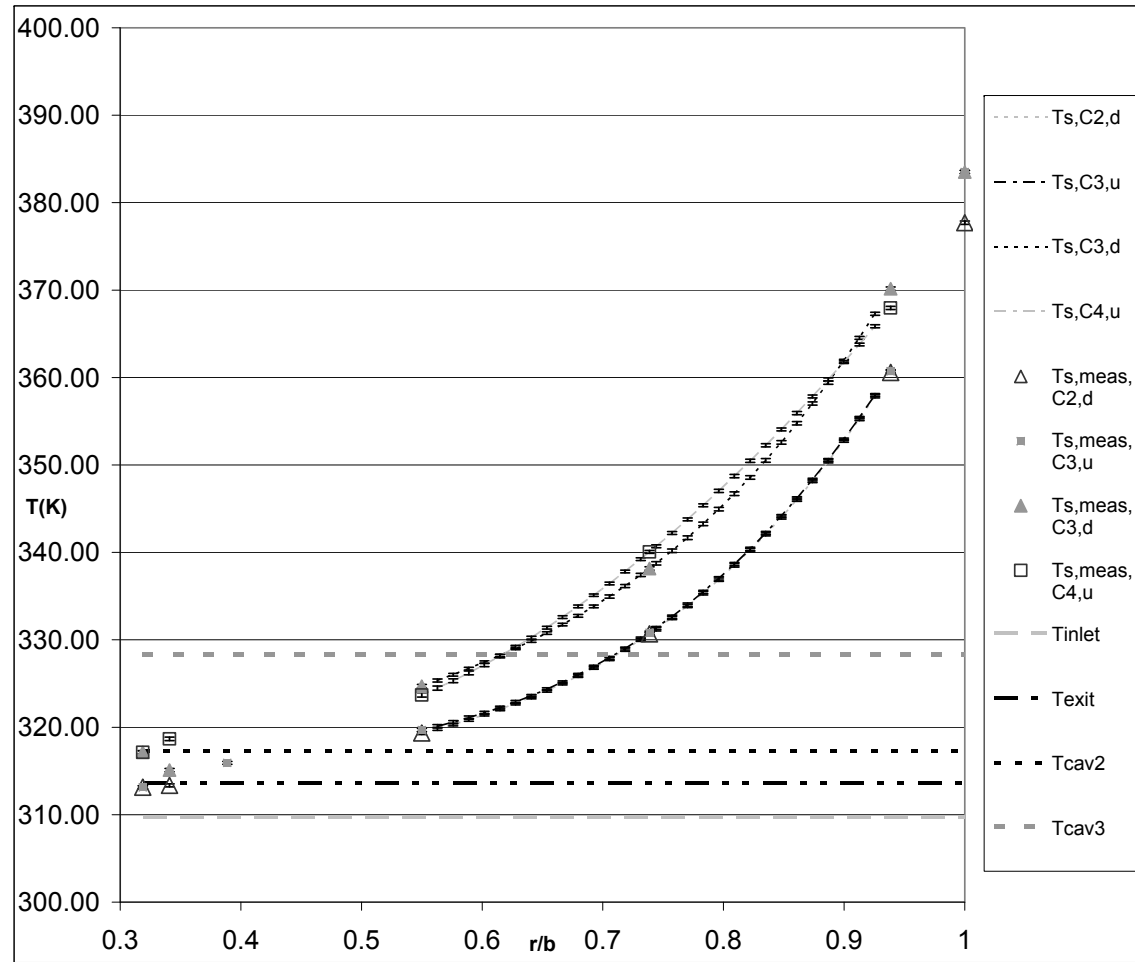
MULTIPLE CAVITY RIG

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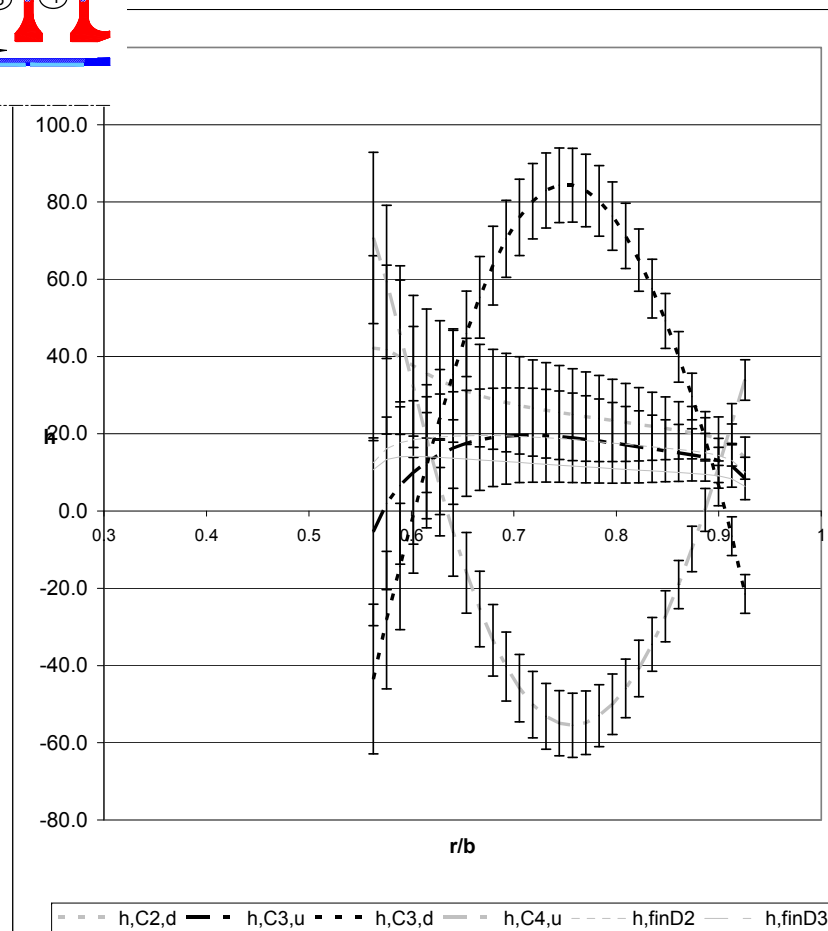
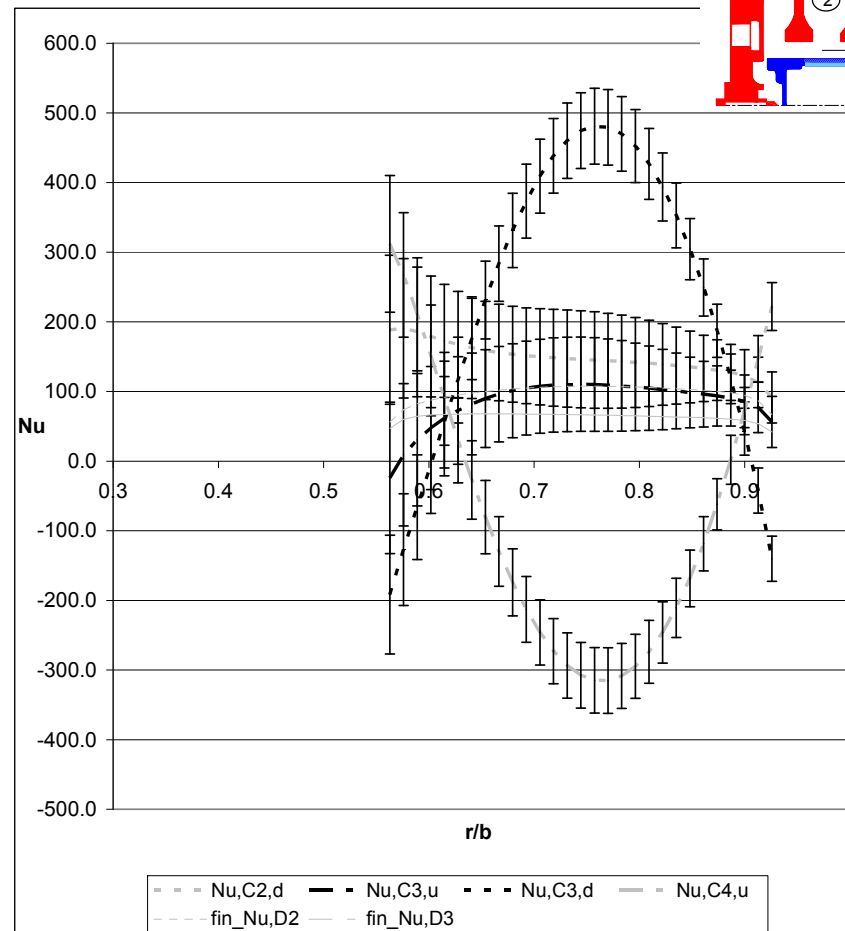
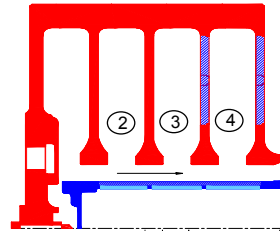
| | | | |
|------------------|--------------------|------------------|--------------------|
| Rez: | 9.18E+04 ± 5.5E+01 | ReΦ: | 4.92E+05 ± 6.9E+02 |
| Ro: | 3.56 ± 2.9E-03 | RotSpeed (rpm): | 665.6 ± 1.7E-02 |
| Mflow (kg/s): | 0.333 | Pcav (abs Pa): | 2.453E+05 |
| Bo*: | 0.084 ± 4.5E-04 | Pr: | 0.707 ± 3.7E-05 |
| Tinlet (K): | 309.75 ± 2.4E-01 | Texit(K): | 313.60 ± 2.6E-01 |
| Tcav2 (K): | 317.27 ± 1.3E+00 | Tcav3 (K): | 328.33 ± 1.0E+00 |
| ΔTmeas (K): | 3.85 ± 3.3E-01 | ΔTcalc (K): | 3.42 ± 6.3E-02 |
| βΔTav: | 0.089 ± 8.3E-04 | finβΔT av: | 0.089 ± 8.3E-04 |
| Ø: | 0.404 ± 5.5E-02 | finØ: | 0.283 ± 3.3E-02 |
| Ts,C2, d,av(K): | 333.50 ± 1.3E-01 | Ts,C3, u,av(K): | 333.65 ± 1.3E-01 |
| Ts,C3, d,av(K): | 340.66 ± 1.4E-01 | Ts,C4, u,av(K): | 341.55 ± 1.2E-01 |
| finTs, D2,av(K): | 333.57 ± 9.3E-02 | finTS, D3,av(K): | 341.11 ± 9.2E-02 |
| Nu,C2, d,av: | 166.7 ± 5.6E+01 | Nu,C3, u,av: | 108.7 ± 5.5E+01 |
| Nu,C3, d,av: | 299.5 ± 4.5E+01 | Nu,C4, u,av: | -136.4 ± 4.2E+01 |
| finNu, D2,av: | 117.8 ± 7.4E+00 | finNu, D3,av: | 75.6 ± 5.8E+00 |
| Tsh,C2 (K): | 377.70 ± 2.0E-01 | Tsh,C3 (K): | 383.50 ± 1.5E-01 |
| Nush,C2: | 39.0 ± 1.1E+00 | Nush,C3: | 31.8 ± 7.5E-01 |
| Grsh,C2: | 4.89E+07 ± 3.8E+05 | Grsh,C3: | 5.30E+07 ± 3.8E+05 |
| θ2: | 1.12 ± 2.7E-02 | θ3: | 1.34 ± 2.6E-02 |



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| Radius(m) | non-dim radius | Tsh/cob D2 | ± | Tsh/cob D3 | ± | TC,C2, dn (K) | ± | TC,C3, up (K) | ± | TC,C3, dn (K) | ± | TC,C4, up (K) | ± | Ts,C2, dn (K) | ± | Ts,C3, up (K) | ± | Ts,C3, dn (K) | ± | Ts,C4, up (K) | ± | finTs, D2 (K) | ± | finTs, D3 (K) | ± |
|-----------|-------------------|------------|------|------------|------|------------------|------|------------------|------|------------------|------|------------------|------|------------------|------|------------------|------|------------------|------|------------------|------|------------------|------|---------------|------|
| 0.0701 | 0.31864 | 313.15 | 0.17 | 317.14 | 0.18 | | | | | | | | | | | | | | | | | | | | |
| 0.075 | 0.34091 | | | | | 313.36 | 0.17 | | | 315.11 | 0.18 | 318.68 | 0.20 | | | | | | | | | | | | |
| 0.0855 | 0.38864 | | | | | | | 315.92 | 0.17 | | | | | | | | | | | | | | | | |
| 0.121 | 0.55 | | | | | 319.35 | 0.17 | 319.68 | 0.19 | 324.70 | 0.18 | 323.70 | 0.24 | | | | | | | | | | | | |
| 0.12385 | 0.56295 | | | | | | | | | | | | | 319.82 | 0.17 | 320.12 | 0.19 | 325.34 | 0.17 | 324.49 | 0.23 | 319.97 | 0.13 | 324.91 | 0.14 |
| 0.1267 | 0.57591 | | | | | | | | | | | | | 320.32 | 0.16 | 320.59 | 0.18 | 326.01 | 0.17 | 325.32 | 0.22 | 320.45 | 0.13 | 325.66 | 0.14 |
| 0.12955 | 0.58886 | | | | | | | | | | | | | 320.85 | 0.16 | 321.10 | 0.18 | 326.71 | 0.16 | 326.21 | 0.21 | 320.98 | 0.13 | 326.46 | 0.13 |
| 0.1324 | 0.60182 | | | | | | | | | | | | | 321.43 | 0.15 | 321.65 | 0.17 | 327.44 | 0.15 | 327.14 | 0.20 | 321.54 | 0.12 | 327.29 | 0.12 |
| 0.13525 | 0.61477 | | | | | | | | | | | | | 322.05 | 0.15 | 322.25 | 0.17 | 328.21 | 0.15 | 328.13 | 0.19 | 322.15 | 0.12 | 328.17 | 0.12 |
| 0.1381 | 0.62773 | | | | | | | | | | | | | 322.71 | 0.15 | 322.90 | 0.16 | 329.02 | 0.15 | 329.17 | 0.18 | 322.80 | 0.12 | 329.09 | 0.11 |
| 0.14095 | 0.64068 | | | | | | | | | | | | | 323.42 | 0.15 | 323.59 | 0.16 | 329.87 | 0.14 | 330.26 | 0.17 | 323.50 | 0.12 | 330.07 | 0.11 |
| 0.1438 | 0.65364 | | | | | | | | | | | | | 324.18 | 0.15 | 324.34 | 0.16 | 330.78 | 0.15 | 331.40 | 0.16 | 324.26 | 0.12 | 331.09 | 0.11 |
| 0.14665 | 0.66659 | | | | | | | | | | | | | 324.99 | 0.15 | 325.15 | 0.16 | 331.73 | 0.15 | 332.58 | 0.16 | 325.07 | 0.12 | 332.16 | 0.11 |
| 0.1495 | 0.67955 | | | | | | | | | | | | | 325.86 | 0.16 | 326.01 | 0.16 | 332.74 | 0.16 | 333.82 | 0.15 | 325.94 | 0.12 | 333.28 | 0.11 |
| 0.15235 | 0.6925 | | | | | | | | | | | | | 326.80 | 0.16 | 326.94 | 0.16 | 333.82 | 0.16 | 335.10 | 0.15 | 326.87 | 0.12 | 334.46 | 0.11 |
| 0.1552 | 0.70545 | | | | | | | | | | | | | 327.80 | 0.17 | 327.94 | 0.16 | 334.95 | 0.17 | 336.43 | 0.15 | 327.87 | 0.12 | 335.69 | 0.11 |
| 0.15805 | 0.71841 | | | | | | | | | | | | | 328.86 | 0.17 | 329.00 | 0.17 | 336.15 | 0.18 | 337.81 | 0.15 | 328.93 | 0.12 | 336.98 | 0.12 |
| 0.1609 | 0.73136 | | | | | | | | | | | | | 330.00 | 0.18 | 330.14 | 0.17 | 337.42 | 0.19 | 339.23 | 0.16 | 330.07 | 0.13 | 338.33 | 0.12 |
| 0.1625 | 0.73864 | | | | | 330.67 | 0.18 | 330.81 | 0.17 | 338.16 | 0.19 | 340.05 | 0.16 | | | | | | | | | | | | |
| 0.16375 | 0.74432 | | | | | | | | | | | | | 331.21 | 0.18 | 331.35 | 0.18 | 338.76 | 0.19 | 340.70 | 0.16 | 331.28 | 0.13 | 339.73 | 0.13 |
| 0.1666 | 0.75727 | | | | | | | | | | | | | 332.50 | 0.19 | 332.65 | 0.18 | 340.18 | 0.20 | 342.22 | 0.17 | 332.58 | 0.13 | 341.20 | 0.13 |
| 0.16945 | 0.77023 | | | | | | | | | | | | | 333.88 | 0.19 | 334.02 | 0.18 | 341.68 | 0.20 | 343.78 | 0.17 | 333.95 | 0.13 | 342.73 | 0.13 |
| 0.1723 | 0.78318 | | | | | | | | | | | | | 335.33 | 0.20 | 335.48 | 0.19 | 343.27 | 0.21 | 345.38 | 0.18 | 335.40 | 0.14 | 344.33 | 0.14 |
| 0.17515 | 0.79614 | | | | | | | | | | | | | 336.87 | 0.20 | 337.02 | 0.19 | 344.94 | 0.21 | 347.03 | 0.18 | 336.95 | 0.14 | 345.99 | 0.14 |
| 0.178 | 0.80909 | | | | | | | | | | | | | 338.51 | 0.19 | 338.66 | 0.19 | 346.70 | 0.21 | 348.73 | 0.18 | 338.58 | 0.13 | 347.72 | 0.14 |
| 0.18085 | 0.82205 | | | | | | | | | | | | | 340.24 | 0.19 | 340.39 | 0.19 | 348.56 | 0.21 | 350.46 | 0.18 | 340.31 | 0.13 | 349.51 | 0.14 |
| 0.1837 | 0.835 | | | | | | | | | | | | | 342.06 | 0.19 | 342.22 | 0.18 | 350.52 | 0.20 | 352.24 | 0.18 | 342.14 | 0.13 | 351.38 | 0.14 |
| 0.18655 | 0.84795 | | | | | | | | | | | | | 343.99 | 0.18 | 344.15 | 0.18 | 352.58 | 0.20 | 354.06 | 0.18 | 344.07 | 0.12 | 353.32 | 0.13 |
| 0.1894 | 0.86091 | | | | | | | | | | | | | 346.02 | 0.18 | 346.18 | 0.17 | 354.75 | 0.19 | 355.93 | 0.17 | 346.10 | 0.12 | 355.34 | 0.13 |
| 0.19225 | 0.87386 | | | | | | | | | | | | | 348.16 | 0.17 | 348.32 | 0.16 | 357.02 | 0.18 | 357.83 | 0.17 | 348.24 | 0.11 | 357.43 | 0.12 |
| 0.1951 | 0.88682 | | | | | | | | | | | | | 350.41 | 0.16 | 350.57 | 0.15 | 359.41 | 0.18 | 359.78 | 0.16 | 350.49 | 0.11 | 359.59 | 0.12 |
| 0.19795 | 0.89977 | | | | | | | | | | | | | 352.77 | 0.16 | 352.93 | 0.15 | 361.92 | 0.17 | 361.76 | 0.16 | 352.85 | 0.11 | 361.84 | 0.11 |
| 0.2008 | 0.91273 | | | | | | | | | | | | | 355.25 | 0.17 | 355.41 | 0.15 | 364.54 | 0.17 | 363.79 | 0.16 | 355.33 | 0.11 | 364.16 | 0.11 |
| 0.20365 | 0.92568 | | | | | | | | | | | | | 357.85 | 0.18 | 358.01 | 0.15 | 367.29 | 0.18 | 365.85 | 0.17 | 357.93 | 0.12 | 366.57 | 0.12 |
| 0.2065 | 0.93864 | | | | | 360.57 | 0.21 | 360.73 | 0.17 | 370.17 | 0.20 | 367.95 | 0.18 | | | | | | | | | | | | |
| 0.22 | 1 | 377.70 | 0.20 | 383.50 | 0.18 | | | | | | | | | | | | | | | | | | | | |
| 0.24565 | | 388.15 | 0.17 | 392.76 | 0.17 | | | | | | | | | | | | | | | | | | | | |

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| non-dim radius | | Nu_r,C2,d ± | | Nu_r,C3,u ± | | Nu_r,C3,d ± | | Nu_r,C4,u ± | | finNu_r,D2 ± | | finNu_r,D3 ± | | h_r,C2,d ± | | h_r,C3,u ± | | h_r,C3,d ± | | h_r,C4,u ± | | fin-h,D2 ± | | fin-h,D3 ± | |
|-------------------|---------|-------------|-------|-------------|-------|-------------|------|-------------|------|--------------|-------|--------------|-------|------------|------|------------|------|------------|------|------------|------|------------|------|------------|------|
| 0.0701 | 0.31864 | | | | | | | | | | | | | | | | | | | | | | | | |
| 0.075 | 0.34091 | | | | | | | | | | | | | | | | | | | | | | | | |
| 0.0855 | 0.38864 | | | | | | | | | | | | | | | | | | | | | | | | |
| 0.121 | 0.55 | | | | | | | | | | | | | | | | | | | | | | | | |
| 0.12385 | 0.56295 | 188.5 | 107.0 | -24.0 | 108.8 | -191.6 | 85.3 | 312.0 | 98.1 | 55.9 | 256.8 | 47.1 | 202.4 | 42.2 | 23.9 | -5.4 | 24.3 | -43.5 | 19.4 | 70.7 | 22.2 | 12.5 | 57.4 | 10.7 | 45.9 |
| 0.1267 | 0.57591 | 190.9 | 100.0 | 9.1 | 102.1 | -127.0 | 80.2 | 267.3 | 89.5 | 73.9 | 63.1 | 60.0 | 49.4 | 41.8 | 21.9 | 2.0 | 22.3 | -28.2 | 17.8 | 59.3 | 19.8 | 16.2 | 13.8 | 13.3 | 11.0 |
| 0.12955 | 0.58886 | 185.6 | 93.1 | 30.8 | 95.1 | -66.1 | 75.1 | 210.8 | 81.2 | 82.4 | 21.1 | 64.4 | 16.2 | 39.8 | 20.0 | 6.6 | 20.4 | -14.4 | 16.3 | 45.9 | 17.6 | 17.7 | 4.5 | 14.0 | 3.5 |
| 0.1324 | 0.60182 | 179.0 | 86.8 | 47.5 | 88.4 | -5.2 | 70.1 | 150.4 | 73.6 | 87.6 | 8.4 | 66.2 | 5.9 | 37.6 | 18.2 | 10.0 | 18.6 | -1.1 | 15.0 | 32.1 | 15.7 | 18.4 | 1.8 | 14.1 | 1.3 |
| 0.13525 | 0.61477 | 172.7 | 81.2 | 61.1 | 82.3 | 55.9 | 65.6 | 89.5 | 66.8 | 91.6 | 4.5 | 67.0 | 2.6 | 35.6 | 16.7 | 12.6 | 17.0 | 11.7 | 13.7 | 18.7 | 14.0 | 18.9 | 0.9 | 14.0 | 0.6 |
| 0.1381 | 0.62773 | 167.2 | 76.5 | 72.6 | 77.2 | 116.4 | 61.6 | 29.9 | 61.0 | 94.8 | 3.3 | 67.4 | 1.6 | 33.8 | 15.5 | 14.7 | 15.6 | 23.9 | 12.7 | 6.2 | 12.5 | 19.2 | 0.7 | 13.9 | 0.3 |
| 0.14095 | 0.64068 | 162.7 | 73.0 | 82.1 | 73.2 | 175.3 | 58.3 | -27.1 | 56.2 | 97.6 | 2.8 | 67.7 | 1.2 | 32.3 | 14.5 | 16.3 | 14.5 | 35.4 | 11.8 | -5.5 | 11.4 | 19.4 | 0.6 | 13.7 | 0.2 |
| 0.1438 | 0.65364 | 159.0 | 70.6 | 89.9 | 70.3 | 231.4 | 55.8 | -80.5 | 52.6 | 100.0 | 2.6 | 67.8 | 1.1 | 31.0 | 13.8 | 17.5 | 13.7 | 45.9 | 11.1 | -16.0 | 10.4 | 19.5 | 0.5 | 13.5 | 0.2 |
| 0.14665 | 0.66659 | 155.9 | 69.3 | 96.2 | 68.5 | 283.7 | 54.2 | -129.7 | 49.9 | 102.0 | 2.4 | 67.8 | 1.0 | 29.9 | 13.3 | 18.4 | 13.1 | 55.3 | 10.5 | -25.3 | 9.8 | 19.5 | 0.5 | 13.2 | 0.2 |
| 0.1495 | 0.67955 | 153.5 | 68.7 | 101.2 | 67.5 | 331.3 | 53.3 | -174.0 | 48.2 | 103.7 | 2.3 | 67.8 | 1.0 | 28.9 | 12.9 | 19.1 | 12.7 | 63.5 | 10.2 | -33.5 | 9.3 | 19.5 | 0.4 | 13.0 | 0.2 |
| 0.15235 | 0.6925 | 151.5 | 68.8 | 104.9 | 67.2 | 373.4 | 53.0 | -212.8 | 47.2 | 105.0 | 2.3 | 67.6 | 1.0 | 28.1 | 12.7 | 19.4 | 12.5 | 70.5 | 10.0 | -40.3 | 8.9 | 19.5 | 0.4 | 12.8 | 0.2 |
| 0.15525 | 0.70545 | 149.9 | 69.1 | 107.5 | 67.4 | 409.3 | 52.2 | -245.9 | 46.8 | 106.1 | 2.2 | 67.5 | 1.1 | 27.3 | 12.6 | 19.6 | 12.3 | 76.0 | 9.9 | -45.9 | 8.7 | 19.3 | 0.4 | 12.6 | 0.2 |
| 0.15805 | 0.71841 | 148.5 | 69.5 | 109.2 | 67.6 | 438.4 | 53.6 | -272.9 | 46.7 | 106.8 | 2.3 | 67.2 | 1.2 | 26.7 | 12.5 | 19.6 | 12.1 | 80.2 | 9.8 | -50.1 | 8.6 | 19.2 | 0.4 | 12.3 | 0.2 |
| 0.1609 | 0.73136 | 147.3 | 69.8 | 110.2 | 67.8 | 460.2 | 54.1 | -293.5 | 46.9 | 107.3 | 2.3 | 67.0 | 1.2 | 26.1 | 12.3 | 19.5 | 12.0 | 83.0 | 9.7 | -53.2 | 8.5 | 19.0 | 0.4 | 12.1 | 0.2 |
| 0.1625 | 0.73864 | | | | | | | | | | | | | | | | | | | | | | | | |
| 0.16375 | 0.74432 | 146.2 | 69.7 | 110.4 | 67.7 | 474.5 | 54.4 | -307.5 | 47.0 | 107.5 | 2.3 | 66.6 | 1.3 | 25.5 | 12.1 | 19.3 | 11.8 | 84.4 | 9.6 | -54.9 | 8.4 | 18.8 | 0.4 | 11.9 | 0.2 |
| 0.1666 | 0.75727 | 145.2 | 69.2 | 110.1 | 67.3 | 481.0 | 54.5 | -314.7 | 47.1 | 107.5 | 2.4 | 66.3 | 1.4 | 25.0 | 11.9 | 18.9 | 11.6 | 84.3 | 9.5 | -55.5 | 8.3 | 18.5 | 0.4 | 11.7 | 0.2 |
| 0.16945 | 0.77023 | 144.1 | 68.8 | 109.3 | 66.4 | 479.4 | 54.3 | -315.1 | 47.1 | 107.3 | 2.4 | 65.9 | 1.5 | 24.4 | 11.5 | 18.5 | 11.3 | 83.0 | 9.4 | -54.8 | 8.2 | 18.2 | 0.4 | 11.4 | 0.3 |
| 0.1723 | 0.78318 | 143.0 | 66.6 | 108.1 | 65.0 | 469.9 | 53.6 | -308.4 | 46.7 | 106.9 | 2.5 | 65.4 | 1.6 | 23.9 | 11.1 | 18.1 | 10.9 | 80.3 | 9.1 | -53.0 | 8.0 | 17.9 | 0.4 | 11.2 | 0.3 |
| 0.17515 | 0.79614 | 141.7 | 64.5 | 106.6 | 63.0 | 452.3 | 52.5 | -294.7 | 46.1 | 106.4 | 2.5 | 65.0 | 1.7 | 23.4 | 10.7 | 17.6 | 10.4 | 76.4 | 8.8 | -50.0 | 7.8 | 17.6 | 0.4 | 11.0 | 0.3 |
| 0.178 | 0.80909 | 140.4 | 61.9 | 104.8 | 60.6 | 426.8 | 50.9 | -273.9 | 45.2 | 105.7 | 2.5 | 64.5 | 1.7 | 22.9 | 10.1 | 17.1 | 9.9 | 71.2 | 8.5 | -45.9 | 7.6 | 17.3 | 0.4 | 10.8 | 0.3 |
| 0.18085 | 0.82205 | 138.9 | 58.8 | 102.9 | 57.7 | 393.6 | 48.9 | -245.9 | 44.0 | 104.9 | 2.5 | 64.0 | 1.8 | 22.4 | 9.5 | 16.6 | 9.3 | 64.9 | 8.0 | -40.8 | 7.3 | 17.0 | 0.4 | 10.6 | 0.3 |
| 0.1837 | 0.835 | 137.2 | 55.3 | 100.8 | 54.4 | 352.8 | 46.6 | -210.7 | 42.5 | 104.0 | 2.6 | 63.4 | 1.8 | 21.9 | 8.8 | 16.1 | 8.7 | 57.6 | 7.6 | -34.5 | 7.0 | 16.6 | 0.4 | 10.4 | 0.3 |
| 0.18655 | 0.84795 | 135.3 | 51.5 | 98.6 | 50.8 | 304.6 | 43.9 | -168.3 | 40.7 | 102.9 | 2.6 | 62.9 | 1.9 | 21.4 | 8.1 | 15.6 | 8.0 | 49.2 | 7.1 | -27.3 | 6.6 | 16.3 | 0.4 | 10.2 | 0.3 |
| 0.1894 | 0.86091 | 133.2 | 47.5 | 96.2 | 47.0 | 249.3 | 41.1 | -118.7 | 38.8 | 101.7 | 2.6 | 62.2 | 1.9 | 20.9 | 7.4 | 15.1 | 7.4 | 39.9 | 6.6 | -19.0 | 6.2 | 15.9 | 0.4 | 10.0 | 0.3 |
| 0.19225 | 0.87386 | 130.7 | 43.7 | 93.6 | 43.4 | 187.2 | 38.2 | -61.9 | 36.9 | 100.3 | 2.8 | 61.5 | 1.9 | 20.3 | 6.8 | 14.5 | 6.7 | 29.7 | 6.1 | -9.8 | 5.9 | 15.6 | 0.4 | 9.8 | 0.3 |
| 0.1951 | 0.88682 | 127.4 | 40.2 | 90.5 | 40.1 | 118.3 | 35.6 | 1.8 | 35.2 | 98.2 | 3.9 | 60.4 | 2.6 | 19.6 | 6.2 | 13.9 | 6.2 | 18.6 | 5.6 | 0.3 | 5.5 | 15.1 | 0.6 | 9.5 | 0.4 |
| 0.19795 | 0.89977 | 122.5 | 37.6 | 85.9 | 37.6 | 42.3 | 33.6 | 71.9 | 33.9 | 94.7 | 8.8 | 58.4 | 6.6 | 18.7 | 5.7 | 13.1 | 5.7 | 6.6 | 5.2 | 11.2 | 5.3 | 14.4 | 1.3 | 9.1 | 1.0 |
| 0.2008 | 0.91273 | 113.2 | 36.2 | 77.1 | 36.3 | -42.2 | 32.4 | 146.9 | 33.5 | 86.9 | 24.7 | 53.8 | 20.2 | 17.1 | 5.5 | 11.7 | 5.5 | -6.5 | 5.0 | 22.6 | 5.2 | 13.1 | 3.7 | 8.3 | 3.1 |
| 0.20365 | 0.92568 | 91.5 | 36.5 | 56.3 | 36.7 | -140.2 | 32.5 | 222.0 | 34.4 | 66.9 | 93.9 | 41.7 | 79.5 | 13.7 | 5.5 | 8.4 | 5.5 | -21.5 | 5.0 | 33.9 | 5.2 | 10.0 | 14.1 | 6.4 | 12.2 |
| 0.2065 | 0.93864 | | | | | | | | | | | | | | | | | | | | | | | | |
| 0.22 | 1 | | | | | | | | | | | | | | | | | | | | | | | | |
| 0.24565 | | | | | | | | | | | | | | | | | | | | | | | | | |